

# Towards the Development of a Conceptual Model for Pre-chamber Spark-Ignition High Efficiency Natural Gas Engines

---

**Rajavasanth Rajasegar <sup>1</sup>, Yoichi Niki <sup>2</sup>, Jose Maria Garcia Oliver <sup>3</sup>,  
Zheming Li <sup>1</sup>, Mark P.B. Musculus <sup>1</sup>**

<sup>1</sup>*Combustion Research Facility, Sandia National Laboratories, Livermore, CA, USA*

<sup>2</sup>*National Maritime Research Institute, Tokyo, Japan*

<sup>3</sup>*CMT - Motores Térmicos, Universitat Politècnica de València, Valencia, Spain*

**Sponsors :** Office of Vehicle Technologies, Department of Energy (DOE).  
**Program Managers :** Kevin Stork, Gurpreet Singh and Mike Weismiller.

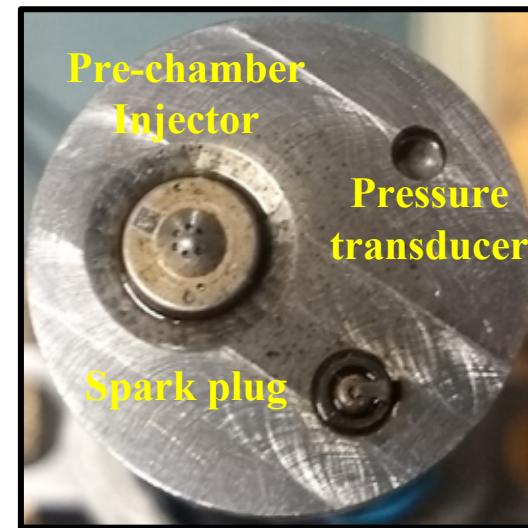
# Abstract

---

The overarching goal of this work is to identify some key elements of a conceptual-model for pre-chamber spark-ignition (PCSI) and to help address the inadequate science base and simulation tools required to describe/predict the fluid-mechanical and chemical-kinetic processes governing PCSI. To this end, experiments performed in a heavy-duty, optical, single-cylinder engine fitted with an active/fueled PCSI module provided fundamental insights on the ignition and subsequent combustion of fuel-lean main-chamber mixtures by a near-stoichiometric pre-chamber. Heat-release analysis coupled with optical diagnostics allowed for tracking spatial and temporal progress of ignition and combustion of lean-burn of NG, while identifying key phenomenological features of PCSI systems.

Pre-chamber Specifications	
Volume [ml]	4.66
Orifices and diameter [mm]	8 equally-spaced, 1.6
Included angle	130
Modified Compression Ratio	11.03
Spark plug	Miniature rimfire Z1

