



Source: Tokyo Electric Power Company



Terry Turbopump Expanded Operating Band Program Sandia Modeling Efforts – FY17 Update

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Nuclear Maintenance Applications Center Terry Turbine Users' Group Meeting And Workshop – July 10-14, 2016



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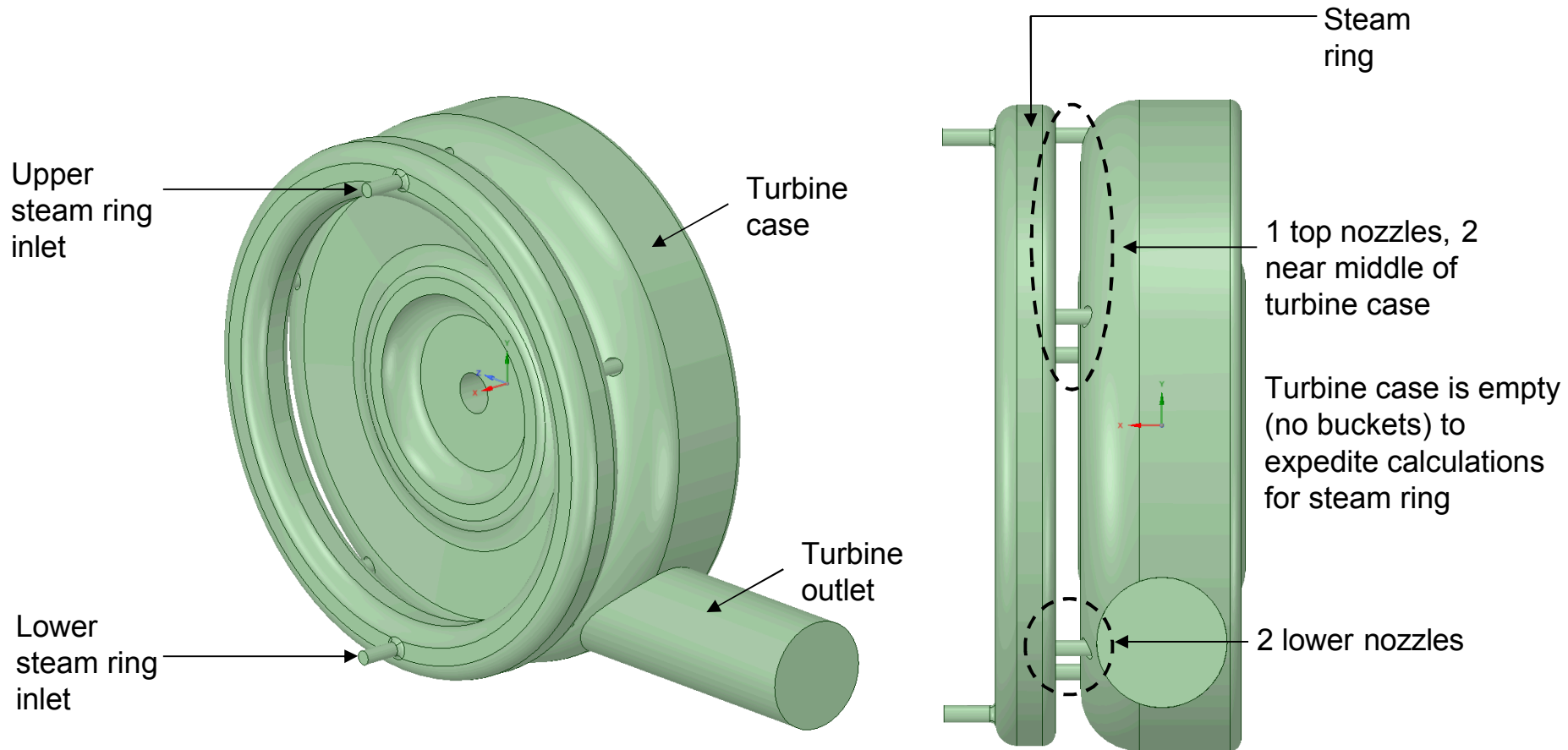
Terry Turbine CFD Efforts

- Corroborate, complement, and inform the experimental and system-modeling efforts
- Using 3D, two-phase, turbulent, compressible, Fluent calculations of key RCIC system components to answer key questions including:
 - RCIC steam ring: does water accumulate and flood?
 - Governor valve: what are the flow characteristics and what is its C_v ?
 - Nozzles and buckets: obtain bucket inlet and outlet velocities for lower steam pressures and for air to support RCIC experiments
 - Turbine wheel windage: does water accumulate around the wheel and retard turbine-pump speed?

