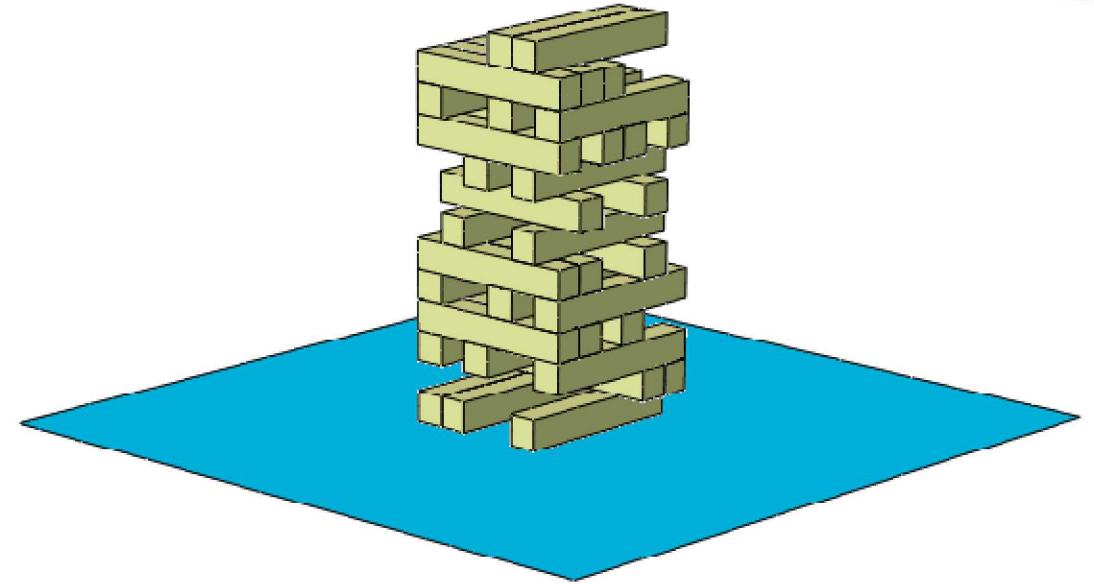




Substructuring in ABAQUS



Ben Moldenhauer¹, Matt Allen¹,

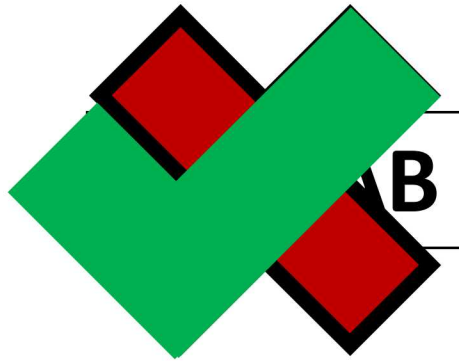
Dan Roettgen² & Brian Owens²

(1) University of Wisconsin – Madison

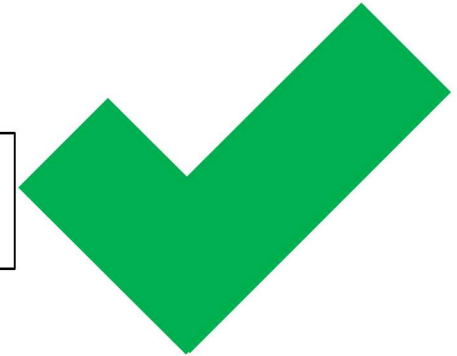
(2) Sandia National Labs

Motivation

- Usual substructuring procedure involves importing FEM results into MATLAB to do all calculations
 - Bookkeeping can be cumbersome, and yields results with limited usability



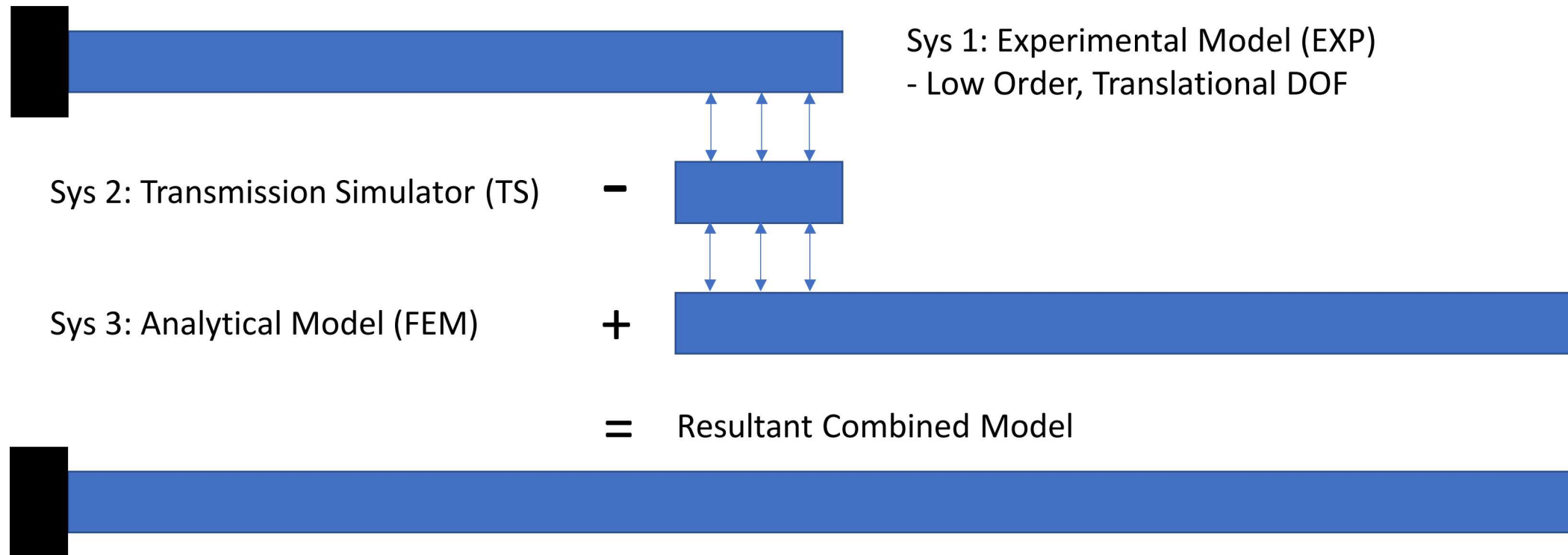
ABAQUS



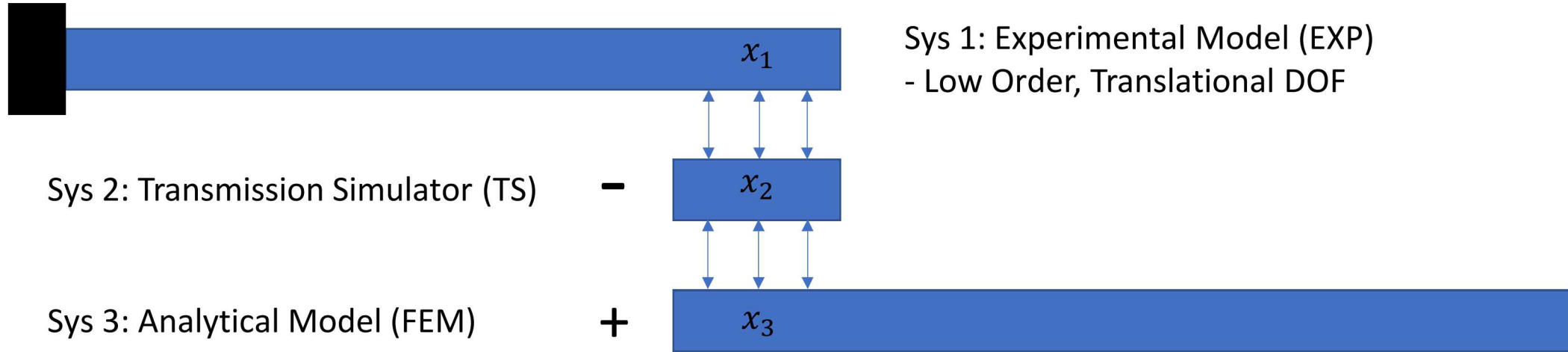
- Perform substructuring calculations during ABAQUS modal solution
 - Allows for much greater post processing options, i.e. 'Full Field' Results - Stress, Strain, Displacement, etc.