Pre-chamber layout

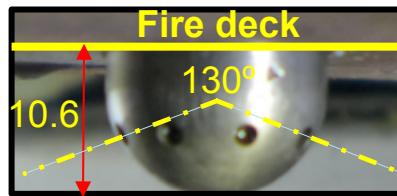


Rimfire Z1 Spark plug



\*All dimensions are in inches

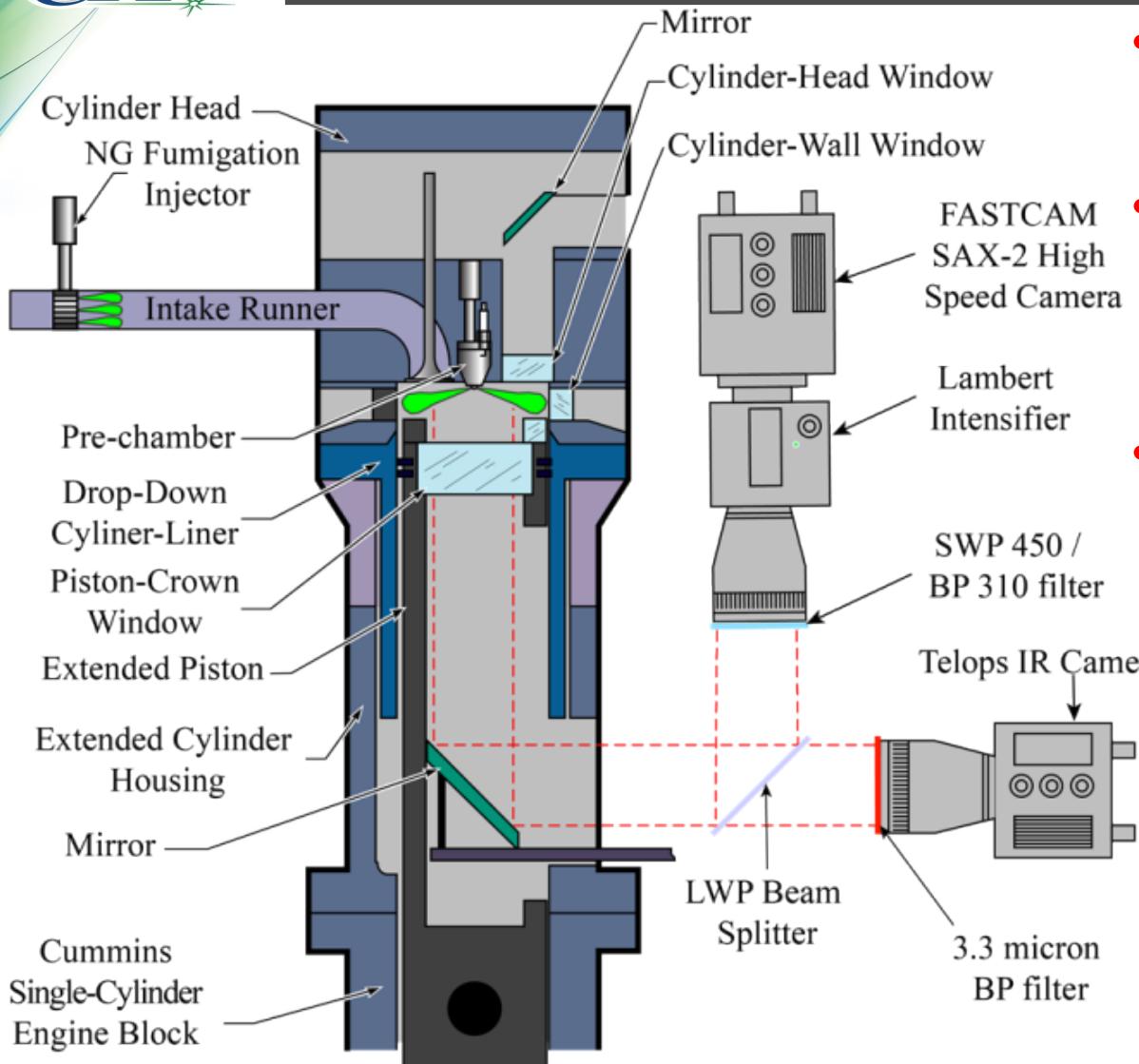
NG Pre-chamber Injector Specifications	
Fuel injector type	Solenoid actuated, Bosch HDEV5 GDI
Orifices and diameter [mm]	6 (3 identical pairs) unequally-spaced, 0.17
Fuel	NG (95% CH <sub>4</sub> , 4% C <sub>2</sub> H <sub>6</sub> 1% C <sub>3</sub> H <sub>8</sub> by vol.)



- Number of holes: 8. Hole size: 1.6 mm.
- Included angle: 130.
- 10.6 mm tip protrusion below fire deck.
- Nozzle plane parallel to cylinder head.
- Uncooled piezoelectric pressure transducer.
- GDI fuel injector (SNL / NREL). Check valve (ANL).
- Miniature “Rimfire Z1” spark plug.

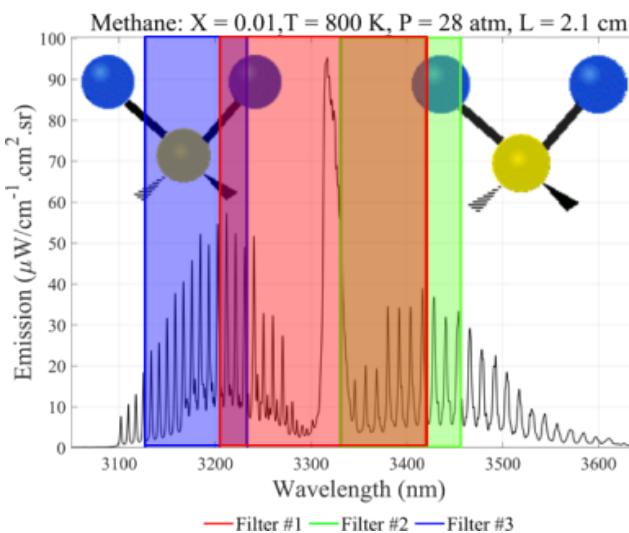
# IR imaging for characterization of pre-chamber jets.

