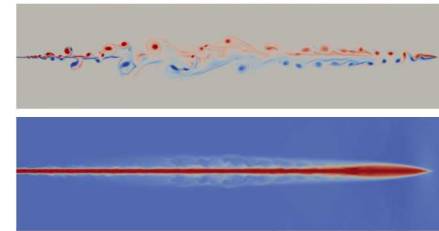
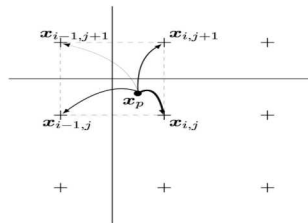
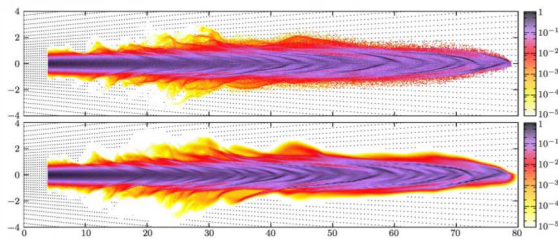


Modeling and simulation of a dense evaporating spray

F. Doisneau, M. Arienti, J. Oefelein

**Combustion Research Facility
Sandia National Laboratories, Livermore, CA 94551**



**Support by Sandia National Laboratories' LDRD program
LDRD – Laboratory Directed Research and Development
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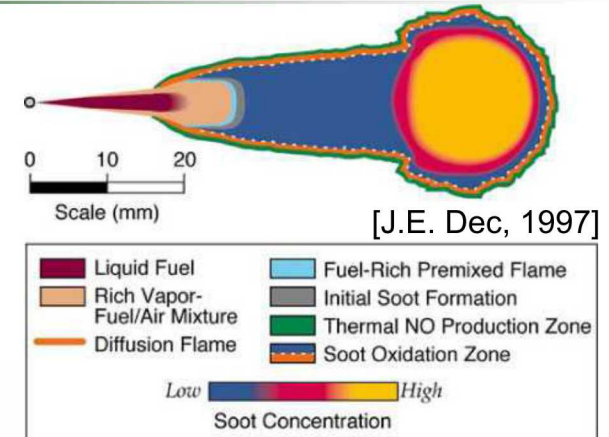
Martin Corporation, for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-AC04-94-OR21400.

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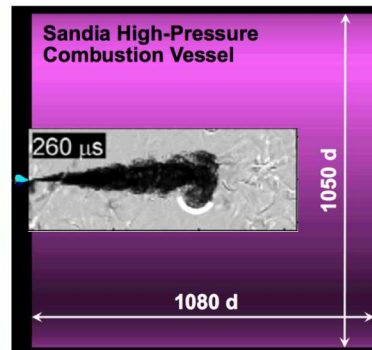
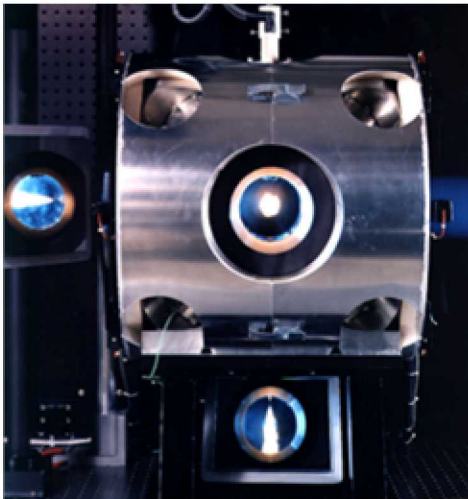
MOTIVATION – Liquid fuel injection

In Diesel and Gasoline engines

- Inlet is turbulent (+ cavitation)
 - $Re \sim 10^5$, $d = 90\mu m$
- High pressure chamber and sonic flow
 - $p = 60bar$, $u_i = 600m/s$
- Atomization process not understood
 - $We \sim 10^4$, $1\mu m < r_l < 100\mu m$



➔ **MULTI-SCALE&MULTI-PHYSICS** drive **MIXING&COMBUSTION**



Experimental background
High pressure vessels
[Pickett 2010, Skeen 2014]

Need for a
High Fidelity Simulation
that is **affordable**

STATE OF THE ART – Injection simulation

No comprehensive simulation approaches

1 DNS with Interface Capturing

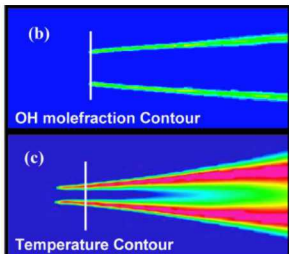
[Menard 2007; Desjardins 2010]

- Accurate and insightful
- Intractably costly

2 Filtered Interface Capturing

[Chesnel 2011]

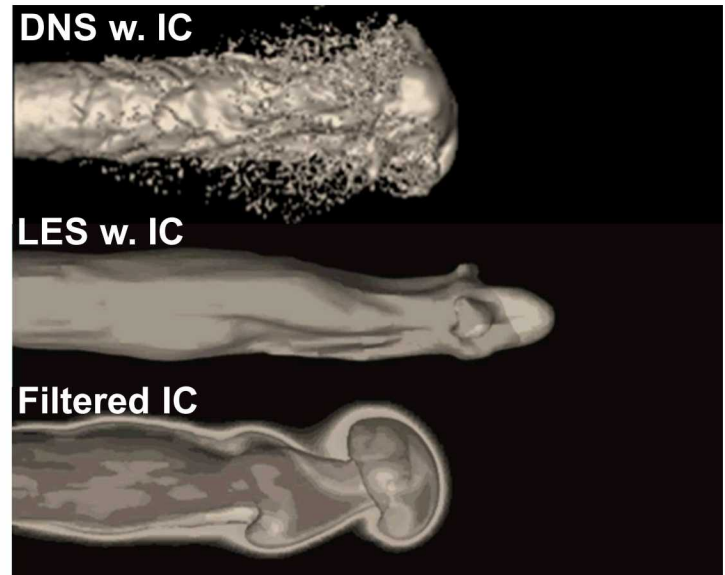
- Promising but empirical



3 Extension of dilute spray with coarse AMR

[Som 2013]

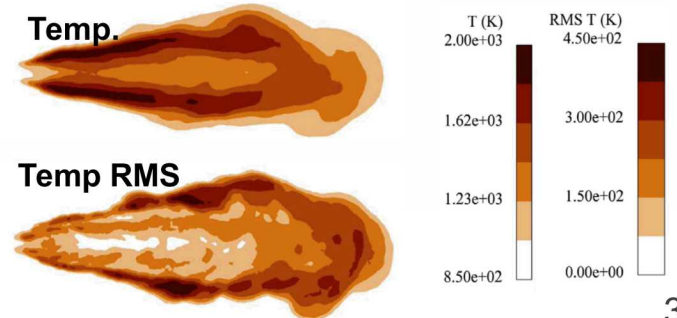
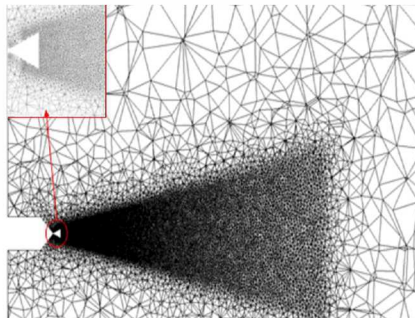
- Essential features missing
- Useful to investigate other physics (e.g. complex chemistry)



1 Prescribed downstream boundary condition

[Tillou 2014]

