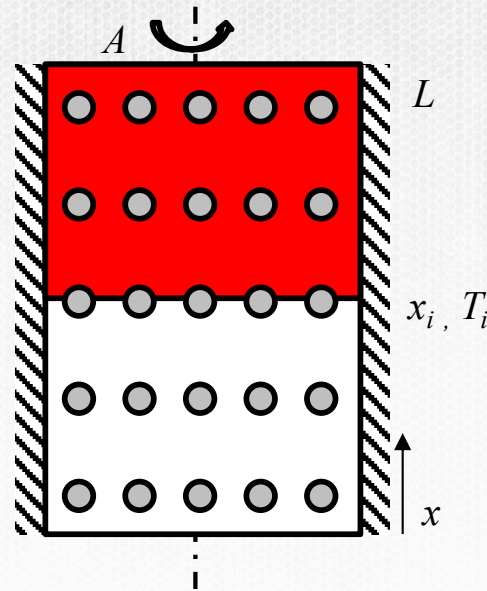




Sodium Pumping via Condensation within a Non-Wetting Porous Structure

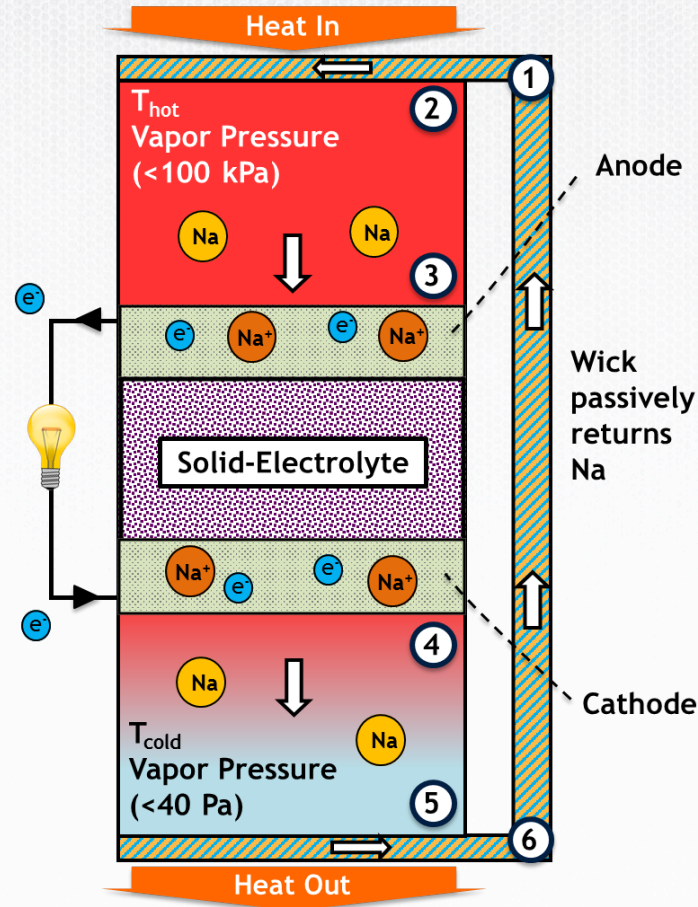


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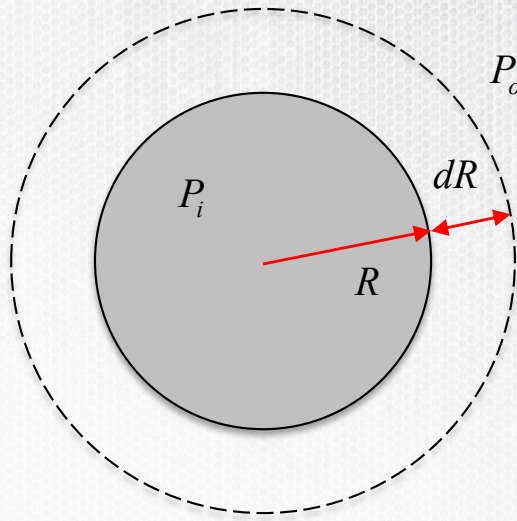
Motivation

- Liquid metals are used in applications with large heat transfer requirements (e.g. heat pipes, electrochemical systems)
- Goal is to pump liquid sodium in a closed loop system using capillary action
- A large capillary pressure increase in the liquid-vapor interface within non-wetting porous surface is required in the Na-TEC



Interfacial Physics

Considering the interface between two fluids, the work required to displace the interface by dR is given by:



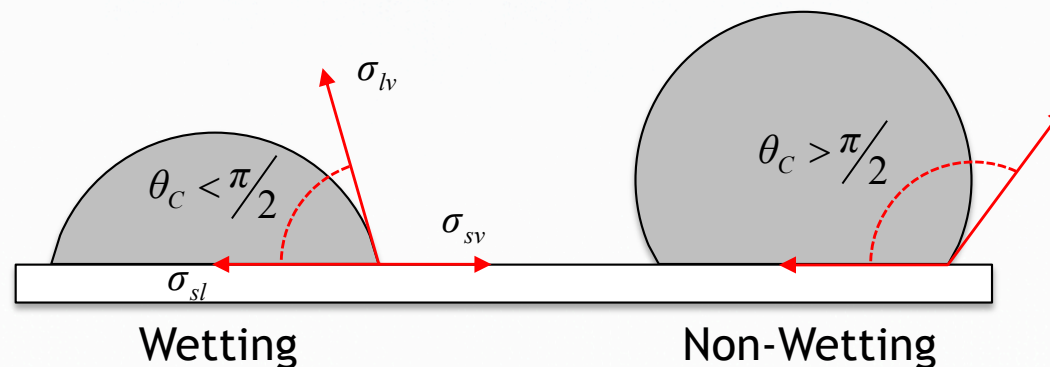
$$\delta W = P_i dV - P_o dV - \sigma dA_s$$

For a sphere at equilibrium:

$$(P_i - P_o)(4\pi R^2 dR) = \sigma(8\pi R dR)$$

Laplace Pressure $P_i - P_o = \Delta P = 2K_{12}\sigma$ where $K_{12} = \frac{1}{2}\left(\frac{1}{R_1} + \frac{1}{R_2}\right) \rightarrow \left(\frac{1}{R}\right)_{\text{sphere}}$

Young-Dupree Equation $\sigma_{sv} = \sigma_{sl} + \sigma_{lv} \cos \theta_C$



Conservation of Mass

If mass transfer occurs by advection and diffusion, the two commonly used models are the advection-diffusion model (ADM) and the dusty-gas model (DGM).

