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# A trade off study of metal and composite turbines using fluid-structure interaction modeling

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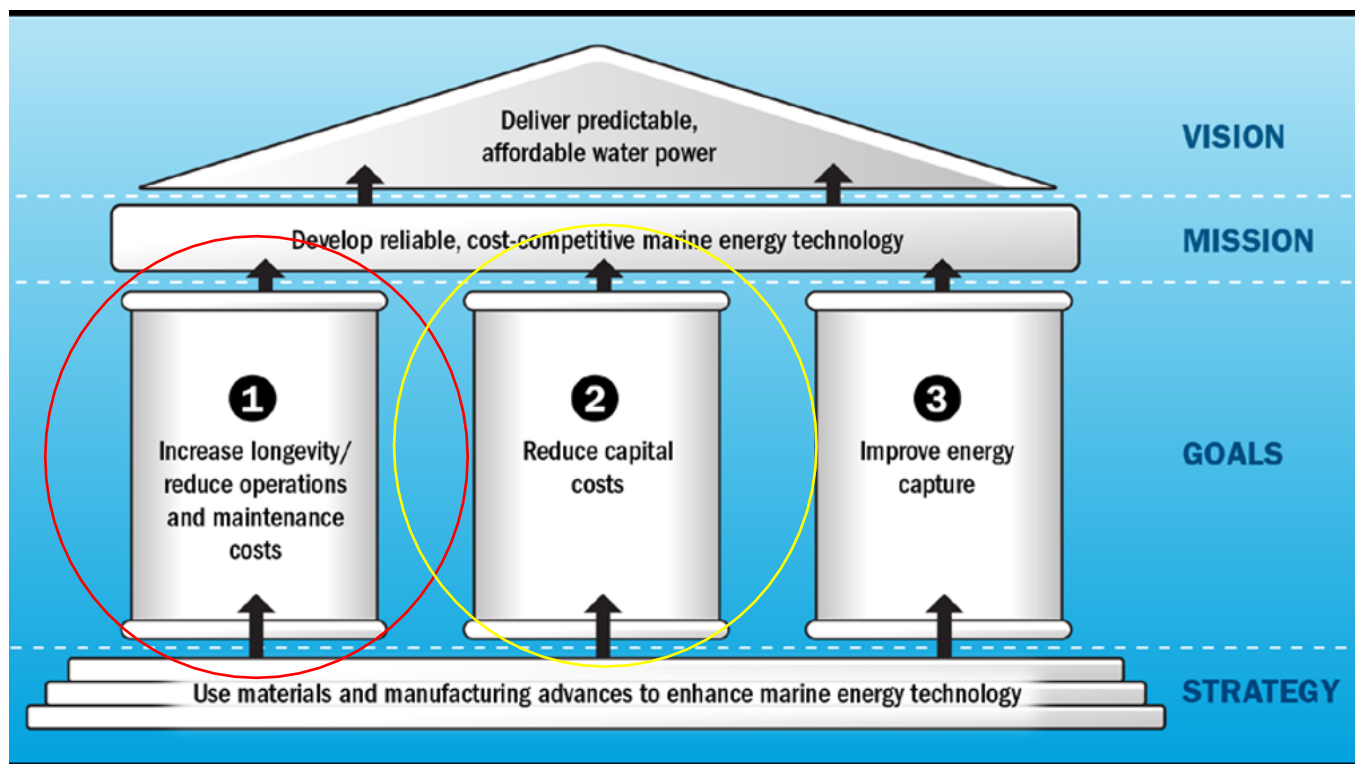
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# Motivation



WPTO's vision for materials and manufacturing in marine energy\*

- The materials selected for marine energy devices must be able to perform under the harsh marine environment.
- WPTO draft Materials and Manufacturing Strategy<sup>^</sup> identified FSI for non-rigid blades, as a near- and mid-term research needs.
- Current Energy Converter (CEC) design studies often *only* include Computational Fluid Dynamics (CFD) modeling with a simple **rigid blade assumption** or Finite Element Analysis (FEA) with simplified load distributions. This simplification can cause errors in predicting the device structural performance, reliability and LCOE.
- An FSI study takes into account the hydro-elastic behavior of the blade material, **yield time-accurate solutions for loading and performance of a deforming rotor**, which could be critical for understanding structural performance and failure modes.