- Most applicable approach today
- Low predictability



OBJECTIVE – Comprehensive simulation

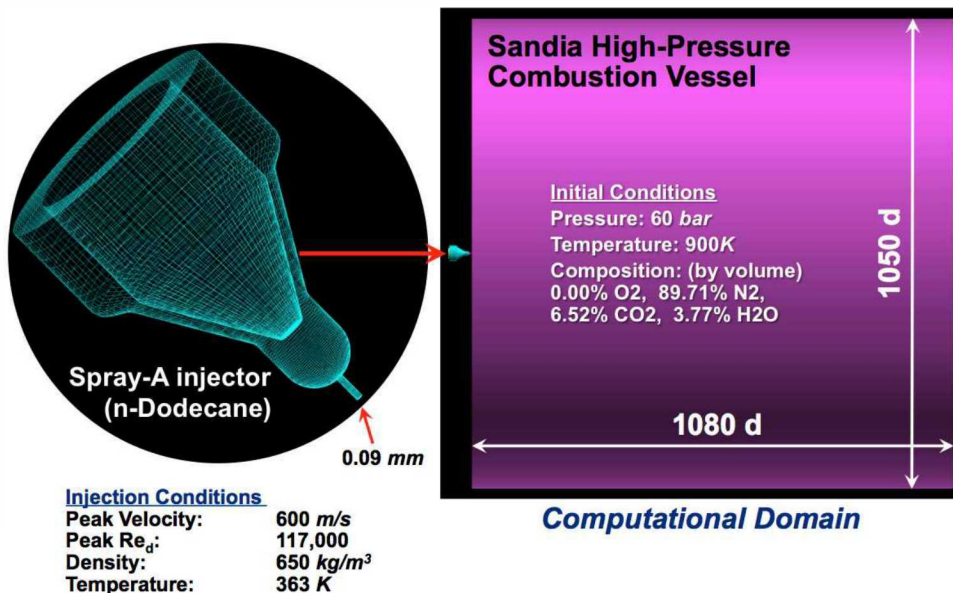
From nozzle outlet to dilute spray

- **Sensitized to nozzle flow** (in an LES sense)
 - $Re \sim 10^5$, $d = 90\mu m$
- **Robust to high pressures, velocities, and loadings**
 - $p = 60\text{bar}$, $u_i = 600\text{m/s}$
- **Compute the whole chamber** (with combustion)
 - with a billion points

$O(1\mu m, 10\text{ns})$



$O(10^5\mu m, 10^6\text{ns})$



We introduce a
Simplified Approach
 of interface flows
 to describe
 more physical scales

MODEL – Two-phase approach

A simplified but promising approach

■ Coupled Eulerian-Eulerian (gas and liquid moments)

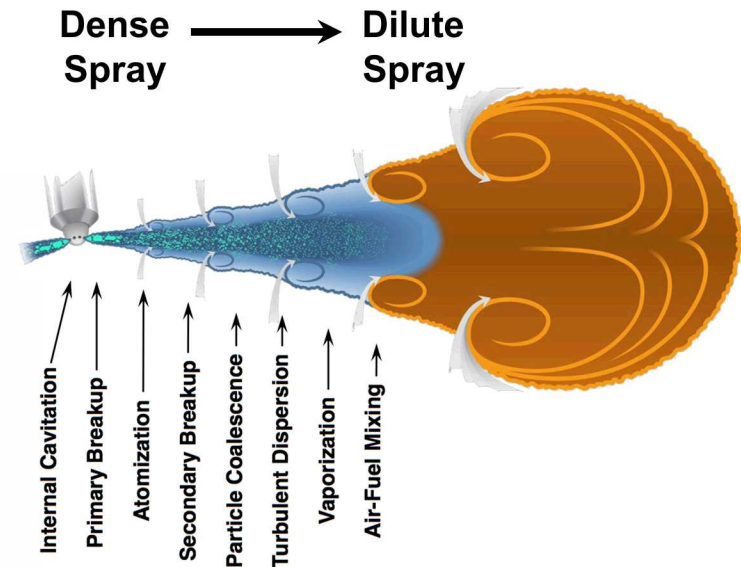
can **emulate at once**

- the inertial behavior of the dense liquid **core**
- the break-up and dispersion of liquid **blobs** (prescribes size of droplets)
- the dilute spray regime with **droplets**

■ Conservation for both **light** and **dense** phases

- no interface tracking
- mesoscale-gradients handled more easily
- as opposed to real gas approaches
- but no built-in thermodynamics!

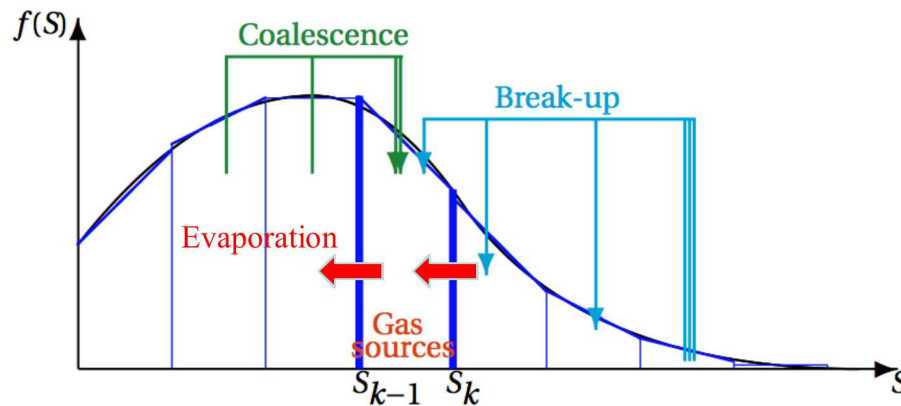
■ ...provided the transfers = need for **closures**!



MODEL – Sectional method

A cost-efficient way to capture polydispersity

- Various drop sizes are treated as a continuum:
Multi-Fluid [Laurent 2001, Doisneau 2013]



$$\begin{aligned}
 N_{\text{sec}} \text{ systems } \left\{ \begin{aligned}
 \partial_t n_k + \partial_x \cdot (n_k \mathbf{u}_k) &= {}^2C_k^n + {}^2B_k^n + {}^2E_k^n \\
 \partial_t m_k + \partial_x \cdot (m_k \mathbf{u}_k) &= {}^2C_k^m + {}^2B_k^m + {}^2E_k^m \\
 \partial_t (m_k \mathbf{u}_k) + \partial_x \cdot (m_k \mathbf{u}_k \otimes \mathbf{u}_k) &= m_k \mathbf{F}_k + {}^2C_k^u + {}^2B_k^u + {}^2E_k^u \\
 \partial_t (m_k h_k) + \partial_x \cdot (m_k h_k \mathbf{u}_k) &= m_k \mathbf{H}_k + {}^2C_k^h + {}^2B_k^h + {}^2E_k^h
 \end{aligned} \right. \quad \Longleftrightarrow \quad \text{Navier-Stokes with sources}
 \end{aligned}$$

...many integral source terms to compute

MODEL – Two-phase approach

Pressureless Gas Dynamics (PGD) decouples Lagrangian advection

- The coupled NS-PGD* system

$$\left\{ \begin{array}{l}
 \partial_t \rho_g Y_f + \partial_x \rho_g Y_f \mathbf{u}_g = \omega_f + \sum_k E_k^{m-g} \\
 \partial_t \rho_g Y_i + \partial_x \rho_g Y_i \mathbf{u}_g = \omega_i, \quad i \in [1; N_{\text{species}}], i \neq f \\
 \partial_t \rho_g \mathbf{u}_g + \partial_x \rho_g \mathbf{u}_g \otimes \mathbf{u}_g = -\partial_x p + \sum_k (-\mathbf{F}_k + \mathbf{u}_k E_k^{m-g}) \\
 \partial_t \rho_g e_g + \partial_x \rho_g e_g \mathbf{u}_g = -p \partial_x \mathbf{u}_g + \sum_k (-H_k + \mathbf{F}_k (\mathbf{u}_g - \mathbf{u}_k) + h_k E_k^{m-g}) \\
 \partial_t m_k + \partial_x m_k \mathbf{u}_k = E_{k+1}^m + B_k^{m+} + C_k^{m+} - (E_k^m + E_k^{m-g} + B_k^{m-} + C_k^{m-}) \\
 \partial_t m_k \mathbf{u}_k + \partial_x m_k \mathbf{u}_k \otimes \mathbf{u}_k = \mathbf{F}_k + \mathbf{u}_{k+1} E_{k+1}^m + \mathbf{B}_k^{u+} + \mathbf{C}_k^{u+} - \mathbf{u}_k (E_k^m + E_k^{m-g} + B_k^{m-} + C_k^{m-}) \\
 \partial_t m_k h_k + \partial_x m_k h_k \mathbf{u}_k = H_k + h_{k+1} E_{k+1}^m + B_k^{h+} + C_k^{h+} - h_k (E_k^m + E_k^{m-g} + B_k^{m-} + C_k^{m-})
 \end{array} \right\} \quad k \in [1; N_{\text{sec}}]$$

pressureless
sections

Needs to be closed

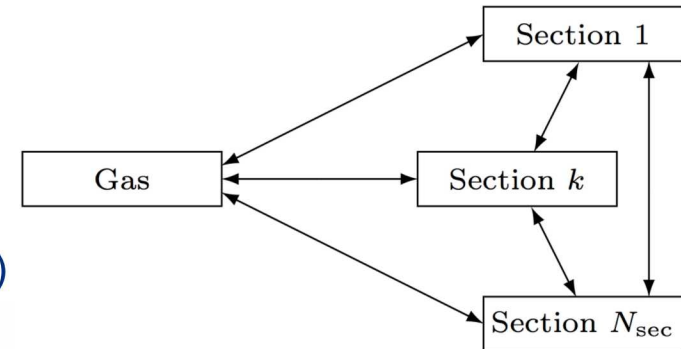
*obtained from kinetic theory or conservation principles

$$\partial_t f + \partial_x \mathbf{c} f + \partial_c \mathbf{F} f + \partial_\theta H f + \partial_r E f = B + C$$

