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Validating 3-axis mechanical shock environments with nonlinear dynamic models

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Outline

- Motivation
- Experimental Setup for Validation
- Model Development
- Results and Discussion
 - Shock Response Spectrum
- Conclusions

Motivation

- Components of aerospace systems are often subject to the pyroshock and other shock loads and, as such, must be qualified for those environments.
- Single-axis testing is a common qualification method, but it cannot perfectly reproduce the environmental loading.
- Three-axis tests can be complex and tedious to set up, so it is desirable to quickly iterate and design the test fixture computationally.

Pilot ejection



Stage separation



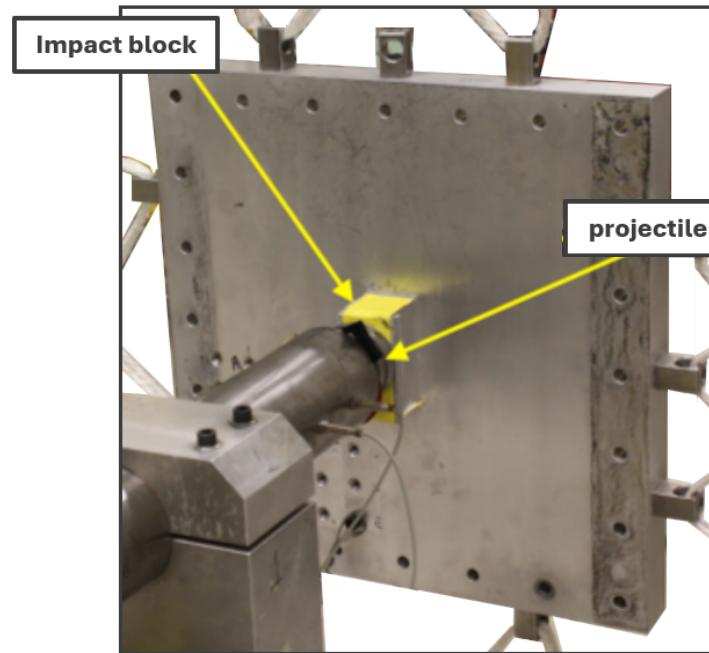
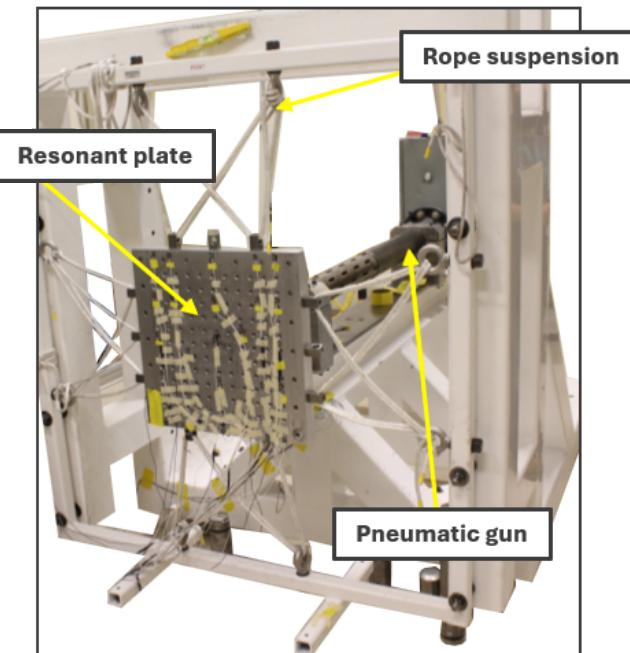
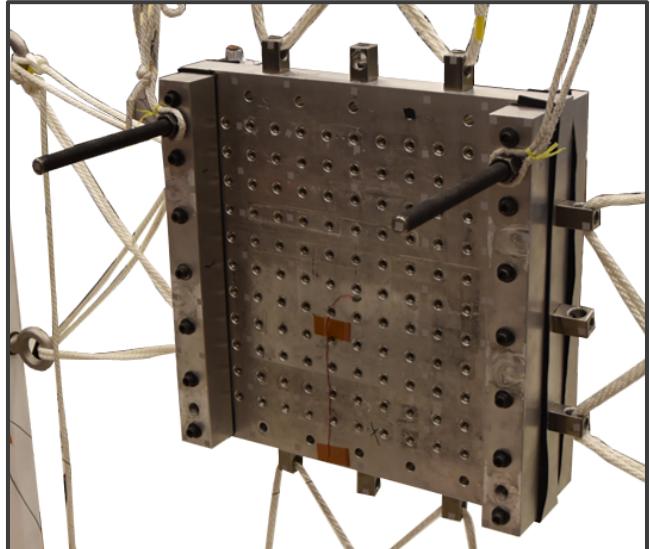
[Open source images]

<https://www.wallpaperflare.com/search?wallpaper=ejection+seat>

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The Resonant Plate Test

- A high-velocity projectile strikes the resonant plate and excites a response.
- Rubber and aluminum bars are added to tune the plates modes
- A felt material is attached to the face of the impact block to tune the impulse shape
- 24 accelerometers captured data for over 77 resonant plate shots with a variety of programmers, projectile sizes, and projectile speeds.

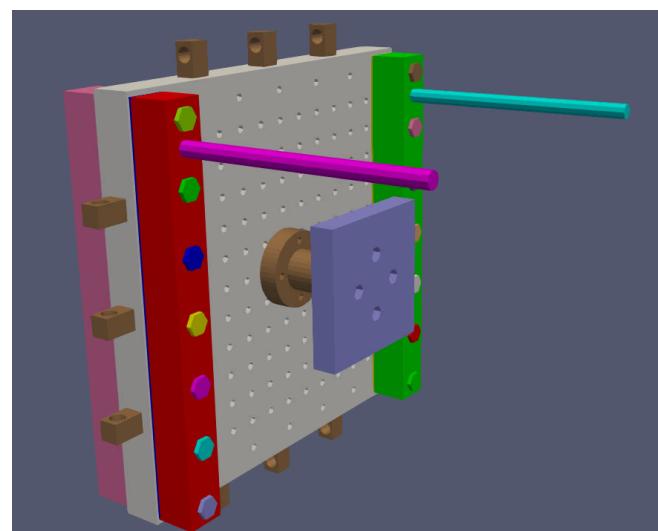
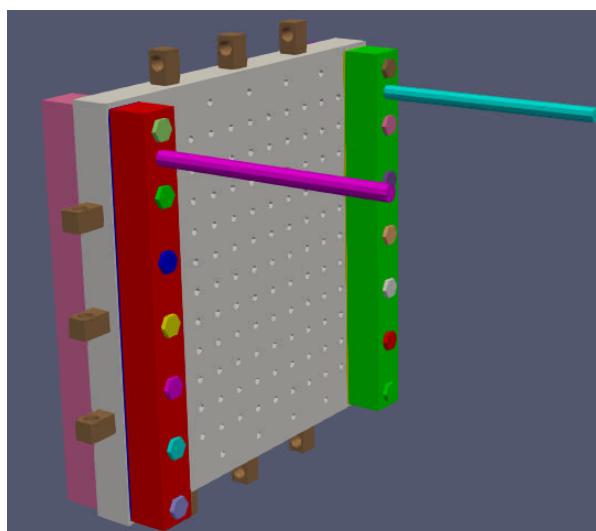
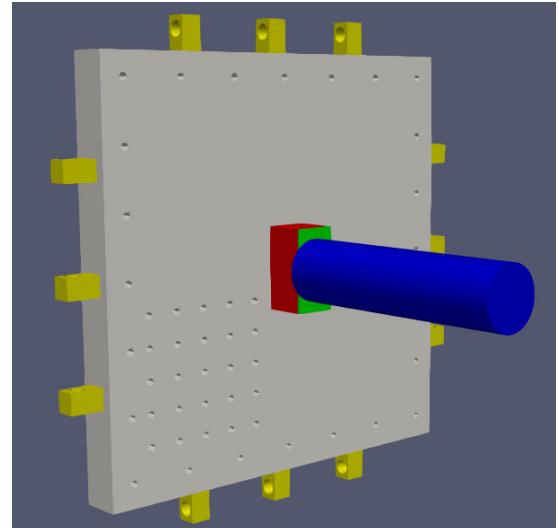
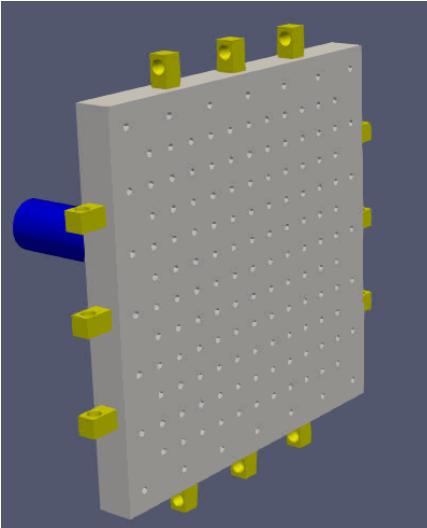


