

ADTF SPOKE CAVITY CRYOMODULE CONCEPT

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ABSTRACT

The Accelerator Driven Test Facility (ADTF) is being developed as a reactor concepts test bed for transmutation of nuclear waste. A 13.3 mA continuous-wave (CW) proton beam will be accelerated to 600 MeV and impinged on a spallation target. The subsequent neutron shower is used to create a nuclear reaction within a subcritical assembly of waste material that reduces the waste half-life from the order of 10^5 years to 10^2 years. Additionally, significant energy is produced that can be used to generate electrical power.

The ADTF proton accelerator consists of room-temperature (RT) structures that accelerate the beam to 6.7-MeV and superconducting (SC) elements that boost the beam's energy to 600-MeV. Traditional SC elliptical cavities experience structural difficulties at low energies due to their geometry. Therefore, stiff-structured SC spoke cavities have been adopted for the energy range between 6.7 and 109 MeV. Elliptical cavities are used at the higher energies. This paper describes a multi-spoke-cavity cryomodule concept for ADTF.

INTRODUCTION

The Accelerator Driven Test Facility (ADTF) is being developed as a reactor concepts test bed for transmutation of nuclear waste. A 13.3 mA CW proton beam will be accelerated to 600 MeV and impinged on a spallation target. Spallation neutrons are used to create a nuclear reaction within a subcritical assembly of waste material. The reaction produces significant usable energy and reaction products with greatly reduced half-lives.

The 600-MeV ADTF proton accelerator consists of a 6.7-MeV RT radio frequency quadrupole accelerating structure, and a 6.7 to 600-MeV superconducting (SC) accelerator. Geometry related structural difficulties limit the use of traditional elliptical SC cavities at low energies. Therefore, stiff-structured, 350 MHz SC spoke cavities [1] have been adopted for the energy range between 6.7 and 109 MeV [2], the Low Energy Linac (LEL). Elliptical cavities are used at the higher energies.

The beam dynamics design [2] dictates that cavities of three β types be used in the LEL: $\beta=0.175$ two-gap cavities [3], and $\beta=0.2$ and $\beta=0.34$ three-gap cavities, where β is the ratio of particle speed to the speed of light.

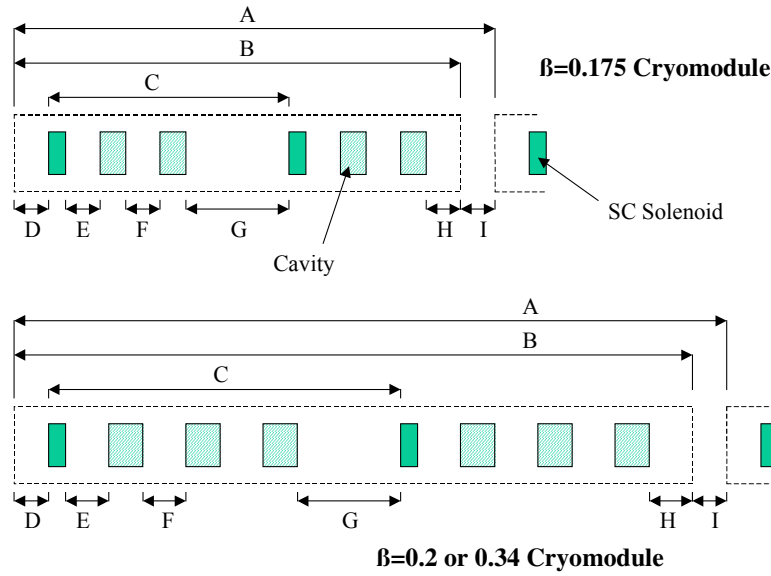


FIGURE 1. Layout schematic of the ADTF LEL Cryomodules. Letters refer to Table 1.