MODEL – The closure problem

LES closures should respect dominant dynamics and equilibrium

- **Need the rate of exchange for**
 - **Momentum** (drag and dynamic subgrid model)
 - **Heat** (heating)
 - **Mass** (vaporization and combustion subgrid model)
- **Derivation from first principles is hard**
 - **All thermophysical properties needed**
 - **Subgrid knowledge needed too** (turbulent & atomizing)
- **Should enforce thermodynamics and equilibria**
 - **Multifluid models** by Saurel's team [D. Furfaro, 2015]



ANALYSIS – Two-way coupling

Two-way coupling as the main driver of characteristic times

- Two-way coupling drives **characteristic times**

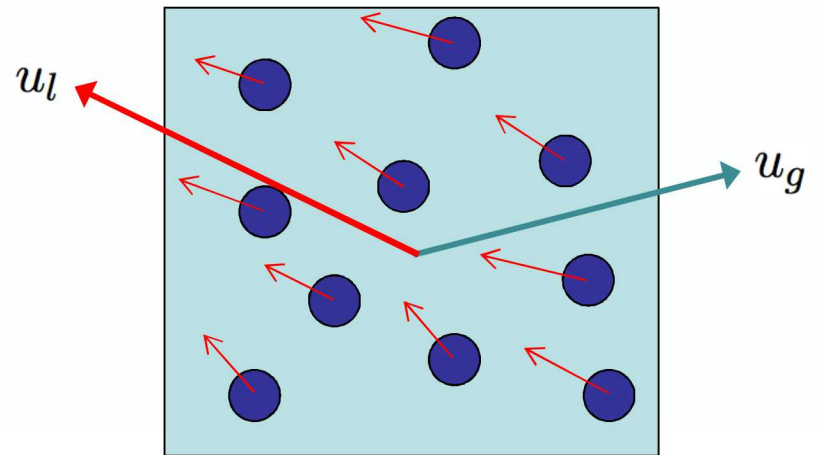
$$C = \frac{m_l}{m_g}$$

– **Example for drag (0D):** [Doisneau, 2013]

$$\begin{cases} \partial_t(m_g u_g) = \frac{m_l}{\tau} (u_l - u_g) \\ \partial_t(m_l u_l) = -\frac{m_l}{\tau} (u_l - u_g) \end{cases} \Leftrightarrow \begin{cases} \partial_t(m_g u_g + m_l u_l) = 0 \\ \partial_t(u_l - u_g) = -\frac{1+C}{\tau} (u_l - u_g) \end{cases}$$

- Correction factor can be large since C reaches ~ 100

$$\tau^{2\text{-way}} = \frac{\tau}{1+C}$$



ANALYSIS – Two-way coupling

Two-way coupling as the main driver of characteristic times

- Two-way coupling drives **characteristic times**

$$C = \frac{m_l}{m_g}$$

- Drag
- Same argument for heating
- Vaporization is driven by heating

$$\partial_t d^2 = -K$$

$$K = \frac{8\lambda_g}{\rho_l c_{p,g}} \log \left(1 + \frac{c_{p,g}(T_g - T_l)}{h_l} \right)$$

- Correction factor can be large
since C reaches ~ 100

$$\tau^{2\text{-way}} = \frac{\tau}{1 + C}$$

NUMERICS – Two-way coupling needs

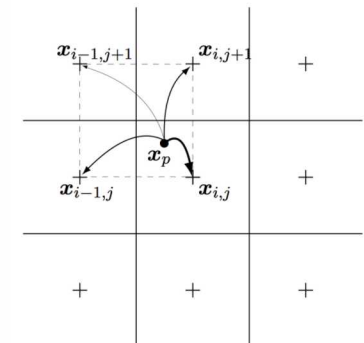
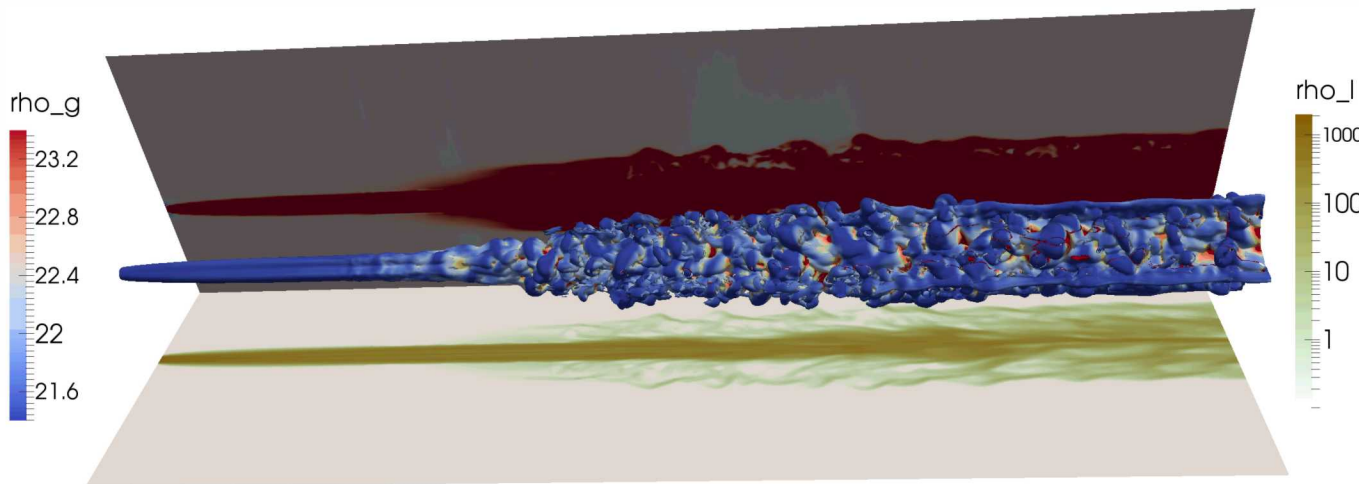
Effort on numerical methods for multi-scale coupled flows

- 1) Time integration tailored **splitting**

$$\begin{array}{|c|} \hline \text{Gas transport } \mathcal{T}_g \\ \hline \end{array} = \begin{array}{|c|} \hline \text{Coupling } \mathcal{R} \\ \mathcal{F} + \mathcal{H} + \mathcal{E} \\ \hline \end{array} + \begin{array}{|c|} \hline \text{Spray sources} \\ \mathcal{B} + \mathcal{C} \\ \hline \end{array}$$

$$\begin{array}{|c|} \hline \text{Section transport } \mathcal{T}_k \\ \hline \end{array} = \begin{array}{|c|} \hline \text{Coupling } \mathcal{R} \\ \mathcal{F} + \mathcal{H} + \mathcal{E} \\ \hline \end{array} + \begin{array}{|c|} \hline \text{Spray sources} \\ \mathcal{B} + \mathcal{C} \\ \hline \end{array}$$

