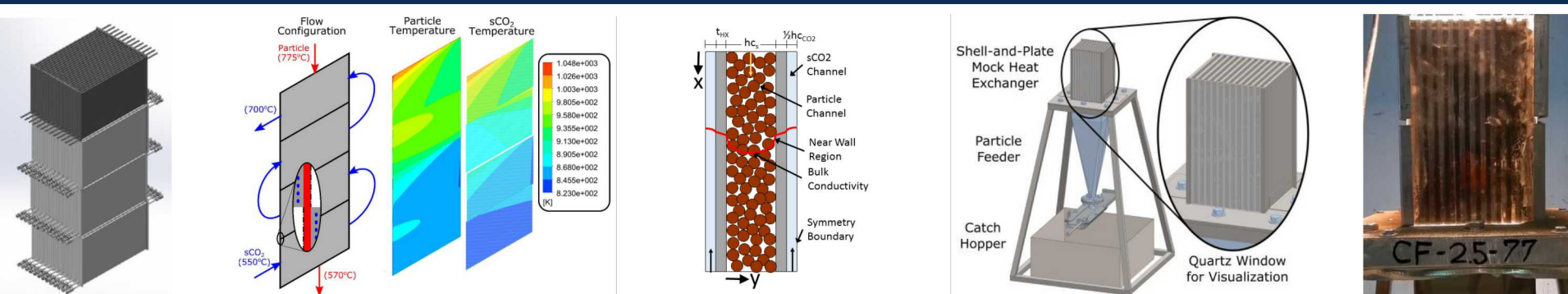


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# Efficient Modeling and Analysis of a Moving Packed-Bed Particle-to-sCO<sub>2</sub> Heat Exchanger

Kevin J. Albrecht and Clifford K. Ho

Sandia National Laboratories, Concentrating Solar Technologies (8823)

PowerEnergy2018-7324, SAND2018-####



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# Outline

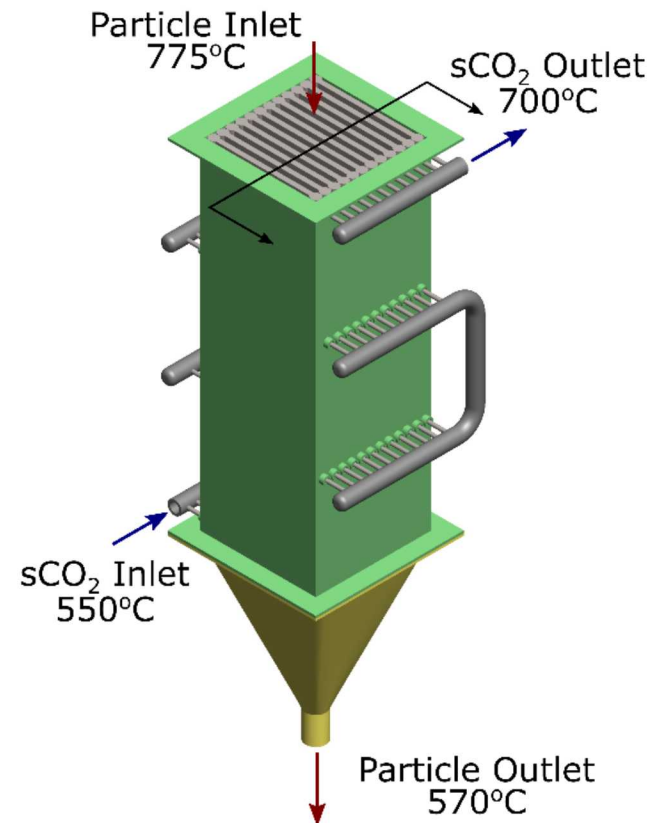
- Introduction/Objectives
- Heat Exchanger Modeling
- Heat Exchanger Design Considerations
- Conclusion/Future Work

# Outline

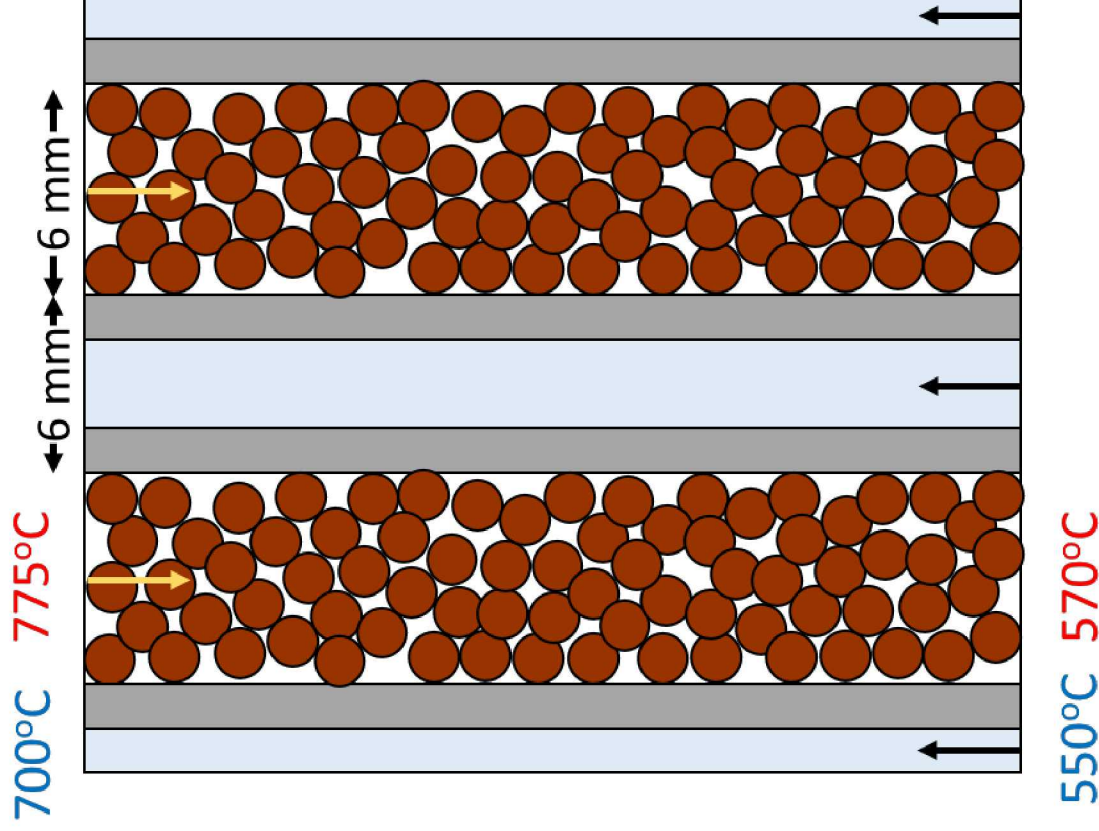
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# Particle receiver development has led to the need for an sCO<sub>2</sub>-to-particle heat exchanger