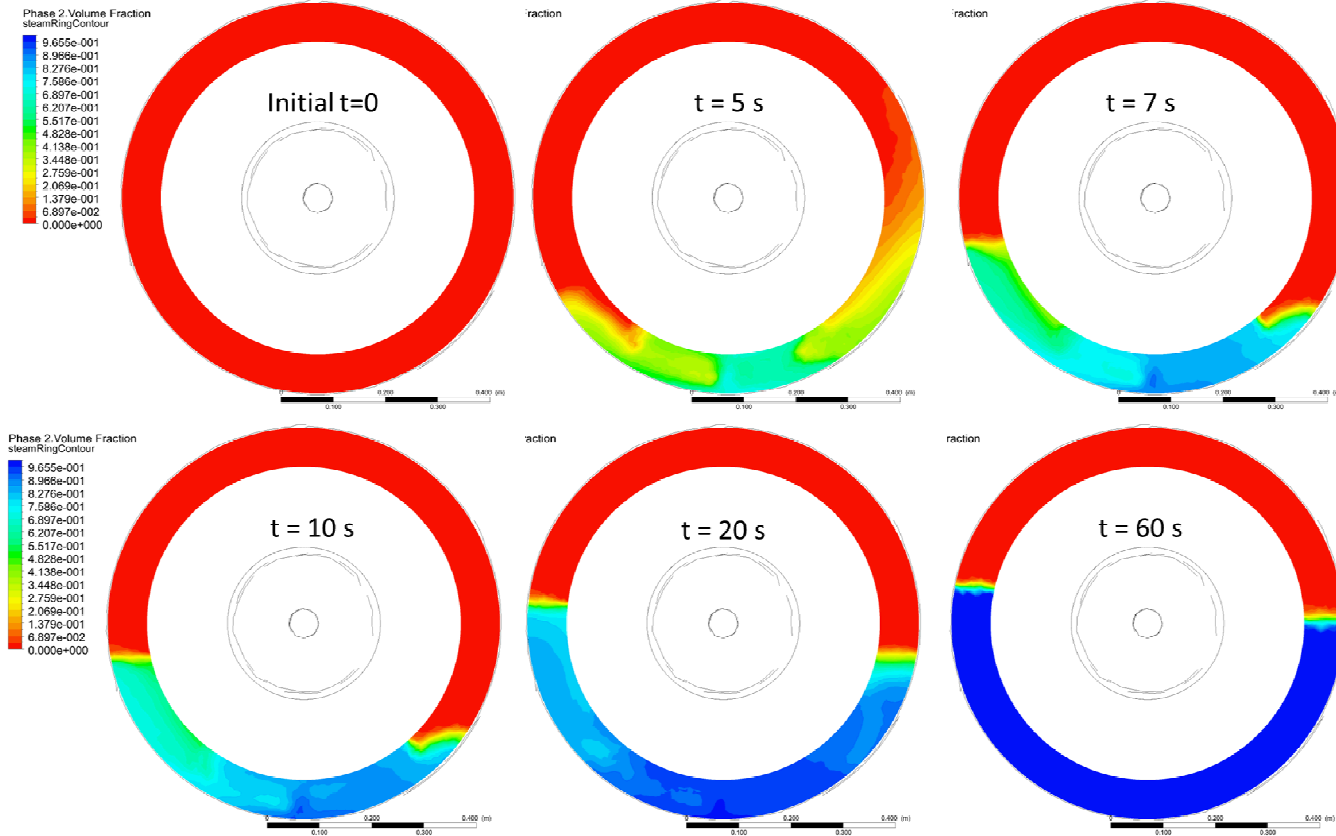
Steam ring analyses

- For a range of pressures and 2-phase conditions (i.e. void fraction), examine flow regime inside steam ring – between governor valve and nozzles
- Calculations largely corroborate system-level assumption concerning flooded nozzles in the lower part of the turbine
 - Water appears to quickly accumulate and result in stratified flow – flooding the lower nozzles – particularly for high liquid content ($> 50\%$ volume frac.)
 - Lower liquid content ($< 5\%$) results in a two-phase mixture covering the lower nozzles
 - Often upper nozzle flow is still 2-phase, but predominately steam
- Transient analyses were performed for a representative steam ring with two inlets and five nozzle
 - Additional calculations could make use of expanded geometry information for the steam ring, particularly the details of its inlets from the governor

Steam ring geometry



Steam ring results



- Mid-plane slice of steam ring – contour of liquid volume fraction (1.0 = all liquid)
- 300 psig inlets
- Top steam ring inlet: 100% steam
- Bottom inlet: 100% liquid
- Saturated vapor-liquid properties assuming no mass/energy transfer between phases
- Fluent's Euler multiphase model (code's most rigorous option)

More steam ring results

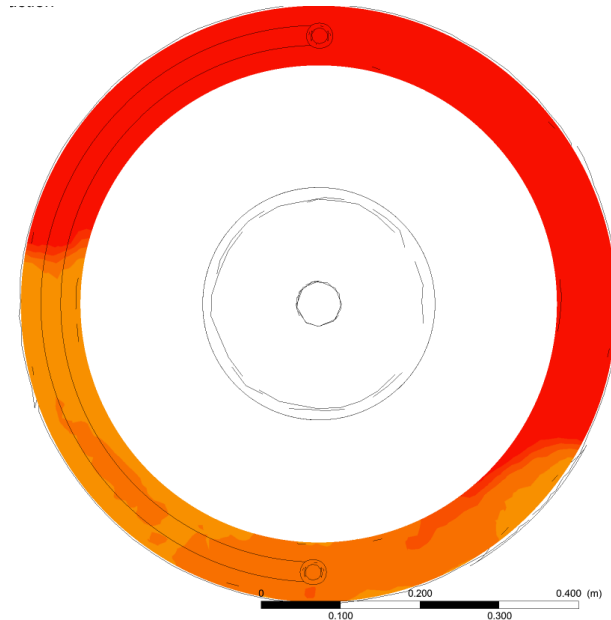
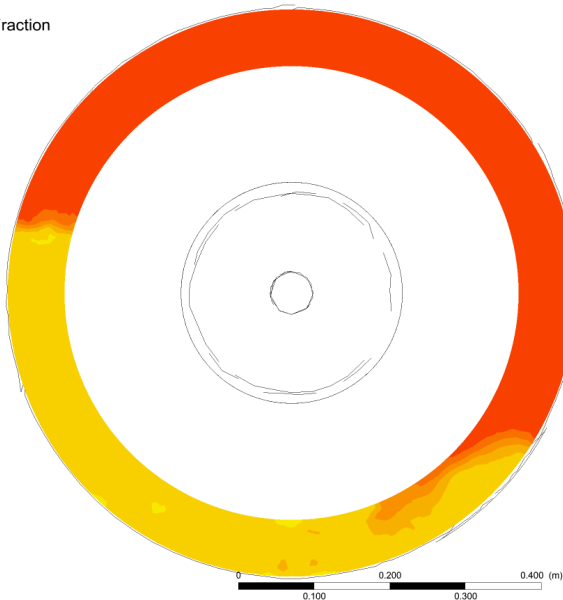
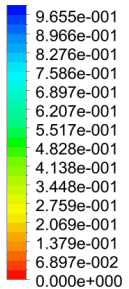
10% avg. liquid volume fraction across inlets