Assume a binary mixture within a porous structure:

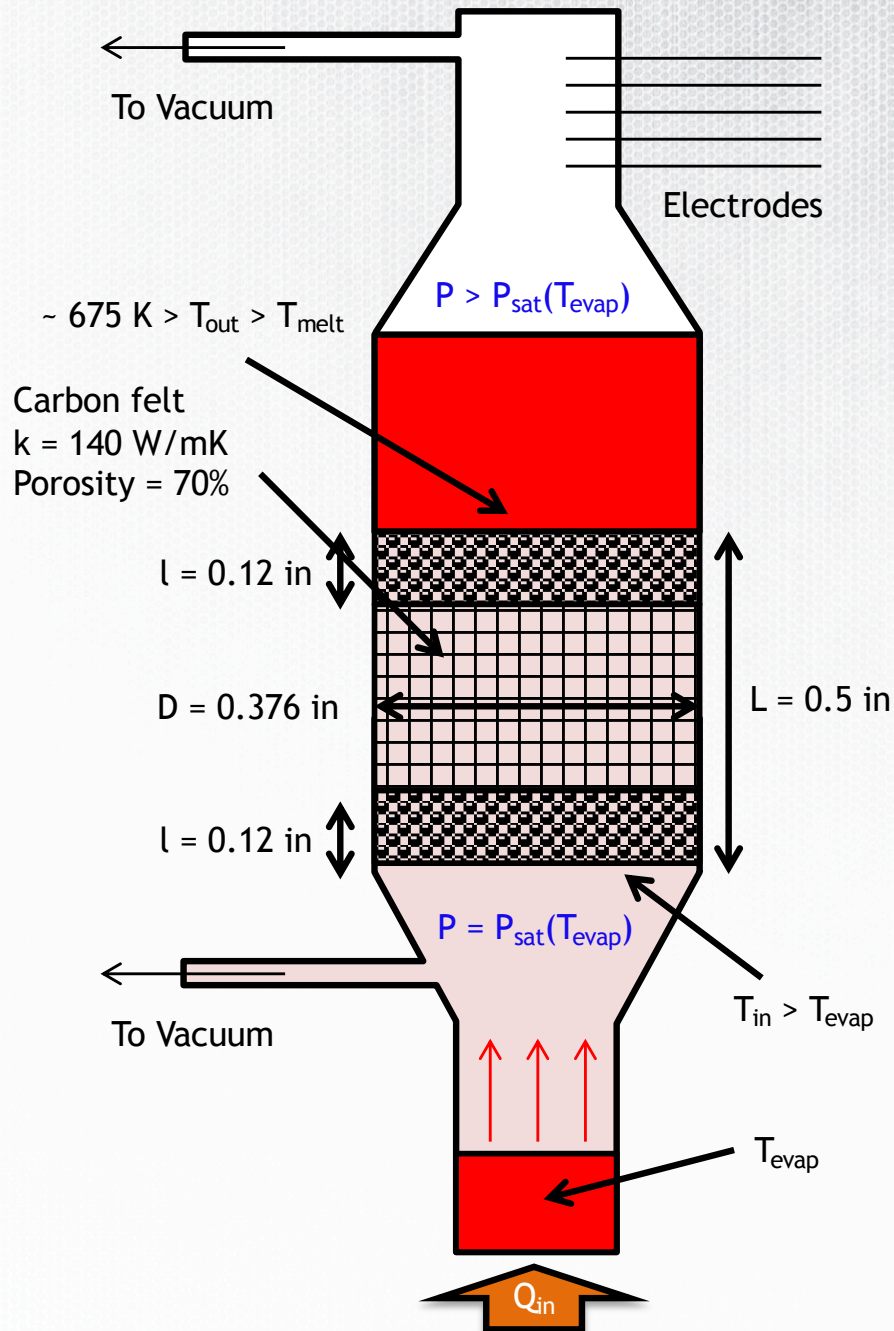
$$\text{Conservation of mass} \quad \varepsilon \frac{\partial \rho_i}{\partial t} + \nabla \cdot \left[\underbrace{\rho_i \left(-\frac{\kappa}{\mu} \nabla P \right)}_{\text{Advection}} - \underbrace{\rho \varepsilon D_{ij} \nabla \omega_i}_{\text{Binary diffusion}} \right] = 0$$

The permeability is given by the Blake-Kozeny expression $\kappa = \frac{D_p^2 \varepsilon^2}{150(1 - \varepsilon^3)}$

To account for Knudsen diffusion, the permeability is modified by the Klinkenberg factor:

$$\kappa = \kappa_\infty \left(1 + \frac{D_{iK} \mu_i}{\kappa_\infty P} \right) \quad \text{from kinetic theory:} \quad \begin{cases} D_{iK} = \varepsilon \frac{4}{3} d_{pore} (2\pi RT)^{-1/2} \\ \mu_i = \rho d_{pore} Kn (2RT / \pi)^{1/2} \end{cases}$$

