

# Efficiency Benefits of Lean or Dilute Well-mixed DISI Engine Operation with Advanced Ignition, Intake Heating and Various Fuel Types

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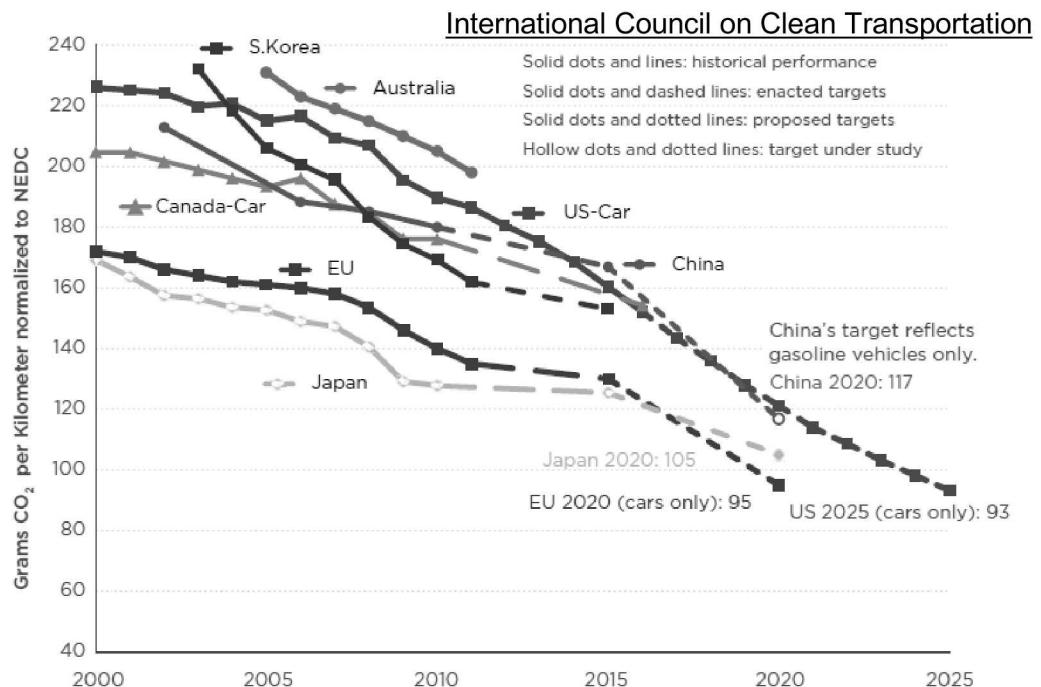
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## Acknowledgement

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# Introduction

- Strong pressure to reduce CO<sub>2</sub> emissions.
- Improved engine efficiency is one key factor.
- Stoichiometric SI operation is standard for gasoline-type engines.
- **Lean and/or dilute SI operation** can improve fuel economy.
- Combustion stability is one key issue.
- World-wide mandates for renewable fuels.
- Motivate examination of **ethanol fuel blends** for advanced combustion.



- Fuels: Gasoline, E30 and E85
- Fuel/air-mixing states:
  - Well-mixed charge.
- Ignition methods:
  - Regular high-energy inductive spark system (106 mJ electrical).
  - Multi-pulse (MP) transient plasma (16mJ electrical).
- Dilution types & fuel/air ratio:
  - Excess air.  $\phi \leq 1$
  - Simulated EGR using  $N_2$ .  $\phi = 1$ , stoichiometric conditions.
- Combustion modes:
  - Deflagration.
  - Deflagration  $\Rightarrow$  end-gas autoignition.