- Lower nozzles flow more water (20-30% liquid)
- Uppers nozzles flow more steam (only ~5% liquid)

5% avg. liquid volume fraction across inlets

- Lower nozzles: 5-20% liquid
- Uppers nozzles are almost fully steam flow

Phase 2, Volume Fraction
steamRingContour

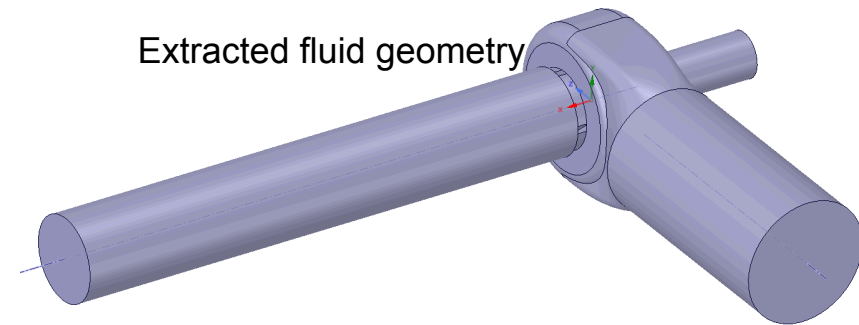
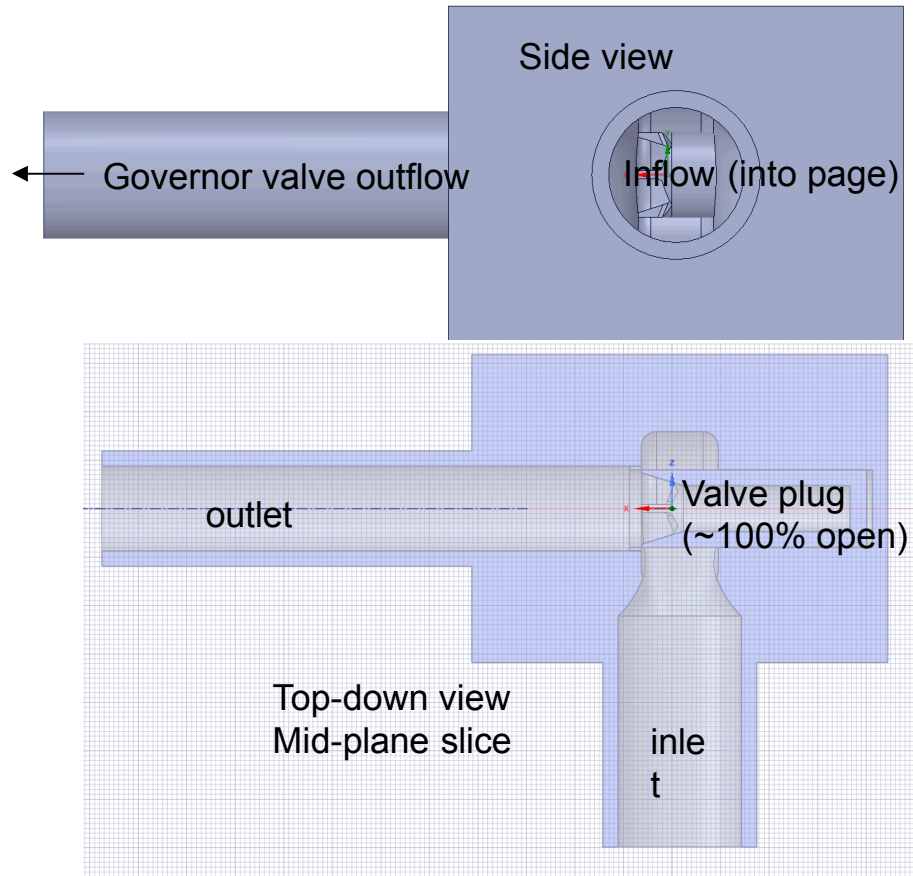


Transient calculations after 60 seconds
(initial condition is all saturated steam at 300 psig)

