

# Spark Assisted Compression Ignition Engine with Stratified Charge Combustion and Ozone Addition

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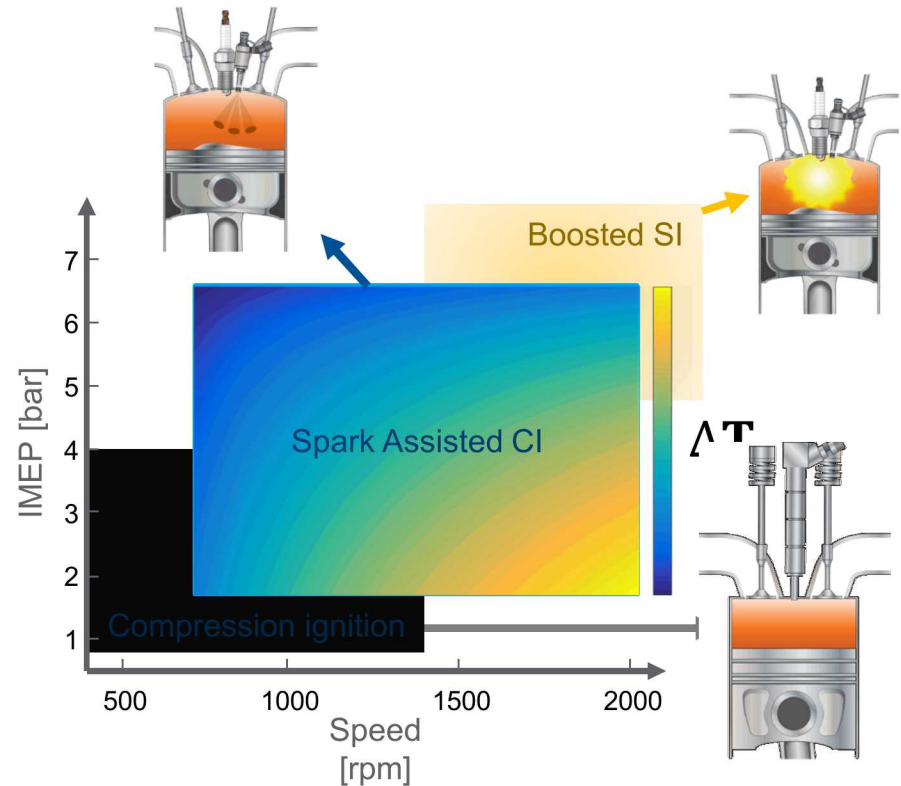
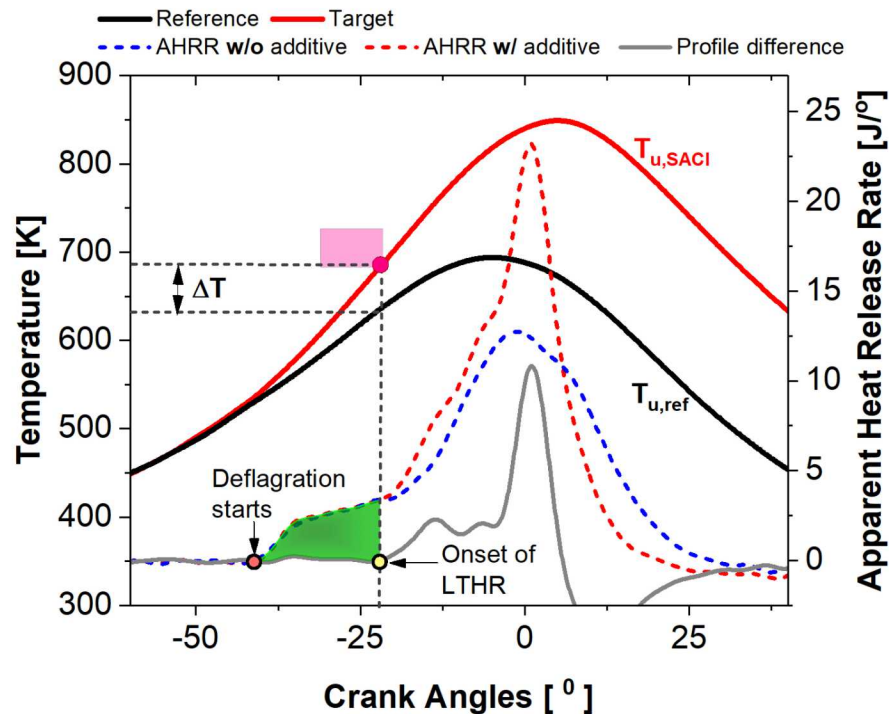


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# Multi-mode combustion strategies can be beneficial for high-efficiency and reduced emission



Advanced ignition can enable  $O_3$  enhanced assisted compression ignition (SACI) (likewise benefits dilute/lean SI & enables spark retard for boosted SI)



