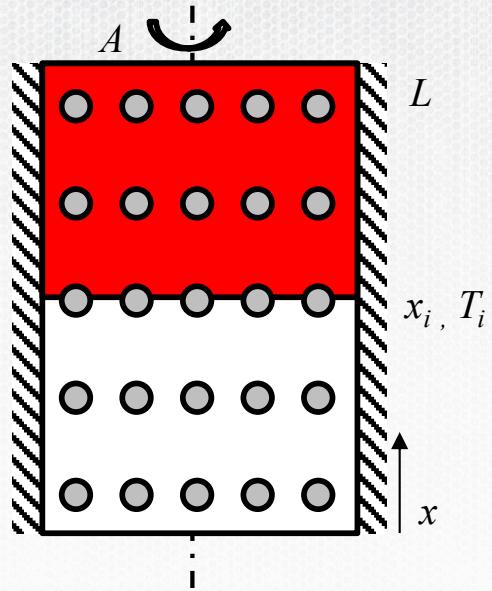




# Sodium Pumping via Condensation within a Non-Wetting Porous Structure

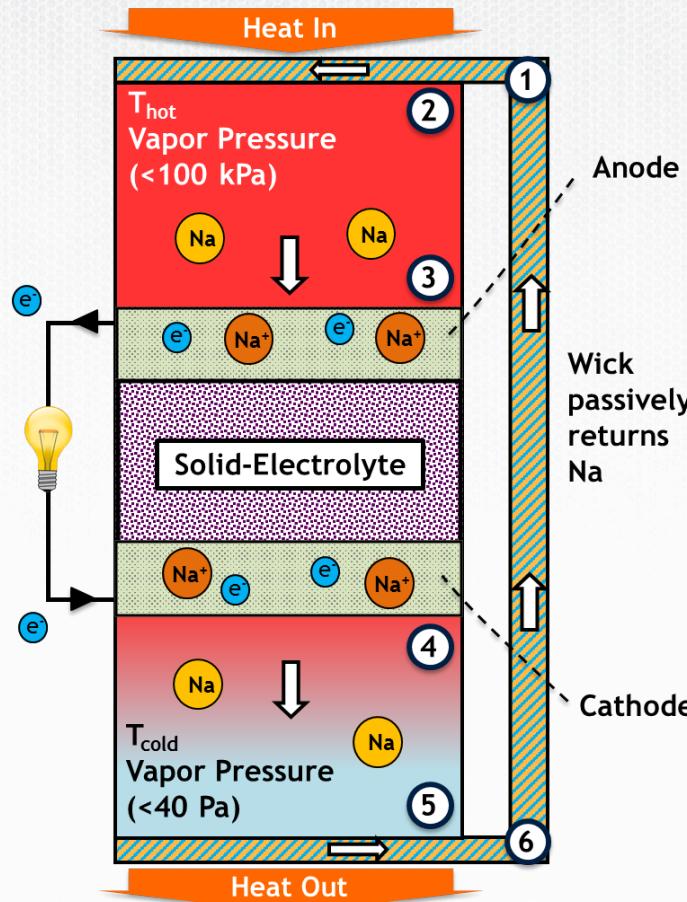


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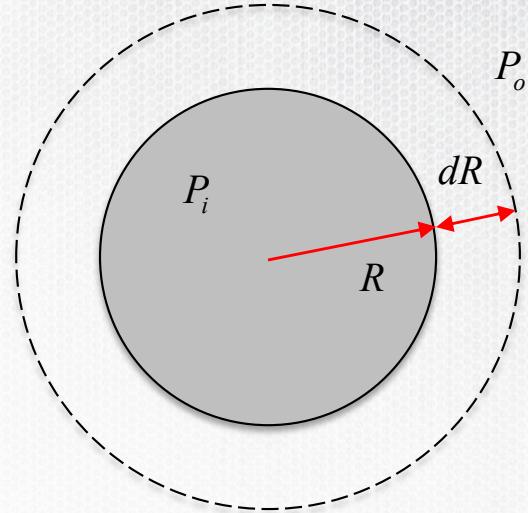
# Motivation

- Liquid metals are used in applications with large heat transfer requirements (e.g. heat pipes, electrochemical systems)
- Goal is to pump liquid sodium in a closed loop system using capillary action
- A large capillary pressure increase in the liquid-vapor interface within non-wetting porous surface is required in the Na-TEC



# Interfacial Physics

Considering the interface between two fluids, the work required to displace the interface by  $dR$  is given by:



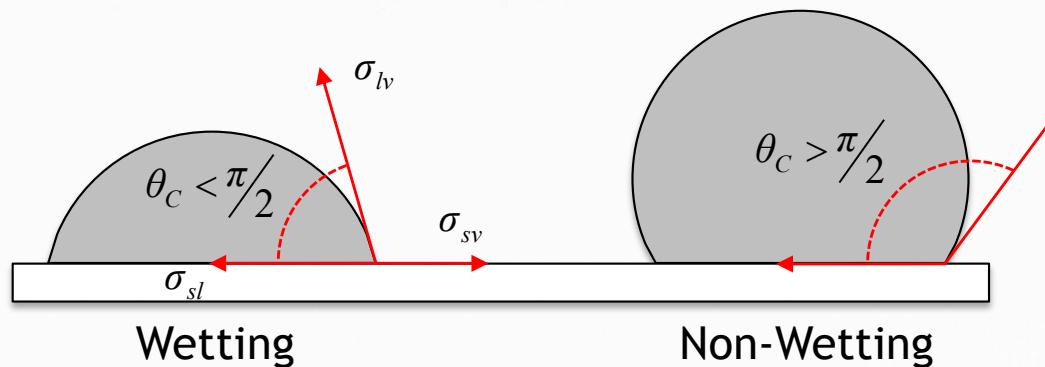
$$\delta W = P_i dV - P_o dV - \sigma dA_s$$

For a sphere at equilibrium:

$$(P_i - P_o)(4\pi R^2 dR) = \sigma(8\pi R dR)$$

Laplace Pressure  $P_i - P_o = \Delta P = 2K_{12}\sigma$  where  $K_{12} = \frac{1}{2}\left(\frac{1}{R_1} + \frac{1}{R_2}\right) \rightarrow \left(\frac{1}{R}\right)_{sphere}$

Young-Dupree Equation  $\sigma_{sv} = \sigma_{sl} + \sigma_{lv} \cos \theta_C$



# Conservation of Mass

If mass transfer occurs by advection and diffusion, the two commonly used models are the advection-diffusion model (ADM) and the dusty-gas model (DGM).

