



# ***Modeling Techniques for Localization and Failure***

**Jay Foulk, Alex Lindblad, Alejandro Mota,  
Jake Ostien, Mike Vielleux, Tracy Vogler**

**2011 TCG Fall Rodeo  
November 16, 2011  
Monterrey, CA**



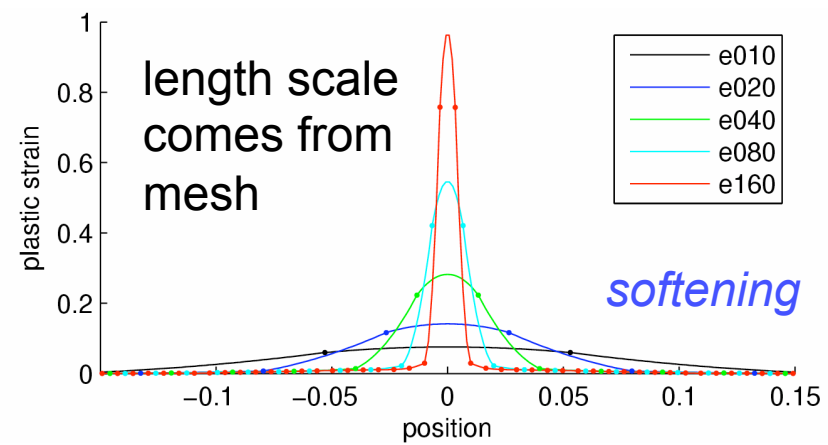
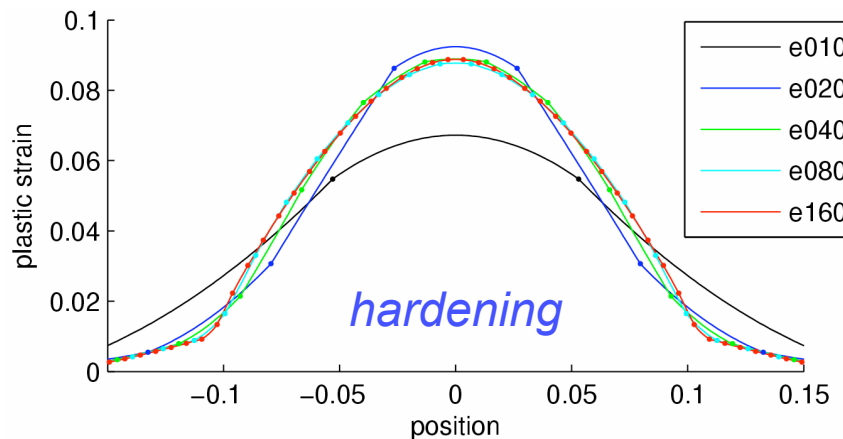
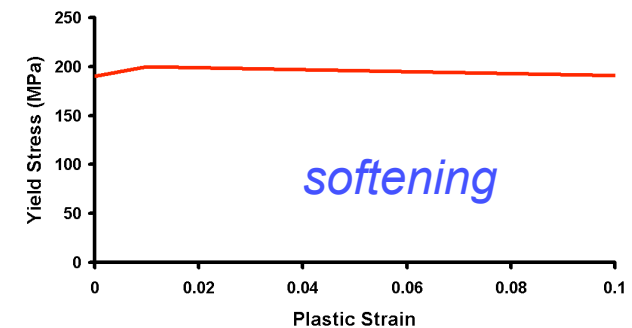
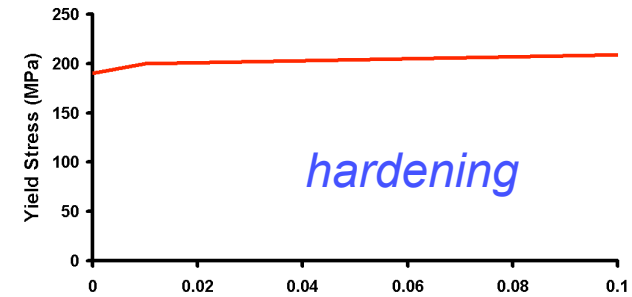
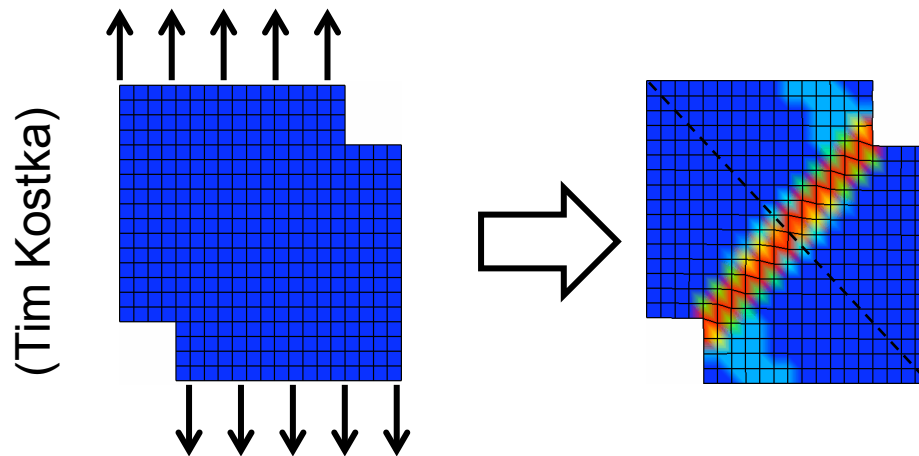
Sandia National Laboratories is a multi-program laboratory managed and operated by Sandia Corporation, a wholly owned subsidiary of Lockheed Martin Corporation, for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-AC04-94AL85000.