Fig. 1 is a schematic layout of the cryomodules. A $\beta=0.175$ lattice element consists of a solenoid magnet and two cavities. For the higher β structures, the lattice consists of a solenoid magnet and three cavities. Cryomodule lengths (Table 1) were dictated by:

- (1) The need to reduce LEL length and total system costs by minimizing distances between elements. This led to inclusion of the solenoids as SC elements in the cryomodule.
- (2) The need for periodic warm spaces for beam diagnostics,
- (3) The desire to maximize cryomodule lengths to minimize the total number of warm to cold transitions, reducing heat loads and distribution system complexity, and
- (4) The need to fit the module elements into the existing cleanroom [4].

This paper describes a cryomodule concept for the multi-spoke ADTF $\beta=0.34$ cavities. Efforts were focused on this unit since it was felt that its greater overall length coupled with the potential interface difficulties with the center cavities in the three cavity arrays (tuner, coupler, etc. - due to position) would make it the most challenging of the three units.

CONCEPTUAL DESIGN DEVELOPMENT

The basic requirements for the LEL cryomodules are given in Table 2. During cryomodule design development, the following basic guidelines were used:

- (1) Where possible, adopt concepts and components from previous programs.

TABLE 1. Dimensions of the three LEL cryomodules. Letters refer to Fig. 1.

		$\beta=0.175$	$\beta=0.2$	$\beta=0.34$
Cryomodule Period	A	4.53 m	6.10 m	6.92 m
Cryomodule Length	B	4.23 m	5.80 m	6.62 m
Focusing Period	C	2.26 m	3.05 m	3.46 m
Warm to Cold Transition (1)	D	0.39 m	0.39 m	0.39 m
Magnet to Cavity Drift	E	0.3 m	0.3 m	0.3 m
Cavity to Cavity Drift	F	0.3 m	0.3 m	0.3 m
Cavity to Magnet Drift	G	1.11 m	1.11 m	1.11 m
Warm to Cold Transition (2)	H	0.42 m	0.42 m	0.42 m
Cryomodule to Cryomodule Drift	I	0.3 m	0.3 m	0.3 m
Magnet Physical Length		0.15 m	0.15 m	0.15 m
Cavity Physical Length		0.20 m	0.30 m	0.43 m

TABLE 2. Cryomodule parameters.

	$\beta=0.175$	$\beta=0.2$	$\beta=0.34$
Q_o	1.72 E 9	1.92 E 9	2.50 E 9
E_{acc}	5 MV/m	5 MV/m	5 MV/m
Frequency	350 MHz	350 MHz	350 MHz
Coupled Power @ 13.3mA(100 mA)	4.7 (35.3) kW	10.35 (77.8) kW	18.6 (139.8) kW
Cavity Operating Temperature	4.5 K	4.5 K	4.5 K
Cavity External Magnetic Field (Goal)	5 milli-Gauss	5 milli-Gauss	5 milli-Gauss
Cavity Mechanical Resonance	> 200 Hz	> 200 Hz	> 200 Hz
Tuning Stiffness	~ 26 kHz/mil	To Be Determined	To Be Determined
	0.31 kHz/lb.		
Cavity Detuning Rate in < 300 msec	67 kHz/sec	To Be Determined	To Be Determined
	67 micron/sec		
Shield Operating Temperature	40-55 K	40-55 K	40-55 K
Solenoid Field	1.8-2.32 T	2.5-4.0 T	4.0-5.4 T
Current	20 A	20 A	20 A
Lead Type	Conduction Cooled	Conduction Cooled	Conduction Cooled

- (2) To minimize cleanroom time and simplify assembly, insert helium vessel assemblies axially into the vacuum vessel. Minimizing radial penetrations is a corollary.
- (3) Adopt design similarity between the three module types. With the exception of length, parts are identical.
- (4) Since ADTF has its roots in the Accelerator Production of Tritium Program (APT) [5],
 - (a) it must be upgradable to 100 mA operations for tritium production.
 - (b) the ADTF cryomodule must fit into the APT tunnel design.