\*DOE WPTO (2022) Summary Report: October 5, 2021 Workshop on Materials & Manufacturing for Marine Energy Technologies; May 2022

<sup>^</sup>DOE WPTO (draft) Materials and Manufacturing Strategy for Marine and Hydrokinetic Energy Technologies Research & Development



# Objectives

**Objectives:** Perform FSI simulations for a reference tidal turbine (DOE Reference Model 1) made of metal and composites (e.g., FRP) and compare structural performance and cost

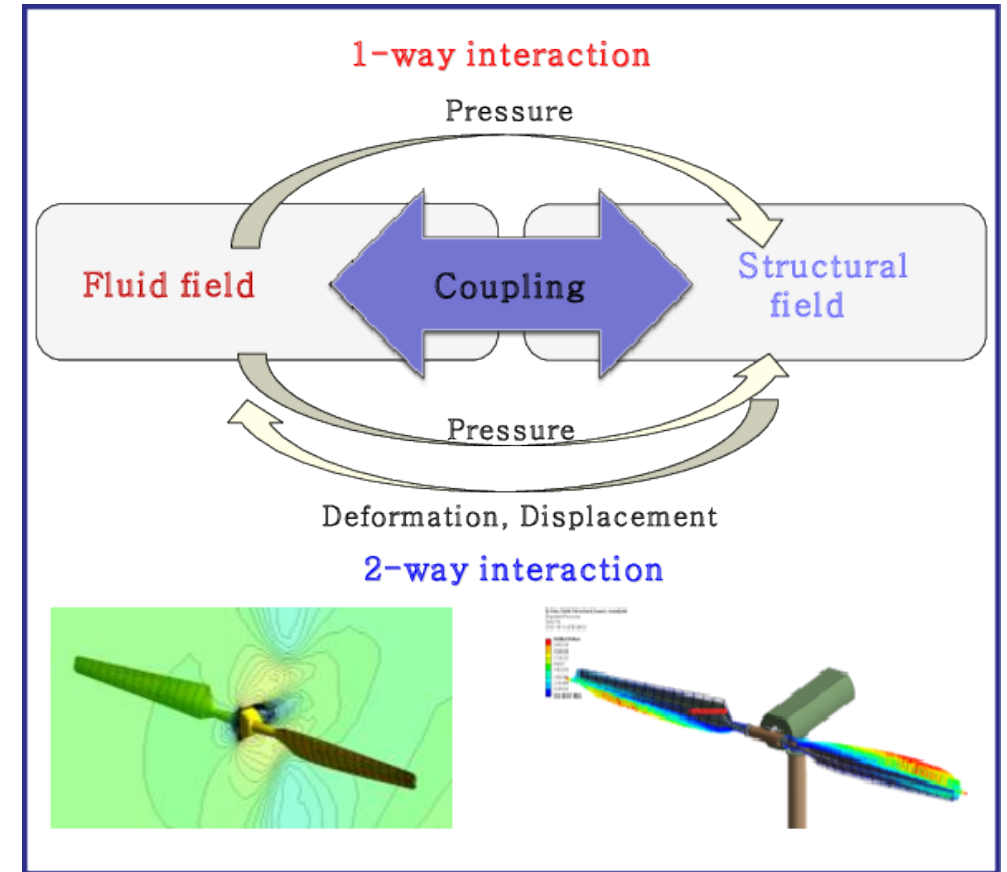
Structural performance metrics to observe include: deflection, stresses, ultimate limit state, fatigue limit state, vibration (flutter)