# Softening Response (due to Heating, Damage, etc.) Leads to Mesh Dependence

softening (loss of ellipticity) behavior corrupts the PDE and results in mesh dependent results



*need to separate numerical issues from physics issues*



# ***Localization and Failure (TRL 1-3)***

## **What are you trying to do?**

- Provide techniques for the modeling of localization and failure (ductile fracture, shear bands, compaction bands, etc.) that are not mesh dependent:
  - localization elements
  - variational non-local method

## **What makes you think you can do it?**

- Follow multiple modeling techniques at different maturity levels → reduce risk
- Significant experience in development and implementation of localization techniques

## **What difference will it make?**

- DoD and DOE have continued need for ability to perform predictive simulations of munitions behavior
- Failure and localization are inevitably the most critical aspects of munitions simulations

## **What / When / To Whom Will You Deliver?**

- Verify localization elements
- Implement, verify, and validate adaptive insertion
- Develop variational mixed formulation
- Implement into Sierra when appropriate

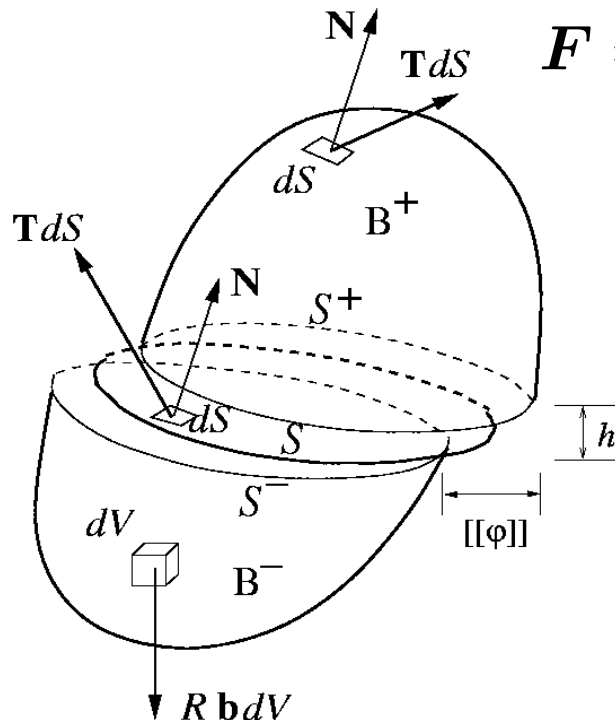
***Q4-FY13 - Demonstrate the capability to obtain a mesh independent solution involving failure for a problem of realistic complexity.***



# Localization Elements (similar to cohesive elements)



*IDEA: Use ANY bulk constitutive model ( $\sigma$ - $\epsilon$ ) to drive surface separation*



$$\mathbf{F} = \mathbf{F}^{\parallel} \mathbf{F}^{\perp}$$

$$\mathbf{F}^{\parallel} = \mathbf{g}_i \otimes \mathbf{G}^i$$

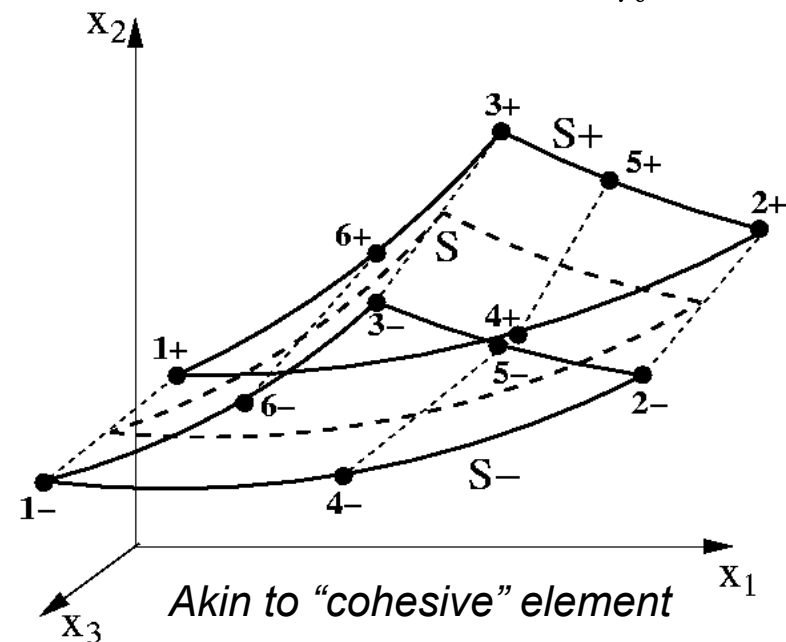
$$\mathbf{F}^{\perp} = \mathbf{I} + \frac{[[\Phi]]}{h} \otimes \mathbf{N}$$

$$\mathbf{F} = \mathbf{F}^{\parallel} + \frac{[[\varphi]]}{h} \otimes \mathbf{N}$$

$h$  = band thickness

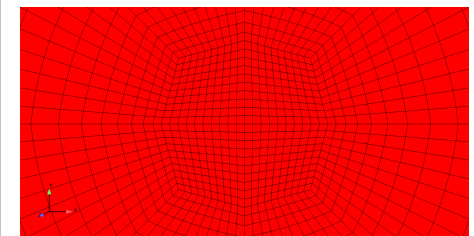
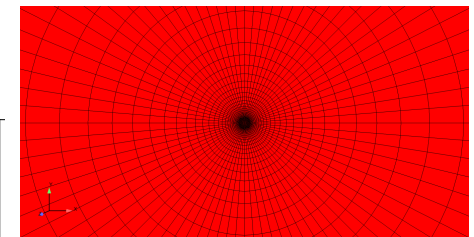
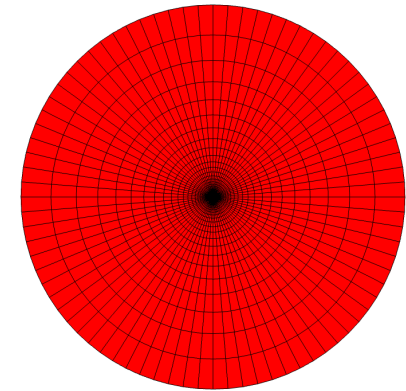
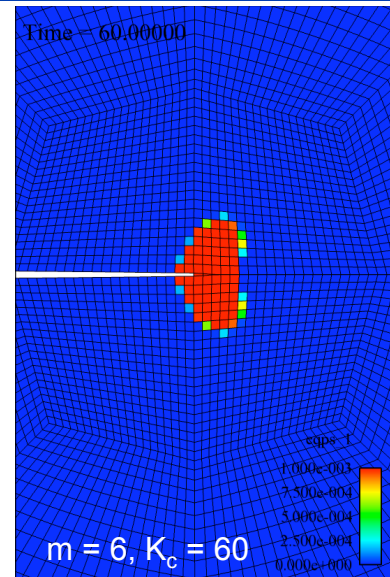
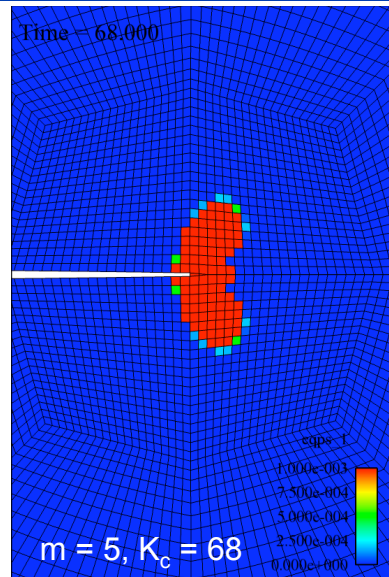
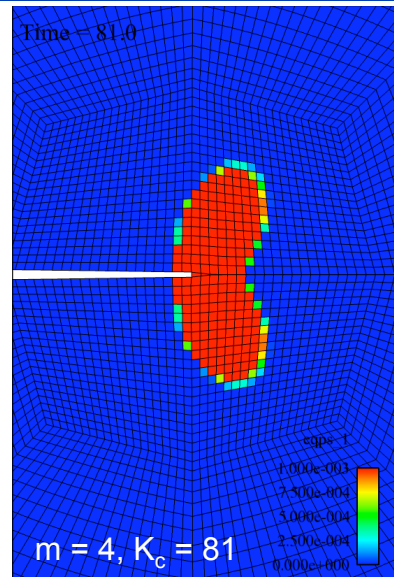
- Finite-deformation kinematics.
- Simulation of strain localization.
- No additional constitutive assumptions

