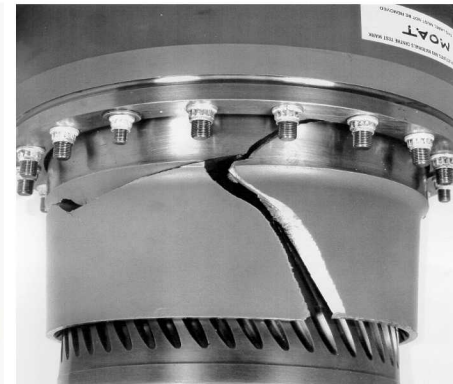
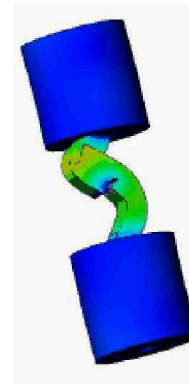


*Exceptional service in the national interest*



**N=O=MAD**



# Predictive Modeling of Bolted Assemblies with Surface Irregularities

**Matthew Fronk**  
*Georgia Institute of  
Technology*

**Gabriela Guerra**  
*New Mexico State  
University*

**Matthew Southwick**  
*University of  
Pittsburgh*

**Robert J. Kuether**  
*Sandia National  
Laboratories*

**Adam Brink**  
*Sandia National  
Laboratories*

**Brendan Nation**  
*Sandia National  
Laboratories*

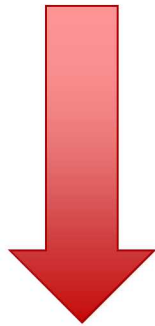
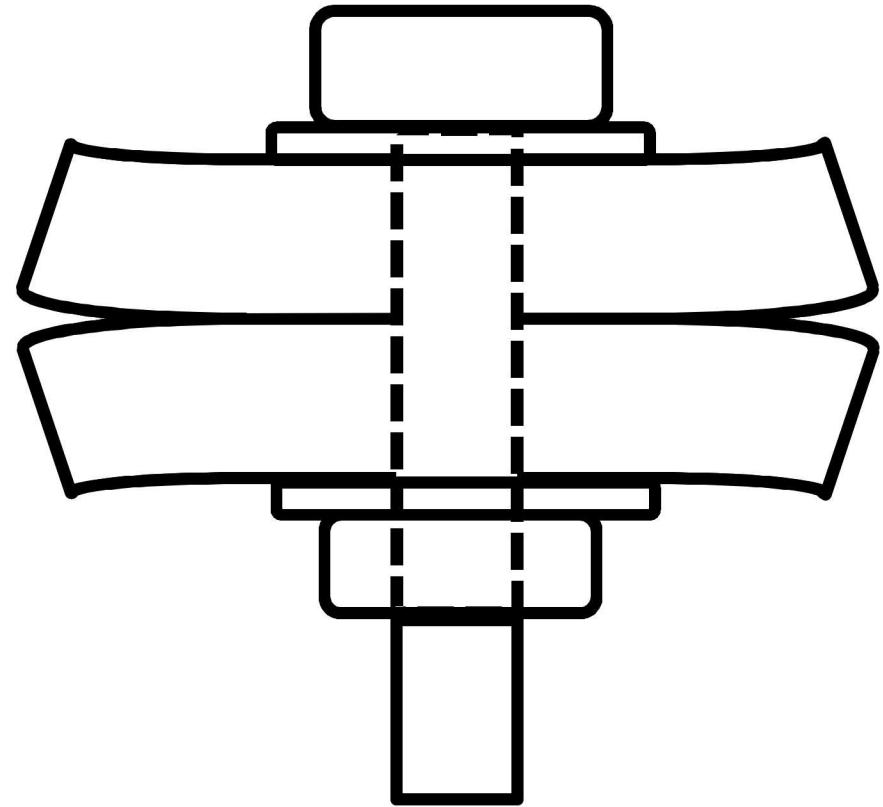
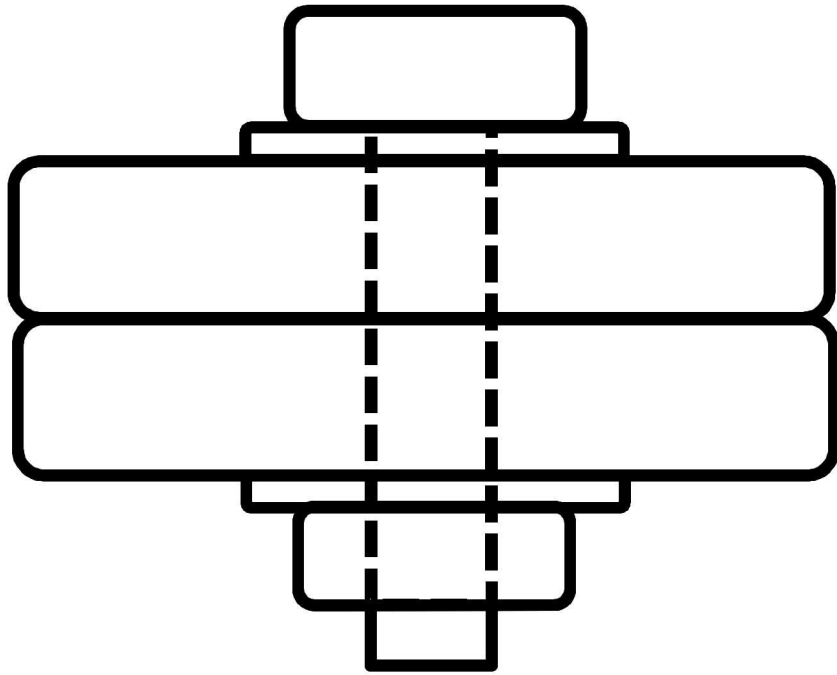
**Paolo Tiso**  
*ETH Zurich*

