

Calibration Of Fiber Optic Rosette Sensors For Measuring Bending Moment On Tidal Turbine Blades



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Background

US DOE Water Power Technology Office
Open Source Tidal Energy Converter (OSTEC):
Design, build and deploy an instrumented tidal
energy converter – generate open-source R&D
datasets



D=2.5 m rotor diameter MHKF1
turbine, blade span 1.05 m



University of New Hampshire / Atlantic Marine Energy Center
Tidal Energy Test Site at Memorial Bridge in Portsmouth, NH
(Piscataqua River – Great Bay Estuary)

Background

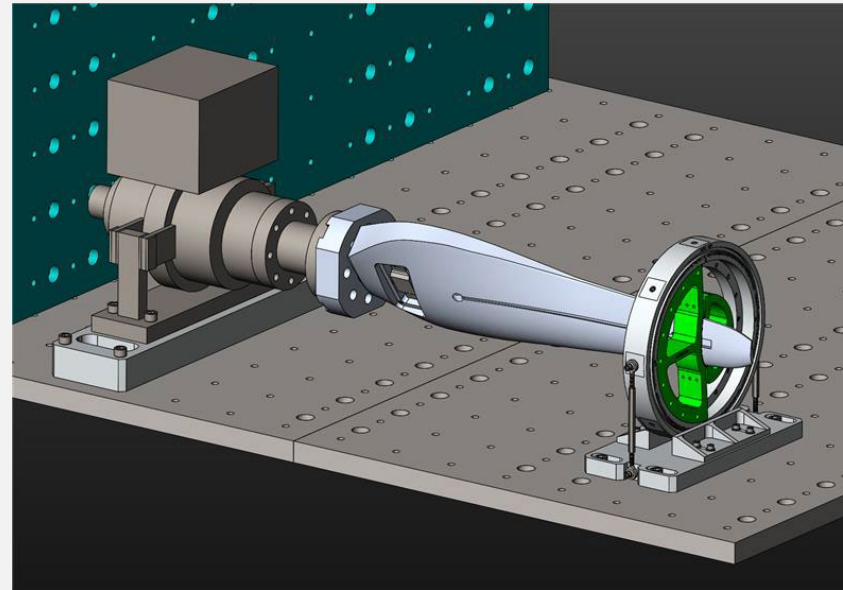
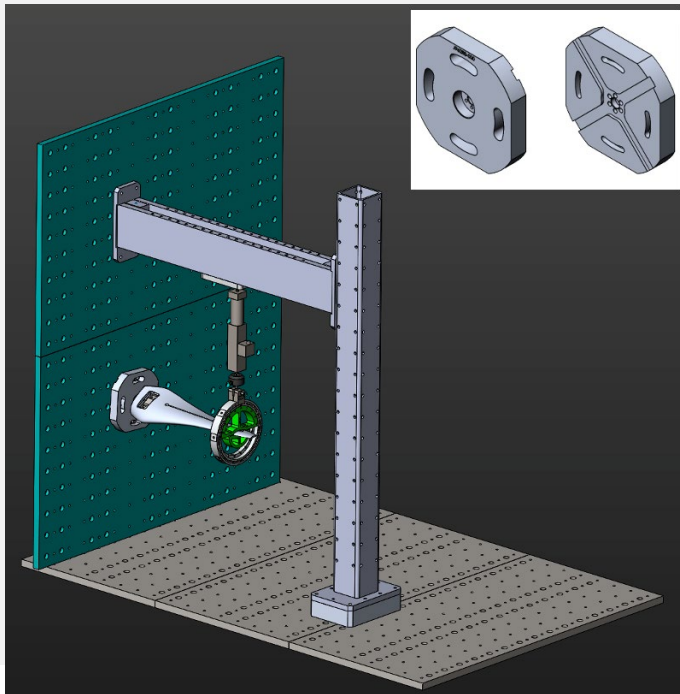
IEC -3 Measurement requirements & sensing

- **Blade root flapwise bending moment** (mandatory on at least one blade; additional blade recommended, specified in table 12 of IEC TS62600-3), measured at the blade root using multiple fiber optic rosette strain sensors (using fiber Bragg-grating (FBG)).
 - **Blade root edgewise bending moment** (mandatory on at least one blade; additional blade recommended, specified in table 12 of IEC TS 62600-3), measured at the blade root using multiple fiber optic rosette strain sensors.
 - **Load exerted in the prime movers**, e.g., blades (recommended, specified in table 9 of the IEC TS 62600-3), measured along the blade using multiple fiber optic uniaxial strain sensors.
- need to develop a calibration curve for that relates the strain measurements and bending moments
- Sandia's Structural Mechanics lab

Structural Laboratory Test Objectives

Test objectives

- Calibrate the bending moment values derived from the blade root fiber optic rosette strain sensors against a set of known loads along the flapwise, edgewise and combined flap- and edge-wise directions.
- Calibrate the twist moment values derived from the rosettes.



Testing was performed at Sandia National Laboratories' Structural Mechanics Laboratory, February – April 2024

