

IMAC XXXVI

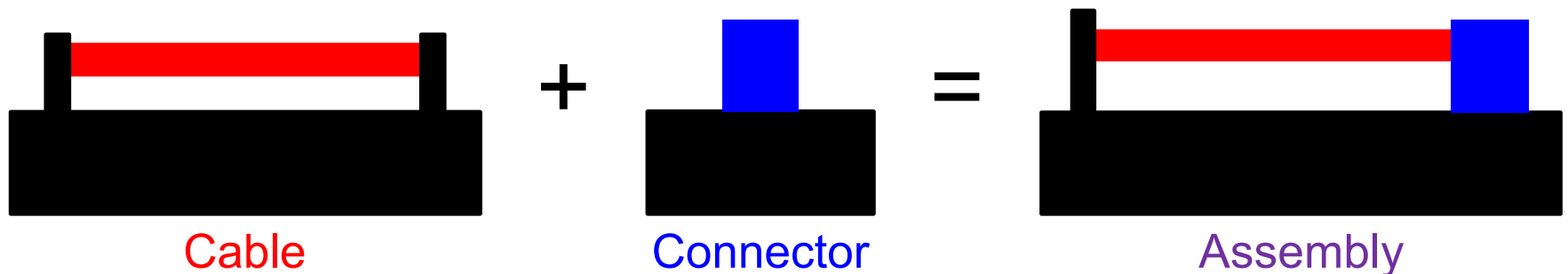
Predicting Assembly Effective Mass from Two Component Effective Mass Models

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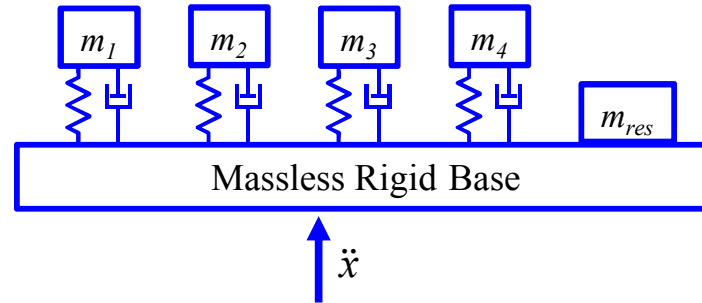
Motivation

- This work focuses on the development of an effective mass modal model of a cable-connector assembly for the purposes of margin quantification
- Developing effective mass modal models of cable-connector assemblies poses several challenges
 - Finite element models (FEMs) typically not developed
 - Multiple cable-connector combinations make analytical modeling and explicit testing of each combination impractical
- **The goal of this work is to develop a method to predict the effective mass and natural frequencies of the cable-connector assembly using limited dynamic information from its components**



Effective Mass

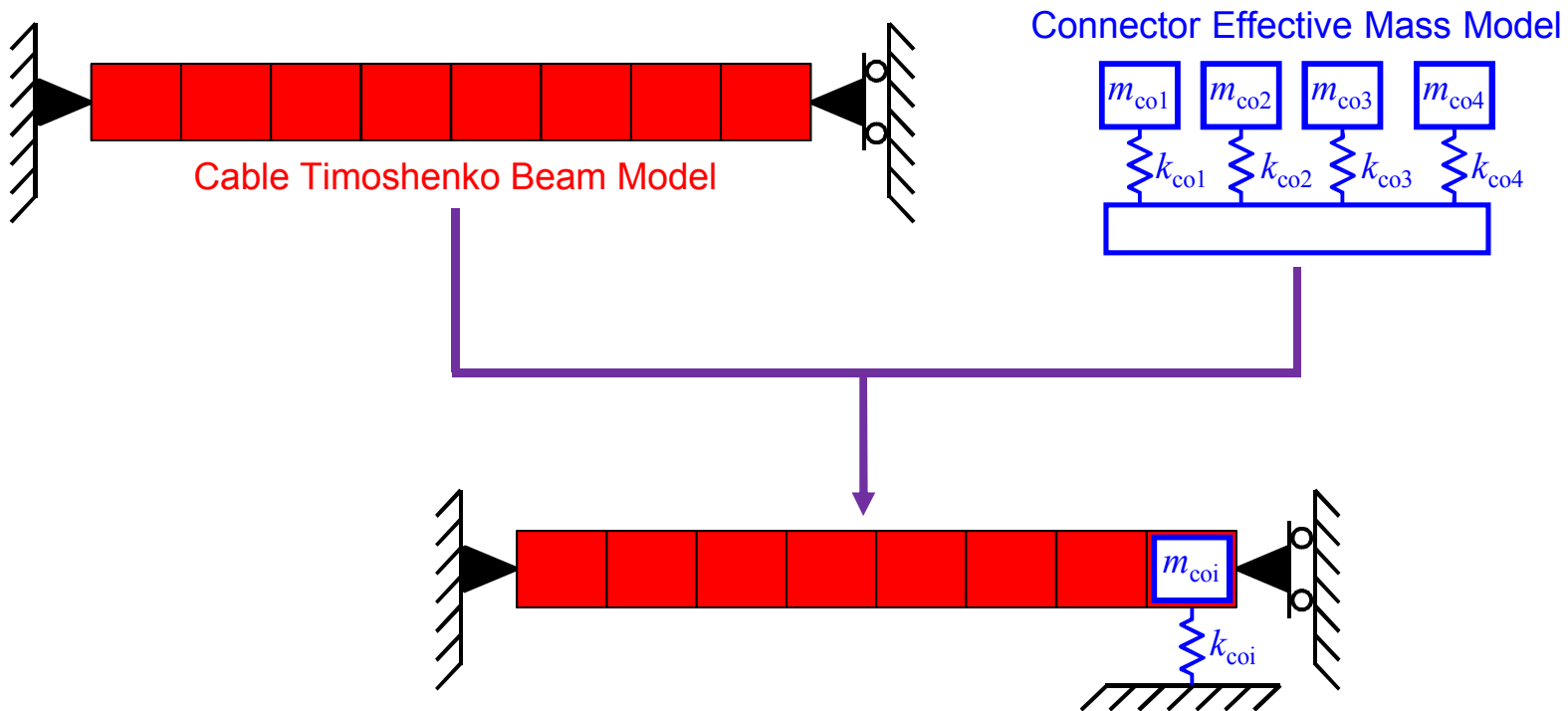
- An effective mass model is a modal model that simulates the response of a component due to a base acceleration input in one direction