- 2) Space transport novel **semi-Lagrangian scheme**

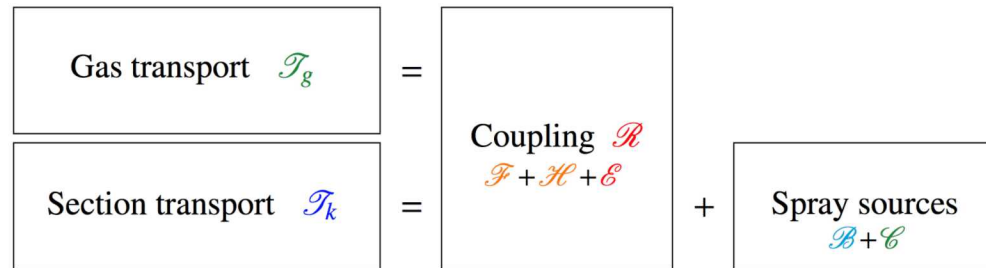


NUMERICS – Time integration

A Tailored Operator Splitting

Operator splitting

- Recycle legacy solvers
- Robust time integration
- Local properties enforced
- Adaptable accuracy



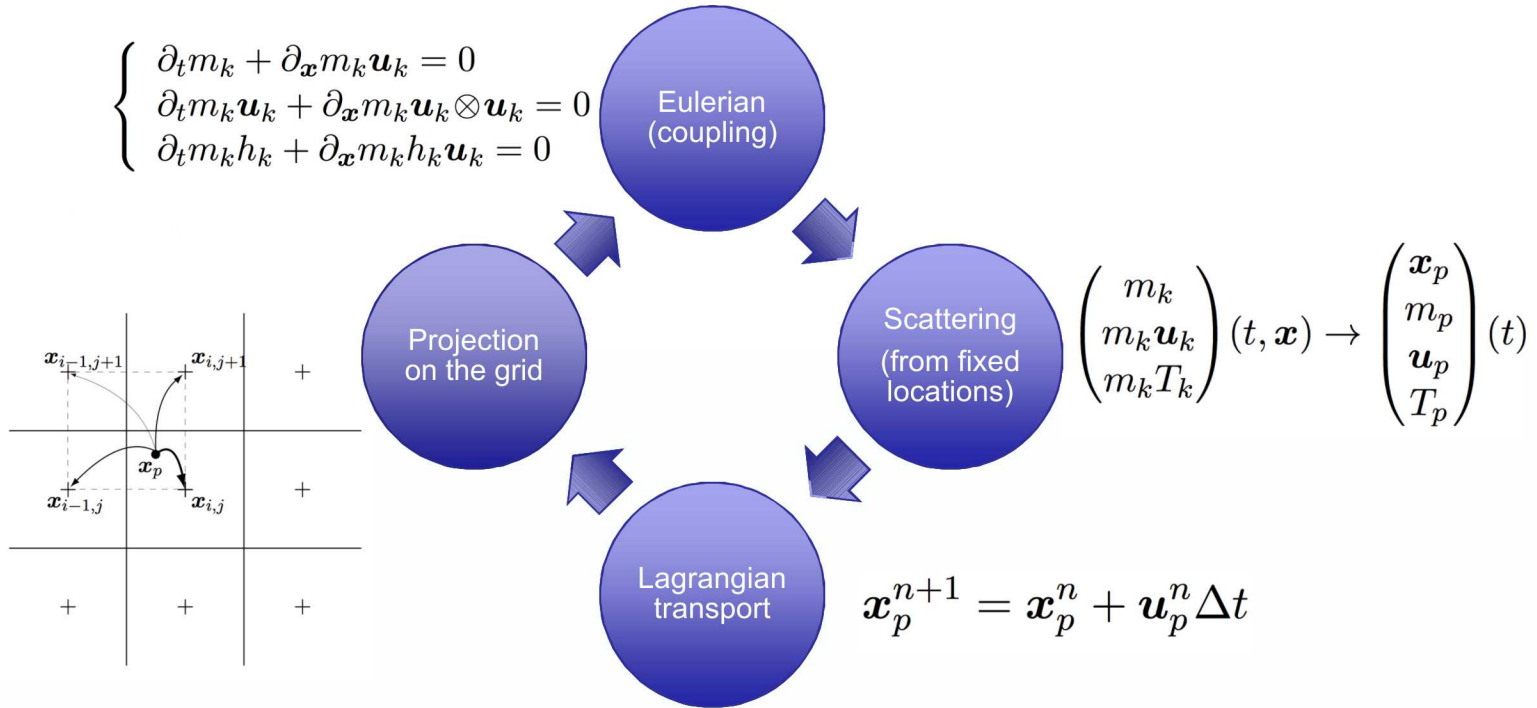
- to integrate all phase exchange terms \mathcal{R} at once (RK4)
 - Realizability, conservativity, equilibrium
 - Strong couplings
- to integrate spray sources $\mathcal{B} + \mathcal{C}$
 - Realizability and convergence
 - Strong particle-particle coupling

$$U^{n+1} = \mathcal{R} \prod_{k=1}^{N_{\text{sec}}} (\mathcal{T}_k) \mathcal{T}_g U^n$$

$$U^n = \begin{pmatrix} \rho_g Y_i \\ \rho_g \mathbf{u}_g \\ \rho_g e_g \\ n_k \\ m_k \\ m_k \mathbf{u}_k \\ m_k h_k \end{pmatrix}^n$$

NUMERICS – PGD transport

A robust and accurate answer to PGD peculiarities

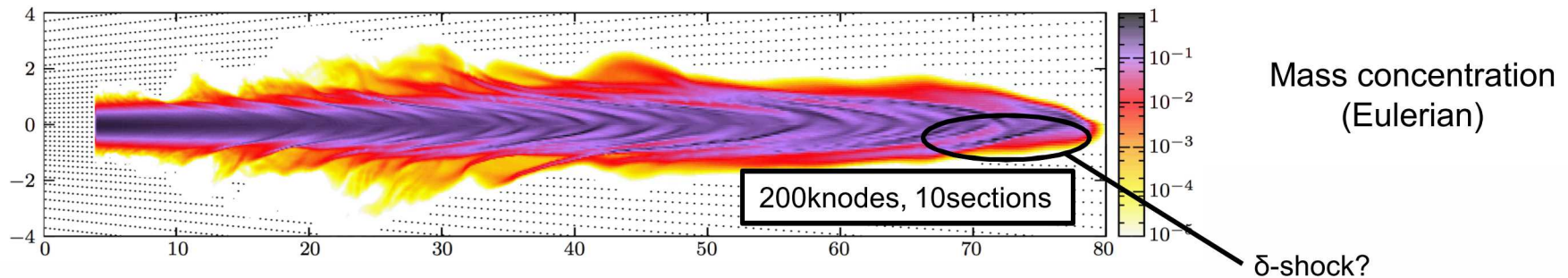


- **Novel semi-Lagrangian PGD transport scheme**
 - **Deterministic: no noise**
 - **Localizes spray info at mesh nodes: good for coupling**
 - **Easier load balancing**
 - **No fluxes to be computed: reduce cost and numerical diffusion**

