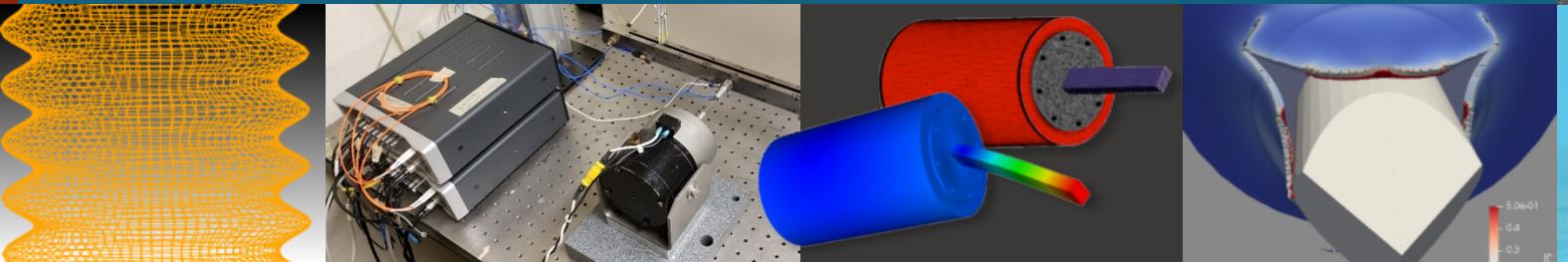




# Layered Plate Damping Systems



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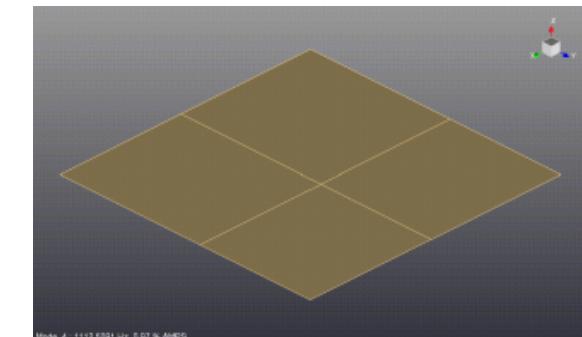
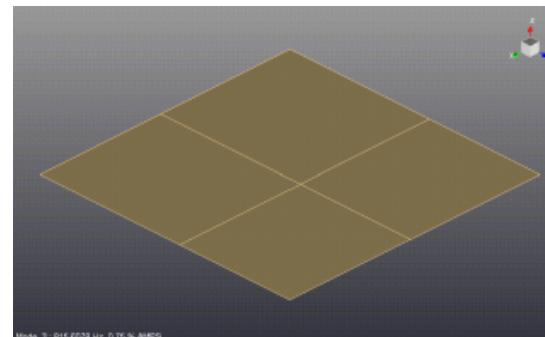
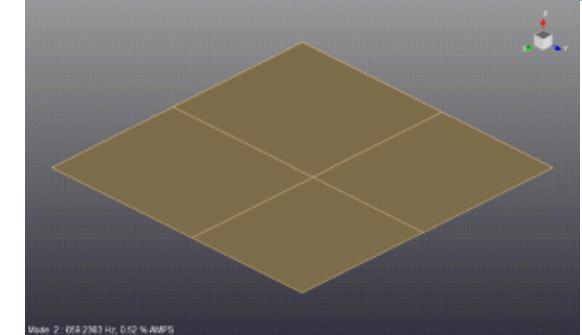
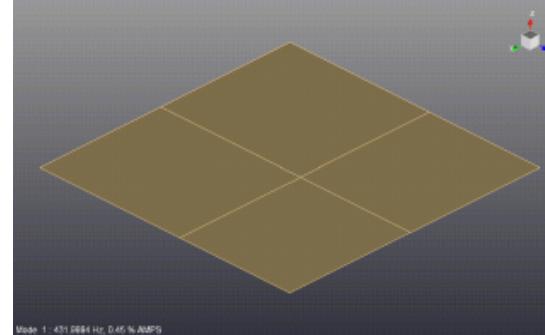
# Introduction



Objective: Determine the critical factors and their effects on layered plates joined by bolts

Method: Experimental modal analysis using a full factorial experimental design

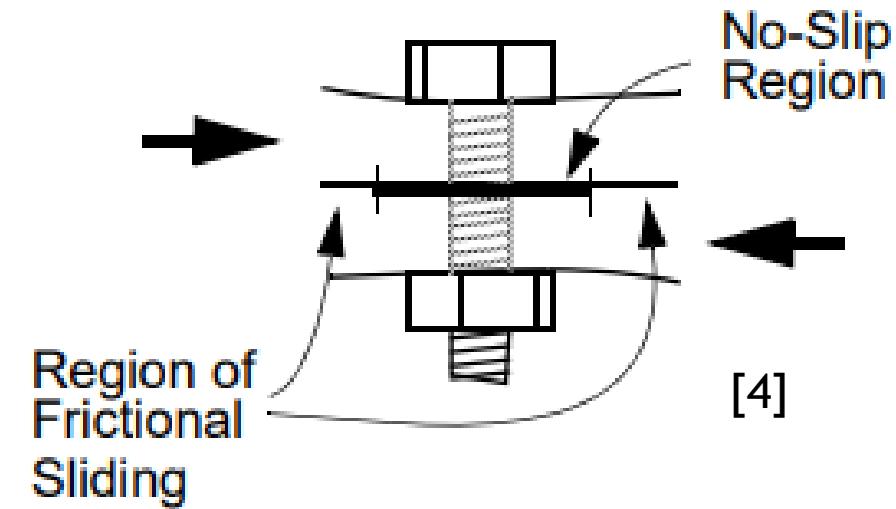
Motivation: Bolted plates and joints are omnipresent. Understanding the dynamic response of bolted plate systems are critical to their use.



# Critical Factors



- Damping is dominated by microslip
  - Microslip- the localized slip between two bolted plates - can be described by dry Coulomb friction
  - Upon excitation, the plate areas furthest from bolts will slip
  - The interface can also separate depending on direction of excitation (also known as slap)
- 5 factors across 2 levels (along w/ 32 unique configurations) selected to assess their relationship with two dynamic properties