## Project Plan (3 years):

**2022:** CFD model development, FSI simulations for metal blades (lab-scale)

**2023:** FSI simulations for composite blades (lab-scale) & metal blades (full-scale)

**2024:** FSI simulations for composite blades (full-scale) & final cost/LCOE calculations



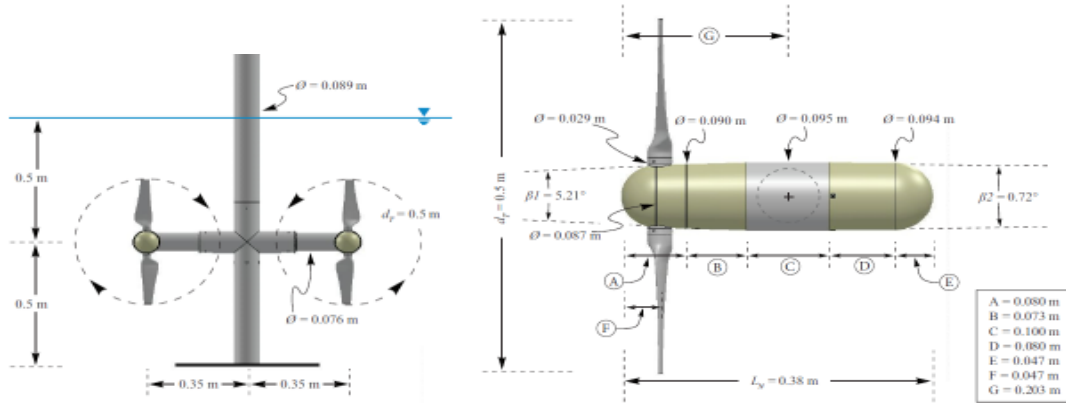
Fluid-structure interaction concept\*



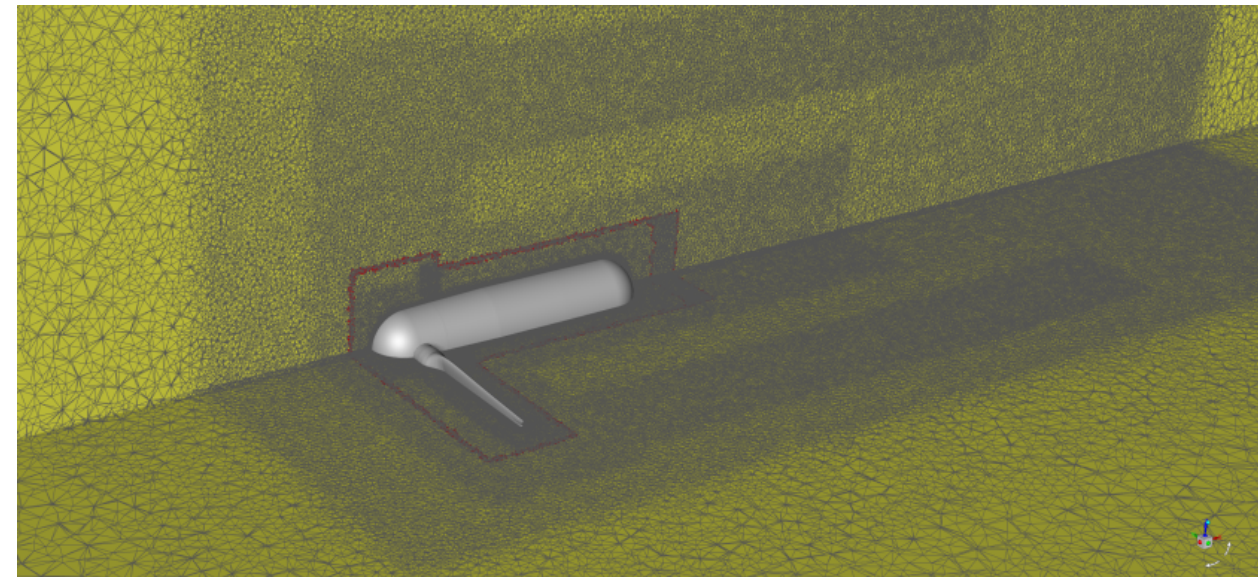
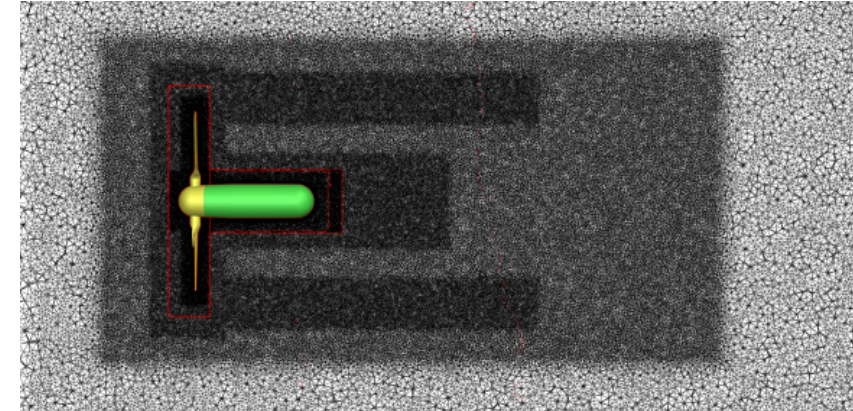
# CFD Setup

## Computational Mesh (Medium grid)

- Tetrahedral mesh with overset multi-blocks, 29.4M cells
- No-slip wall: rotor, nacelle, bottom and right side
- Free surface effect is ignored (Slip wall)
- SST k-omega model
- $y^+ = 1.4$  on the rotor and nacelle wall
- Simulated on 128-516 cores (3-7 days)



**1:40 scale RM1 turbine\***



**Computational Mesh for rotor and nacelle overset blocks and background domain**

\*Hill, C.; Neary, V.S.; Guala, M.; Sotiropoulos, F. Performance and Wake Characterization of a Model Hydrokinetic Turbine: The Reference Model 1 (RM1) Dual Rotor Tidal Energy Converter. *Energies* 2020