Yang, Mota and Ortiz, IJNME, 2005





# Resolution and lumping dissipation

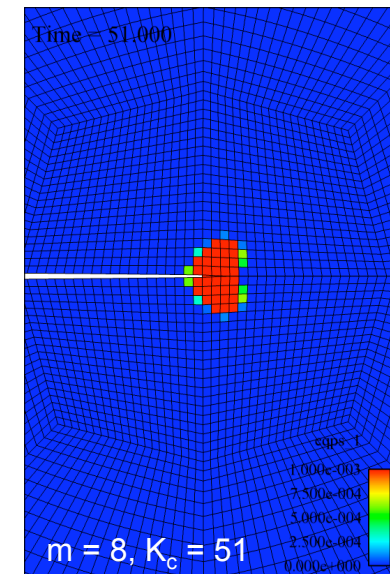
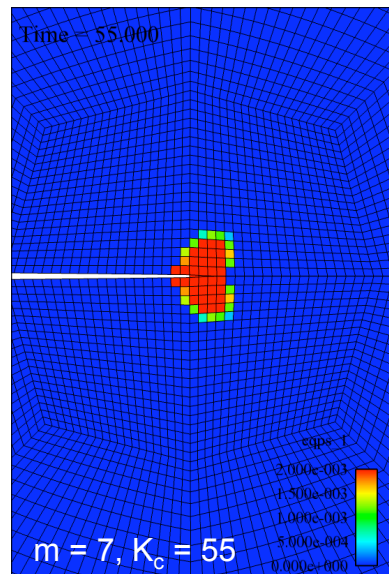


*K-field boundary condition*

The size of the plastic zone at propagation,  $Da = 60 \mu m$ , The mesh size  $s$  is  $30 \mu m$ .

$K_{Ic} = 70 \text{ MPa}\cdot\text{m}^{0.5}$   
 $h = 30 \mu m$

*Quasi-statics with SierraSM*



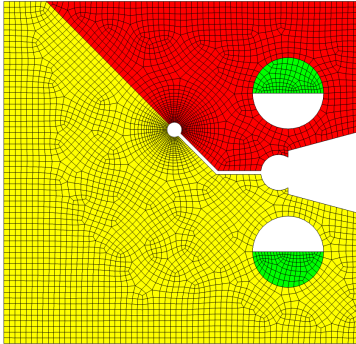




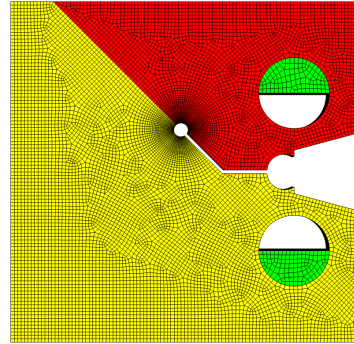
# Damage is Convergent



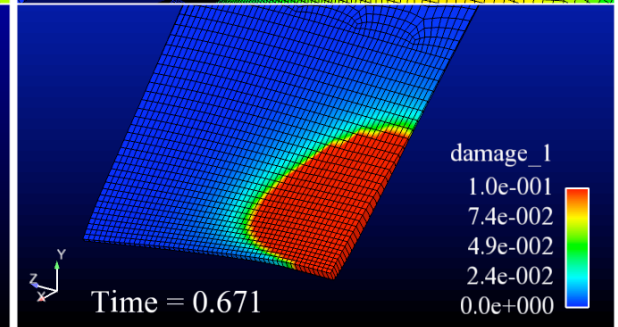
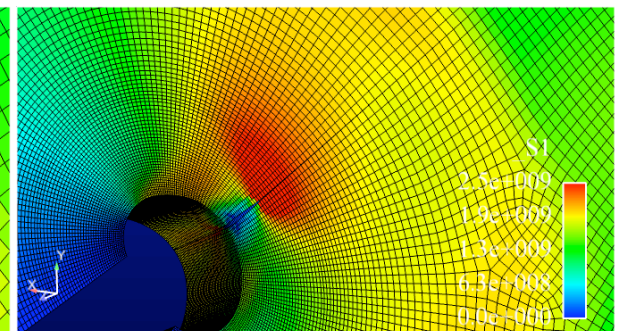
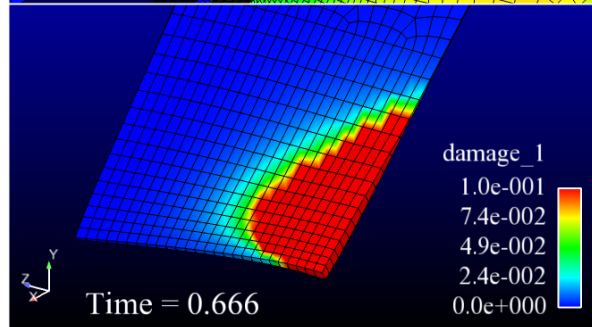
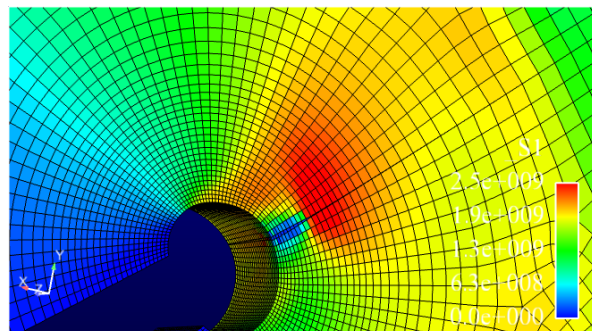
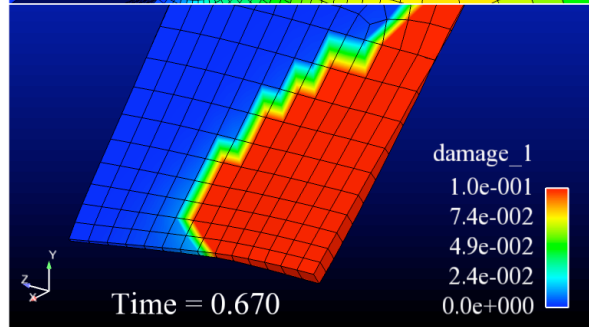
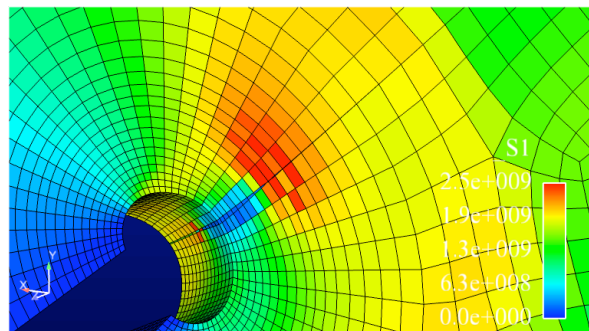
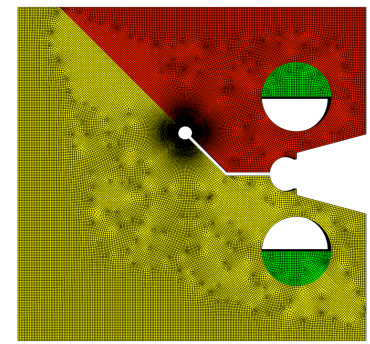
Mesh: 02  
Label: Medium  
Nodes: 30k  
Elem: 24k  
s ~ 120  $\mu$ m