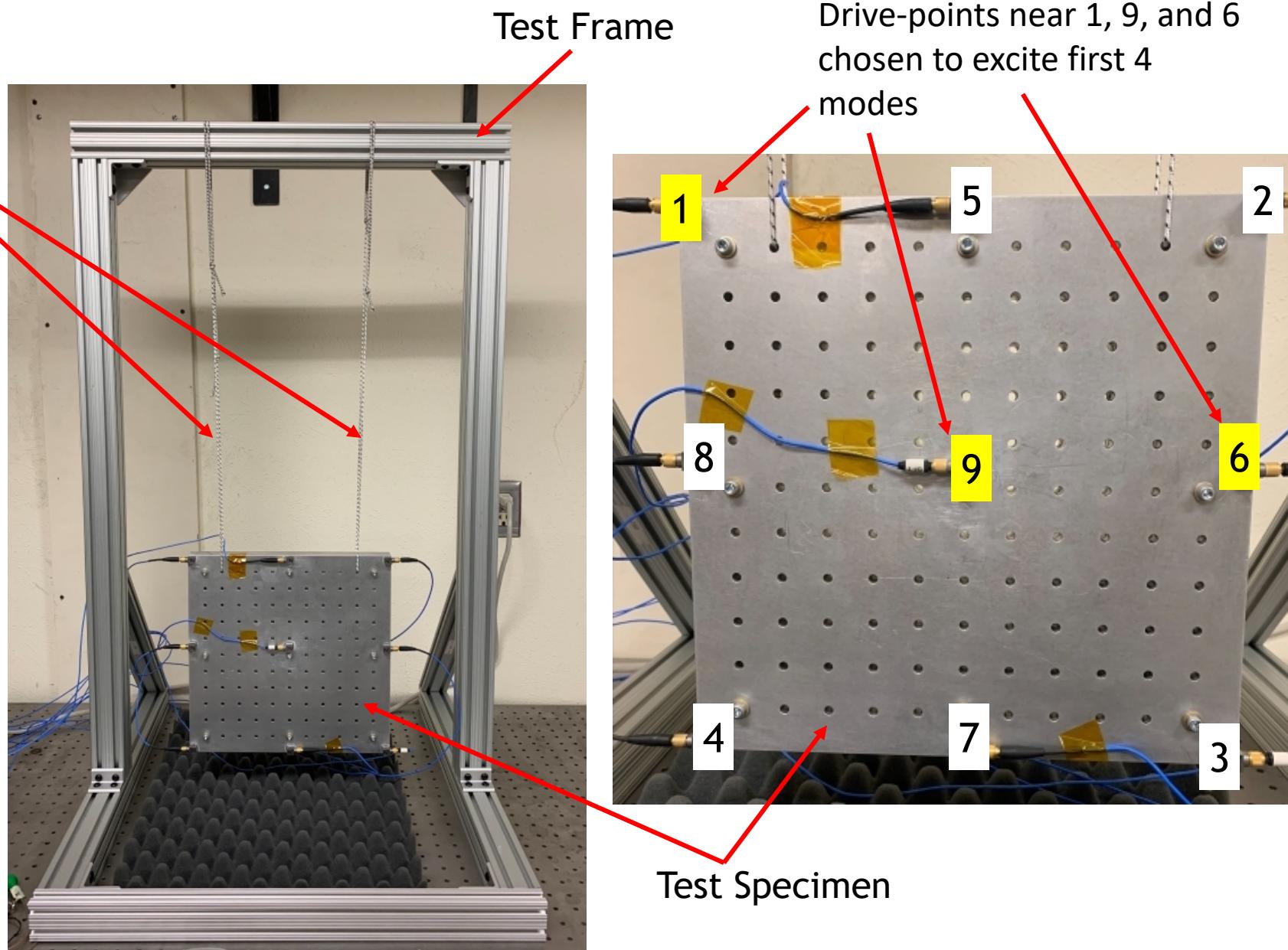
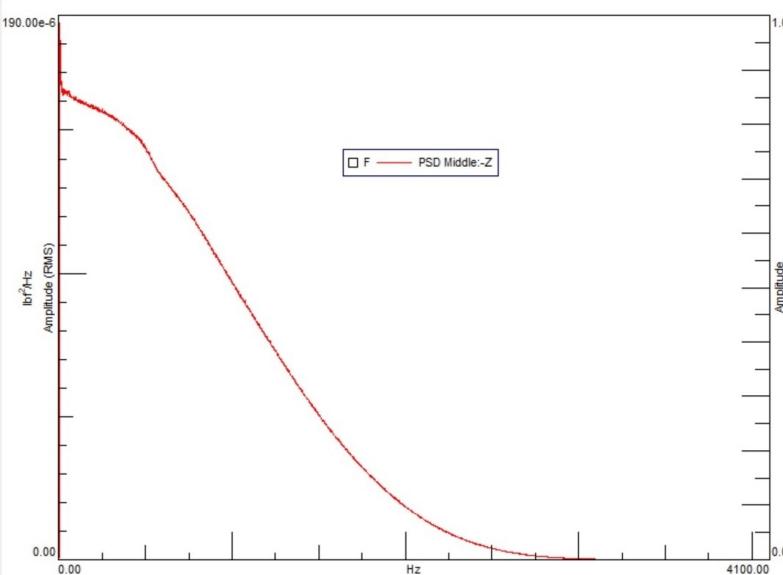


	Low	High	Expected Relationship with Frequency	Expected Relationship with Damping
Bolt Preload (in-lb)	25	50	Direct [2]	Inverse [2]
Bolt Spacing (in)	2	7.1	Unknown	Direct [3]
Total Thickness (in)	0.75	1.0	Direct	None (Minimal)
Number of Interfaces	1	2	Inverse	Direct
Impact Amplitude (lbf)	20-30	60-70	Direct	Direct [4]

# Experimental Test Set-up



- Free boundary conditions supported by bungee cords
- Accelerometer placement based on mode shapes of interest
- White plastic impact tip (084B04 PCB) inputs content in frequency range of interest