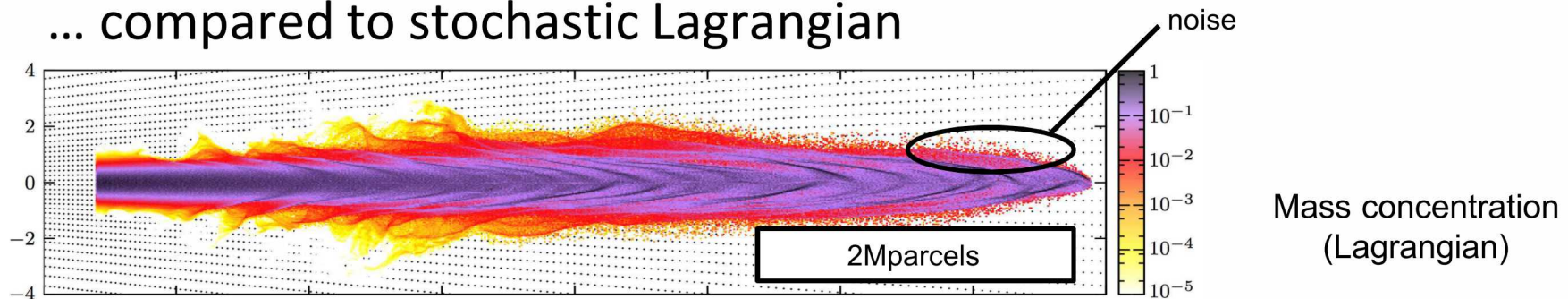
NUMERICS – PGD transport

2D test with prescribed flow field

- Obtained **cost-efficient** and **accurate** results



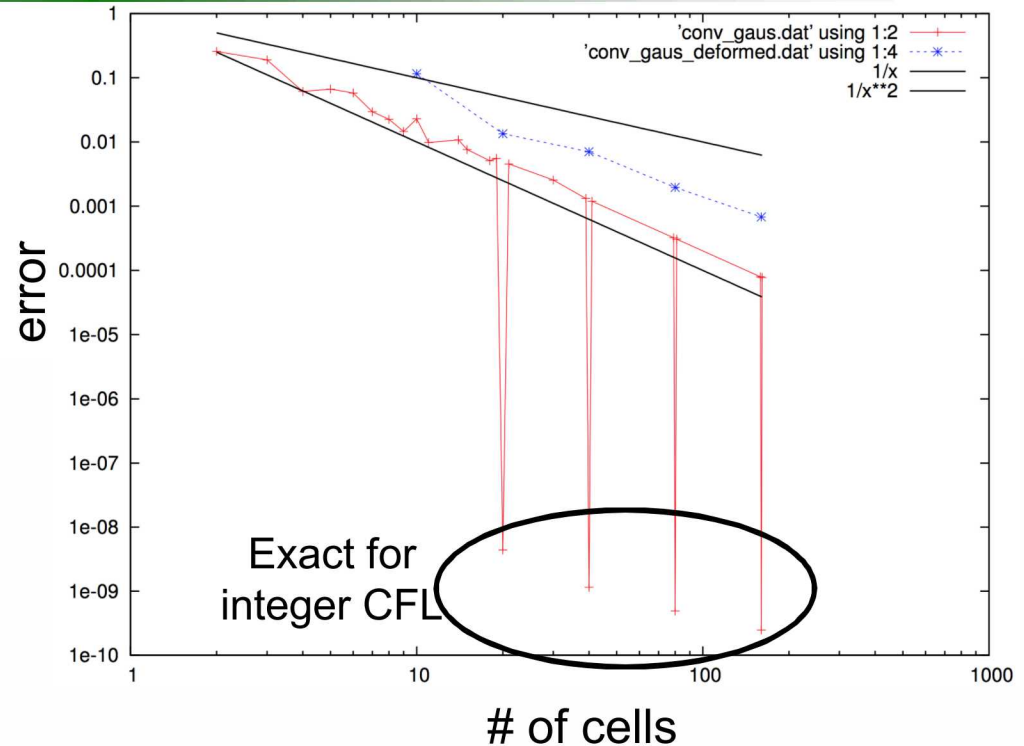
... compared to stochastic Lagrangian



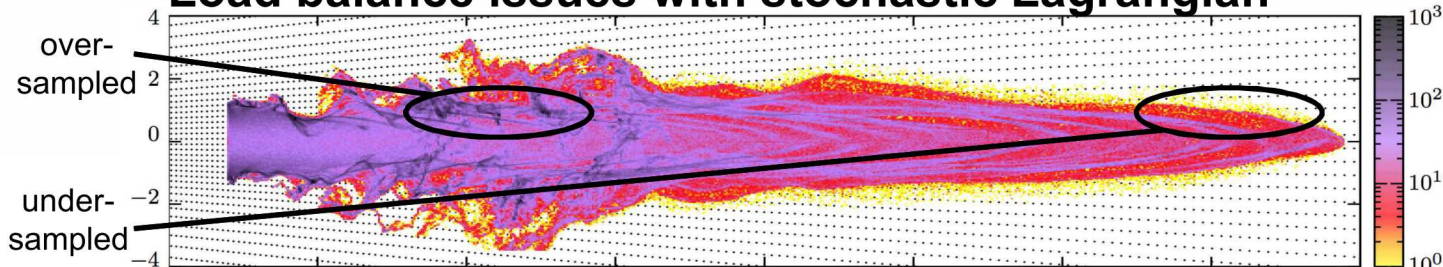
NUMERICS – PGD transport

Transport is 2nd order in space

- **No CFL constraint**
(unconditionally stable)
- **Handles vacuum**
- **Handles δ -shocks**
- **Predictable load**



Load balance issues with stochastic Lagrangian



Number of
numerical parcels
(Lagrangian)

TEST 1 – Momentum Coupling

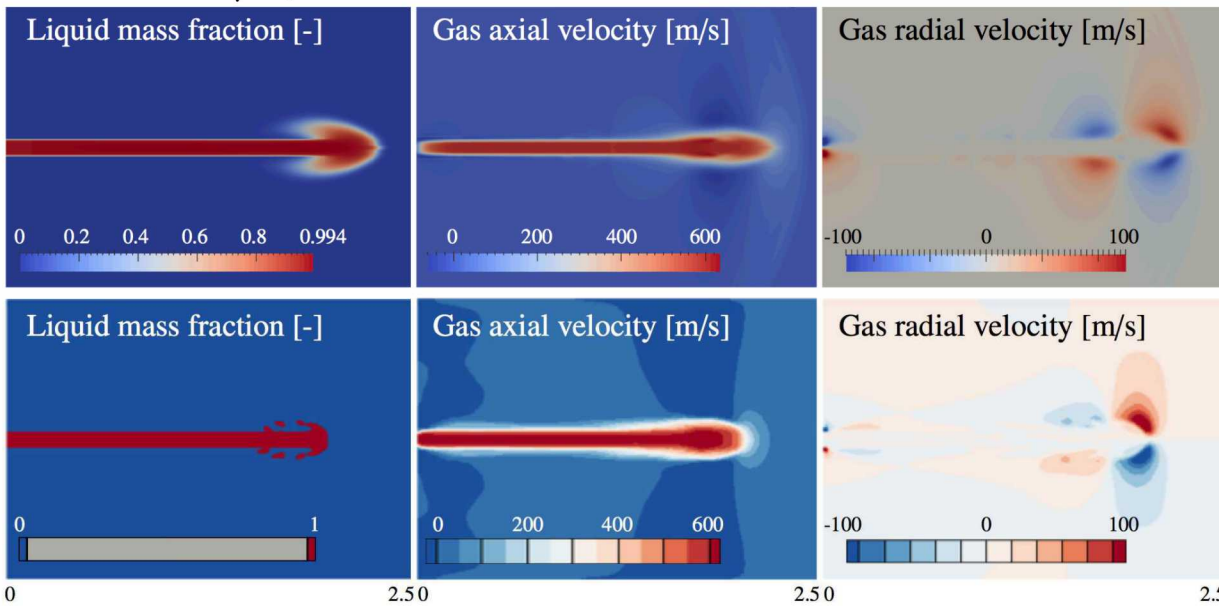
Comparison between E-ES and CLSVOF

✓ Supersonic injection (toy problem)

- velocity plug-flow boundary
- no thermal transfer
- $T_{\text{end}} = 4\mu\text{s}$

