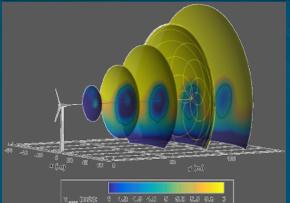




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# A Continuous Representation of Actuator Disk Forcing



*PRESENTED BY*

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# Introduction

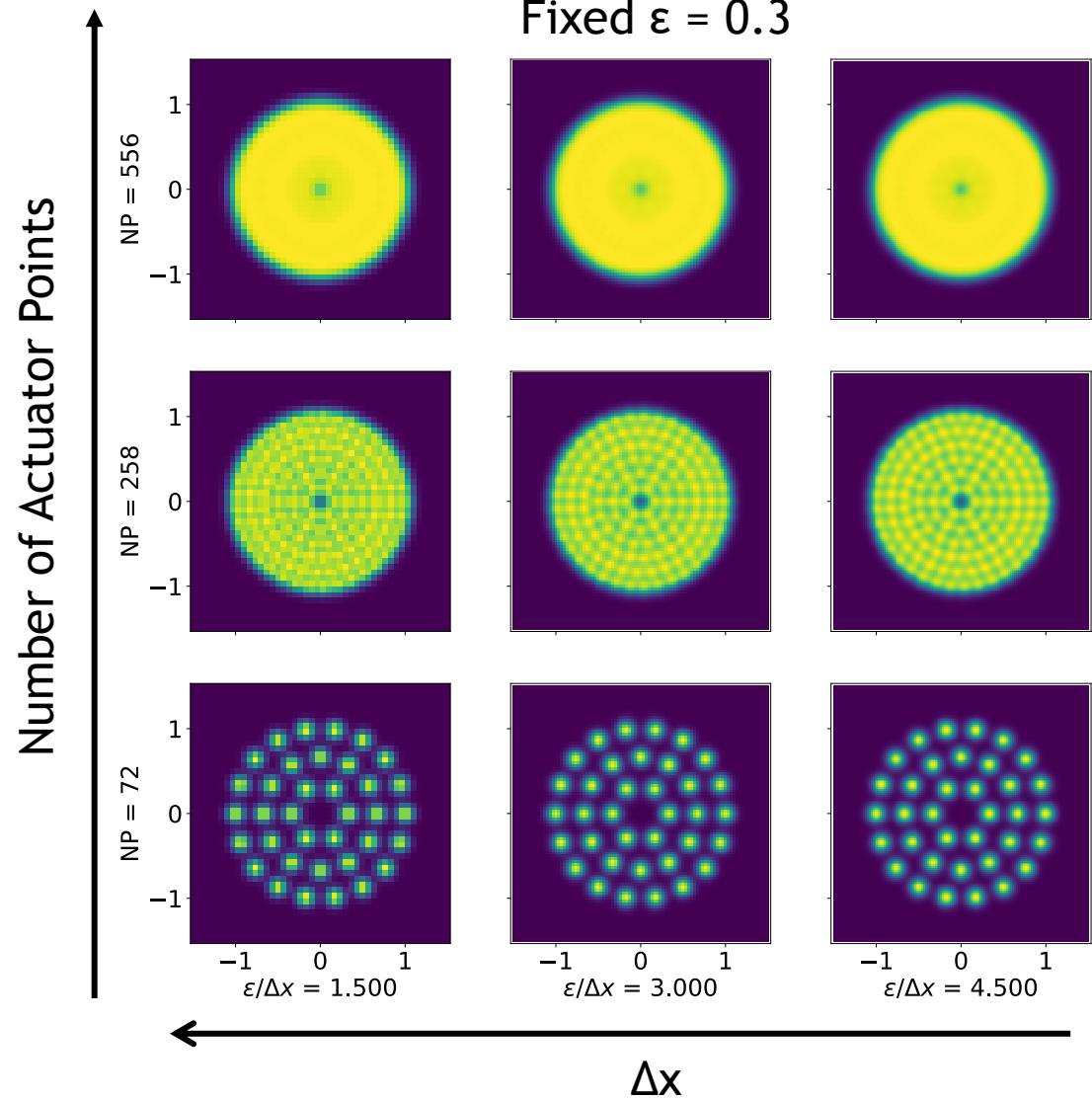


Actuator methods (line and disk) have 3 major components

- Interpolate fluid properties
- Aerodynamic model
- Spreading function – typically Gaussians

Focus on spreading function for disk application

- Two primary purposes:
  - Transfer the force from the aerodynamic model
  - Create a geometric representation which determines wake development
- Point sampling with Gaussians can get expensive
- Dependency on Gaussian width can be confusing
- Explore alternative to Gaussian



## Spreading Function:



Actuator forcing term:

$$\mathbf{F}(\mathbf{x}) = \mathbf{f}_j \xi_j(\mathbf{x})$$

Spreading function defines the geometric representation of the actuator

$$\xi_j(\mathbf{x}) = \frac{1}{(\sqrt{\pi}\epsilon)^3} e^{-\frac{|\mathbf{x}-\mathbf{x}_j|^2}{\epsilon^2}}$$

Can also be thought of as a projection onto a sub-space where the actuator geometry is defined (i.e. sub-space of  $\mathbb{R}^3$ )

$$\vartheta(\mathbf{x}) = \sum_j^N \xi_j(\mathbf{x})$$
$$\mathbf{x} \in (-\infty, \infty), \vartheta(\mathbf{x}) \in \mathbb{R}^3$$

## Foundations for Looking Beyond the Gaussian



Gaussian is not the only option for spanning the actuator sub-space

Requirements for the projection function:

- Normalized so point force is conserved i.e.  $\int \xi_j^*(\mathbf{x}) d\mathbf{x} = 1$
- Ideally it will be smooth and strictly positive as well