## CL imaging for ignition/combustion location and mode of propagation.

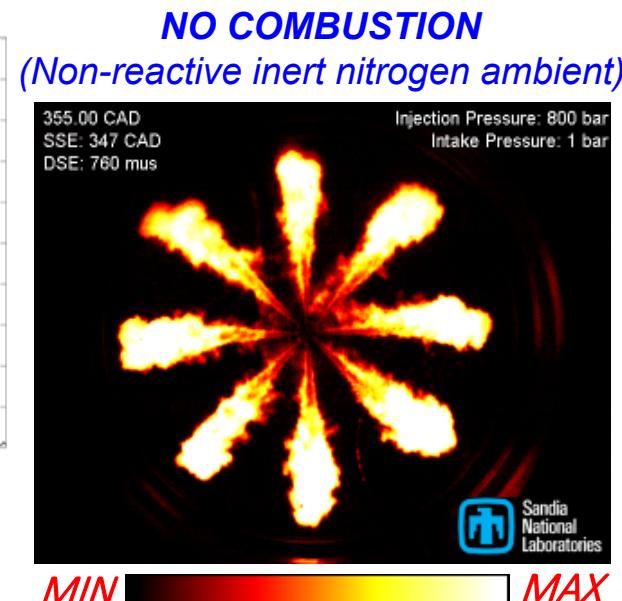


Schematic layout of the optical engine with Infrared and high-speed Chemiluminescence imaging diagnostics setup

- All hydrocarbons emit in the IR near 3.3  $\mu\text{m}$  due to *thermally excited vibration of C-H bonds (“C-H stretch”)*.
- Filtered IR emission is strong enough for imaging when fuel is heated to above 700 K by compression, providing a means to *quickly and easily detect hot in-cylinder fuel*.
- *Concentration and Temperature are intertwined*.



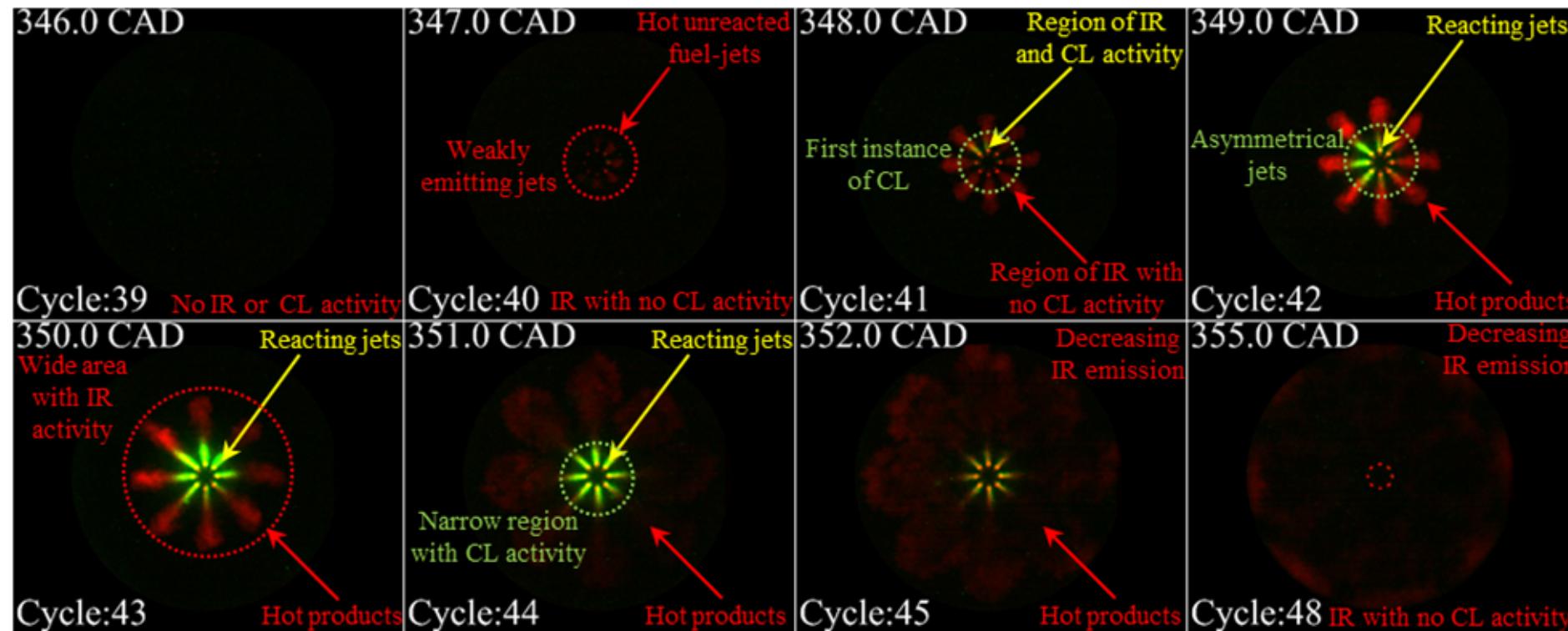
**Methane Emission Spectrum**  
(HITRAN Simulation)