Structural Laboratory Test Objectives

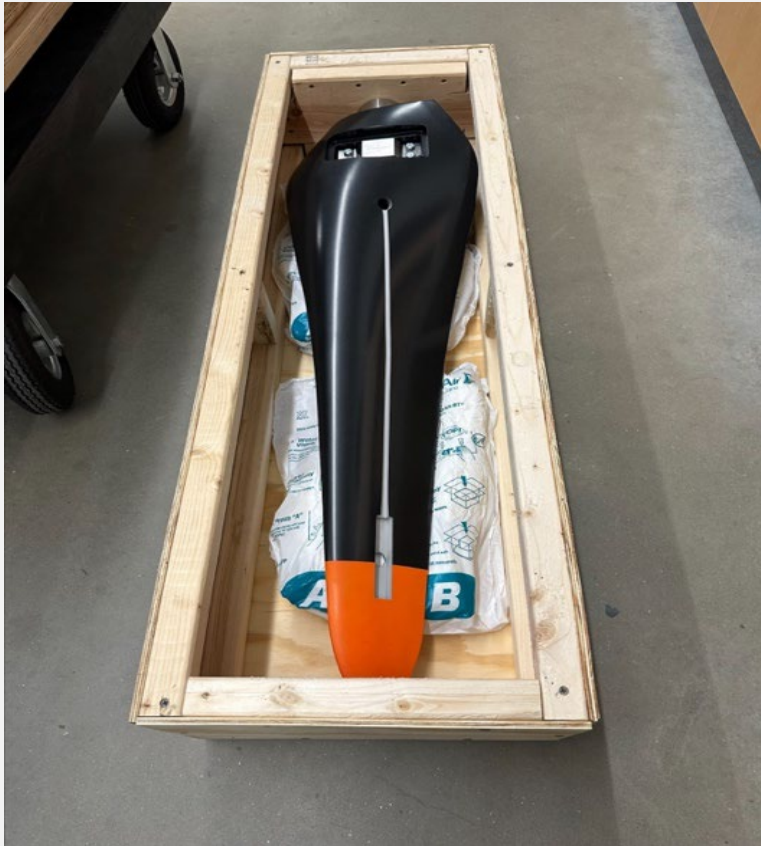
Test objectives

- Calibrate the bending moment values derived from the blade root fiber optic rosette strain sensors (referred as rosettes throughout this report) against a set of known loads along the flapwise, edgewise and combined flap- and edge- wise directions.
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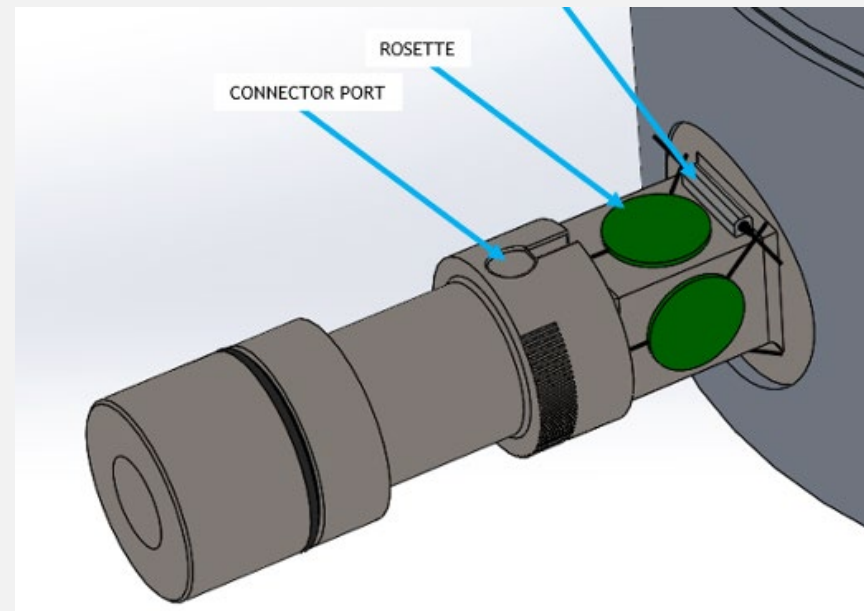
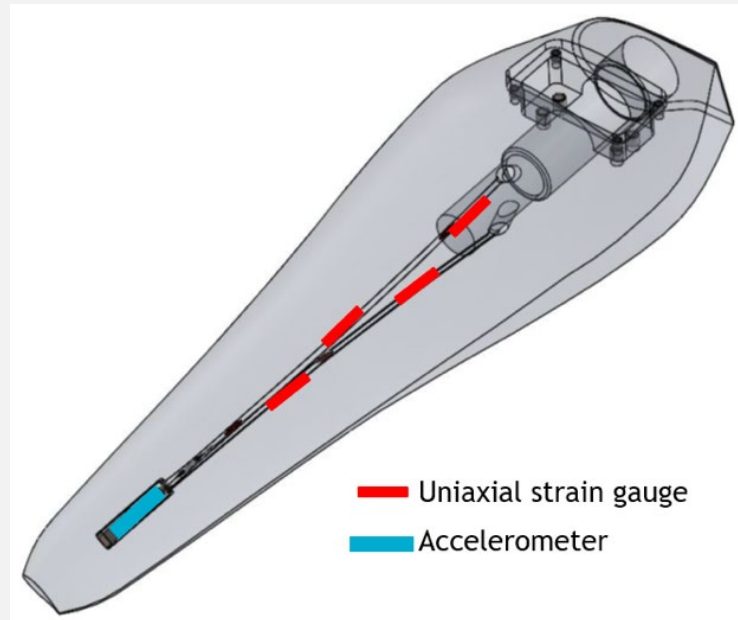
	Moment (ft-lb (Nm))	Equivalent point force applied (lb (kg))
Flapwise	3485 (4725)	1223 (555)
Edgewise	765 (1037)	268 (121)
Twist	116 (157)	