**Dane Quinn**  
*The University of Akron*



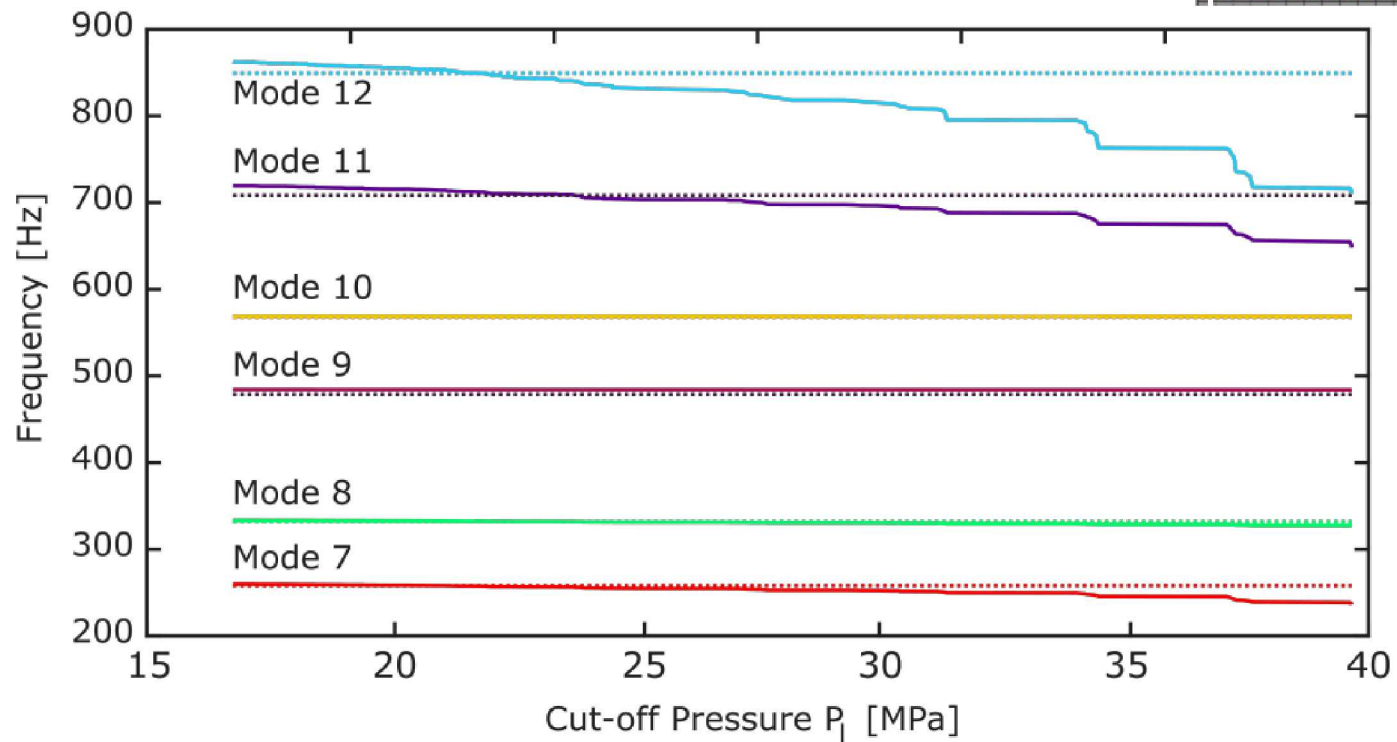
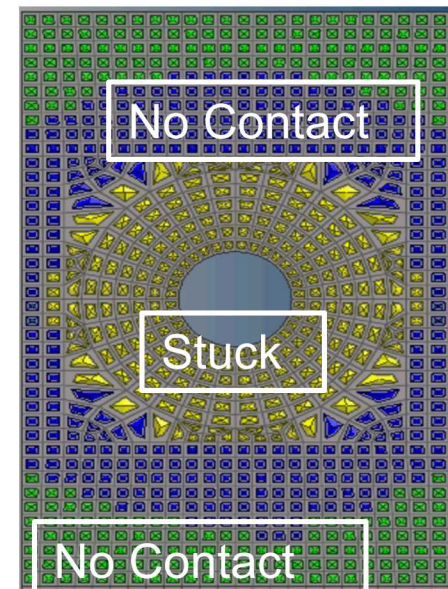
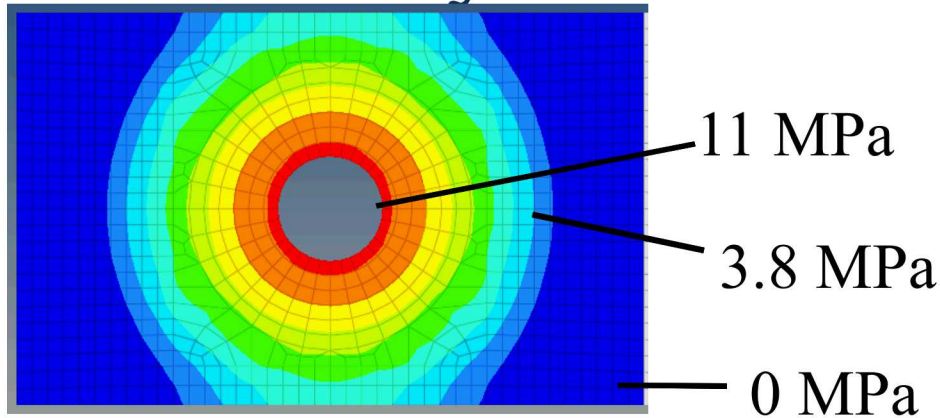
Sandia National Laboratories is a multi-mission laboratory managed and operated by National Technology and Engineering Solutions of Sandia, LLC., a wholly owned subsidiary of Honeywell International, Inc., for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-NA-0003525.