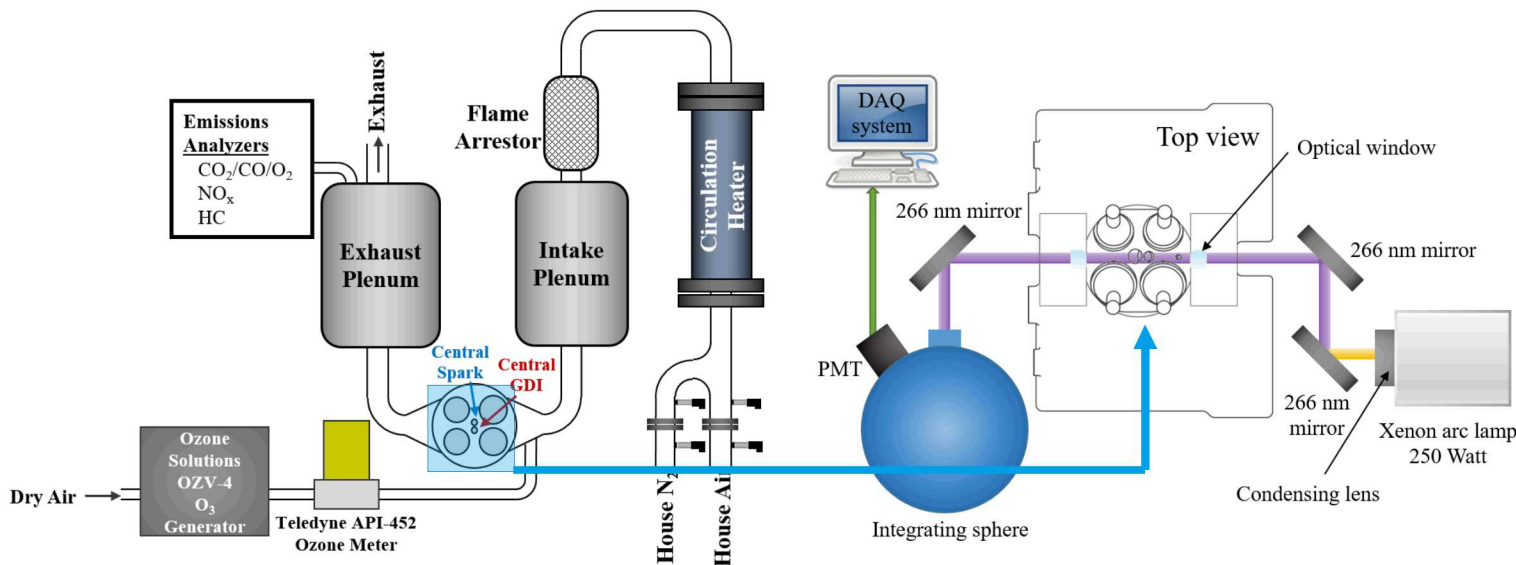
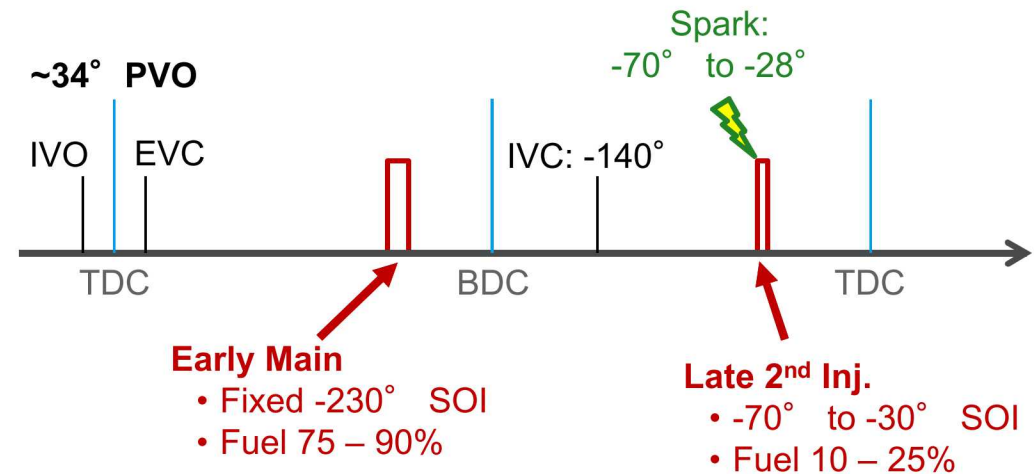
## O<sub>3</sub> impact on LTHR Chemical Pathways

- $O_3$  pyrolysis depends on temperature & composition
- Fuel hydrogen abstraction by atomic oxygen ( $O^\bullet$ )
- Chain branching reactions form more alkyl radical ( $R^\bullet$ ) and hydroxyl ( $OH^\bullet$ )



| Combustion Mode                   | Spark Assisted CI           |
|-----------------------------------|-----------------------------|
| Head Design                       | 4-valve pent-roof           |
| Injector                          | Central VCO, 8 holes        |
| Displaced volume [L]              | 0.55                        |
| Stroke/Bore                       | 1.11                        |
| Compression Ratio                 | 13:1                        |
| Intake Pressure [bar]             | 1                           |
| Intake Temperature [°C]           | 42                          |
| Intake O <sub>3</sub> Conc. [ppm] | 0 – 34                      |
| Engine Speed [rpm]                | 800, 1000, 1200, 1400, 1600 |
| Equivalence ratio                 | 0.27 – 0.56                 |
| RGF [%]                           | 10 – 18                     |
| 2nd injection [%]                 | 0 – 25                      |

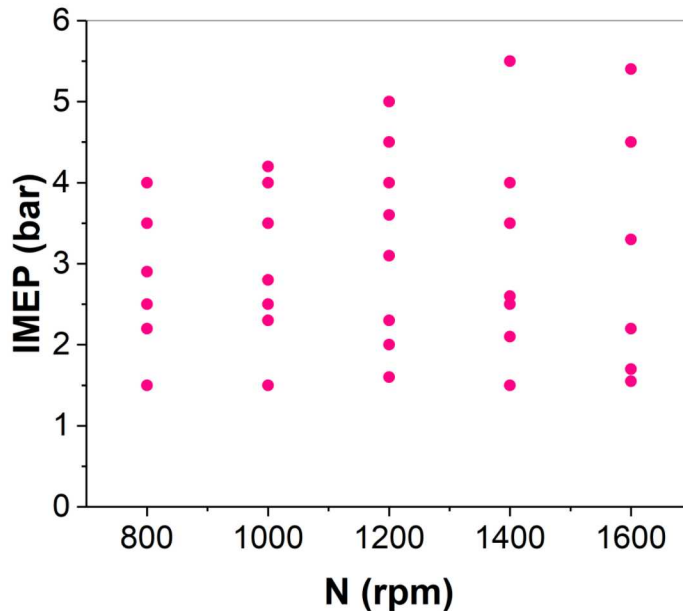
## Partially stratified strategy



$$X_{O_3} = \frac{p_{O_3}}{p} = \ln \left( \frac{I_{ref}}{I} \right) \frac{k_B T}{p B \sigma_{O_3}}$$

$k_B$ : Boltzmann constant  
 $\sigma_{O_3}$ : O<sub>3</sub> absorption cross-section  
 $B$ : Bore diameter  
 $T$ : Bulk temperature  
 $p$ : Total pressure  
 $I, I_{ref}$ : Intensity





## Sweep parameters

- $O_3$  concentration (0 – 34 ppm)
- Spark timing ( $70^\circ - 28^\circ$  bTDC)
- 2<sup>nd</sup> injection SOI ( $64^\circ - 36^\circ$  bTDC)
- Fuel split, 2<sup>nd</sup> injection fueling fraction 0 – 25%

For N rpm, P IMEP

Constraint 1

- Indicated thermal efficiency  $\uparrow$

Constraint 2

- $NO_x \downarrow (< 5 \text{ g/kg-fuel})$

Constraint 3

- Ringing intensity  $\downarrow (< 1 \text{ MW/m}^2)$

For a **fixed  $O_3$  concentration**  
iterate for

Fuel split,  
 $m_{f, 2nd \text{ inj.}}$

- $< 25\%$   
in 2<sup>nd</sup>  
inj.