Mesh: 03  
Label: Fine  
Nodes: 142k  
Elem: 126k  
s ~ 60  $\mu$ m



Mesh: 04  
Label: Finest  
Nodes: 1M  
Elem: 1M  
s ~ 30  $\mu$ m

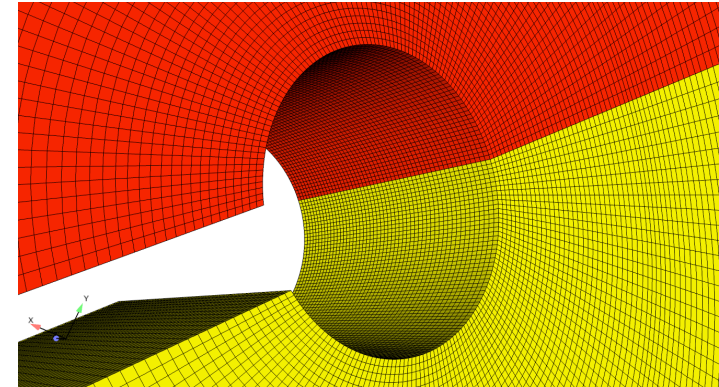
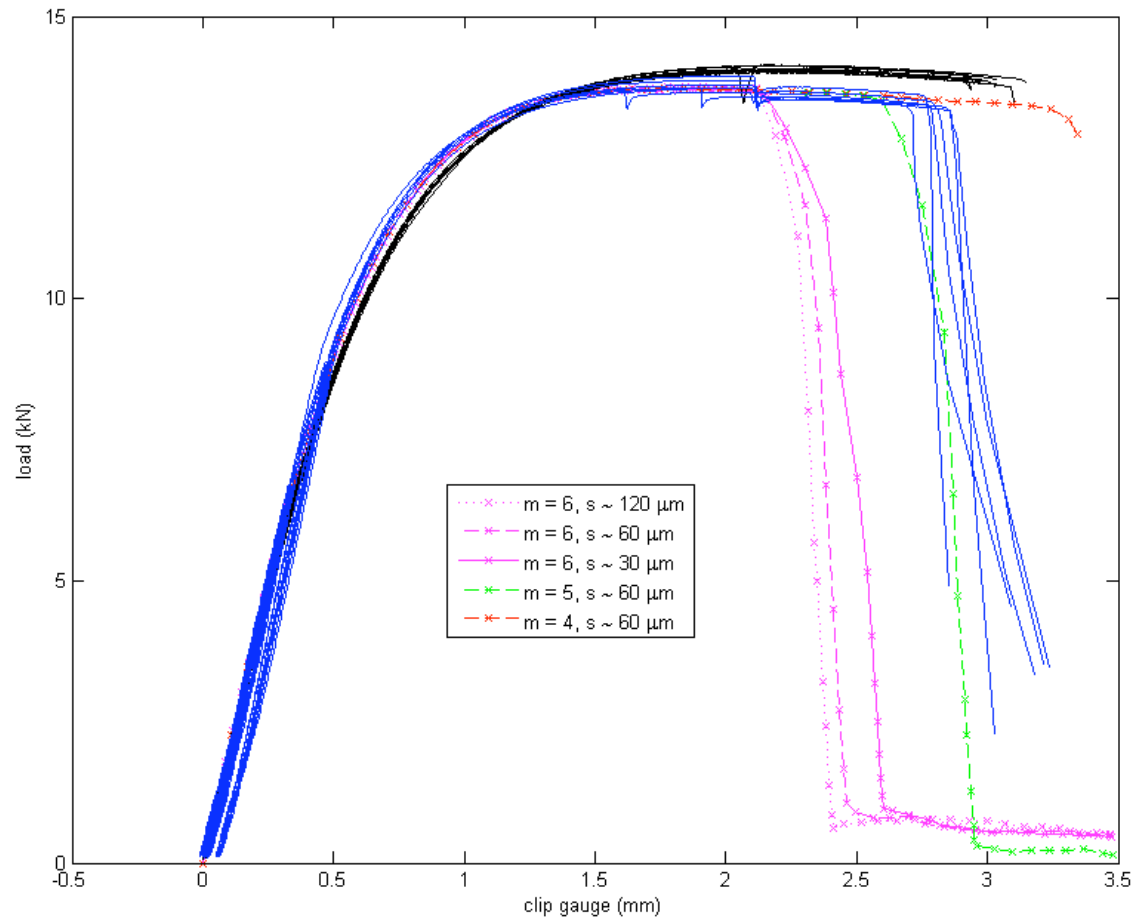


Although at slightly different times, the evolution of damage is comparable for 03 & 04.