# Introduction



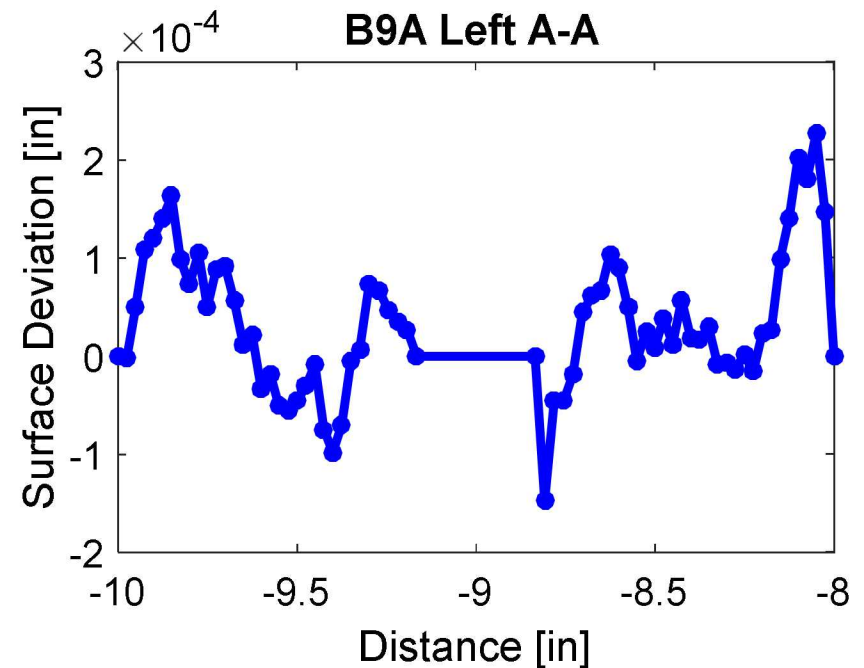
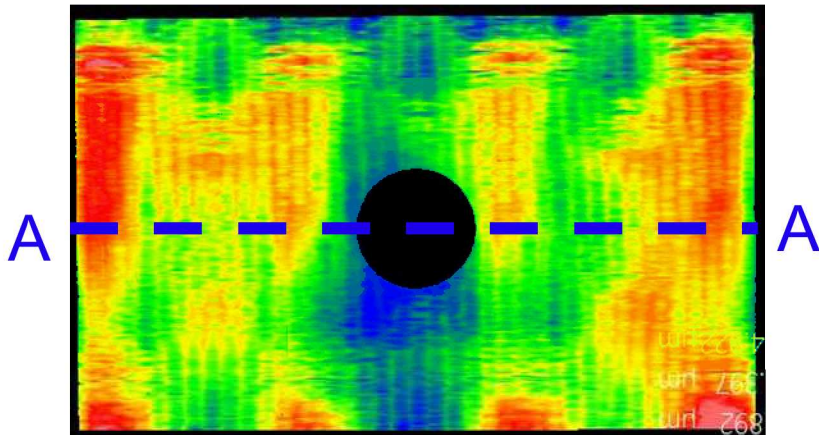
**Preload  $\rightarrow$  Joint stiffness**

# Prior study: NOMAD 2017



# Introduction

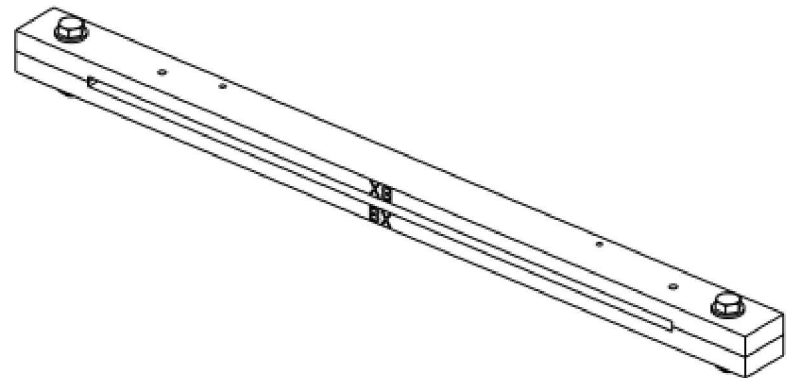
- To what extent do surface irregularities in an interface effect the dynamic properties of a joint?





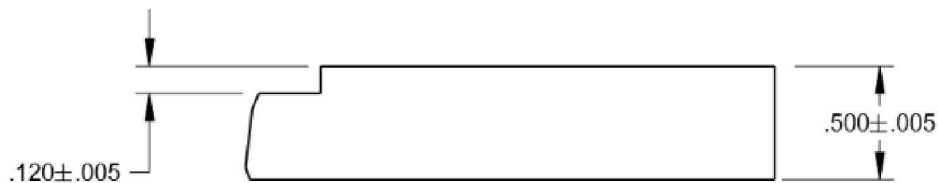
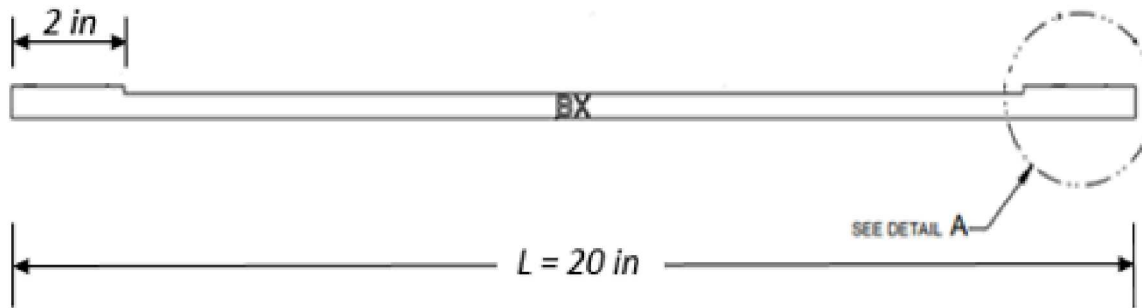
# Agenda

1. System Description
2. Single Beam Structure
  1. Experimental Modal Analysis
  2. Model Updating
    1. Density
    2. Young's Modulus
    3. Surface Geometry
3. Assembled Beam Structure
  1. Experimental Modal Analysis
  2. Computational Analysis
  3. Validation
4. Conclusions

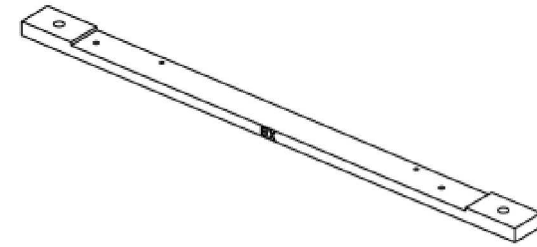


3D ROTATED VIEW  
REFERENCE ONLY  
SCALE 1/2

# System Description

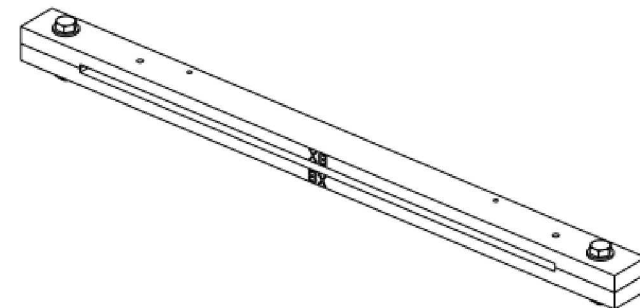


DETAIL A  
SCALE 2/1



3D ROTATED VIEW  
REFERENCE ONLY  
SCALE 3/8

Single beam



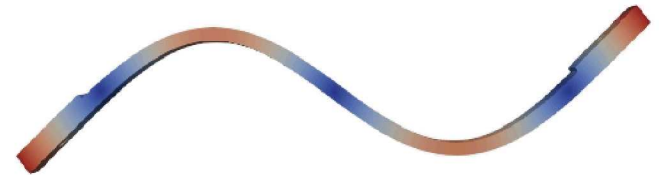
3D ROTATED VIEW  
REFERENCE ONLY  
SCALE 1/2