Maximum load, from various Design Load Cases (DLCs), simulated using OpenFAST

Test articles



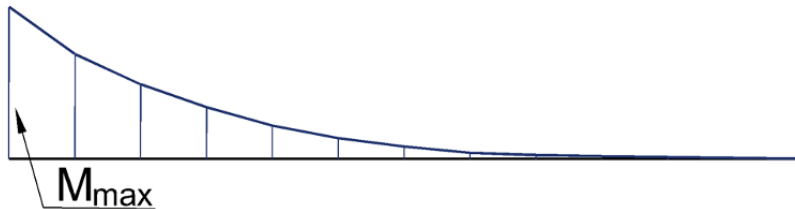
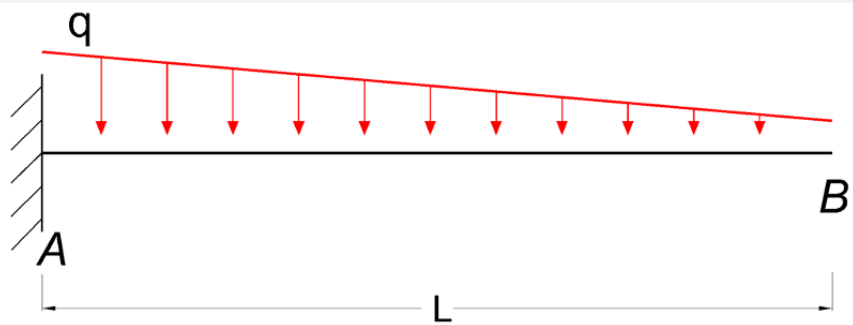
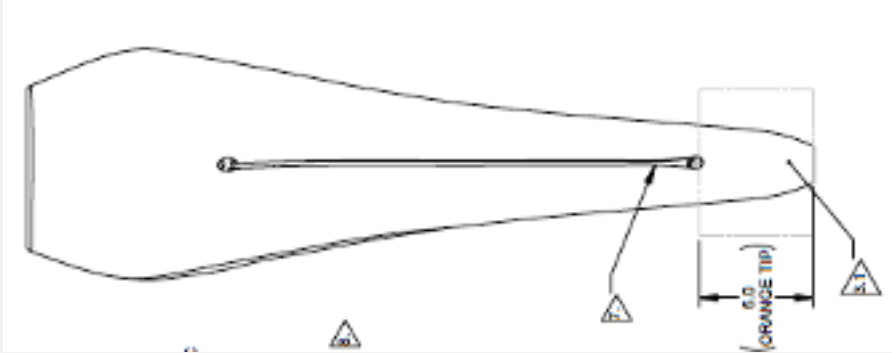
Sensor locations