- Particle-based CSP plants enable the use of sCO<sub>2</sub> power cycles
- Multiple particle receivers have been demonstrated at the megawatt scale
- Minimal work has been conducted on particle-to-sCO<sub>2</sub> heat exchangers

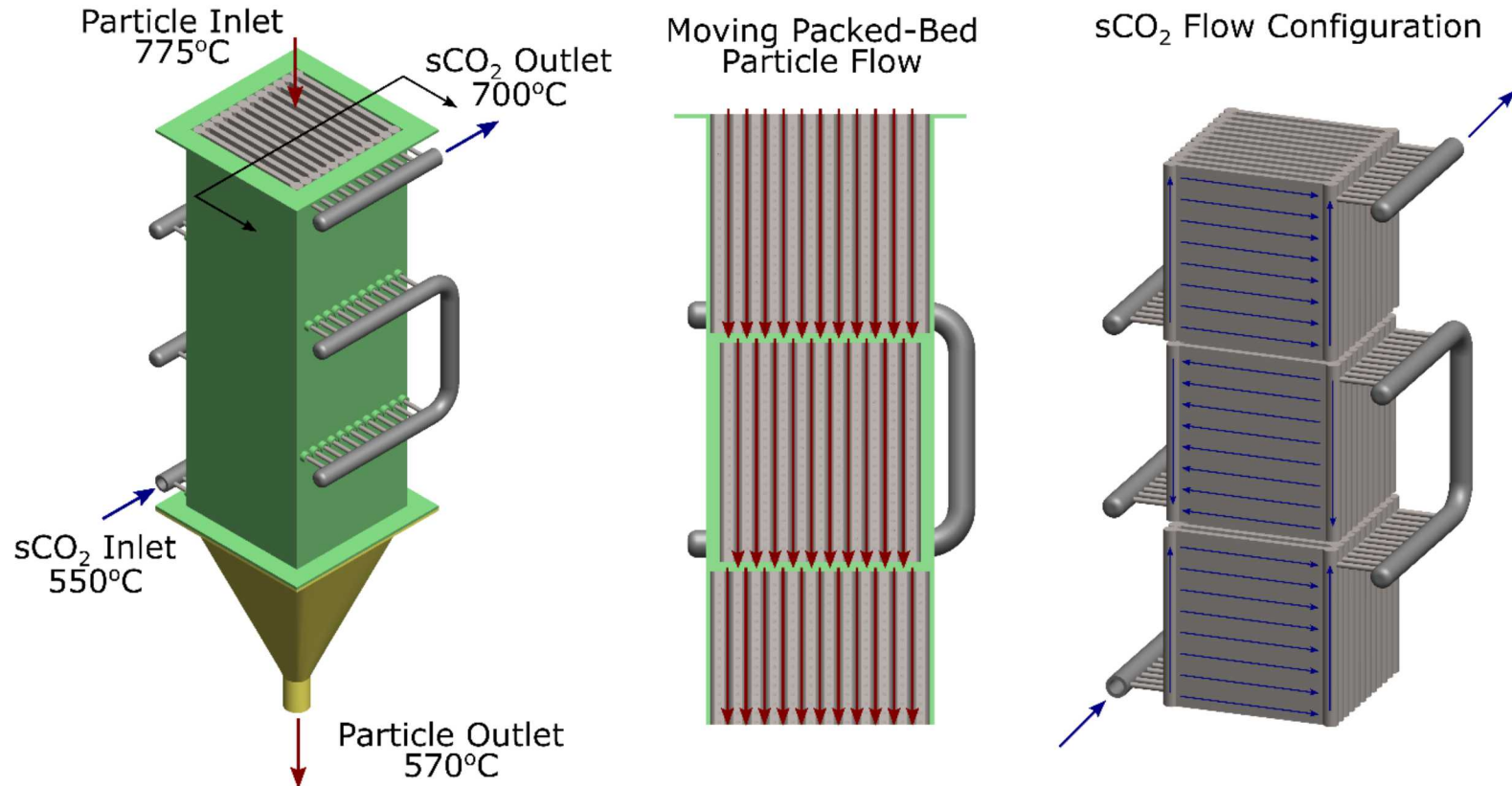


# Shell-and-plate moving packed-bed heat exchanger configuration



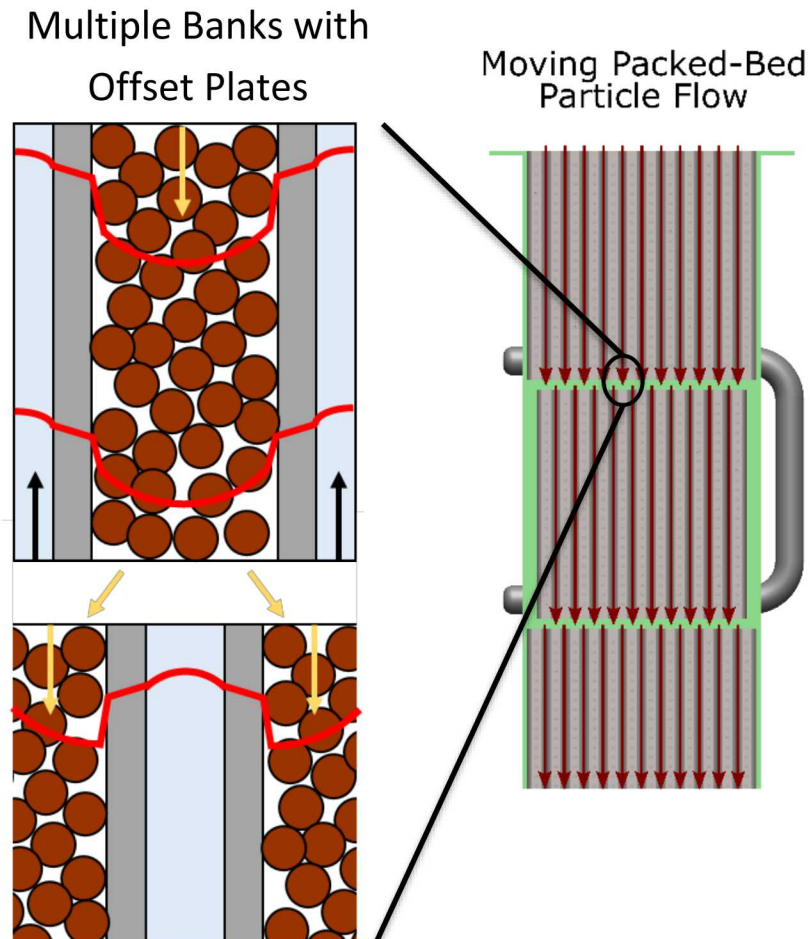