Assume a binary mixture within a porous structure:

Conservation of mass  $\varepsilon \frac{\partial \rho_i}{\partial t} + \nabla \cdot \left[ \rho_i \underbrace{\left( -\frac{\kappa}{\mu} \nabla P \right)}_{\text{Advection}} - \rho \varepsilon D_{ij} \nabla \omega_i \underbrace{\varepsilon D_{ij} \nabla \omega_i}_{\text{Binary diffusion}} \right] = 0$

The permeability is given by the Blake-Kozeny expression  $\kappa = \frac{D_p^2 \varepsilon^2}{150(1-\varepsilon^3)}$

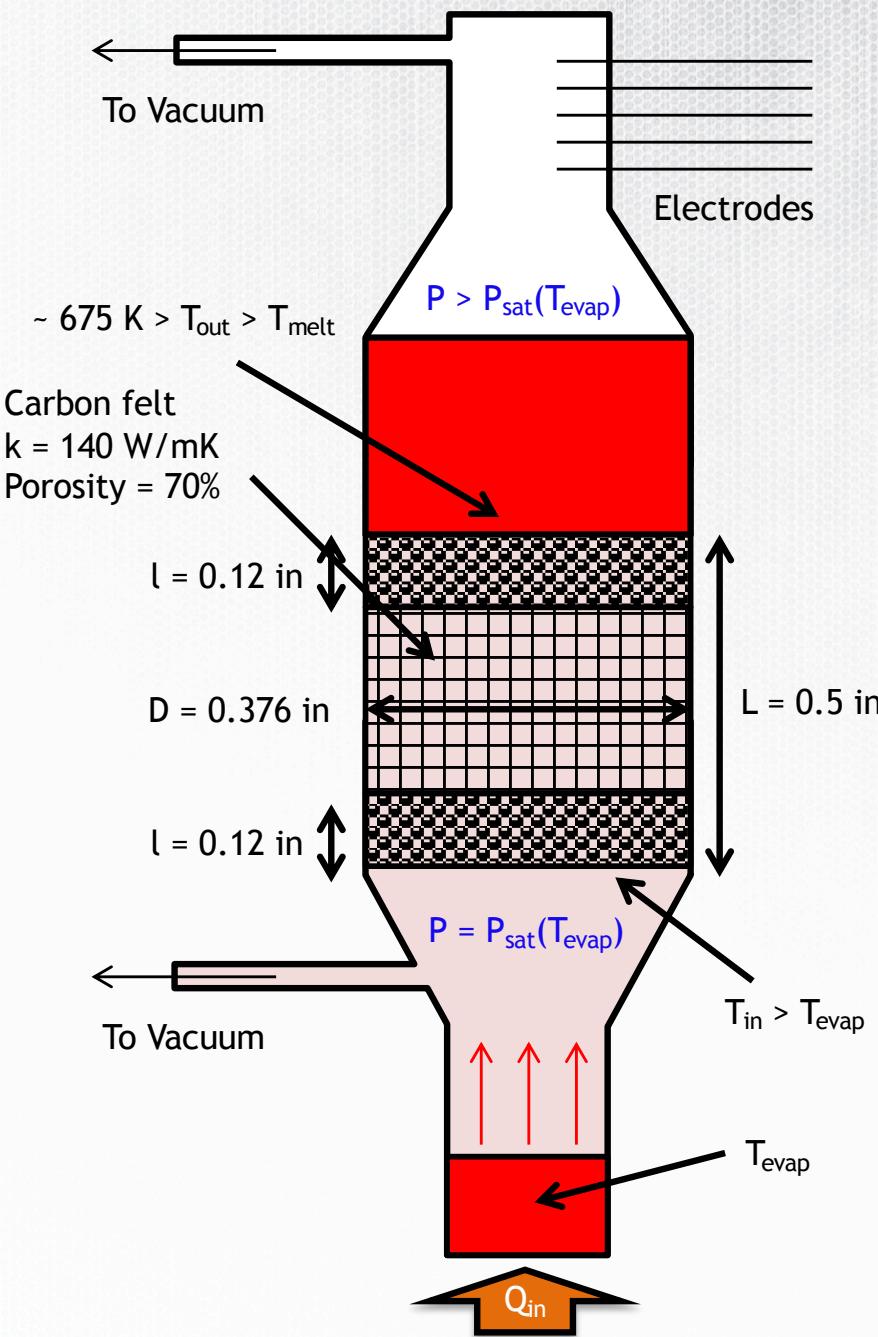
To account for Knudsen diffusion, the permeability is modified by the Klinkenberg factor:

$$\kappa = \kappa_\infty \left( 1 + \frac{D_{iK} \mu_i}{\kappa_\infty P} \right)$$