*Quasi-statics with SierraSM* NOTE: Smooth notch – the specimen was not pre-cracked.



# Load-Displacement Is Convergent



*Boyce's lab:*

- Load line rate is 0.0127 mm/s

*Cordova's lab*

- Load line rate before 2.03 mm is 0.0027 mm/s
- Load line rate after 2.03 mm is 0.00025 mm/s

blind predictions differed somewhat from experimental data but showed correct trend

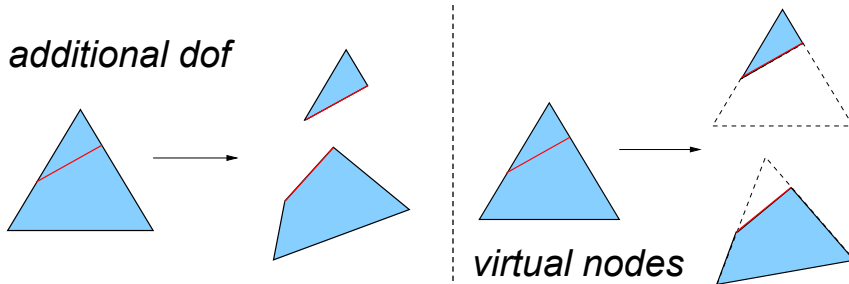


# Moving the Discontinuity

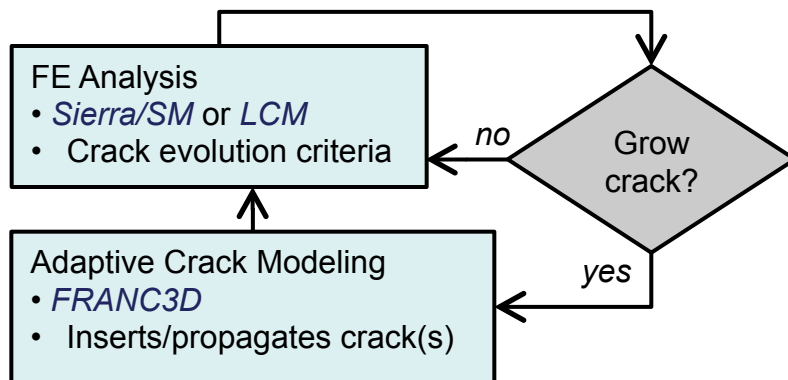
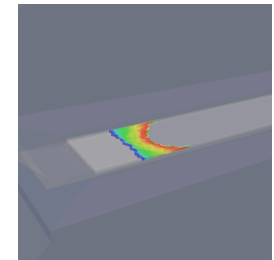


## X-FEM through virtual node method in SierraSM

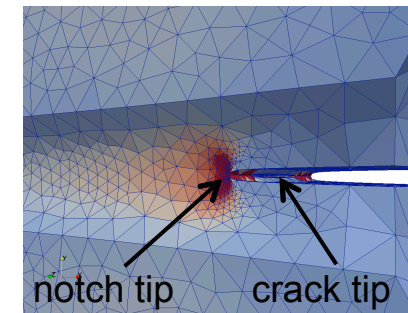
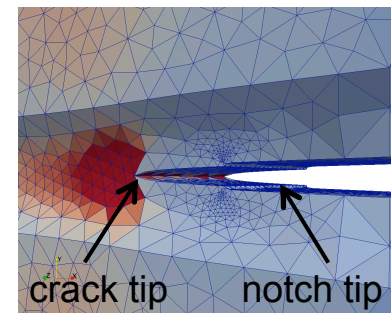
additional dof



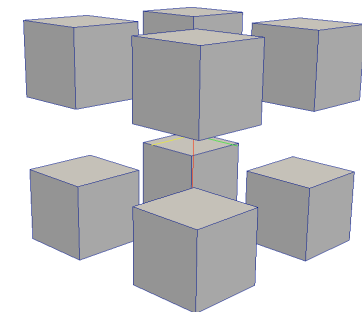
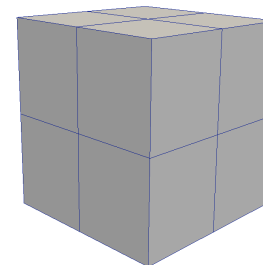
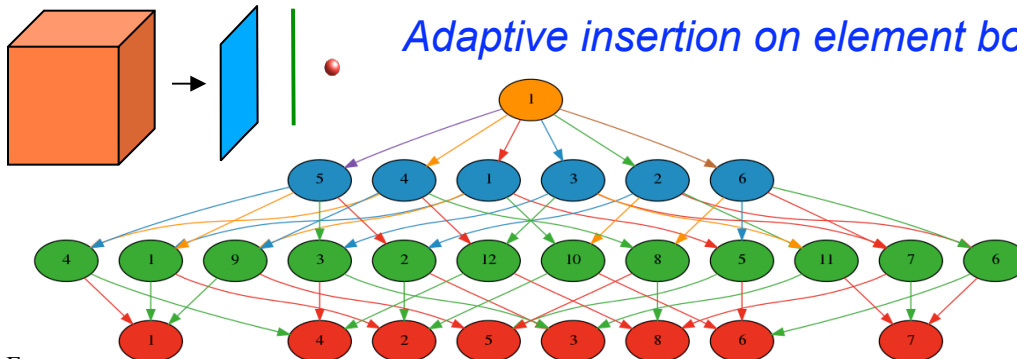
- Virtual node method enriches displacement field elements by duplicating cut elements
- Results in same number of degrees of freedom as Heaviside-enriched XFEM
- Two approaches shown to be equivalent



## Adaptive remeshing with refinement/coarsening



## Adaptive insertion on element boundaries w/STK







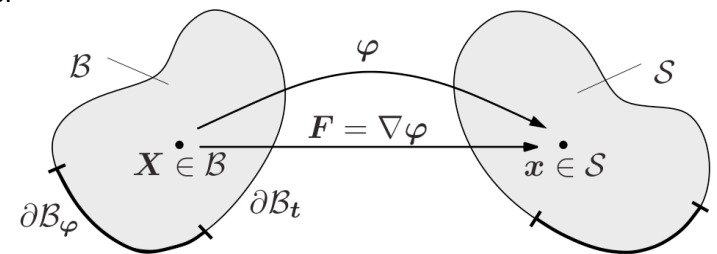
# Variational Nonlocal Method



*IDEA: Derive nonlocality optimized for parallel computation for ANY bulk ( $\sigma$ - $\epsilon$ ) model*