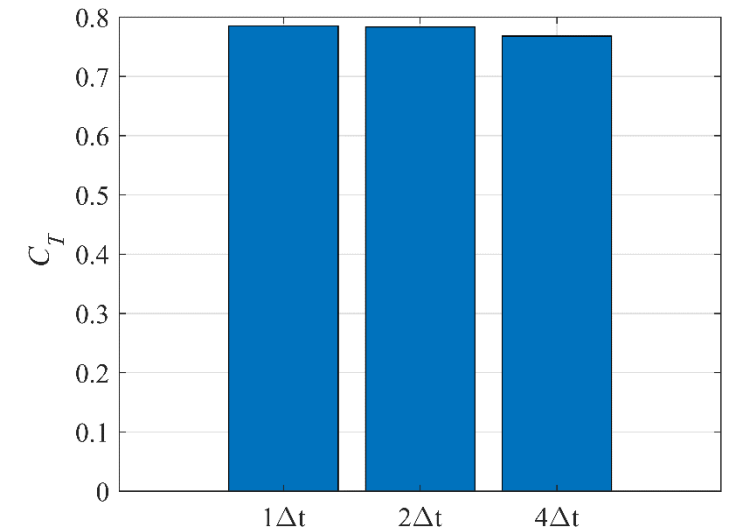
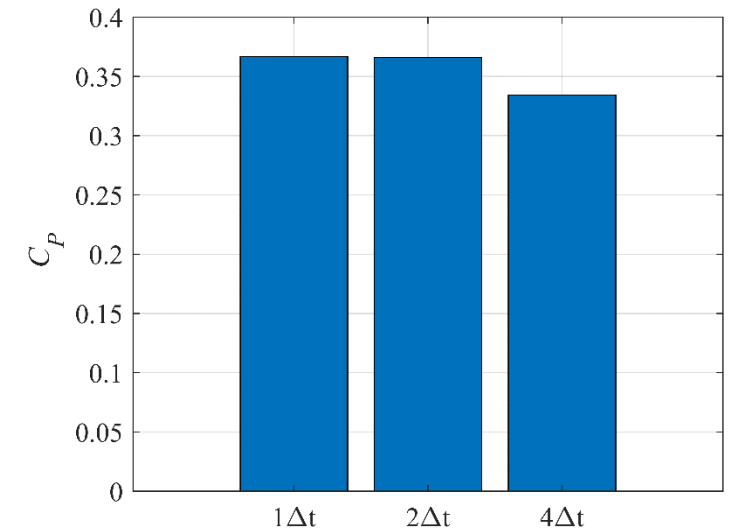


# Temporal Convergence

## Timestep size dependency (w/o blockage)

	Time step size	$C_P$ (diff, %)	$C_T$ (diff, %)
$N_1$	1° rotation per $\Delta t$	0.3667 (-)	0.7850 (-)
$N_2$	2° rotation per $\Delta t$	0.3660 (0.20)	0.7833 (0.22)
$N_3$	4° rotation per $\Delta t$	0.3343 (8.86)	0.7681 (2.15)
$U_{k_1}$		0.008%	0.054%

$U_{k_1}$  is uncertainty of  $N_1$  obtained from the method of Stern et al. (2006); and Xing and Stern (2010)



Estimated  $C_P$  and  $C_T$  depends on the time step size

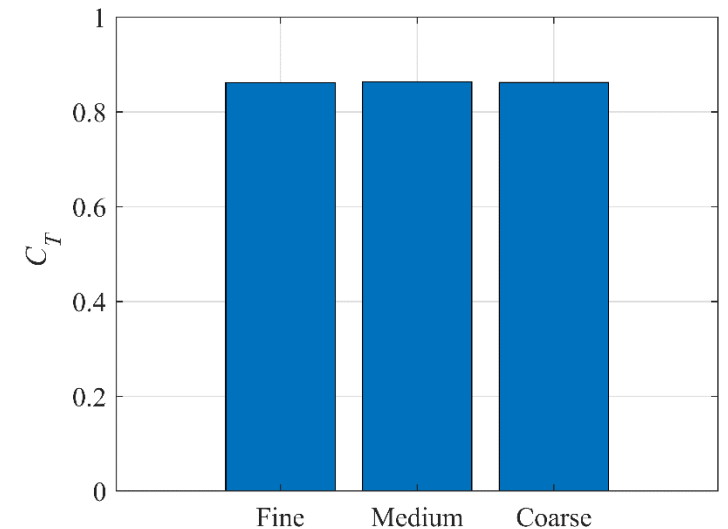
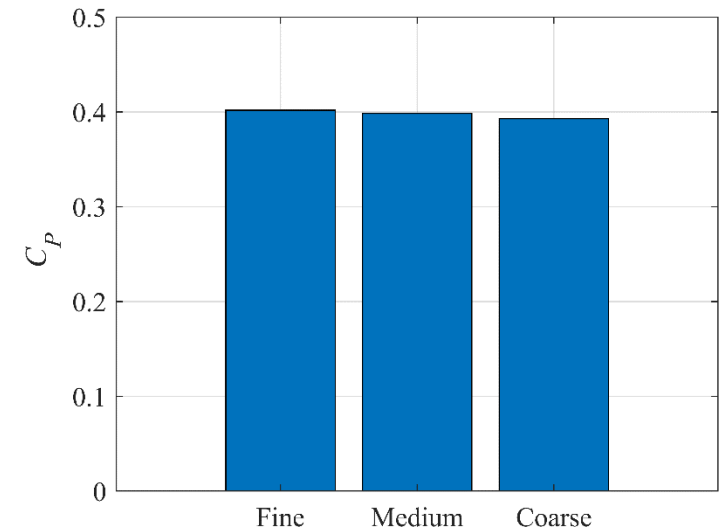