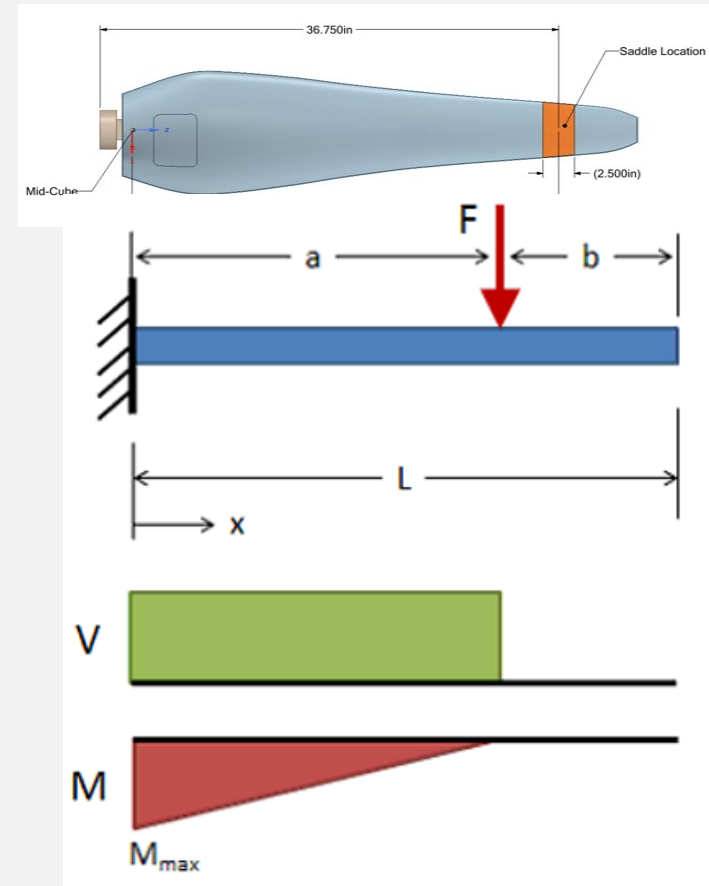
Simplification to point force

Figures are for illustration only

~Simplified load distribution on a rotor blade



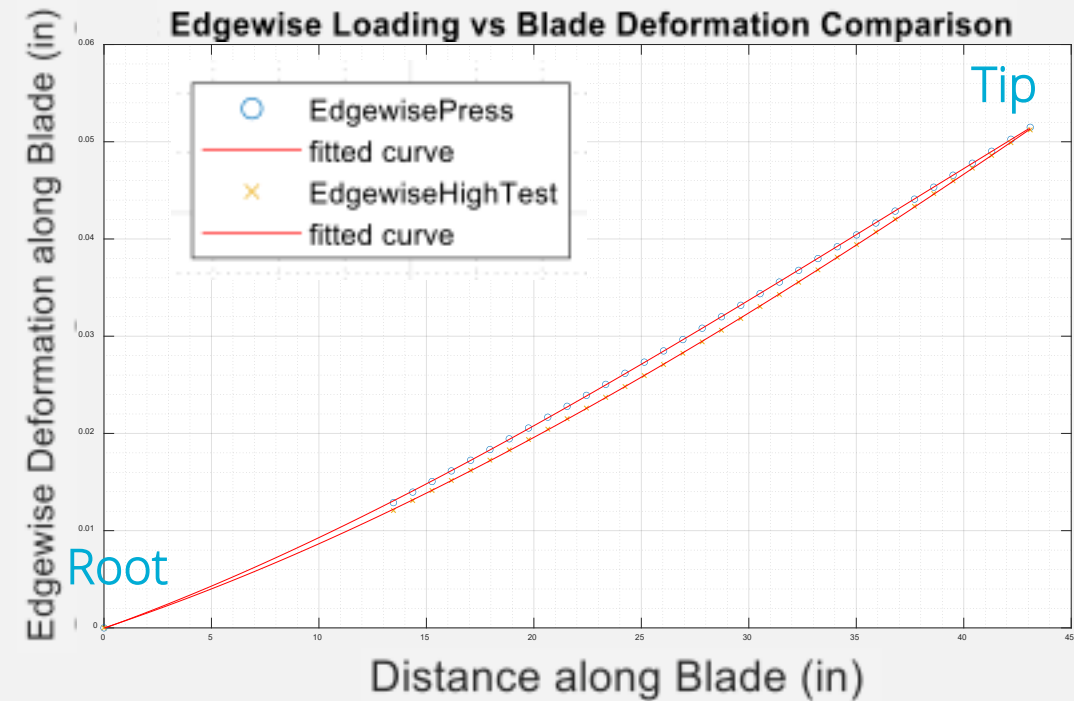
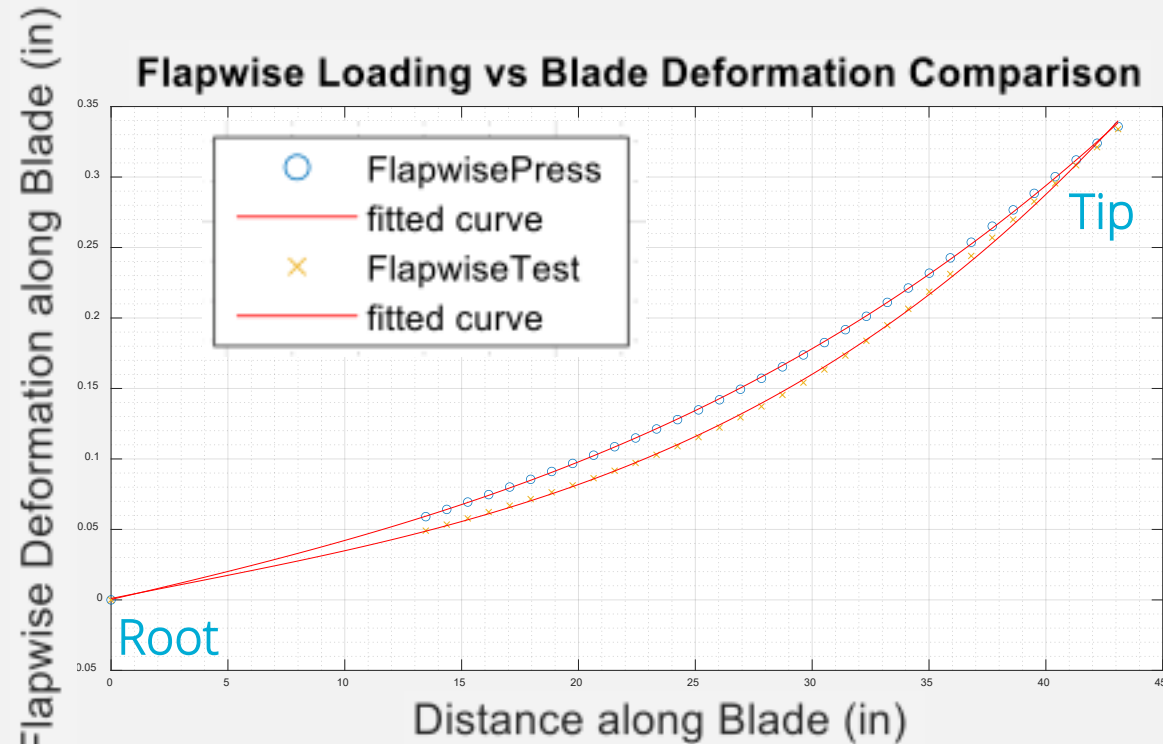
Implementation in structural laboratory



For a rigid body to be static, the net sum of forces and moments acting on it must be zero