Substructuring Theory – Transmission Simulator Method

- Substructuring techniques allow for the dynamics of several subcomponents to be joined, approximating the assembly dynamics



Transmission Simulator Math

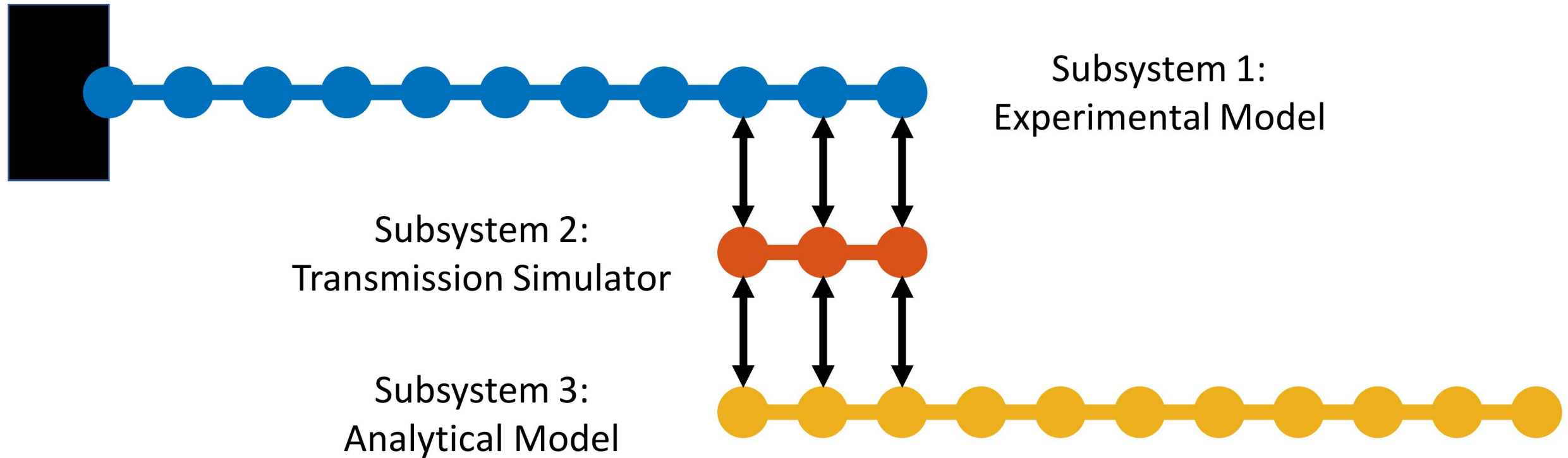


$$\begin{bmatrix} -1 & 1 & 0 \\ 0 & 1 & -1 \end{bmatrix} \begin{bmatrix} \{x_1\} \\ \{x_2\} \\ \{x_3\} \end{bmatrix} = [0] \rightarrow \begin{bmatrix} -\Phi_1 & \Phi_2 & 0 \\ 0 & \Phi_2 & -\Phi_3 \end{bmatrix} \begin{bmatrix} \{q_1\} \\ \{q_2\} \\ \{q_3\} \end{bmatrix} = [0] \rightarrow \begin{bmatrix} \Phi_2^\dagger & 0 \\ 0 & \Phi_2^\dagger \end{bmatrix} \begin{bmatrix} -\Phi_1 & \Phi_2 & 0 \\ 0 & \Phi_2 & -\Phi_3 \end{bmatrix} \begin{bmatrix} \{q_1\} \\ \{q_2\} \\ \{q_3\} \end{bmatrix} = [0]$$

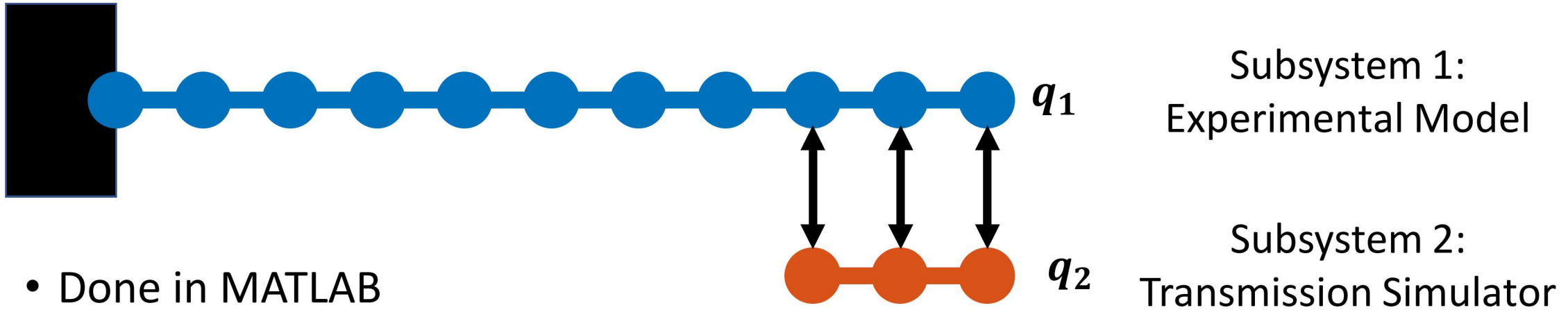
$$\rightarrow \begin{bmatrix} -\Phi_2^\dagger \Phi_1 & \Phi_2^\dagger \Phi_2 & 0 \\ 0 & \Phi_2^\dagger \Phi_2 & -\Phi_2^\dagger \Phi_3 \end{bmatrix} \begin{bmatrix} \{q_1\} \\ \{q_2\} \\ \{q_3\} \end{bmatrix} = [0] \rightarrow \begin{bmatrix} -\Phi_2^\dagger \Phi_1 & I & 0 \\ 0 & I & -\Phi_2^\dagger \Phi_3 \end{bmatrix} \begin{bmatrix} \{q_1\} \\ \{q_2\} \\ \{q_3\} \end{bmatrix} = [0]$$

$$\begin{aligned} Bq &= 0 \\ \text{null}(B) &= L \\ \hat{M} &= L^T M L, \hat{K} = L^T K L \\ [\Phi, \lambda] &= \text{eig}(\hat{K}, \hat{M}) \end{aligned}$$

Analytical Test Case & Method Procedure



Uncouple TS From Cantilever Beam



- Done in MATLAB
- Translation DOF constrained
- Constrained M and K can be “corrected” to be positive definite and yield real Eigen solutions

$$\begin{bmatrix} -\Phi_2^\dagger \Phi_1 & I \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \end{bmatrix} = [0]$$

→ $\Phi_{\text{Uncoupled}} \ \& \ \omega_{n\text{Uncoupled}}$

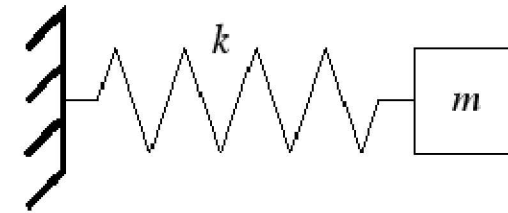
These must be real valued to import the modal model into ABAQUS

Importing Uncoupled Model into ABAQUS

- MATLAB script writes an auxiliary input file that:

- Generates nodes for each modal DOF q_{1-2}

- Assign unit point mass $m = 1$
- Attach grounded spring with $k = \omega_{n_{1-2}}^2$
- Constrain DOF 2-6

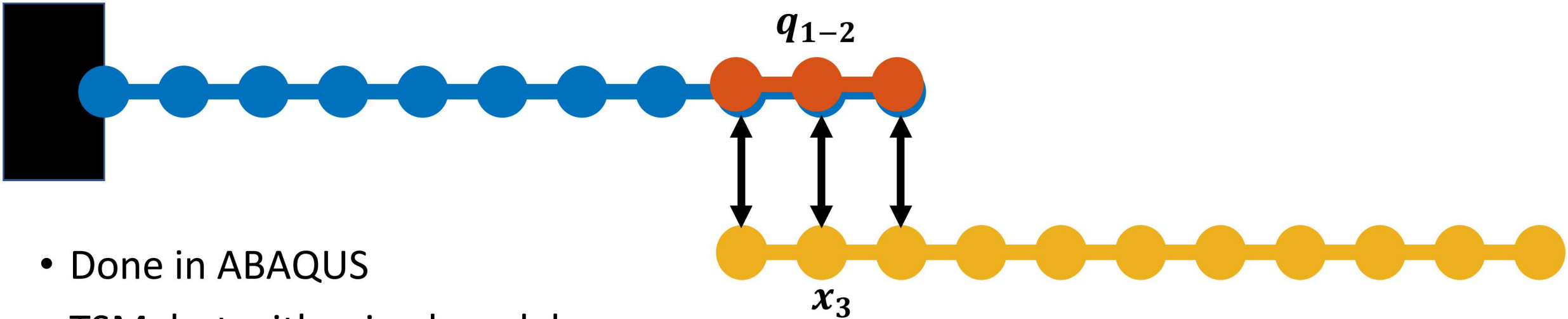


- Constraints: Linear Equation Multi-Point Constraints

- Equation terms defined by: $\begin{bmatrix} -\Phi_2^\dagger \Phi_{1-2} & \Phi_2^\dagger \end{bmatrix} \begin{bmatrix} q_{1-2} \\ x_3 \end{bmatrix} = [0]$
- Must put equations in RREF - ABAQUS eliminates first DOF in each EQ, so it can only appear once