Metrics for comparing Gaussian with alternative projection functions

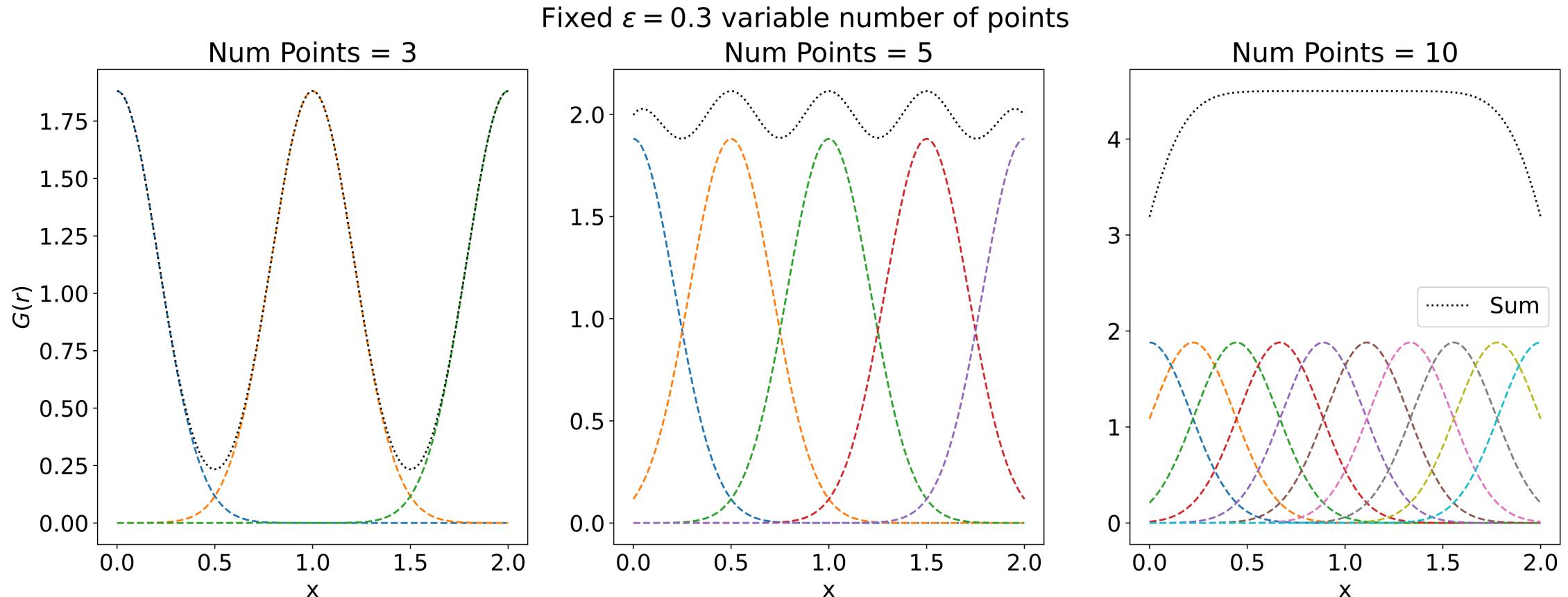
- Accuracy:
  - Measure the value of the normalized projection function on the background mesh

$$R = 1.0 - \frac{\Delta x^2}{N} \sum_j^N \xi_i^*(\mathbf{x})$$

- Performance:
  - Count the number of projection function calls required to compute the actuator sub-space

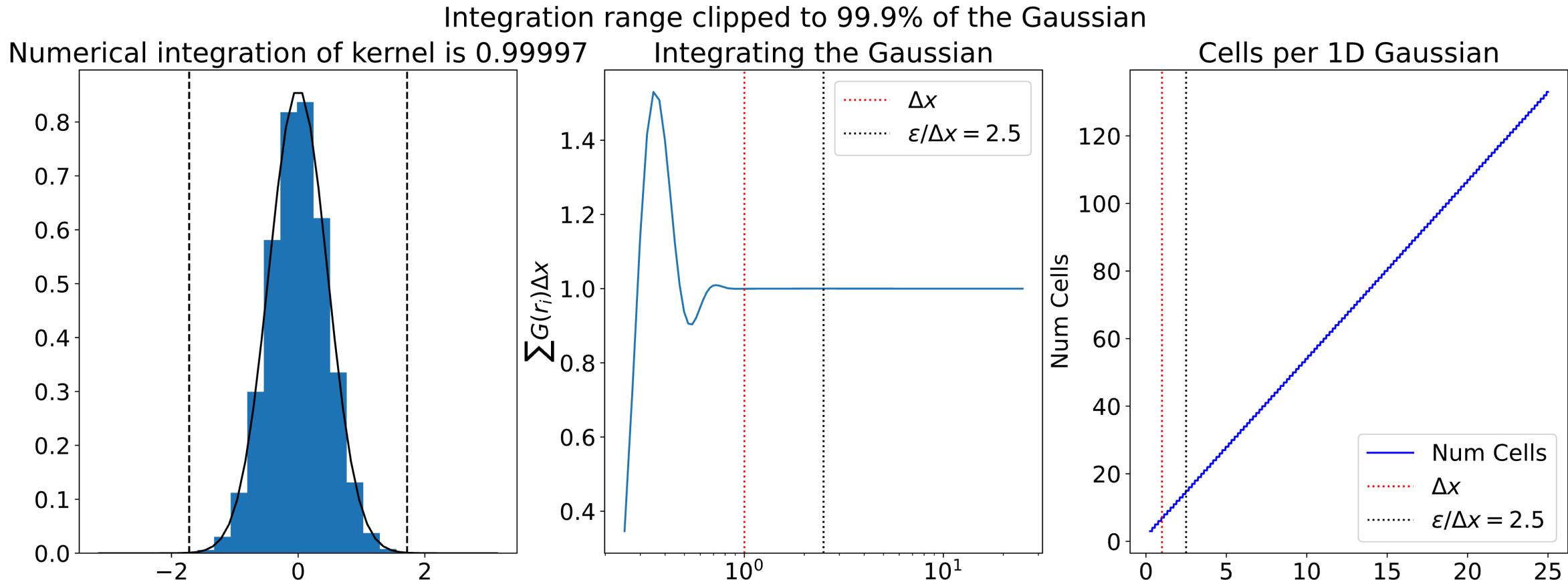
$$\sum_i^M \sum_j^N \xi_j^*(\mathbf{x}_i)$$

# Traditional Method: Gaussians



Forcing must become diffuse to get a smooth sub-space (geometry)

- Lots of overlap between forcing from each actuator point



Accuracy is quite good, even with mid-point integration

Requires large foot print on the background mesh (10 cells > in each direction) based on common metric (Sorensen and Shen, 2002; Martinez-Tossas, Churchfield and Leonardi, 2014)

## 7 Another Word on Epsilon



Forcing sub-space geometry is dependent on combination of number of actuator points and epsilon