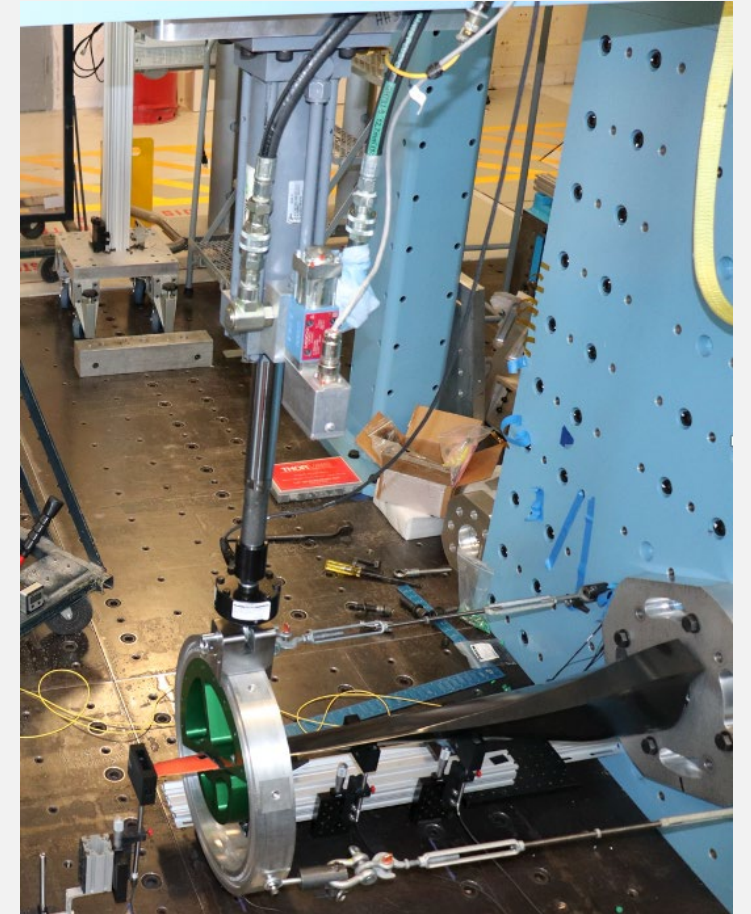
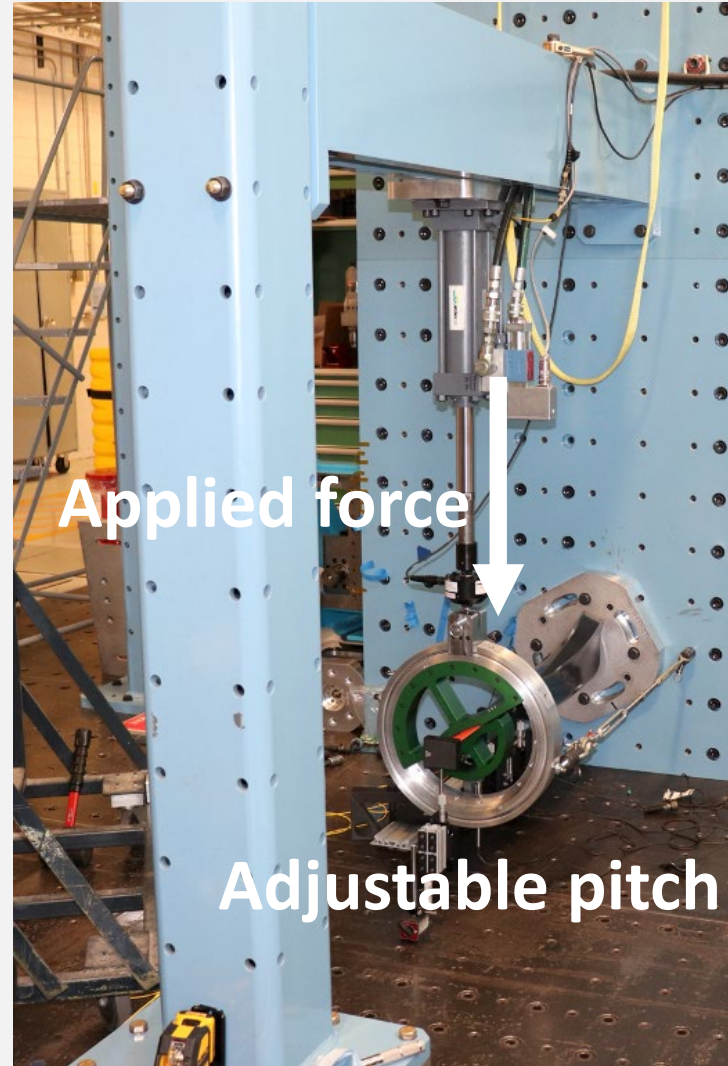
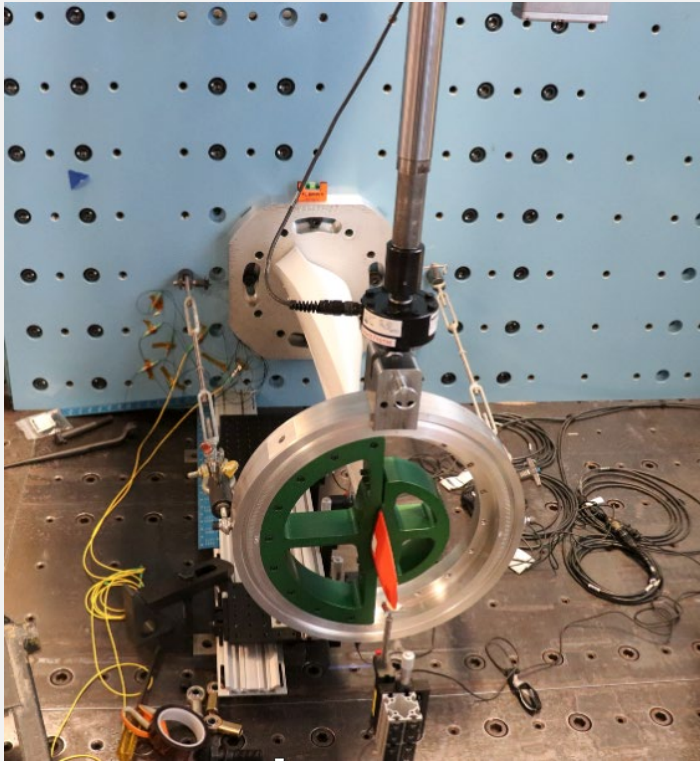
Simplification to point force (continued)