# Moving Packed-Bed Heat Exchanger Geometry



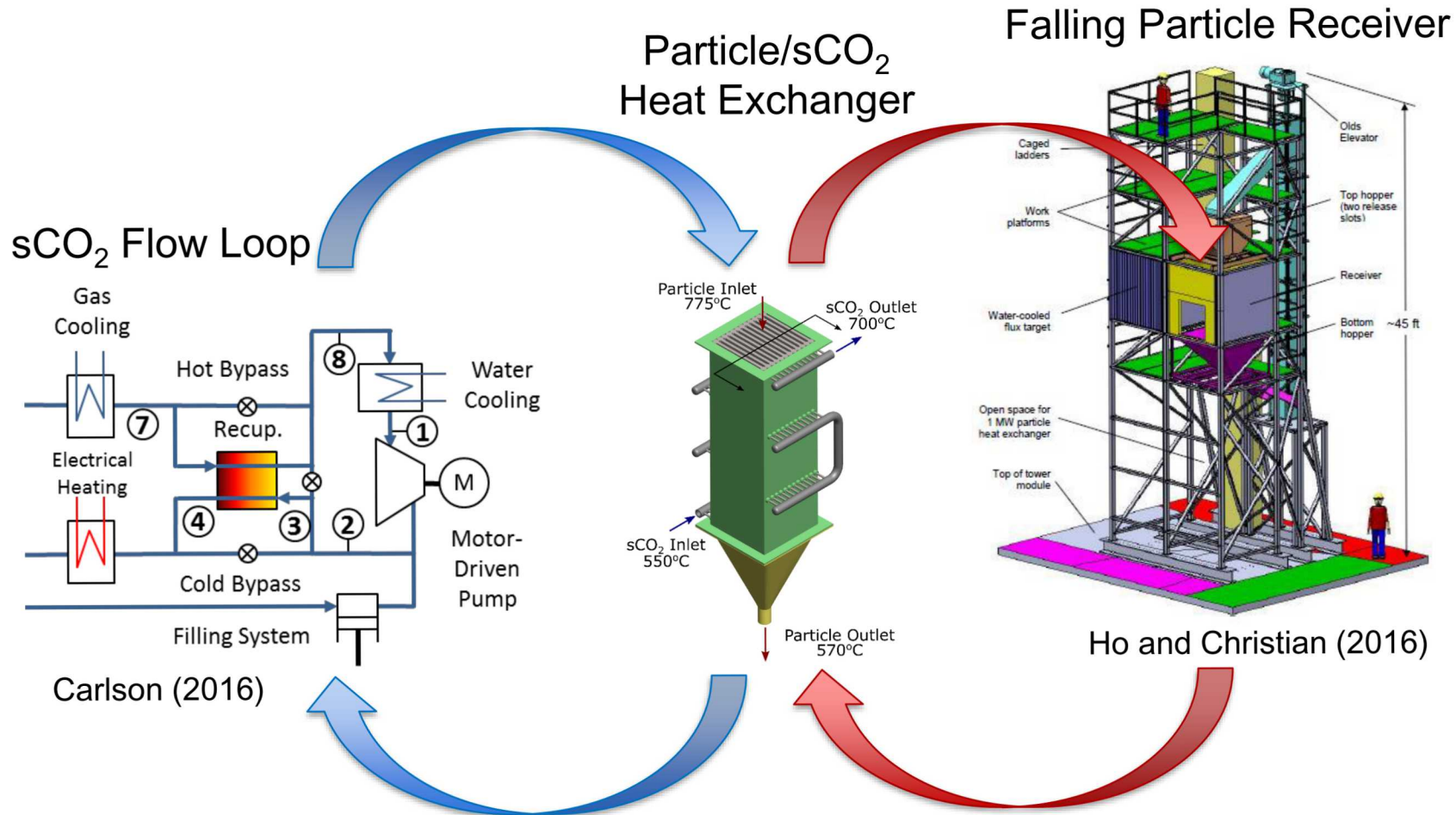
- Constructed from multiple cross-flow banks arranged in a counter-flow configuration

# Multibank approach allows for particle heat transfer enhancement



- Multibank configuration allows for cross-channel mixing and multiple thermal entries