Couple Analytical Subsystem onto Previous Result

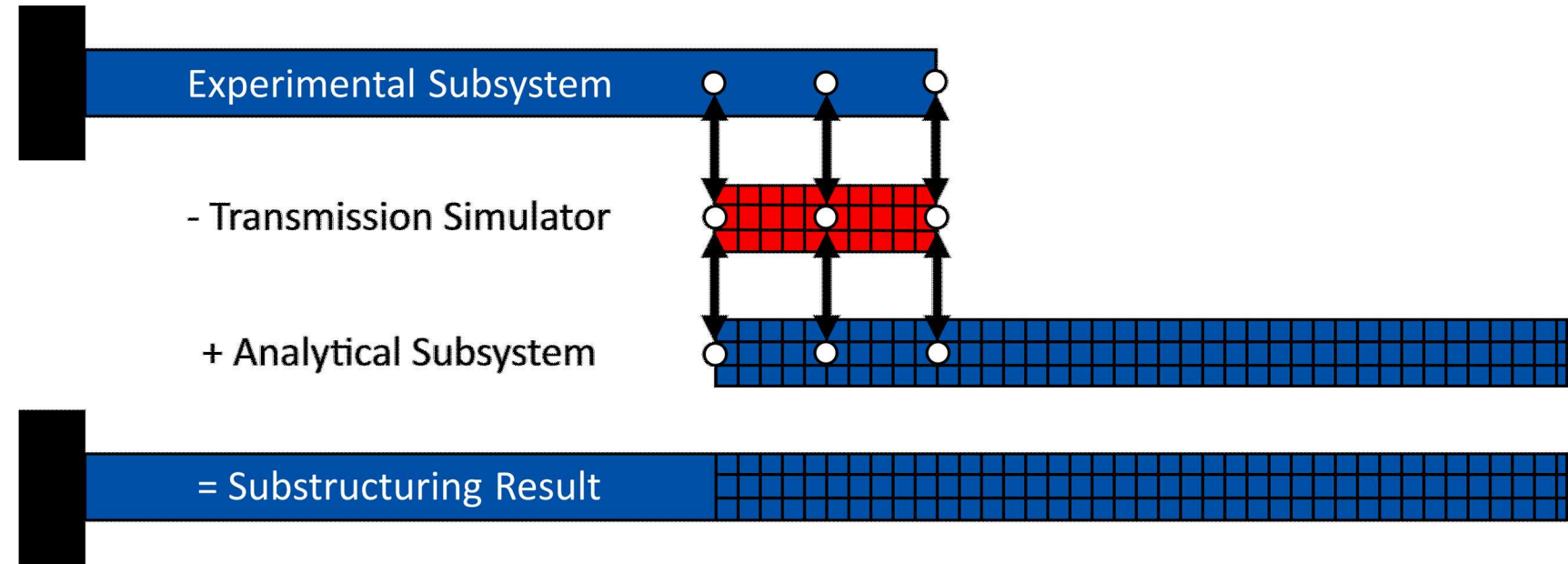


- Done in ABAQUS
- TSM, but with mixed modal and physical coordinates
- Yields prediction of resultant combined beam (a longer cantilever beam)

$$\begin{bmatrix} -\Phi_2^\dagger \Phi_{\text{Uncoupled}} & \Phi_2^\dagger \end{bmatrix} \begin{bmatrix} q_{\text{Uncoupled}} \\ x_3 \end{bmatrix} = [0]$$

$$\rightarrow \Phi_{\text{Coupled}} \ \& \ \omega_{n\text{Coupled}}$$

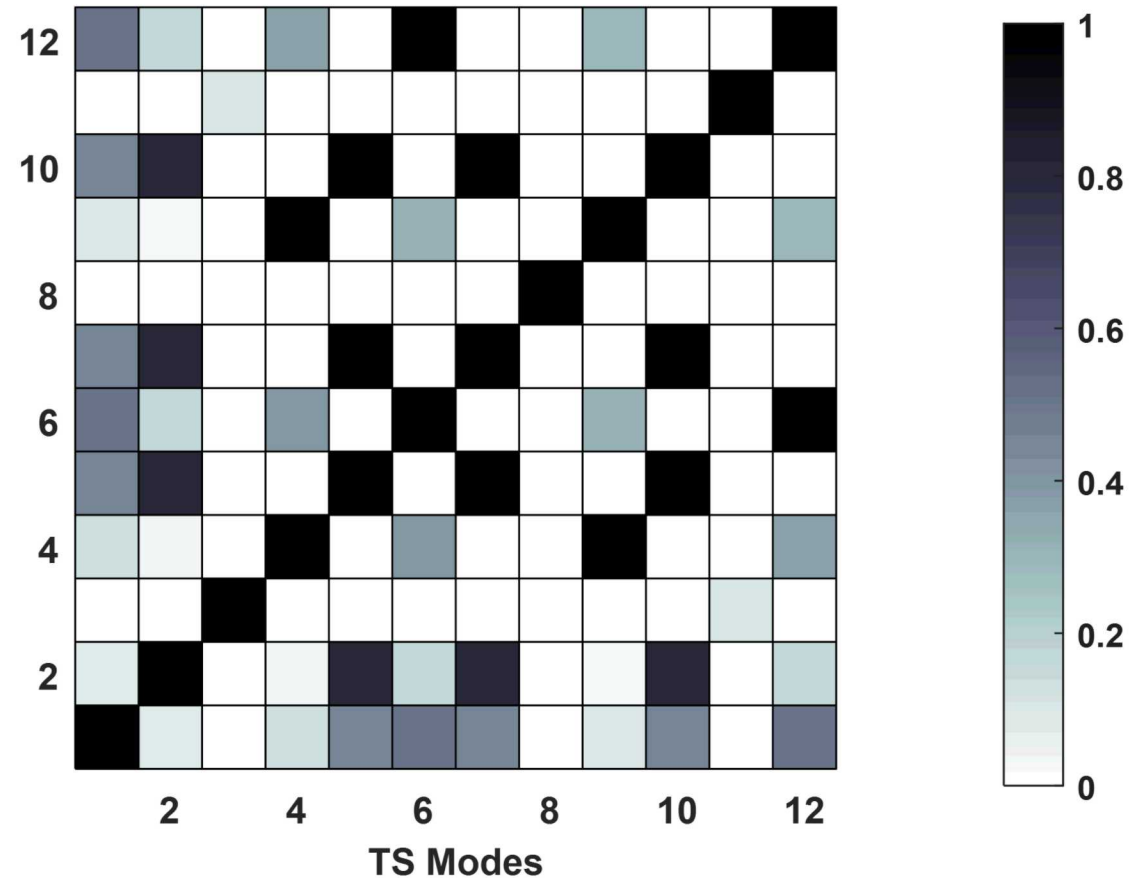
ABAQUS Beam Models



- 2 Node Linear Beam Elements (B21)
- Aluminum properties:
 - 2700 [kg/m³], .33 [-], 70e9 [GPa]
 - 1 [cm²] Cross Section
- EXP: 1 [m], 500 Elements, cantilever
- TS: 0.3 [m], 500 Elements, free-free
- FEM: 1 [m], 500 Elements, free-free
- Truth: 1.8 [m], 900 Elements, cantilever

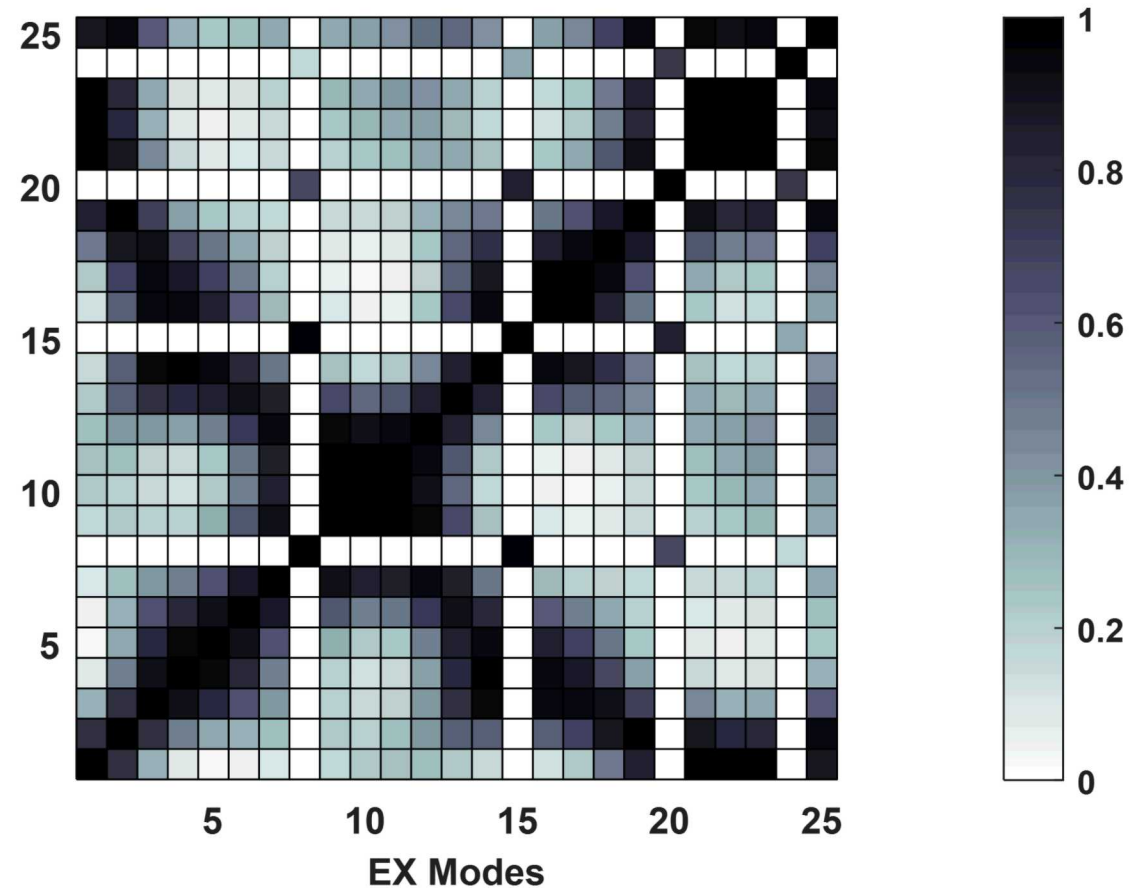
Beam Results: Subsystem Modes Used

- Experimental Model:
 - 10 Flexural Modes - 2050 Hz
 - 4 Axial Modes – 8910 Hz
- Transmission Simulator:
 - 3 RBMs
 - 1 Flexural Mode – 1297 Hz
 - 1 Axial Mode – 12730 Hz
- Analytical Model: All Modes