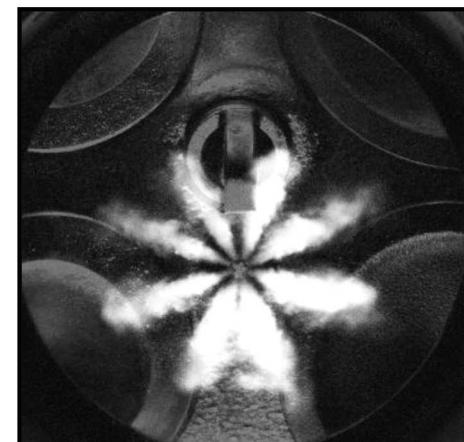
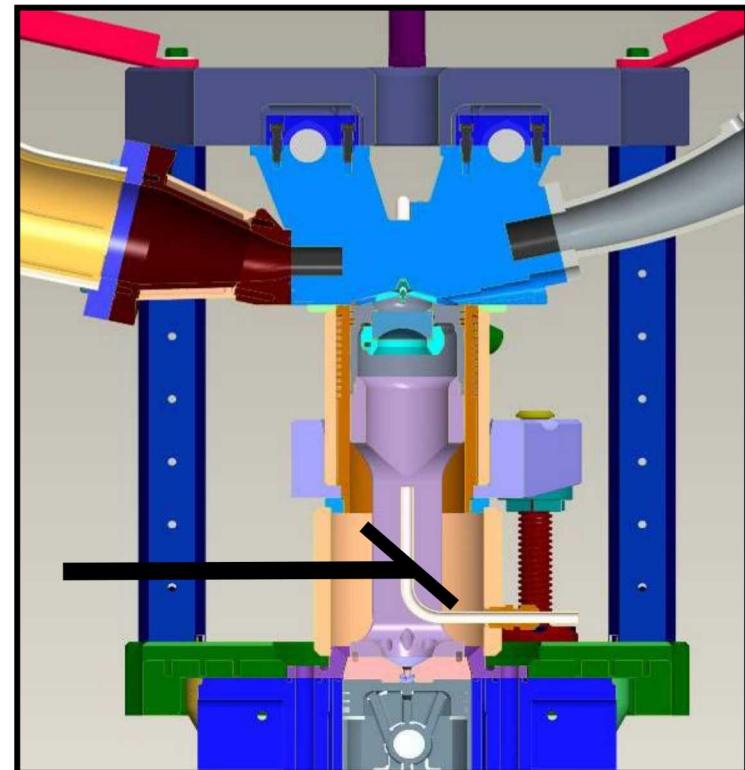
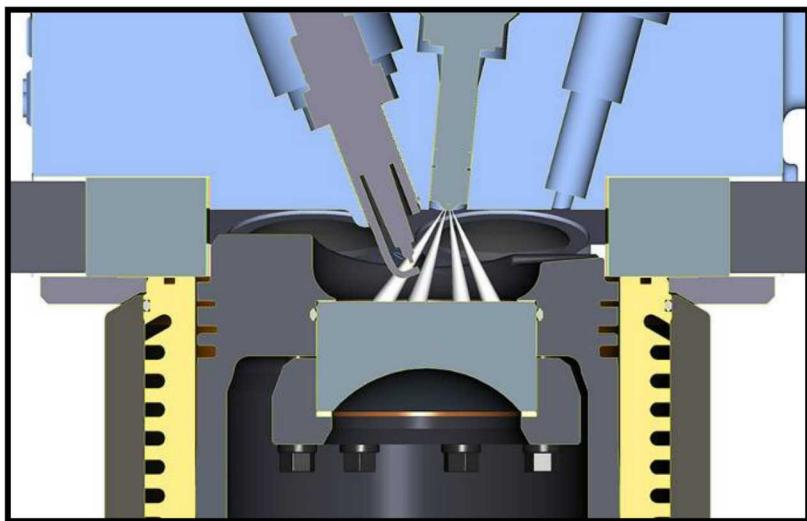
# Objectives

- Examine selected data sets to identify opportunities for improved thermal efficiency relative to that of traditional stoichiometric operation.
  - A. Lean well-mixed SI operation using regular spark.
    - Gasoline & E85: Fuel effects?
    - Enabling factors:
      - Intake charge heating.
      - Mixed-mode combustion (deflagration  $\Rightarrow$  autoignition)
    - Limitations:
      - Combustion variability.
      - Combustion inefficiency.
      - Lack of combustion-phasing control.
    - $\text{NO}_x$  emissions / dilute operation.
  - B. Opportunities with enhanced ignition.
    - Multi-pulse (MP) transient plasma.
    - Quantify control authority.
  - Combine:
    - All-metal performance testing with emissions measurements.
    - High-speed imaging of spray, ignition and flame development.