- Must adequately span the distance between actuator points (avoid porosity)
- Must meet requirements of background grid for integration accuracy

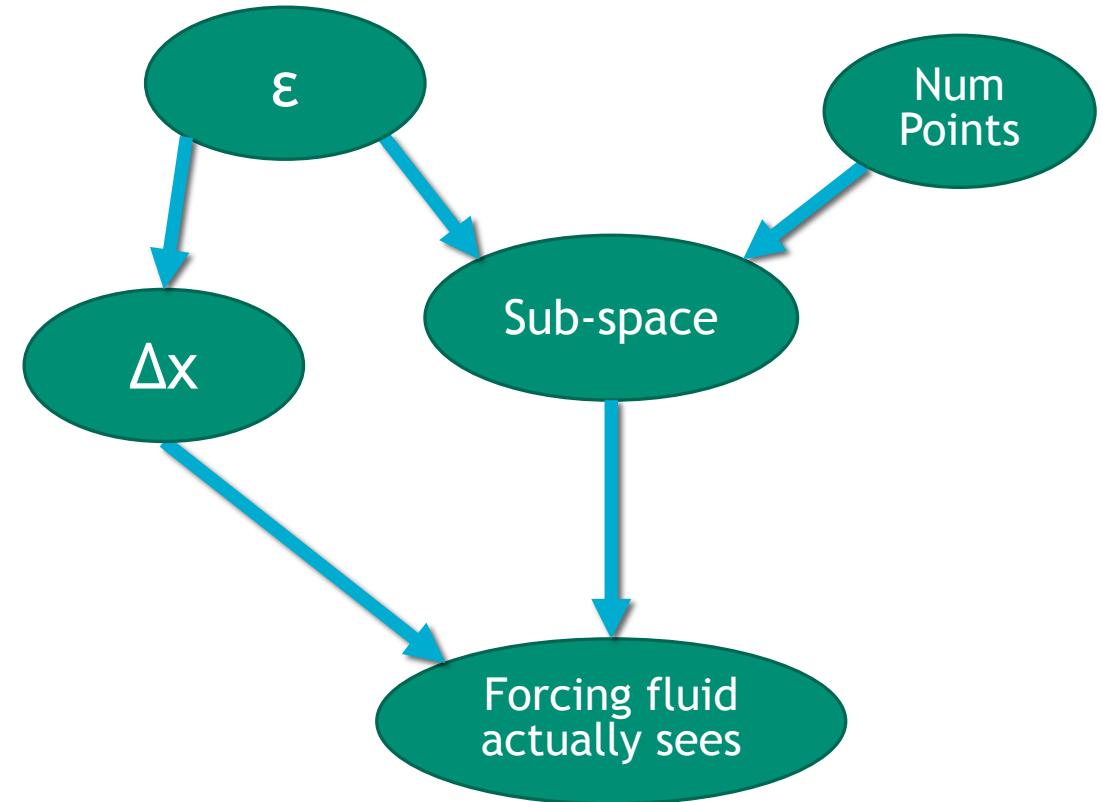
One solution is to use more points

- Acceptable for ALM but costly for ADM
- 1D vs 2D geometry

Another is to increase epsilon

- Changes geometry (makes disk bigger at the edge of the domain)
- Diffuses force over larger area
- Computational cost increase on a given grid due to larger foot print