Governor valve calculations

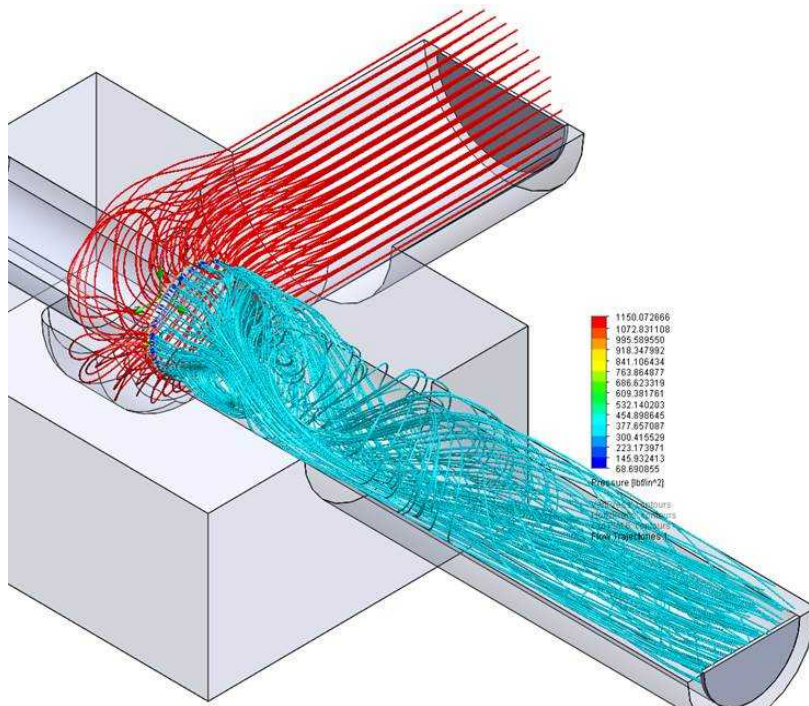
- Obtain C_v information for governor valve
 - Leverage CAD/flow modeling using SolidWorks from 2016
- Examine two-phase flow characteristics
 - Expanded model may be able to inform analysis of the steam ring
 - Need piping details from governor valve to steam ring inlet(s)
- Will enable rigorous benchmarking to Unit 2 strip chart data before power was lost
 - Allows for more calibration of uncertain model inputs (multipliers, friction coefficients, etc.)
 - May increase understanding of Unit 2 avoiding overspeed trip after loss of power

Governor valve geometry



Preliminary governor valve results

From 2016 using Solidworks Flow
Streamlines with pressure contour
1100 psig inlet; 300 psig outlet



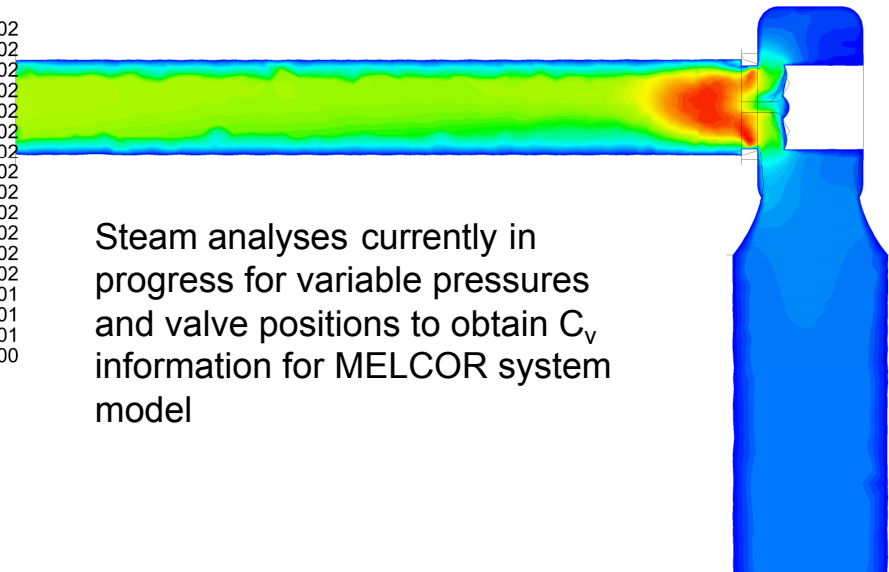
Fluent test calculation using ideal-gas steam to validate mesh and
geometry

Velocity magnitude contour for 1100 psig inlet and 300 psig outlet
Valve position at about 70% open

Velocity
Contour 1

4.756e+002
4.459e+002
4.162e+002
3.864e+002
3.567e+002
3.270e+002
2.973e+002
2.675e+002
2.378e+002
2.081e+002
1.784e+002
1.486e+002
1.189e+002
8.918e+001
5.945e+001
2.973e+001
0.000e+000

[m s⁻¹]



Steam analyses currently in
progress for variable pressures
and valve positions to obtain C_v
information for MELCOR system
model

Nozzle calculations with low pressure steam

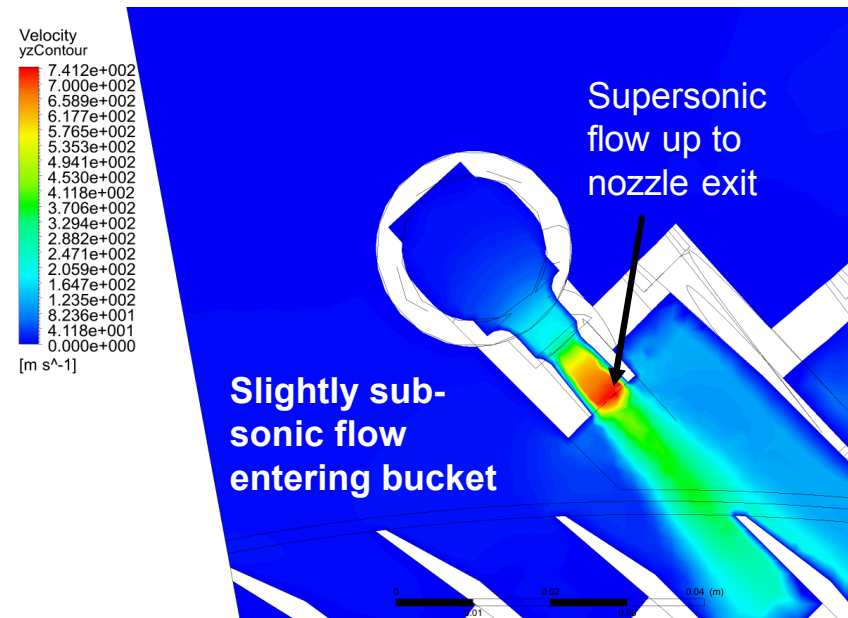
- Inform/support experimental efforts
 - Steam pressures from 40 to 70 psia
 - Outlet 15 psia