- It can be used to calculate the actual energy in the component during the base acceleration environment
 - Useful metric for comparing energy at failure compared to energy in a qualification test (margin)
- Effective mass models can be computed from a FEM or extracted from a modal test of a component on a fixture
- Typically there are three effective mass modal models for a component, one for each of the X, Y, and Z translational directions
 - Normally rotational directions are ignored, since standard vibration table tests are usually focused in one translational direction

Proposed Method

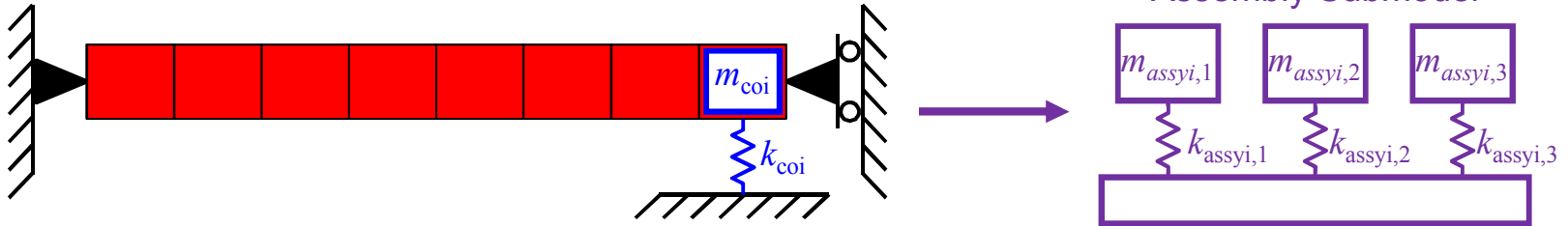
- Create simple numerical Timoshenko beam model for Cable using desired length and stiffness properties (fit from experimental data)
- Extract an effective mass modal model of the Connector from modal test
- Connect an individual Connector effective mass and stiffness to the tip of the Cable



Proposed Method

- Compute the effective mass model of this assembly to produce an Assembly Submodel using:

- $m_i = P_i^2$
 - $P_i = \{\phi\}_R \mathbf{M} \{\phi\}_i$

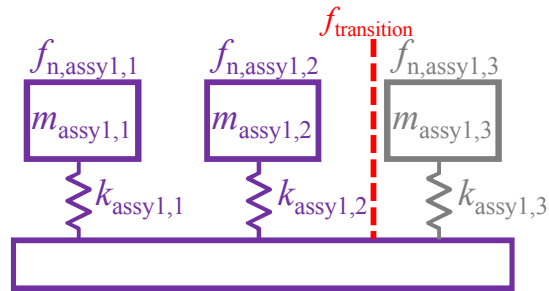


- Repeat this process with the next Connector effective mass and stiffness to create many Assembly Submodels
- Only connector modes with large m_{coi} are used in this process so that the cable tip is loaded as it is in the assembly
 - A connector mode with very small effective mass would essentially produce an assembly submodel of a pin-free cable which does not accurately represent the physical cable-connector assembly