# Spatial Convergence

## Mesh size dependency study (w/ blockage)

	# of cells	$C_P$ (diff, %)	$C_T$ (diff, %)
$G_1$	66.2 $M$ ( $y^+ = 1$ )	0.4018 (-)	0.8617 (-)
$G_2$	29.4 $M$ ( $y^+ = 1.4$ )	0.3984 (0.83)	0.8632 (0.18)
$G_3$	14.5 $M$ ( $y^+ = 2$ )	0.3928 (2.24)	0.8622 (0.06)
$U_{k_1}$		1.007%	-

$U_{k_1}$  is uncertainty of  $G_1$  obtained from the method of Stern et al. (2006); and Xing and Stern (2010)



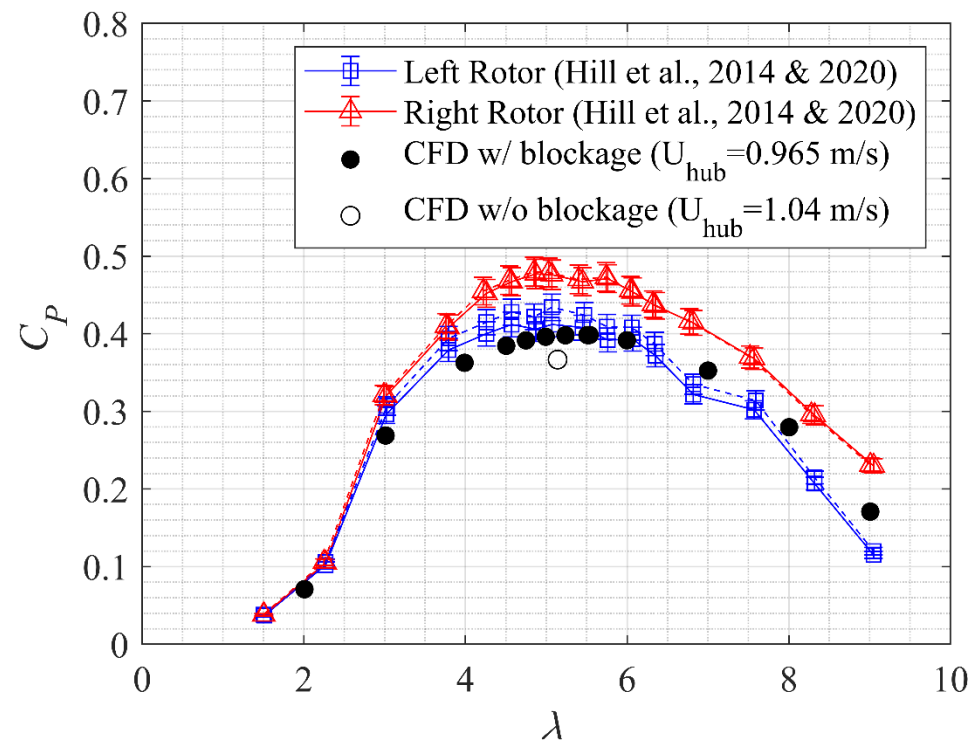
Estimated  $C_P$  and  $C_T$  depends on the mesh size



# Turbine Performance

## Coefficient of power

- Discrepancy between CFD w/o blockage and Exp. (Hill et al, 2014 & 2020) results due to the extensive blockage effect (14.3%)



@ 204 rpm	$C_p$	Uncertainty
Exp. Left Rotor (TSR = 5.07)	0.412, 0.434	3.9 %
Exp. Right Rotor (TSR = 5.03)	0.476, 0.479	
CFD w/o blockage (TSR = 5.14)	0.367	
CFD w/ blockage (TSR = 5.54)	0.402	