**FILTERED IR SIGNAL  $\equiv$  HOT FUEL**

**PC only fueling** → **IR** : Unburned fuel-jet precedes burning jet; spreads over wider area;  
 → **CL** : Asymmetry in PC jets; restricted to PC periphery.

**Composite snapshots : IR images overlaid on Broadband CL image (simultaneous but non-sequential)**

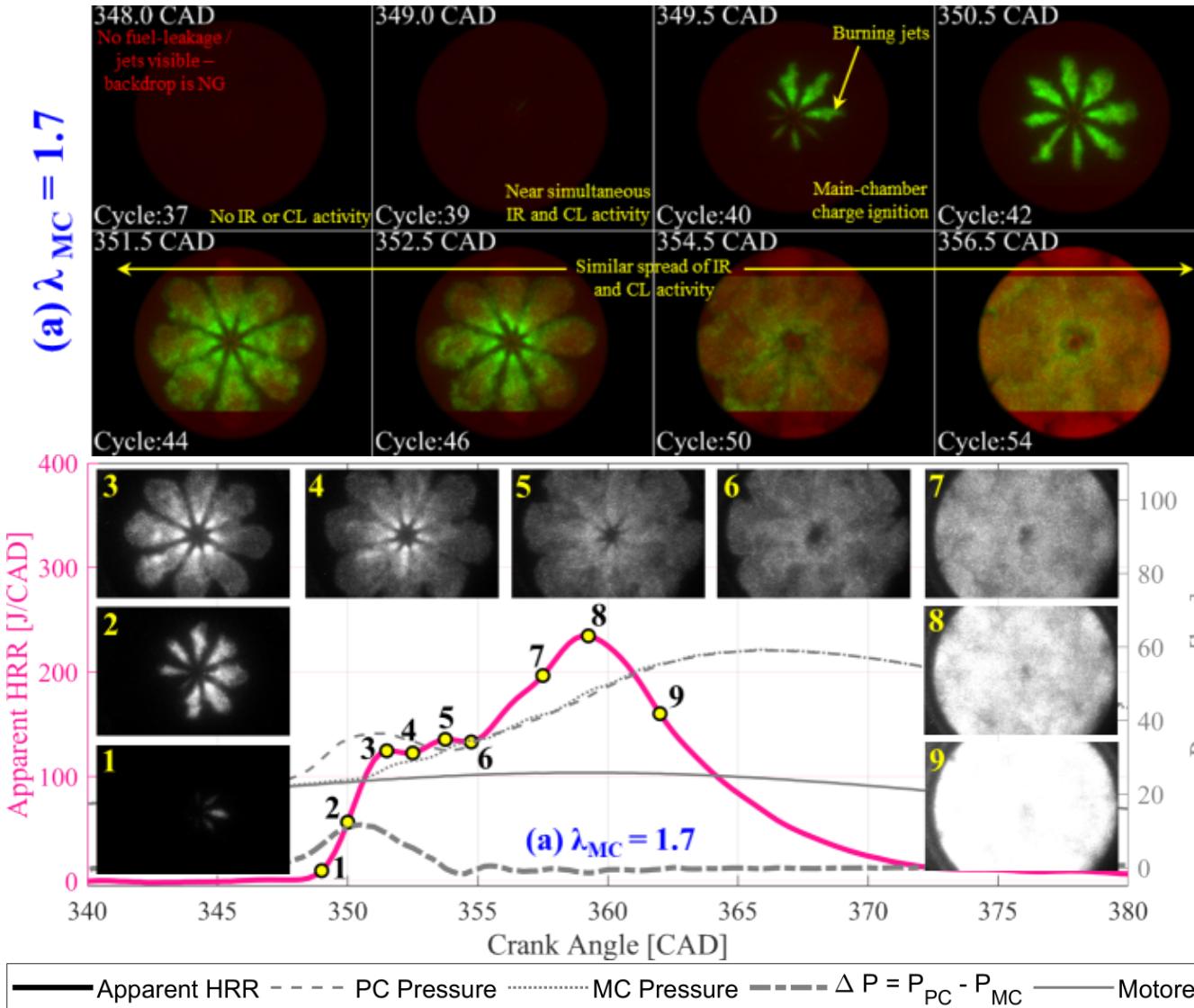


**PC only fueling: No NG in MC.  $\lambda_{PC} = 0.93^*$ , SSE: 325.75 CAD, DSE: 1517  $\mu$ s (~ 11° CA), ESE: 336.5 CAD.**

- **IR imaging (Red)**: Far-field, wider gas jet, high temperature but absence of radical.