- Also examine air flow
 - 40 to 70 psia
 - Outlet 15 psia

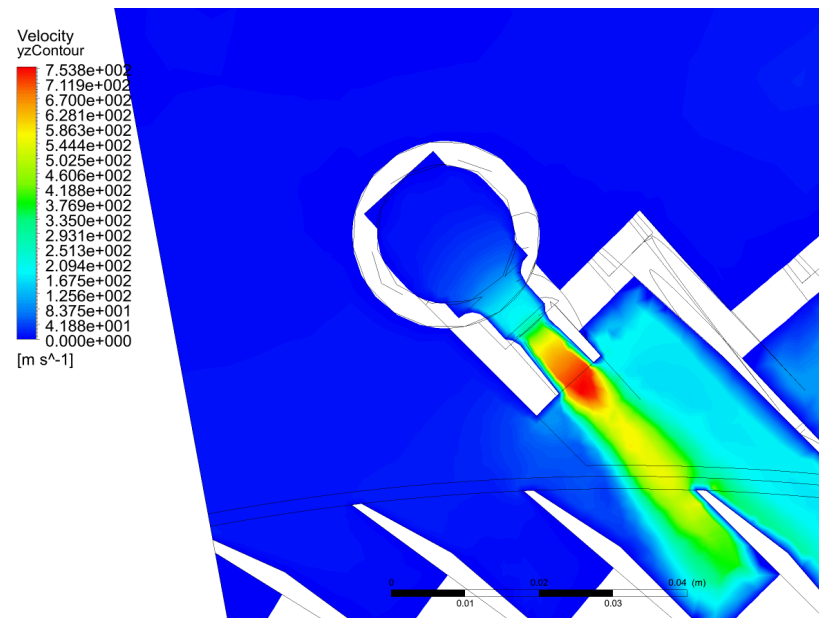
Some low steam pressure nozzle results

Such low pressure drop of the Terry nozzle typically results in shock formation near the nozzle exit plane – supersonic velocities quickly reduce to sonic speed at nozzle exit

40 psia steam inlet



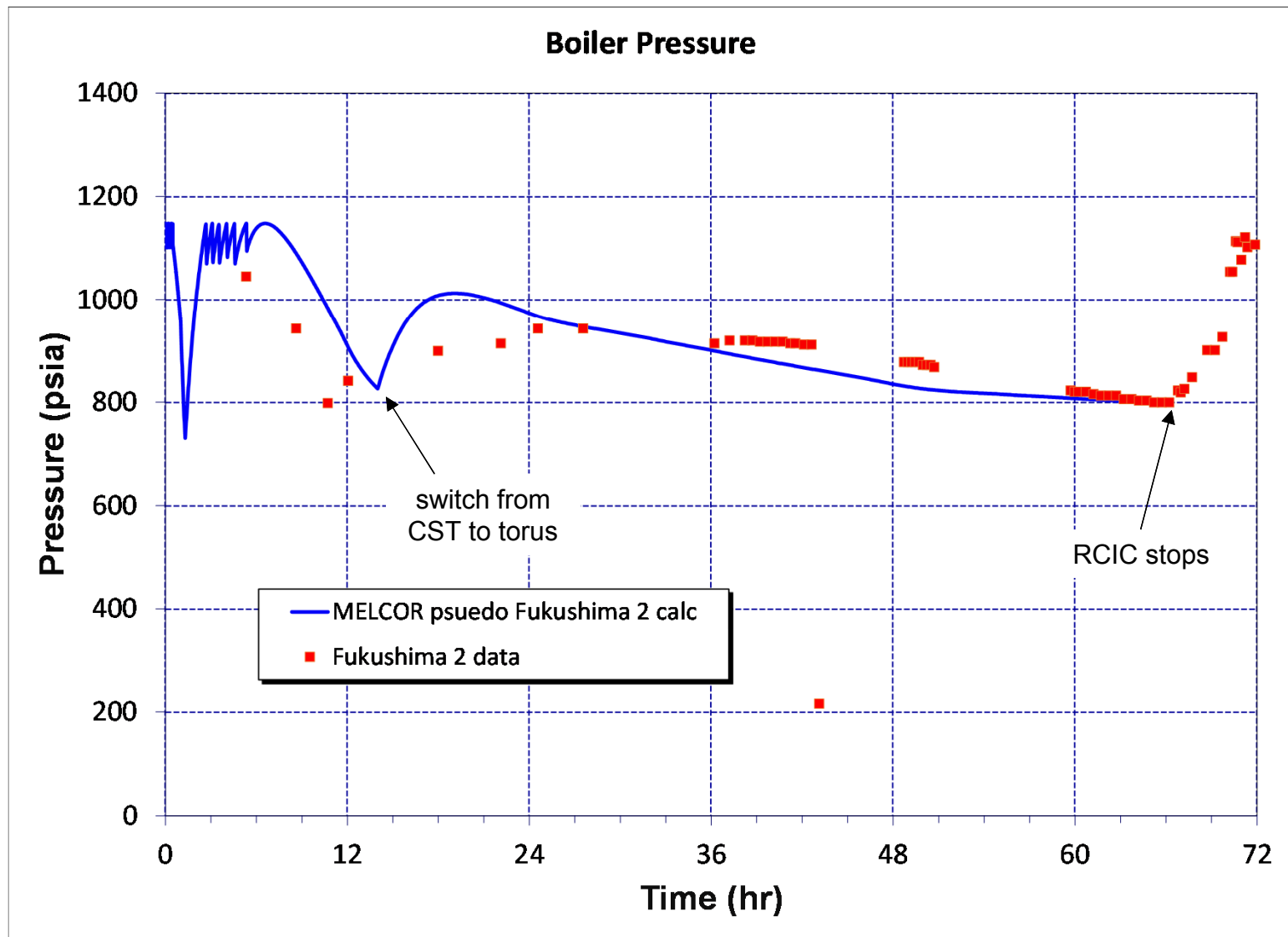
70 psia steam inlet;
higher pressure drop 'pushes' shock out of the nozzle