Spark  
timing

- $70^\circ - 28^\circ$   
bTDC

2<sup>nd</sup> inj.  
SOI

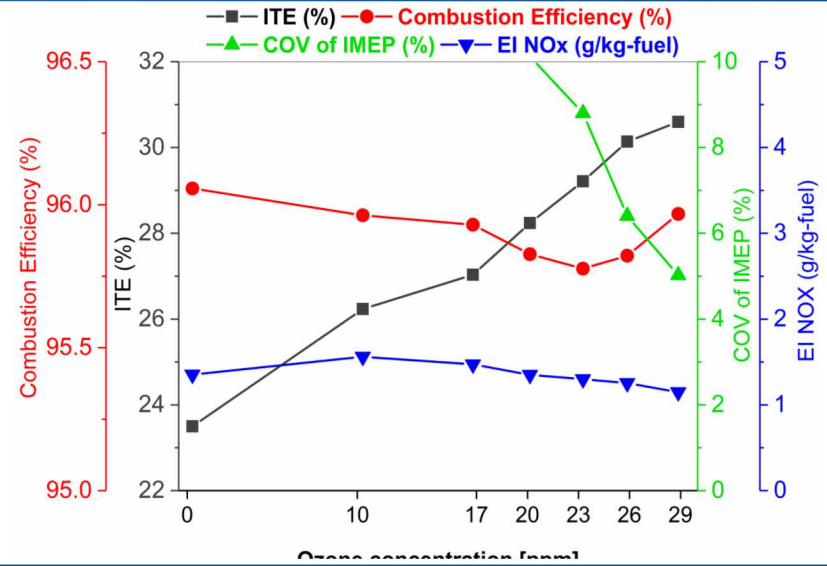
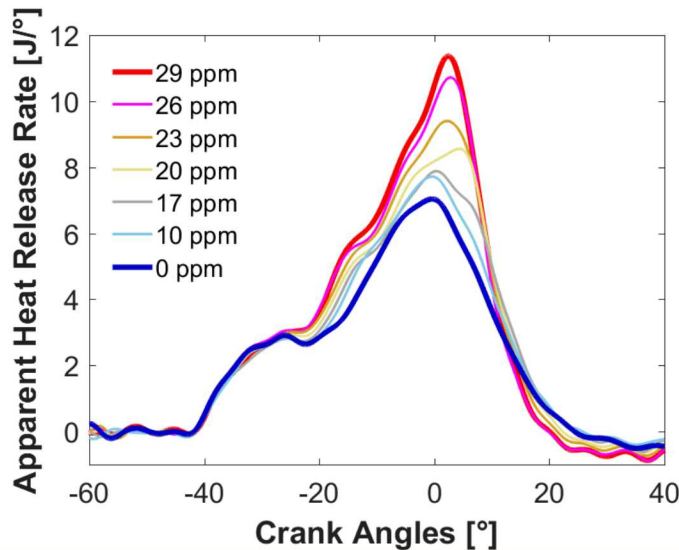
- $64^\circ - 36^\circ$   
bTDC

Optimize for maximum brake  
torque (MBT)

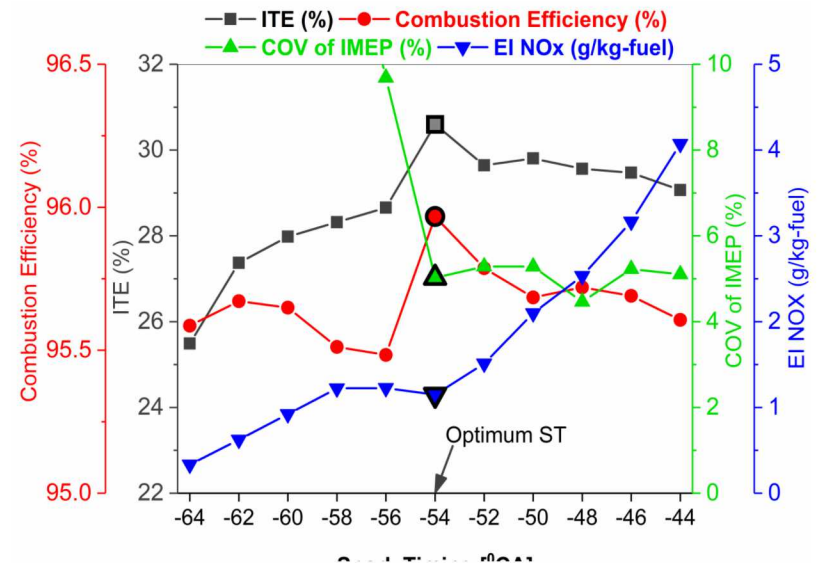
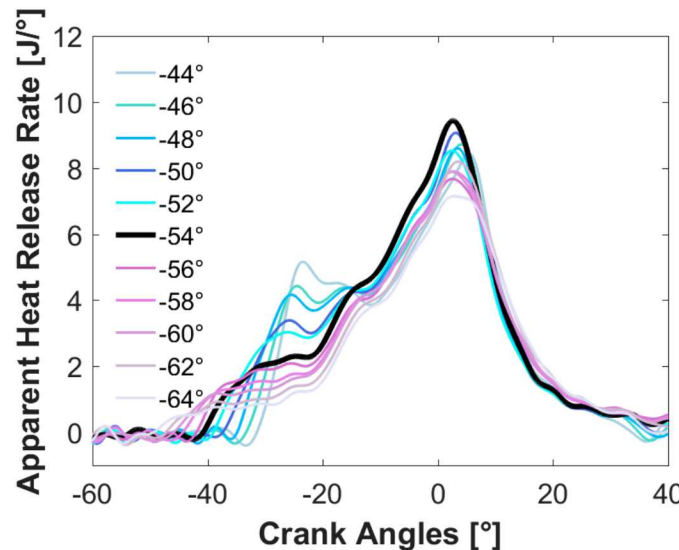
# Baseline (1000 rpm, 2.8 bar IMEP) parametric sweep: O<sub>3</sub> and ST