Raptor with E-ES

$\Delta x = 12.5 \mu\text{m}$, $\Delta t = 8 \text{ ns}$



CLSVOF

$\Delta x = 13.3 \mu\text{m}$, $\Delta t \sim 6 \text{ ns}$

✓ Agreement on **gas entrainment**

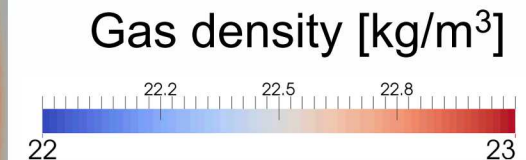
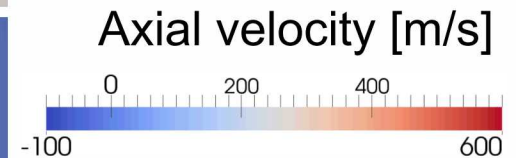
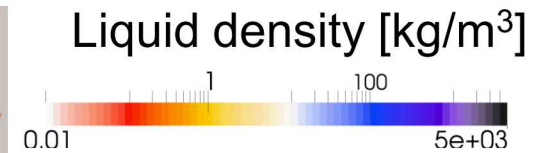
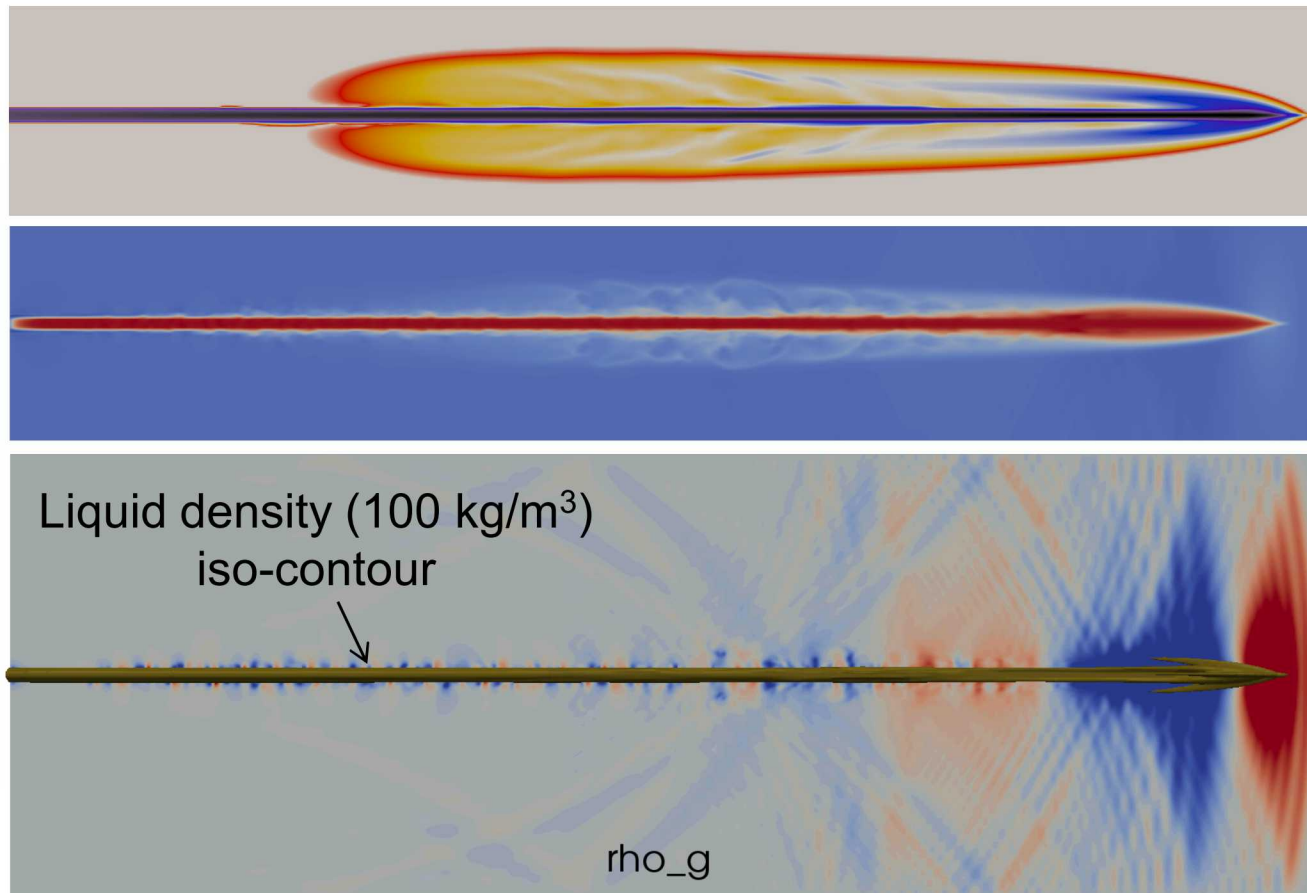
✱ Liquid density discrepancy from **pressureless** assumption

✱ Jet tip is different because of **lack of surface tension**

TEST 2 – Induced turbulence

Entrainment and induced turbulence by jet injection

- Executed with **RAPTOR + E-ES**

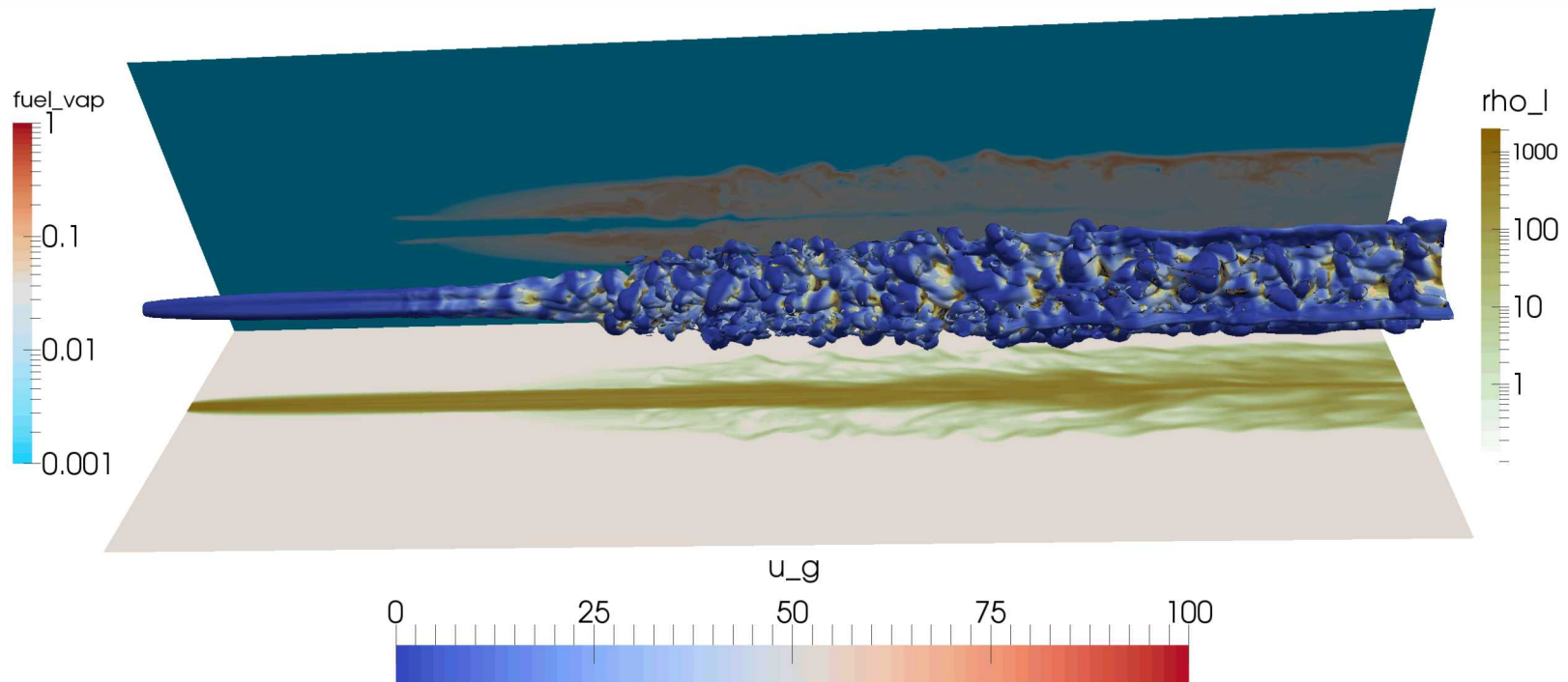


TEST 3 – Fuel vaporization

Fuel vapor footprint

- Executed with **RAPTOR E-ES**

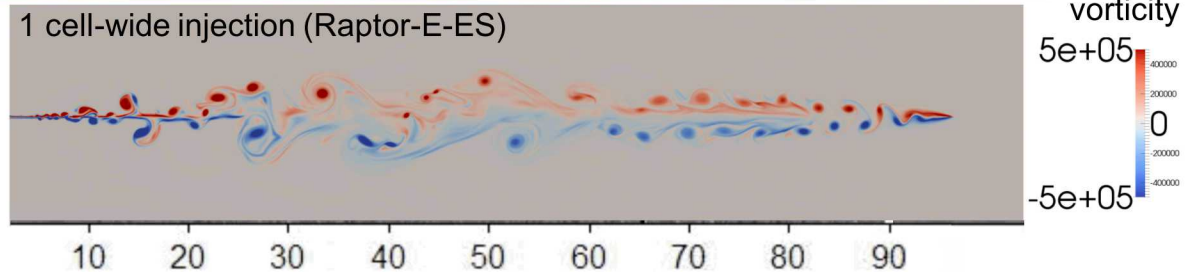
- Box 3x3x10mm
- $d_{inj}=90\mu\text{m}$, $T_{end}=40\mu\text{s}$
- quiescent gas at 60bar, 900K
- n-dodecane at 702kg/m³, 600m/s
- 50Mcells (cartesian mesh)
- $\Delta x=12.5\mu\text{m}$, $\Delta t=8\text{ns}$, $T_{end}=40\mu\text{s}$
- 1 section (prescribed initial size)
- PGD transport (δ -shocks)
- d²-law