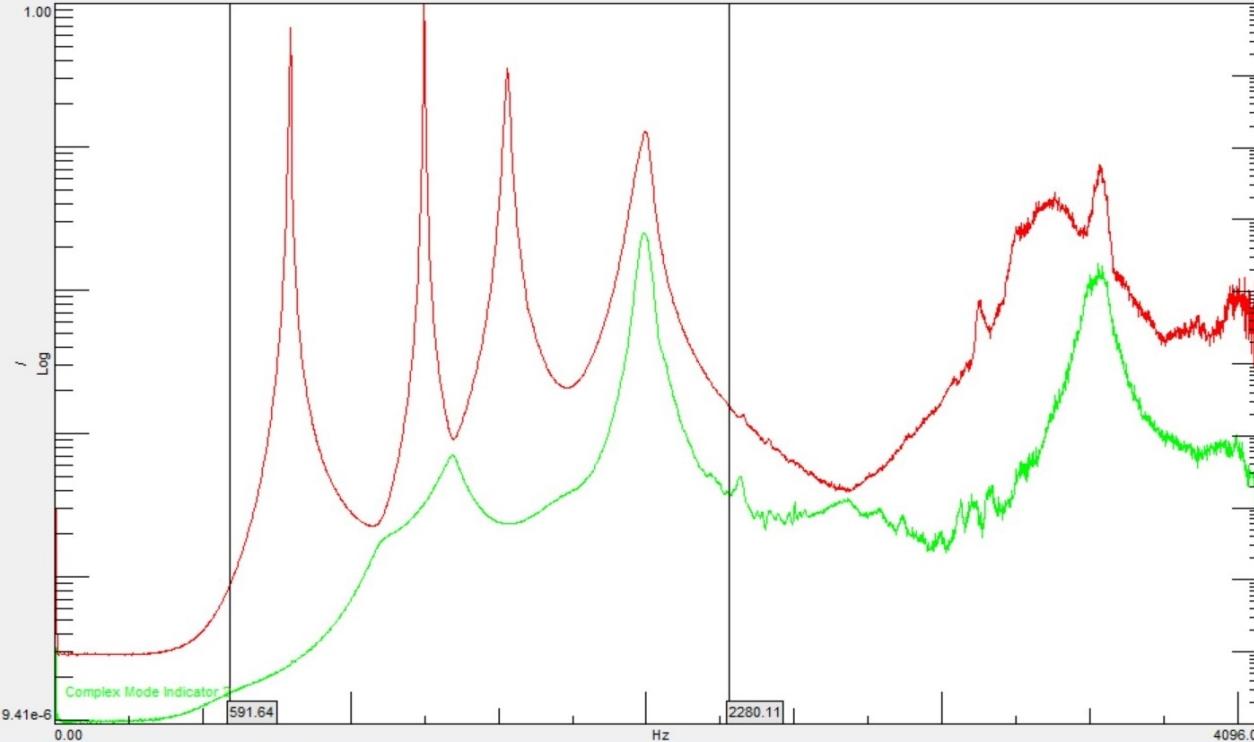


# Experimental Metrics and Evaluation

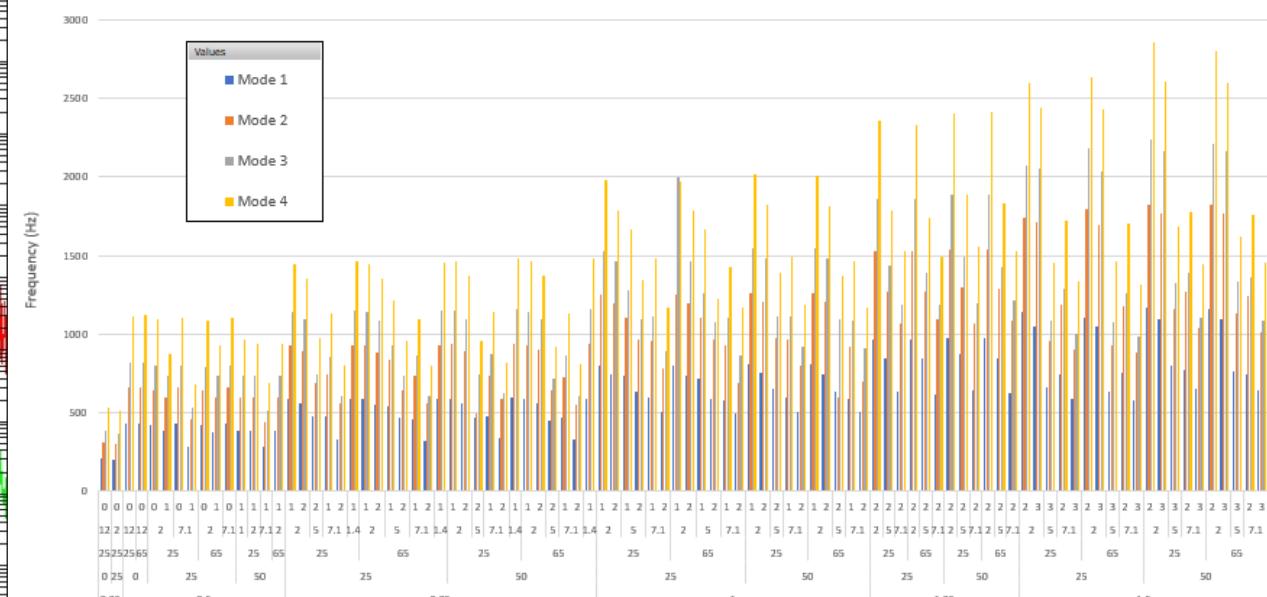


- Siemens LMS used to acquire data and extract natural frequencies, mode shapes, and modal damping via curve fitting
  - FRFs averaged over three impacts at each drive point (9 impacts) from measurements at each accelerometer
- Excel and Minitab used for data organization and visualization
  - Factor importance and effects on natural frequency and modal damping

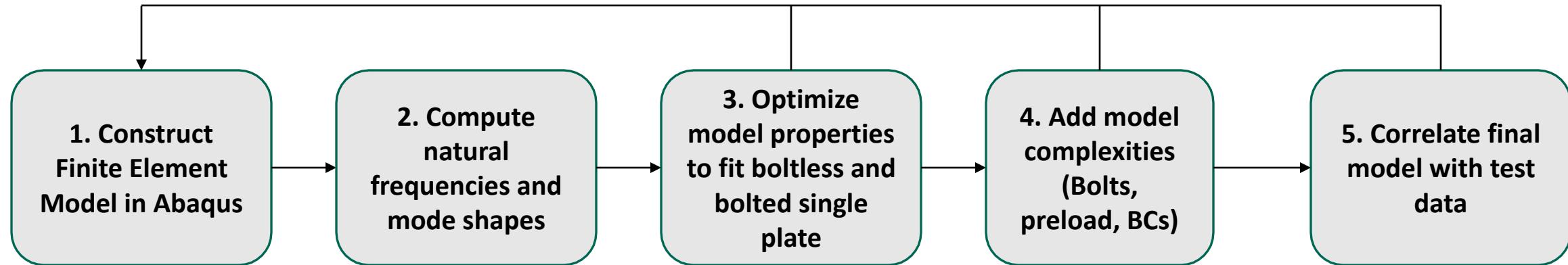
Averaged Frequency Response Function (FRF)  
(Configuration 5, Bolt Pattern 3, 25 in-lb Bolt Preload, Low Impact Amplitude)



Natural Frequencies over all 32 tests for 1<sup>st</sup> four modes plotted on a PivotChart in Excel