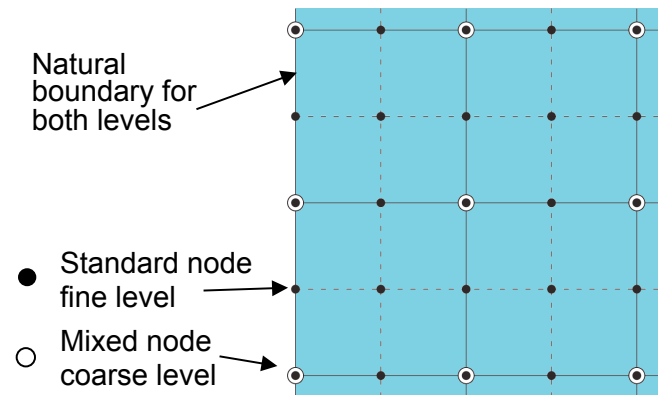
$$\Phi[\varphi, \bar{\mathbf{Z}}, \bar{\mathbf{Y}}] := \int_B W(\mathbf{F}, \bar{\mathbf{Z}}, \mathbf{Q}, T) dV + \int_B \bar{\mathbf{Y}} \cdot (\bar{\mathbf{Z}} - \mathbf{Z}) dV - \int_B \rho_0 \mathbf{B} \cdot \varphi dV - \int_{\partial_T B} \mathbf{T} \cdot \varphi dS$$

Deformation Mapping
Helmholtz Free Energy
Nonlocal Internal Variable
Constraint Enforced by Lagrange Multiplier



Deformation Mapping

- Motivated through studies of non-locality
- Fully variational approach that by-passes ad hoc assumptions.
- No modifications to constitutive models.
- Nonlocal domain is defined.
- Natural parallelization by domain decomposition of coarse discretization.
- Does not require cut-off approaches at boundary.





# Mesh Dependence in Baseline Case

Simple finite-deformation elastic model with damage:

$$W(\mathbf{C}, \zeta) = (1 - \zeta)W_0(\mathbf{C})$$

$$\boldsymbol{\epsilon} = \frac{1}{2} \log(\mathbf{C})$$

$$\bar{\boldsymbol{\epsilon}} = \text{dev}(\boldsymbol{\epsilon}), \quad \theta = \text{tr}(\boldsymbol{\epsilon}),$$

$$W_0(\mathbf{C}) = W_0^{\text{vol}}(\theta) + W_0^{\text{dev}}(\bar{\boldsymbol{\epsilon}}),$$

$$W_0^{\text{vol}}(\theta) = \frac{\kappa}{4} [\exp(2\theta) - 1 - 2\theta],$$

$$W_0^{\text{dev}}(\bar{\boldsymbol{\epsilon}}) = \frac{\mu}{2} [\text{tr}(\exp \bar{\boldsymbol{\epsilon}}) - 3].$$

$$\zeta(\alpha) := \zeta_{\infty} [1 - \exp(-\alpha/\iota)]$$

$$\alpha(t) := \max_{s \in [0, t]} W_0(s)$$

$\zeta_{\infty}$ : maximum possible damage  
 $\iota$ : damage saturation parameter