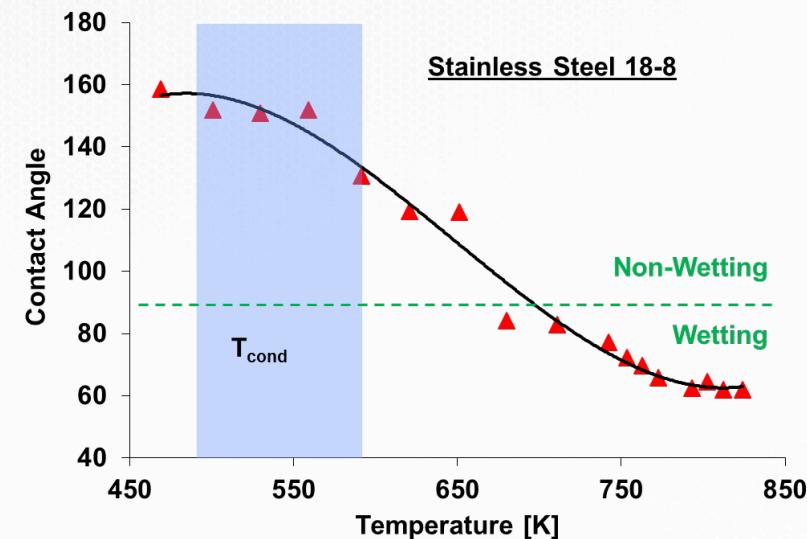
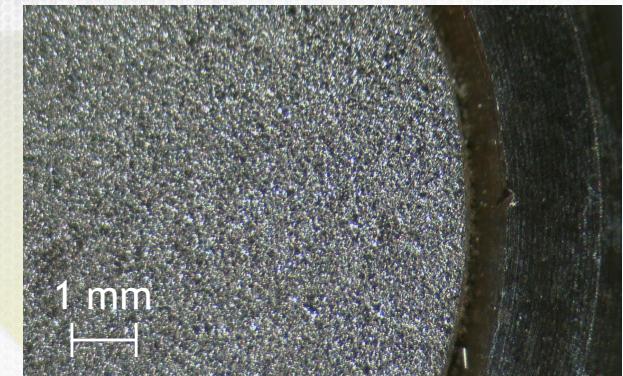
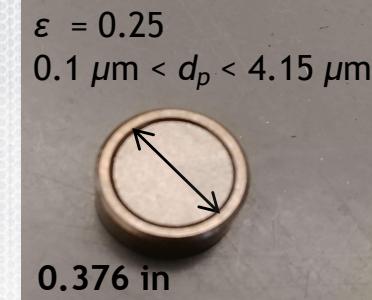
from kinetic theory:

$$\left\{ \begin{array}{l} D_{iK} = \varepsilon \frac{4}{3} d_{pore} (2\pi RT)^{-1/2} \\ \mu_i = \rho d_{pore} Kn (2RT/\pi)^{1/2} \end{array} \right.$$

# Vacuum Pressure Experiment

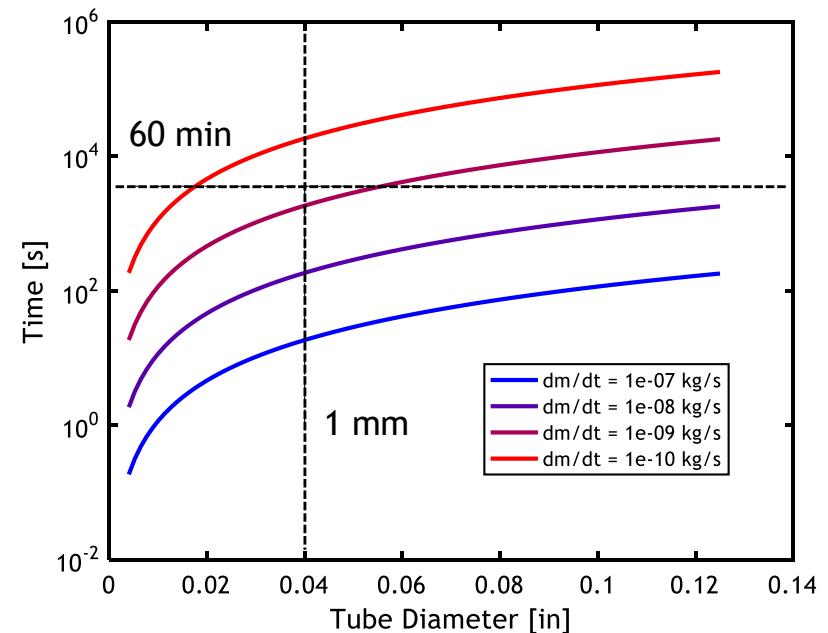
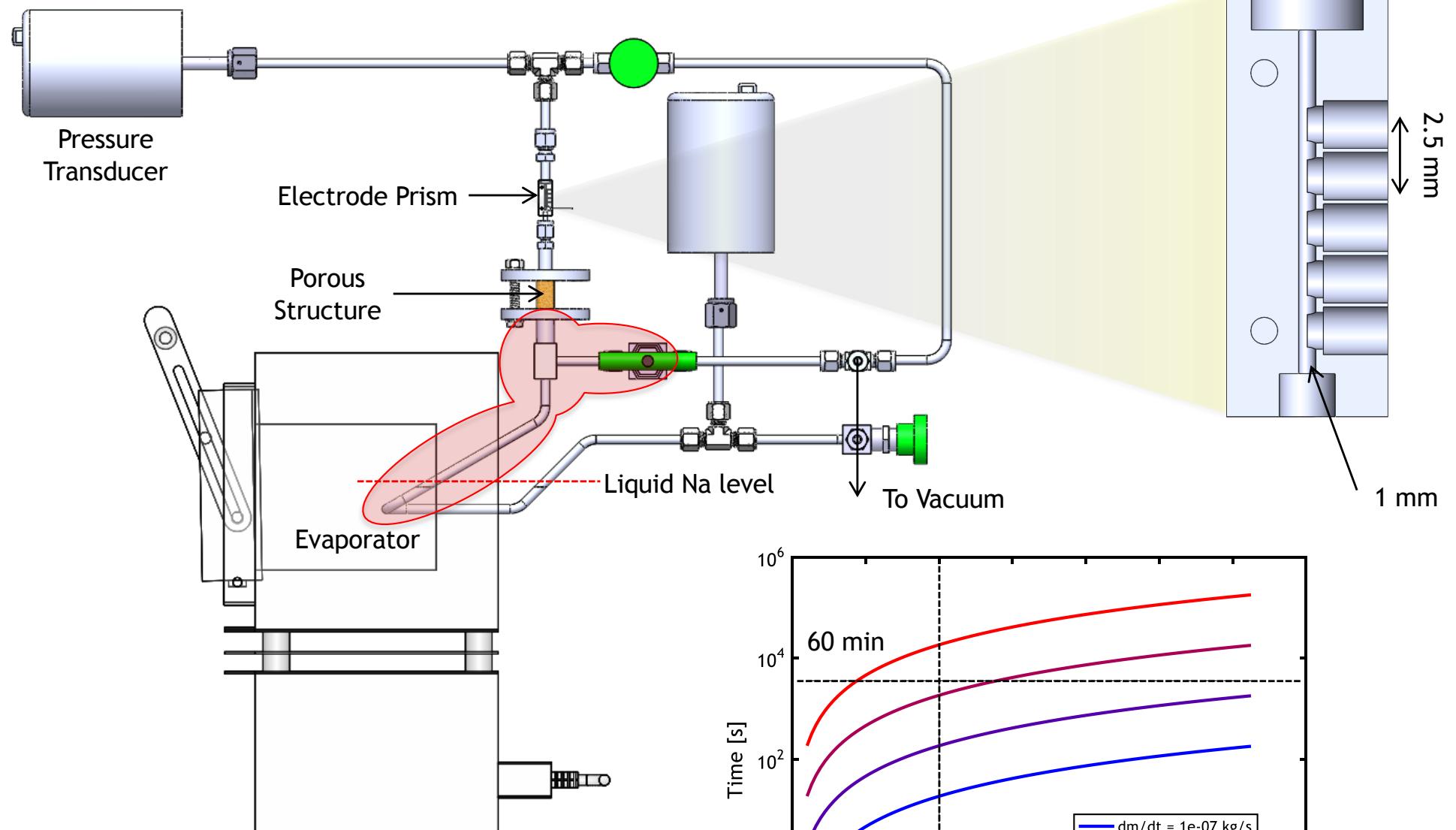