Blade deflection for distributed and point load cases, simulated using FEM



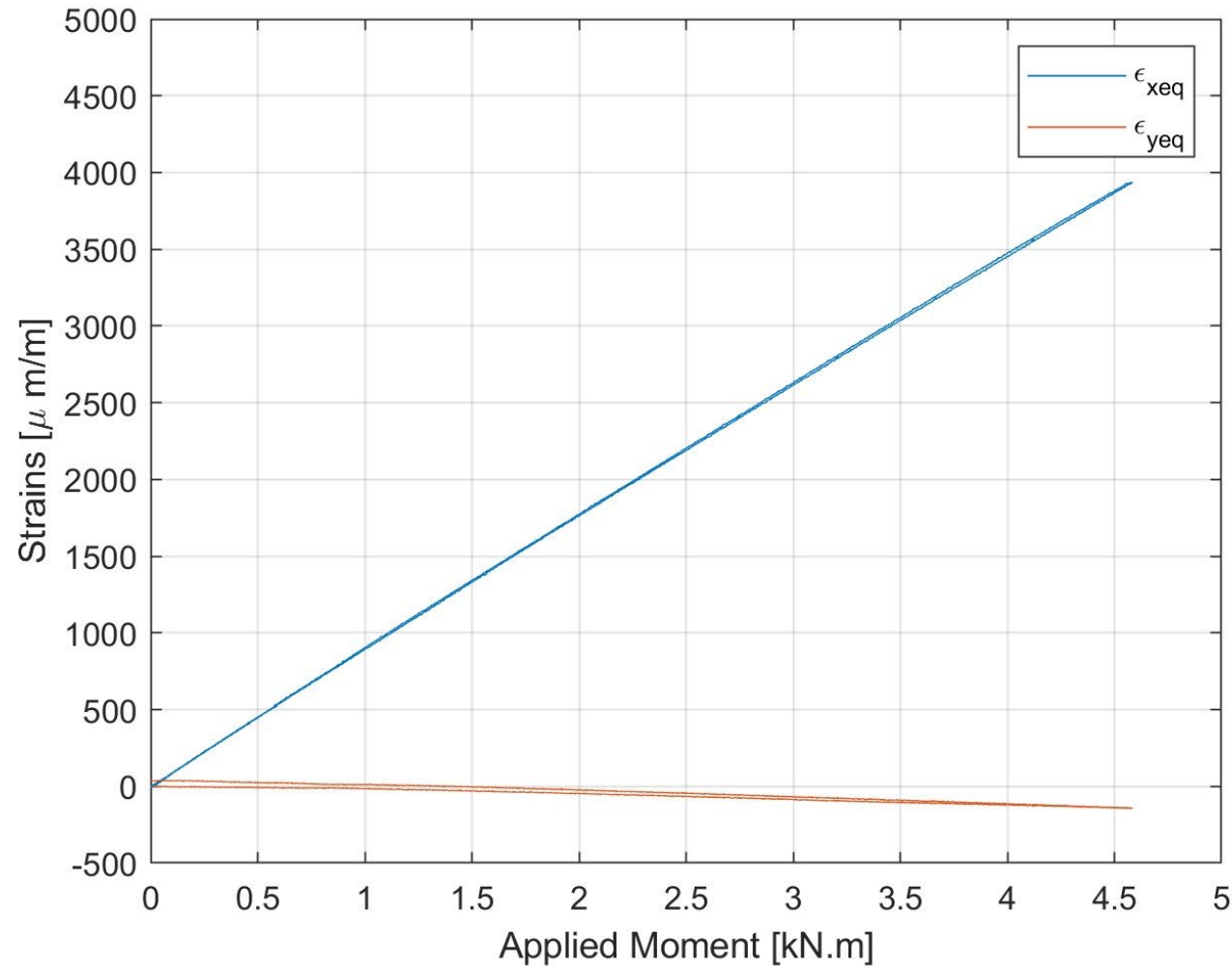
Point-force application location is selected from location with good deflection profile agreement