- **Broadband CL imaging (Green)**: Near-nozzle zone, narrow chemically active jet.
- Asymmetric PC jets : PC charge stratification & stochastic nature of ignition → non-uniformities in shape and direction of initial spark kernel development inside PC → Cycle-to-cycle variations.

$\lambda_{MC} = 1.7$ : Ejection of unburned fuel-jet is not discernible due to presence of NG in MC.  
 Simultaneous appearance of IR & OH\* CL signal at 349.5 CAD ( $\Delta P \sim 3$  bar) for all  $\lambda_{MC}$ .

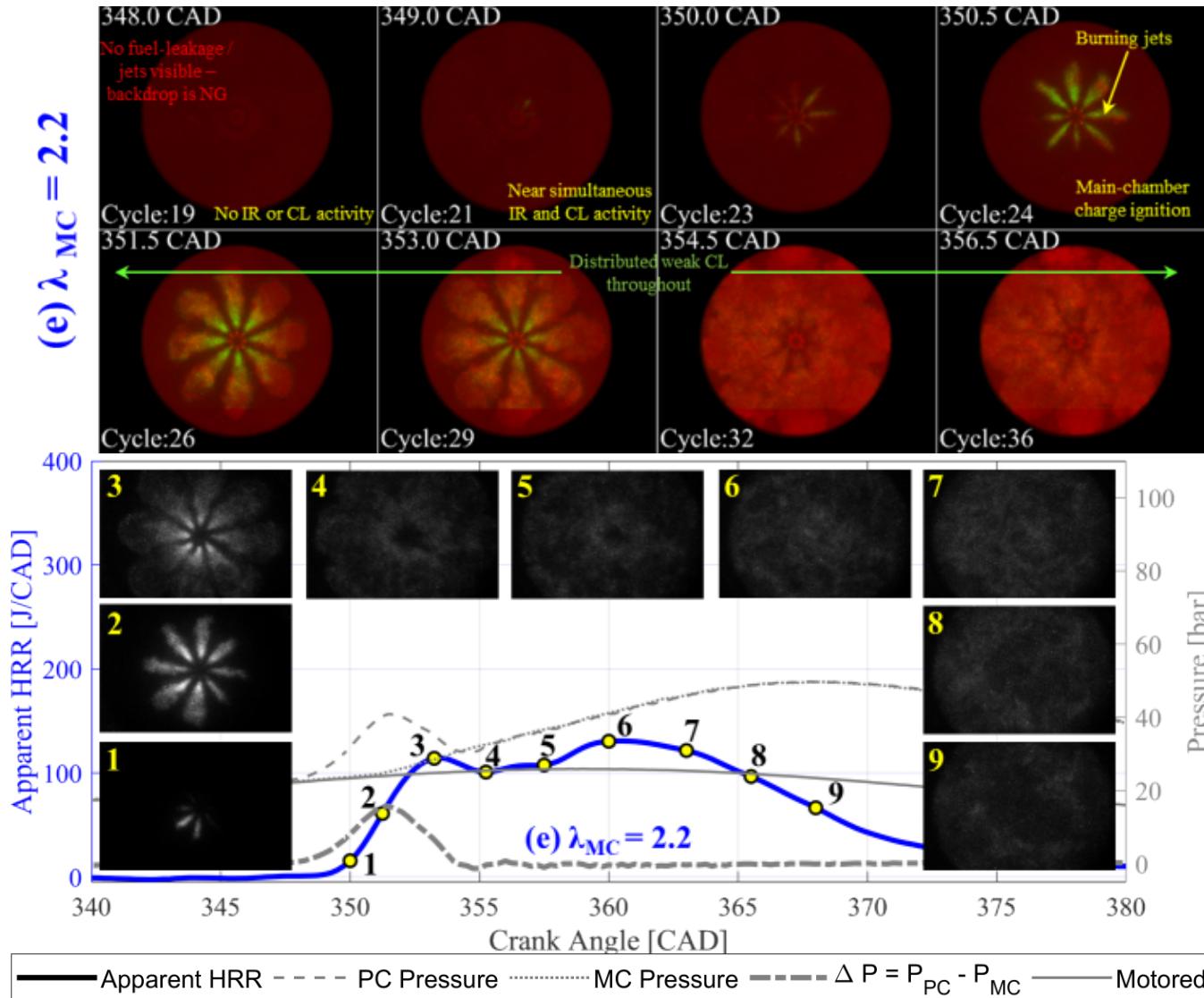
Composite snapshots : IR images overlaid on OH\* CL image (simultaneous but non-sequential)