$$E = 200 \text{ GPa}$$

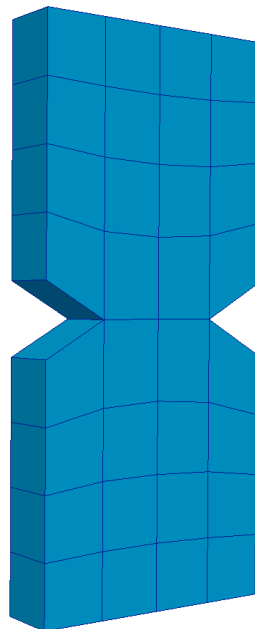
$$\nu = 0.25$$

$$\kappa = 133 \text{ GPa}$$

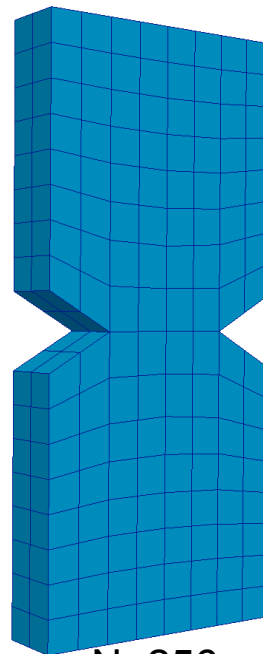
$$\mu = 80 \text{ GPa}$$

$$\zeta_{\infty} = 1.0$$

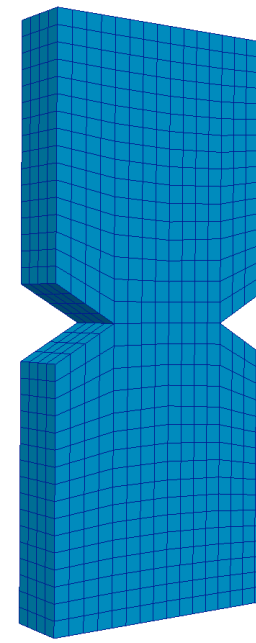
$$\iota = 100 \text{ GJm}^{-3}$$



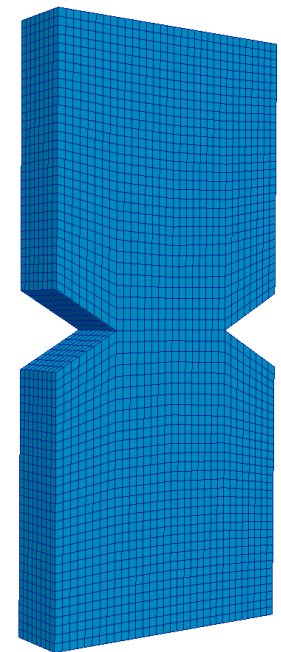
N=32  
h~1mm



N=256  
h~0.5mm



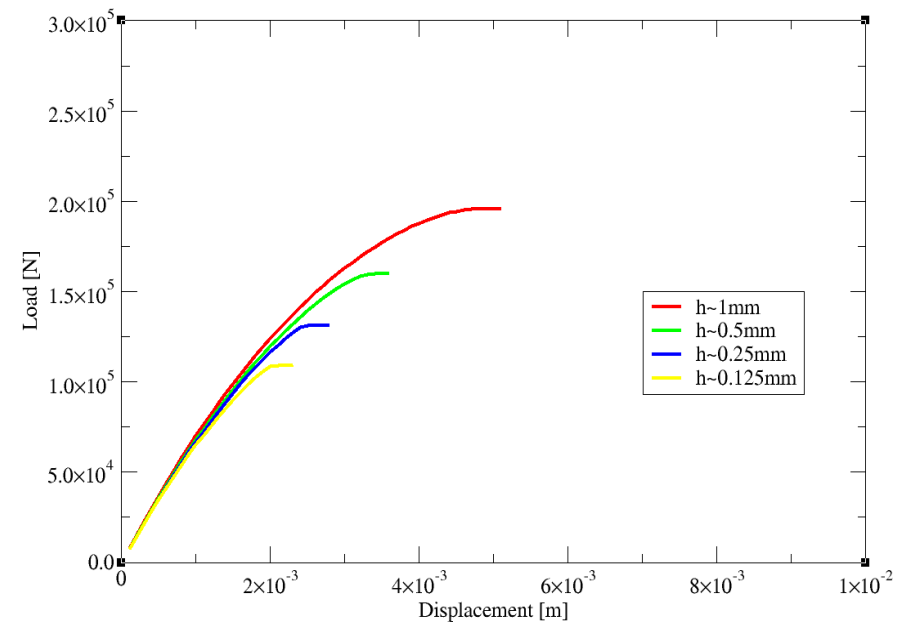
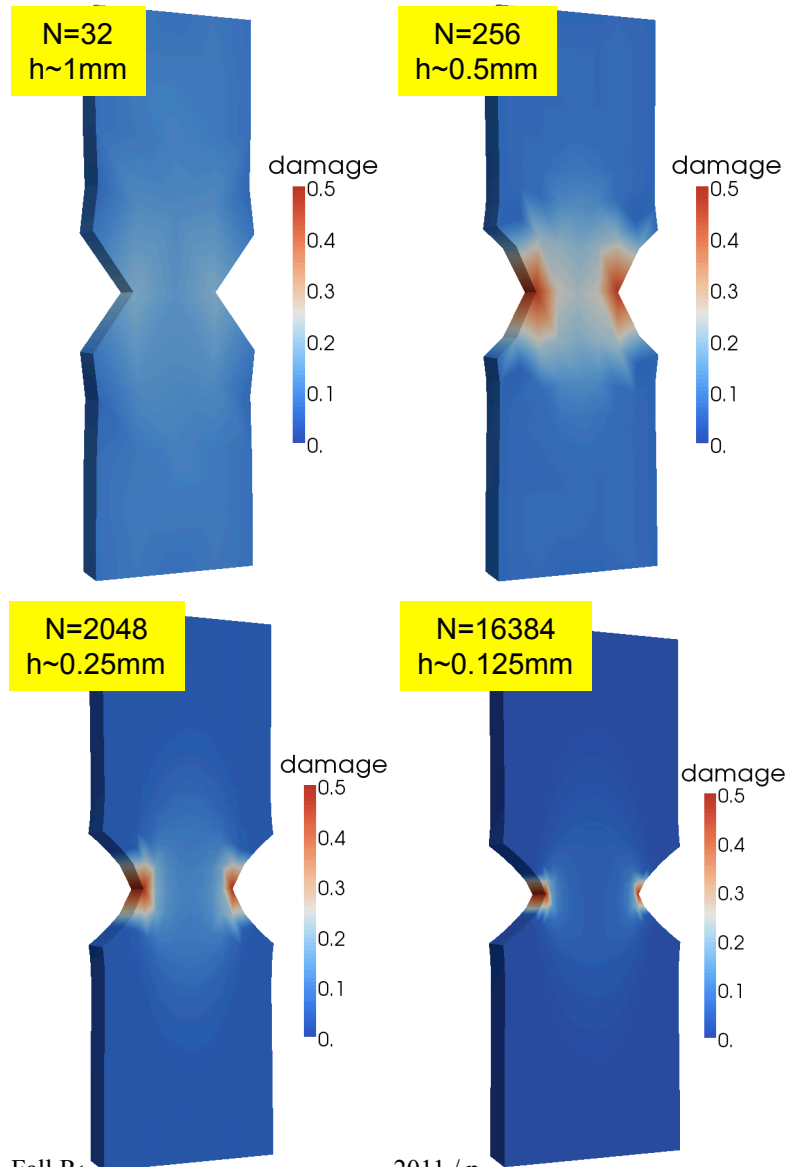
N=2048  
h~0.25mm



N=16384  
h~0.125mm



# Mesh Dependence in Baseline Case (2)



damage zones and load-displacement curves display mesh dependent behavior: as mesh is refined damaged region shrinks and failure load drops



# Mesh Partitioning Tools Used to Provide Coarse Scale



$$\bar{\mathbf{Y}} = \frac{1}{\text{vol}(D)} \int_D \mathbf{Y} dV,$$

$$\bar{\mathbf{Z}} = \frac{1}{\text{vol}(D)} \int_D \mathbf{Z} dV,$$

$$\text{vol}(\bullet) := \int_{(\bullet)} dV,$$

Constant interpolation leads to decoupling and simple averaging:

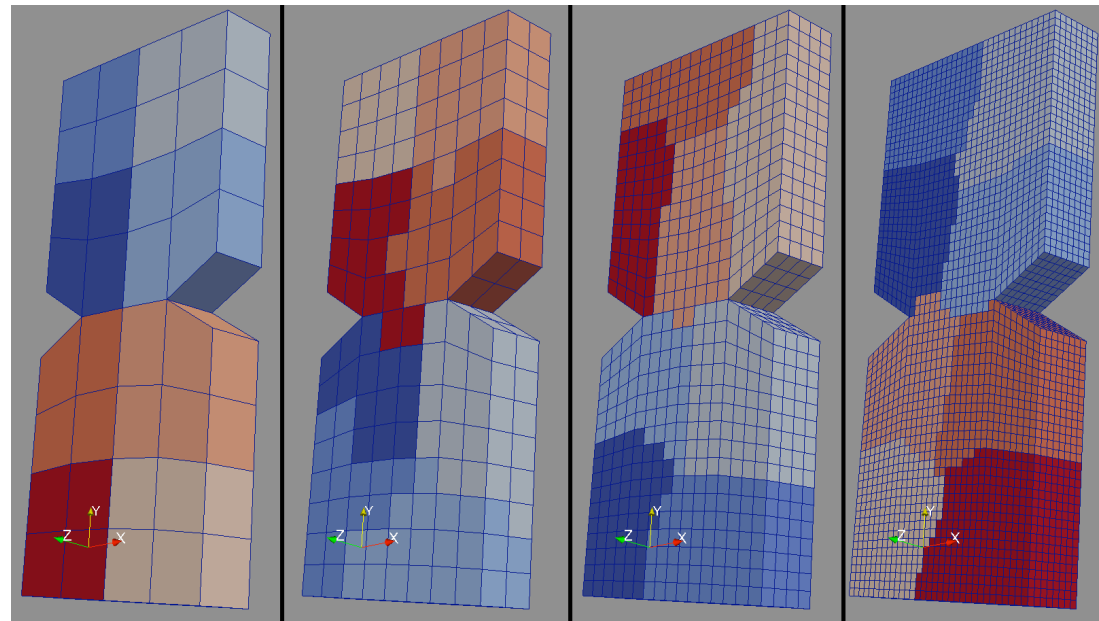
$$\text{vol}(D) = \sum_{i=0}^n \text{vol}(E_i),$$