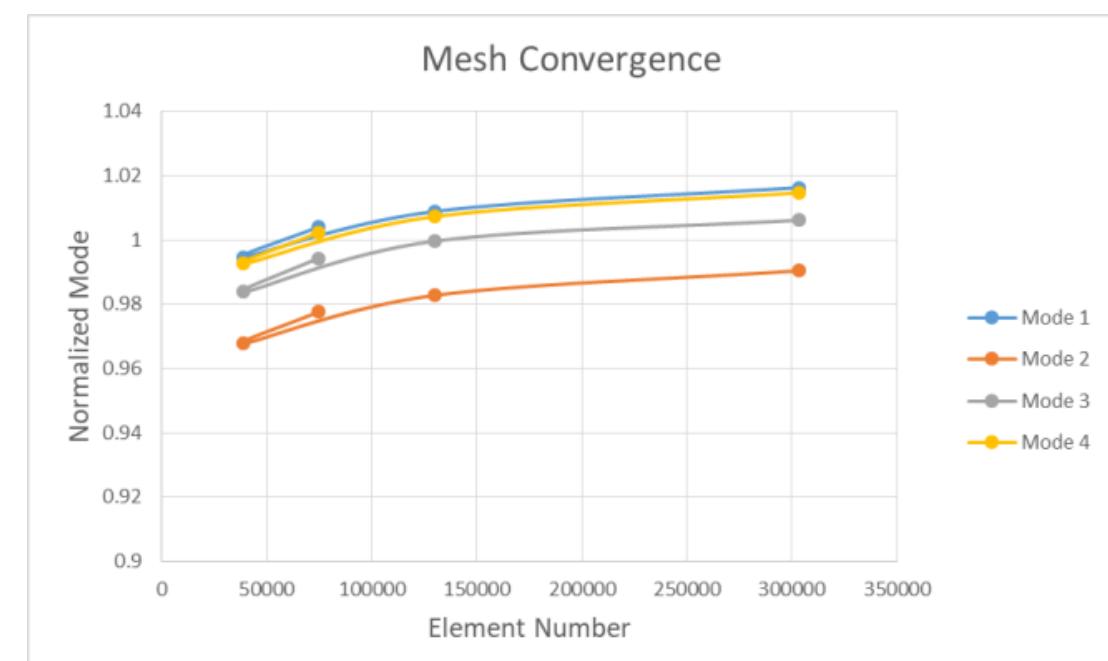


# Finite Element (FE) Modeling Process



- Bolted plate system modeled in Abaqus
- Frequency and Modal Dynamics steps in Abaqus Standard used during analysis
- Material property values reflect model properties after Step 3

	Aluminum (Plate)	Steel (Bolts)
Elastic Modulus $\left(\frac{lb}{in^2}\right)$	$10^7$	$2.8 * 10^7$
Density $\left(\frac{lb}{in^3}\right)$	0.093	0.28
Poisson's Ratio	0.33	0.29



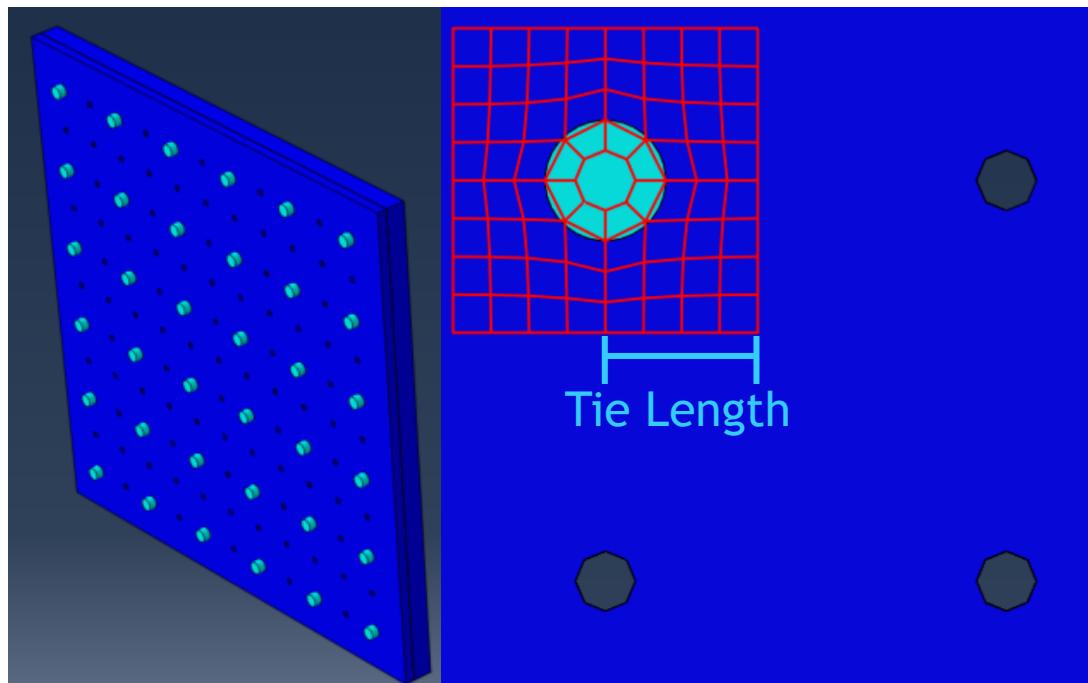
# Bolt Constraints and Connections



## Tie constraints

- Square partitions tie bolt heads to plate as well as adjacent plates together
- Tie area varied between configurations to assure frequency match

Configuration	3	4	5	6
Tie Length (in)	0.38	0.4	0.8	1

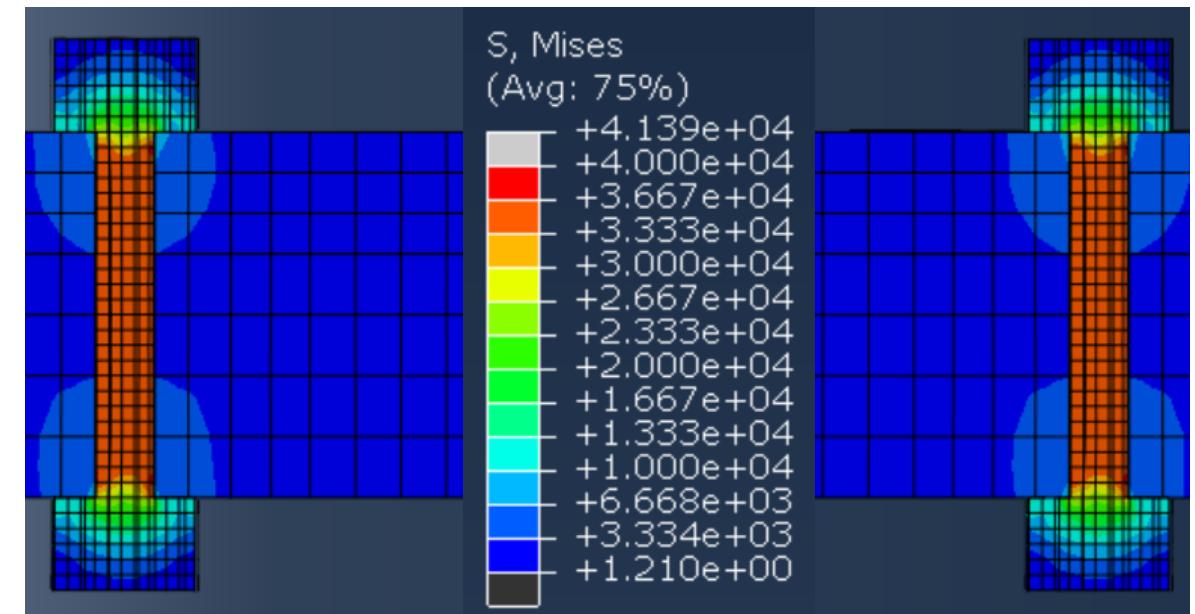


## Bolted Connections:

- Simplified, symmetric bolt used in analysis
- Preload ( $T$ ) related to clamping force ( $F$ ) via:

$$F = \frac{T}{K_i * d} \quad [5]$$

- Preload validation:
  - Symmetric stress profile obtained in FEA consistent with above formula after translation to force