Summary of test data shots

| Run Set Name | Test Runs | Pressure (psi) | Felt Thk (in) | Projectile Length (in) | Average Projectile Speed (ft/s) |
|--------------|-----------|----------------|---------------|------------------------|---------------------------------|
| Set 14 | [14:17] | 10 | 1/8" Grey | 6" | 21.5 |
| Set 21 | [21:27] | 14 | 1/8" Grey | 6" | 28.1 |
| Set 28 | [28:31] | 10 | 1/8" Grey | 6" | 21.3 |
| Set 32 | [32:35] | 10 | 1/2" Grey | 6" | 21.4 |
| Set 36 | [36:39] | 20 | 1/2" Grey | 6" | 35.7 |
| Set 41 | [41:47] | 20 | 1" Grey | 6" | 36.0 |
| Set 48 | [48:51] | 40 | 1" Grey | 6" | 53.5 |
| Set 54 | [54:57] | 50 | 1" Grey | 6" | 59.8 |
| Set 58 | [58:61] | 20 | 1" Grey | 12" | 23.9 |
| Set 62 | [62:65] | 40 | 1" Grey | 12" | 37.1 |
| Set 66 | [66:68] | 60 | 1" Grey | 12" | 46.1 |
| Set 69 | [69:72] | 15 | 1/2" Grey | 12" | 19.5 |
| Set 73 | [73:76] | 25 | 1/2" Grey | 12" | 27.8 |
| Set 78 | [78:81] | 35 | 1/2" Grey | 12" | 34.1 |
| Set 82 | [82:85] | 15 | 1/8" Grey | 12" | 19.1 |
| Set 86 | [86:91] | 25 | 1/8" Grey | 12" | 27.5 |

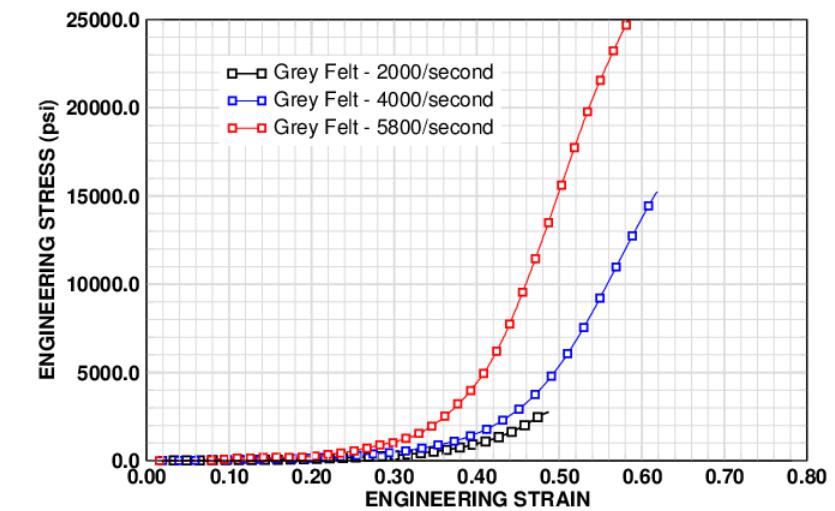
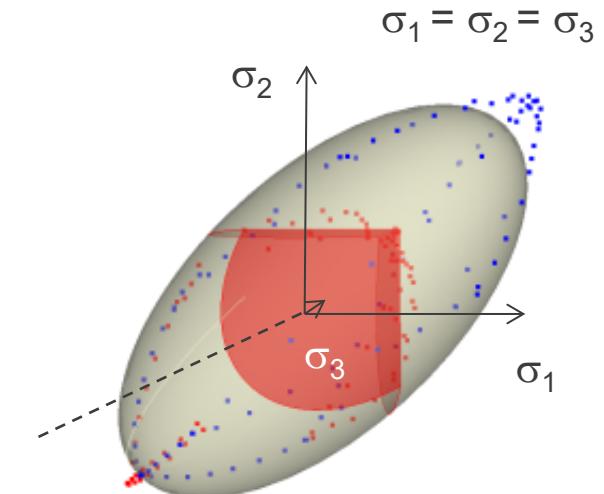
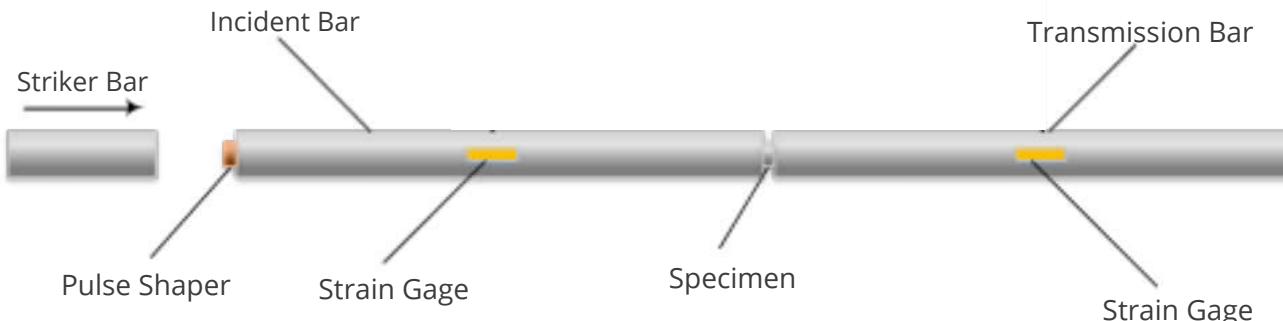
Model Development

- The test fixture has been computationally modeled, including surface holes for mounting the unit under test.
- The nonlinear model is simulated with explicit dynamics with the following features:
 - Prescribing an initial velocity on the projectile
 - Contact and friction
 - Large deformations
 - Nonlinear material models for felt, rubber, and elastic-plastic behavior
- To validate the model, the fixture is simulated in stages to not propagate errors into the components response:
 - Bare plate
 - Plate with damping
 - Plate with damping and component