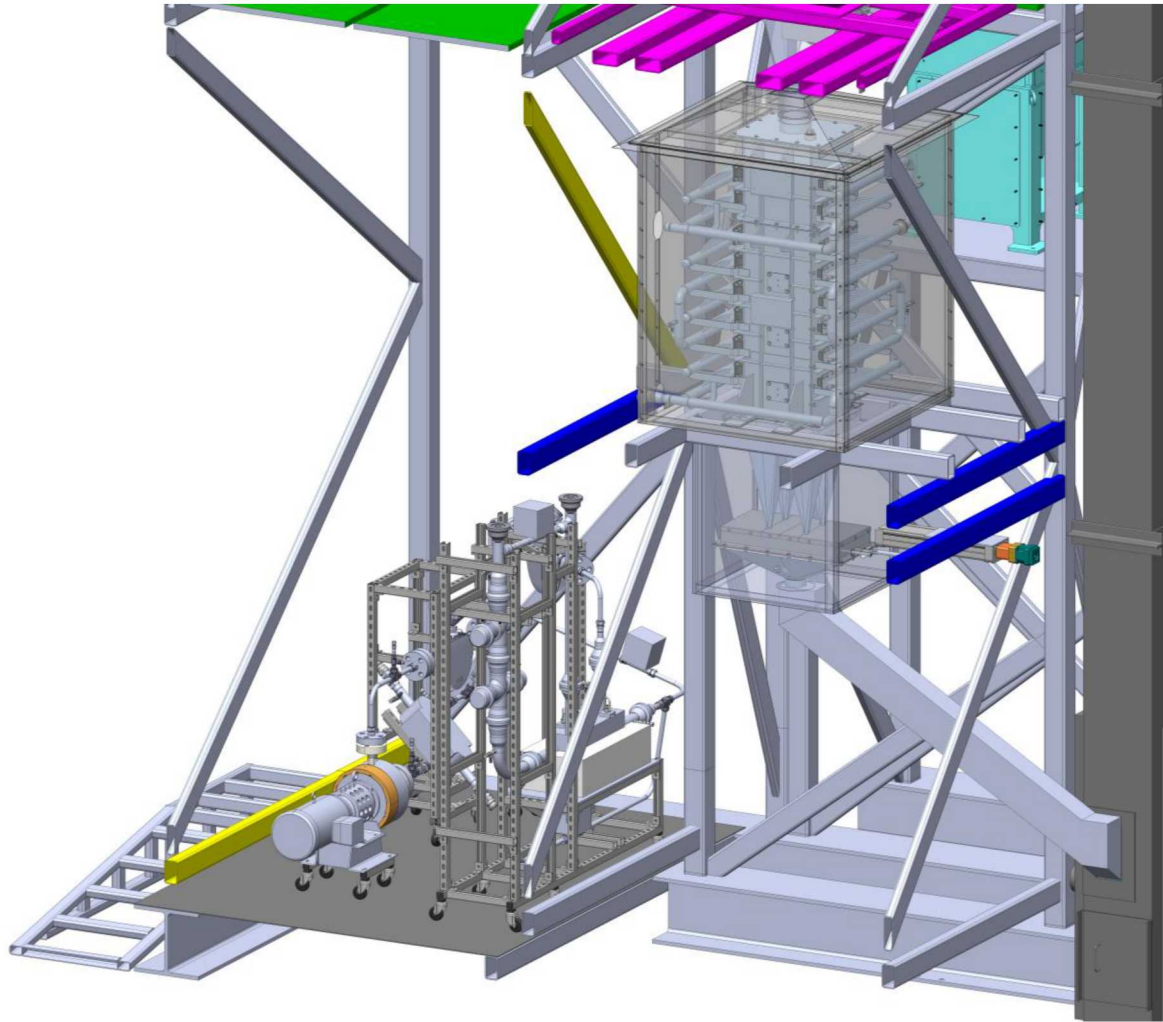
# Objective



- Develop and test 100kW<sub>th</sub> prototype particle-to-sCO<sub>2</sub> heat exchanger