Vacuum Pressure Experiment

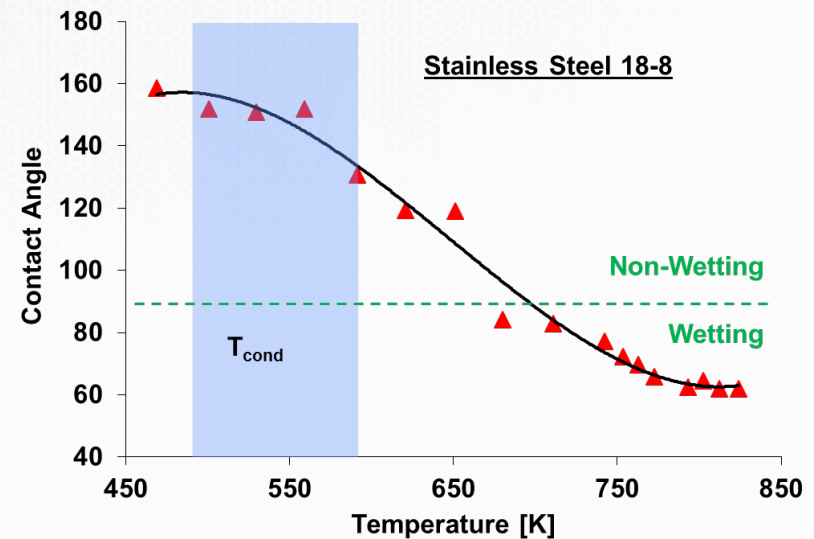
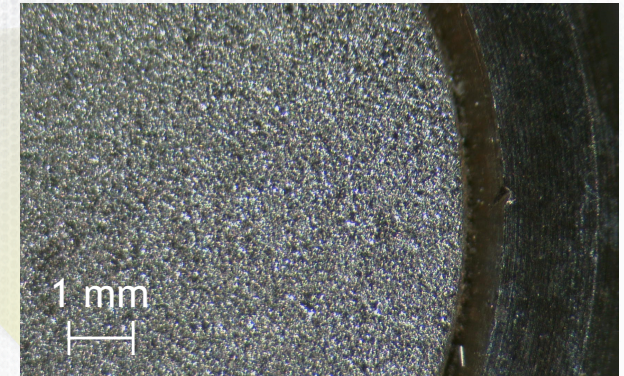


SS 316 thermally pressed inside cylinder

$$\varepsilon = 0.25$$

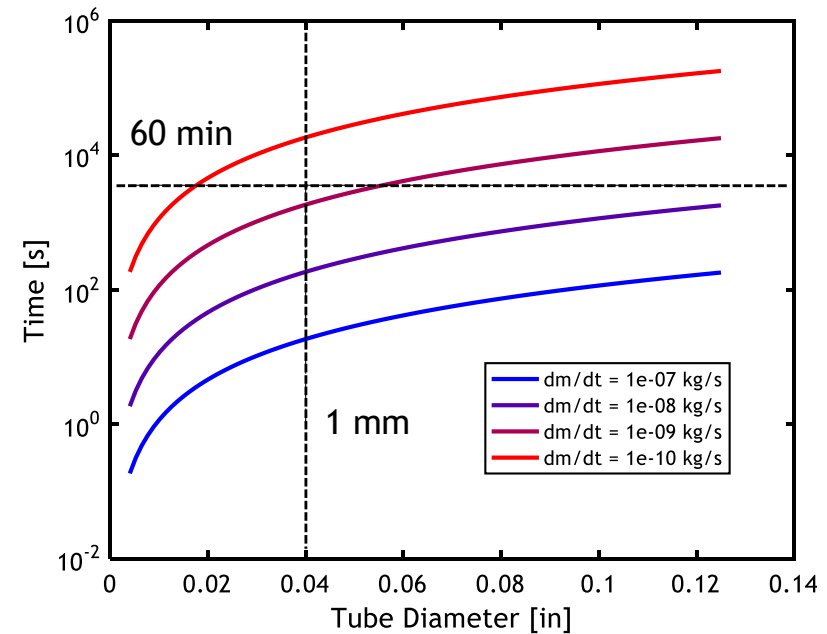
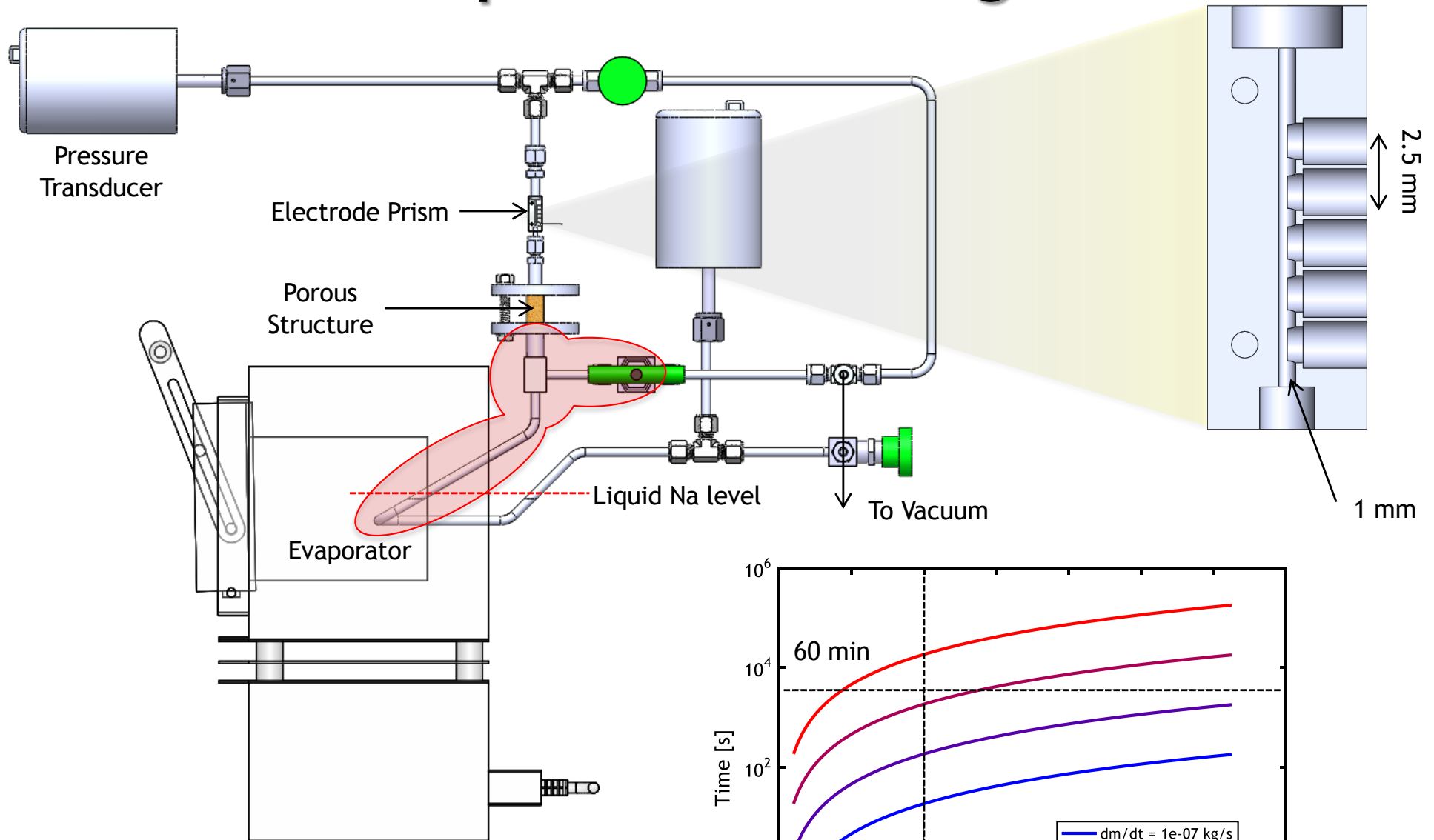
$$0.1 \mu\text{m} < d_p < 4.15 \mu\text{m}$$