Ozone sweep



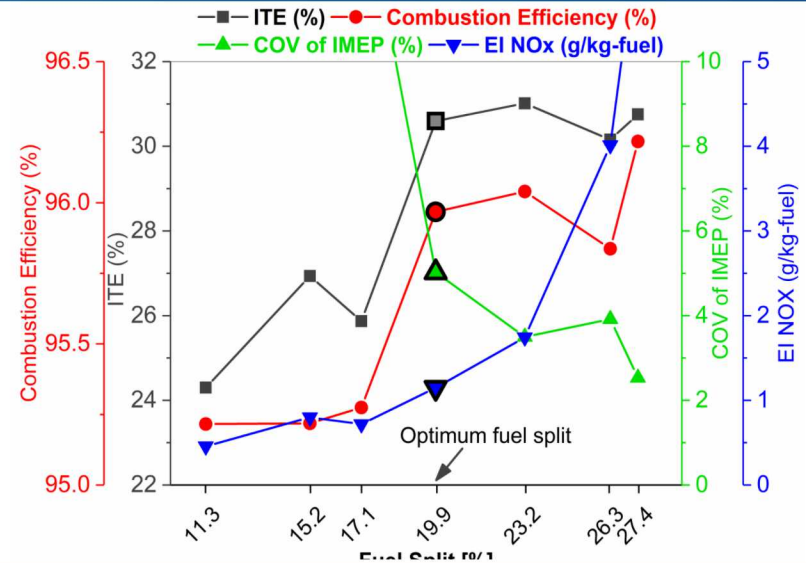
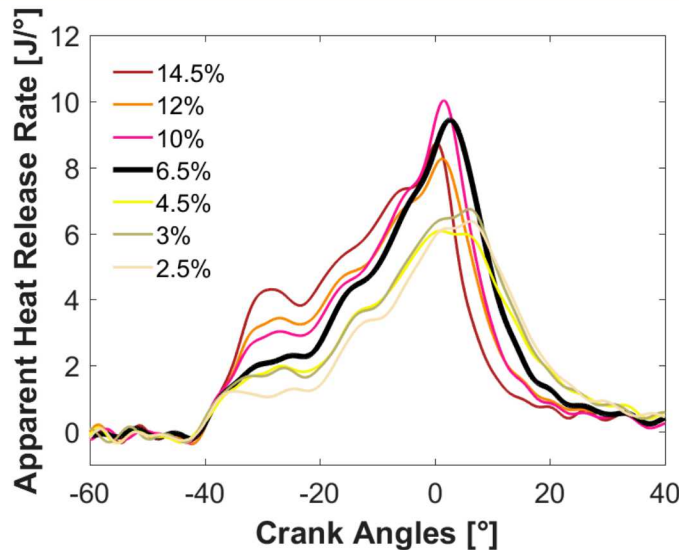
ST sweep



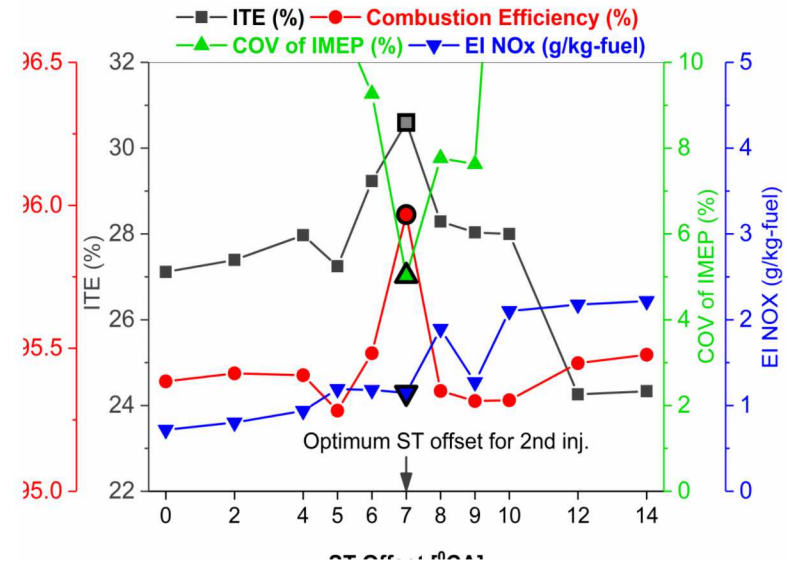
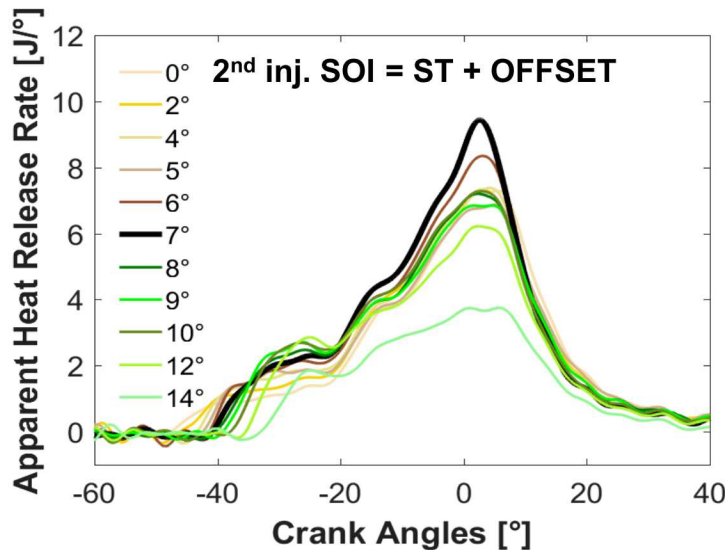
# Baseline (1000 rpm, 2.8 bar IMEP) parametric sweep: Fuel split and 2<sup>nd</sup> inj. SOI



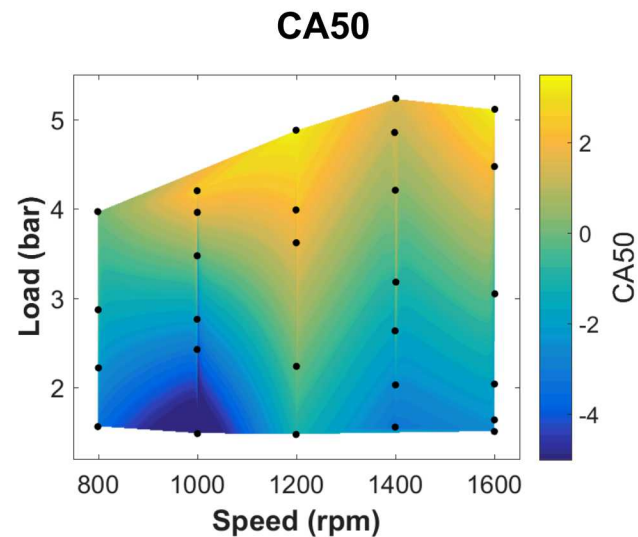
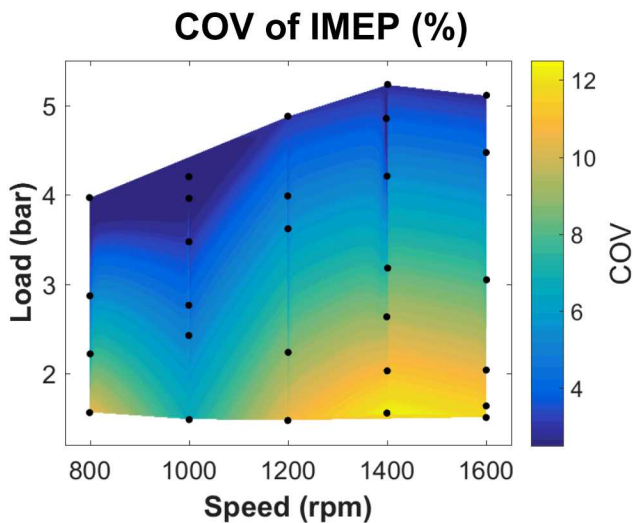
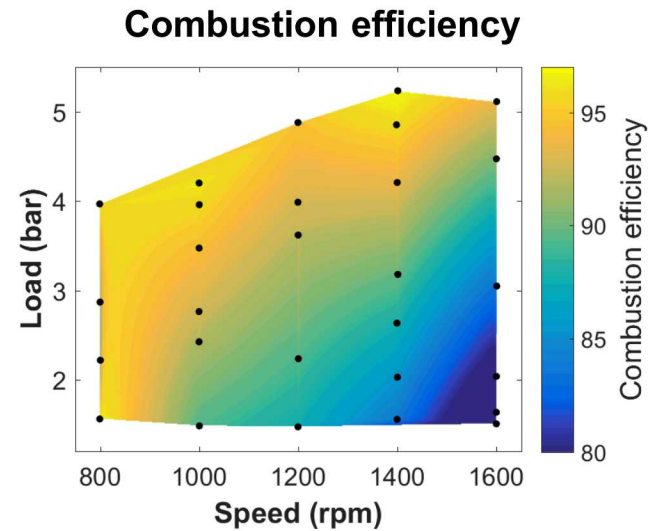
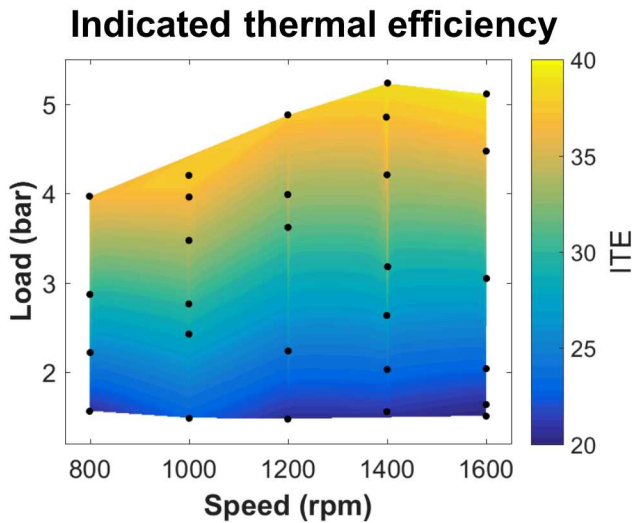
Fuel split sweep