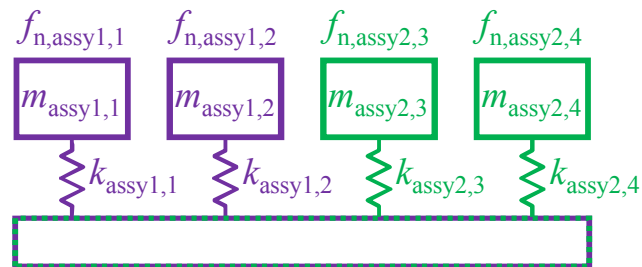
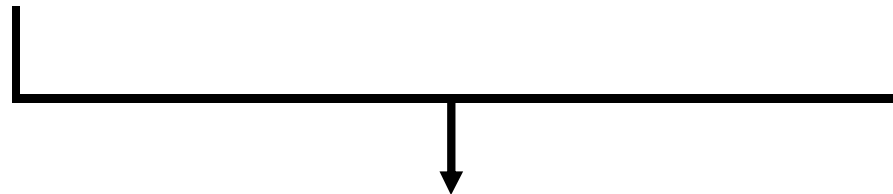
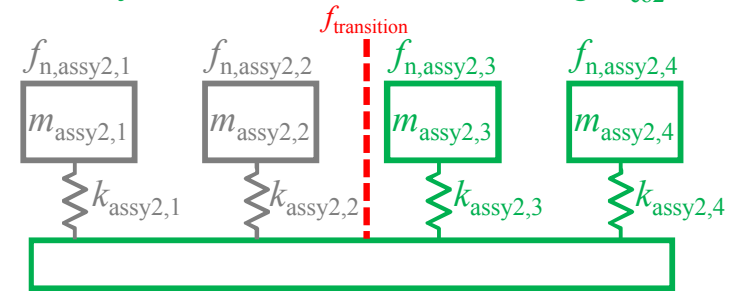
Proposed Method

- Arrange Assembly Submodels to create the final effective mass modal model of the cable-connector assembly (Final Assembly Model)
 - Only modes within a specific bandwidth are kept from each Assembly Submodel