0.376 in



Taylor & Ford, U.K. Atomic Energy Authority Report, 1955

Experimental Design

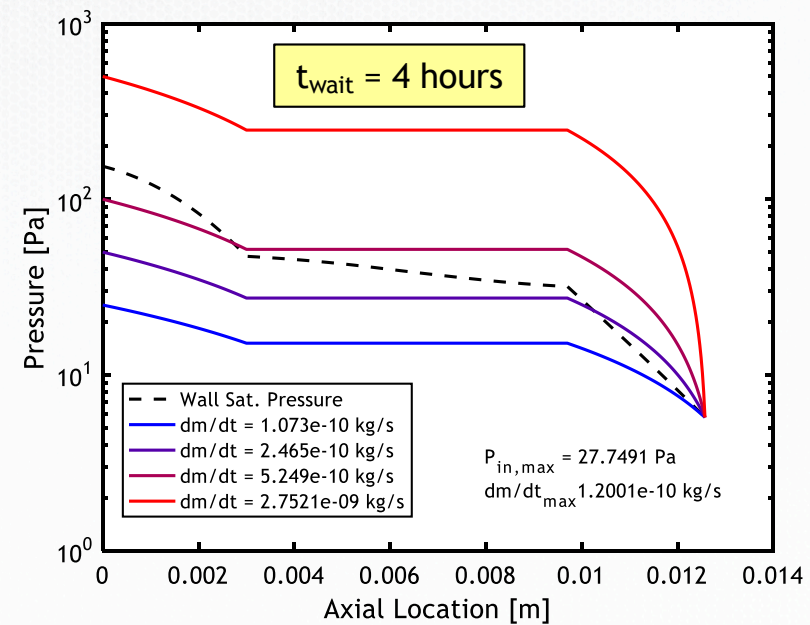
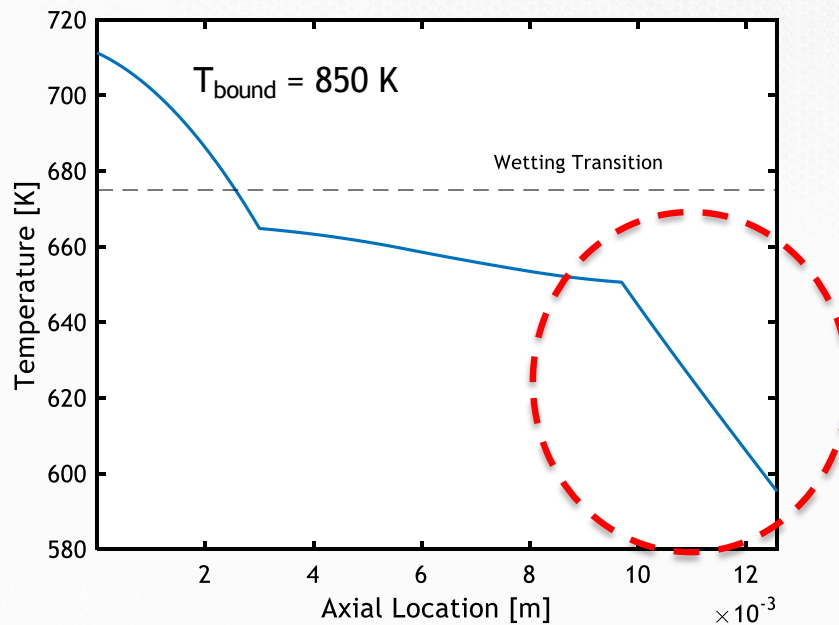