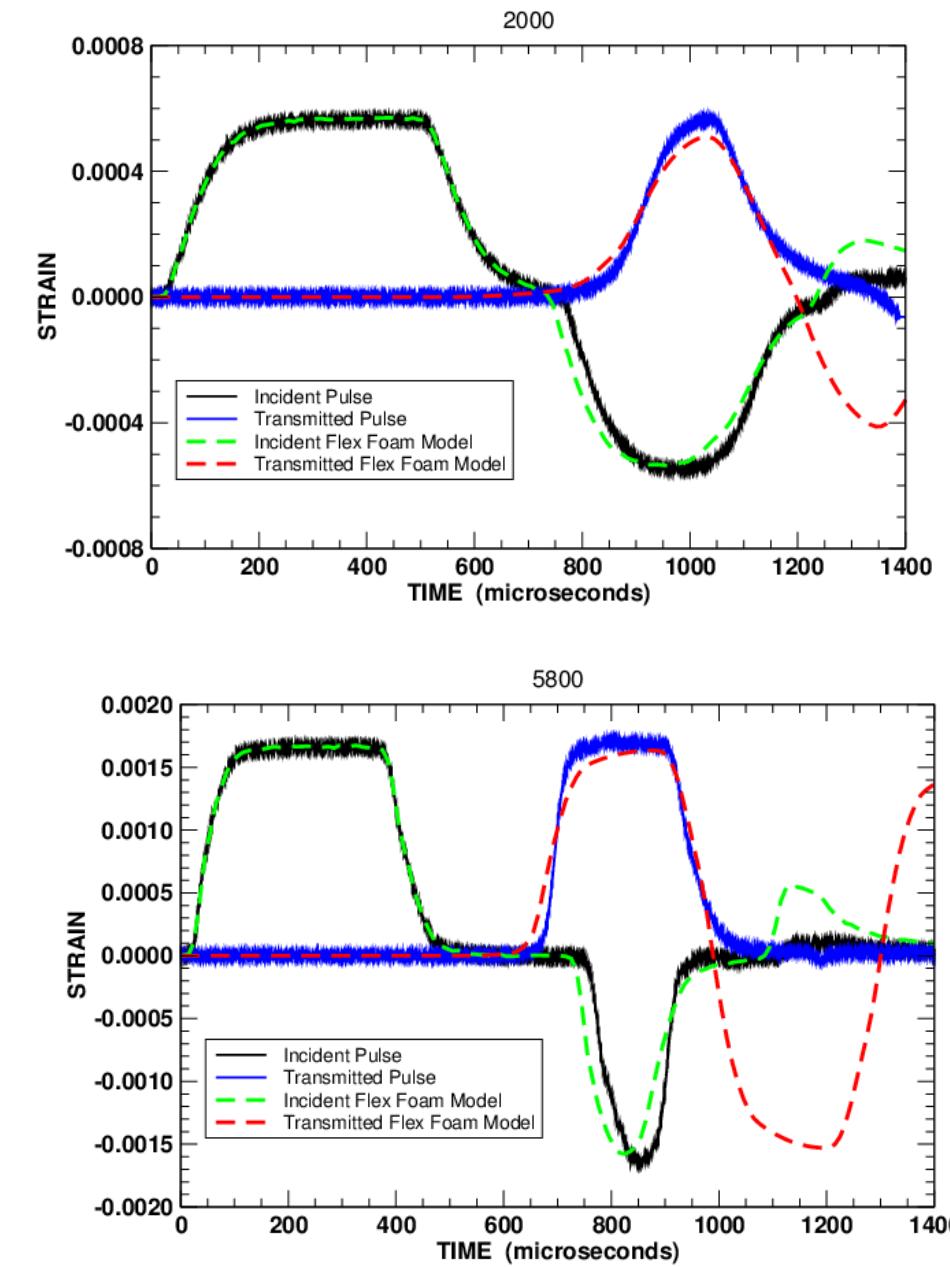
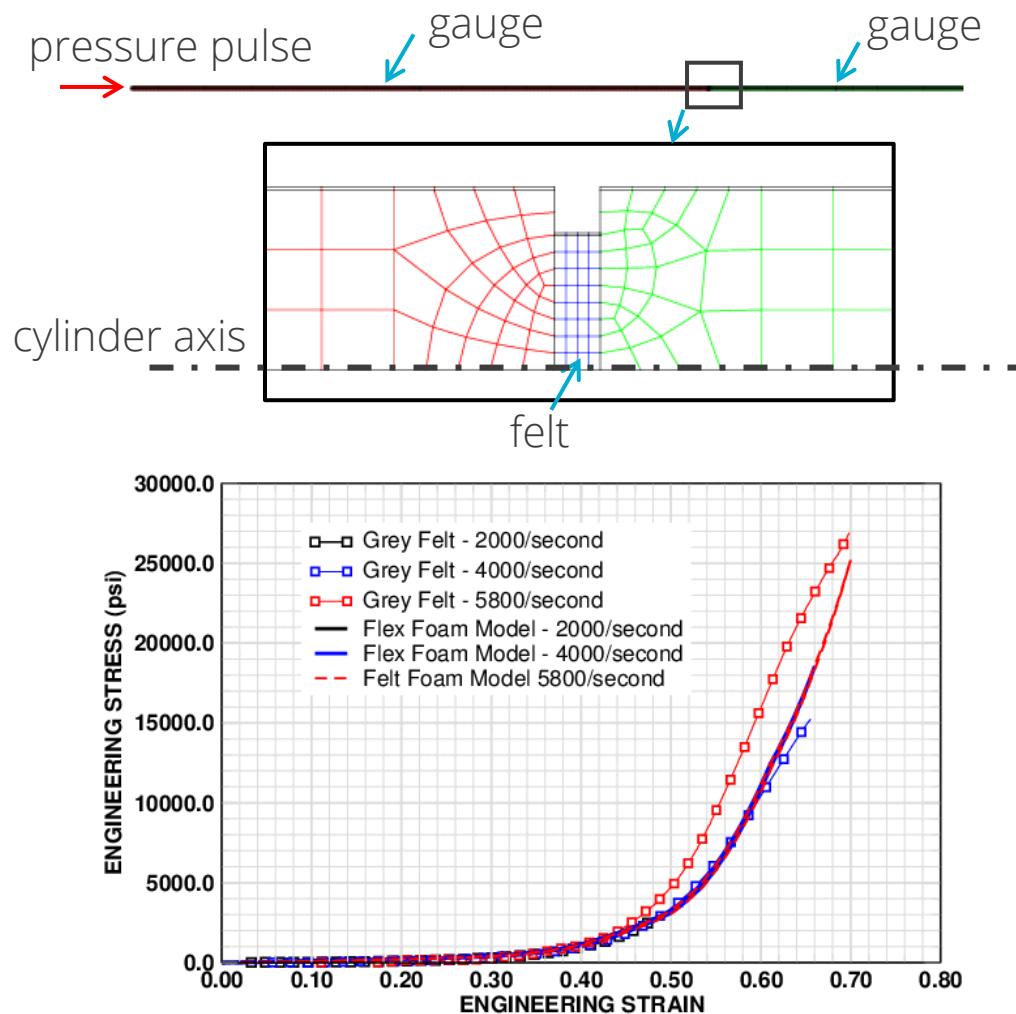
Split-Hopkinson Bar Experiment for Material Parameters

- Since felt is a porous material, it can withstand a lot of compression before large amounts of stress are introduced.
- The felt will fail quickly under tension, behavior that is captured by a foam model with an ellipsoidal yield surface truncated by red damage surfaces for loading in tension.
- The Split-Hopkinson bar induces a compression wave to move through the material specimen.
- By measuring the strain through the bars, the Young's modulus of the material can be approximated.



Material Model Validation

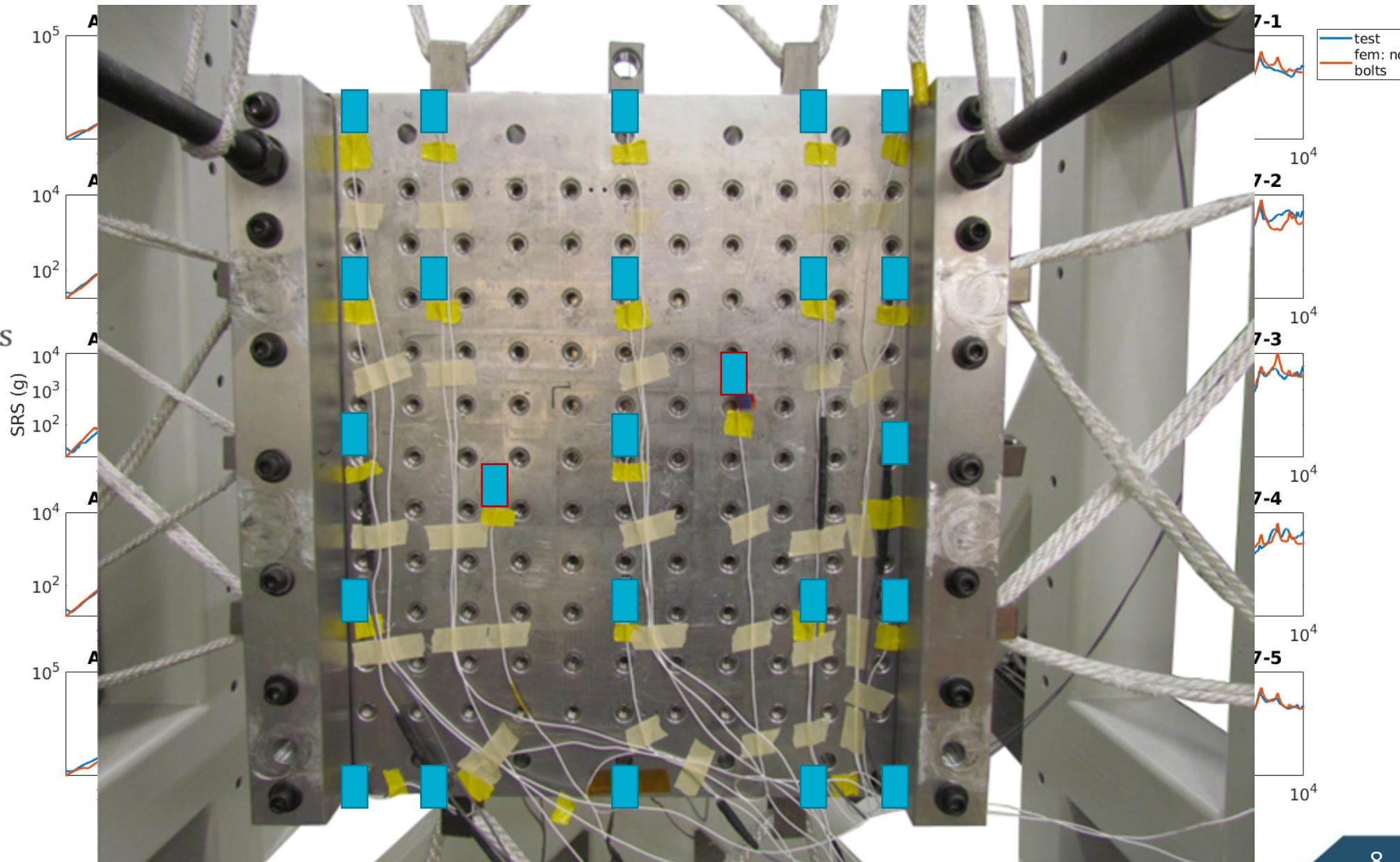
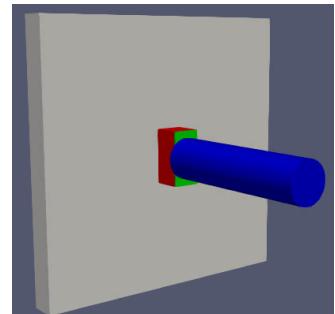
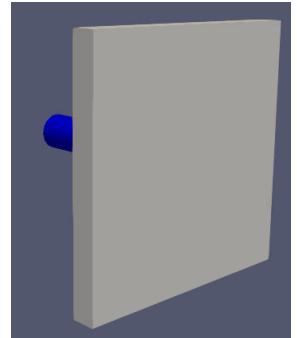
- Once the material parameters were approximated, the Hopkinson bar experiment was simulated to validate the foam model used for felt.