Bending moment testing



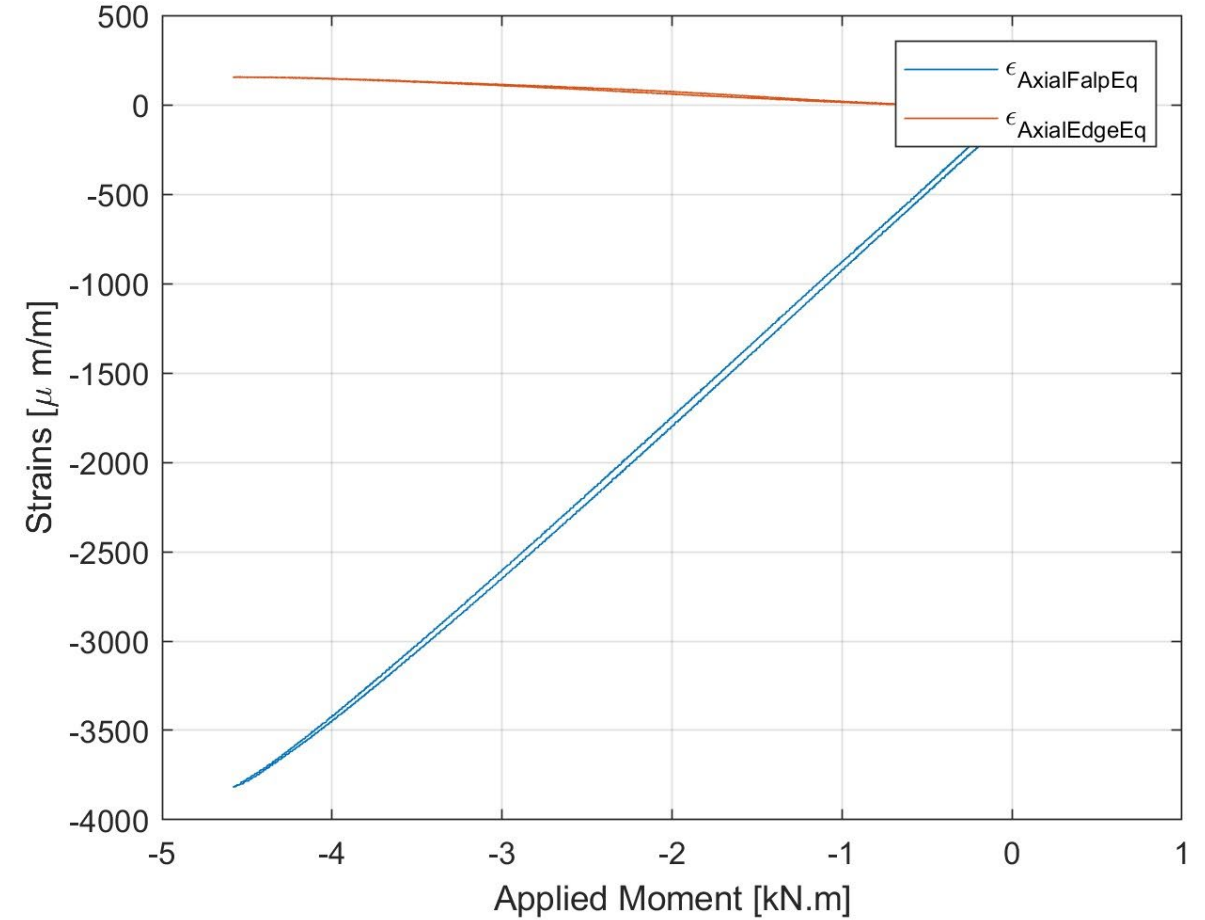
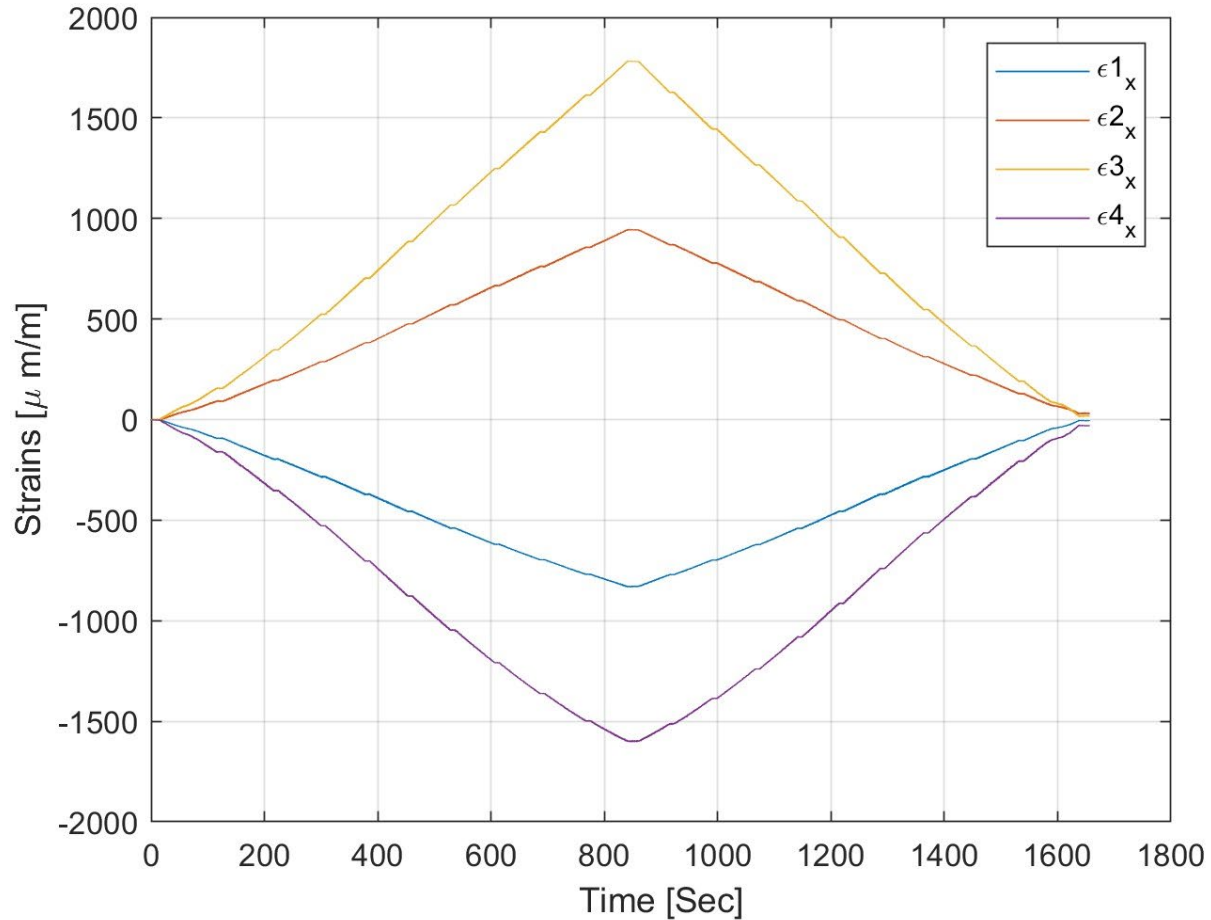
Example result:

Blade 3 - Run 21 (1,185 (524 N) lbf flapwise ; 4,575 Nm)