CONCLUSION

Conclusion

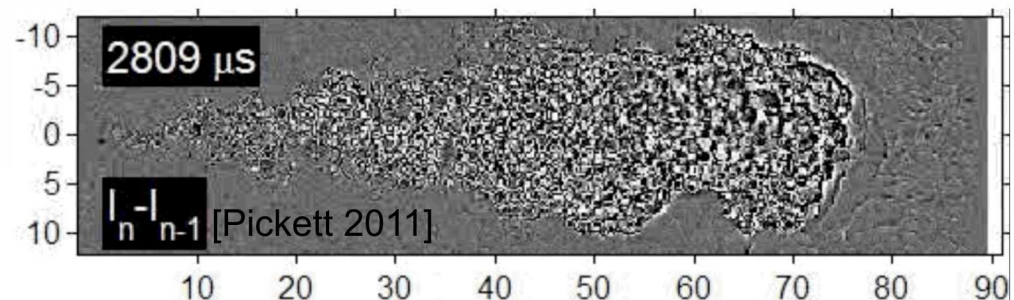
- Kinetic theory
- Two-way coupling
- Dedicated numerics



Spray tools are promising to efficiently **handle injection**

Perspectives

- **Dense core dynamics**
 - pressure/crossings
 - turbulence
 - surface tension
- **High-pressure mixing**
 - atomization
 - “evaporation”
 - LES closure
- **Combustion**
 - chemistry
 - LES closure
 - numerics
- **Verification**
 - vs CLSVOF
 - vs stochastic Lagrangian
 - vs Real-Gas solver
- **Validation** vs ECN results (spray A)



- [Chesnel 2011] Chesnel, J., Reveillon, J., Menard, T., and Demoulin, F.-X. (2011) **Large eddy simulation of liquid jet atomization** *Atomization and Sprays*, 21(9).
- [J.E. Dec, 1997] Dec, J. E. **A conceptual model of DI diesel combustion based on laser-sheet imaging** *SAE transactions*, 106(3):1319-1348.
- [Desjardins 20XX] Desjardins, O. and Pitsch, H. (2010) **Detailed numerical investigation of turbulent atomization of liquid jets** *Atomization and Sprays*, 20(4).
- [Doisneau 2013] Doisneau, F., Laurent, F., Murrone, A., Dupays, J., and Massot, M. (2013). **Eulerian Multi-Fluid models for the simulation of dynamics and coalescence of particles in solid propellant combustion** *Journal of Computational Physics*, 234 :230–262.
- [Laurent 2001] Laurent, F. and Massot, M. (2001) **Multi-fluid modeling of laminar poly-dispersed spray flames: origin, assumptions and comparison of the sectional and sampling methods** *Combustion Theory and Modelling*, 5(4):537-572
- [Menard 2007] Menard, T., Tanguy, S., and Berlemont, A. (2007). **Coupling level set/VOF/ghost fluid methods : Validation and application to 3D simulation of the primary break-up of a liquid jet** *International Journal of Multiphase Flow*, 33(5) :510–524.
- [Pickett 2010] Pickett L.M., Genzale C.L., Bruneaux G., Malbec L.-M., Hermant L., Christiansen C., Schramm J. (2010) **Comparison of diesel spray combustion in different high-temperature, high-pressure facilities** *SAE Int. J. Engines* 3(2):156-181.
- [Pickett 2011] Pickett L.M., Manin J., Genzale C.L., Siebers D.L., Musculus M.P.B., Idicheria C.A. (2011) **Relationship between diesel fuel spray vapor penetration/dispersion and local fuel mixture fraction** *SAE Int. J. Engines* 4(1):764-799.
- [Skeen 2014] Skeen, S.A. and Manin, J. and Pickett, L.M. **Simultaneous formaldehyde PLIF and high-speed schlieren imaging for ignition visualization in high-pressure spray flames** *Proceedings of the Combustion Institute*, 35(3):3167-3174.
- [Som 2013] Xue, Q. and Som, S. and Senecal, P. K. and Pomraning, E. (2013). **Large eddy simulation of fuel-spray under non-reacting IC engine conditions** *Atomization and Sprays*, 23(10).
- [Tillou 2014] Tillou, J., Michel, J.-B., Angelberger, C., and Veynante, D. (2014). **Assessing LES models based on tabulated chemistry for the simulation of diesel spray combustion** *Combustion and Flame*, 161(2) :525–540.