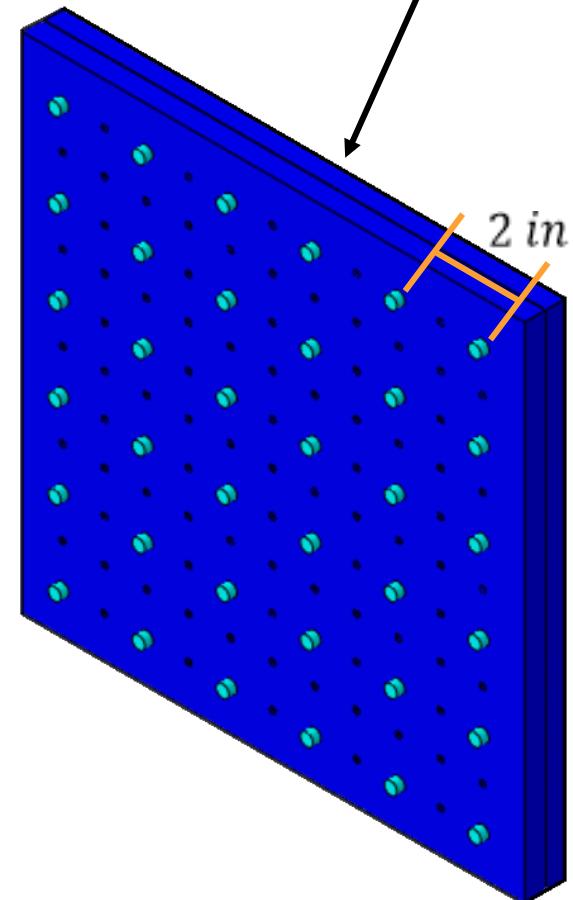
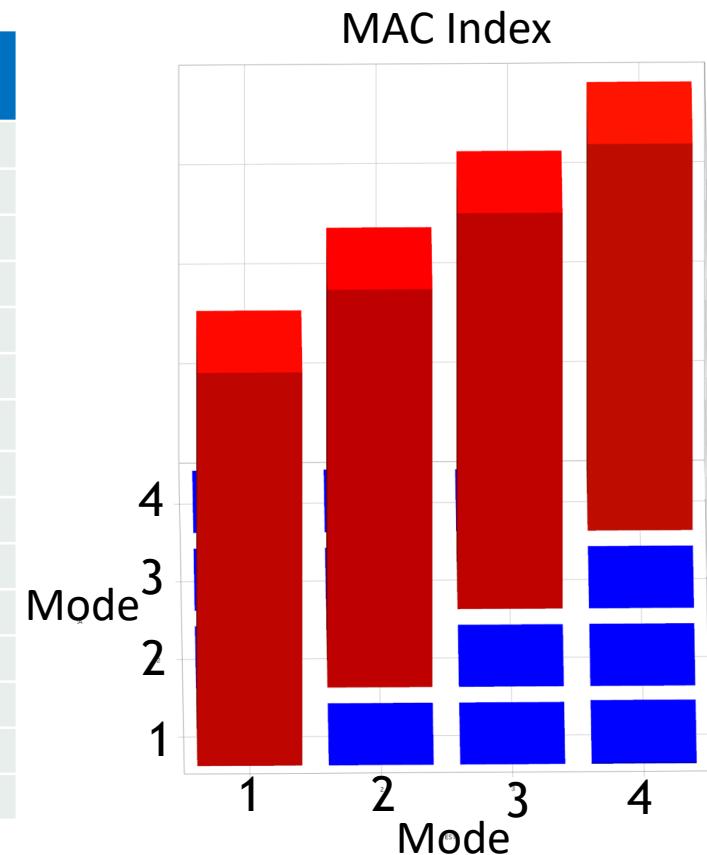
# Modal Comparison between FE and Experimental



Configuration	Thickness (in)	Number of Interfaces
3	0.5	0
4	0.75	2
5	1	1
6	0.75	1

- Requisite change in density from  $0.098 \frac{lb}{in^3}$  to  $0.093 \frac{lb}{in^3}$  ( $-5.1\%$ )
- Configuration used for material property calibration
- Modal Assurance Criterion (MAC) values suffers dramatically at degenerate mode

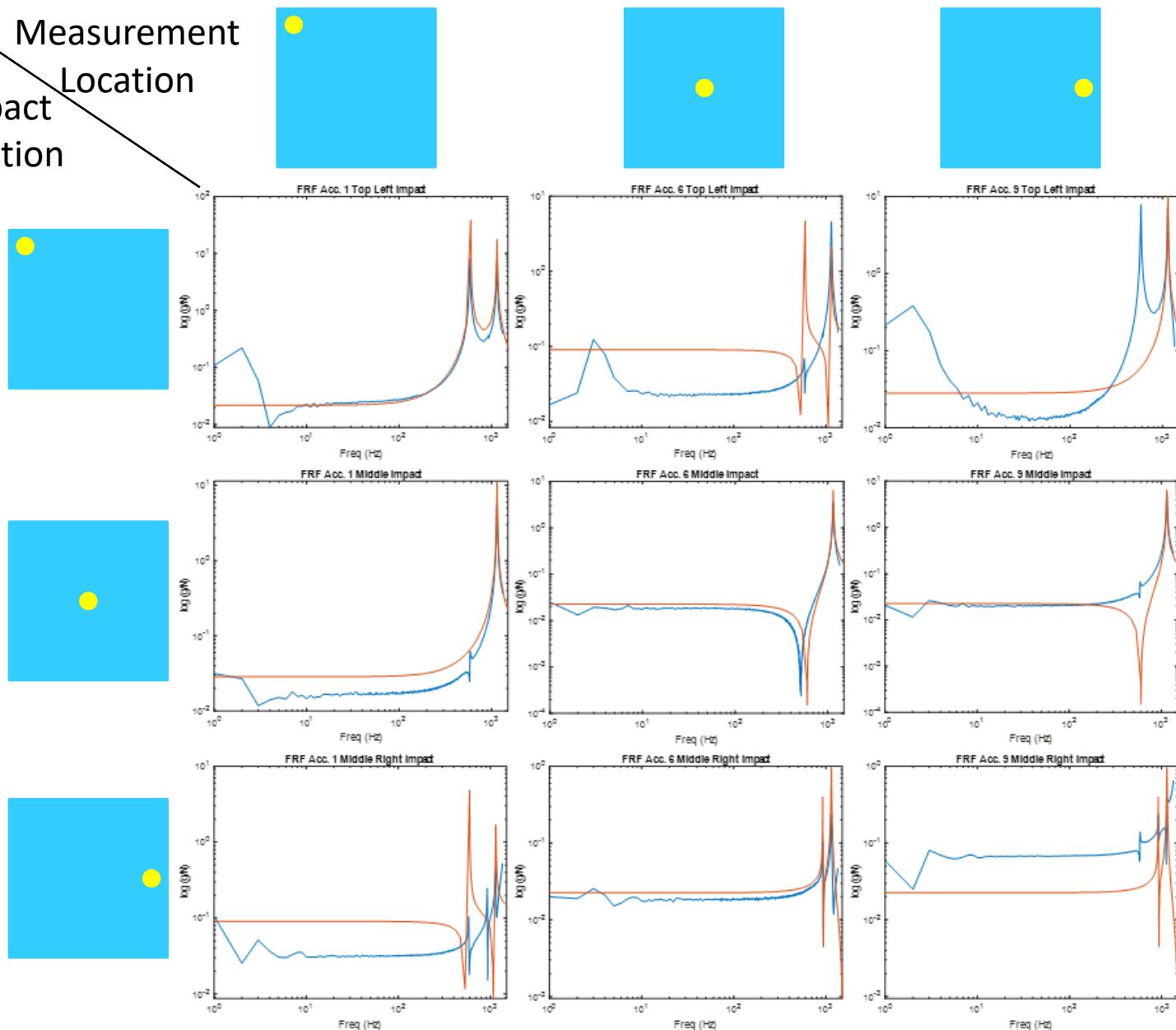
Configuration	Mode	Percent Difference (%) FE vs Exp Natural Frequency	MAC (%)
3	1	0.464	99.5
4	1	1.95	99.7
5	1	-0.81	98.7
6	1	1.13	99.1
3	2	-2.239	99.9
4	2	1.13	99.9
5	2	-2.08	100
6	2	-1.21	99.8
3	3	-0.564	99.4
4	3	1.08	99.7
5	3	-1.14	99.5
6	3	0.07	99.6
3	4	0.228	12.8
4	4	1.45	63.3
5	4	-1.9	84.5