# Research Engine

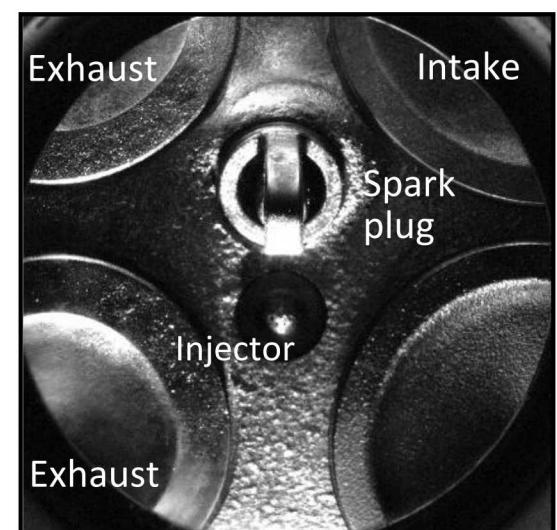
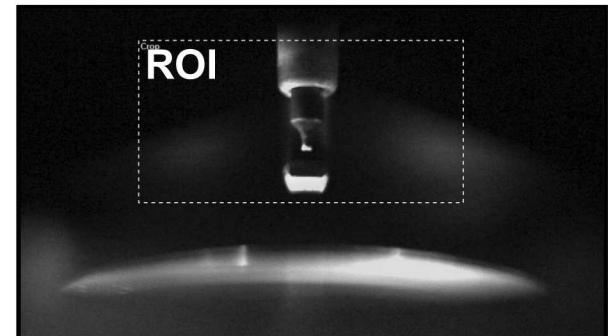
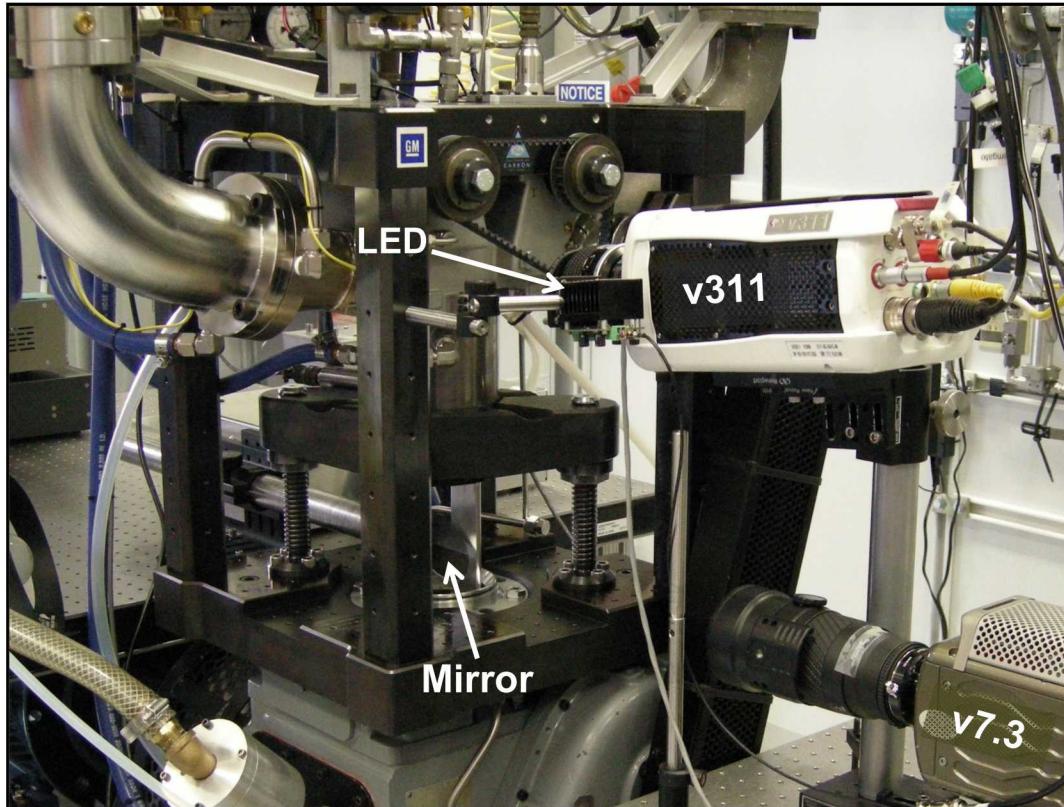


- Drop-down single-cylinder engine.
- Automotive size. 0.55 liter swept volume.
- Identical geometry for **All-metal** and **Optical**.
- Designed for spray-guided stratified-charge operation  $\Rightarrow$  Piston bowl.
- Injections during intake stroke  $\Rightarrow$  well-mixed charge.