Goal to remove coupling between actuator geometry and integration requirements



# Alternative Functions

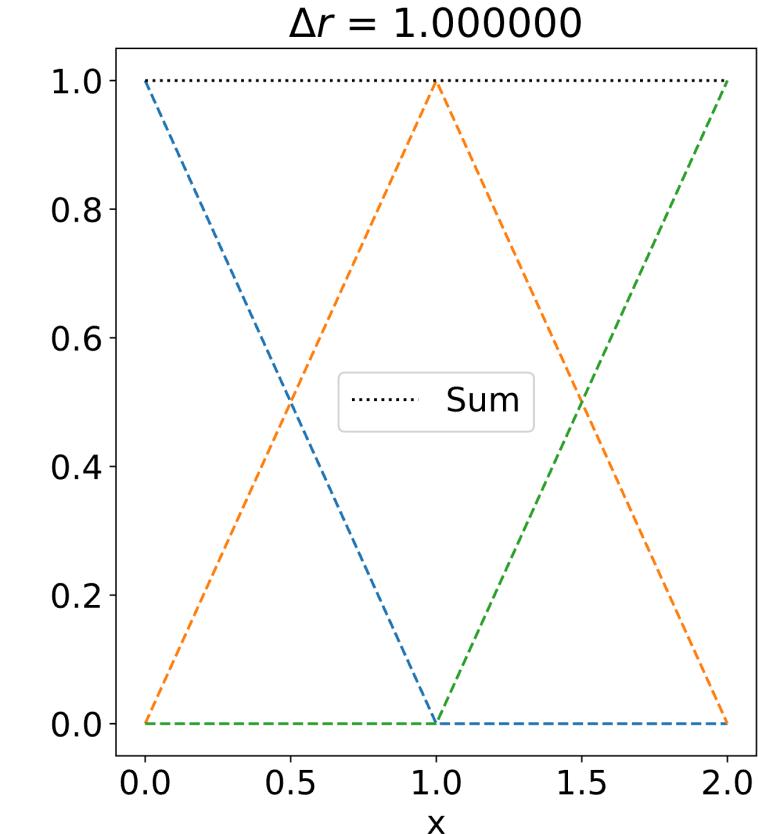
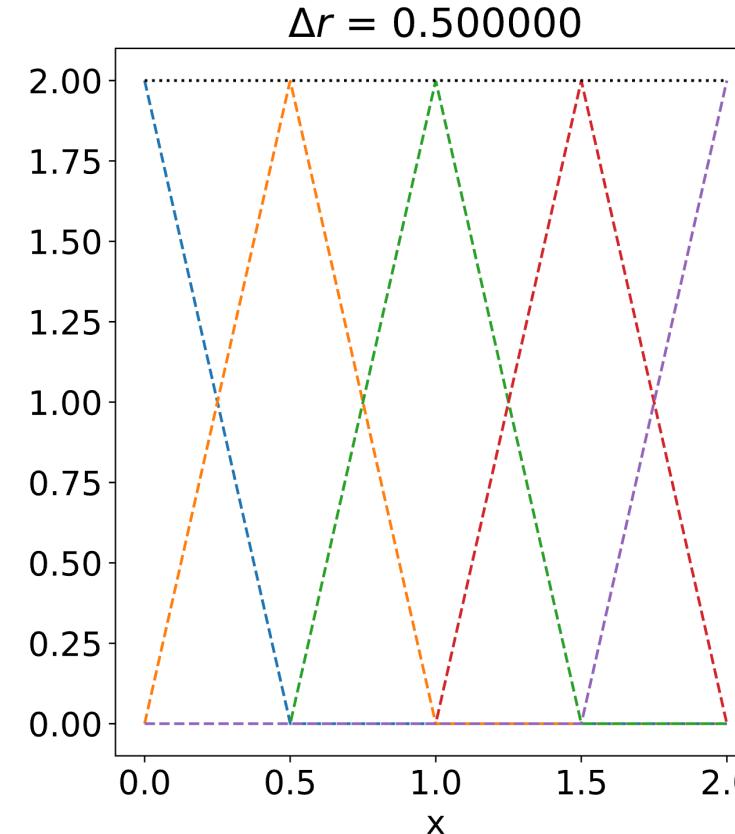
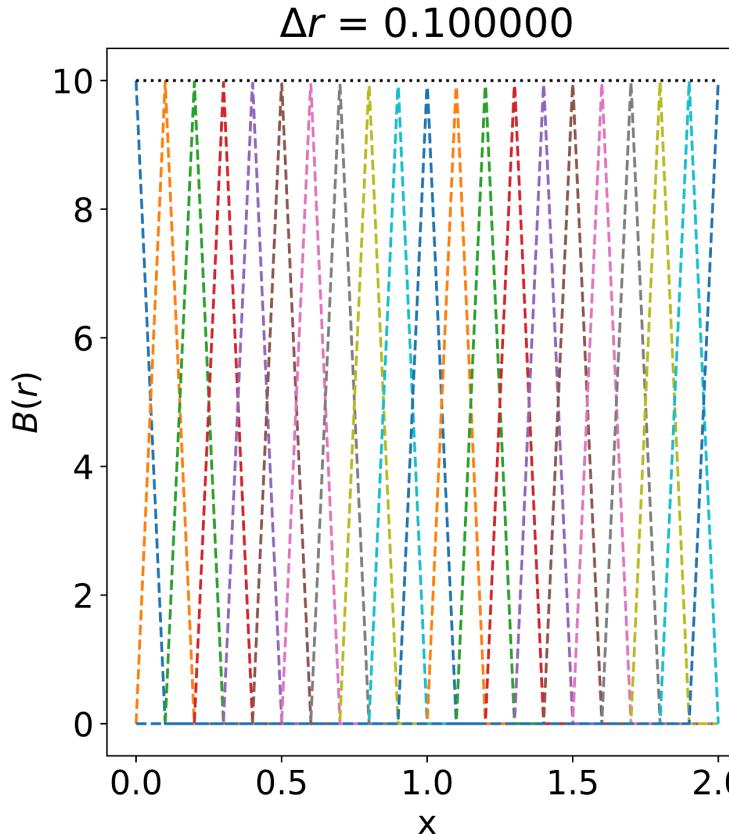


Replace Gaussians with functions that

- Have compact support i.e.  $x \in [-\Delta r, \Delta r]$ ,  $\vartheta^*(x) \in \mathbb{R}^1$
- Are partitions of unity:  $\sum_j^N \xi_j^*(x) = 1$

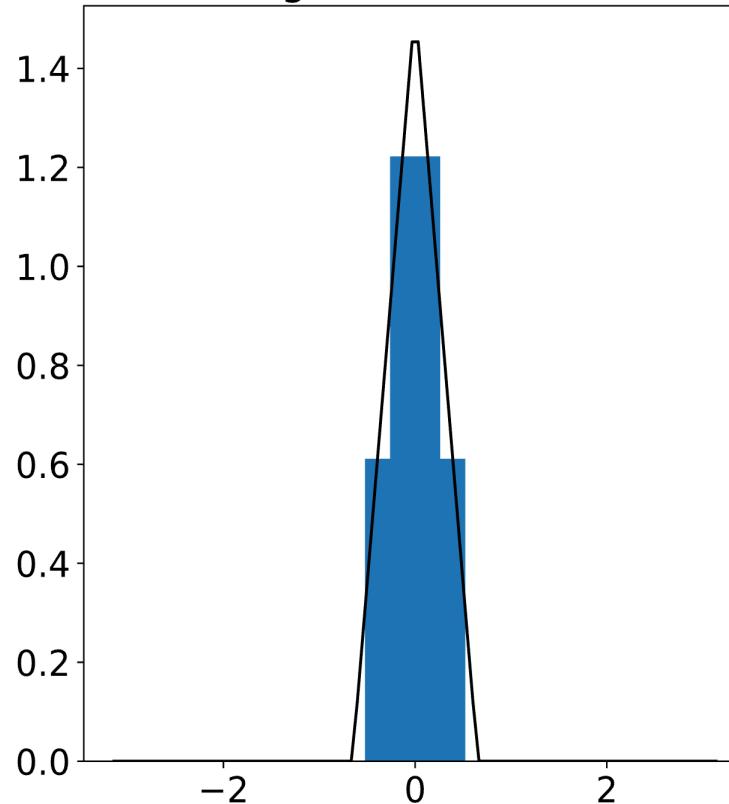
Example is a normalized linear basis:  $\xi_j^*(x) = \frac{\max(0, 1 - \frac{|x - x_j|}{\Delta r})}{\Delta r}$

Note the peak value changes due to compact support combined with normalization. Same thing occurs with Gaussians as  $\varepsilon$  is changed.