Note that since the completion of this work, requirement (4)(b) has been eliminated.

The drive to minimize length led to the placement of the principle cryogenic and other penetrations in a Tee shaped section at the center of the module in the gap made available by the beam dynamics design (Section G in Fig. 1).

Fig. 2(a) shows a $\beta=0.175$ two-gap spoke cavity. Cavities must be tuned individually. Tuning requirements dictated that both end walls of a spoke cavity be flexed, resulting in a tuner assembly that straddles the cavity. If an individual cavity and tuner were housed in a helium vessel, the helium vessel size would increase, and require multiple penetrations. To minimize spacing between cavities, essentially flat helium vessel heads would be necessary, requiring thicker material or elaborate stiffeners. If multiple cavities with tuners were housed in a single vessel, the length would dictate elaborate penetrations to accommodate thermal contractions. Therefore, it was decided to house each cavity in an individual titanium helium vessel (Fig. 2(b)) with the tuner outside of the vessel (Fig. 3). A

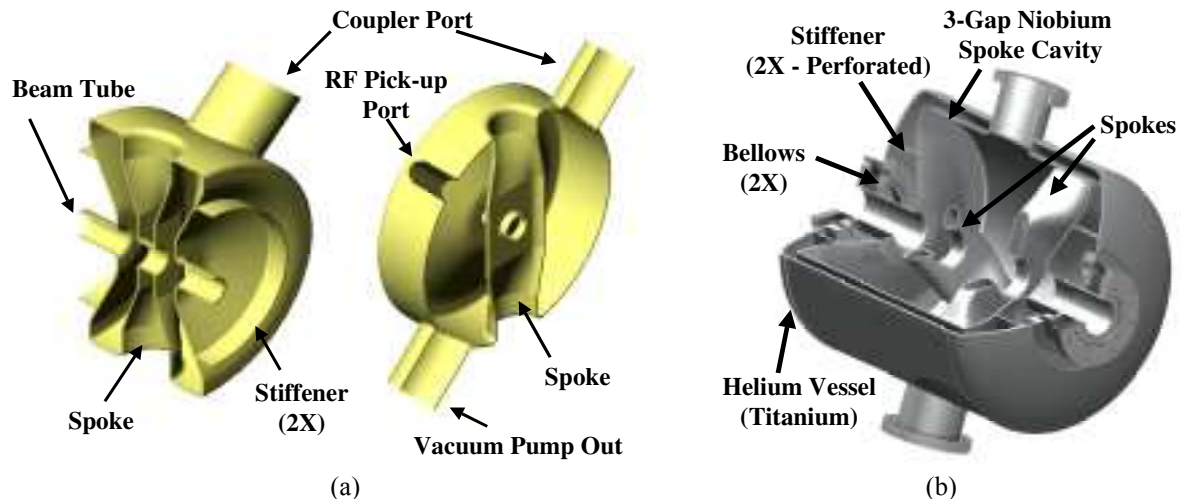


FIGURE 2. (a) A $\beta=0.175$ two-gap niobium spoke cavity. (b) A $\beta=0.34$ three-gap cavity and helium vessel.

TABLE 3. Estimated heat loads (Watts) for the $\beta=0.34$ Cryomodule. No margins are assigned.