Results Comparison: Bare Plate with No Mounting Holes

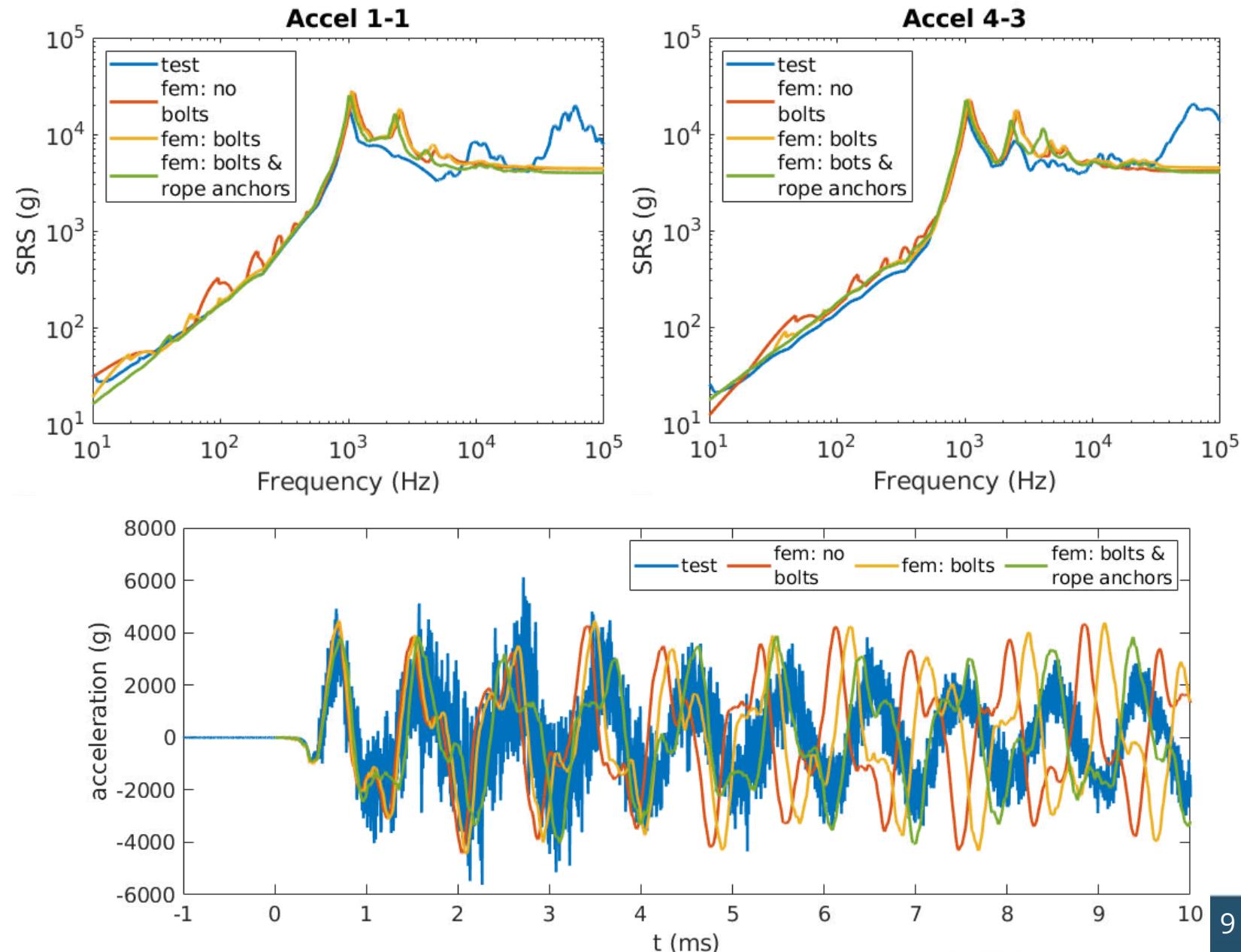
Test configuration:

- Plate with no damping or components
- 12" projectile
 - $v_i = 20.2 \text{ ft/s}$
- 1/8" felt



Results Comparison: Bare Plate

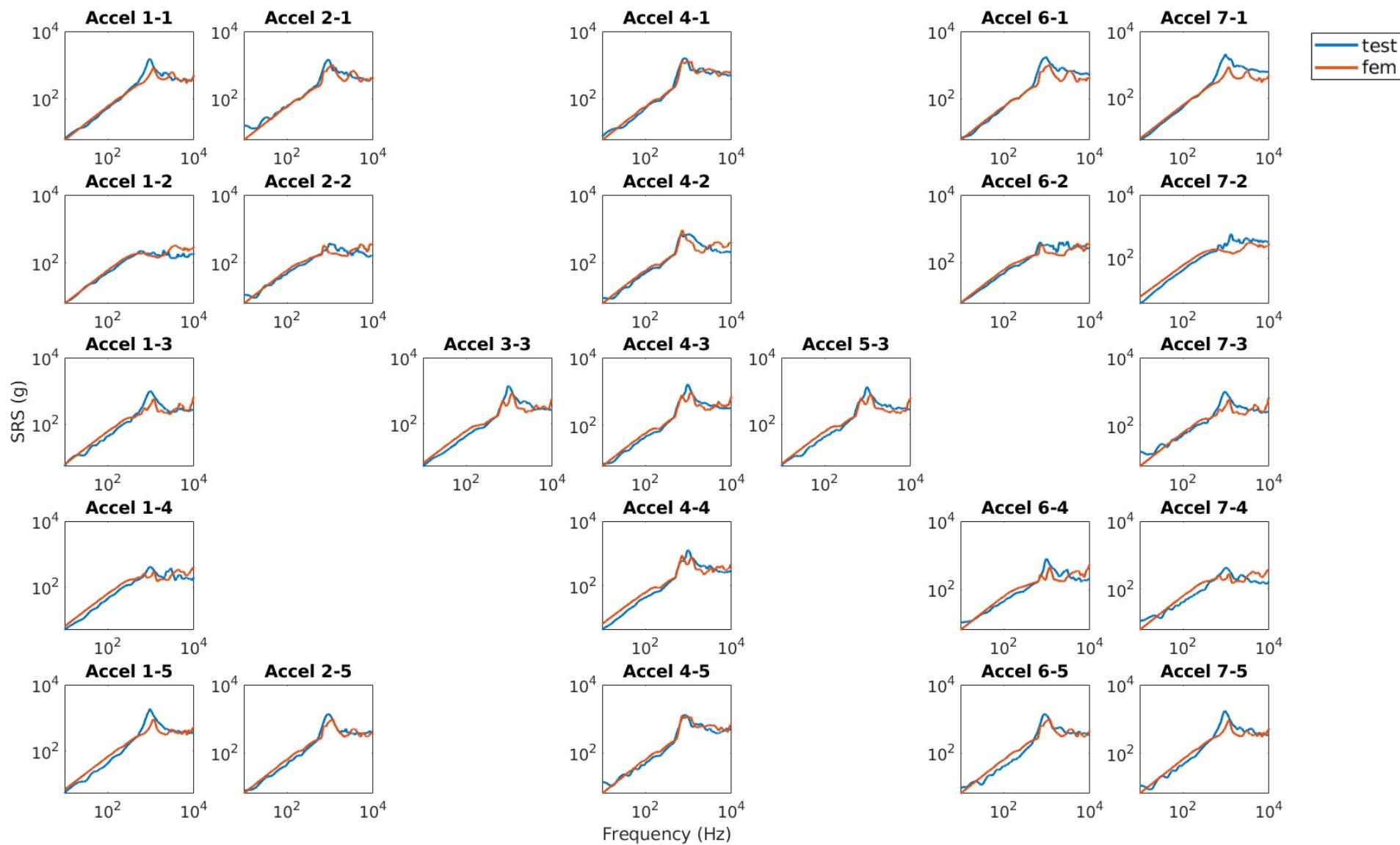
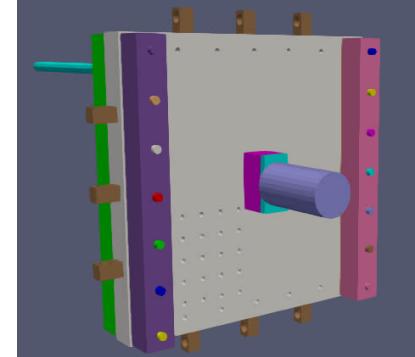
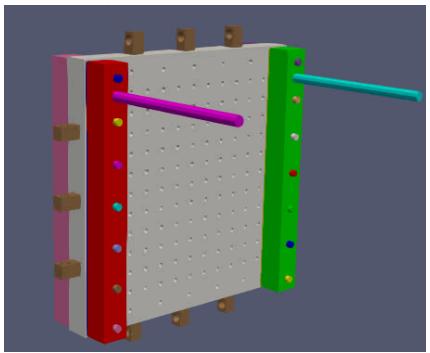
- Adding the mounting holes for the bolts had a minimal effect on the response.
- The addition of the rope anchors reduced the frequency to better match the peaks in the SRS.
- Through the first 10 ms of the time history, the configuration with the rope anchors and the bolt holes matches the response of the test excellently.