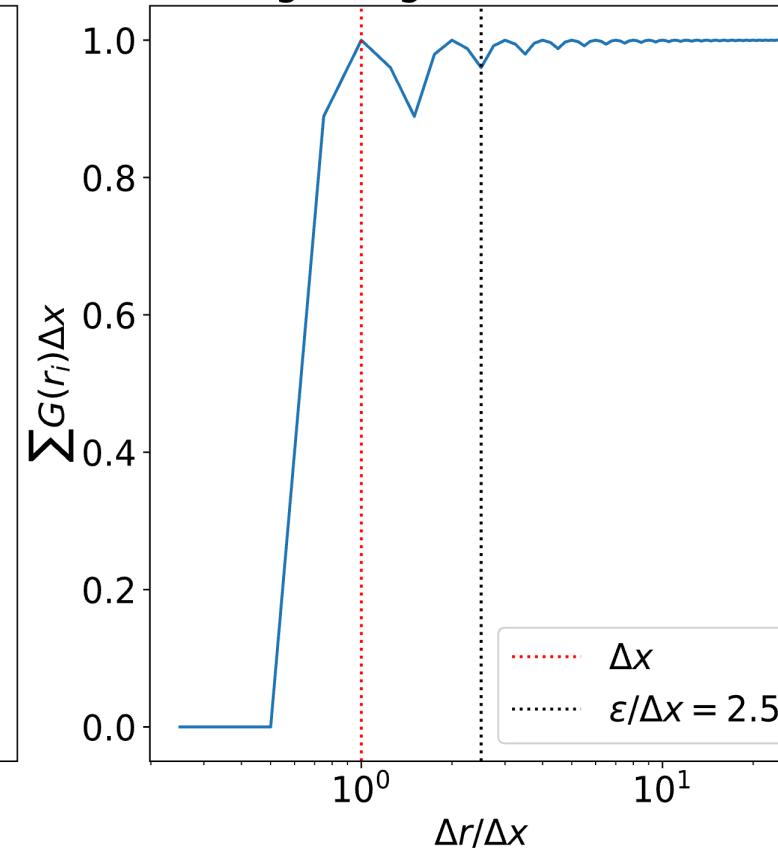




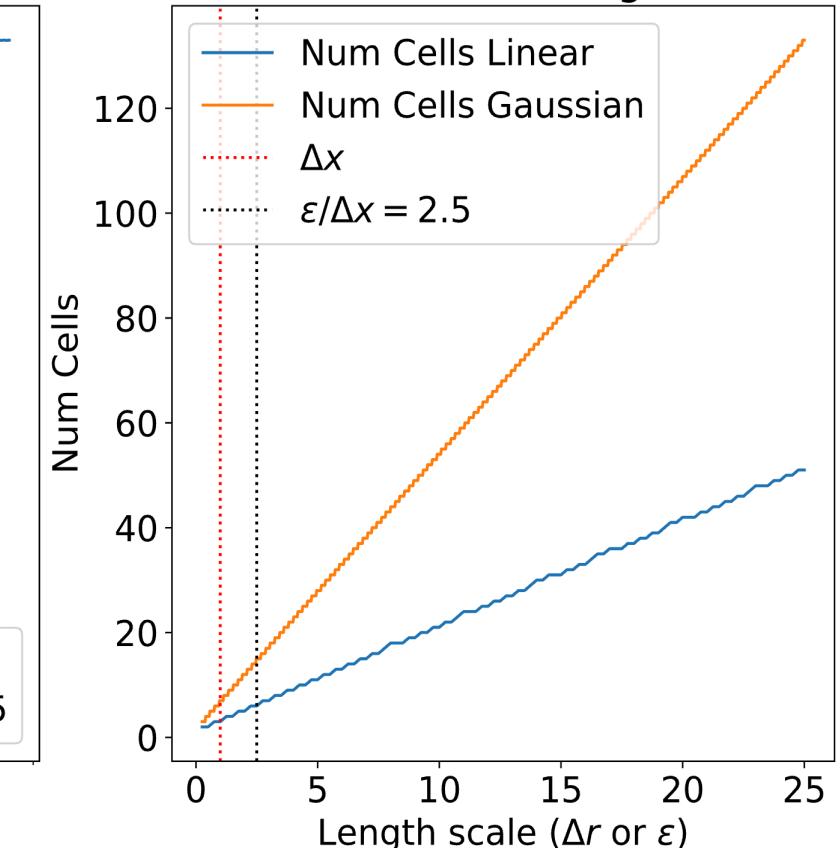
Numerical integration of kernel is 0.96000



Integrating the Linear Basis



Cells used for integration



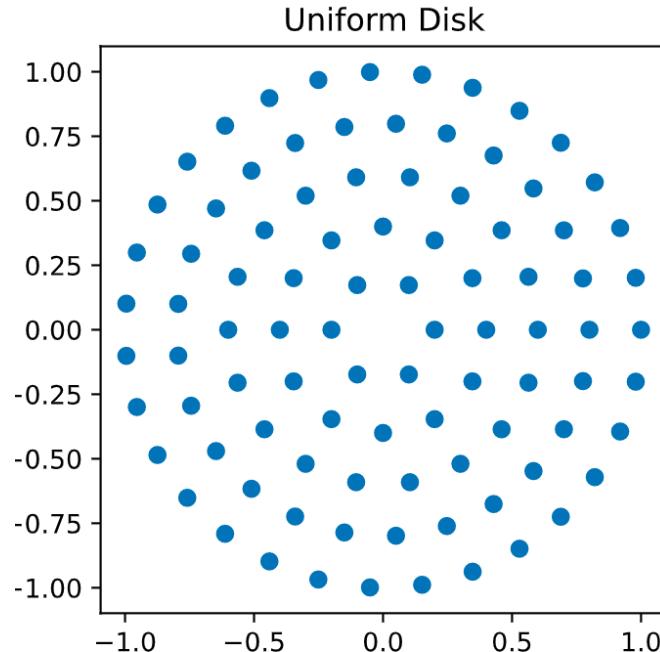
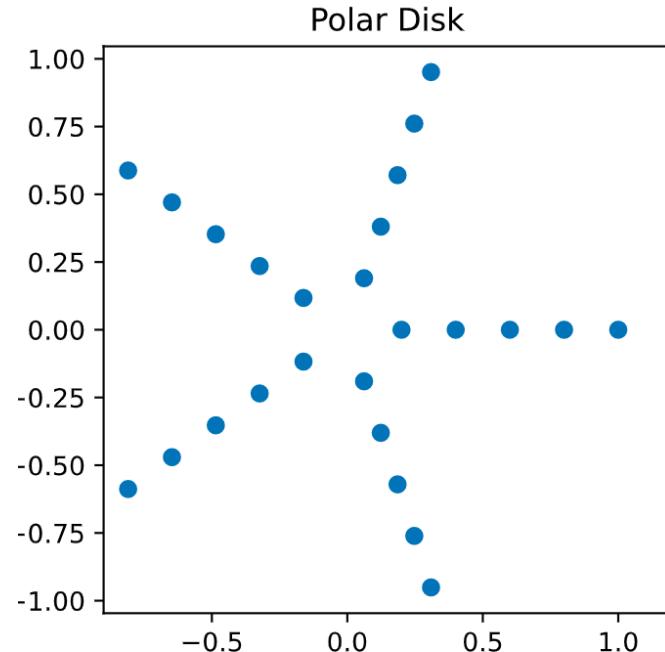
Integration foot print is smaller (restricts support to distance between actuator points)