# 100 kW<sub>t</sub> heat exchanger and sCO<sub>2</sub> flow loop integrated with falling particle receiver module

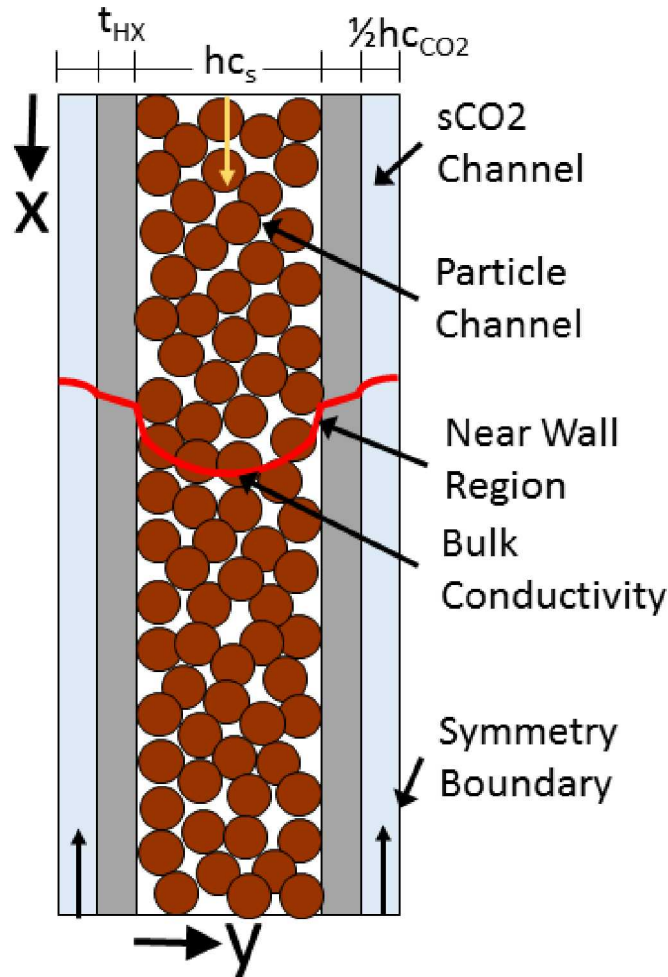


- Heat Exchanger to be commissioned in late summer 2018

# Outline

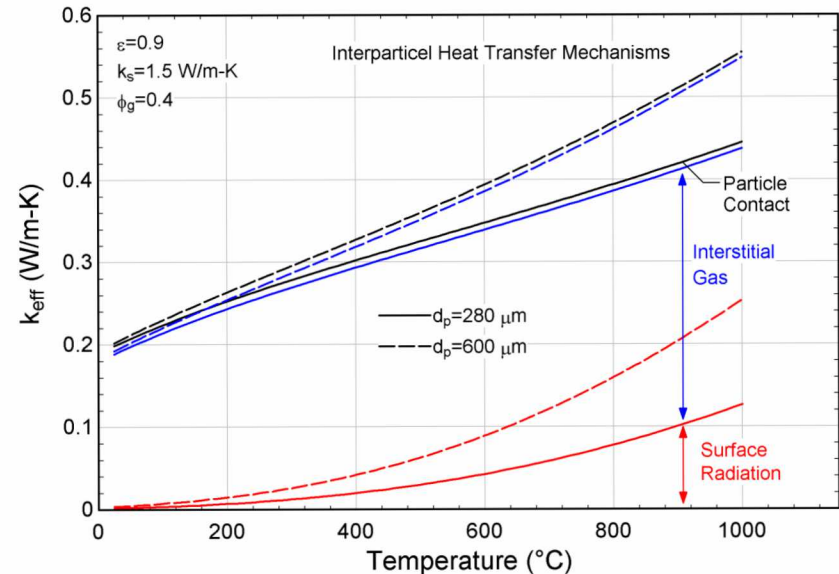
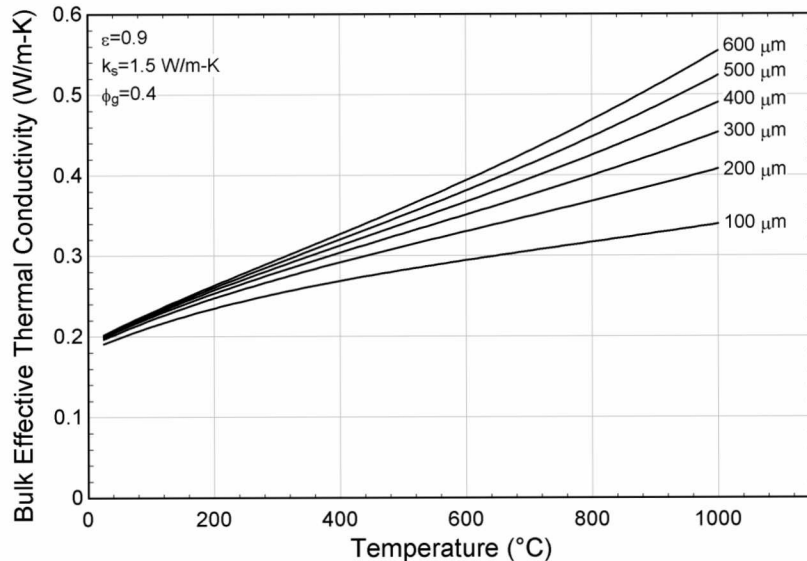
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# Moving Packed-Bed Heat Transfer Modeling



- Correlations required to predicted thermal resistance in near-wall region
- Thermal transport in particle packed bed is characterized with bulk effective thermal conductivity
- Couple particle and sCO<sub>2</sub> domain temperature profiles

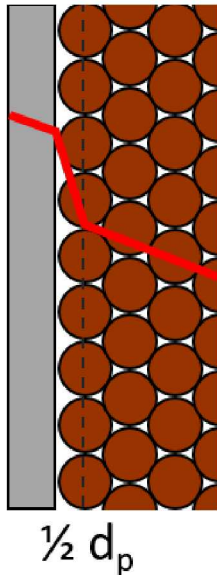
# Contributions to Effective Packed-Bed Thermal Conductivity



- Packed-bed thermal conductivity rises with temperature due to increased conductivity of the gas phase and radiation contributions
- Particle diameter has a large impact on thermal conductivity at elevated temperature due to radiation

W. van Antwerpen, C. G. du Toit and P. G. Rousseau, "A review of correlations to model the packing structure and effective thermal conductivity in packed beds of mono-sized spherical particles," Nuclear Engineering and Design, vol. 240, no. 7, pp. 1803-1818, 2010.

# Particle packing in the near-wall region increases thermal resistance



$$\phi_{g,nw} = \frac{(1 - \phi_g)(0.7293 + 0.5139Y)}{1 + Y}$$

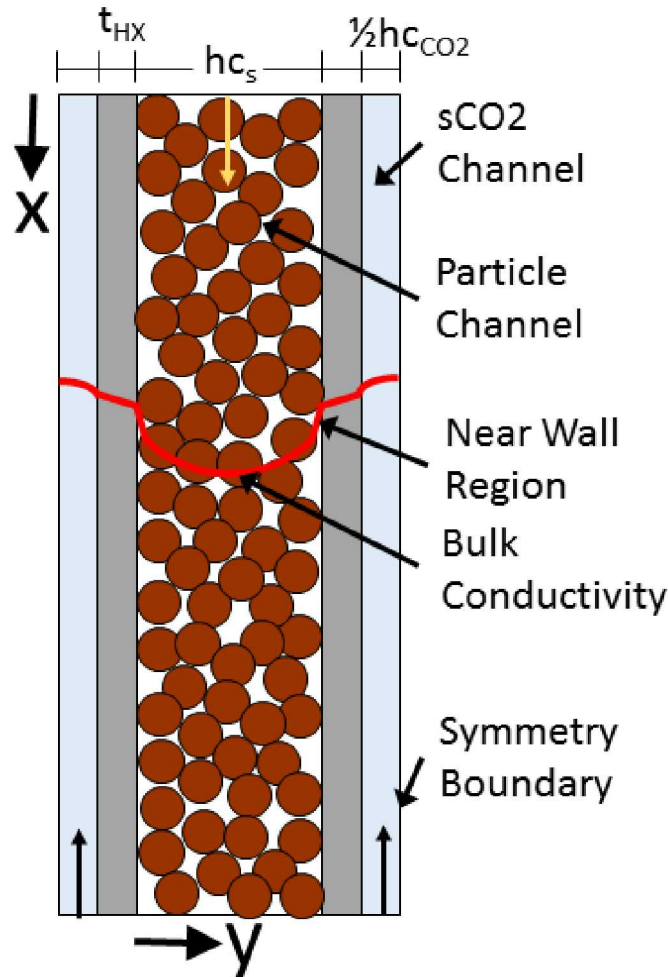
$$R_c'' = \frac{d_p}{2k_{s,nw}^{eff}}$$

J. M. Botterill and A. O. Denloye, "A Theoretical Model of Heat Transfer to a Packed or Quiescent Fluidized Bed," *Chemical Engineering Science*, vol. 33, no. 4, pp. 509-515, 1978.

- Particle packing in near wall region results in high void fractions and reduced effective thermal conductivity
- Resistance is considered to be  $\frac{1}{2}$  particle diameter



# 2-D Single Component Continuum Modeling Approach



2D S.S. Particle Conservation of Energy:

$$\rho_s v_s c_{p,s} \frac{\partial T_s}{\partial x} = \frac{\partial}{\partial y} \left( k_{s,\text{eff}} \frac{\partial T_s}{\partial y} \right)$$

1D S.S. sCO2 Conservation of Energy:

$$\rho_{\text{CO}_2} v_{\text{CO}_2} c_{p,\text{CO}_2} \frac{dT_{\text{CO}_2}}{dx} = \frac{2q''}{h_{\text{CO}_2}}$$