2<sup>nd</sup> inj. SOI

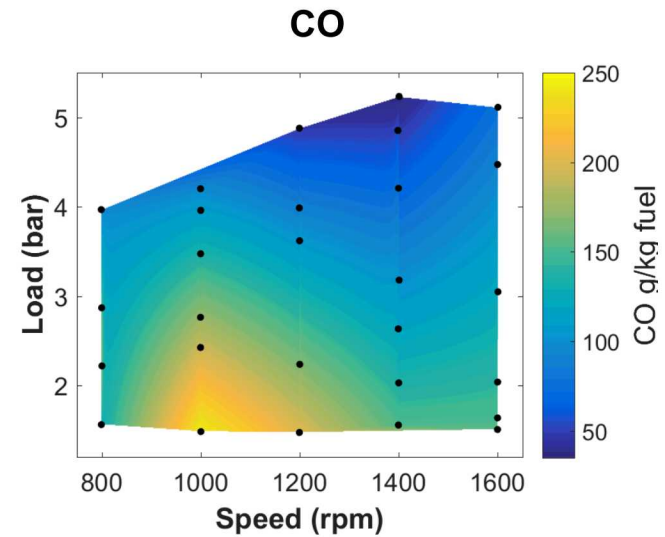
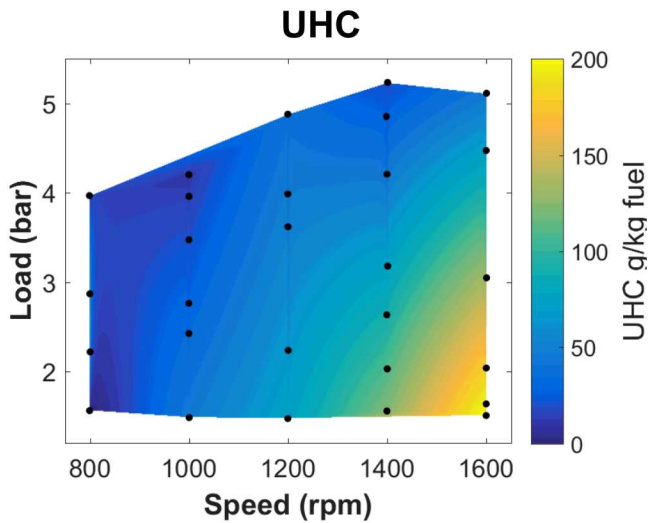
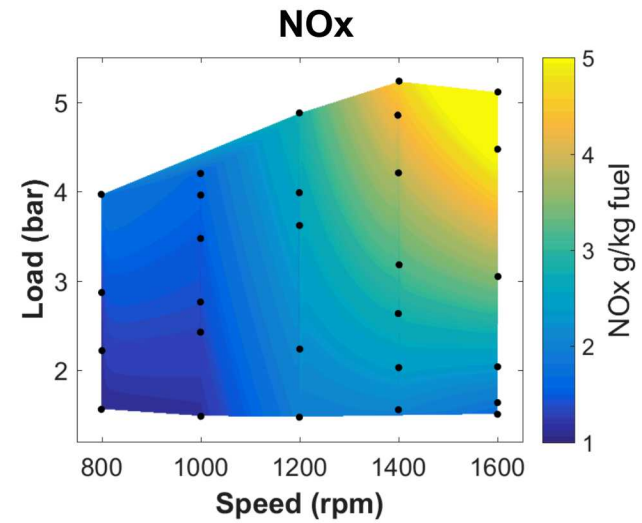
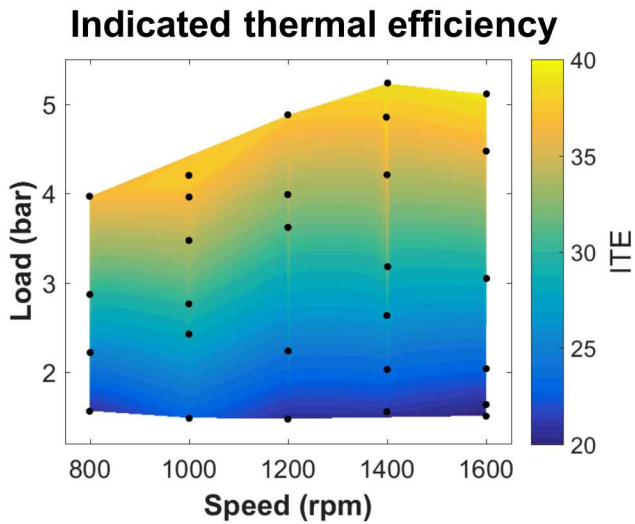