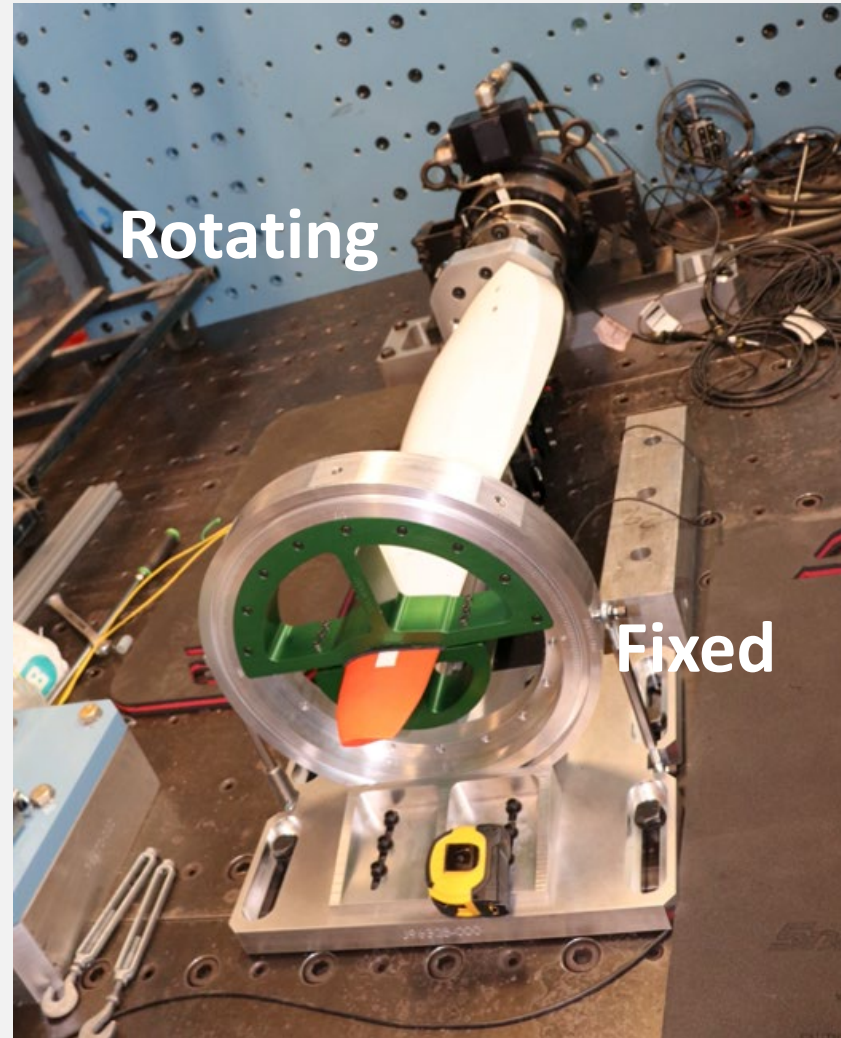


Applied moment (by actuator) Vs Equivalent strain epsilon x : - ~linear relationship

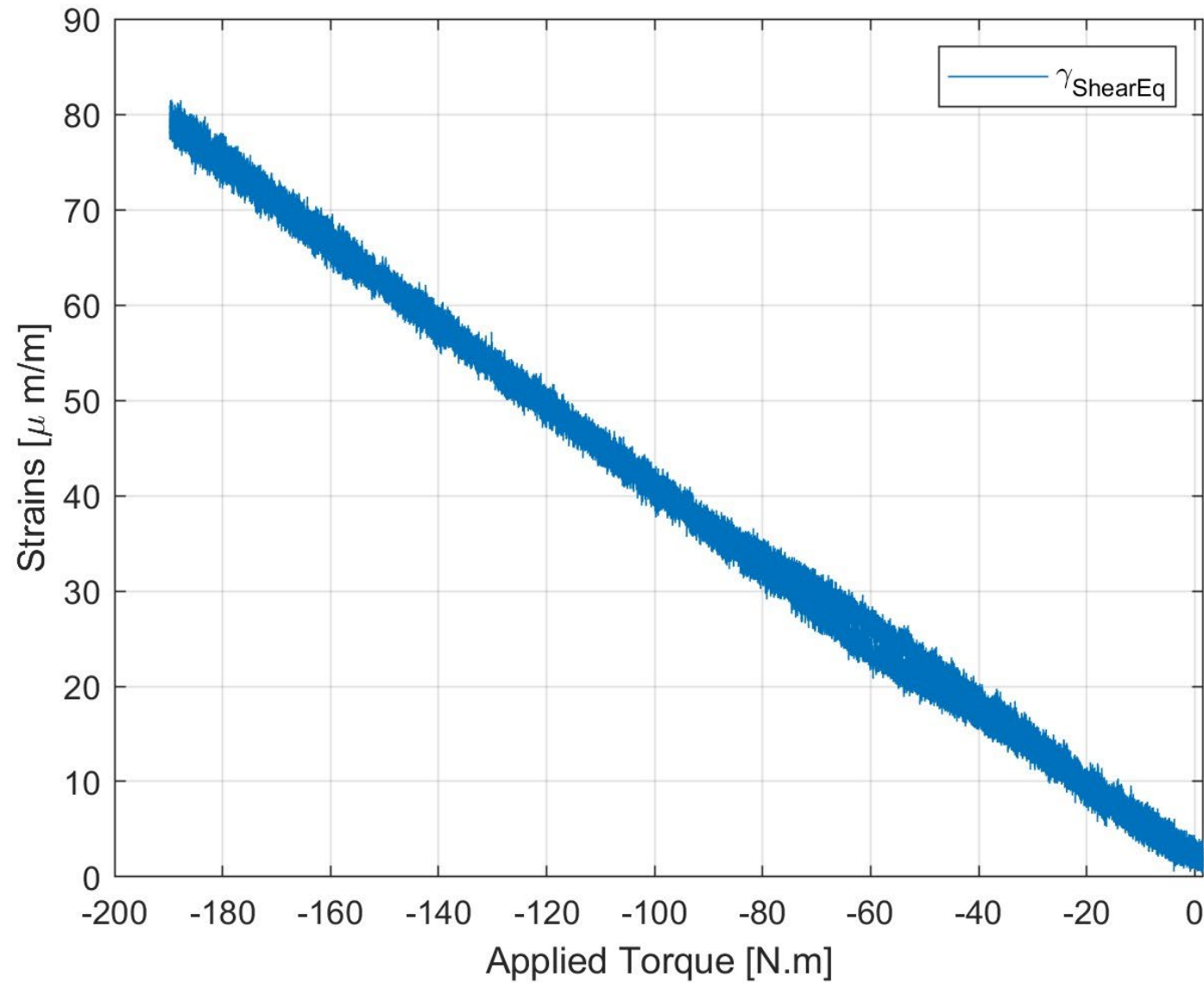
Example result: Run 24 (Flapwise (1180 lbs) 5.25 kN.m)



Torsion testing



Example result: Run 47 (Torque (139 ft -lbs) 188.5 N.m)



Preliminary conclusions

- Virtually linear relationship is observed on strain vs the various quantities of interests (flapwise bending moment, edgewise bending moment, torsion)
- ready for system integration, dry system testing, then open-water testing
- Bending moment can be derived from the strain measurements using theoretical equations (~10% difference from the prescribed value)

Acknowledgements

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