# Frequency Response Functions (FRFs)



Measurement Location  
Impact Location  
Impact Location



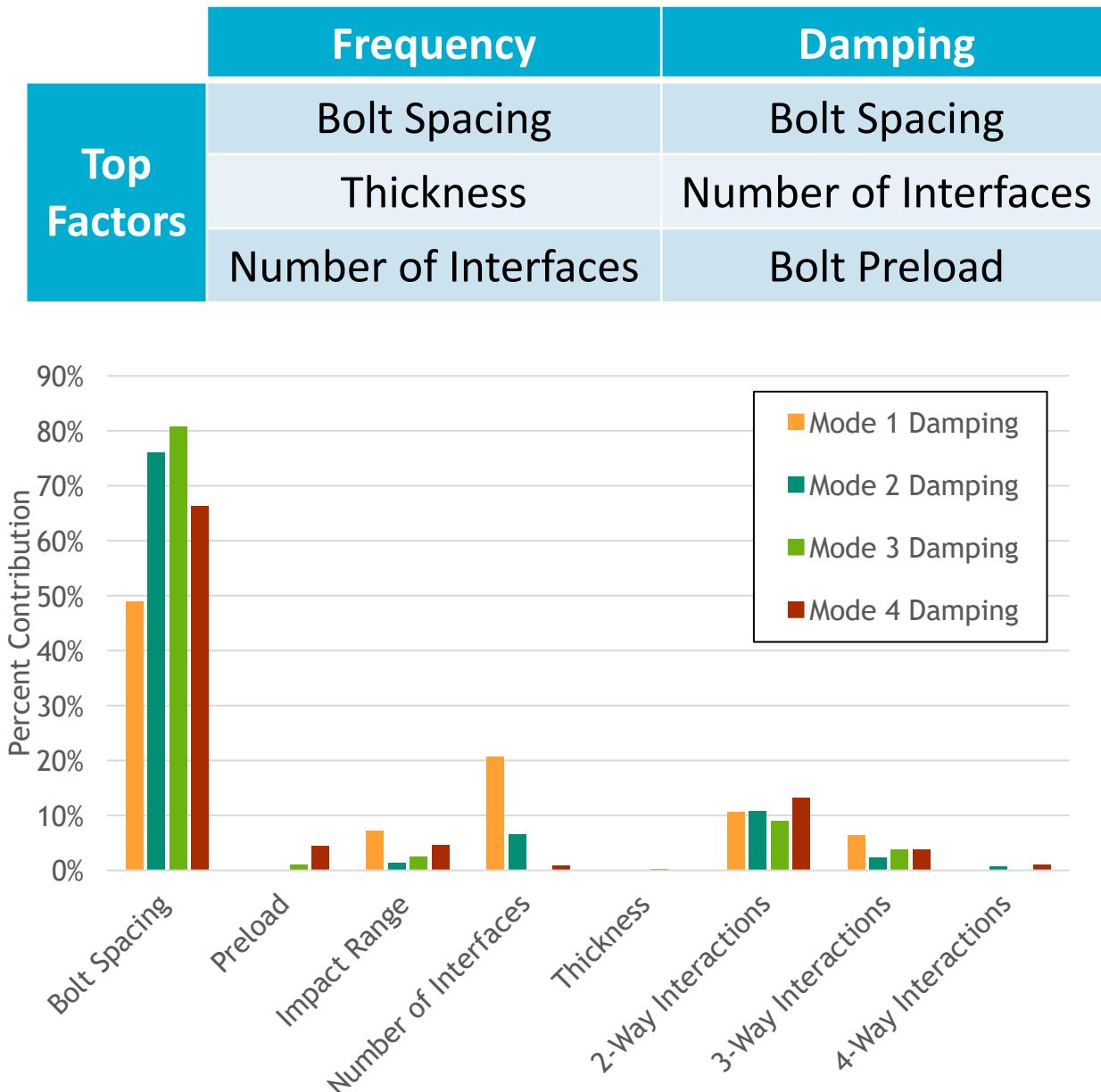
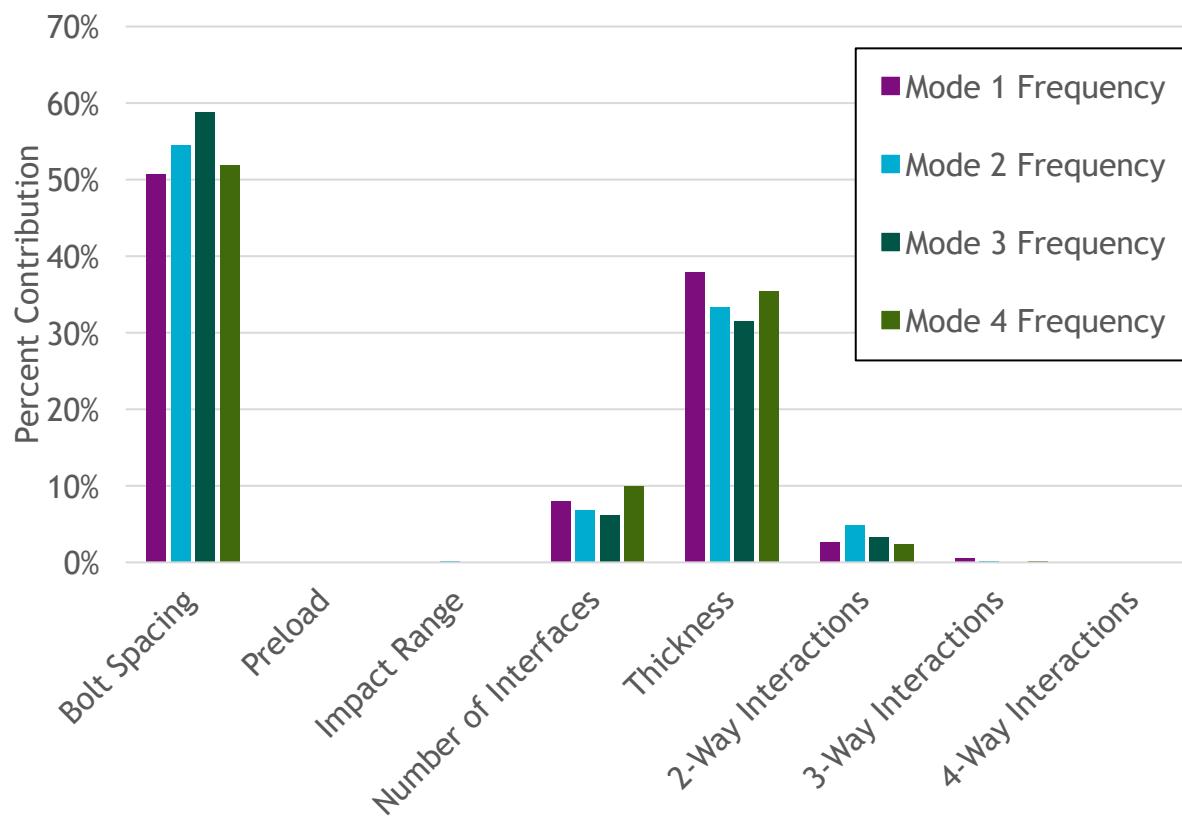
— Experimental  
— Simulation