Measured and estimated  $C_p$  vs.  $\lambda$  (coefficient of power vs. tip-speed ratio).  
Solid and dashed lines are from Hill et al, 2014 and 2020, respectively)

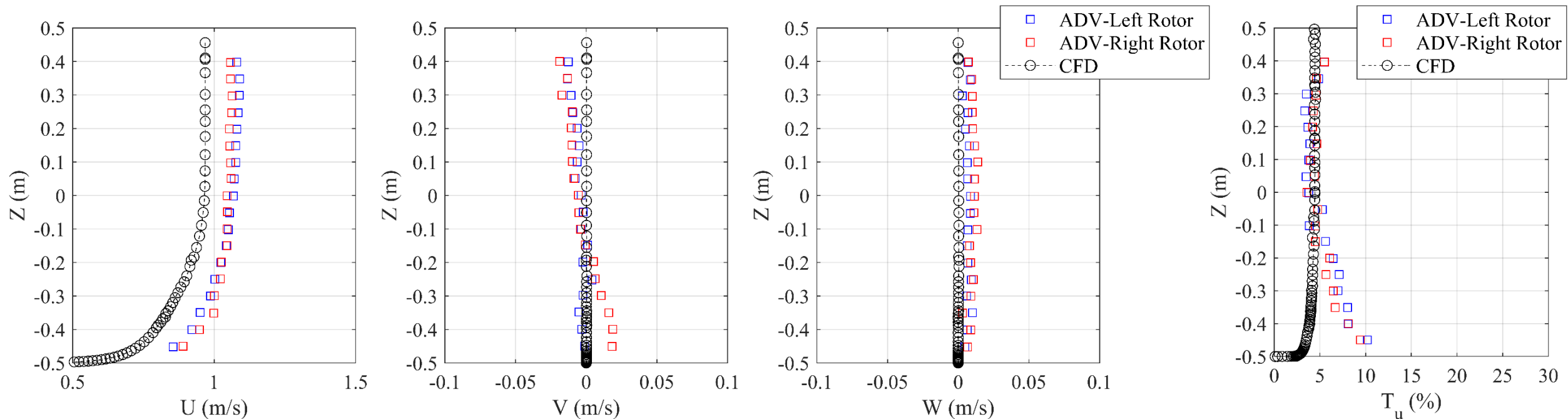


# Inflow Characteristics

## Velocity and turbulence intensity profiles

- $U_{\text{hub,Exp}} \approx 1.04 \text{ m/s} @ x = -3d_T$
- $U_{\text{hub,CFD}} \approx 0.965 \text{ m/s} @ x = -3d_T$

$d_T$ : Turbine diameter  
 $T_u$ : Turbulence intensity  
ADV: Acoustic Doppler Velocimetry



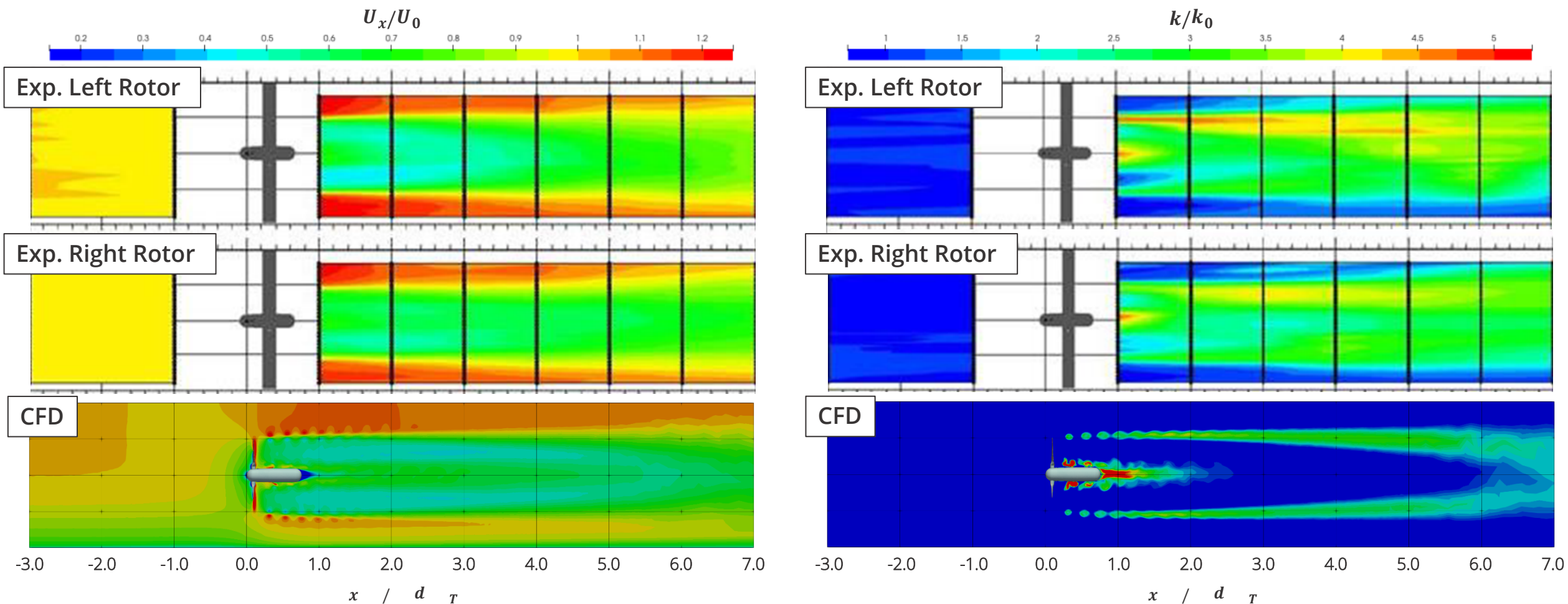
Measured (red and blue square) and estimated (black circle) profiles for velocity components and turbulence intensity