SS 316 thermally pressed inside cylinder



Taylor & Ford, U.K. Atomic Energy Authority Report, 1955

# Experimental Design



# Maximum Mass Flowrate

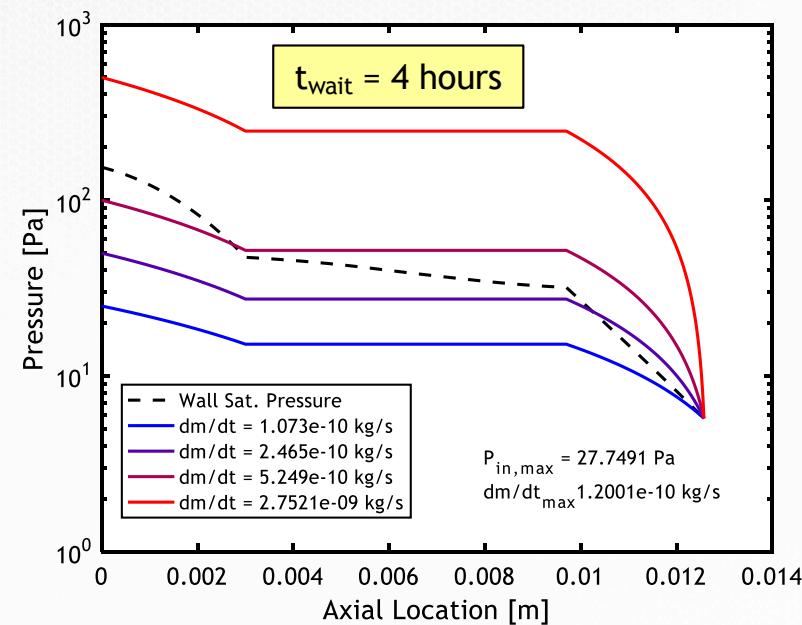
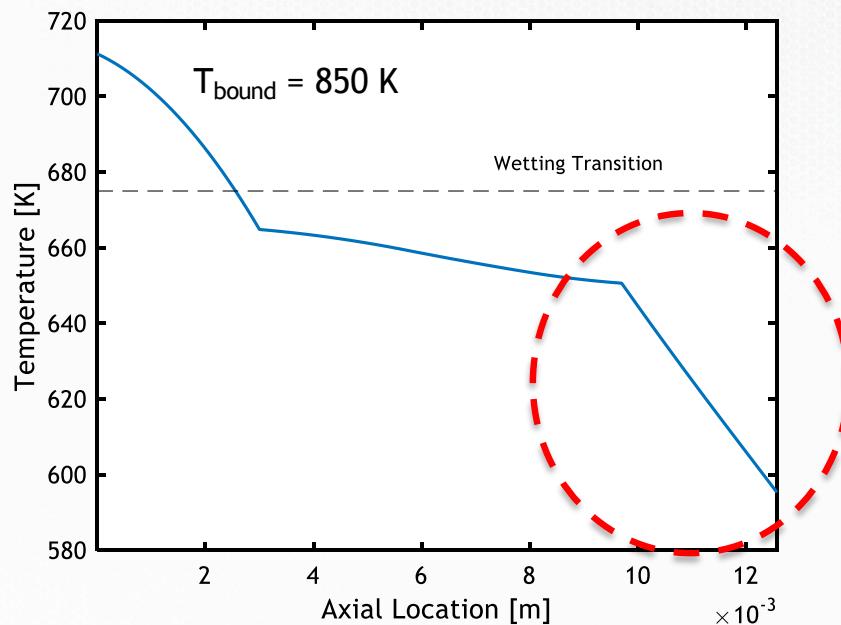
Assume there is no presence of argon, so transport is just with a single species of sodium vapor. Solve the compressible Darcy equation with the following considerations:

Kelvin Equation at interface

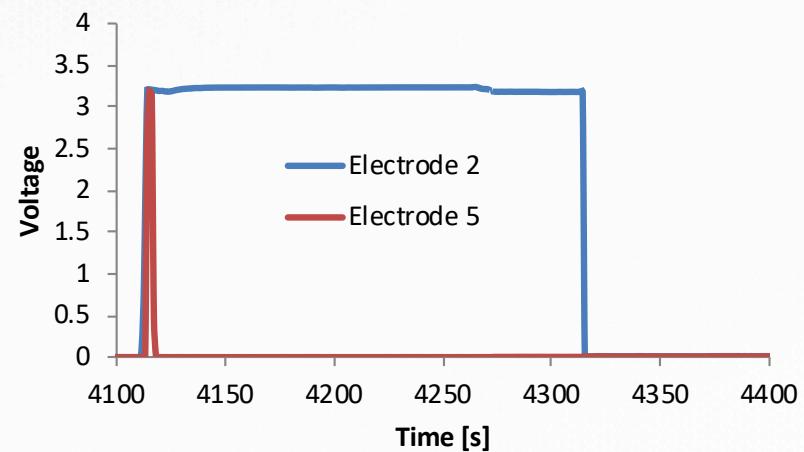
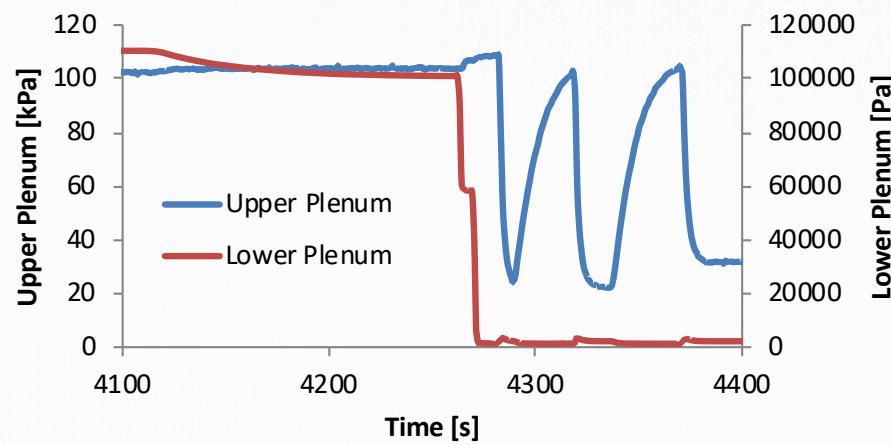
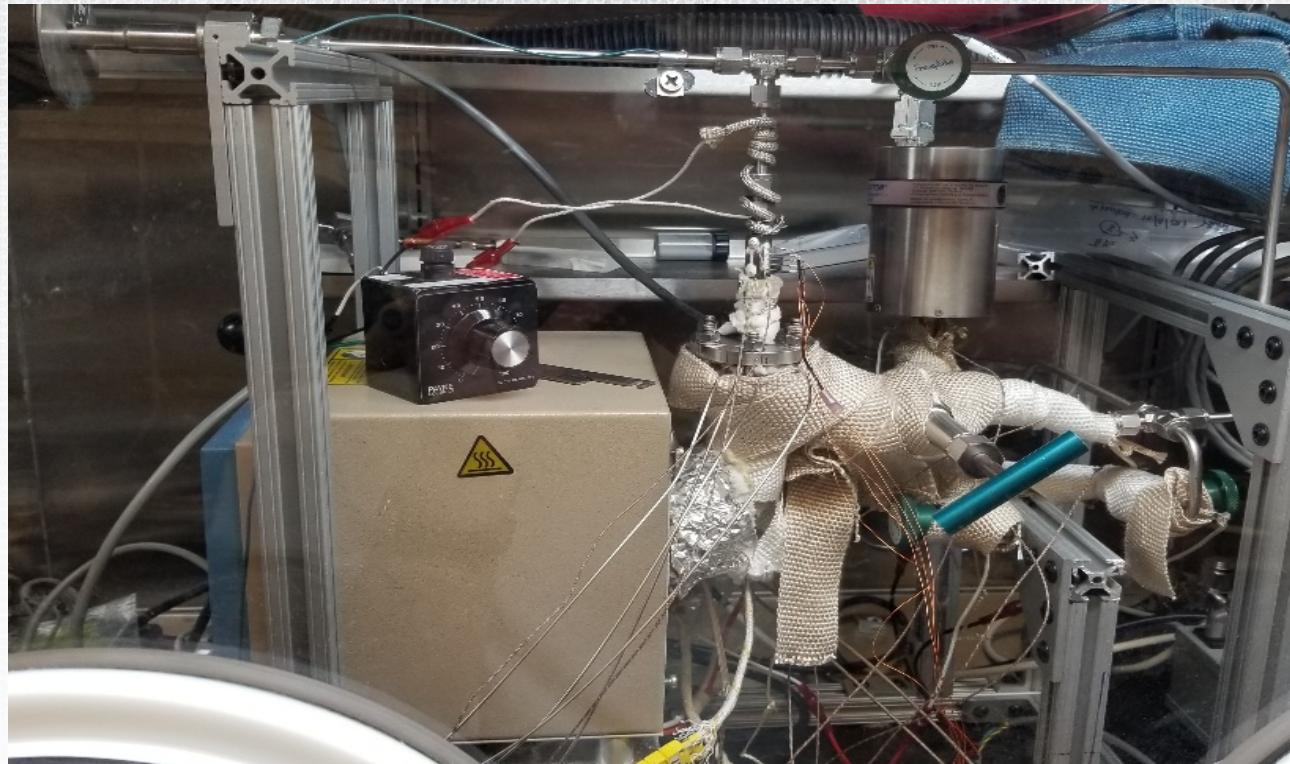
$$P(x_i) = P_{sat}(T_i) \exp\left(\frac{P_L}{\rho_l T_i \bar{R}}\right)$$