# Effect of O<sub>3</sub>: Engine performance

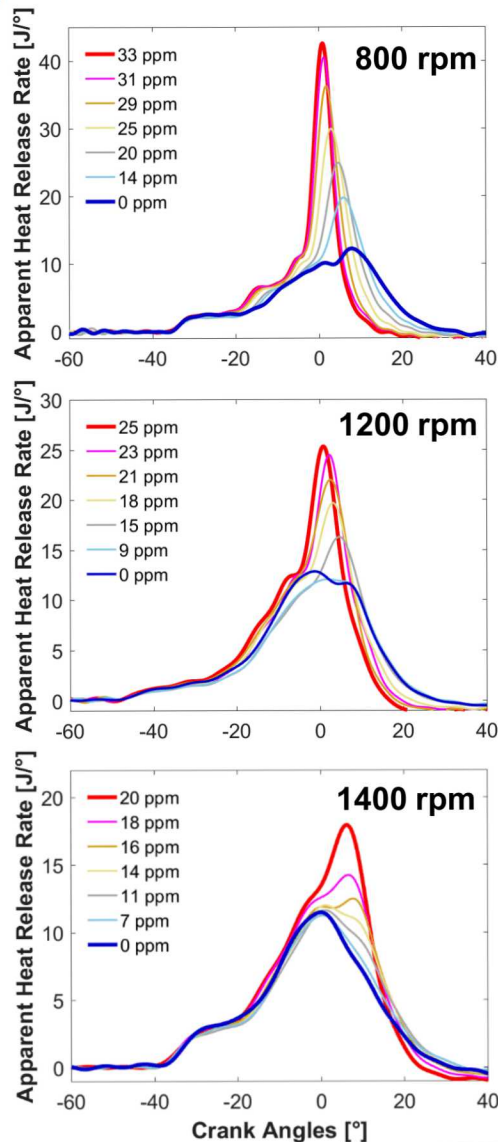




# Effect of O<sub>3</sub>: Emissions

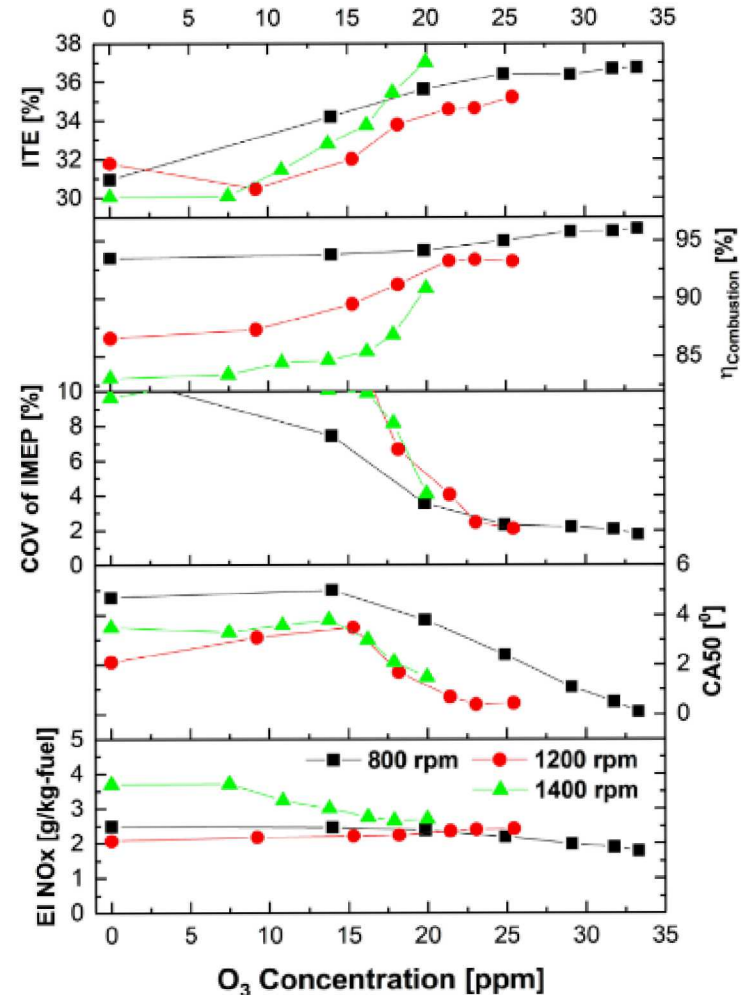


# Effect of engine speed @Fixed load of 4 bar

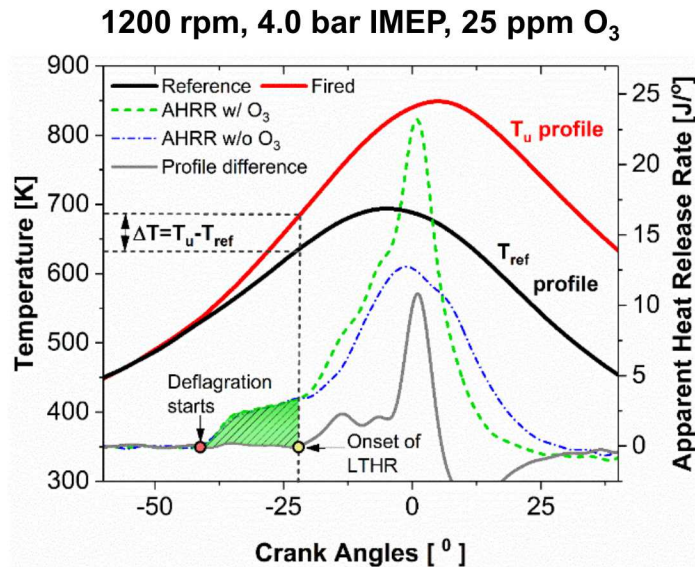


- LTHR steadily weakened with increase in engine speed
- Higher O<sub>3</sub> increased end gas HTHR and advanced its phasing

- O<sub>3</sub> addition increases combustion efficiency, decreased cyclic variability, and produces optimal combustion phasing
- Peak ITE ~ 38%
- NO<sub>x</sub> emissions remained flat or declined with O<sub>3</sub> addition



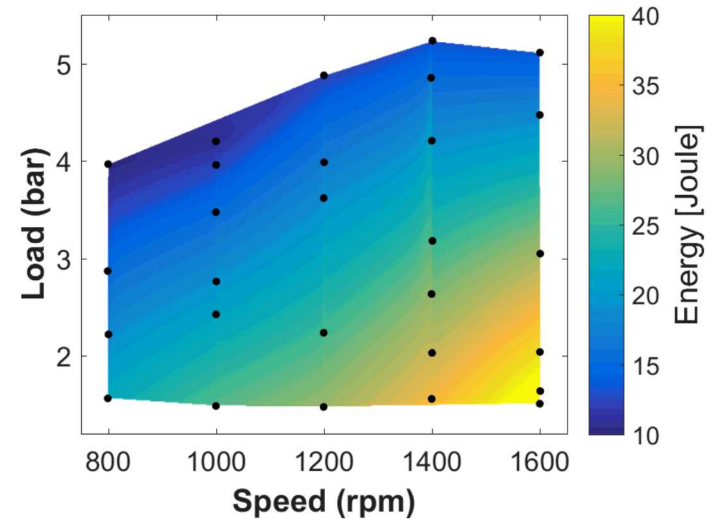
# Energy requirements for end gas auto-ignition