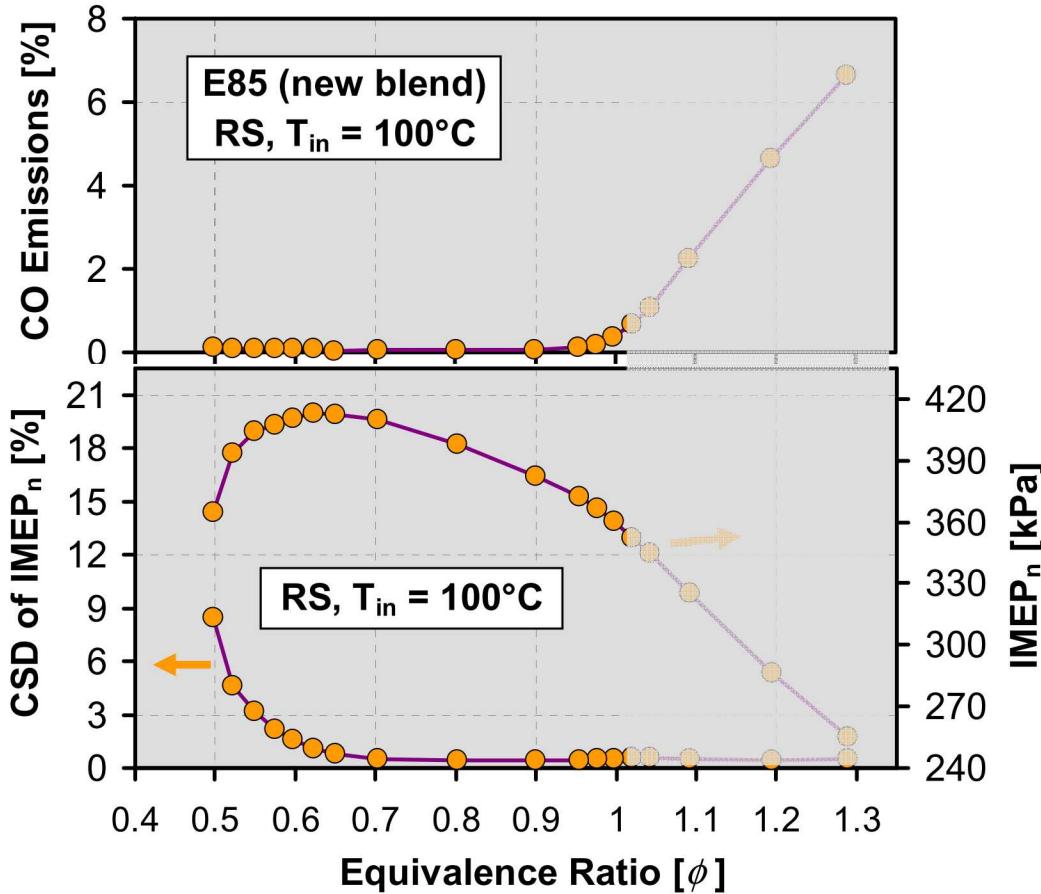
# Imaging Setup

- High-speed imaging: 20 kHz.
- Phantom v7.3 & v311, monochrome.



# Operating Conditions

- Constant fuel mass  $\Rightarrow \text{IMEP}_n$  reveals efficiency changes.
- Focus on lean combustion.