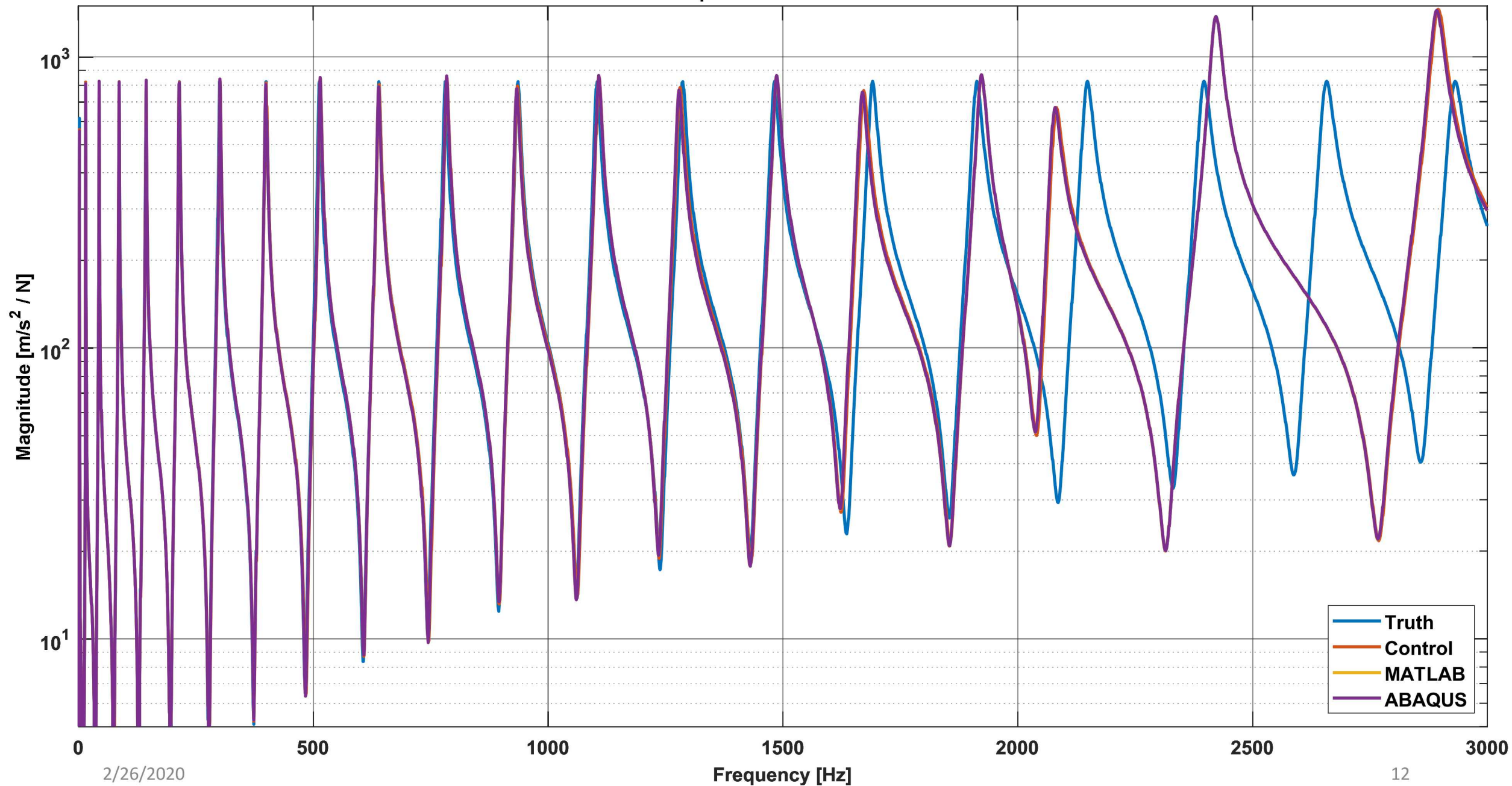


Beam Results: Subsystem Modes Used

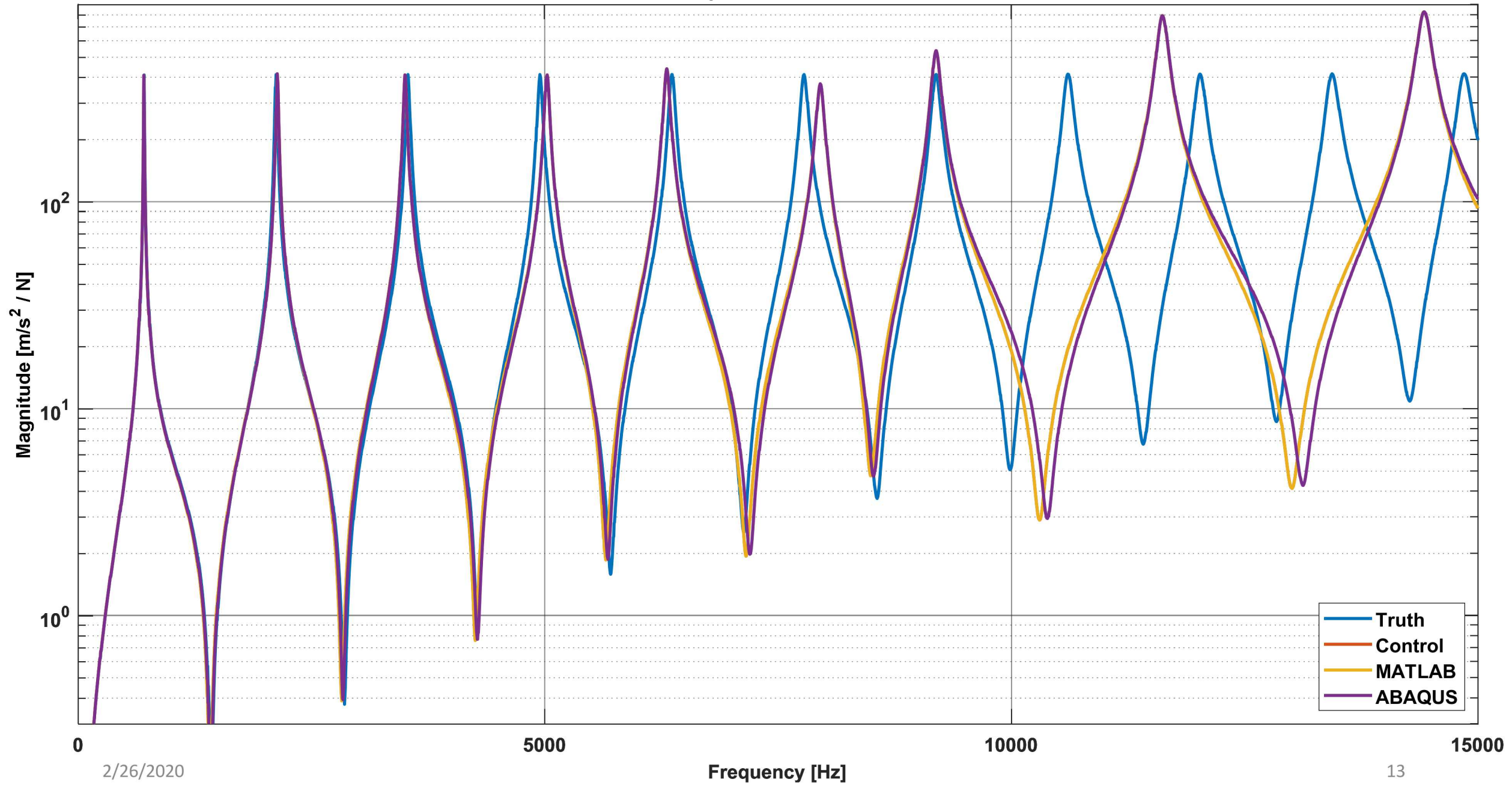
- Experimental Model:
 - 10 Flexural Modes - 2050 Hz
 - 4 Axial Modes – 8910 Hz
- Transmission Simulator:
 - 3 RBMs
 - 1 Flexural Mode – 1297 Hz
 - 1 Axial Mode – 12730 Hz
- Analytical Model: All Modes