Beam assembly

# Single Beam Structure: Experimental Modal Analysis



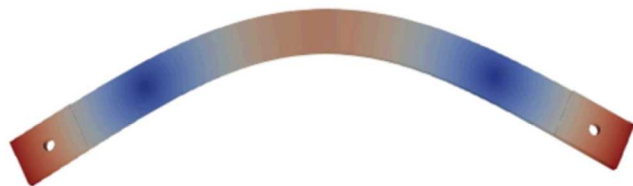
<b>B9A</b>	<b>B9B</b>	<b>FEA</b>
187.5 Hz	187.1 Hz	189.3 Hz



<b>B9A</b>	<b>B9B</b>	<b>FEA</b>
526.3 Hz	525.1 Hz	523.3 Hz



<b>B9A</b>	<b>B9B</b>	<b>FEA</b>
610.8 Hz	610.3 Hz	603.8 Hz



<b>B9A</b>	<b>B9B</b>	<b>FEA</b>
1042.7 Hz	1042.8 Hz	1037.9 Hz



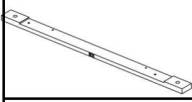
<b>B9A</b>	<b>B9B</b>	<b>FEA</b>
1545.5 Hz	1541.8 Hz	1530.7 Hz

# Single Beam Structure: Model Updating of Density




*Serialize components*


4340 Alloy Steel Beams

	B9A	B9B
Mass (slug)	7.3E-03	7.3E-03
Density (slug/in <sup>3</sup> )	7.3E-04	7.4E-04


Hex Head Screw, .3125-25 UNF-2B

	Bolt 1 (Right)	Bolt 2 (Left)
Mass (slug)	1.0E-04	1.0 E-04
Density (slug/in <sup>3</sup> )	6.7E-04	6.7E-04

Washer .344ID, .688OD, .065 THK

	Washer 1 (Right)	Washer 2 (Right)	Washer 3 (Left)	Washer 4 (Left)
Mass (slug)	1.1E-05	1.2E-05	1.1E-05	1.2E-05
Density (slug/in <sup>3</sup> )	6.3E-04	6.4E-04	6.3E-04	6.4E-04

Nut, Hex, .3125-25 UNF-2B

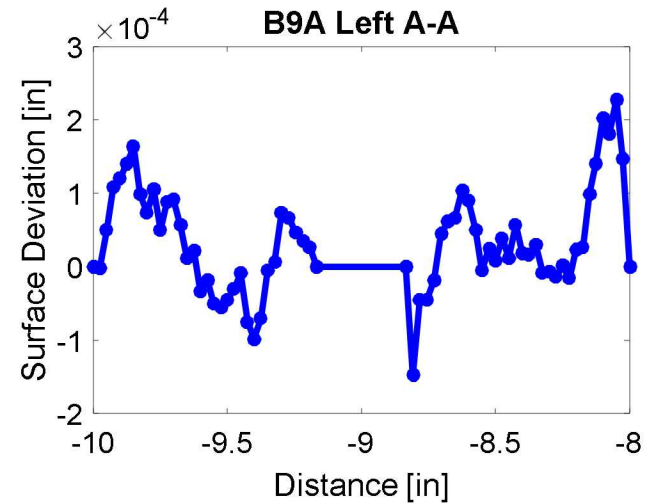
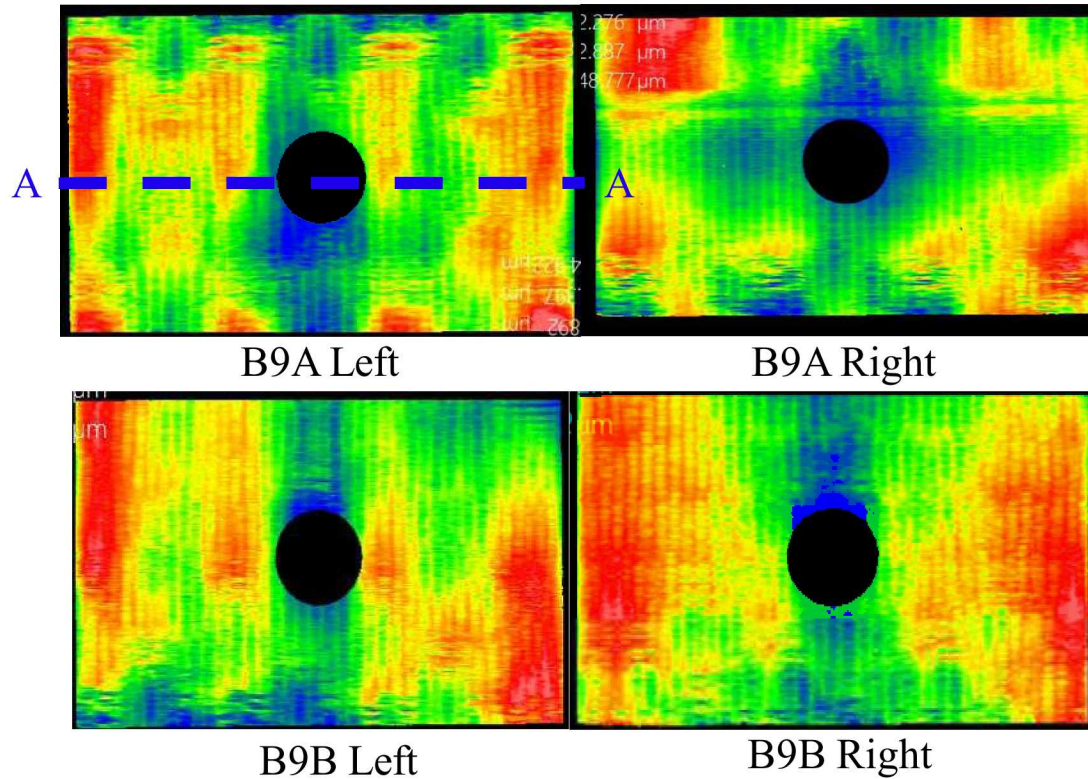
	Nut 1 (Right)	Nut 2 (Left)
Mass (slug)	2.87E-05	2.8E-05
Density (slug/in <sup>3</sup> )	9.1E-04	9.0E-04