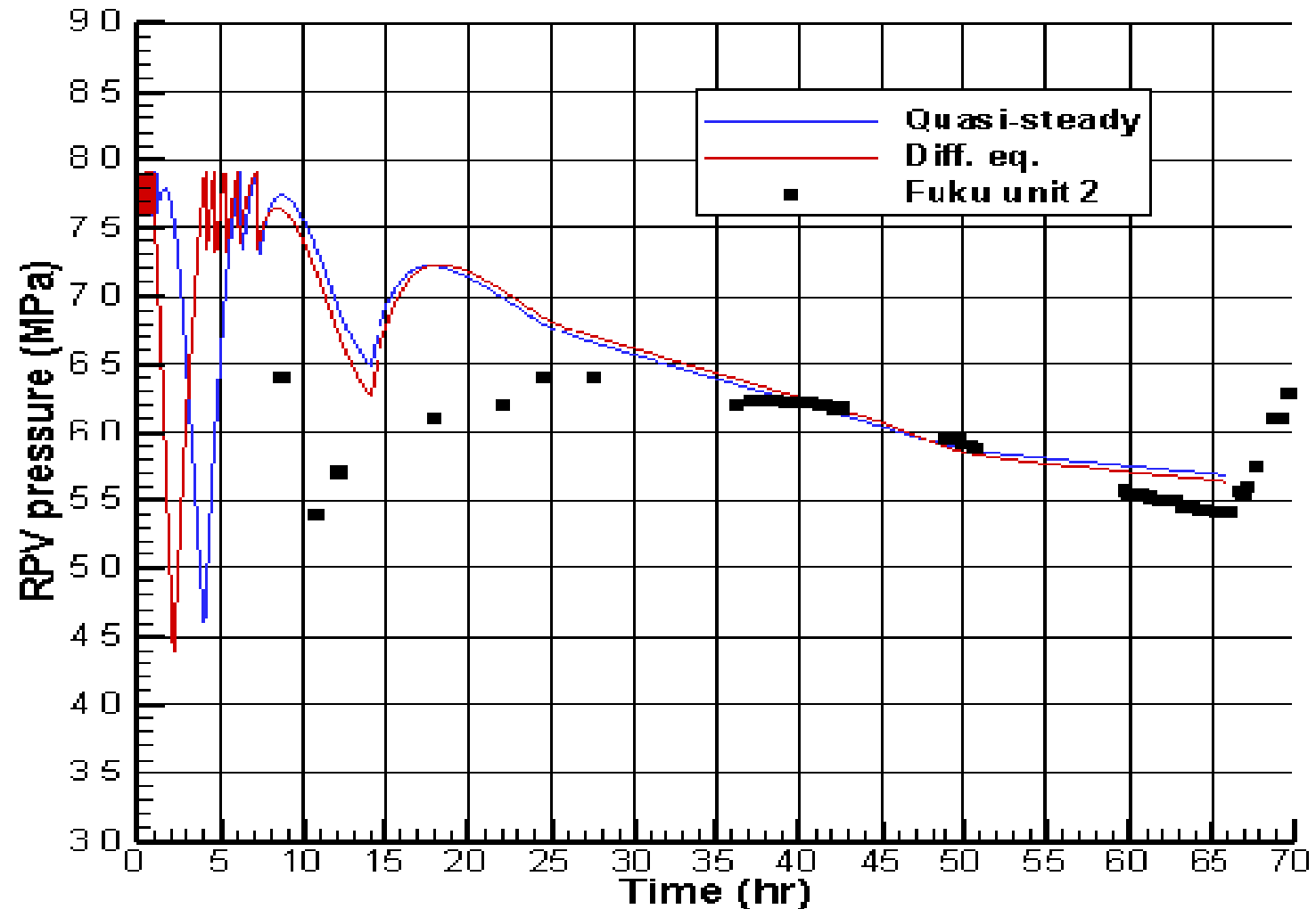
System-level Modeling

- These are test models and test results
- This work is explorative
- Investigating several different explanations and modeling approaches
 - The results are going to change
- The results are qualitative
 - They reproduce the general trends of Fukushima Unit 2