Assembly Submodel 1 Created Using m_{co1} and k_{co1}

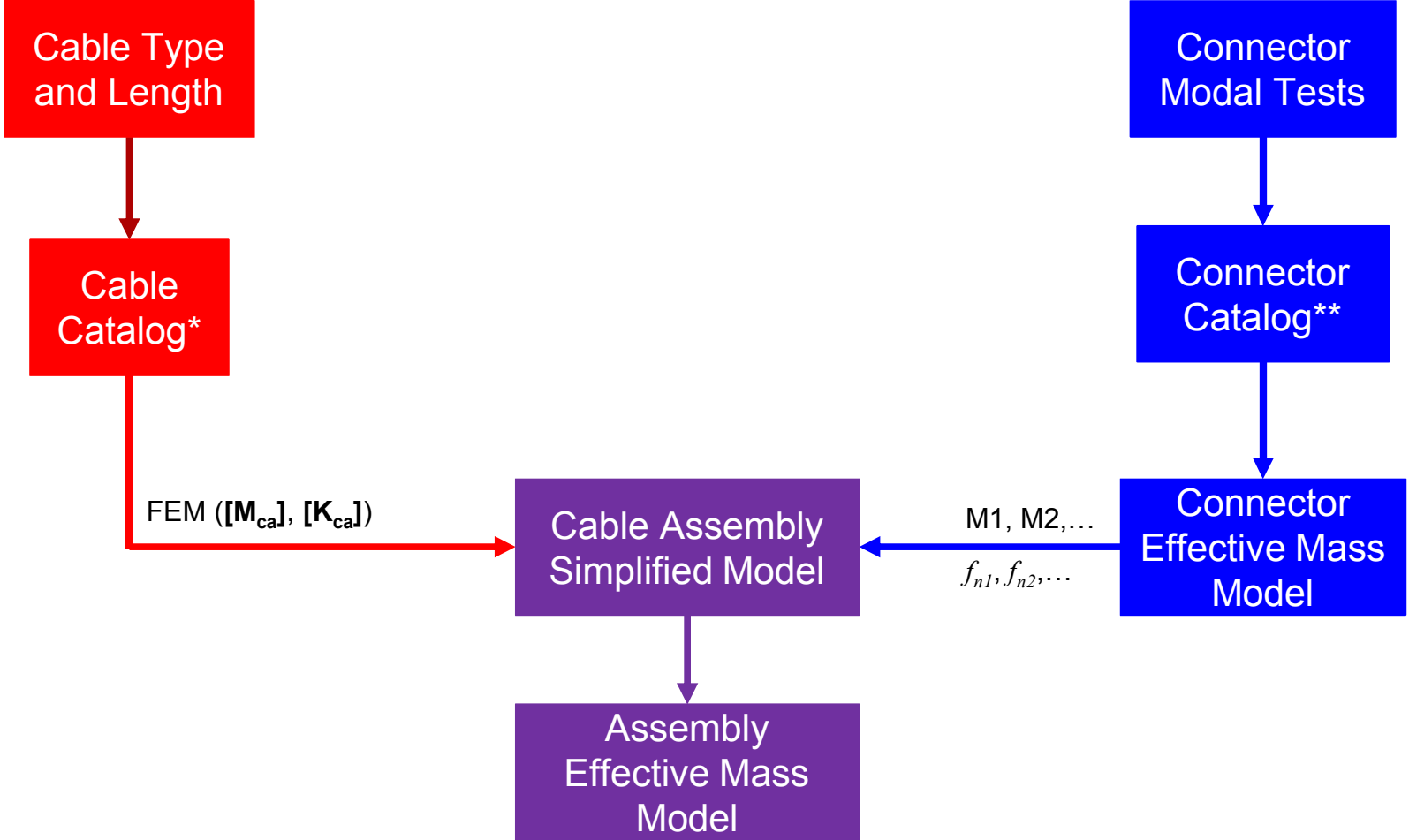


Assembly Submodel 2 Created Using m_{co2} and k_{co2}



Final Assembly Model

Practical Implementation

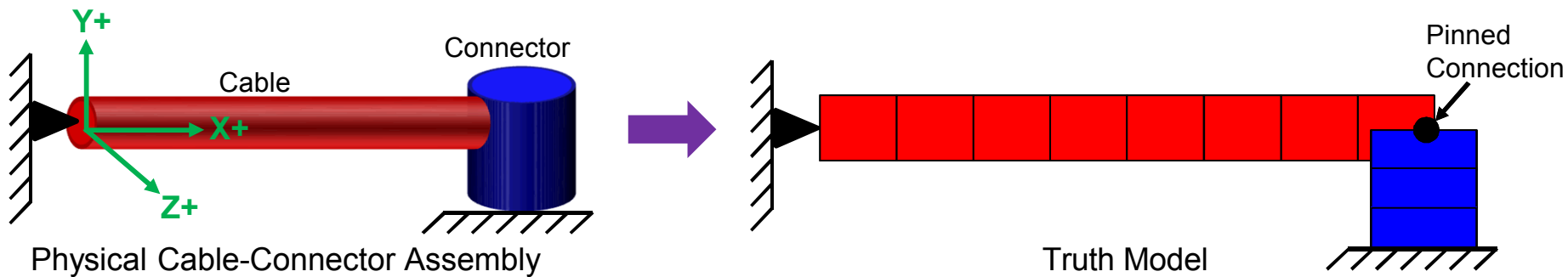


*Catalog consists of EI and KAG values empirically derived from modal tests on different cable types and lengths

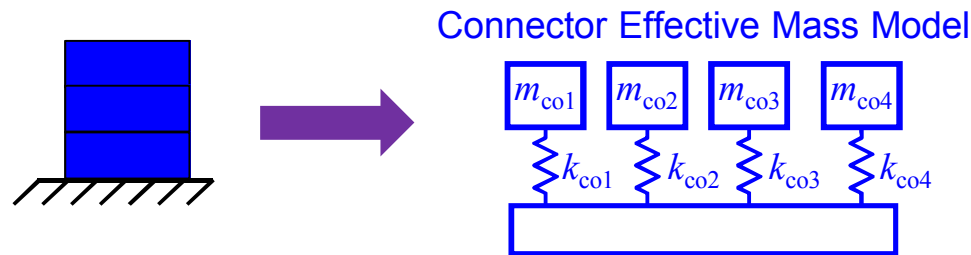
**Catalog consists of modal test results from various connector types

Analytical Verification

- Analytical truth model created to investigate the efficacy of the proposed method
 - Cable was modeled as a Bernoulli-Euler beam
 - Connector represented by cantilevered beam