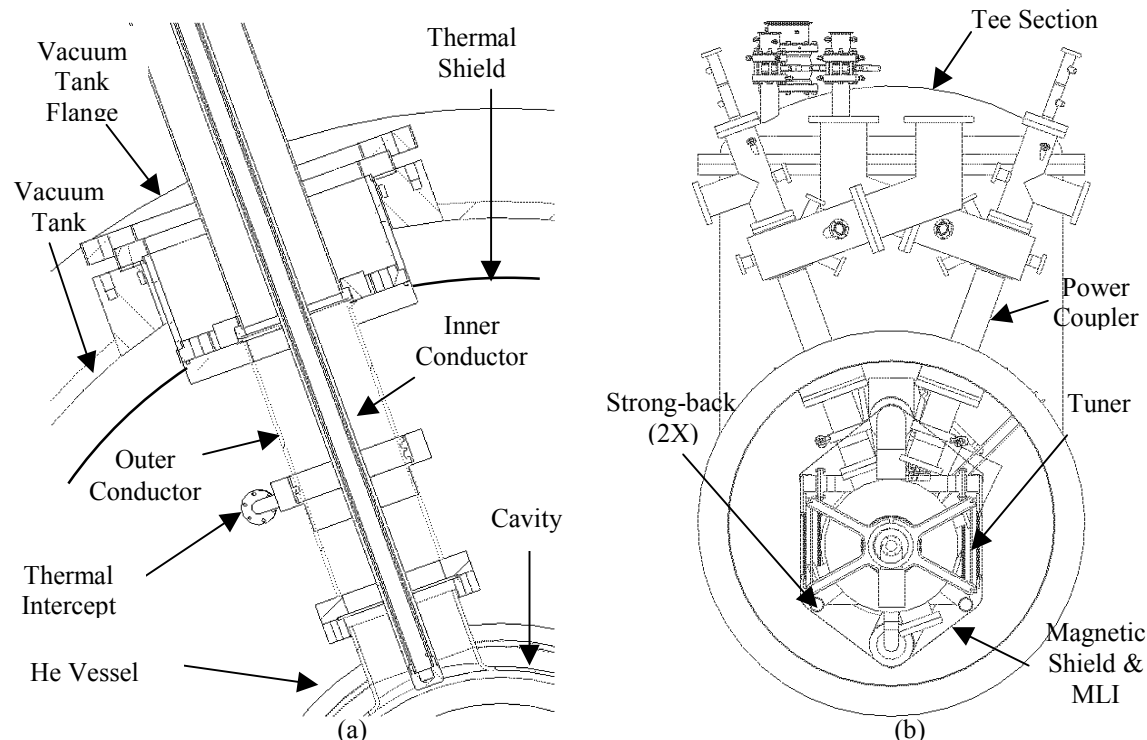
Component (Number)	4.5 K	Shield
Cavities (6)	87.8	0
Couplers @ 100 mA (6) [9]	24	123.6
Beam Tube (2)	1.4	2.94
Current Lead Pair (2) [11]	0.8	3.2
Magnet/Strongback Supports	6	48
Bayonets (4) Male & Female	2.93	4.81
Relief Lines (3)	0.19	3.4
Valves (4)	1	10
Cables, Radiation, HOM	9.22	27.84
Total	133.3	223.8

X 0.146 m.). Heat loads were calculated [9] for a single-point thermal intercept, using techniques developed for APT [10] (see Table 3). The coupler is also the only helium-vessel-assembly support structure, simplifying assembly with fewer penetrations through shields and blankets. Fewer penetrations through the magnetic shields reduce magnetic-fringe fields. Figs. 5 (a) and 5 (b) are sections of the coupler and cryomodule, respectively.

While the helium vessel is titanium, due to the short length of the vessel, the center support by the coupler, and the bellows on each end of the vessel, it may be reasonable to consider a stainless steel vessel for this application. This is under review.

The solenoids are mounted on compression posts similar to those described in [12]. To minimize complexity, a low, 20-A operating current was selected. The subsequent inductance resulted in a long charge time. Use of a persistent mode switch is being considered. This would permit higher current while reducing lead joule heating, resulting in lower inductance and shorter ramp times.