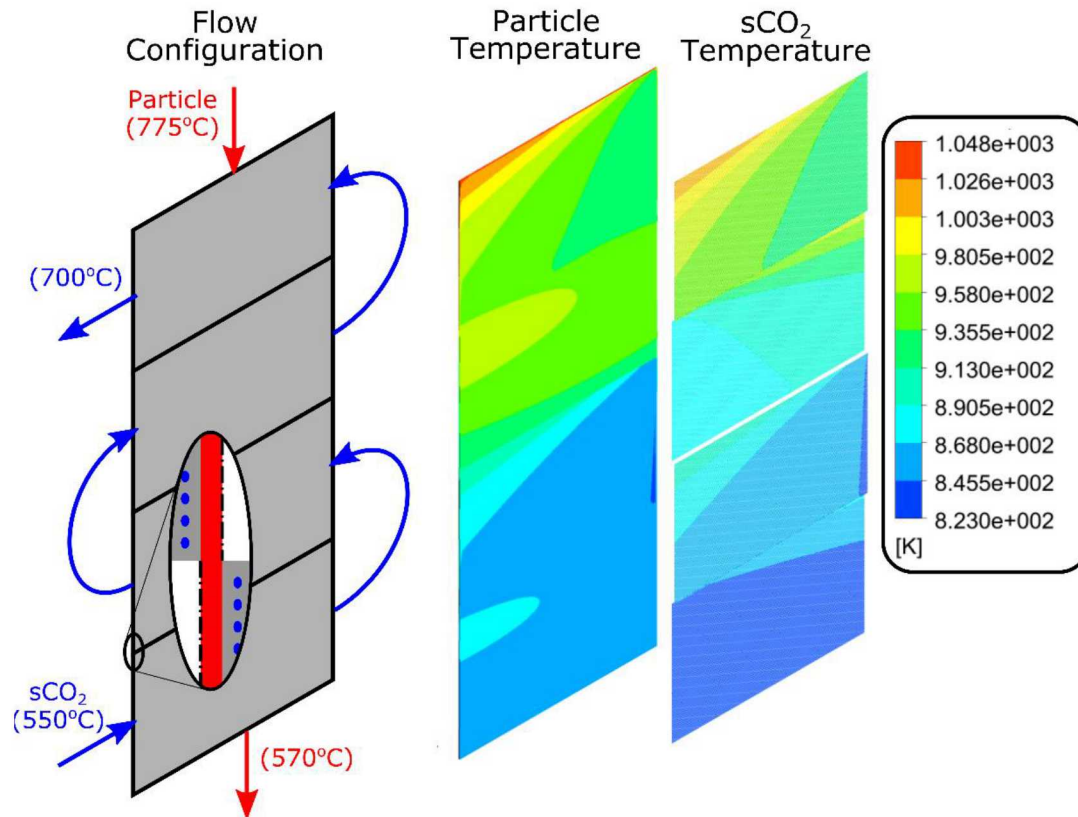
Regions Coupled Through Resistive Network:

$$q''(x) = k_{s,\text{eff}} \left. \frac{dT_s}{dy} \right|_{0,x} = \frac{1}{R''} (T_s(0, x) - T_{\text{CO}_2}(x))$$

$$R''(x) = R_c'' + \frac{t_{\text{HX}}}{k_{\text{HX}}} + \frac{1}{h_{\text{CO}_2}}$$

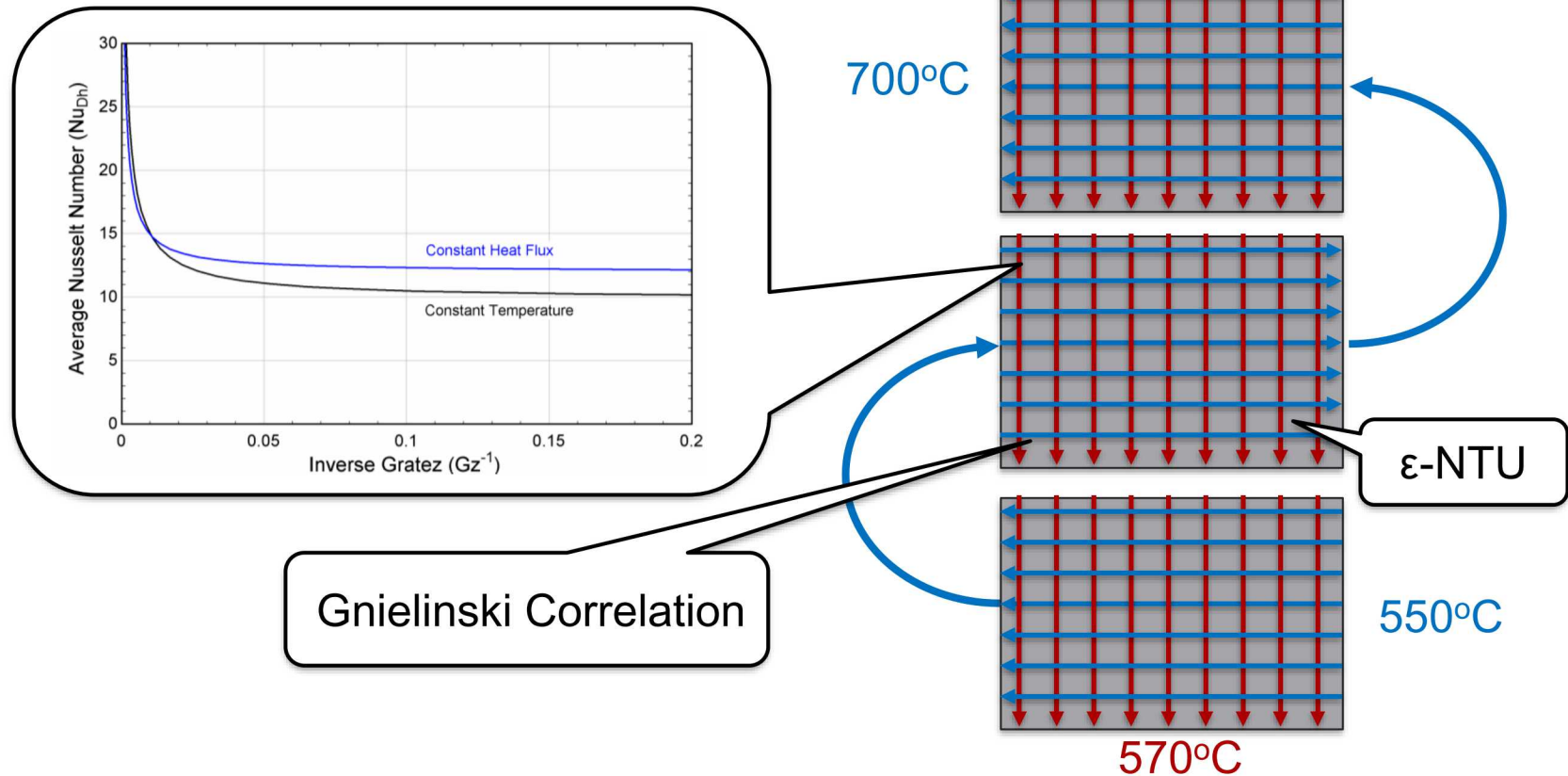
- Does not capture cross/counter flow limitations or plate geometry