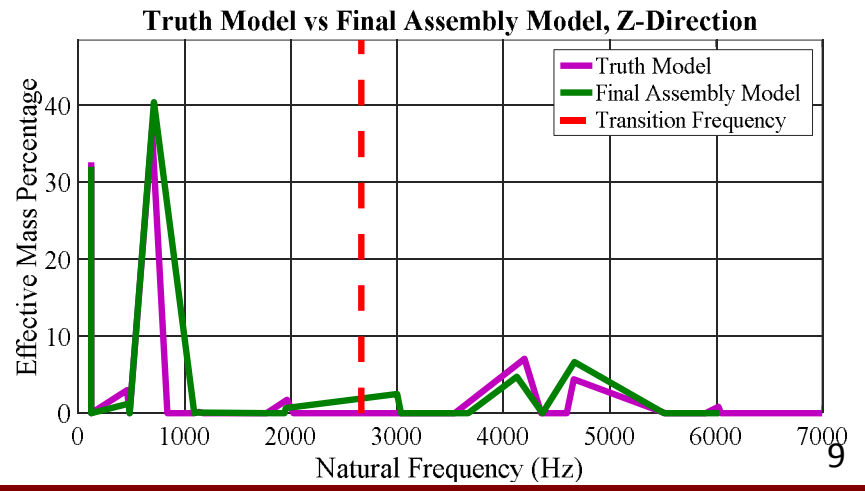
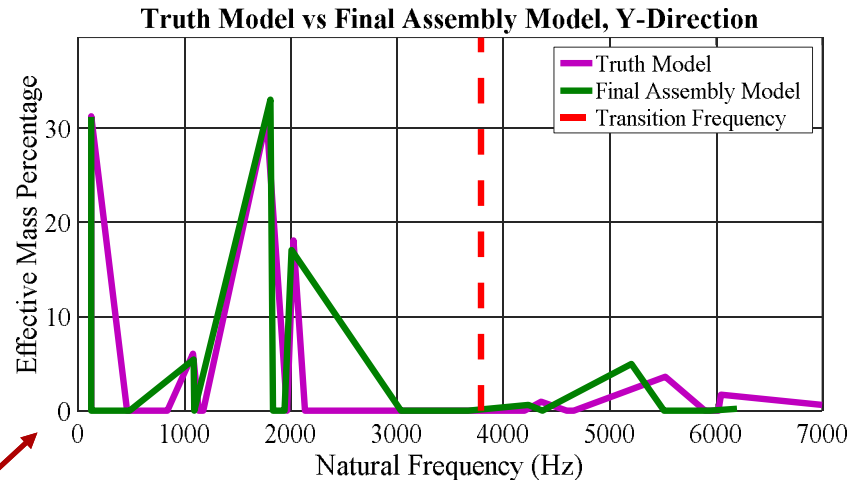
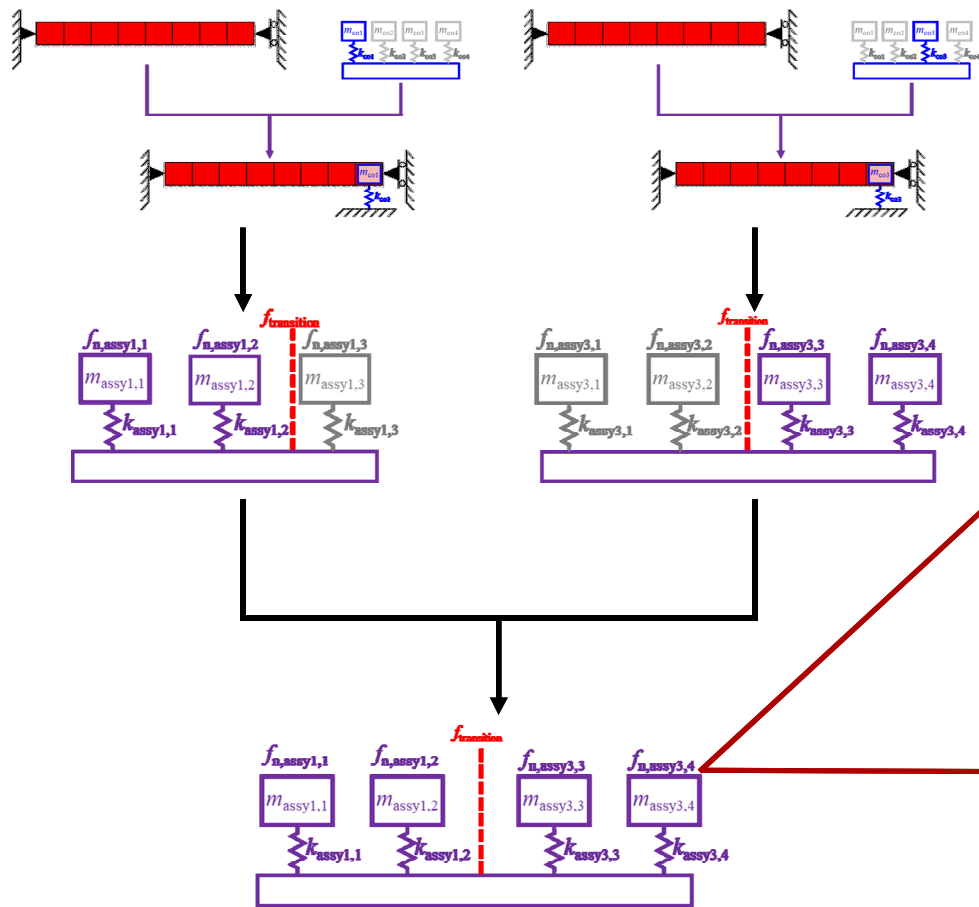
- The effective mass model of the connector was derived from the cantilevered beam used in the truth model



- The focus of this work is on the transverse directions of the cable (Y and Z) as X direction frequencies and effective mass were determined by a different method