Maximum Mass Flowrate

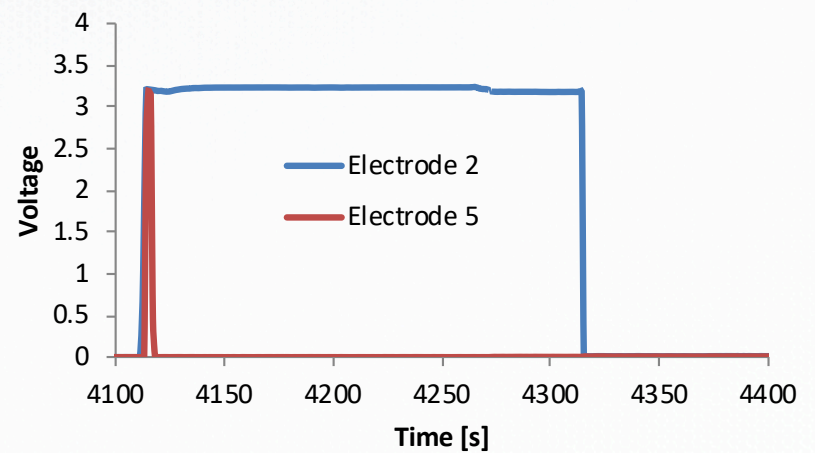
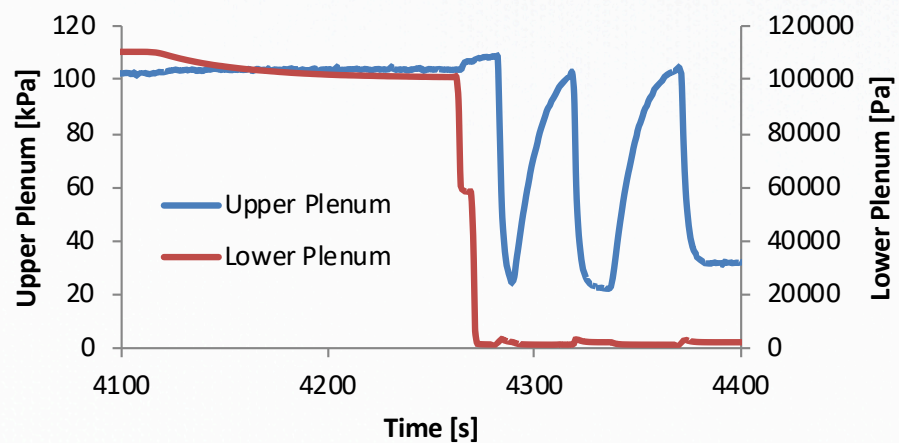
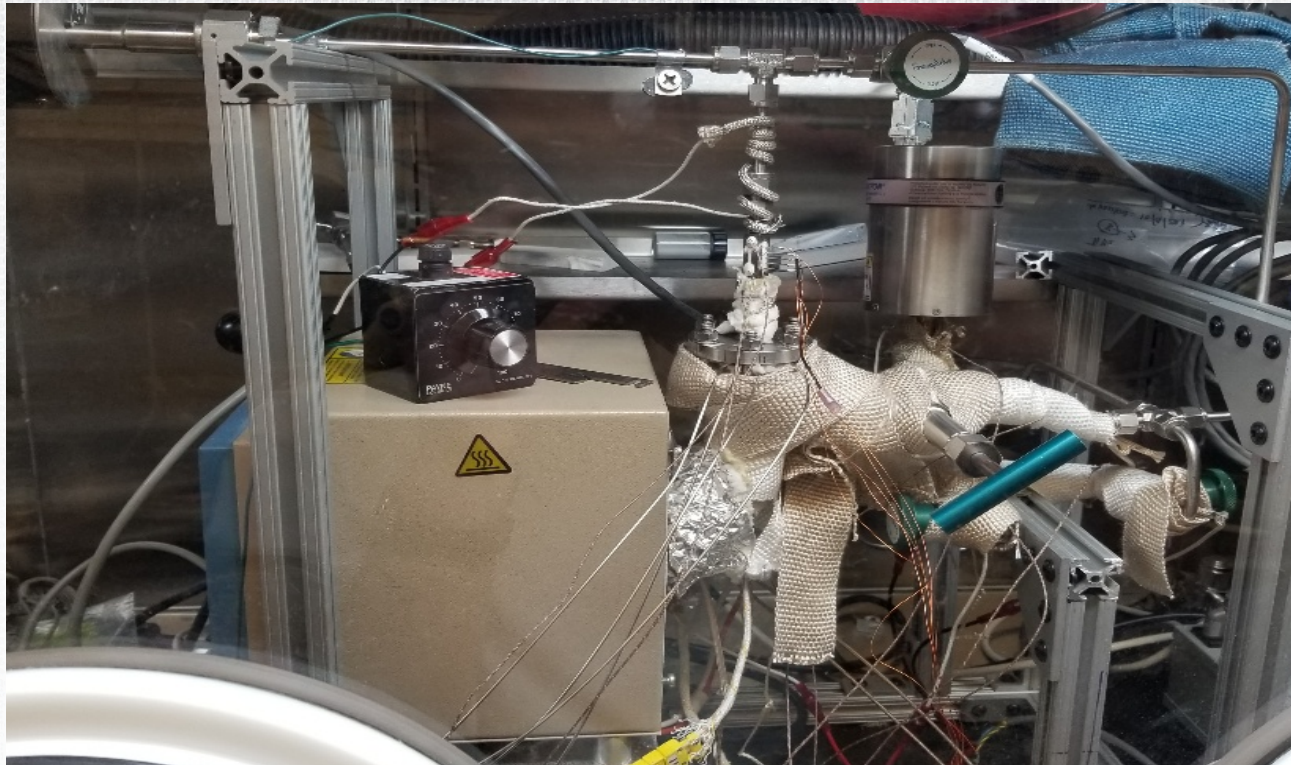
Assume there is no presence of argon, so transport is just with a single species of sodium vapor. Solve the compressible Darcy equation with the following considerations:

Kelvin Equation at interface
$$P(x_i) = P_{sat}(T_i) \exp\left(\frac{P_L}{\rho_l T_i \bar{R}}\right)$$

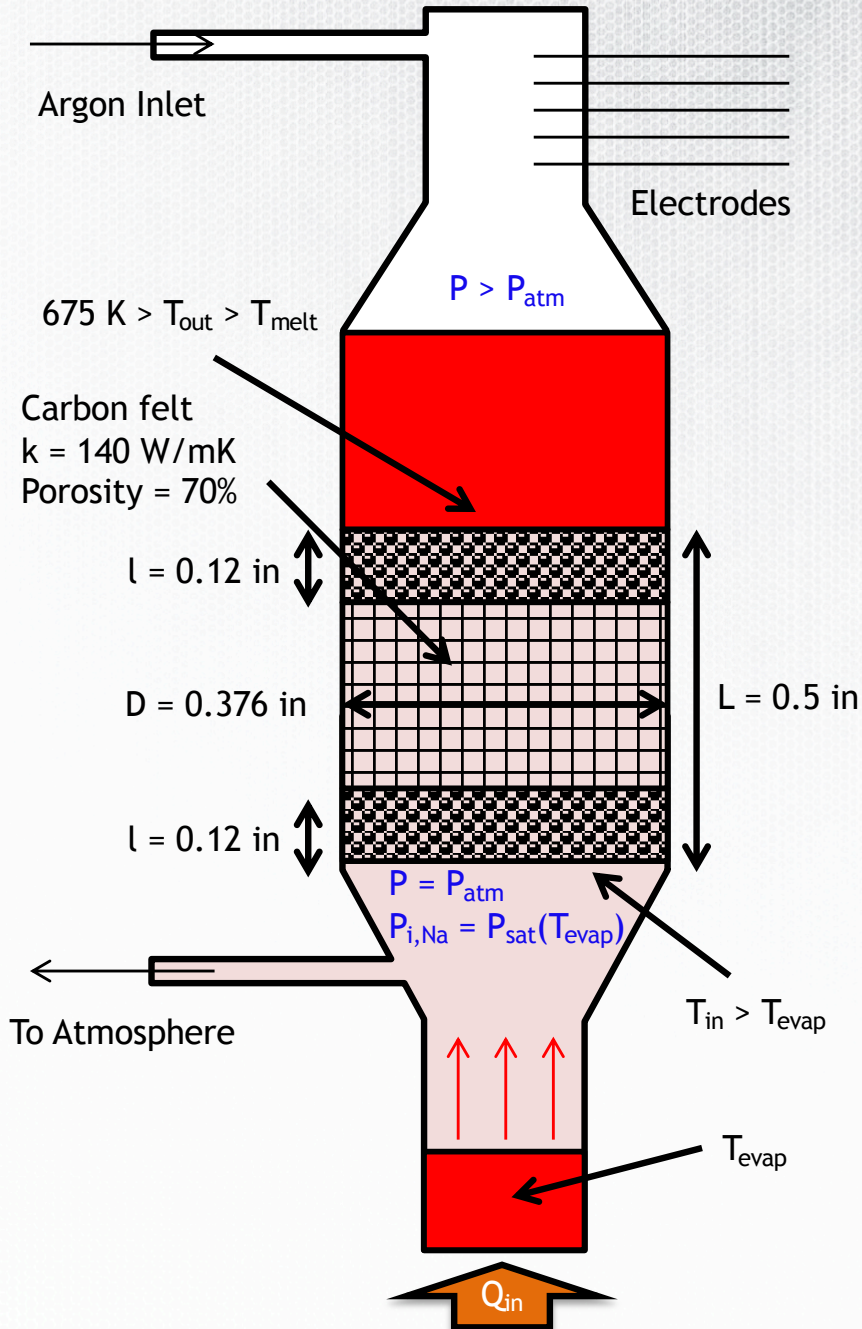
Momentum Conservation for slip flow
$$u = -\frac{\kappa_\infty (1 + 8\beta Kn)}{\mu} \frac{dP}{dx}$$