Beam Tip Flexural Drive Point FRF

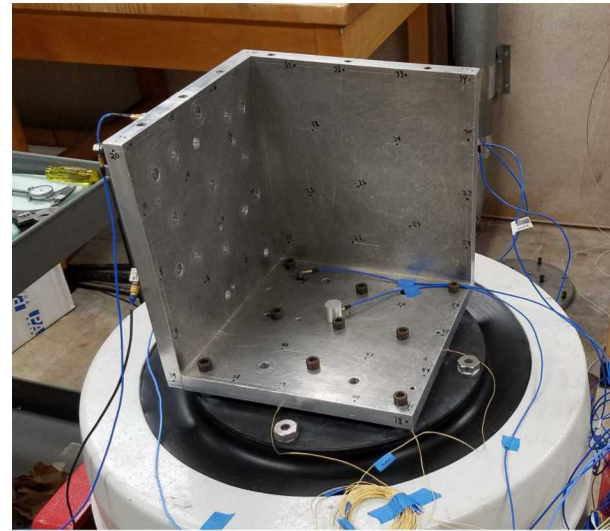


Beam Tip Axial Drive Point FRF

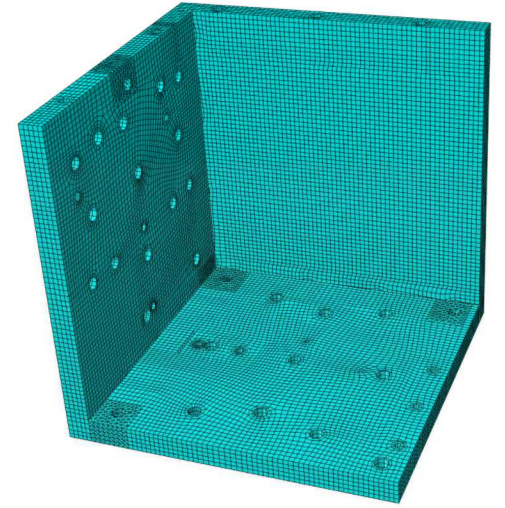


Half Cube Test Case

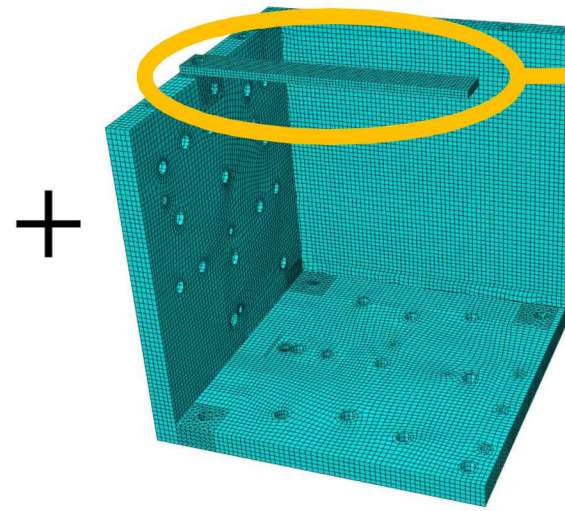
- Experimental data and FEMs from project looking at attaching fixtures to a shaker
- Challenging scenario based on experimental data and a dynamically complex structure
- Experimental Truth data is available to compare with