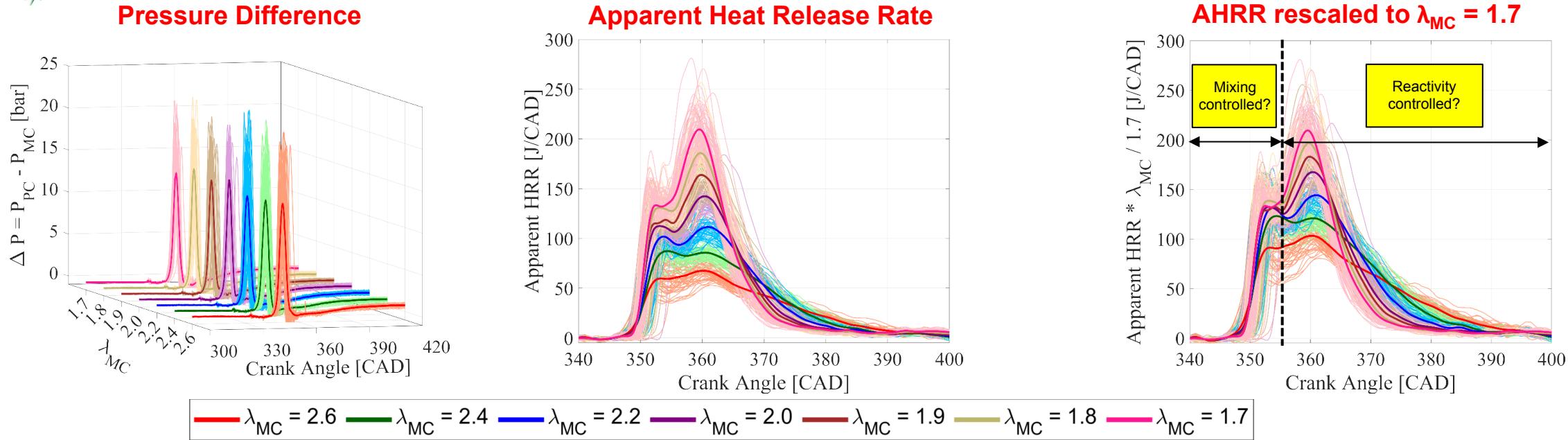
AHRR,  $\Delta P$  & pressures of PC and MC along with sequential snapshots of OH\* CL at key timings.  $\lambda_{PC} = 0.93^*$ .

$\lambda_{MC} = 2.2$ : Near instantaneous ignition of MC charge (within 1°CA). Bimodal AHRR. PC jets merge → local extinction due to fresh charge unavailability → AHRR local minimum (~355 CAD).

Composite snapshots : IR images overlaid on OH\* CL image (simultaneous but non-sequential)



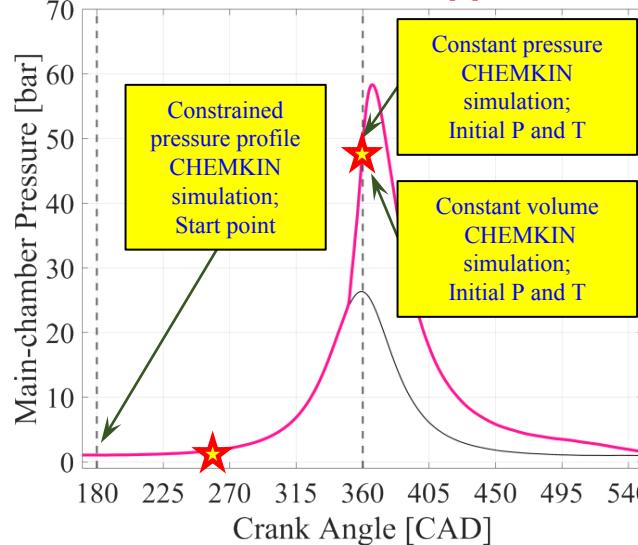
AHRR,  $\Delta P$  & pressures of PC and MC along with sequential snapshots of OH\* CL at key timings.  $\lambda_{PC} = 0.93^*$ .