Results Comparison: Plate with Damping Bars

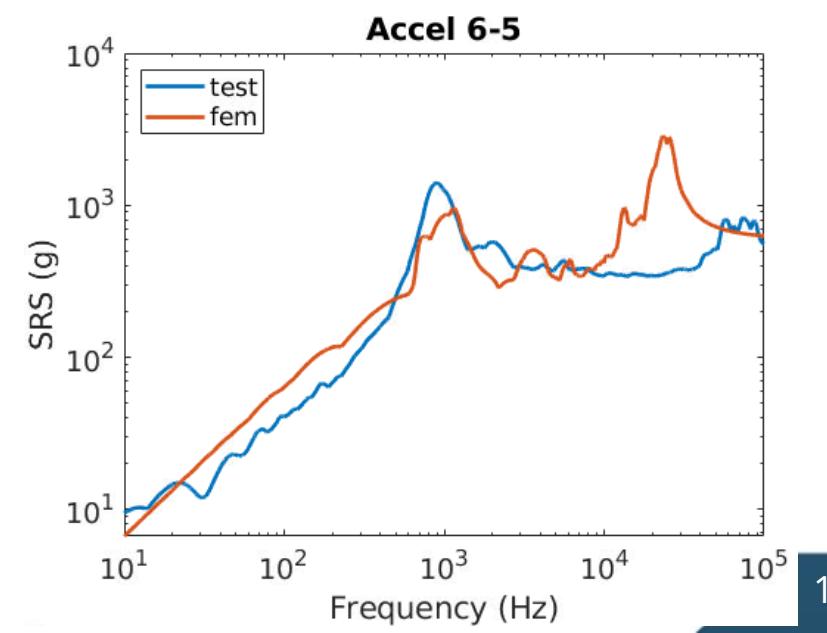
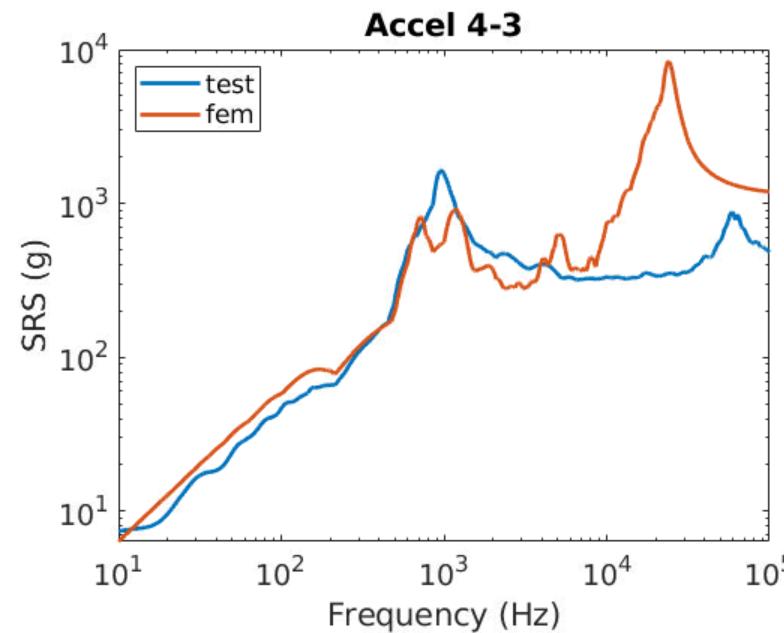
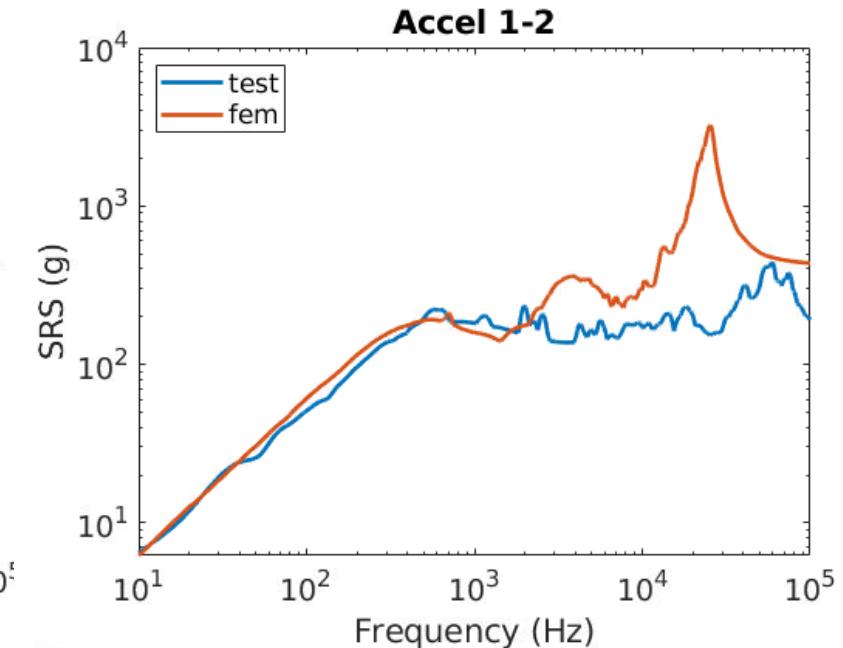
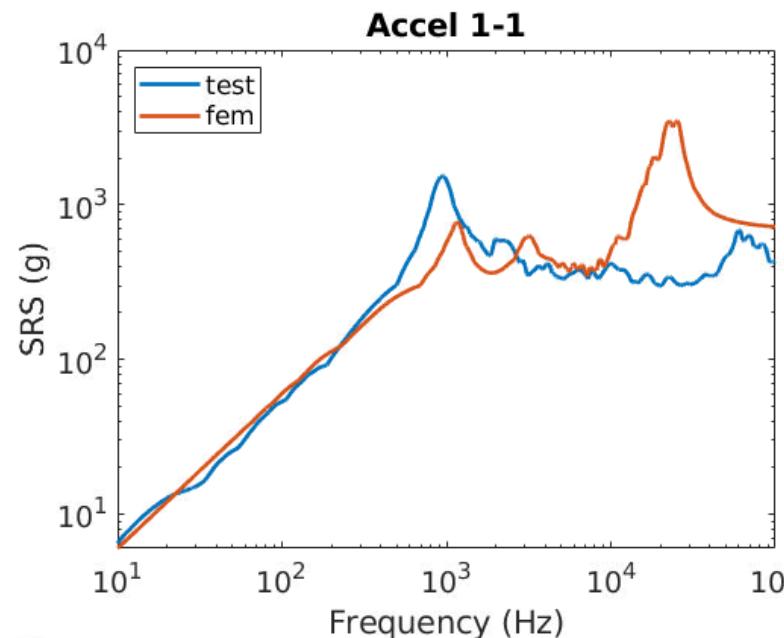
Test configuration:

- Plate with damping but no components
- 6" projectile
 - $v_i = 19.5$ ft/s
- 1/2" felt



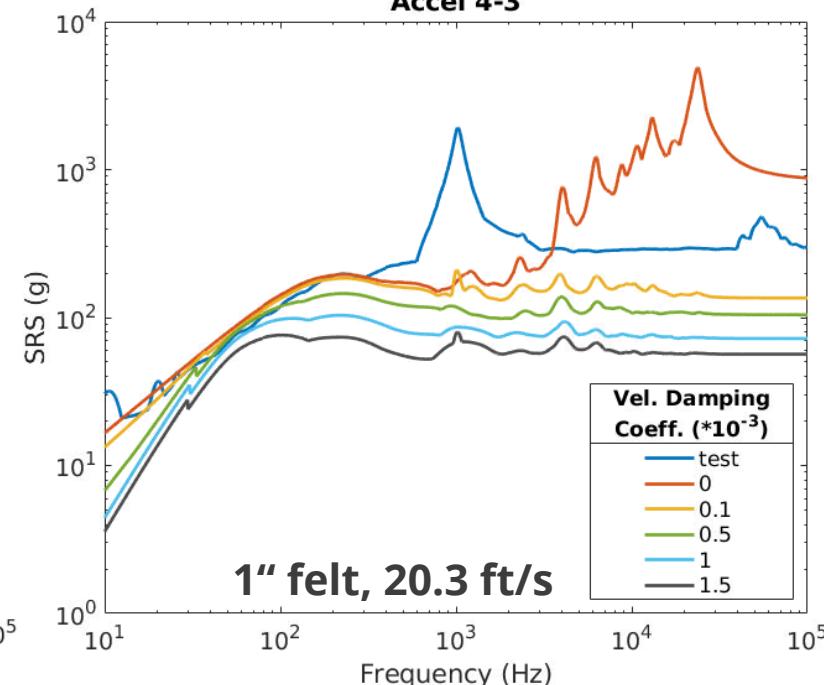
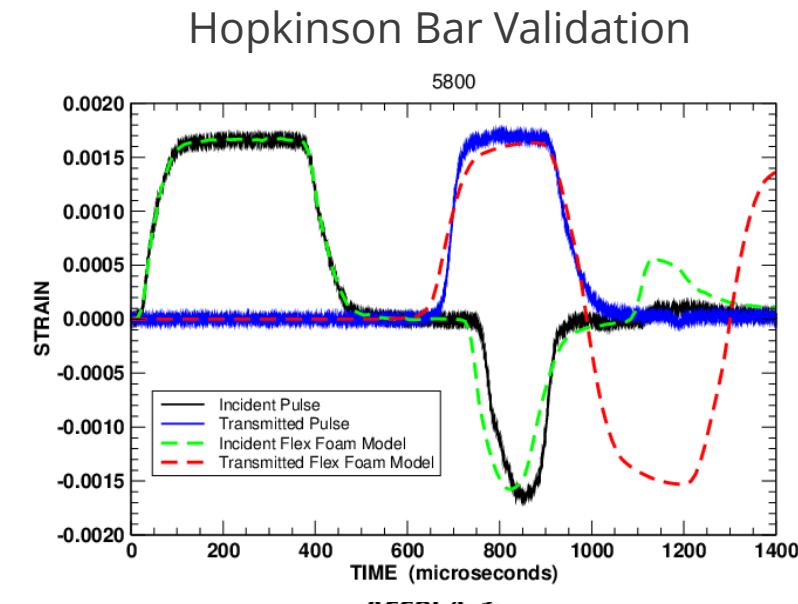
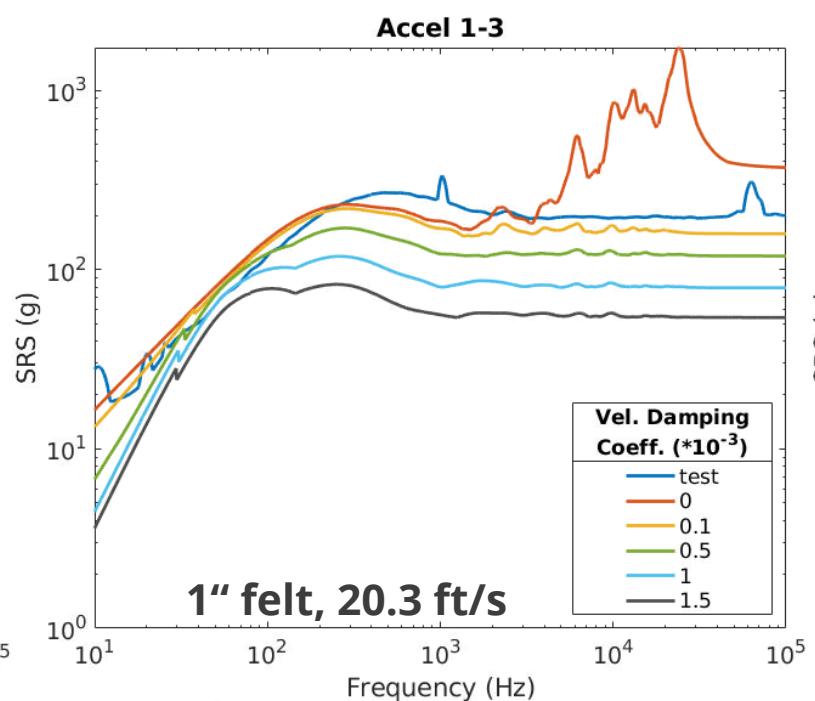
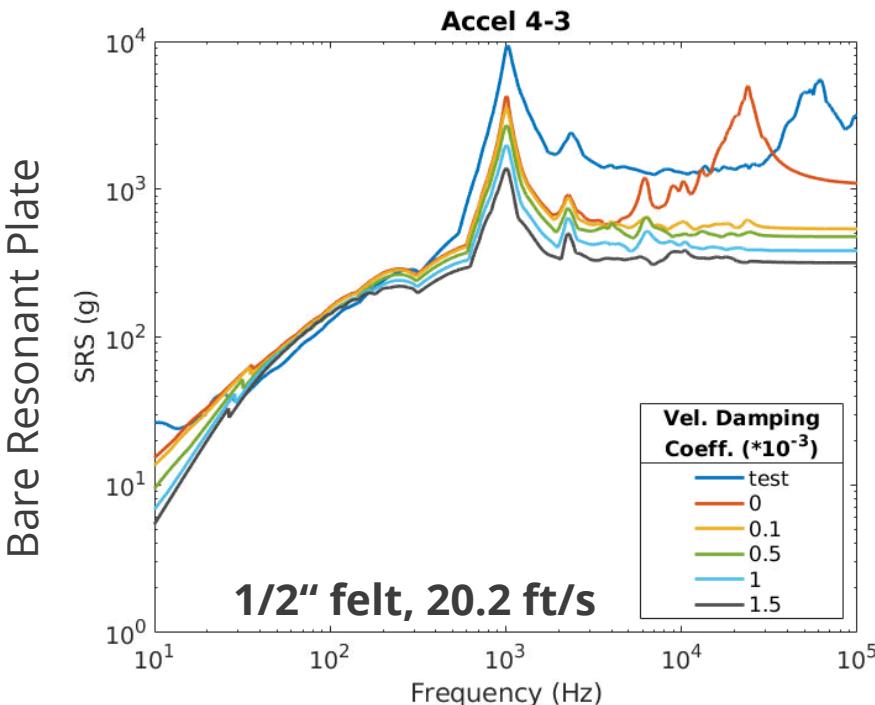
Results Comparison: Bare Plate with Damping Bars

- First resonant peak generally matches frequency band, but some discrepancies in amplitude.
- High-frequency amplitudes still present in test at ~60 kHz.
- Large amplitudes occur in the FEM around 25 kHz.
 - What causes this?
 - Damping bars/rubber?



Applying Velocity Damping to the Felt Volume

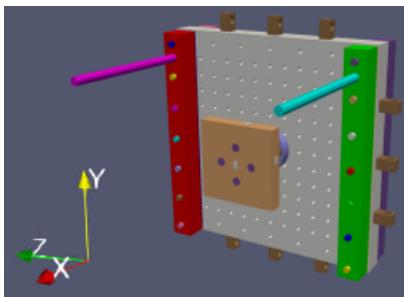
- The change in felt thickness was found to be the cause of the high-frequency amplitude in the simulation.
- The felt material model contains reflection waves in the Hopkinson bar simulation that the test does not.
- Adding a small velocity damping eliminates the high frequency content without effecting the main resonant amplitude.