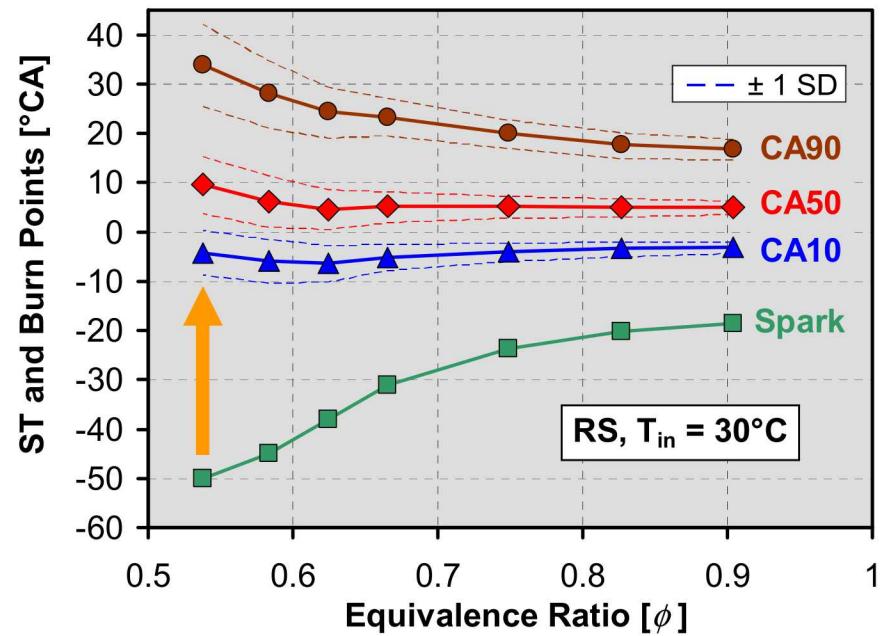
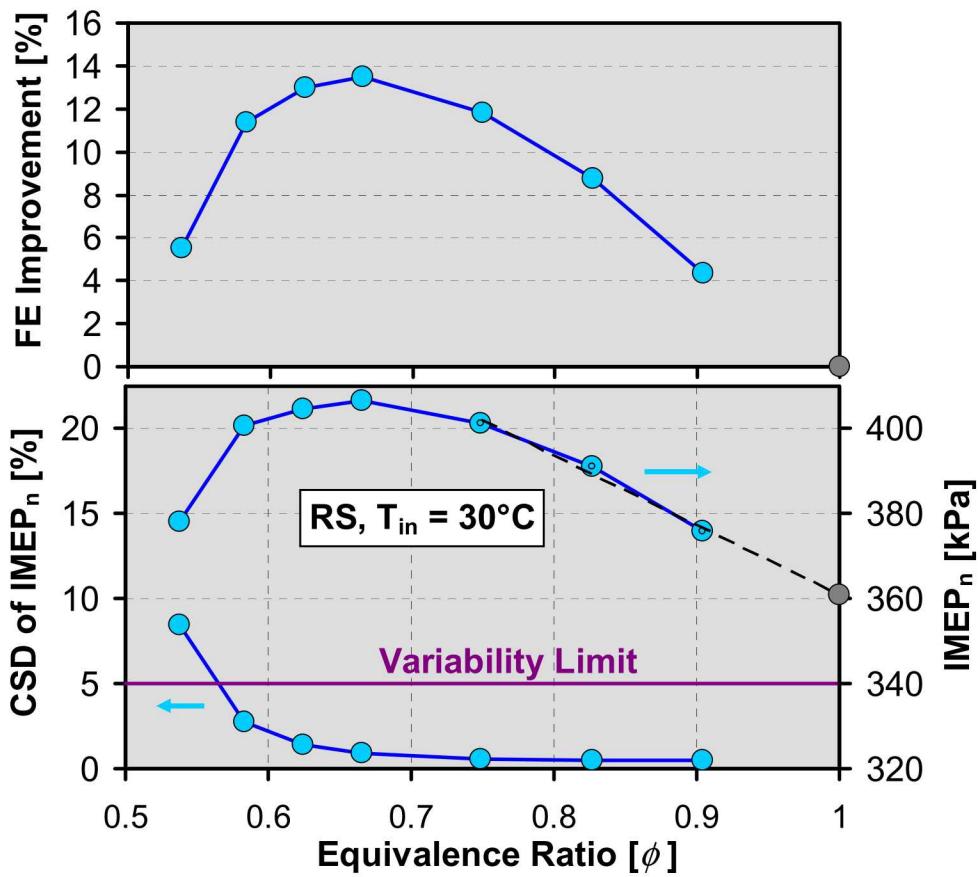
Parameter	This Data Set
Fuel Type	E85
Engine Speed	1000 rpm
Fuel per Cycle	21.6 mg
Intake Pressure	39 - 88 kPa
Target CA50	5°C

- Grouped as hardware, static parameters & operating variables.
- Low residual gas level, 4 – 6% by mass.
- Fuel mass adjusted to  $IMEP_n = 360 - 370 \text{ kPa}$  for  $\phi = 1$  operation.