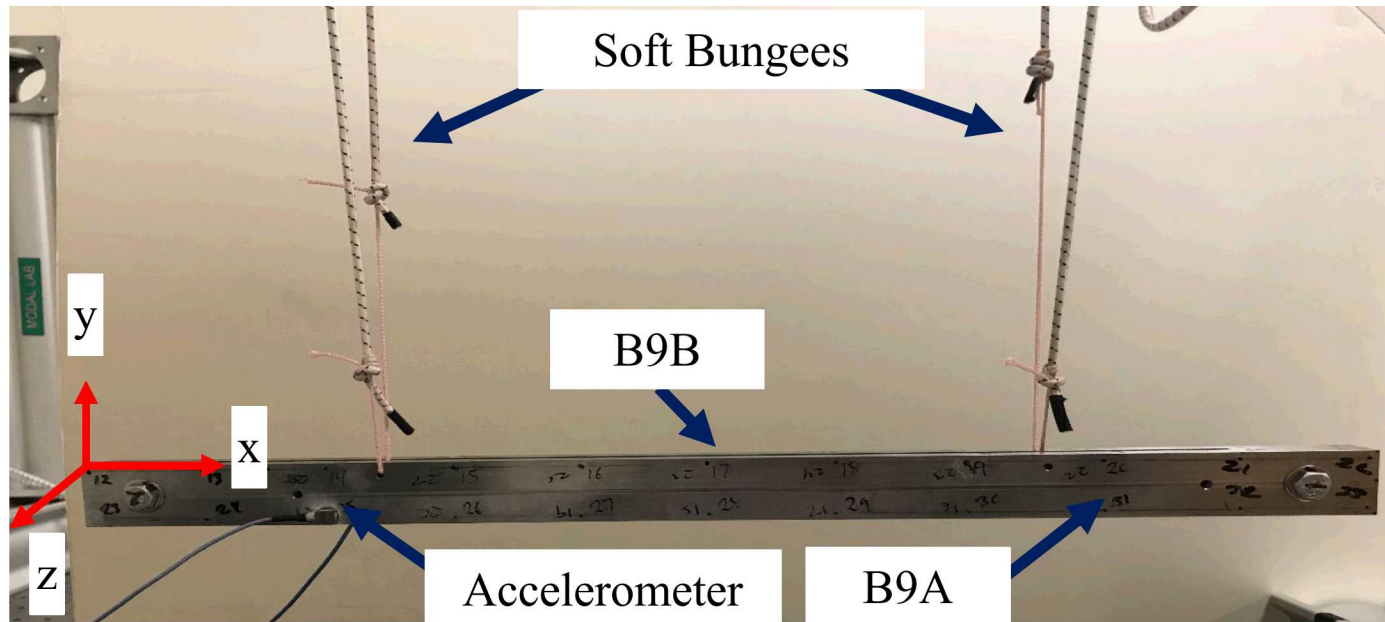
# Single Beam Structure: Model Updating of Young's Modulus

Mode	B9A Natural Frequency [Hz]		B9B Natural Frequency [Hz]	
	Experimental	Young's Modulus Updated FE	Experimental	Young's Modulus Updated FE
<b>1<sup>st</sup> y- bending</b>	187.5	187.7 (+0.1%)	187.1	187.3 (+0.1%)
<b>2<sup>nd</sup> y- bending</b>	526.3	527.4 (+0.2%)	525.1	526.4 (+0.2%)
<b>3<sup>rd</sup> y- bending</b>	610.8	608.1 (-0.4%)	610.2	606.9 (-0.5%)
<b>1<sup>st</sup> z- bending</b>	1042.7	1046.5 (+0.4%)	1042.9	1044.5 (+0.2%)
<b>1<sup>st</sup> torsion</b>	1545.6	1541.8 (-0.2%)	1541.8	1538.8 (-0.2%)