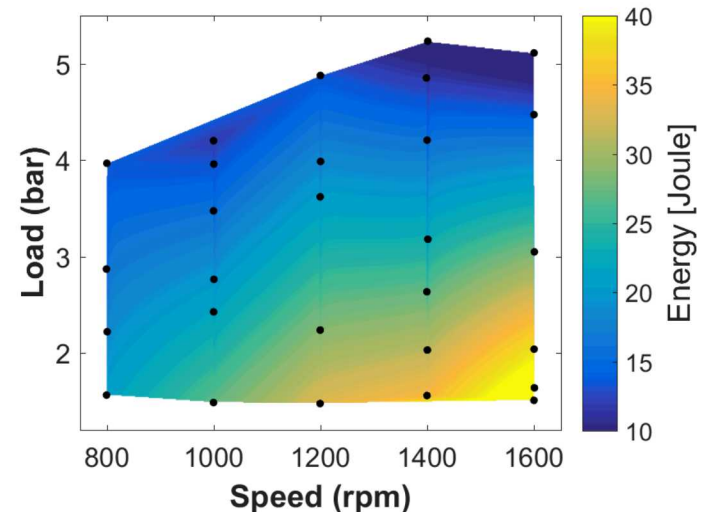
- End-gas auto-ignition  $\propto$  added O<sub>3</sub>
  - Effectiveness  $\downarrow$  with engine speed  $\uparrow$
- End-gas  $\Delta T$  required for optimal auto-ignition increases with higher engine speeds & lower loads
- Good match between integrated heat release to LTHR onset and temperature difference methods

Conceptual model of end-gas energy requirements will enable development of a simplified modeling framework

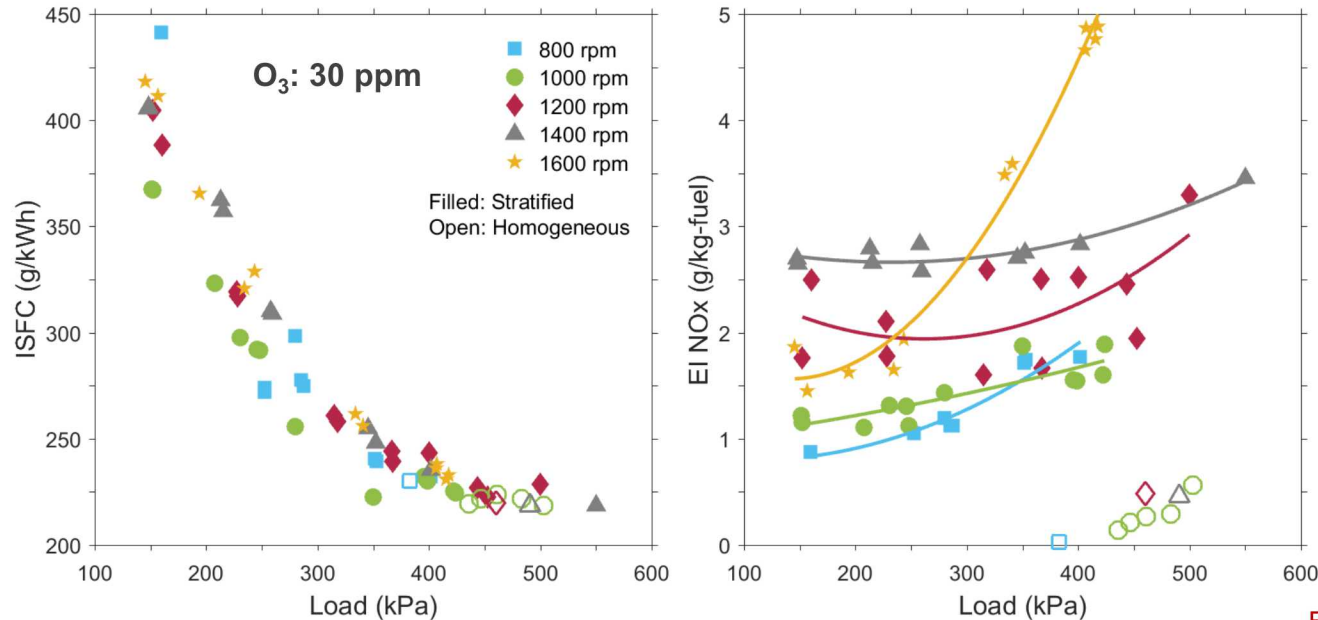
Energy by integration



$$E = m_u C_{p,u} (T_{u,ref} - T_{u,SACI})$$



# O<sub>3</sub>-addition greatly benefits SACI emissions & efficiency



Biswas & Ekoto,  
SAE WCX, Apr 2019

## O<sub>3</sub>-Enhanced SACI

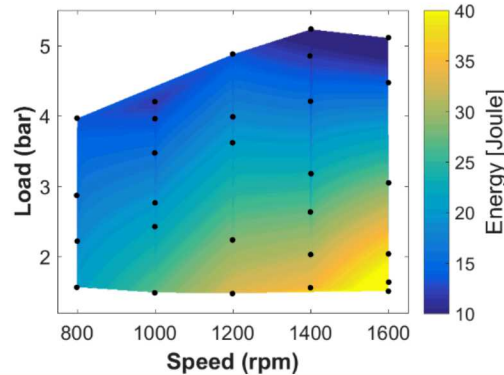
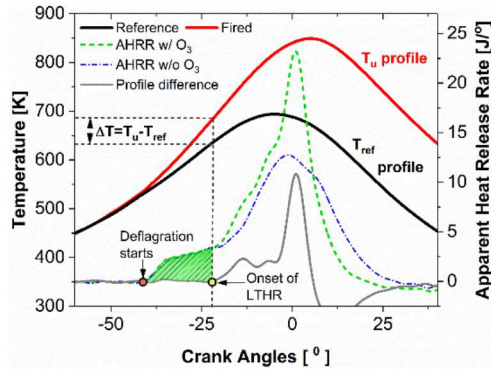
### • Partially Stratified Operation

- ↑ Robust speed/load range
- ↑ 9% lower indicated specific fuel consumption (ISFC) w/ O<sub>3</sub> at lowest loads
- ↓ Problematic nitrogen oxide (NOx) & particulate matter (PM) emissions