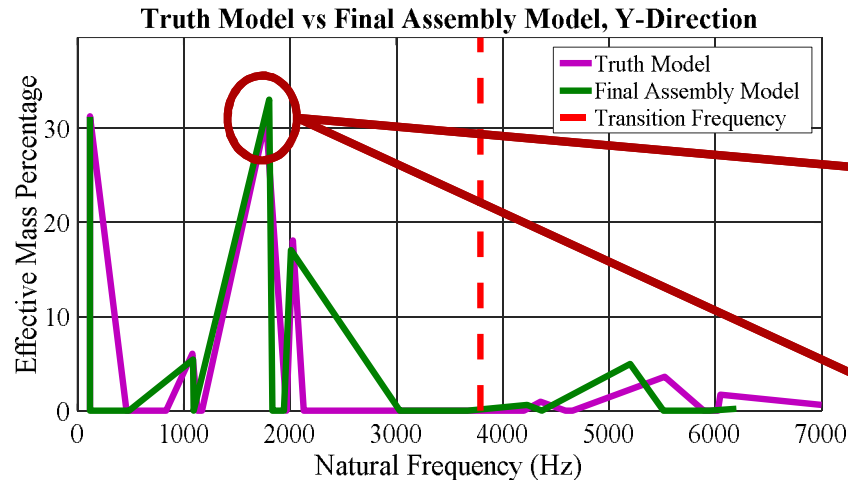
Analytical Verification Results

- For each direction, there were two connector modes with large effective mass below 7000 Hz, so there is one transition frequency

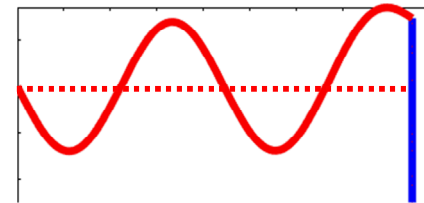


Analytical Verification Results

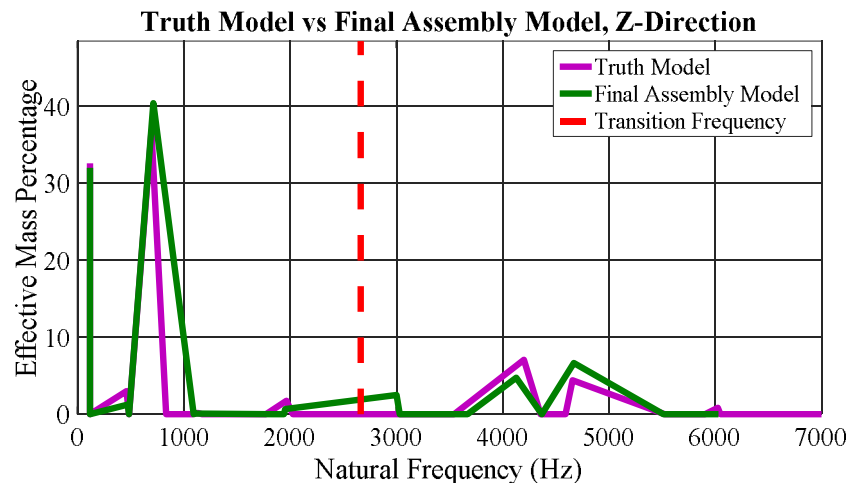
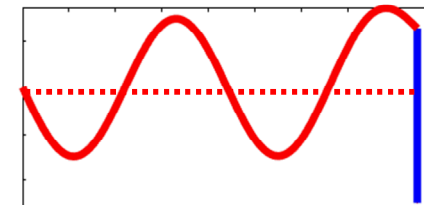
- For each direction, there were two connector modes with large effective mass below 7000 Hz, so there is one transition frequency



Truth Model

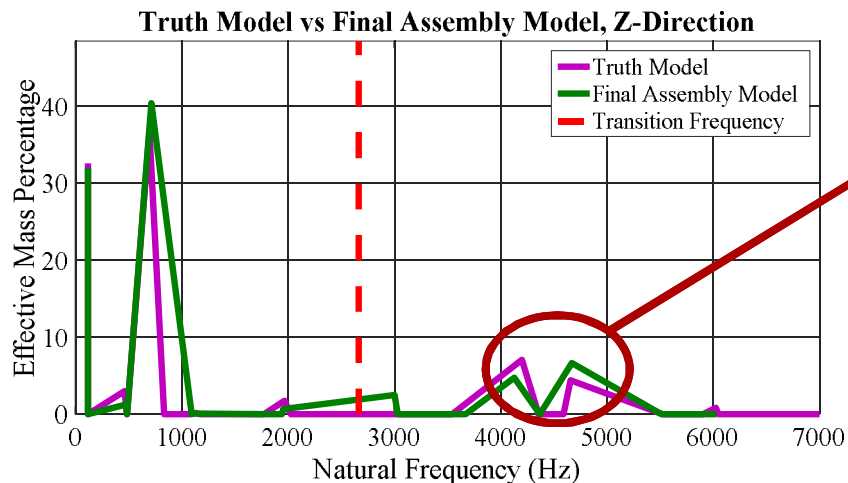
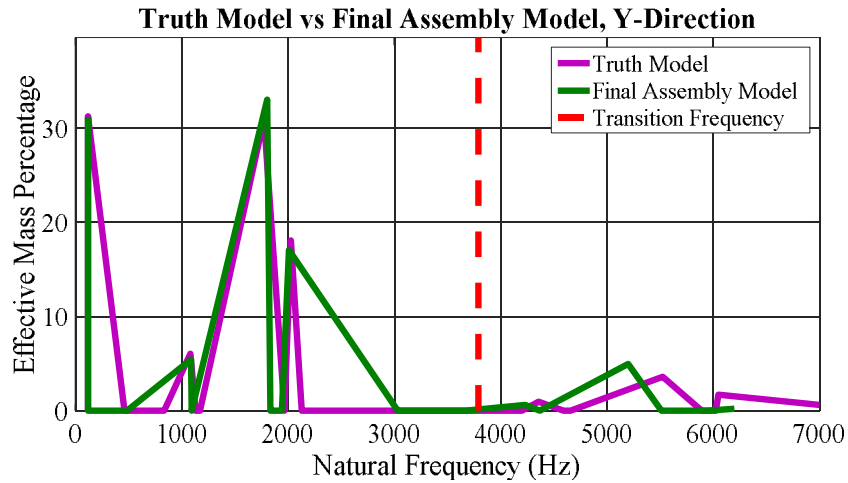