Momentum Conservation for slip flow

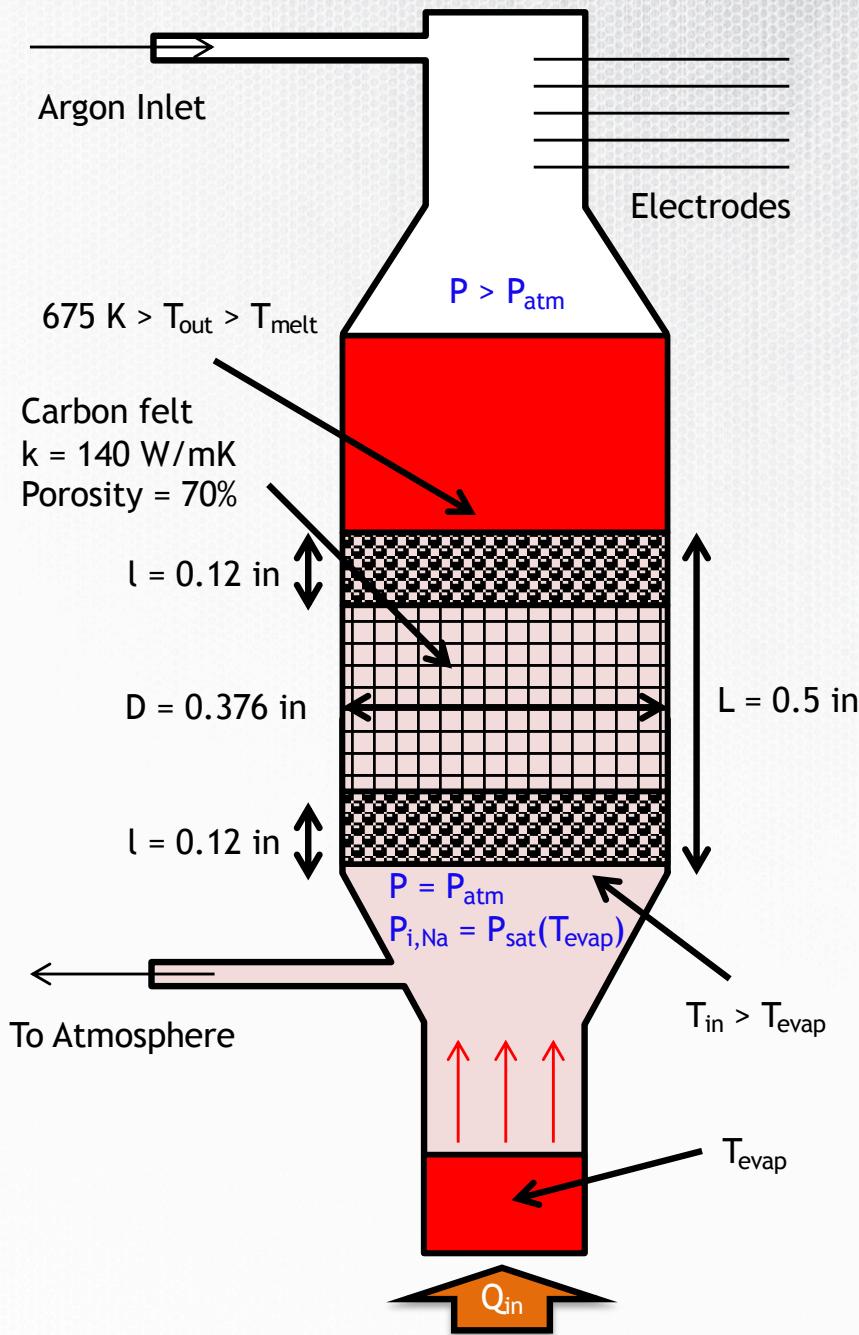
$$u = -\frac{\kappa_\infty (1+8\beta Kn)}{\mu} \frac{dP}{dx}$$