### • Homogeneous Operation

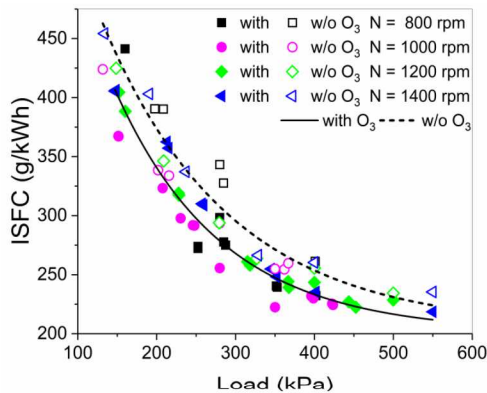
- ↑ Similar efficiency to the partially stratified strategy
- ↑ Ultra-low PM/NOx emissions
- ↓ Does not work well at lower loads and higher speeds



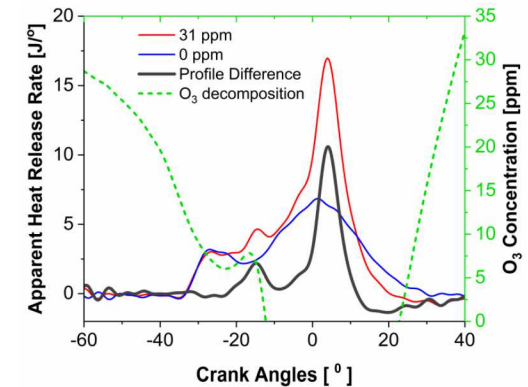


O<sub>3</sub> addition lowers the required thermal energy for auto-ignition

- Enables lower late-injection quantities, which helps to keep NOx/PM emissions under control



- Rapid decomposition of O<sub>3</sub> coincided with the onset of end-gas auto-ignition
- O<sub>3</sub> addition accelerates the onset of high-temperature heat release



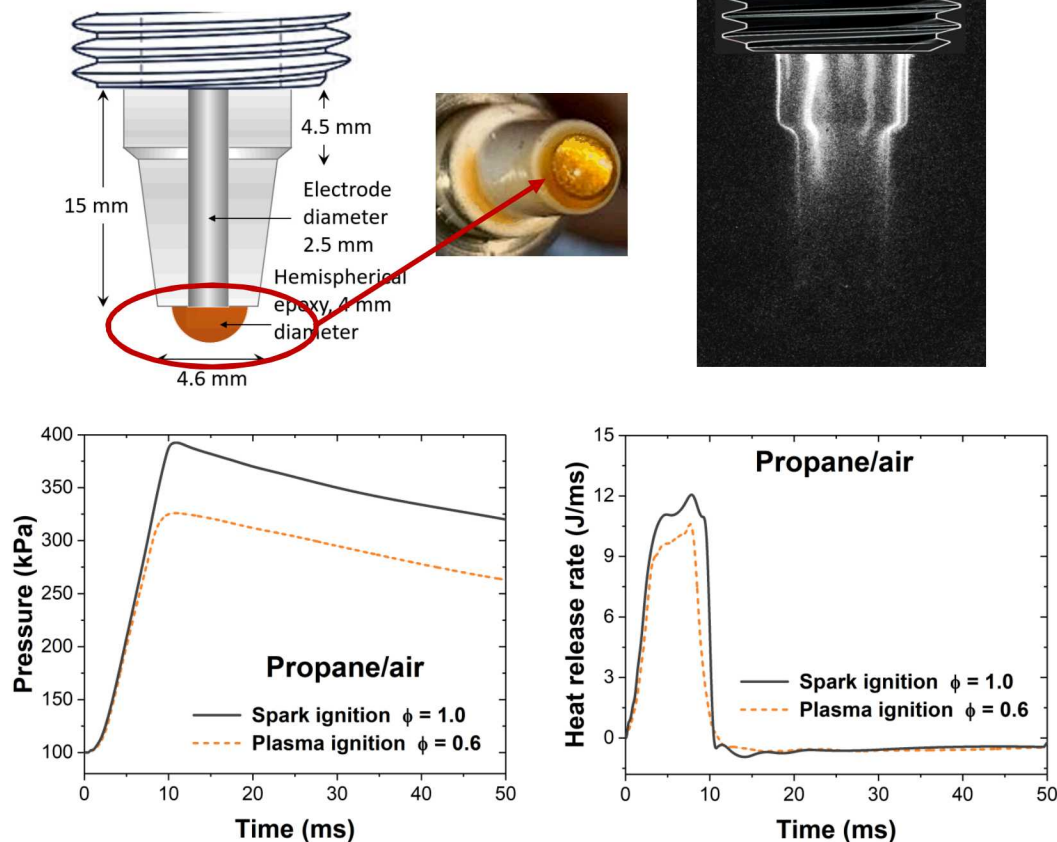
Up to 9% reduction in ISFC with O<sub>3</sub> addition enables controlled lean SACI operation for a range of low load/low speed conditions

***Advanced ignition has the potential to entirely eliminate 2<sup>nd</sup> injection***

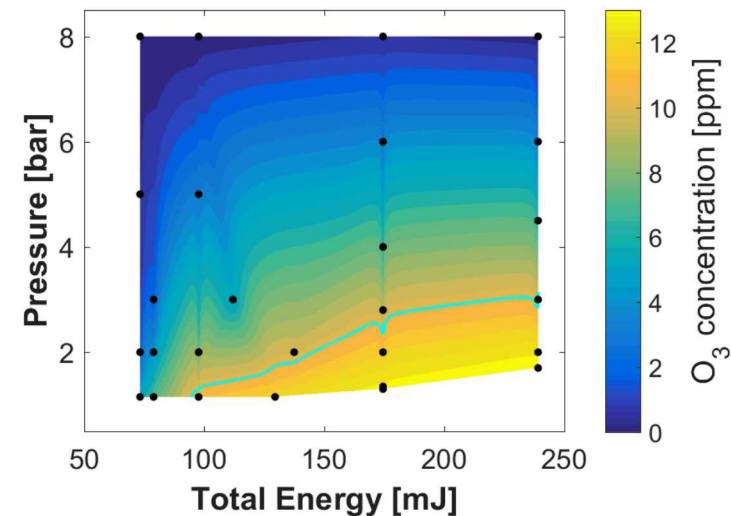
# Low-temperature plasma igniters are advantageous for high-efficiency mixed-mode engine combustion



Plasma ignition produces stronger ignition kernels and faster flame propagation, expands dilute/lean ignition limits



Plasma discharges in air produces enough  $O_3$  to enable end gas auto-ignition



Cyan line represents 10 ppm of  $O_3$

To generate 50 ppm of  $O_3$ ,  
 $50/10 = 5$  bursts of 10 pulses  
are required (Total 50 pulses)



Thank you!