Final Assembly Model



Analytical Verification Results

- For each direction, there were two connector modes with large effective mass below 7000 Hz, so there is one transition frequency



Mode splitting effect, overall energy of these modes captured

Proof of Concept on Physical Hardware

- Previous work by Goodding, et al. shows that cables can be represented by a Timoshenko beam with the rotational inertia term neglected
- We approximated a planar Timoshenko beam by augmenting the Bernoulli-Euler elemental stiffness matrix with

$$\begin{bmatrix} \frac{KAG}{L} & 0 & -\frac{KAG}{L} & 0 \\ 0 & 0 & 0 & 0 \\ -\frac{KAG}{L} & 0 & \frac{KAG}{L} & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

- El and KAG parameters of the cable Timoshenko beam model were fit via sensitivity analysis to experimental modal data of 2 cable lengths

Length (in)	Mode	% f_n Difference Between Experiment and Model
8	1	9
	2	2
	3	-2
	4	5
6	1	-10
	2	0
	3	-11

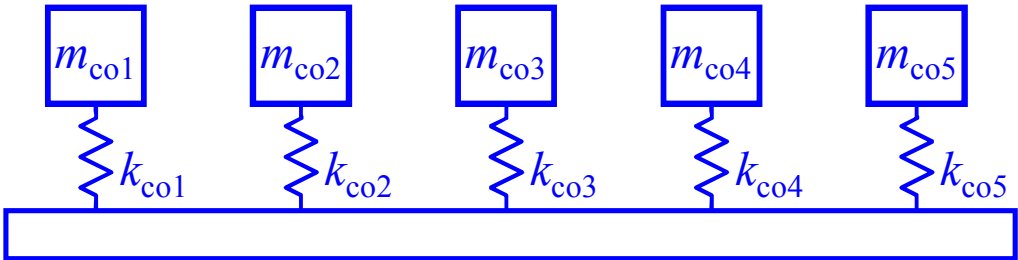
Proof of Concept on Physical Hardware

- The connector effective mass modal model was extracted from modal test data of the connector mounted on a fixture

Mode	Frequency (Hz)	Description	Normalized Effective Mass (%)		
			X	Y	Z
1	768	connector bending in Z	0	2.5	52.3*
2	785	connector bending in X	44.4*	1.4	0
3	1170	cable stub bending in Y	10.1	7	0
4	1264	cable stub bending in Z	0	0.1	0.4
5	1926	connector axial	0	80.2	0.1

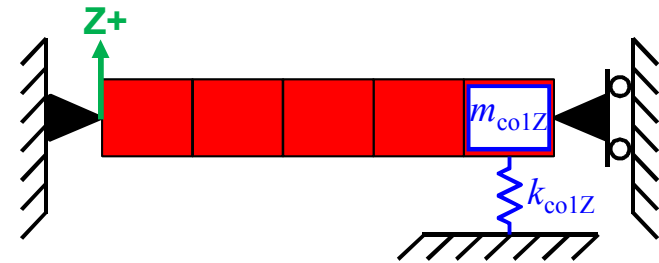
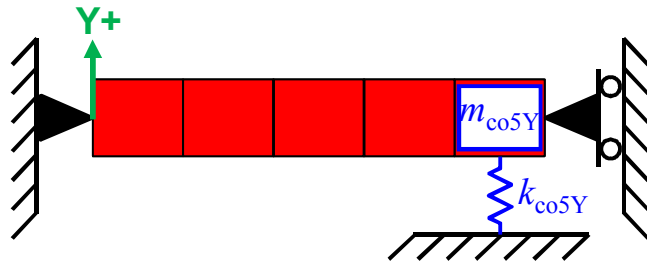
*These values were reconfigured so that effective mass of connector bending is either purely in the X or Z direction