# Single Beam Structure: Model Updating of Surface Geometry



# Beam Assembly: Experimental Modal Analysis

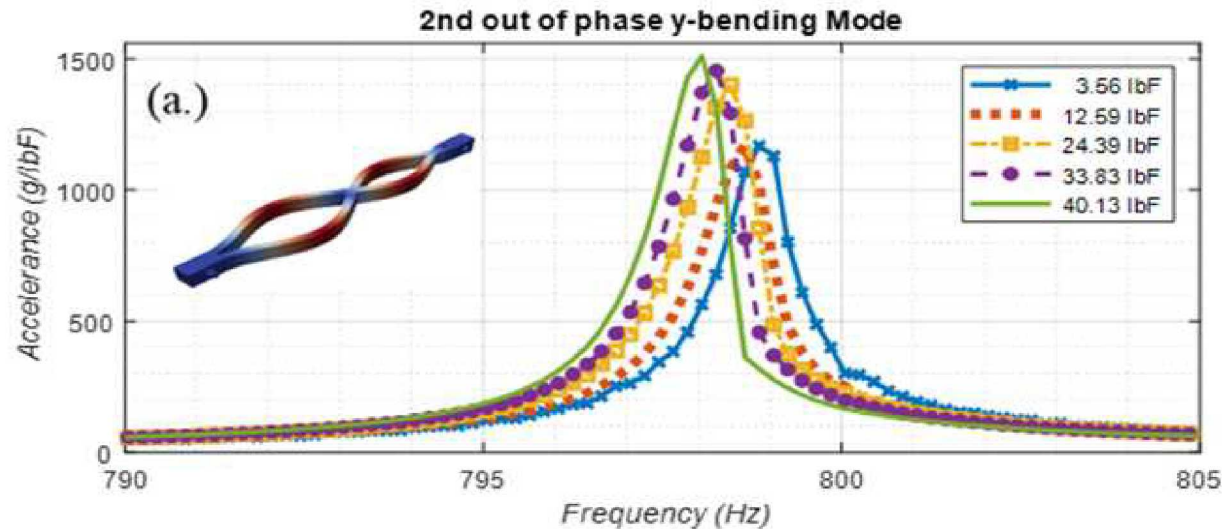


Alignment Tool

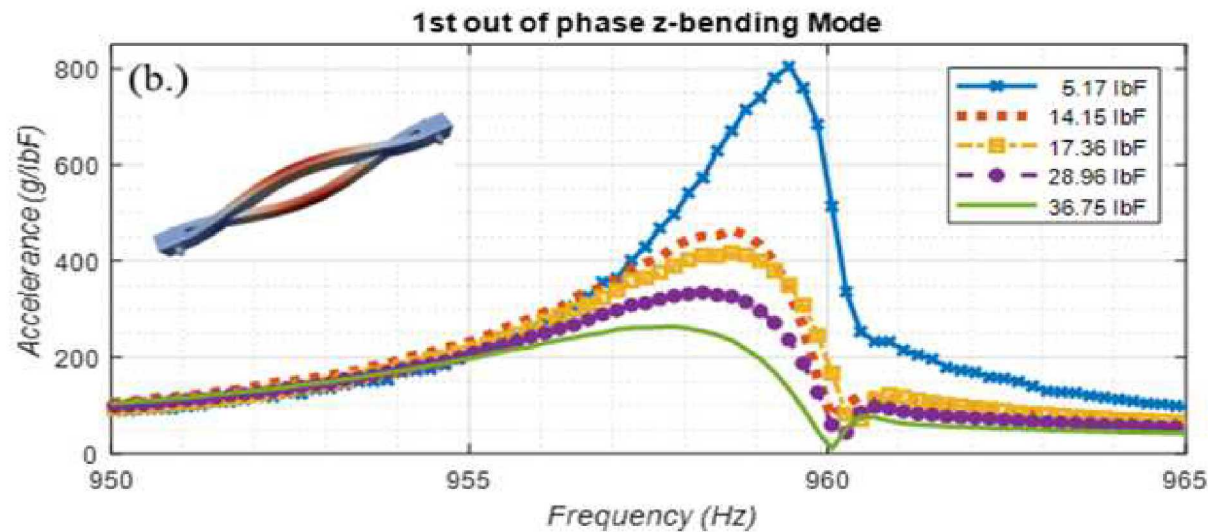


Torque Wrench

# Beam Assembly: Linearity Study



Stiffness decreases

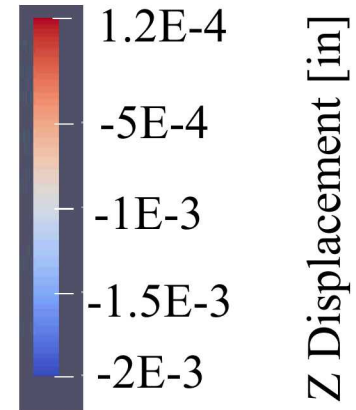


High shearing

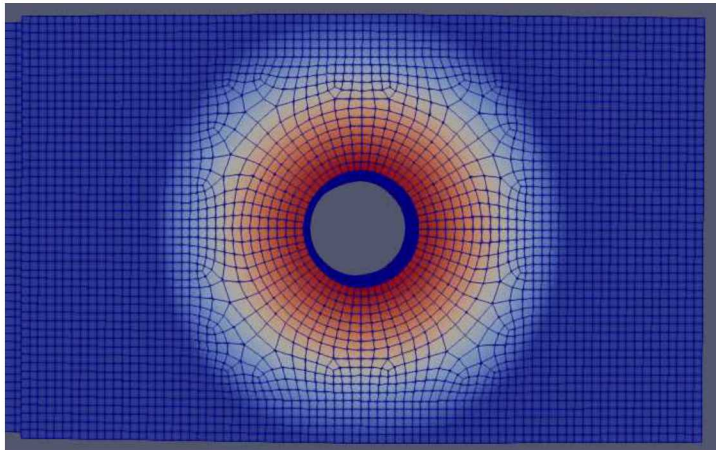


# Beam Assembly: Computational Analysis (Nominally Flat)

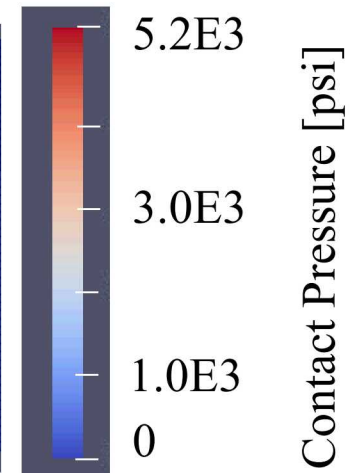
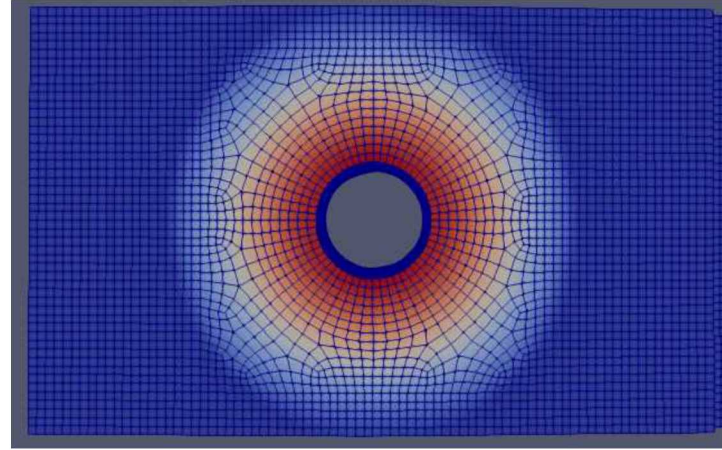
(a.)



(b.)

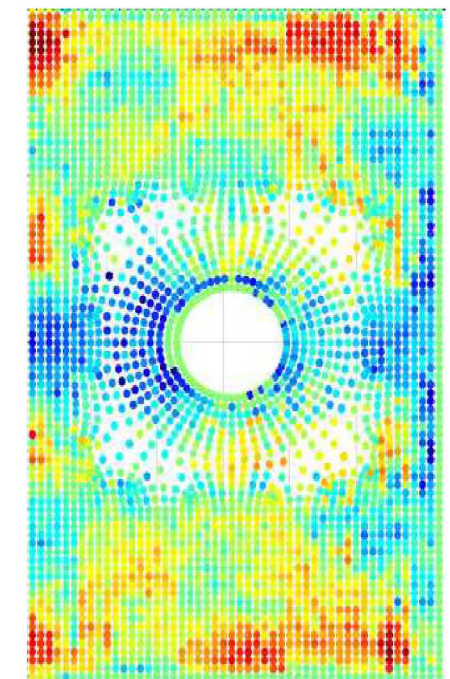


(c.)