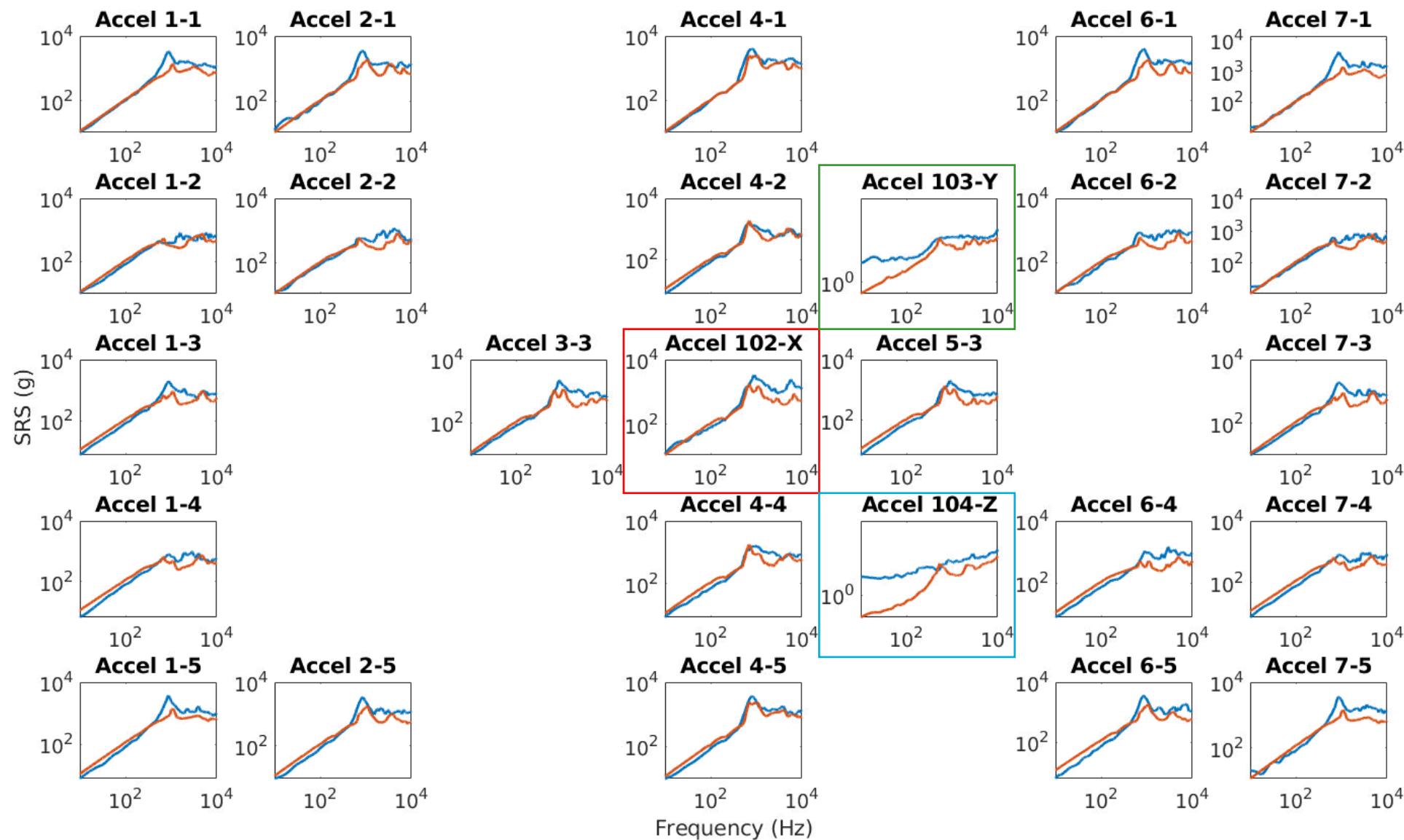
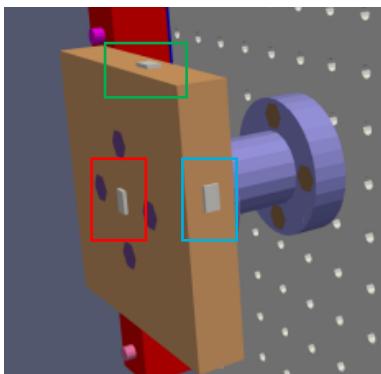
Results Comparison: Plate with Damping Bars and Component

Test configuration:

- Plate with damping and component
- 12" projectile
 - $v_i = 20.6$ ft/s
- 1/2" felt

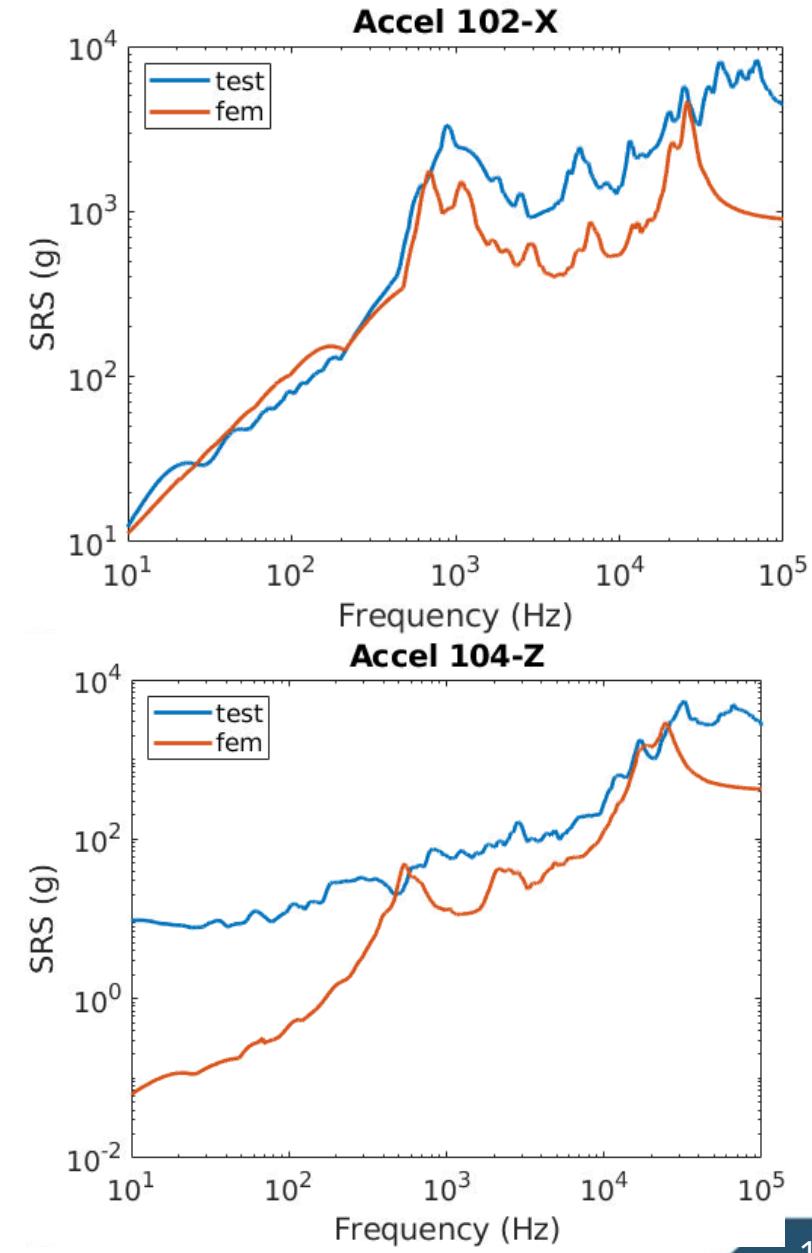
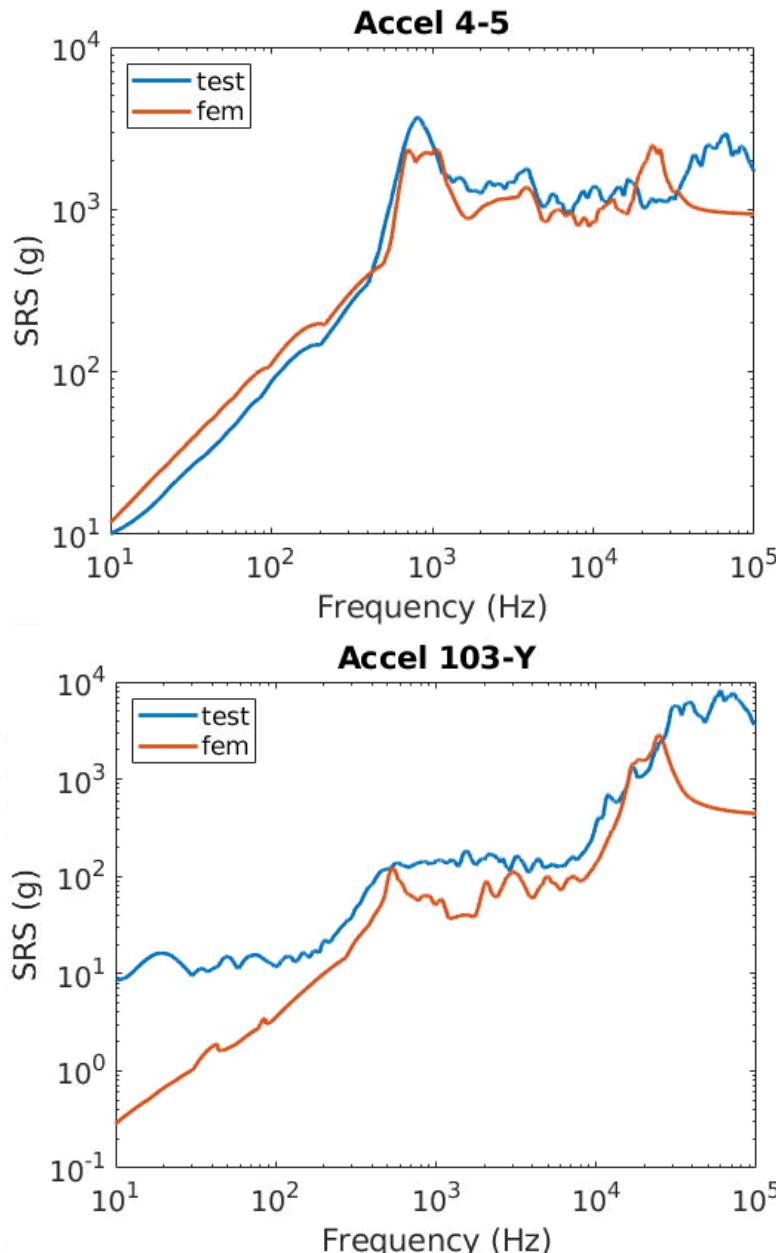