# Turbine Wake Characteristics

## Normalized streamwise velocity and turbulent kinetic energy

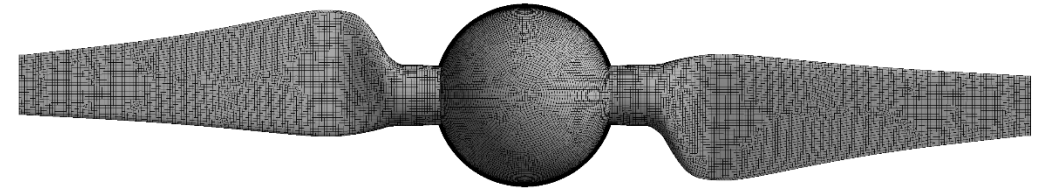


Measured (top) and estimated (bottom) normalized streamwise velocity (left column) and turbulent kinetic energy (right column) in x-z plane

# FEA Model Setup

## Geometry and mesh

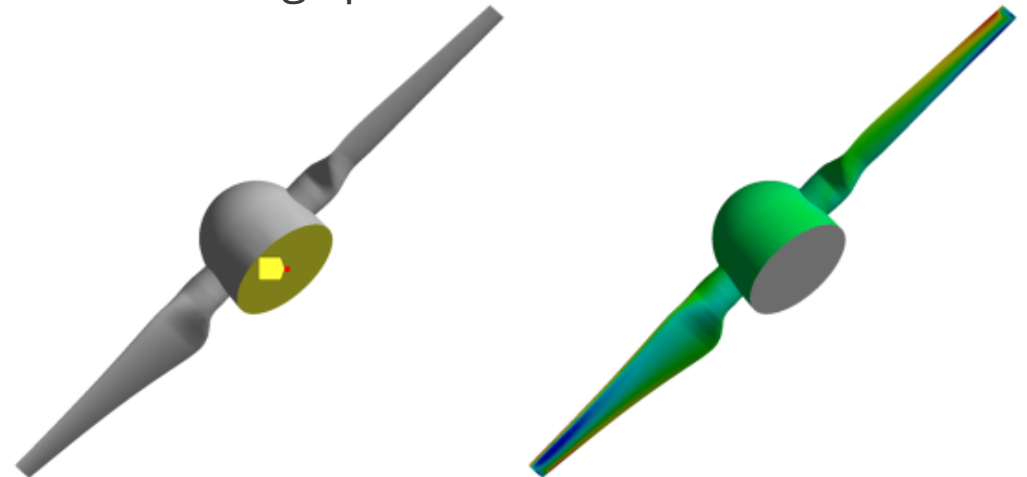
- Rotor only
- Hexahedral mesh with quadratic element order
- Modelled as a solid made from aluminum alloy



Generated mesh for FEA simulation (# of elements = 1.1M)

## Boundary conditions

- Assigned angular velocity corresponding to the turbine rotating speed
- Displacement support at the turbine hub center
- A fluid-solid interface on the rotor surface