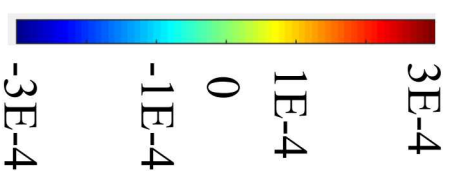
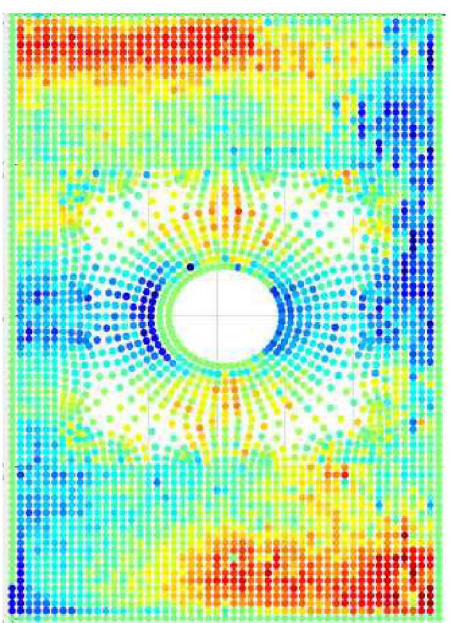




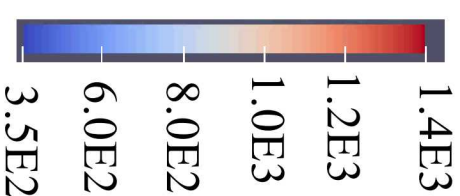
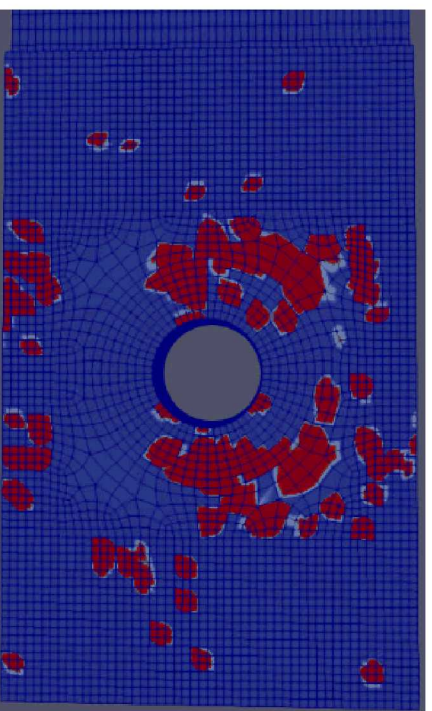
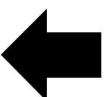
# Beam Assembly: Computational Analysis (Perturbed Mesh)



+

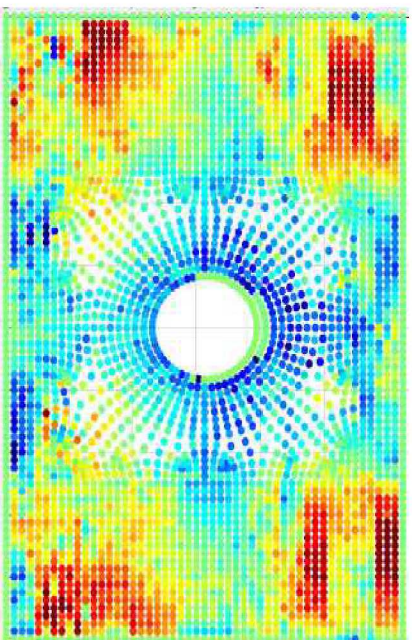


Surface Deviation [in]

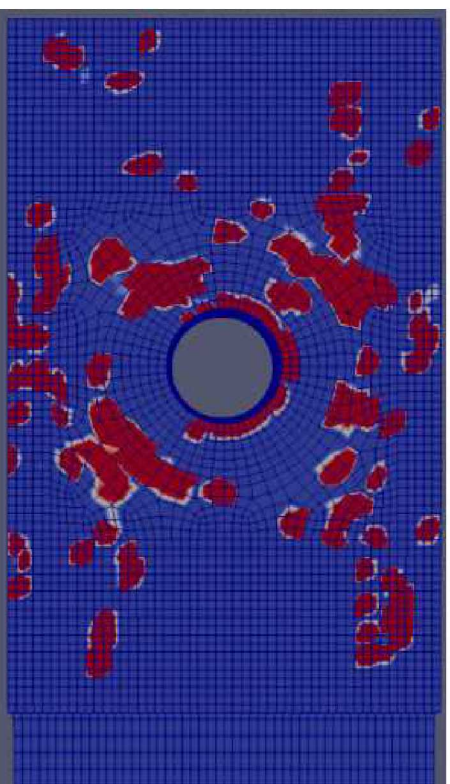
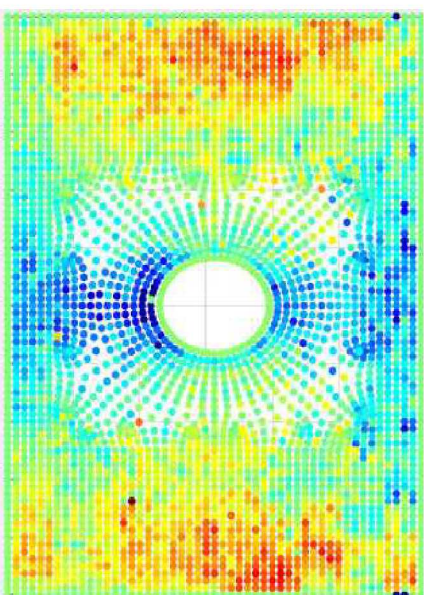


Contact Pressure [psi]

# Beam Assembly: Computational Analysis (Perturbed Mesh)



+



1.4E3  
1.2E3  
1.0E3  
8.0E2  
6.0E2  
3.5E2

Contact Pressure [psi]

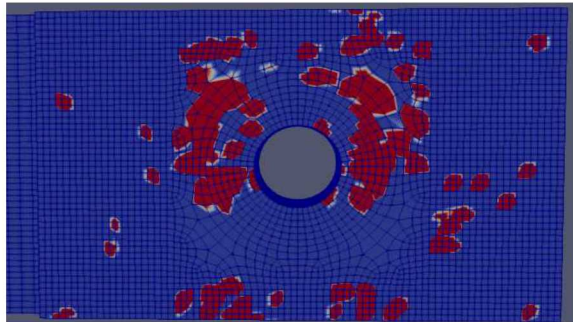


3E-4  
1E-4  
0  
-1E-4  
-3E-4  
Surface Deviation [in]