- PC jets' exit velocity  $\sim 150 - 200$  m/s independent of  $\lambda_{MC}$  (based on initial OH\* images).
- PC jets' tip penetration speed  $\sim 100$  m/s independent of  $\lambda_{MC}$  (entrainment and viscosity effects).
- *PC jets' speed* is *two orders of magnitude higher* than the maximum achievable *turbulent flame speed* at these conditions  $\rightarrow$  **Negligible flame propagation**
- *Mixing controlled combustion until 1<sup>st</sup> peak in AHRR*: MC charge consumed by PC jets penetration ( $\Delta P$ ) and entrainment of surrounding mixture.
- No sequential auto-ignition (Reactivity-controlled combustion).
- Deviations at  $\lambda_{MC} > 2.2 \rightarrow$  Incomplete combustion or lower temperature slowing the kinetics.

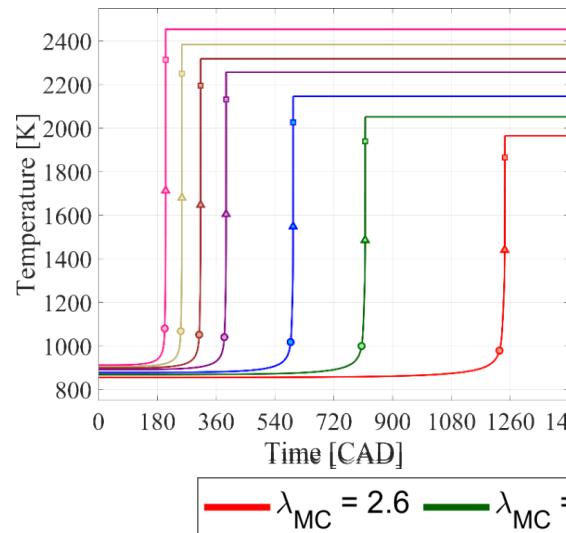
# 0-D CHEMKIN simulations to assess the extent of reactivity controlled combustion in the later stages of heat release.

## CHEMKIN Simulation Approaches

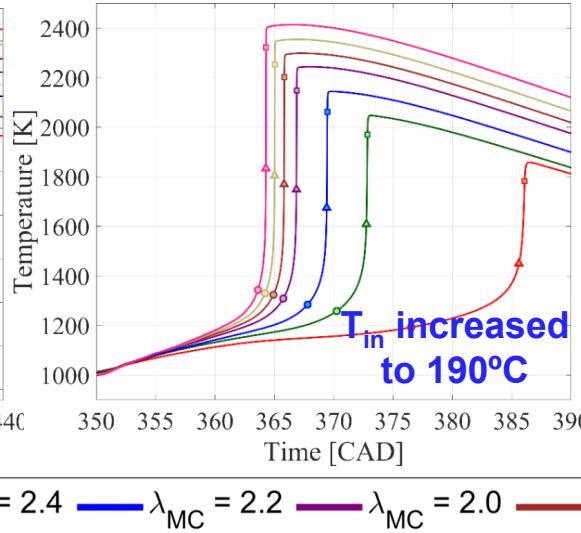


- 3 CHEMKIN simulation approaches using 0-D CHR using AramcoMech version 2.0.
- Unrealistically long ID's** under constant pressure and volume simulations.
- Fuel-air mixture does not ignite** under the imposed pressure profile simulations until *intake T is artificially increased to 190°C*.
- Despite increased intake T, the trends diverge quickly: Estimated durations increase drastically with  $\lambda_{MC}$ ; Experimental values vary by <5 CAD.