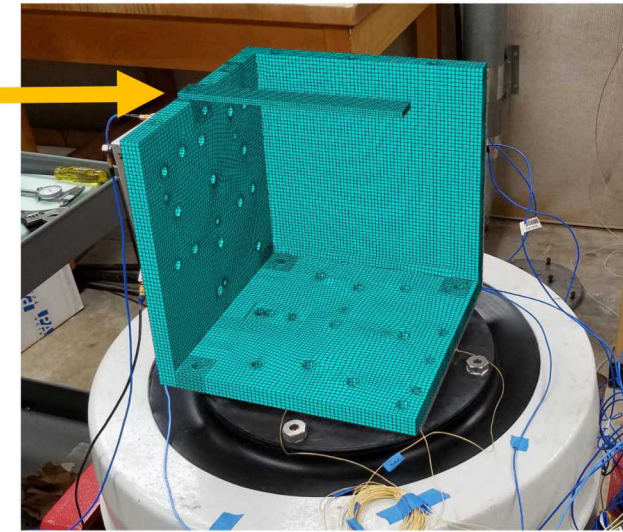
Experimental Subsystem



Transmission Simulator



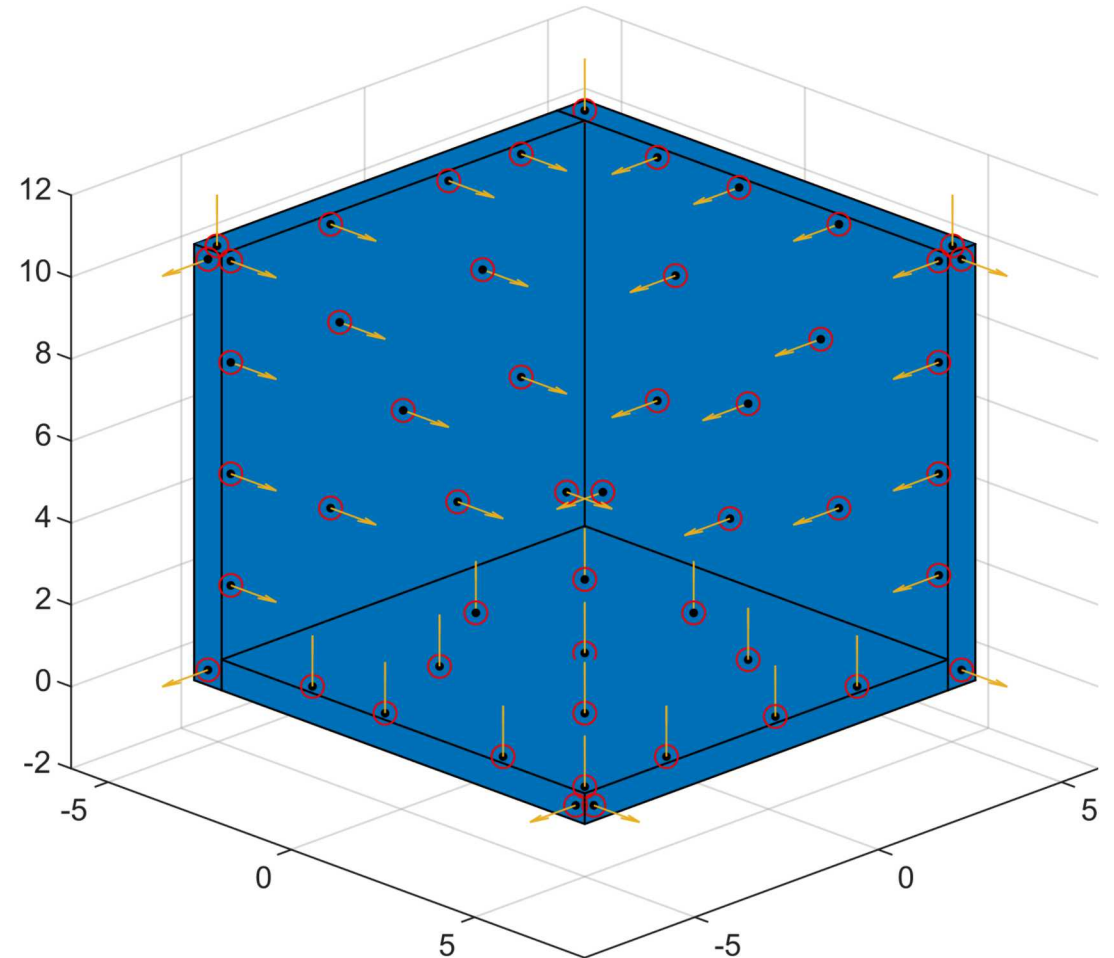
Analytical Subsystem



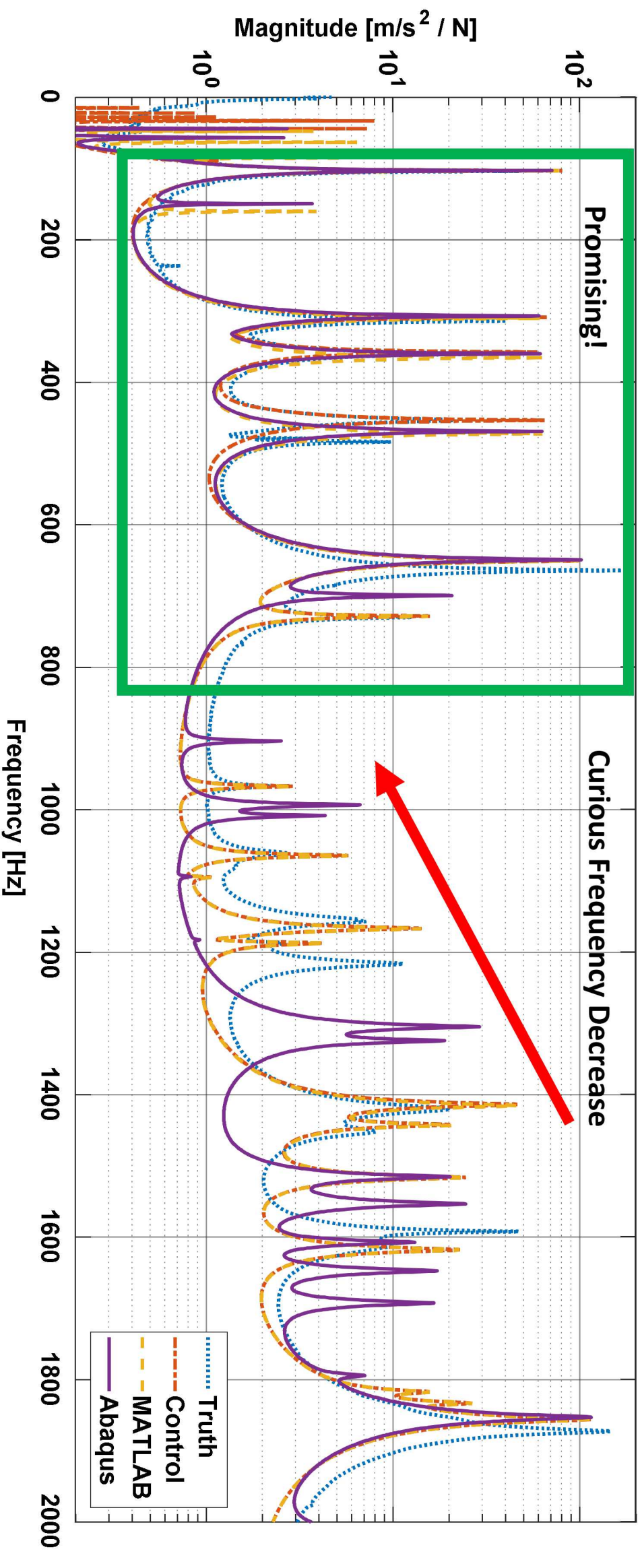
Substructured Model

Mode Selection for the Subsystems

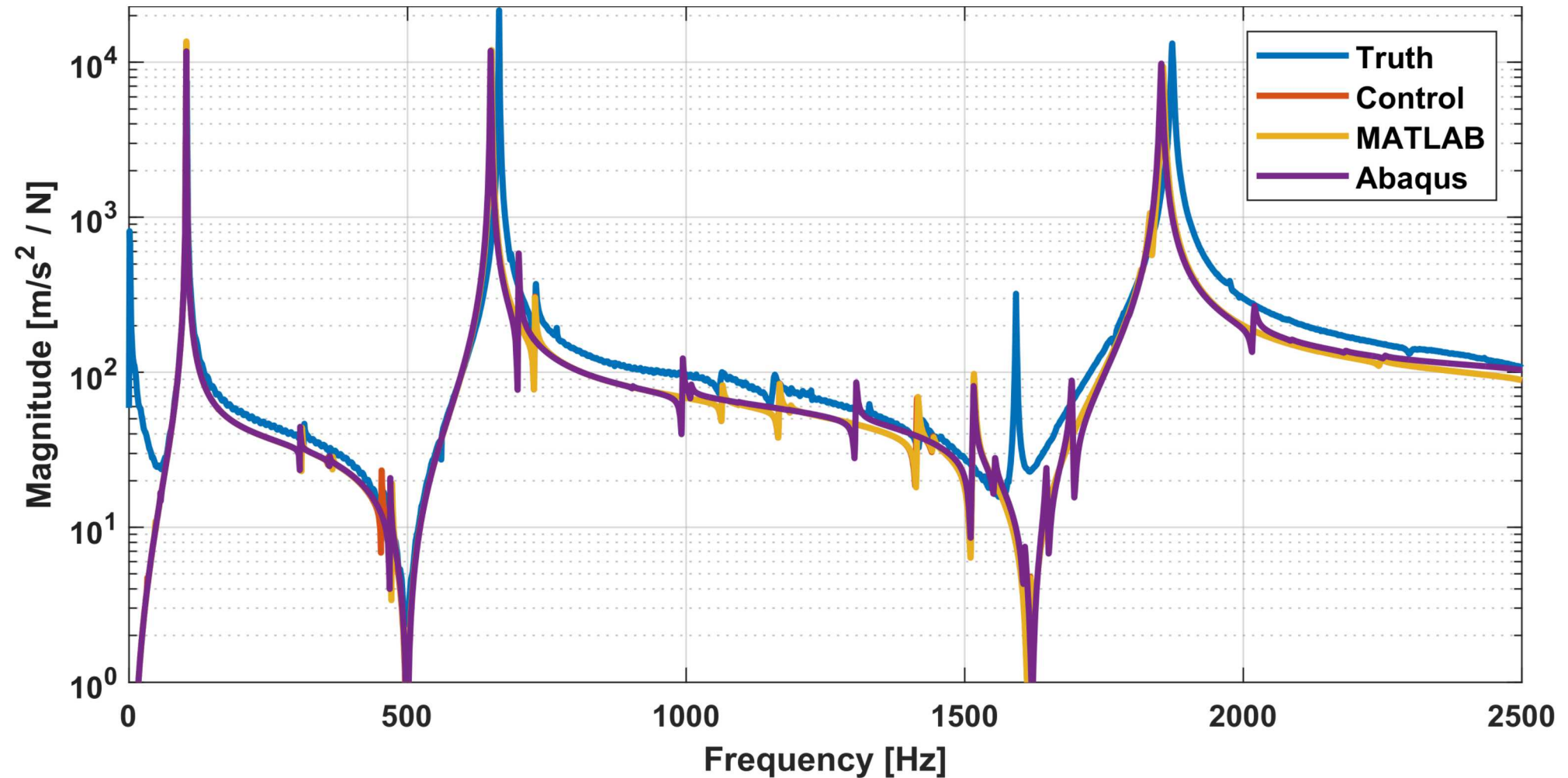
- 51 DOF used in experiment
- Experimental: 32 modes up to 3400 Hz
- TS: 22 modes up to 3800 Hz
- Analytical: All (100 modes up to 12200 Hz when imported into MATLAB)