Failed Vacuum Experiment



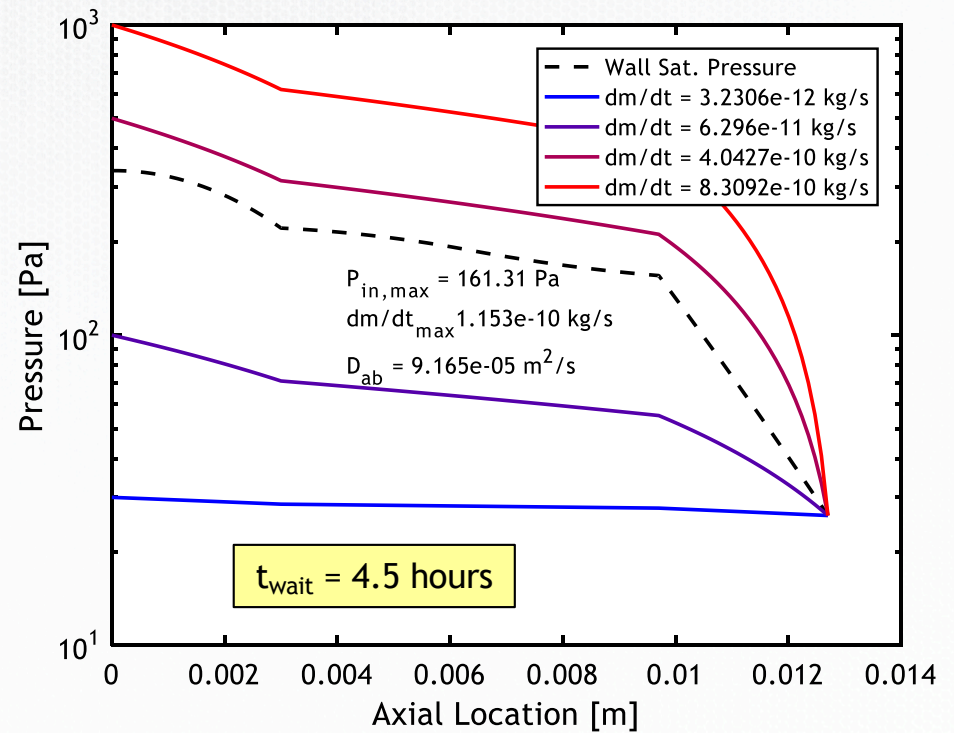
Gauge Pressure Experiment



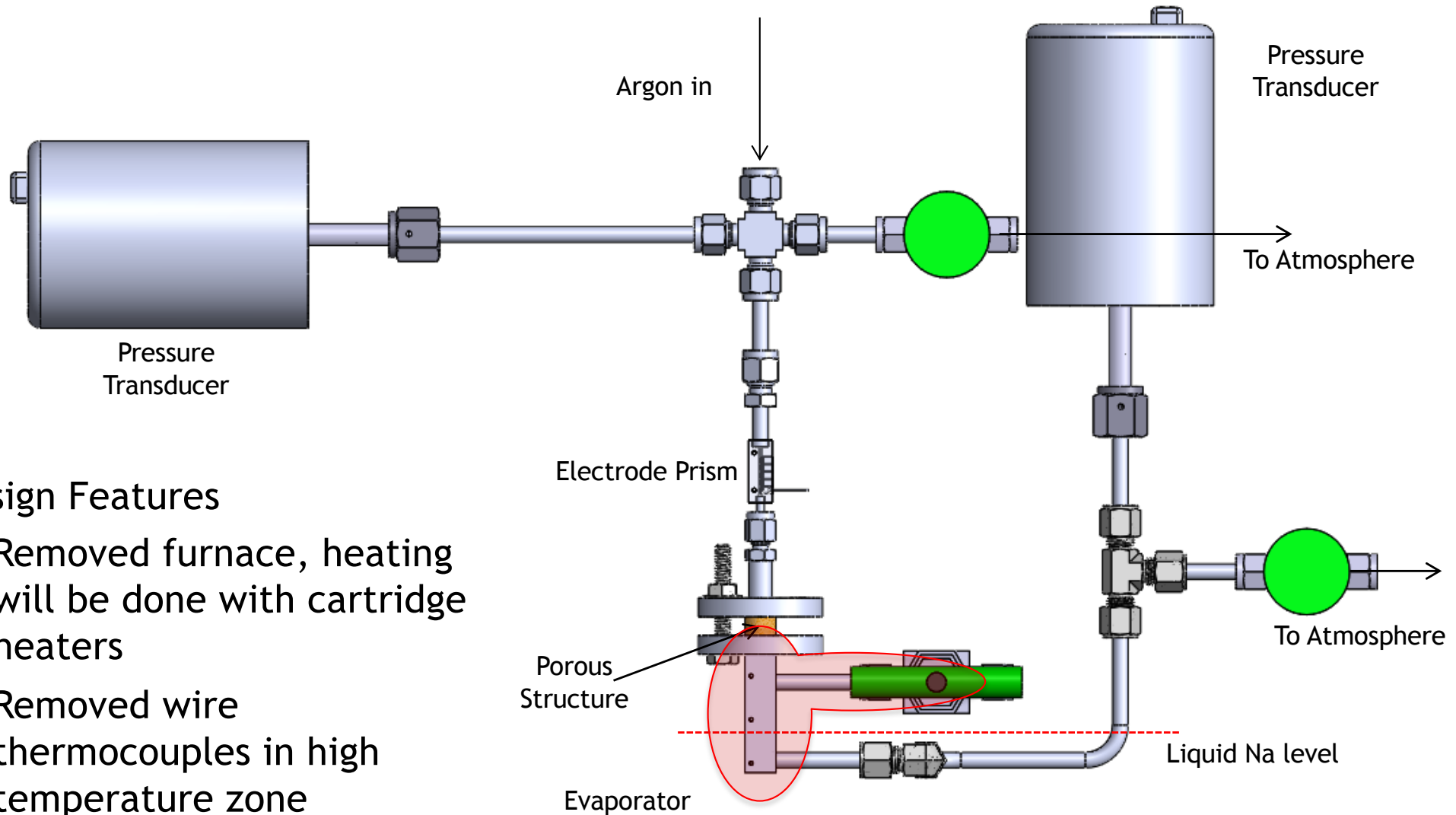
$$u = \frac{\rho_{Na} u_{Na} + \rho_{Ar} u_{Ar}}{\rho} = \omega_{Na} u_{Na}$$