Fig. 6 is the $\beta=0.34$ cryomodule flow schematic. Supercritical helium at 4.6 K and 4 atm is supplied to the module. The flow is split with a portion expanded by a JT valve to fill the thermosyphon reservoir. The remaining flow is re-cooled and directed serially to the solenoids. A re-cooler between solenoids is sized so that the downstream magnet is not

**FIGURE 5.** (a) Coupler - vacuum vessel interface. (b) Cryomodule section. Thermal shield not shown.

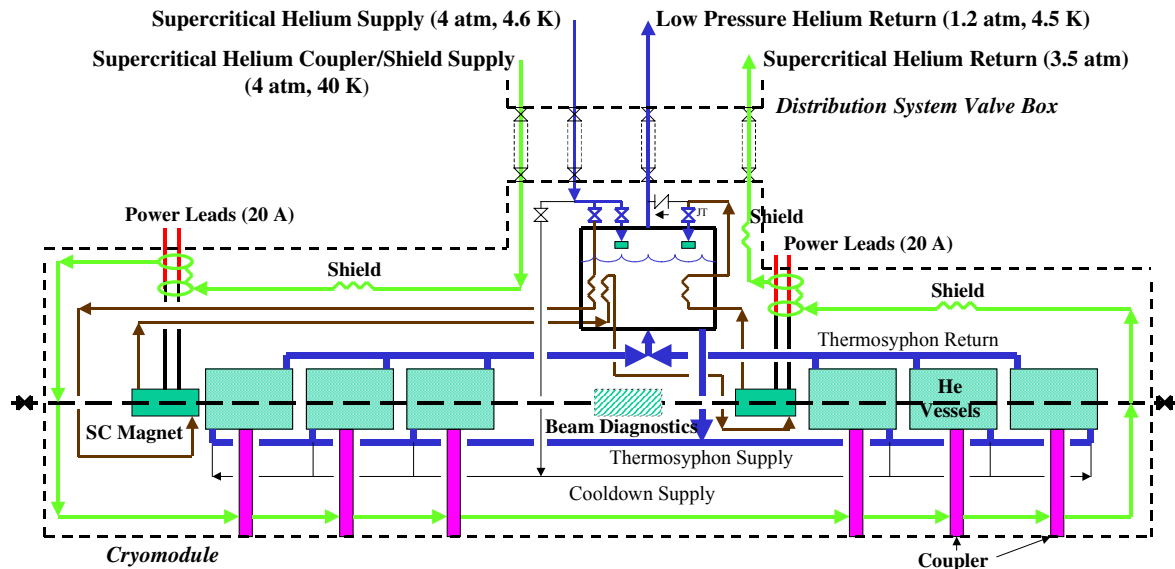


FIGURE 6. Flow schematic for the $\beta=0.34$ cryomodule.

impacted by an upstream magnet quench. A recooler after the downstream magnet removes quench or other heat from the flow. The supercritical flow is then throttled to the thermosyphon reservoir pressure to produce liquid and allow the return of useful cold gas to the cryoplant. This feature eliminates the need for a separate distribution-system return line.

The shields and intercepts are cooled by a flow of supercritical helium at 4 atm and $40 < T < 55$ K. Current leads are conductively cooled with a 40 K intercept.

CRYOMODULE ASSEMBLY

Initial cryomodule assembly will be performed in a cleanroom. The cavity vendor provides the mated cavity and helium vessel. After chemistry, high pressure rinsing, and testing, a coupler is mated to the cavity coupler port to form the helium vessel assembly. At this point, the relative positions of beam centerline, the coupler inner flange and assembly mounting pads are known. The assembly is mounted on a pre-aligned strong-back (Fig. 7) and fiducials are added to outer coupler flanges for relating its position to that of the beam tube centerline. Beam tubes are then installed and the assembly is temporarily locked-down to the strong-back. The solenoid magnet assemblies are similarly mounted on