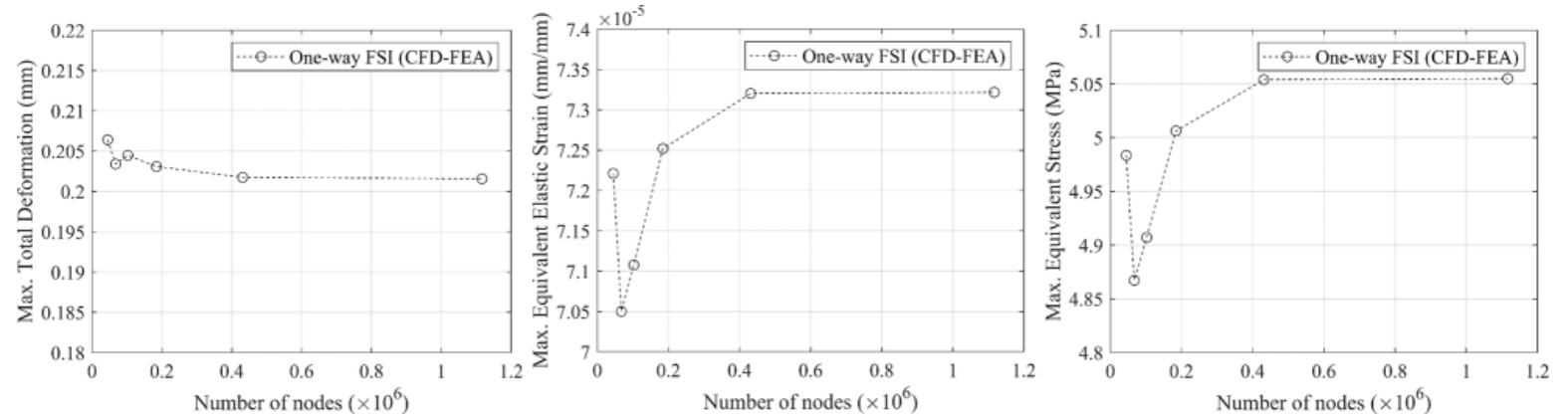
Remote displacement point (left) and pressure on the fluid-solid interface (right)



# One-way FSI

## Simulation results at 204 rpm (TSR = 5.5)

- Mesh size dependency

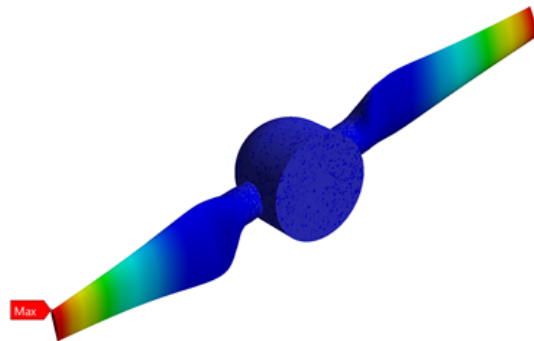


Variation of estimated maximum deformation (left), strain (middle), and stress (right) with mesh density

- Estimated total deformation and equivalent stress

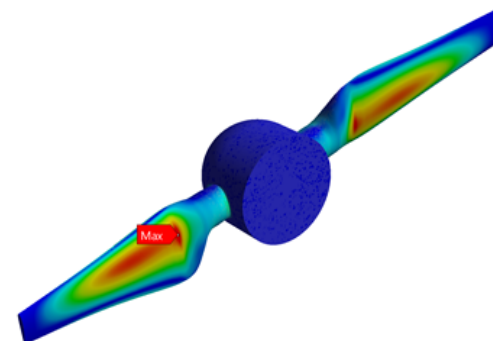
Total Deformation  
Type: Total Deformation  
Unit: mm

0.20074 Max  
0.17844  
0.15613  
0.13383  
0.11152  
0.089219  
0.066915  
0.04461  
0.022305  
2.0923e-7 Min



Equivalent Stress  
Type: Equivalent (von-Mises) Stress  
Unit: MPa

5.0539 Max  
4.4925  
3.9312  
3.3699  
2.8086  
2.2473  
1.686  
1.1247  
0.56334  
0.002026 Min



Max. total deformation = 0.2 mm

Max. equivalent stress = 5.05 MPa

Instantaneous contour plots of total deformation (left) and equivalent stress (right) on rotor



# Next Steps

## 2022:

- CFD model development
- Structural model development
- 2-way FSI simulations, for metal blades **s** model (lab-scale)
- Power performance & wake flow analyses
- Lots of learning:
  - Mesh optimization
  - CFD & Structural coupling
  - Challenges on running on different HPCs (Sandia's HPCs, ANSYS Cloud, etc.)

## 2023:

- FSI simulations for composite blades (lab-scale) & metal blades (full-scale)
- Power performance, hydrodynamic and structural hydroelastic analyses
- Preliminary cost/LCOE analysis

## 2024:

- FSI simulations for composite blades (full-scale)
- Final cost/LCOE calculations
- Final report/publications



# THANK YOU

Questions? Comments?

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