# CFD Codes with closure relationships can be used to predict overall heat transfer coefficient



- CFD proved to be too computationally intensive for large parametric studies
- Model is still useful for prototype design, thermomechanical analysis and transient simulation

# Simplified modeling methodology for parametric studies

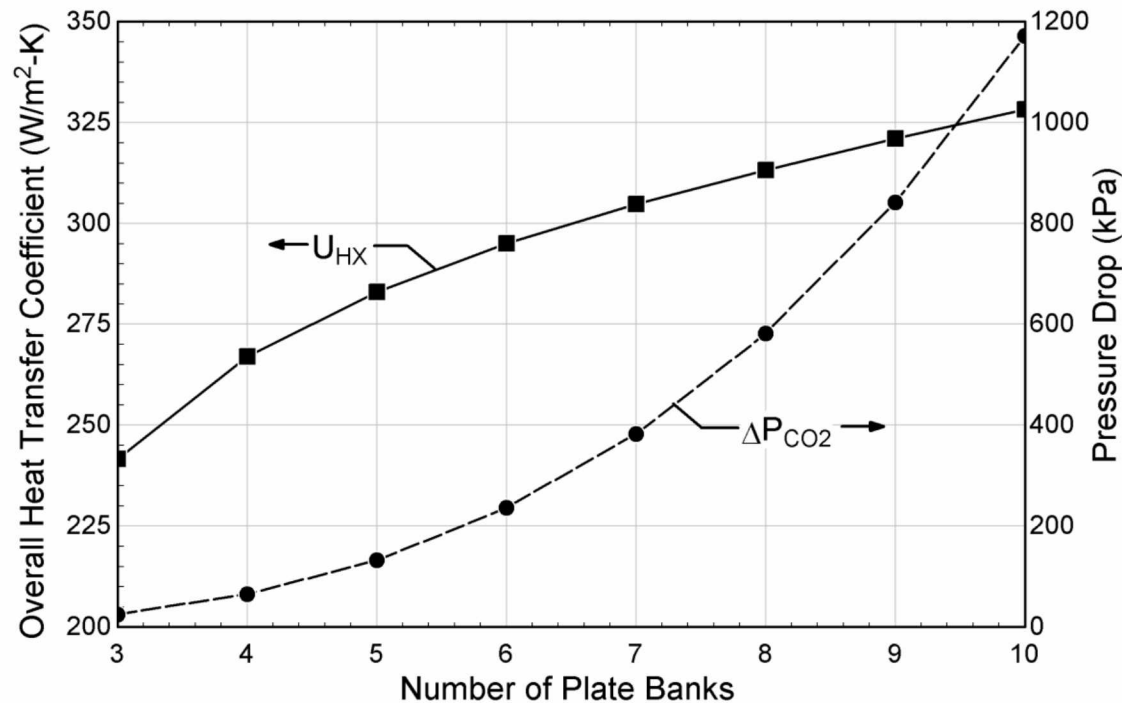


- Effectiveness-NTU correlations to capture flow configuration
- Correlations for particle-wall and  $\text{sCO}_2$ -wall convection coefficient
- Link bank inlets and outlets in counter-flow configuration

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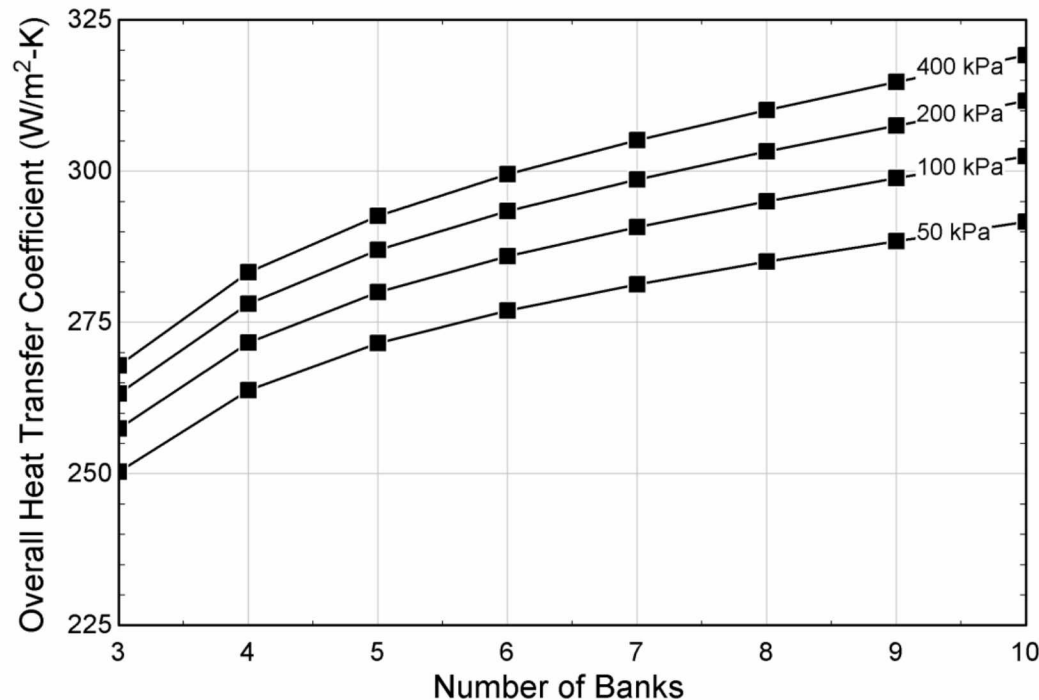
# Increasing the number of plate banks improves the overall heat transfer coefficient



- Increasing the number of cross-flow banks allows for more cross-channel mixing to increase particle side heat transfer
- Increased pressure drop results from additional banks with fixed sCO<sub>2</sub> channel geometry