$$\int_D \mathbf{Y} dV = \sum_{i=0}^n \int_{E_i} \mathbf{Y} dV,$$

$$\int_D \mathbf{Z} dV = \sum_{i=0}^n \int_{E_i} \mathbf{Z} dV.$$

use mesh partitioner (Zoltan in Sierra) to create domains  $D$

$$\text{vol}(D) = (\text{length scale})^3 = (1.6\text{mm})^3$$



N=32  
h~1mm

N=256  
h~0.5mm

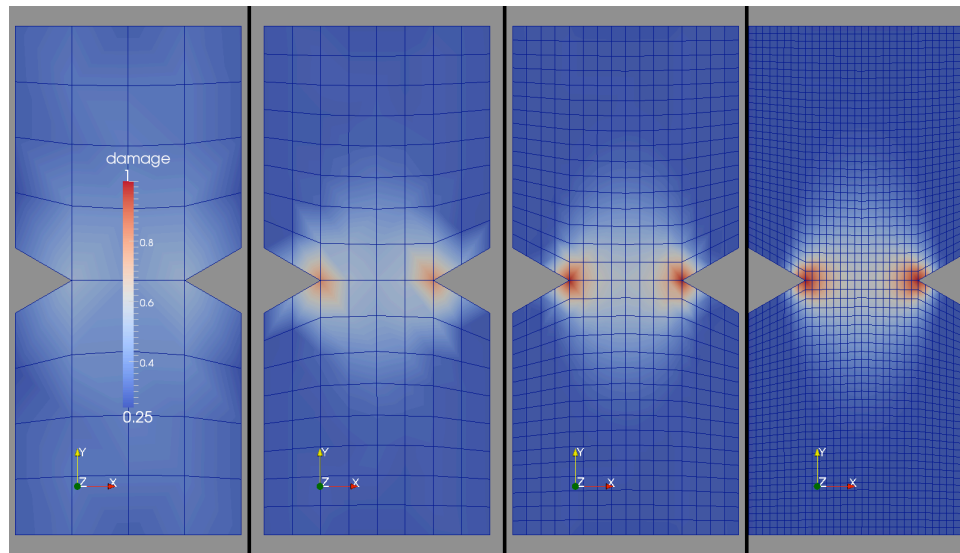
N=2048  
h~0.25mm

N=16384  
h~0.125mm





# Variational Nonlocal Technique is Convergent



N=32  
h~1mm

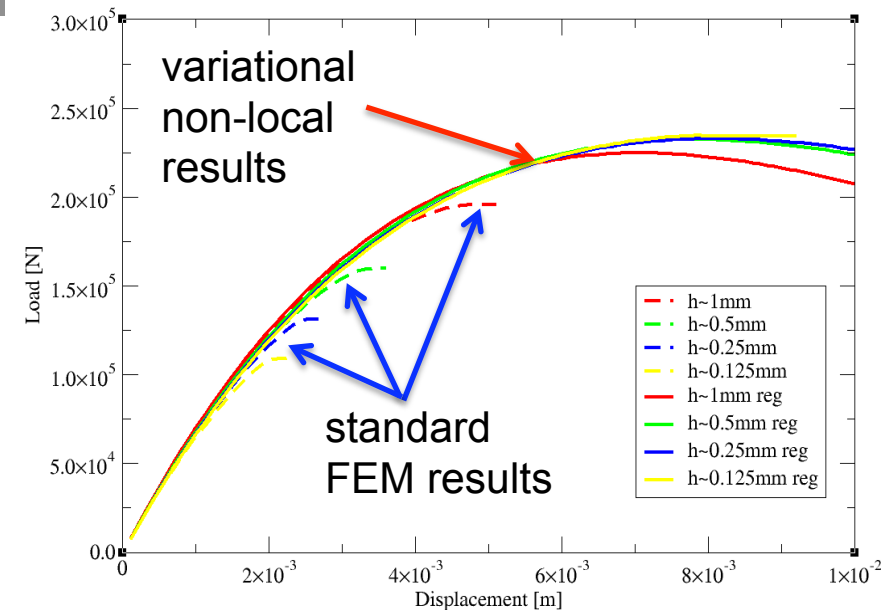
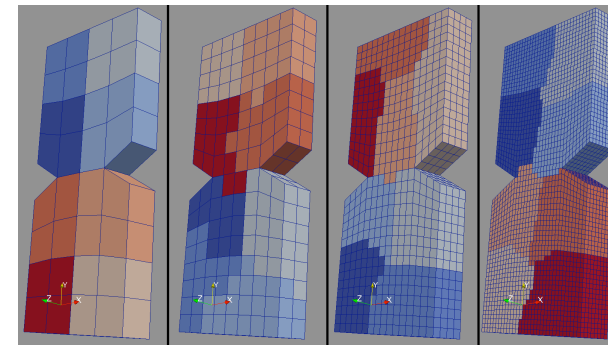
N=256  
h~0.5mm

N=2048  
h~0.25mm

N=16384  
h~0.125mm

- Regularization effective
- Derived naturally from variational principle
- No special boundary considerations
- Simple form with unit interpolation functions

*Initial studies in 3-D confirm 1-D findings, the fields are damage are converging*





# Conclusions



- Developing multiple methods to reduce mesh dependence in problems involving failure and localization, but there is no silver bullet!
- Methods are convergent and have a space of applicability
- Localization elements have broad applicability (leverage bulk response) and provide the regularization needed
  - issues when element size is of order of  $h$
  - robust insertion techniques
- Variational non-local technique establishes length that is natural to the FEM mesh
  - derived naturally from variational principle
  - no special boundary considerations
  - simple form with unit interpolation functions.

*We measure success by analyst adoption and not by model development or implementation.*