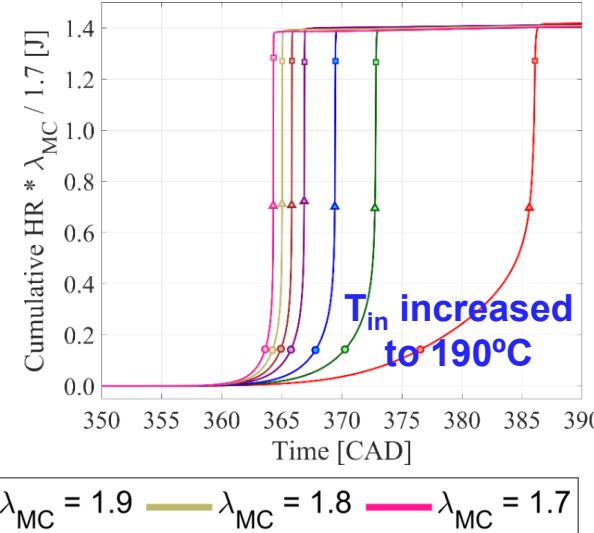
## Constant Volume Simulation



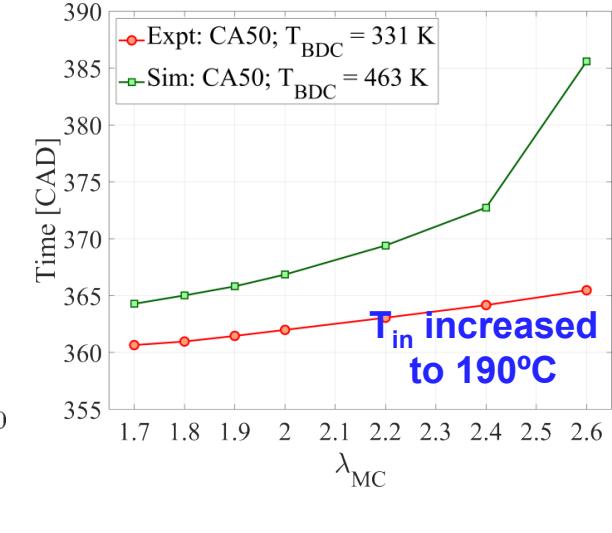
## Imposed Pressure Simulation



## CHR rescaled to $\lambda_{MC} = 1.7$



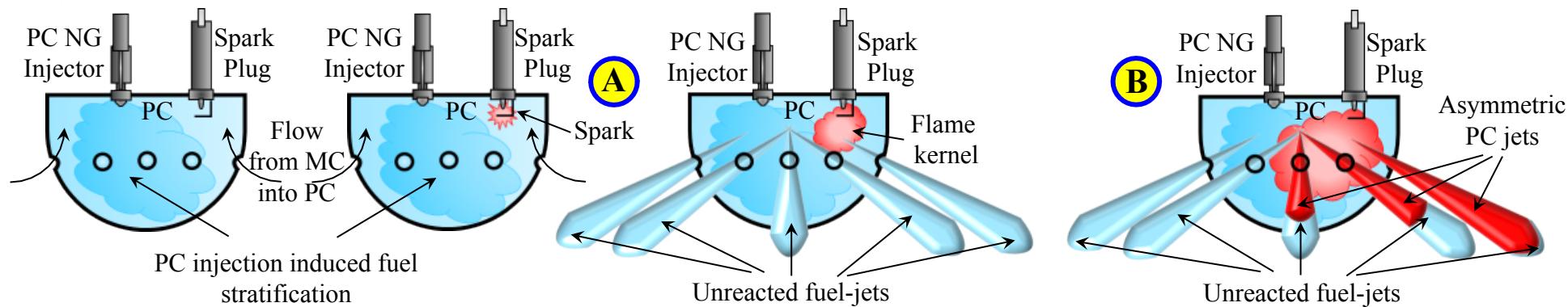
## CA 50 of left over charge



# Conclusions: Elements of initial conceptual model for ignition of fuel-lean MC mixtures by a near-stoichiometric PC

**A. Unburned fuel mixture is pushed out of the PC before MC ignition, creating a spatially symmetric unburned turbulent fuel-jet pattern in the MC.**

**B. MC ignition is generally asymmetric as combustion emerges from the PC at different times in different jets.**



**C. AHRR generally occurs with two stages with a AHRR dip between them.**

- The 1<sup>st</sup> AHRR stage is smaller and shorter in duration and seems to be mixing controlled, driven by jet turbulence.
- The 2<sup>nd</sup> AHRR stage is larger and longer in duration also seems to be mixing controlled, potentially driven by in-cylinder bulk flows and associated turbulence and moderated by chemical kinetics as MC mixtures become leaner.

**D. As the MC lean-limit is approached, the general picture of combustion processes seems to remain essentially unchanged, with extended combustion duration.**