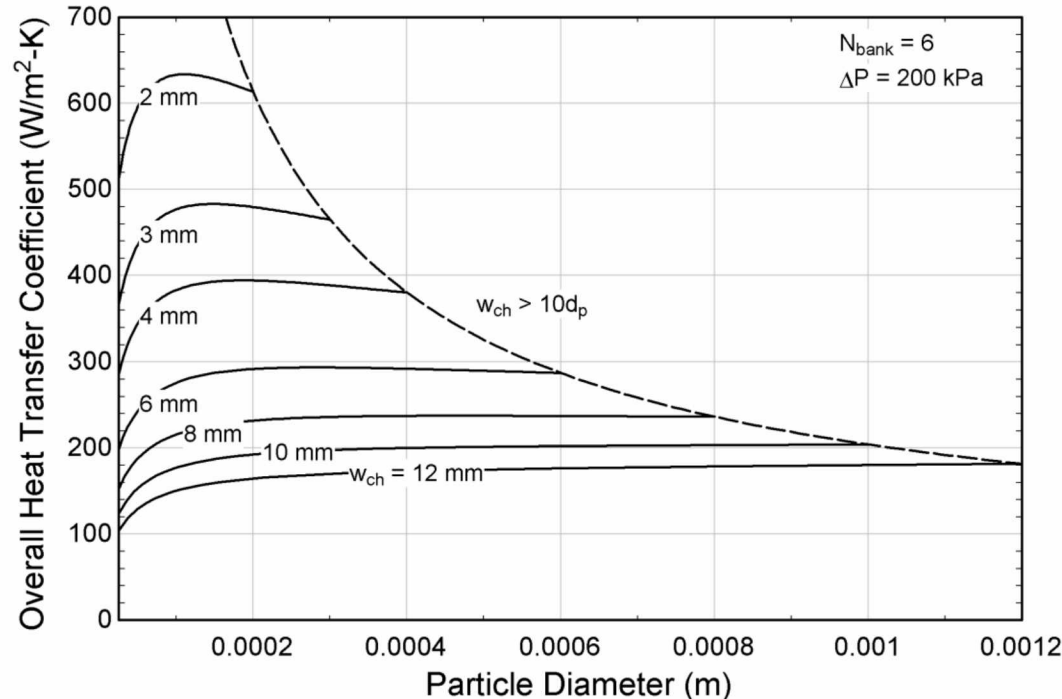


# Comparison of palate banks with constant sCO<sub>2</sub> pressure drop



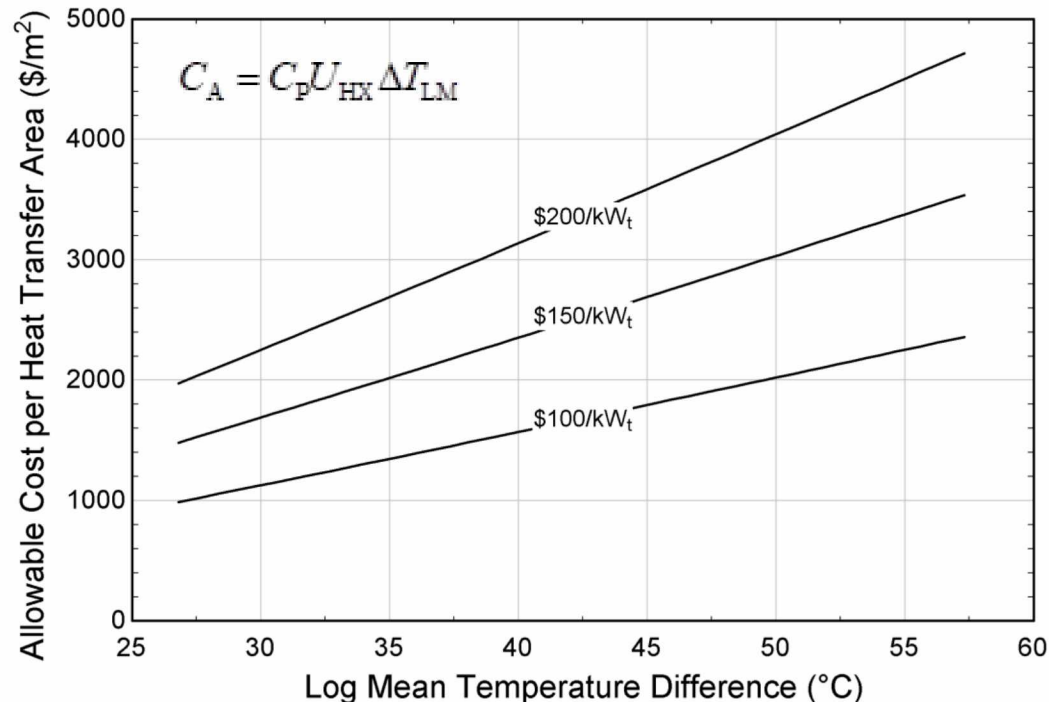
- Constant sCO<sub>2</sub> pressure drop reduces the overall heat transfer coefficient improvement from increasing number of banks

# Influence of Particle Size and Plate Spacing



- Smaller channel width improve heat transfer due to shorter heat diffusion lengths
- Smaller particle diameters are favored in smaller channels due to particle-wall contact resistance becoming limiting

# Primary Heat Exchanger Cost Considerations



- Performance, operating conditions, and cost targets determine allowable cost of heat transfer surface area
- SunShot 2020 cost targets for power cycle heat addition are \$150/kW<sub>t</sub>, which requires heat transfer surface area below \$2400/m²
- Full technoeconomic analysis needed to optimize operating conditions and full plant design

# Conclusion

- Avenues for improving moving packed-bed heat exchanger performance identified
  - Plate spacing reduction, corresponding particle size reduction, and increased number of banks
- Average particle-wall heat transfer coefficients approaching  $400 \text{ W/m}^2\text{-K}$  are possible with channel dimensions of 4 mm
- Coupled technoeconomic study with predictive models of moving packed-bed heat exchanger and falling particle receiver is needed
- Future work:
  - Heat exchanger 100 kW prototype will be commissioned in August and allow for model validation
  - Particle size distribution for reduced near wall thermal resistance

# Acknowledgements

- Ashley Byman and Robert McGillivray of Solex Thermal Science
- This work was supported by the DOE SunShot Program (SuNLaMP-0000000-1507)



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# Plate Geometry