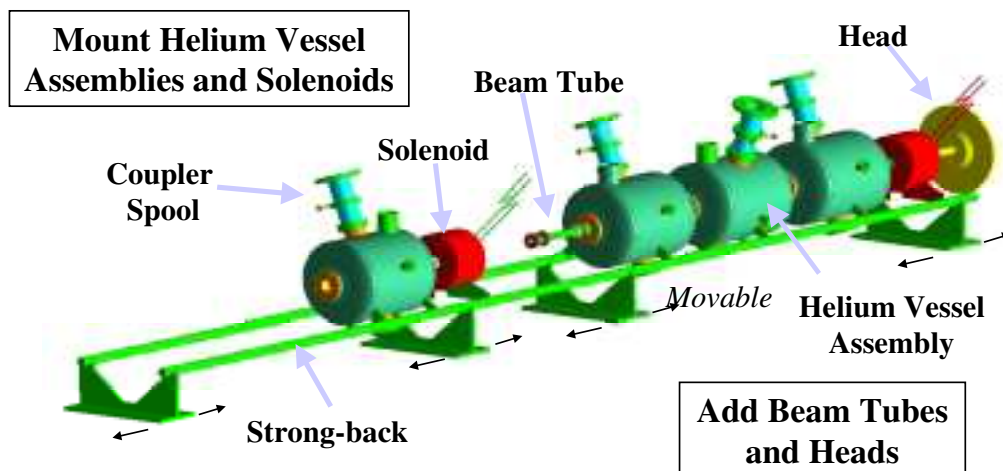


FIGURE 7. Cleanroom assembly.

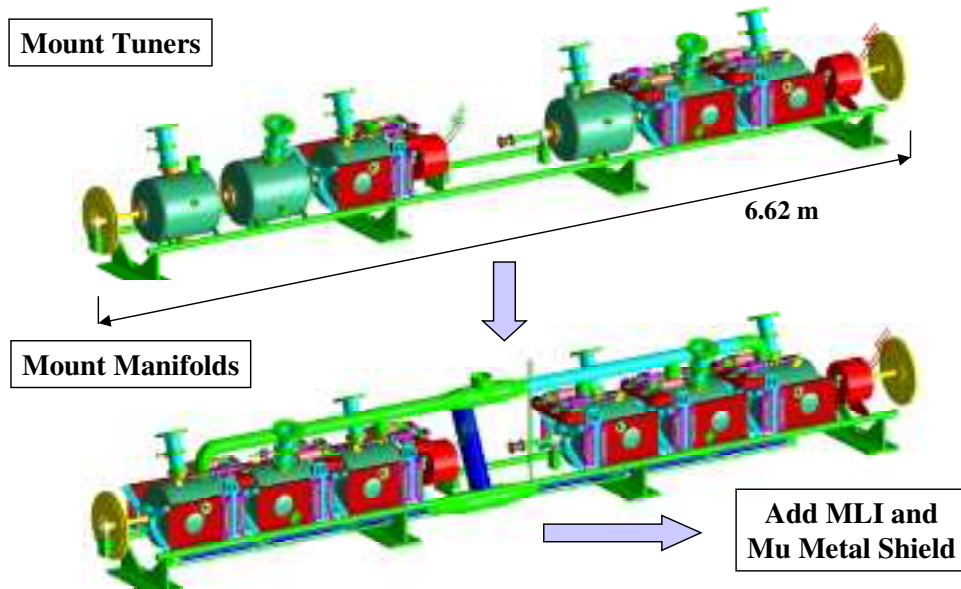


FIGURE 8. Completion of the cold mass assembly.

the strong-back and the beam tubes are installed (cavity to solenoid, solenoid to ambient). Once all the assemblies are mounted, the alignment data is transferred to the fiducials on the outer coupler flanges and final lockdown to the strong-back occurs. All ports are sealed and the remainder of cryomodule assembly is performed outside the clean room.

Once outside the cleanroom, the tuner assemblies are mounted to the helium vessel assemblies (Fig. 8). The thermosyphon and cooldown supply manifolds are welded to the appropriate ports. A 15-layer multi-layer insulation (MLI) blanket and a formed Mu metal shield (1 mm thick) are placed around the manifolds, completing the cold mass assembly.