Fukushima 2 MELCOR Model

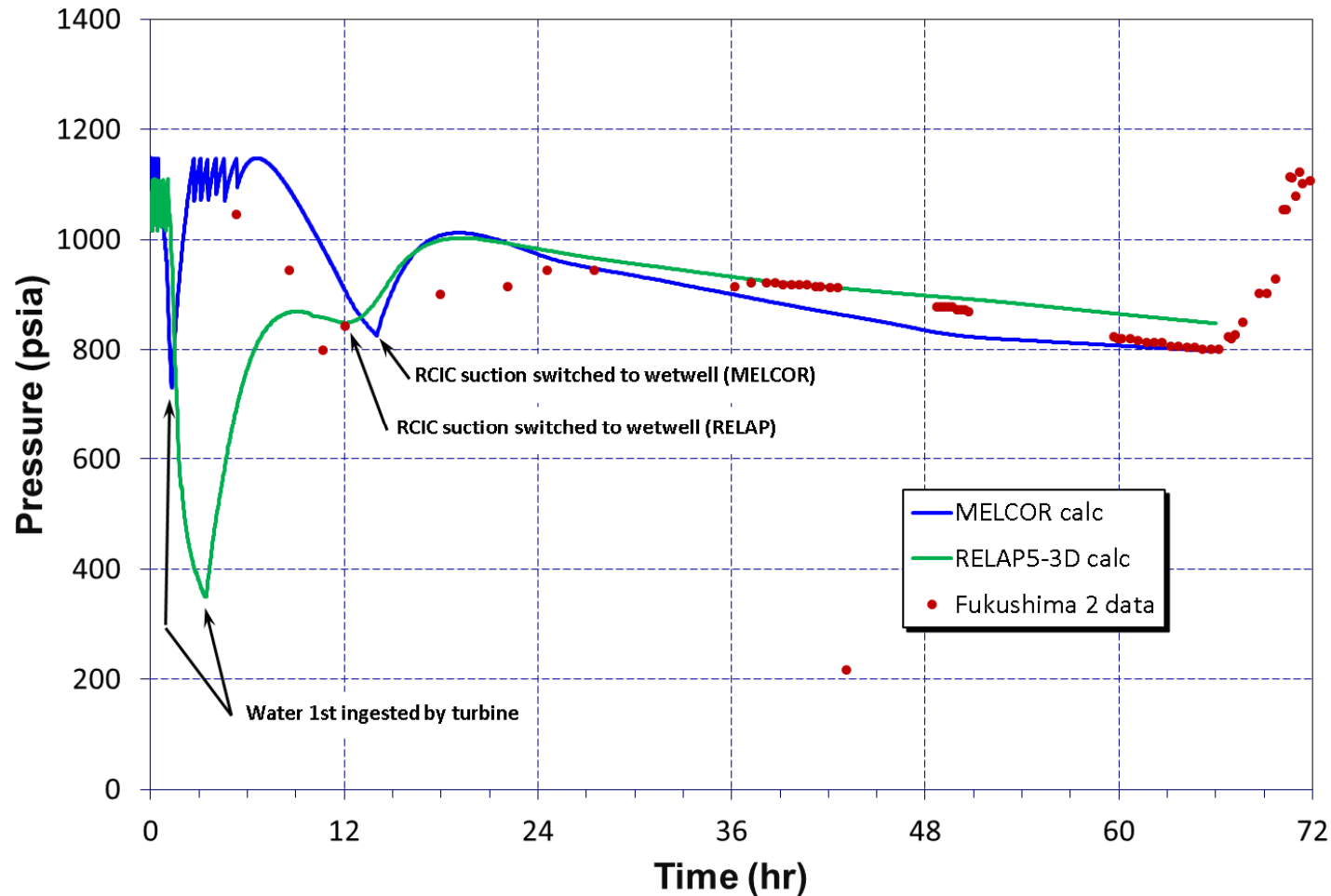


Quasi-Steady vs Time-Dependent Fukushima 2 MELCOR Calculations



RELAP5-3D Modeling

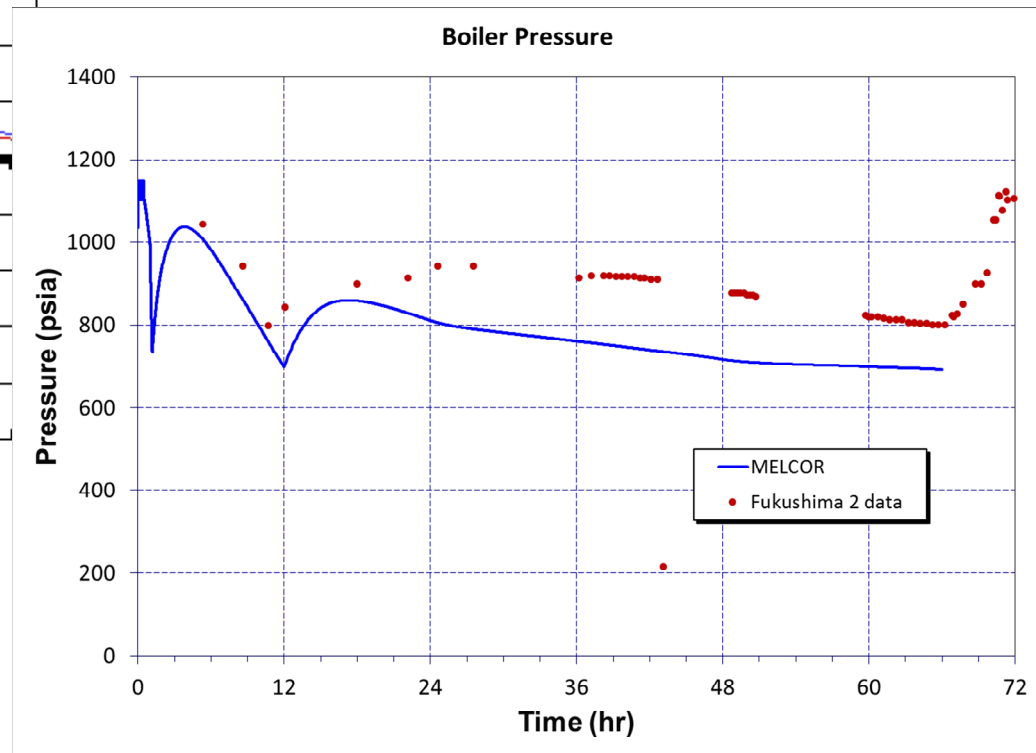
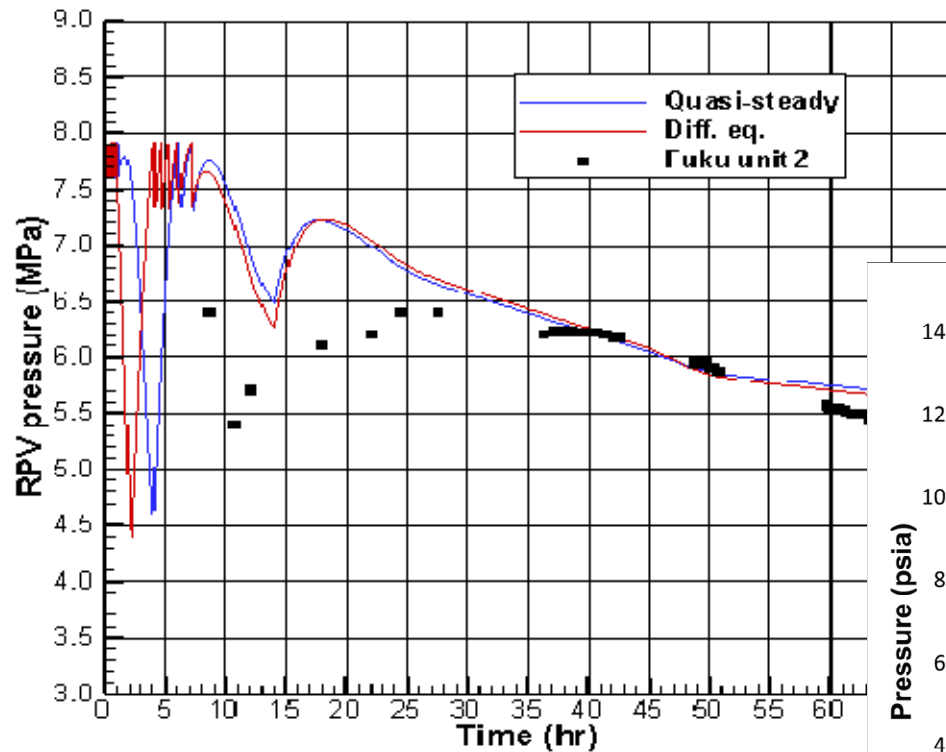
Fukushima II MELCOR and RELAP5-3D Simulations - RPV Pressure