# Failed Vacuum Experiment

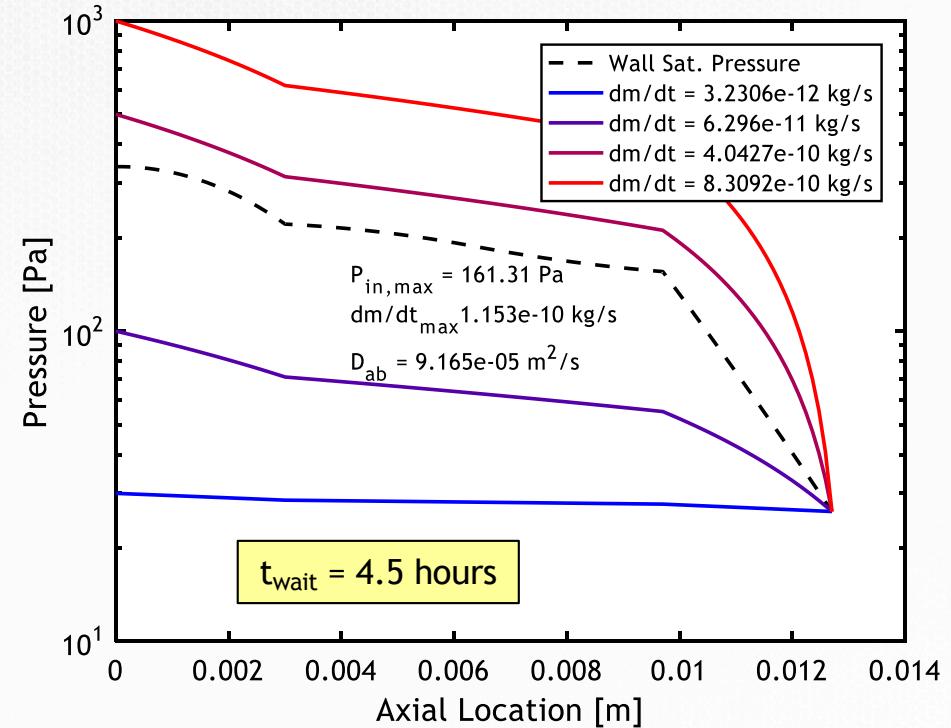


# Gauge Pressure Experiment

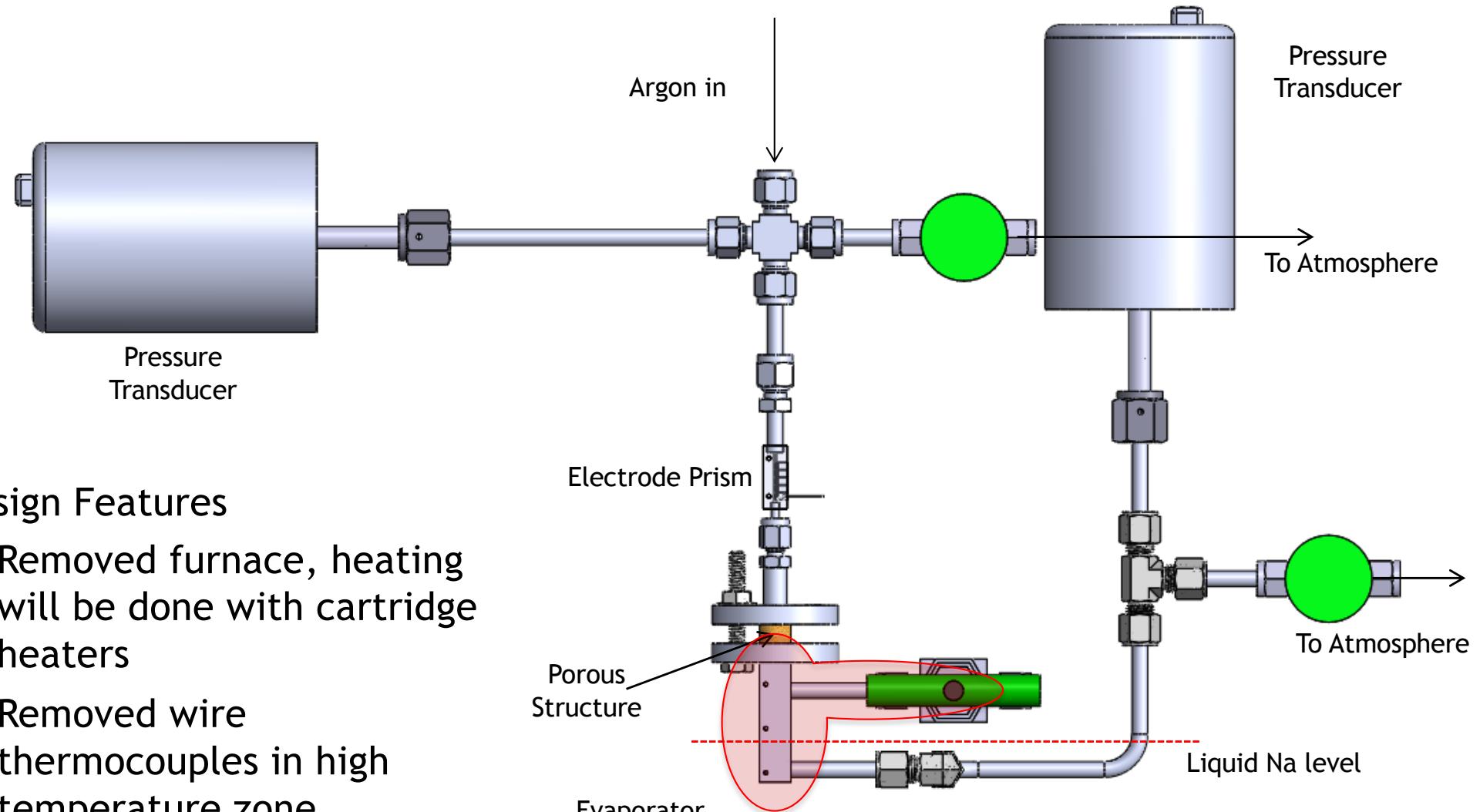


$$u = \frac{\rho_{Na} u_{Na} + \rho_{Ar} u_{Ar}}{\rho} = \omega_{Na} u_{Na}$$

If  $\omega_{Na} \ll 1$   
use dilute gas approximation to  
neglect advection



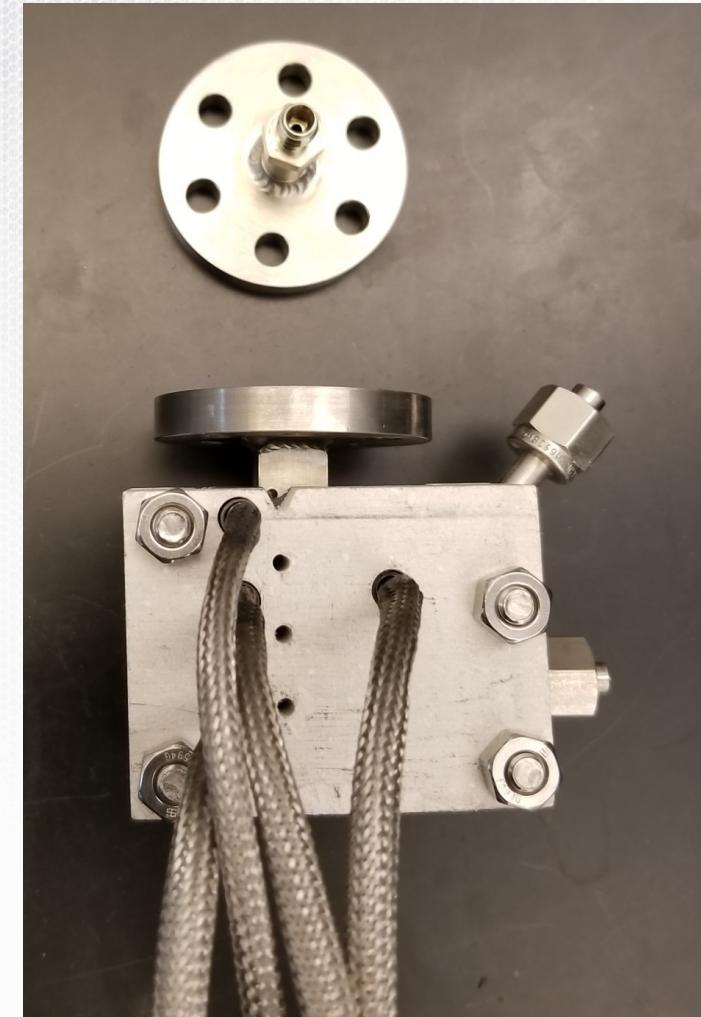
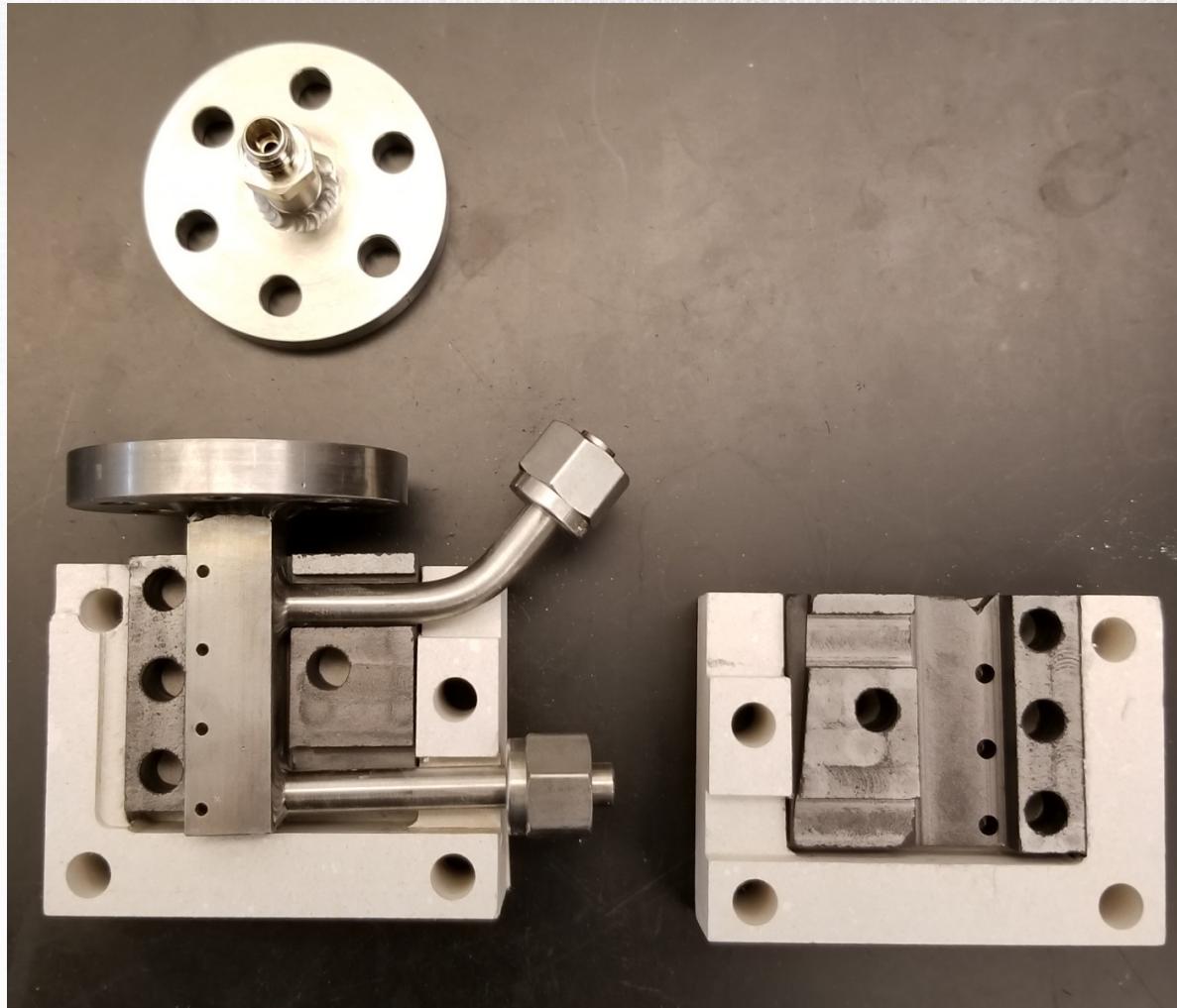
# New Experimental Design



## Design Features

- Removed furnace, heating will be done with cartridge heaters
- Removed wire thermocouples in high temperature zone
- Reduced thermal mass of flanges by 36%

# New Heater Design



# Summary and Future Work

## Summary:

- Discussed transport model for combined advection/diffusion
- Introduced the experimental design used to demonstrate sodium pumping
- Described experimental modification

## Future Work:

- Finish building new experimental set-up. Gather data
- Validate experiment with conjugate heat transfer Comsol modeling
- Use DGM for modeling purposes
- Run experiment with different porous materials

# Acknowledgements

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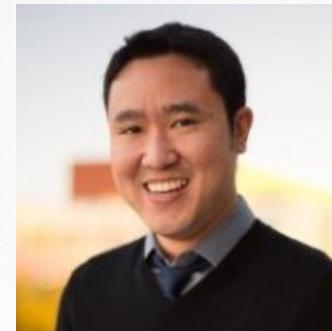


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