Accuracy is comparable to the Gaussian for the same foot print with mid-point rule

## Comparison on a Fixed Grid



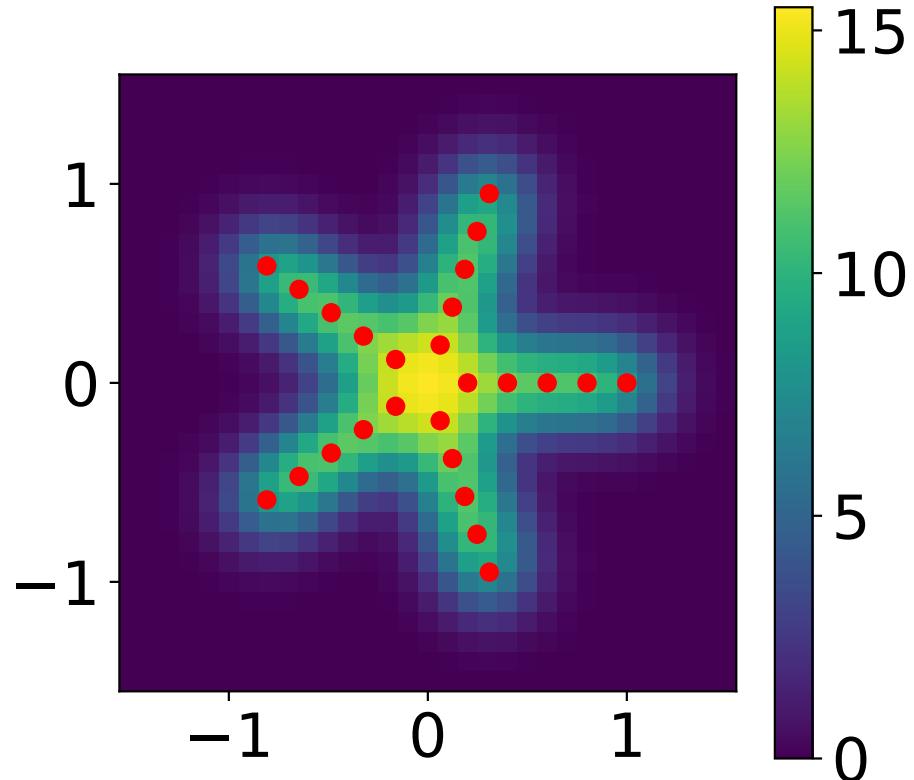
- Discretize the unit circle with actuator points
- Use a background mesh with uniform grid-spacing of 0.1 from -1.5 to 1.5