If $\omega_{Na} \ll 1$

use dilute gas approximation to neglect advection



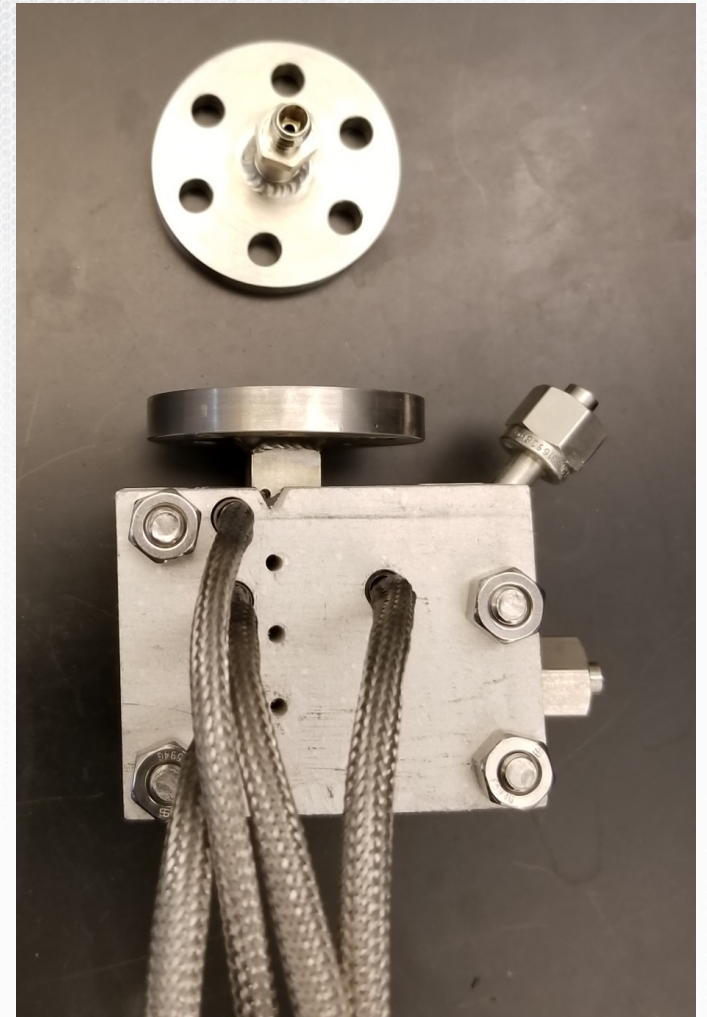
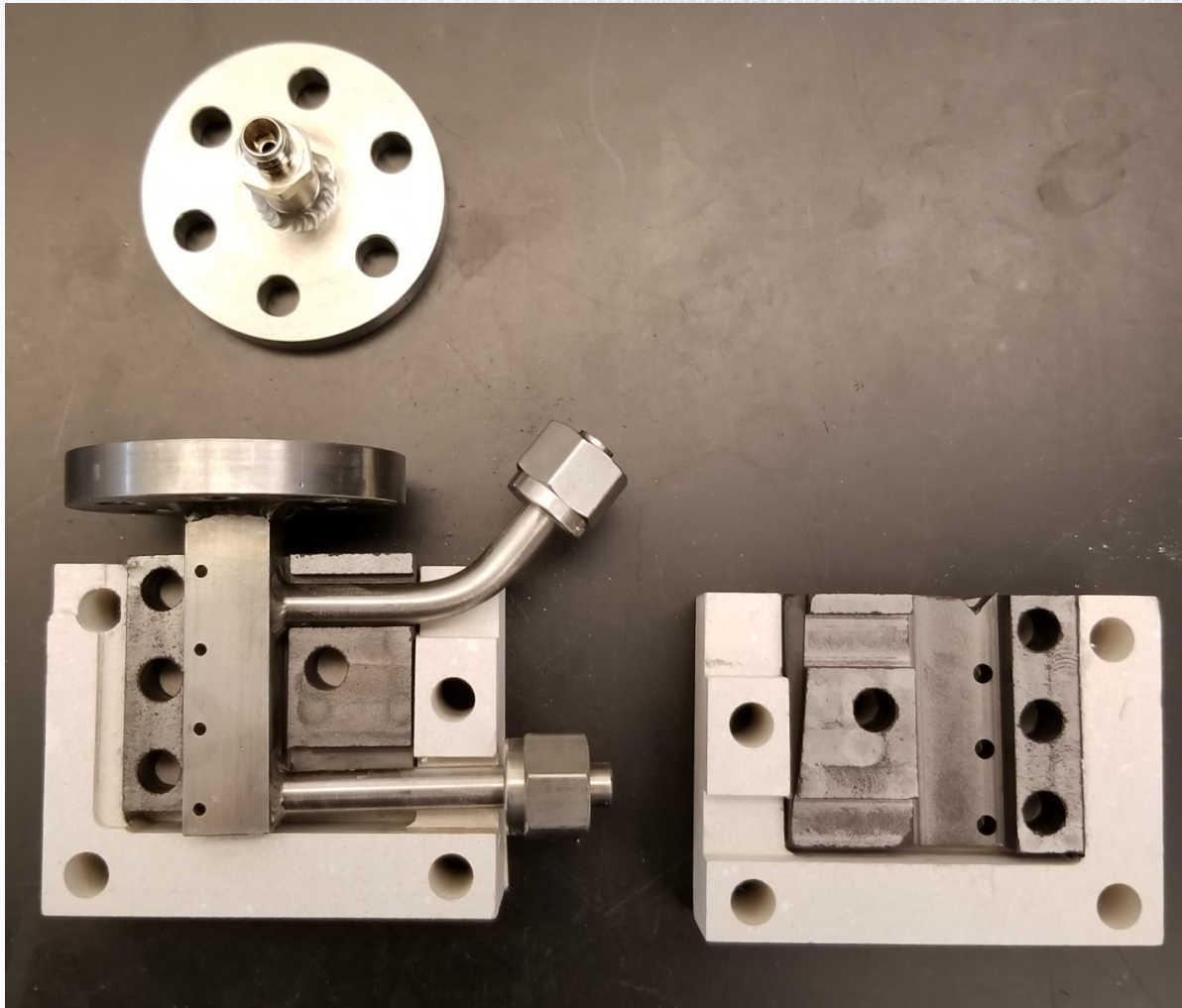
New Experimental Design



Design Features

- Removed furnace, heating will be done with cartridge heaters
- Removed wire thermocouples in high temperature zone
- Reduced thermal mass of flanges by 36%

New Heater Design



Summary and Future Work

Summary:

- Discussed transport model for combined advection/diffusion
- Introduced the experimental design used to demonstrate sodium pumping
- Described experimental modification

Future Work:

- Finish building new experimental set-up. Gather data
- Validate experiment with conjugate heat transfer Comsol modeling
- Use DGM for modeling purposes
- Run experiment with different porous materials

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