Truth Data vs Substructuring Predictions

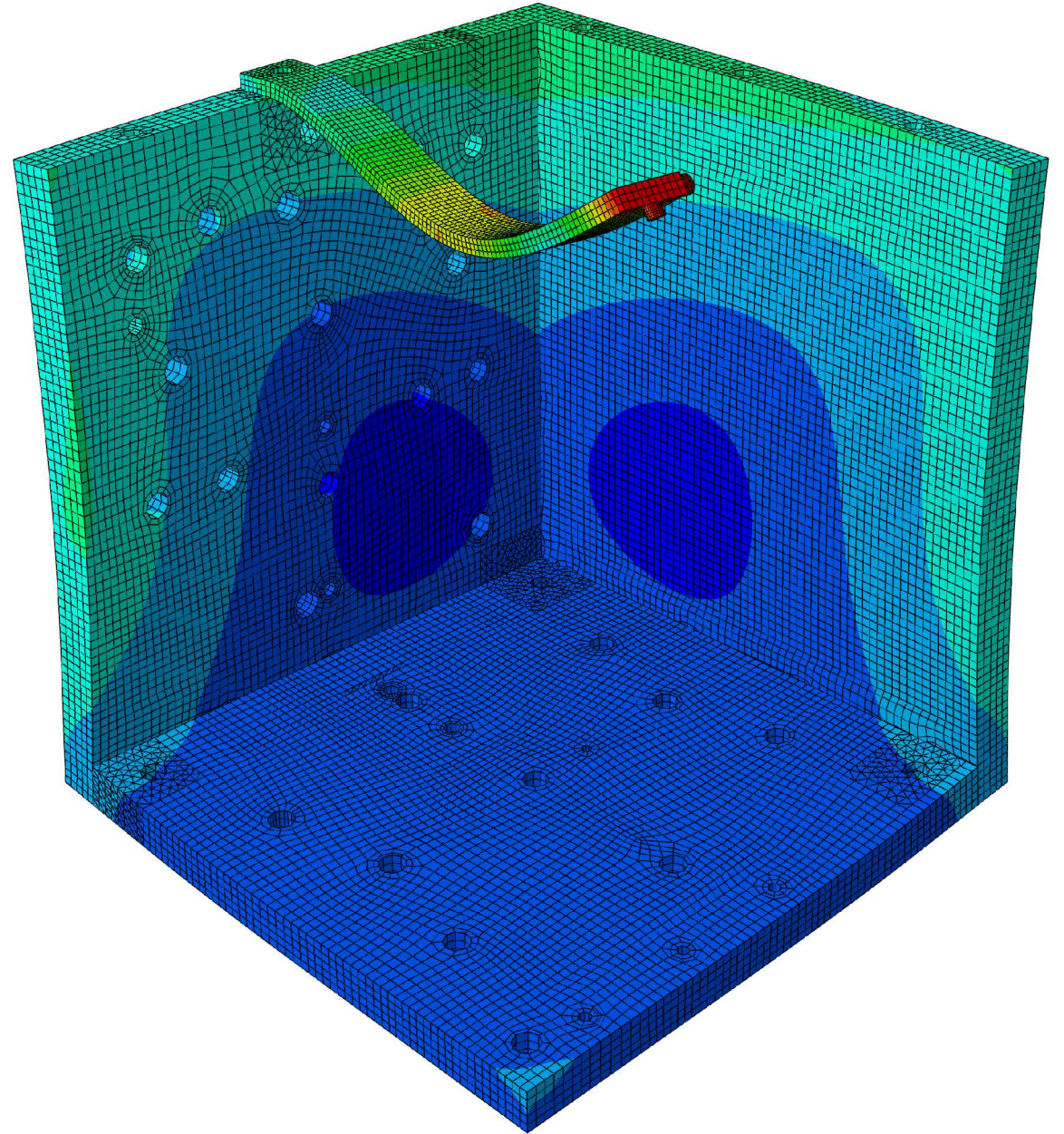


Beam Tip FRF Accurate For All Methods



Abaqus Produces Full FEM Results

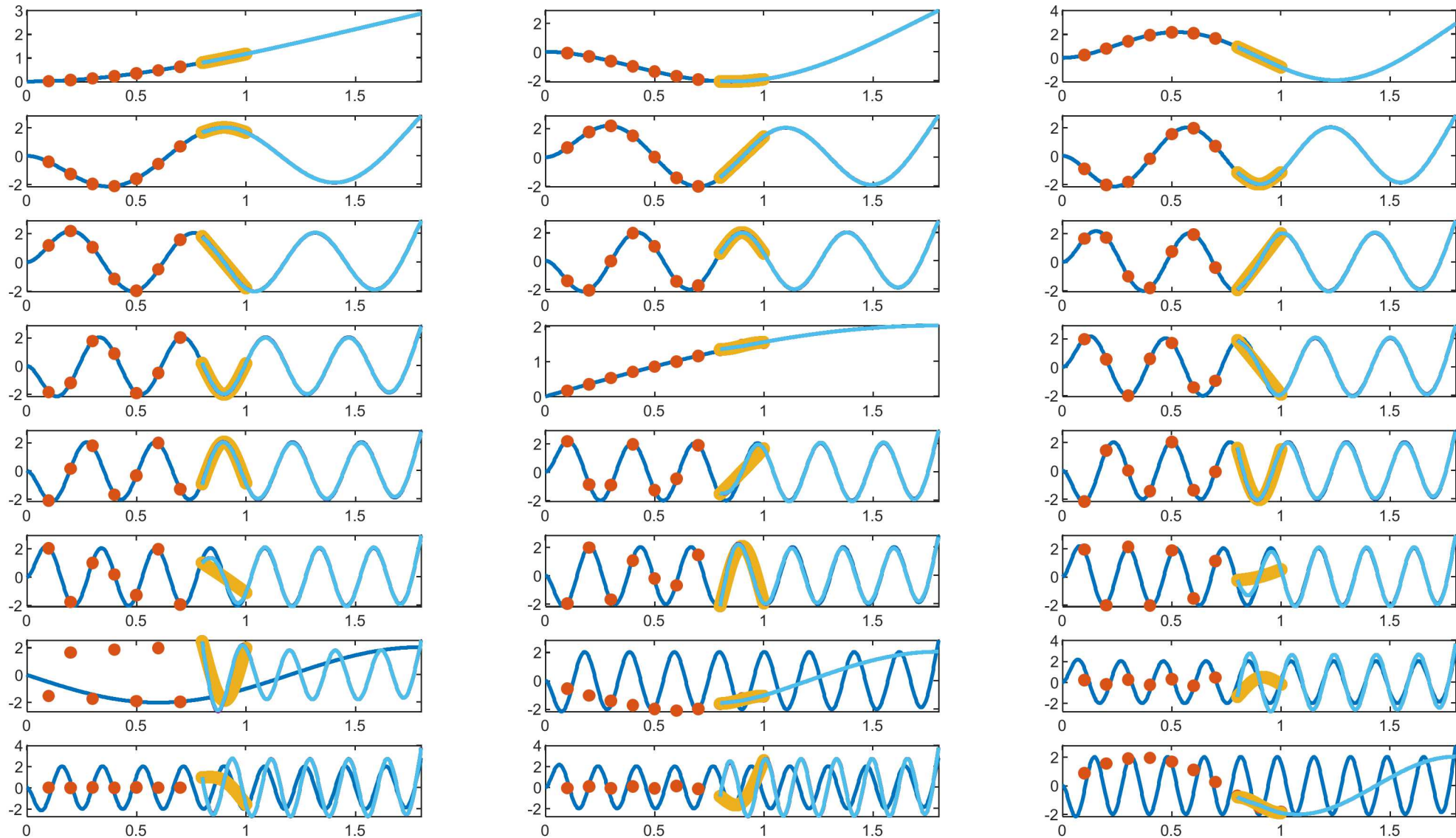
- Mode near 700 Hz is shown
- Participation of close beam mode is visible
- Bottom is correctly constrained



Conclusions

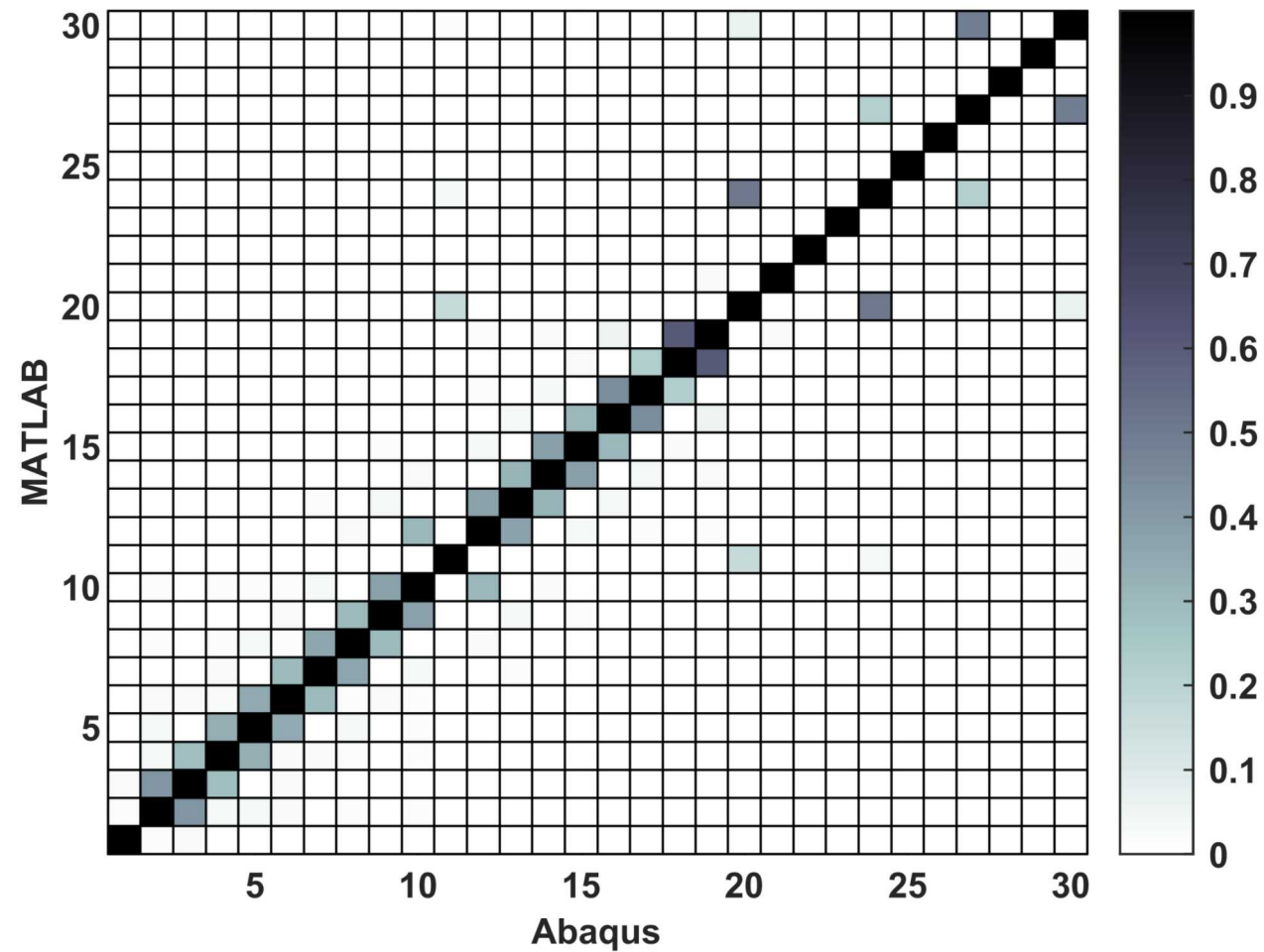
- Substructuring calculations can be performed in Abaqus
 - Uncouple TS from EXP in MATLAB, then import into Abaqus and apply Linear Eq MPCs
 - Decoupling step can be problematic and seems to always degrade results
- Beam numerical case study yielded good results in an ideal scenario
 - Near perfect agreement between MATLAB and Abaqus results
 - Accurate to truth data through full input frequency range
- Half cube experimental test case is a challenging, real-world application
 - Resulting Abaqus assembled model is accurate, but over a limited frequency range when compared to what can be computed in MATLAB
 - Abaqus result seems 'weighed down' by the coupling of every analytical subsystem mode – MATLAB results are better if a truncated set is used.
 - Abaqus solution allows for the assembled model to be easily post-processed in the FEA software i.e. full FEM deformation and stress/strain analysis