# Performance and Accuracy: Gaussian

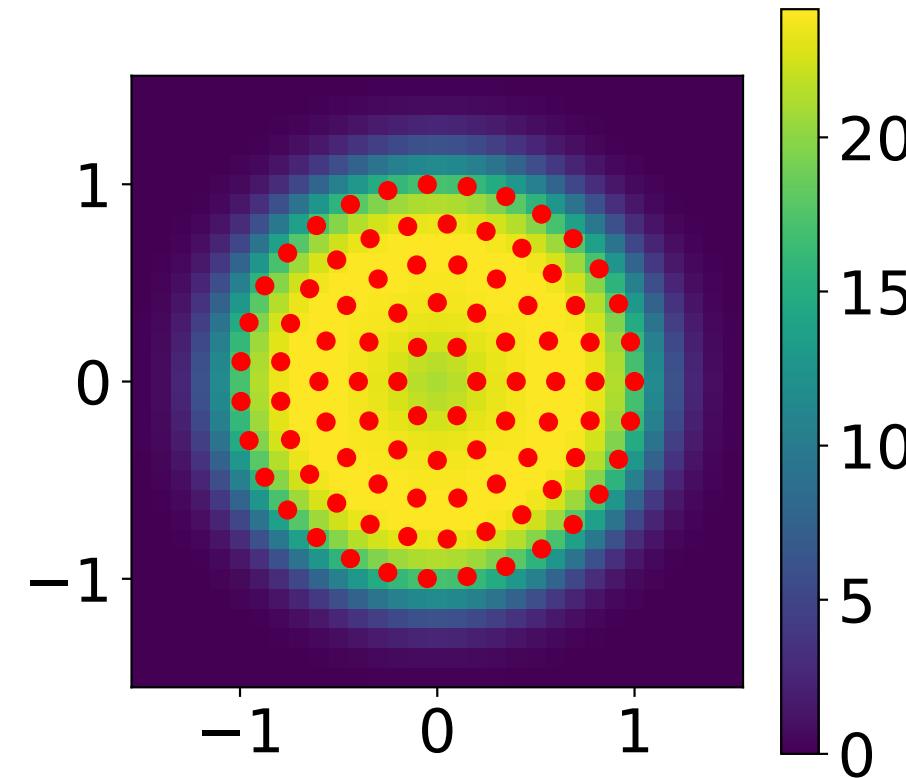


Polar



Num Function Calls: 3378  
Integration Accuracy: 0.99896587

Uniform



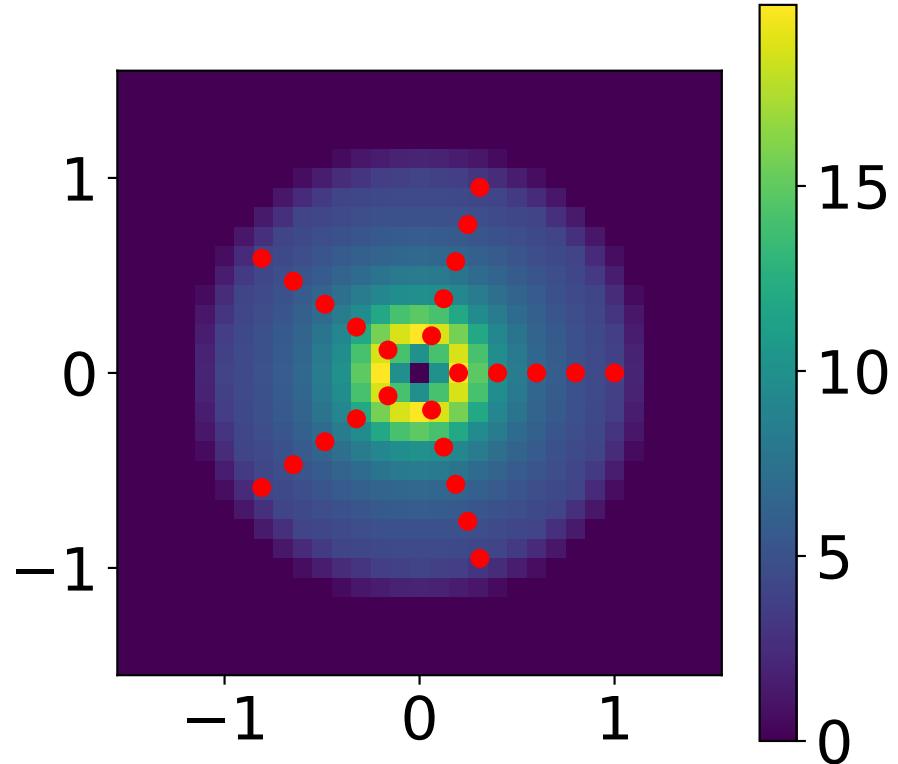
Num Function Calls: 12443  
Integration Accuracy: 0.99896058

# Performance and Accuracy: Linear Basis



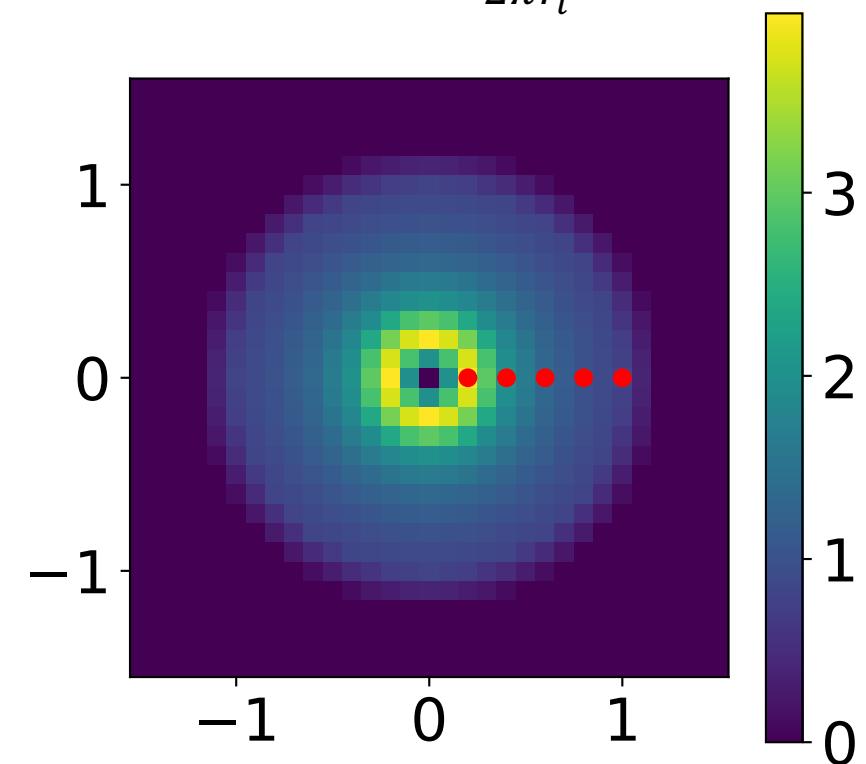
**Polar (5x5)**

$$\eta_{ij}(r, \theta) = \xi_i^*(r, \Delta r = \Delta R) \times \xi_j^*(r\theta, \Delta r = r_i \Delta \theta)$$



**Polar (5x1)**

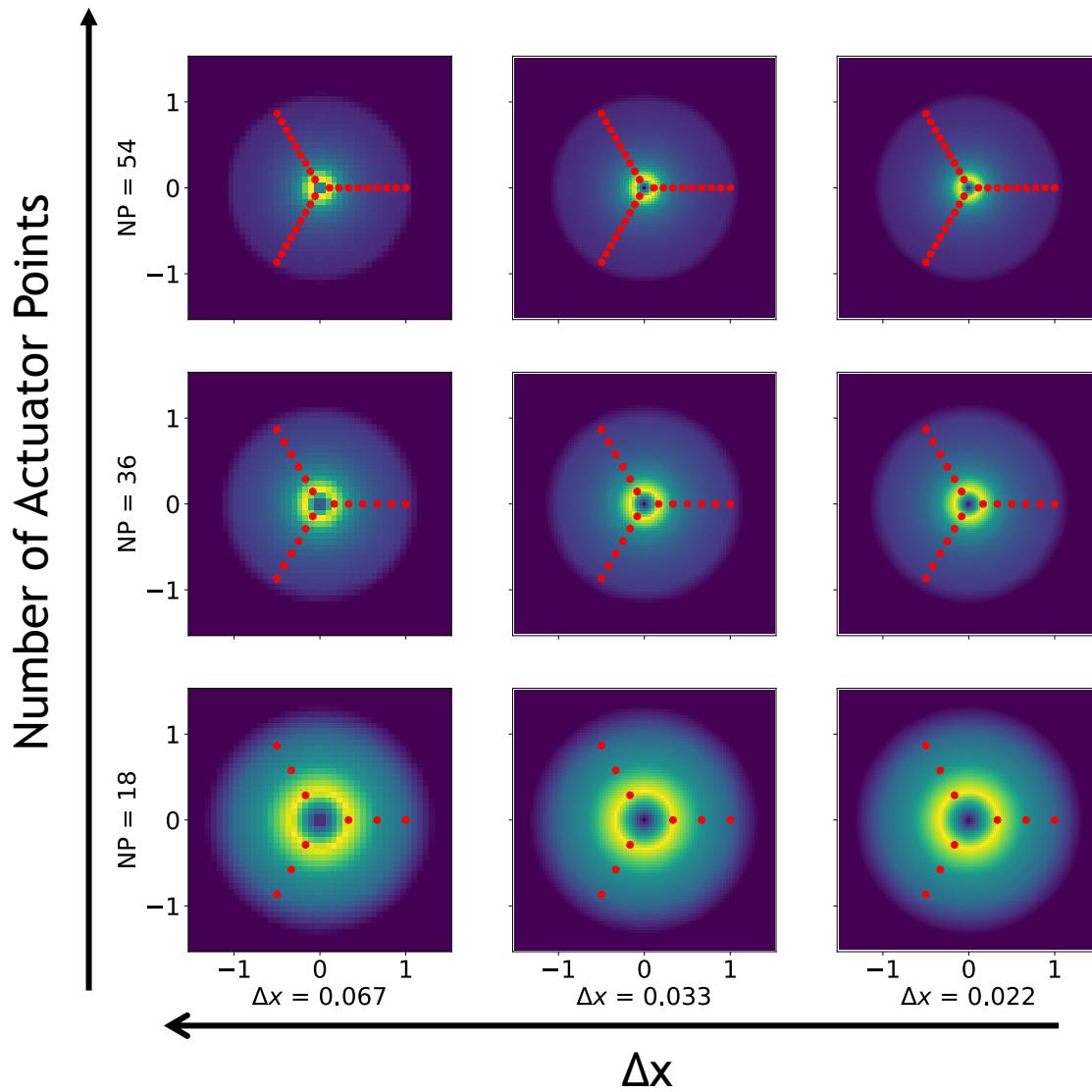
$$\eta_i(r) = \frac{\xi_i^*(r, \Delta r = \Delta R)}{2\pi r_i}$$