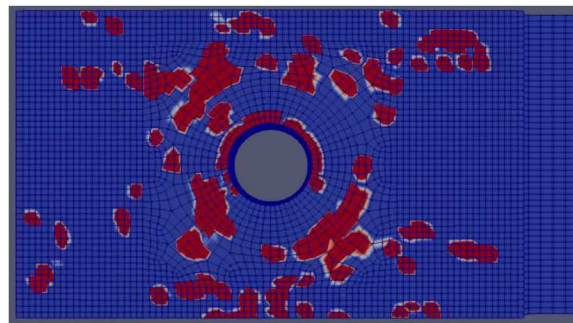


# Beam Assembly: Pressure Film Verification

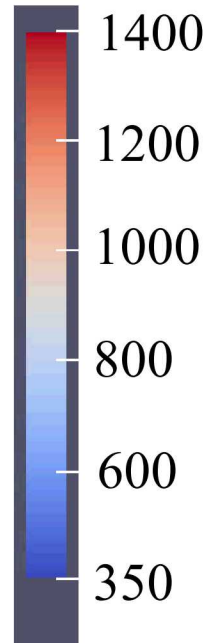
Left



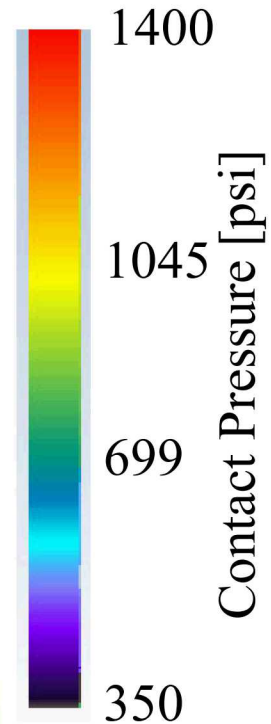
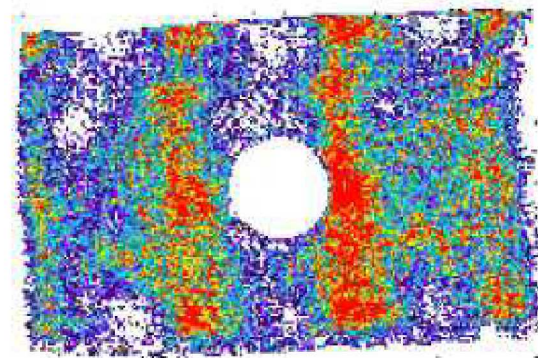
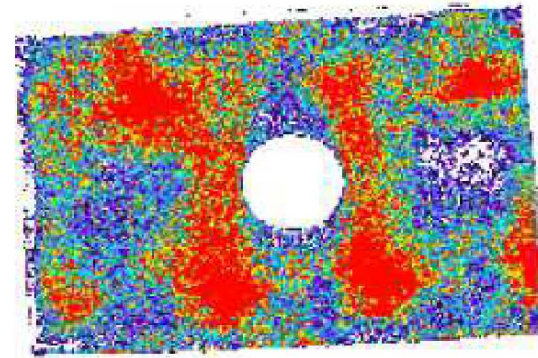
Right



FEA Results



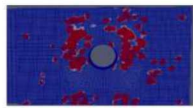
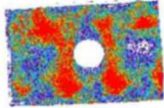


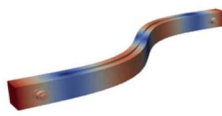
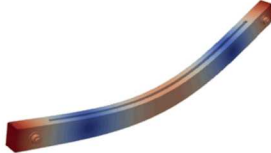


Contact Pressure [psi]



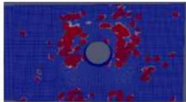
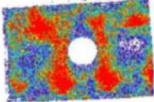


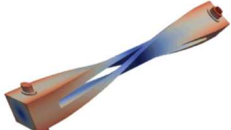
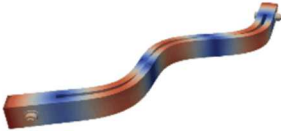


Pressure Film Data

# Validation

	<i>Flat surface, fully stuck [Hz]</i>	<i>Flat surface, preloaded [Hz]</i>	<i>Perturbed surface, preloaded [Hz]</i>	<i>Experiment [Hz]</i>
Mode Shape				
	297.0 (+2.2%)	283.0 (-2.6%)	289.2 (-0.4%)	290.4
	359.2 (+1.5%)	354.5 (+0.2%)	355.5 (+0.5%)	353.8
	513.9 (+1.1%)	513.3 (+1.0%)	513.5 (+1.0%)	508.4
	597.0 (-0.3%)	597.0 (-0.3%)	597.1 (-0.3%)	598.6

# Validation (Cont.)

	<i><b>Flat surface, fully stuck [Hz]</b></i>	<i><b>Flat surface, preloaded [Hz]</b></i>	<i><b>Perturbed surface, preloaded [Hz]</b></i>	<i><b>Experiment [Hz]</b></i>
<b>Mode Shape</b>				
	814.6 (+2.1%)	773.0 (-3.2%)	791.6 (-0.8%)	797.6
	982.9 (+2.8%)	925.9 (-3.2%)	948.7 (-0.7%)	955.6
	1183.7 (-0.2%)	1176.0 (-0.8%)	1178.9 (-0.6%)	1185.6
	1359.4 (+1.1%)	1353.0 (+0.6%)	1353.7 (+0.7%)	1344.4



# Conclusions

- High-fidelity modelling of a benchmark jointed structure was conducted
  - Model updating of density, Young's modulus, surface geometry
- Applying tied MPC's to nodes in contact from a preload analysis identifies the pressure distribution of an interface
  - Qualitative agreement with pressure film data
- Significant improvements in natural frequency predictions occur for out-of-phase bending modes
  - Perturbed surface heights contact each other far from the bolt hole
- Future iterations include:
  - Spatially averaging surface heights, testing more topographies

# Acknowledgments

- This research was conducted at the 2018 Nonlinear Mechanics and Dynamics (NOMAD) Research Institute supported by Sandia National Laboratories.
- Sandia National Laboratories is a multi-mission laboratory managed and operated by National Technology and Engineering Solutions of Sandia, LLC., a wholly owned subsidiary of Honeywell International, Inc., for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-NA-0003525.

# Acknowledgments (Cont.)

- The authors would also like to thank Bill Flynn from Siemens Industry Software NV for supplying the data acquisition and testing systems used to collect the experimental measurements presented throughout this work. SAND2018-11908 C