- Drive-point experimental FRFs matched in finite element by adjusting tie length on the bolt constraints
- Good agreement at natural frequencies
- Low fidelity at frequencies approaching zero (DC) as result of experimental set-up

# Factor Importance



- Bolt preload and impact amplitude has negligible effect on frequency
- Thickness has minimal effect on damping
- Factor interactions much higher for damping than for frequency
- 2-way factor interactions not negligible for either indicator

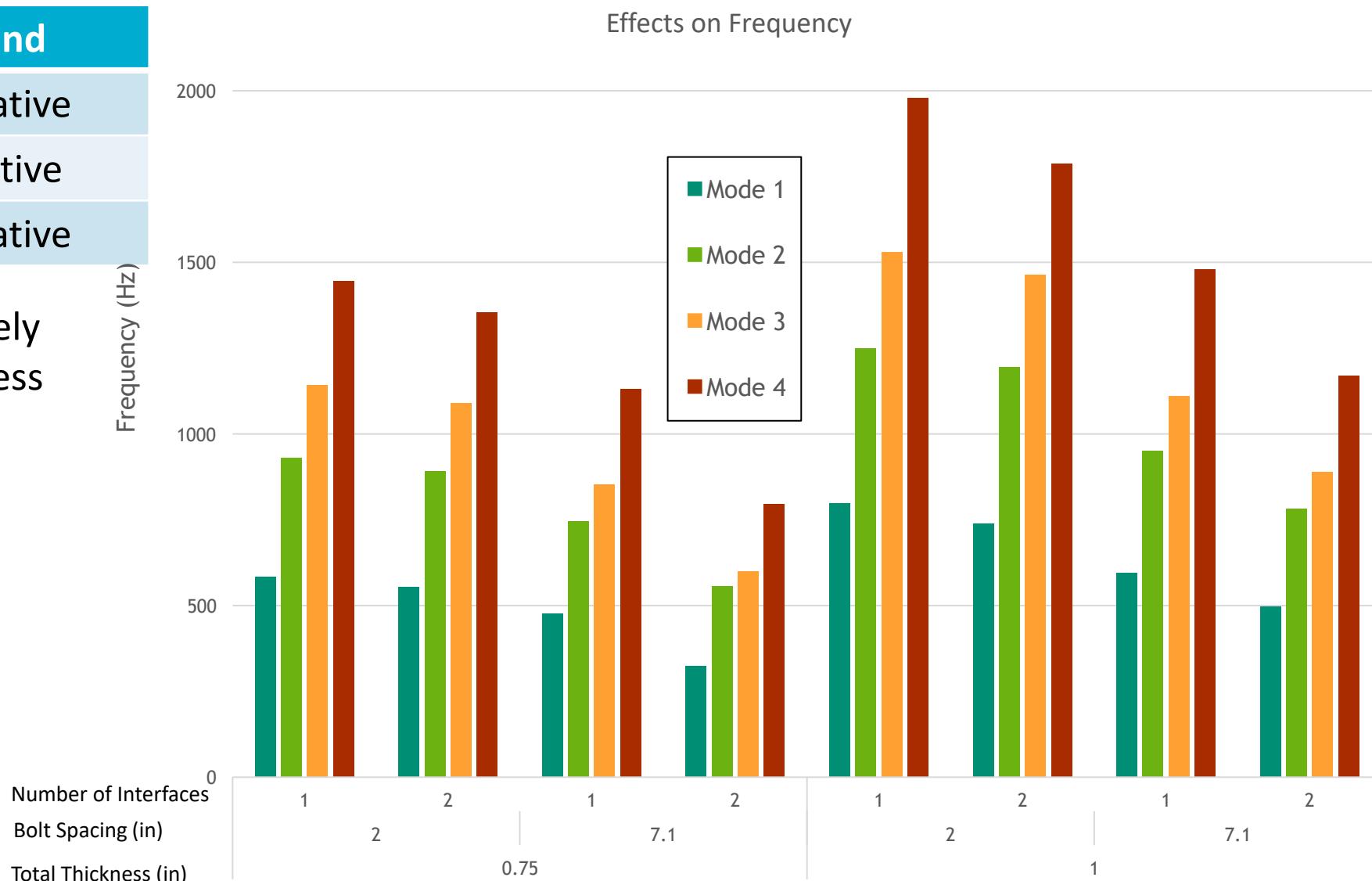


# Factor Effects on Frequency



Factor	Trend
Bolt Spacing	Negative
Thickness	Positive
Number of Interfaces	Negative