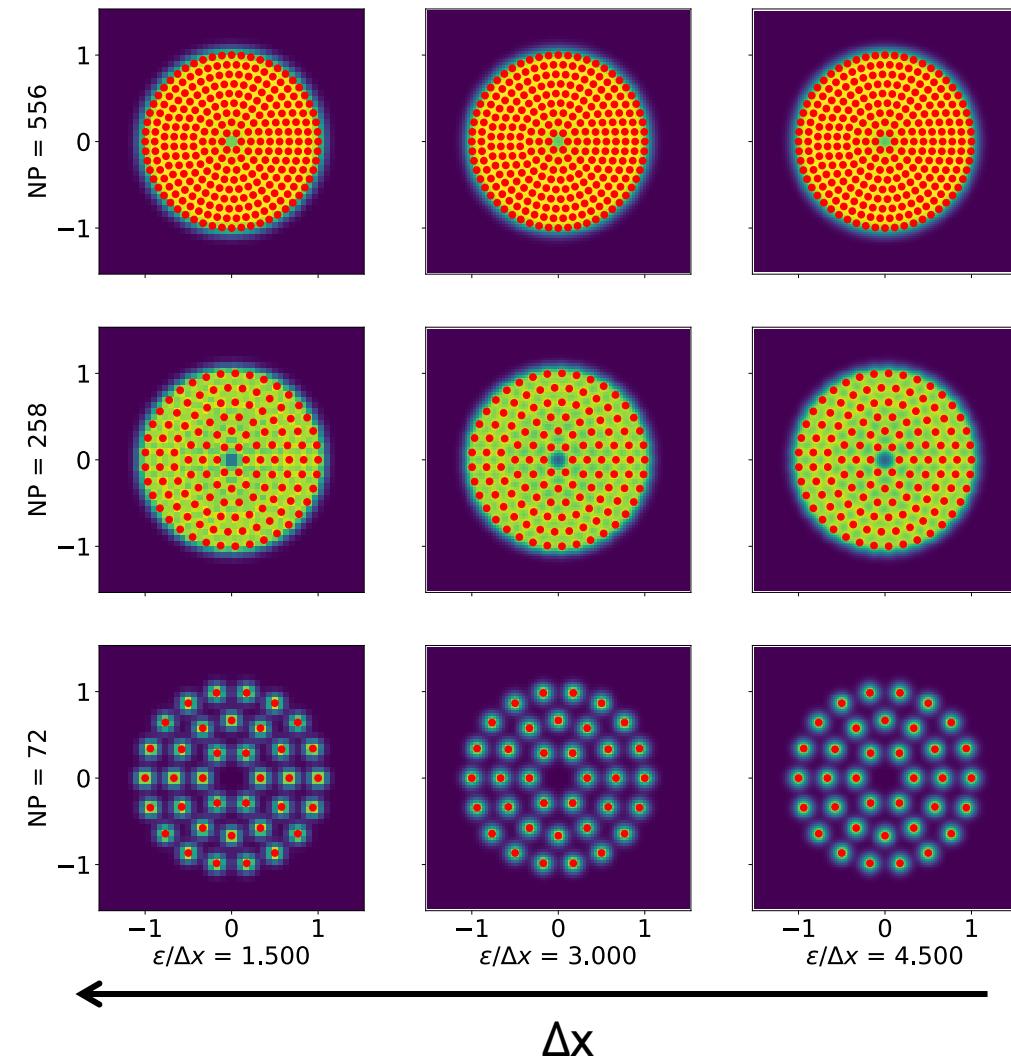
Num Function Calls: 1474  
Integration Accuracy: 1.00008085

Num Function Calls: 737  
Integration Accuracy: 1.00008085

# Point Sample Invariance



- Geometric representation of the disk is really only varies at the edges
  - Due to dependence on radial point spacing



- Practically invariant when compared to the Gaussian
- Significant reduction in points required to capture the disk



## What about normal direction, disk edge?

- Can use a Gaussian, or any other normalized basis function
- Edges of sub-space can be the same as the standard model by using Gaussians

## Conclusions:

- Changing the Gaussian kernels to functions with 1) compact support that are 2) partitions of unity offers several advantages
- Effectively makes geometric resolution independent from other factors (background mesh integration, aerodynamic model, etc.)
  - Geometric representation of the actuator disk can be maintained with minimal actuator points
  - Accuracy of projection to the fluid domain is not noticeably impacted even at coarser actuator point resolutions
- Less diffuse forcing with proportional decrease in computational cost due to reduction in kernel overlap

## Next Steps:

- Evaluate other basis functions and fine tuning
- Extend theory to actuator line applications

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