Similar results – a wide open governor & nozzle size seem critical

Homologous Pump Curve Result

- Better represents the initial Unit 2 data, but not the later phase

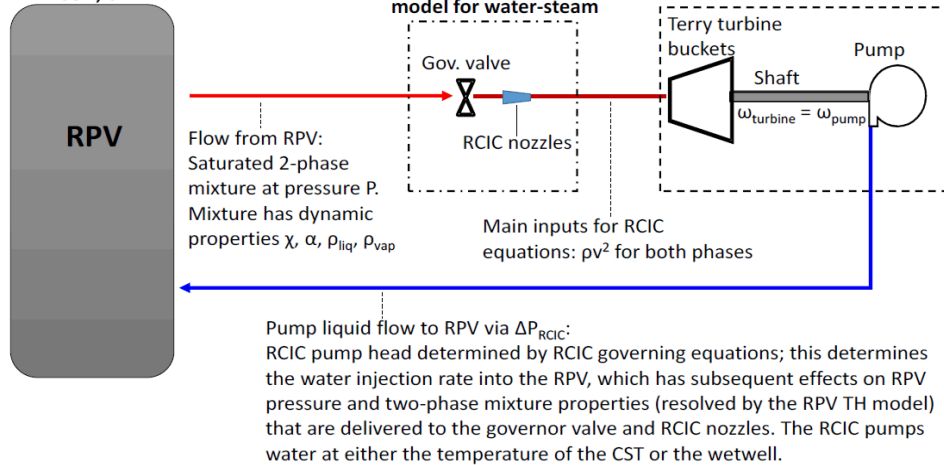


Questions

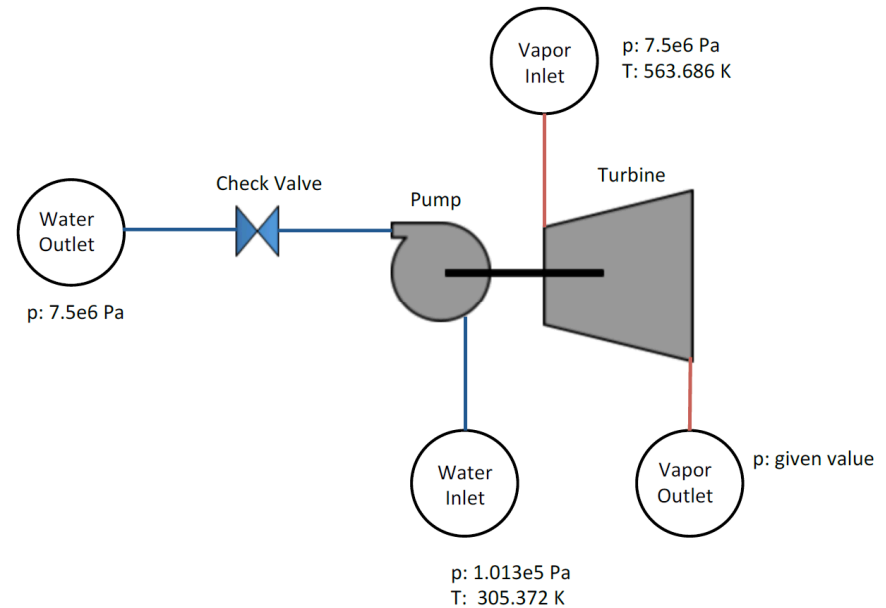
1) Models for RPV thermal-hydraulics: simple equations, MELCOR, or RELAP

2) Choked flow: two phase sonic velocity model for water-steam

3) RCIC governing equations



Simplified representation of physical coupling in MELCOR test model



RELAP-7 Terry turbine RCIC system test model