The cold mass assembly is inserted into the run of a prefabricated Tee section [13]. This section consists of a Tee-shaped vacuum shell with a copper thermal shield, a 1 mm-thick room-temperature magnetic shield, and MLI blankets (4 @ 15 layers ea.) preinstalled (similar to CEBAF [14]). Once inserted, vacuum tanks with preinstalled thermal shields, MLI blankets, and magnetic shields, are mated to the Tee section (Fig. 9). Bridging of shields and MLI blankets between Tee and vacuum cylinders is then completed.

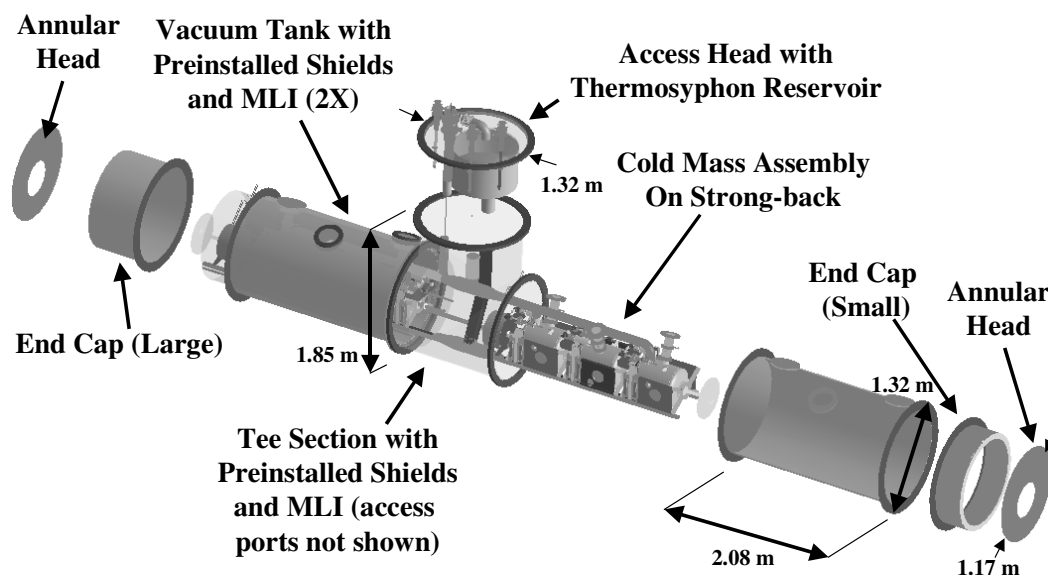


FIGURE 9. Final cryomodule assembly. Thermal, magnetic and MLI shields not shown.

The helium vessel assemblies and solenoids are attached to the vacuum cylinders and aligned. The assemblies are unlocked from the strong-back and the strong-back is removed. Current feedthroughs are made-up and the Tee-section head assembly is installed in the Tee's branch. Pipe connections are made through ports in the side of the Tee. Finally, after installing the appropriate end shields, the vacuum end caps and annular heads are mounted.

SUMMARY

A cryomodule concept for the ADTF LEL spoke cavities has been described. A number of concepts and components from previous programs have been adopted. Assembly is simple and cleanroom time is minimized by axial insertion of the cold mass assemblies into the vacuum vessels. Use of the coupler as the sole helium-vessel-assembly support structure minimizes the potential for thermal shorts and magnetic fringe fields, and reduces the overall part count. Thermosyphon flow has been adopted to cool the cavities, which reduces potential vapor locking and improves heat transfer. This concept can easily be extended to the lower- β units. Since completion of this work, the constraint of fitting the cryomodule into the APT tunnel has been relieved and coupler position will be revisited.

ACKNOWLEDGEMENTS

This work is supported by the U.S. Department of Energy through funding source #NE-AF-5700000.

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