Accel 102, 103, 104 placement



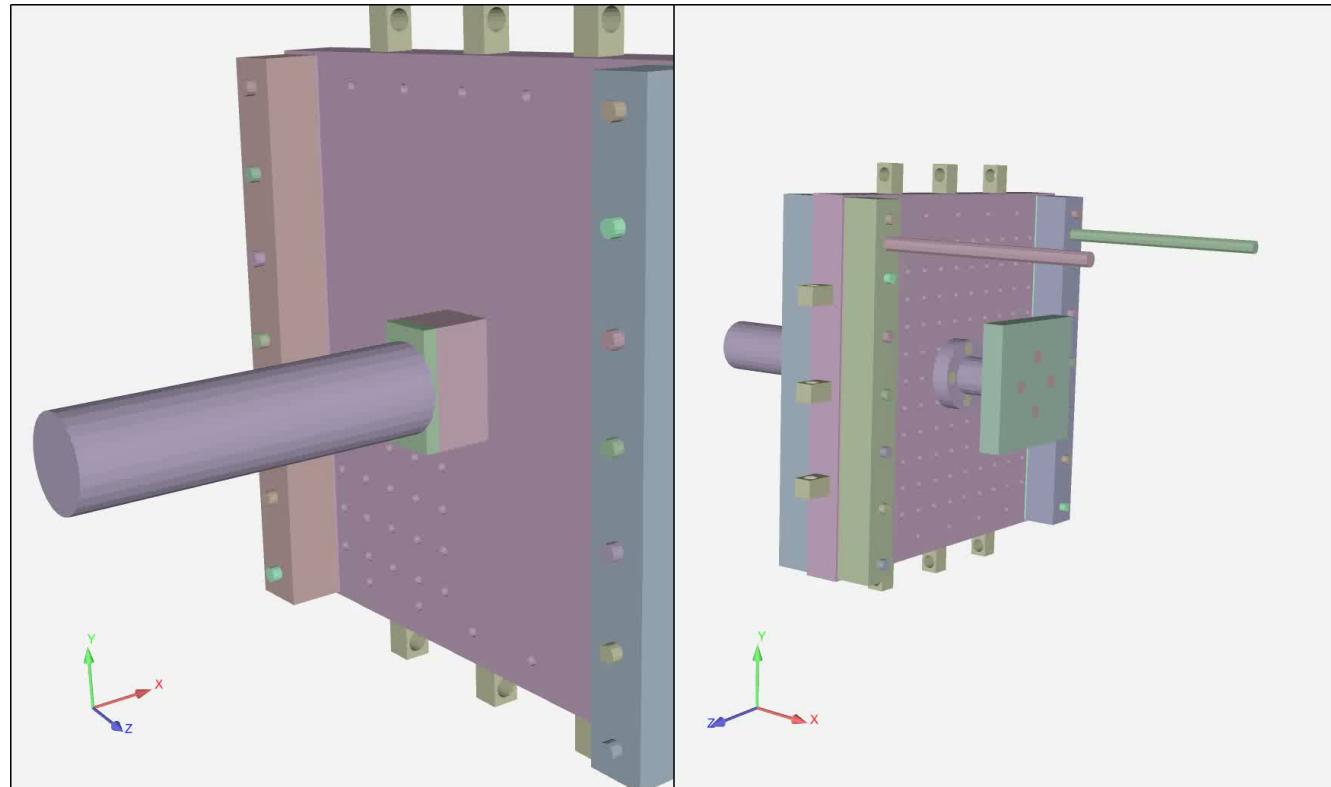
Results Comparison: Plate with Damping Bars and Component

- Accel 4-5 matches the test quite well. The first resonant peak is not sharp, but the general shape of the SRS is obtained through 10 kHz.
- Accel 102-X also matches the general shape of the test. The First resonant peak is split into two, and the amplitude is decreased between 1-10 kHz.
- Accel 103-Y and 104-Z have a small response at low frequencies.



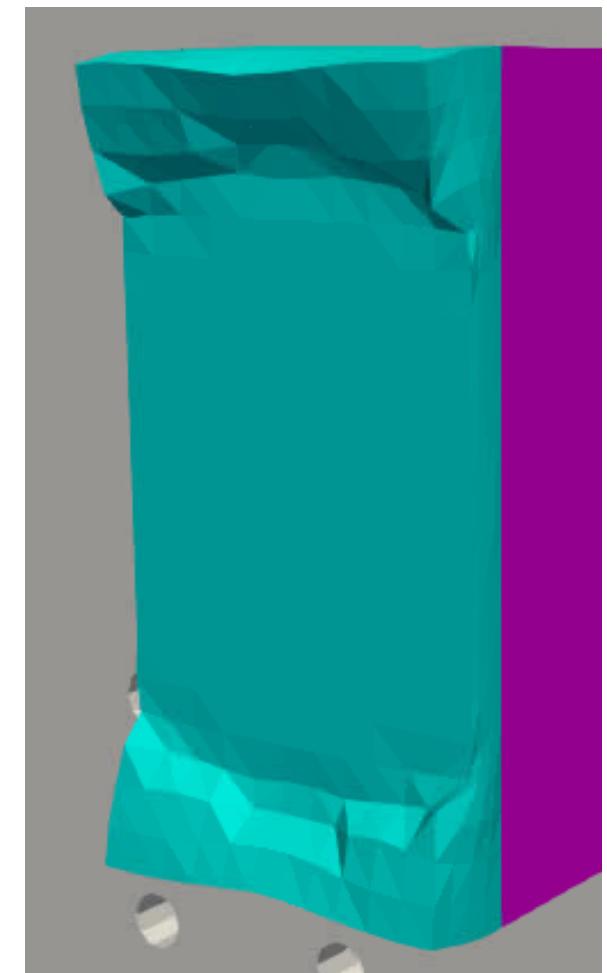
Conclusions

- A nonlinear model incorporating contact and friction, nonlinear materials, large deformations, and explicit dynamics was developed and simulated in Sierra SM.
- The resonant plate without damping bars and the component under test nearly identically matched the SRS response of the test through 10 kHz.
- Introducing velocity damping to the material felt volume eliminates .



Future work

- Optimize the felt material model to eliminate need of velocity damping
- Include support ropes to assess the importance of the boundary conditions on the simulation.
- Investigate the preload of the bolts on the damping bars.
- Offset the projectile/impact location to assess three-axis response.
- Design and optimize a fixture to meet specifications for a three-axis shock test.



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