Modeshapes

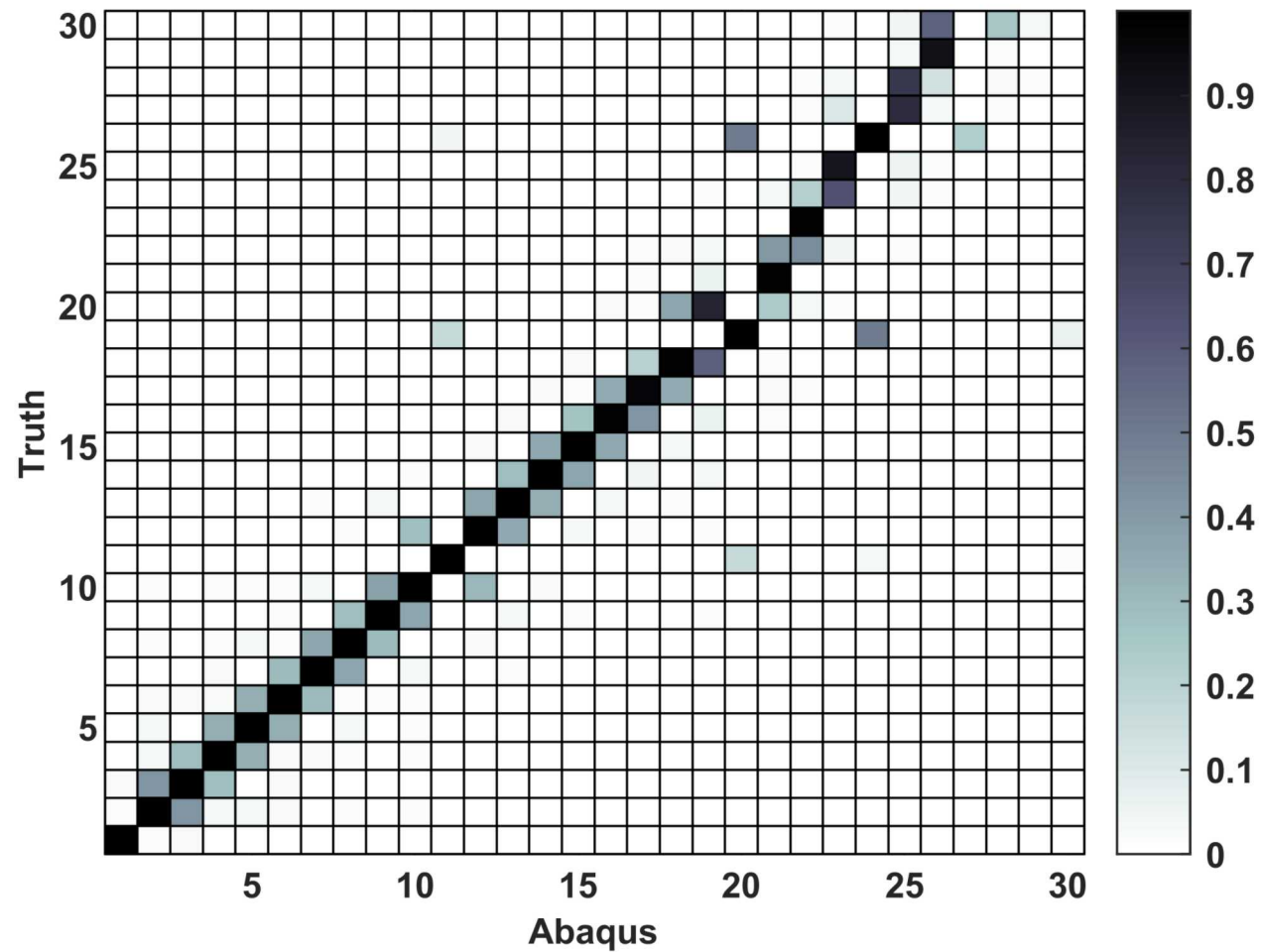


— Truth
 ● Control: EXP
 ● Control: TS
 — Control: FEM
 — MATLAB
 — ABAQUS

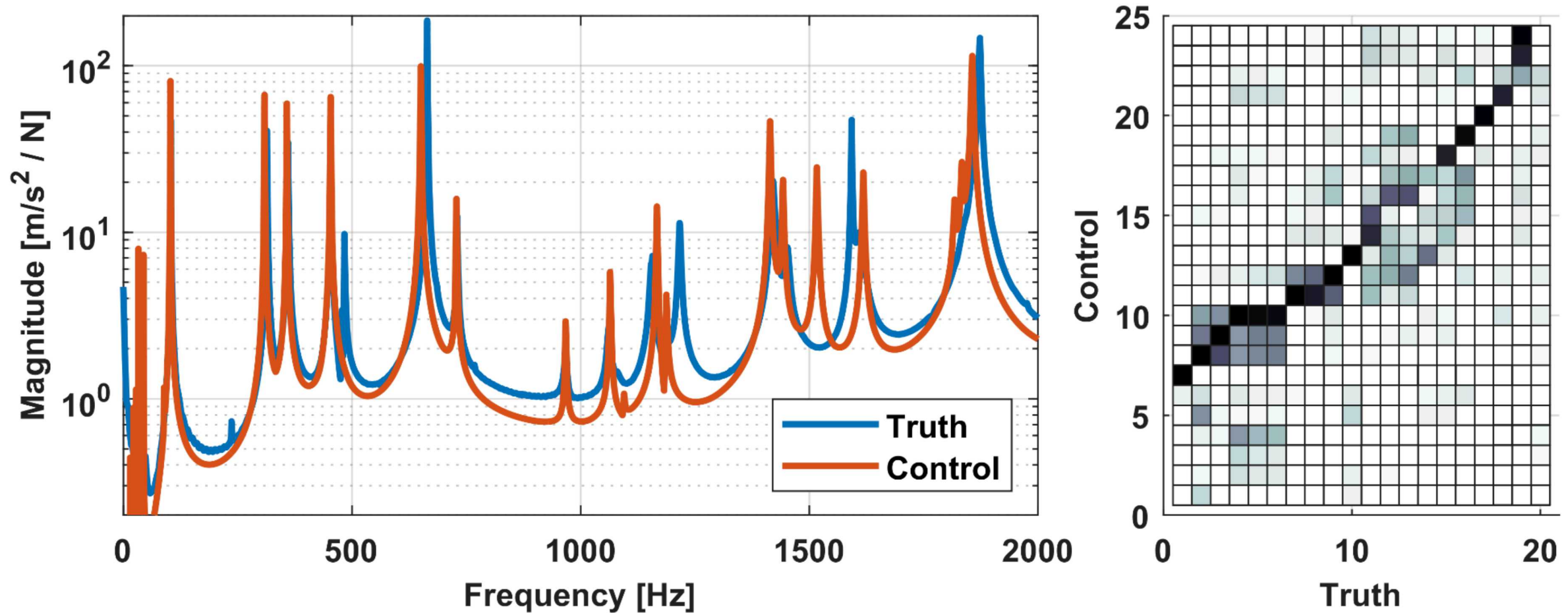
MAC: ABAQUS to MATLAB



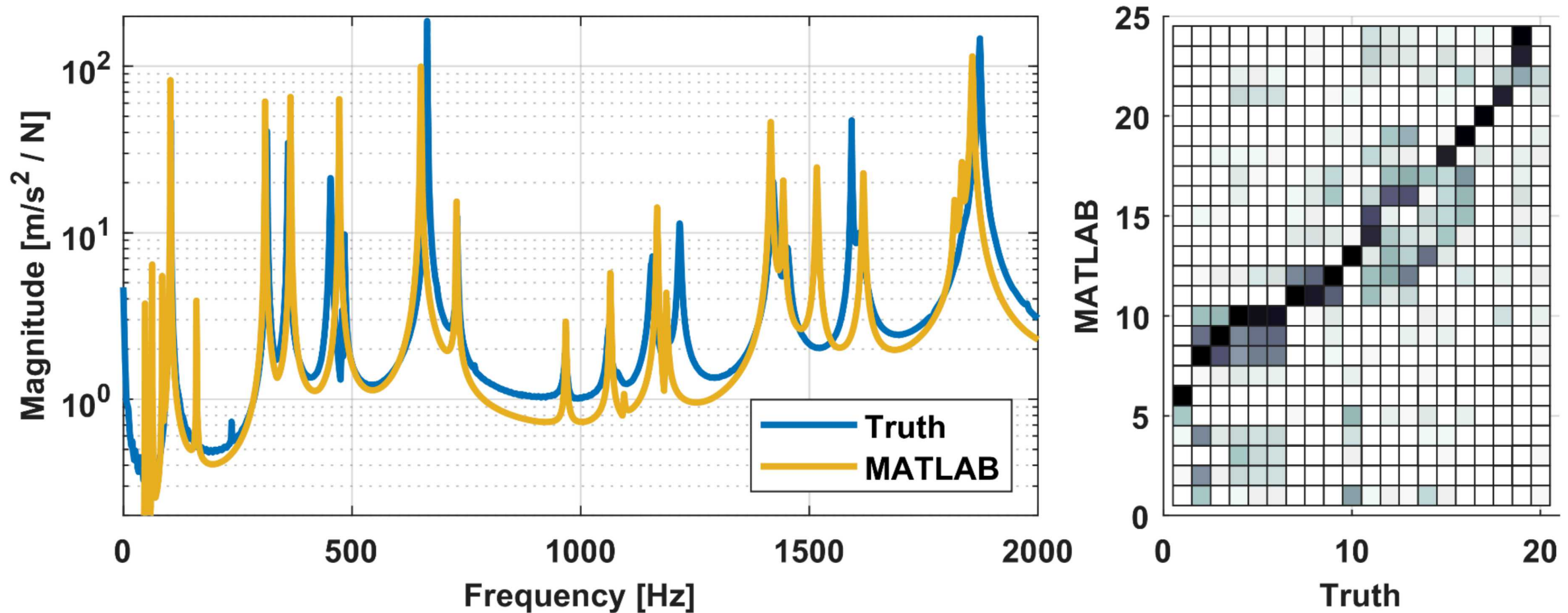
MAC: ABAQUS to Truth



Results: Truth Data vs Standard MATLAB



Results: Truth Data vs Hybrid MATLAB



Results: Truth Data vs Abaqus Output