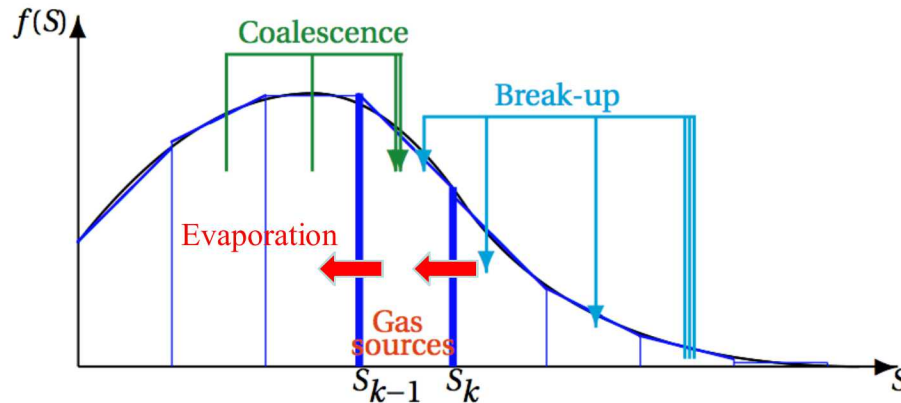


BACK-UP

MODEL – Sectional method

A cost-efficient way to capture polydispersity

- Various drop sizes are treated as a continuum



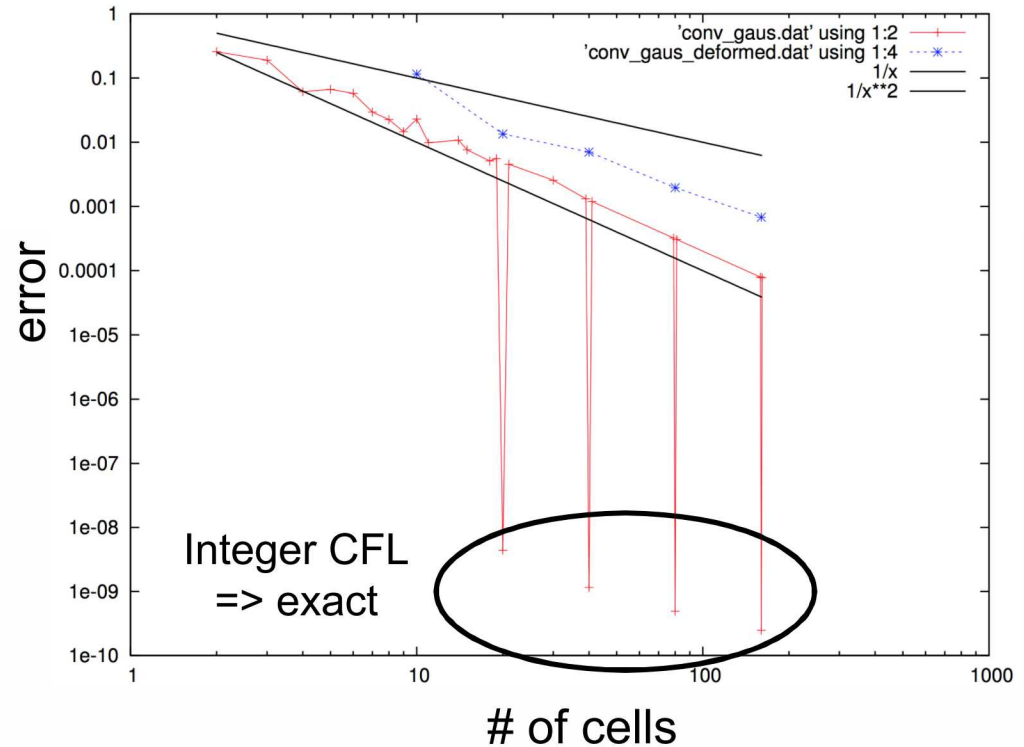
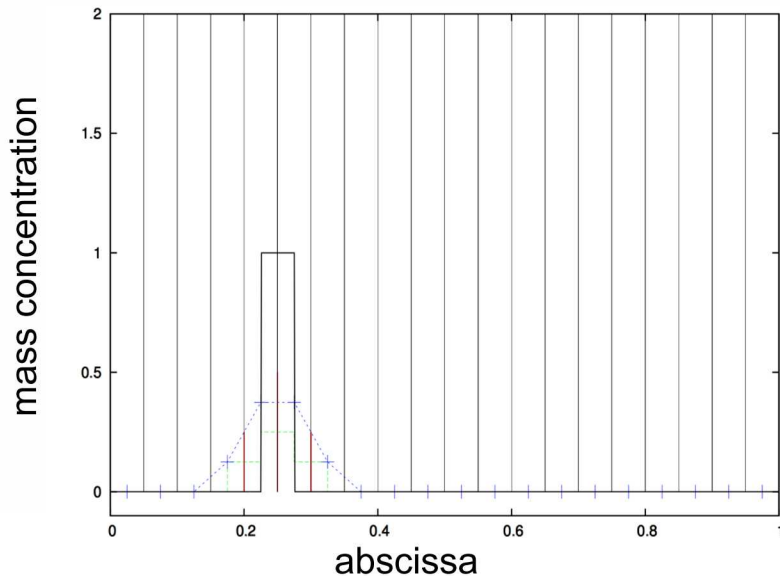
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 \end{aligned} \right. \quad \Longleftrightarrow \quad \text{Navier-Stokes with sources}
 \end{aligned}$$

...many integral source terms to compute

NUMERICS – PGD transport

Transport is 2nd order in space

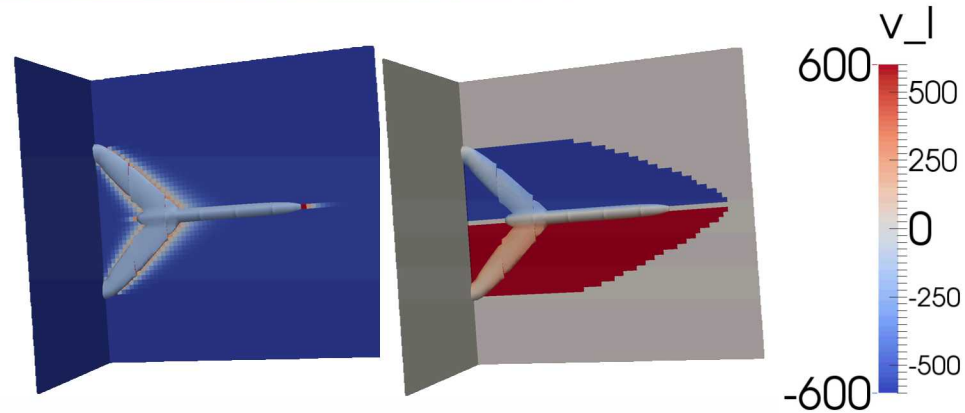
- No CFL constraint (unconditionally stable)
- Handles vacuum
- Handles δ -shocks



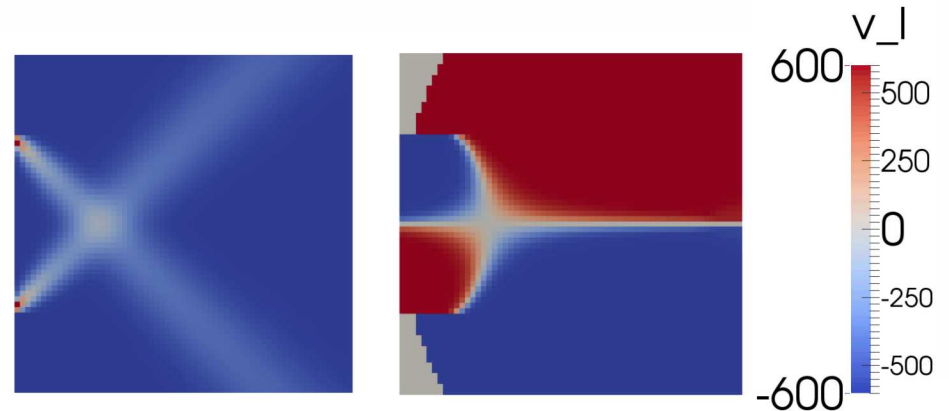
NUMERICS – Beyond PGD transport

Prevention of δ -shocks

- δ -shocks are an artifact from the PGD assumption
 - no grid convergence
 - erroneous density and gradients
 - troublesome with two-way coupling



- Higher order moment methods are developed to solve this



NUMERICS – Raptor

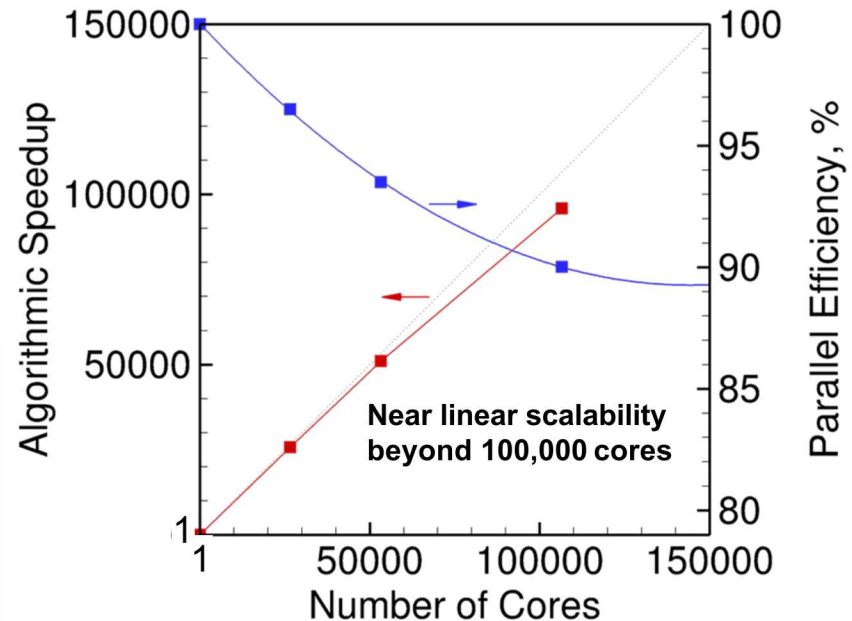
A general solver optimized for LES

- Theoretical framework

- Fully-coupled, compressible conservation equations
- Real-fluid equation of state (high-pressure phenomena)
- Detailed thermodynamics, transport and chemistry
- Multiphase flow, spray
- Dynamic SGS modeling (**No Tuned Constants**)
- **Advanced UQ methods for error/sensitivity analysis**

- Numerical framework

- Staggered finite-volume differencing (non-dissipative, discretely conservative)
- Dual-time stepping with generalized preconditioning (all-Mach-number formulation)
- Detailed treatment of geometry, wall phenomena, BC's



- High-performance computing framework (Advanced parallel programming model that makes optimal use of advanced MP-computer architectures)
- Results from strong and weak scaling on Oak Ridge National Laboratory CRAY XK7 (Titan), June 2013
 - Test case – jet-in-cross-flow, 500-million cells
 - **Strong scaling:** 24,000 to 120,000 cores, > 90% efficiency
 - **Weak scaling:** 500-million-cells/24,000-cores to 2-billion-cells/120,000-cores, < 4% increase in CPU time
- Currently being refactored for hybrid multi-core parallelism and GPU acceleration (MPI/OpenMP/OpenACC)