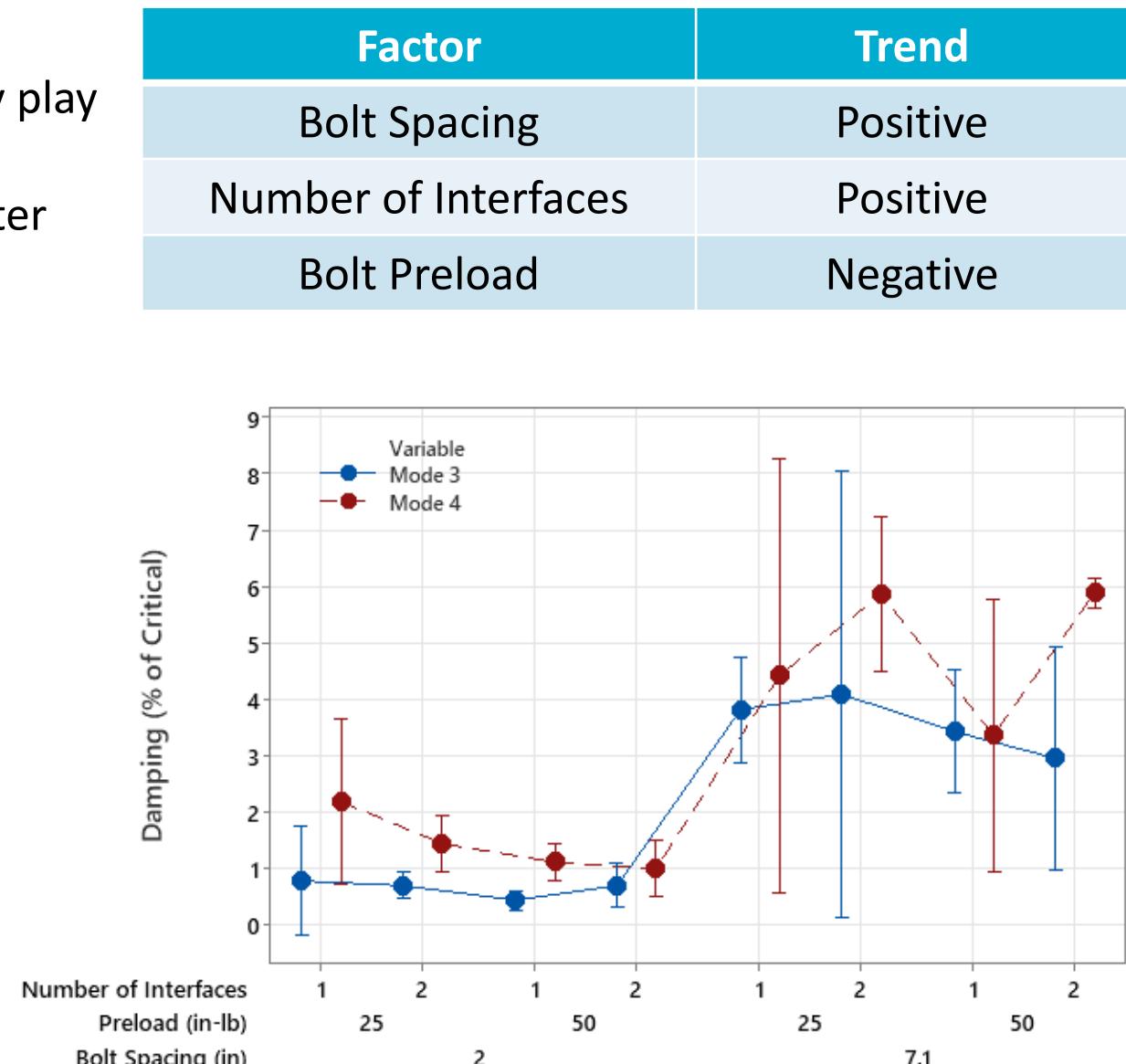
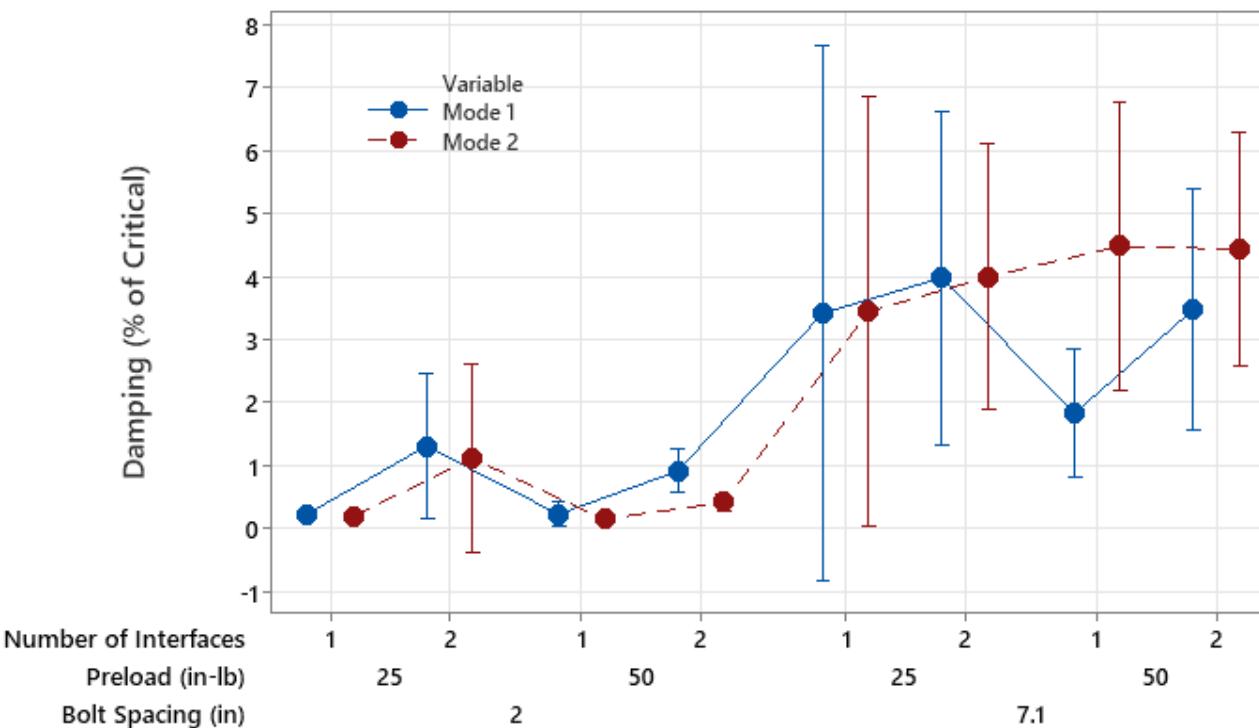
- Frequency shares approximately cubic relationship with thickness
- As bolt spacing increases, the bolted system becomes less coupled and less stiff
- More interfaces for the same thickness results in a less coupled system



# Factor Effects on Damping



- Variability in damping also correlates positively with bolt spacing
- Lack of clear trends indicate factor interactions likely play key role in damping
- Decreased coupling between plates introduces greater amounts of slip at the interfaces and thus greater damping



# Summary



- Full factorial experimental design over 5 factors and 2 levels assessed the factor importance and effects on modal frequency and damping
- Bolt spacing is the most important factor in predicting both natural frequency and damping
- Factor interactions affect modal damping much more than it does the natural frequency
- Finite element and experimental mode shapes have good agreement for first three mode shapes using tie constraints as bolt connections but disagree at the first asymmetric mode



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# Questions?