Connector Effective Mass Model



Proof of Concept on Physical Hardware

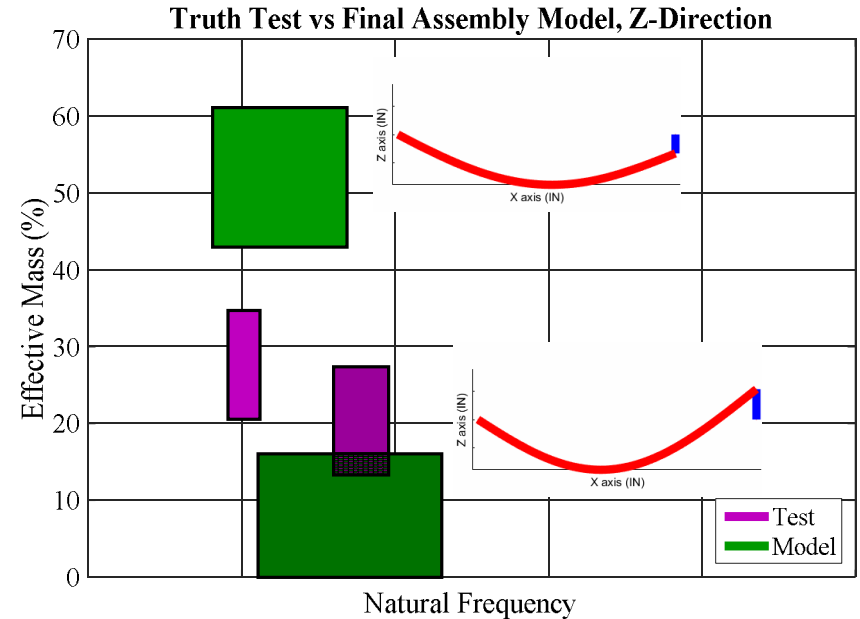
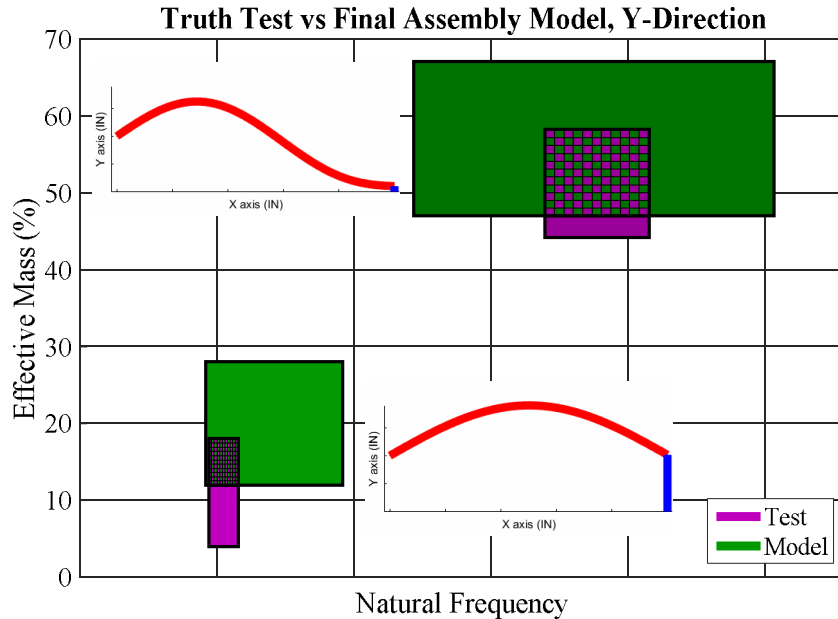
- A numerical model of a cable was created using the fit EI and KAG parameters
- For each direction, this was assembled with the connector mode with the largest effective mass and the final assembly model was calculated



- The results were compared to a truth test
 - Later discovered that the cable length in the truth test was shorter than those used in the EI and KAG fitting process (extrapolation)
- This process included uncertainty bounds based on engineering judgement for the predicted and measured natural frequency and effective mass

Parameter	Source		
	Connector Experiment	Max Error from Analytical Verification Results	Unit-to-Unit Variability
Frequency	±10% of natural frequency	±6% of natural frequency	±30% of natural frequency
Effective Mass	±7% of total mass	±3% of total mass	±10% of calculated effective mass

Proof of Concept Results



- Cable mode shape in model matches that seen in experiment except for the Connector Axial mode
 - Experiment shows cable in 3rd bending, but the model has 1.5th or 2nd bend
- Mode splitting effect for the Z-direction modes
 - Individual mode-to-mode effective masses are inaccurate, but the overall energy of the two modes is captured

Conclusions

- Ideally the final assembly model results would totally encompass the test results, but the proposed method shows promise
- Generally, the predicted mode shape of the cable matches that seen in the truth test
 - Exception: connector axial mode
- Mode splitting when connector modes are close to cable modes can cause large errors in effective mass for a single mode, but the overall dynamic effect of both modes is captured
- The accuracy of the proposed method is greater for cable-connector assembly modes with large effective mass
- Potential causes for error and areas for further improvements
 - Better quantification of uncertainty
 - More accurate model form?