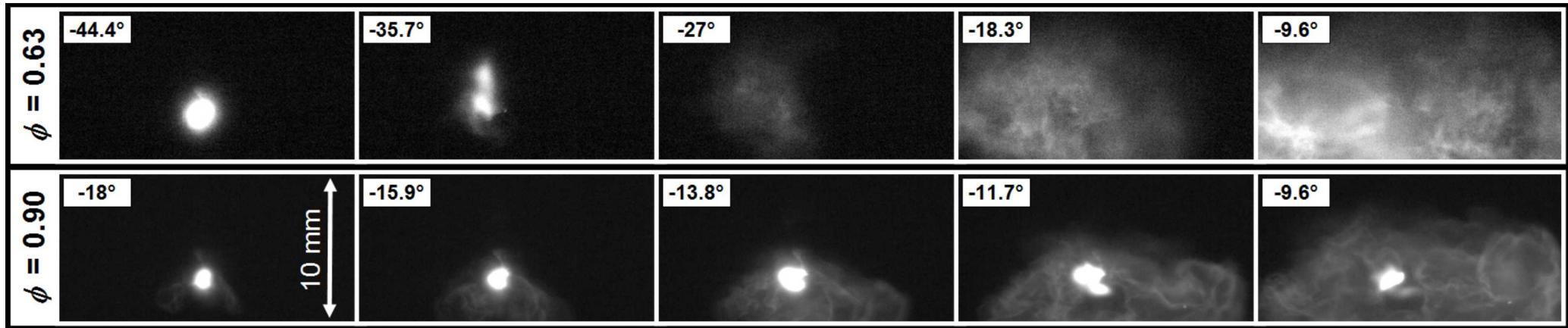
Parameter	This Presentation
CR	12
Piston Bowl	$\varnothing 46 \text{ mm}$
Valve Timings	For Minimal Residual Level
Injector & Spray Targeting	Bosch 8 x 60° Straddling Spark
Swirl Index	2.7
Tumble Index	0.62
Engine Speed	1000 rpm
Injection Pressure	170 bar
$P_{\text{exhaust}}$	100 kPa
$T_{\text{coolant}}$	75°C
$IMEP_n, \phi = 1$	360-370 kPa
Fuel Type	Gasoline, E30, E85
Fuel Mass	15.6, 17.8, or 21.6mg
$\phi$	1.3 - 0.44
Intake Pressure	37 – 96 (107) kPa
$T_{\text{in}}$	30°C or 100°C
EGR / $[\text{O}_2]_{\text{in}}$	11.9 - 21% $\text{O}_2$

# $\phi$ - Sweep, E85, Regular Spark (RS)

- IMEP<sub>n</sub> increases by 13% at  $\phi = 0.67$ .
- Lower  $\phi \Rightarrow$  Unacceptable IMEP<sub>n</sub> variability.
- Spark – CA10 induction period becomes excessive.

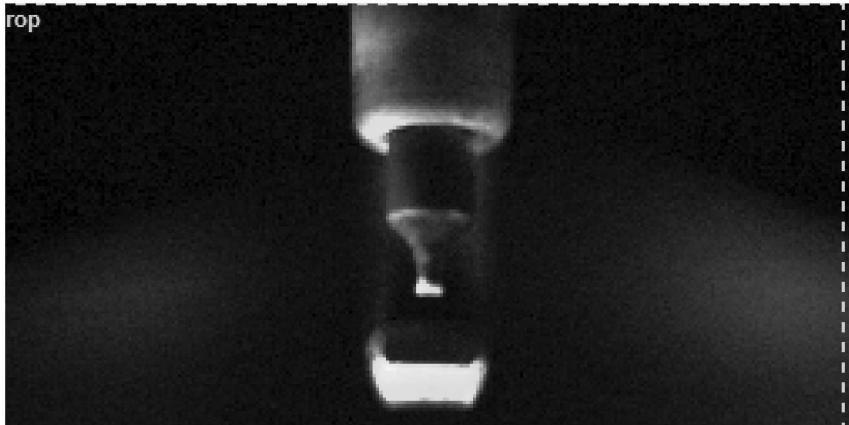


- Lean operation  $\Rightarrow$  fragile flame kernel.



# Spark Discharge Variations

- Flow field has cycle-to-cycle variability.
- Evidenced by plasma-channel variations.



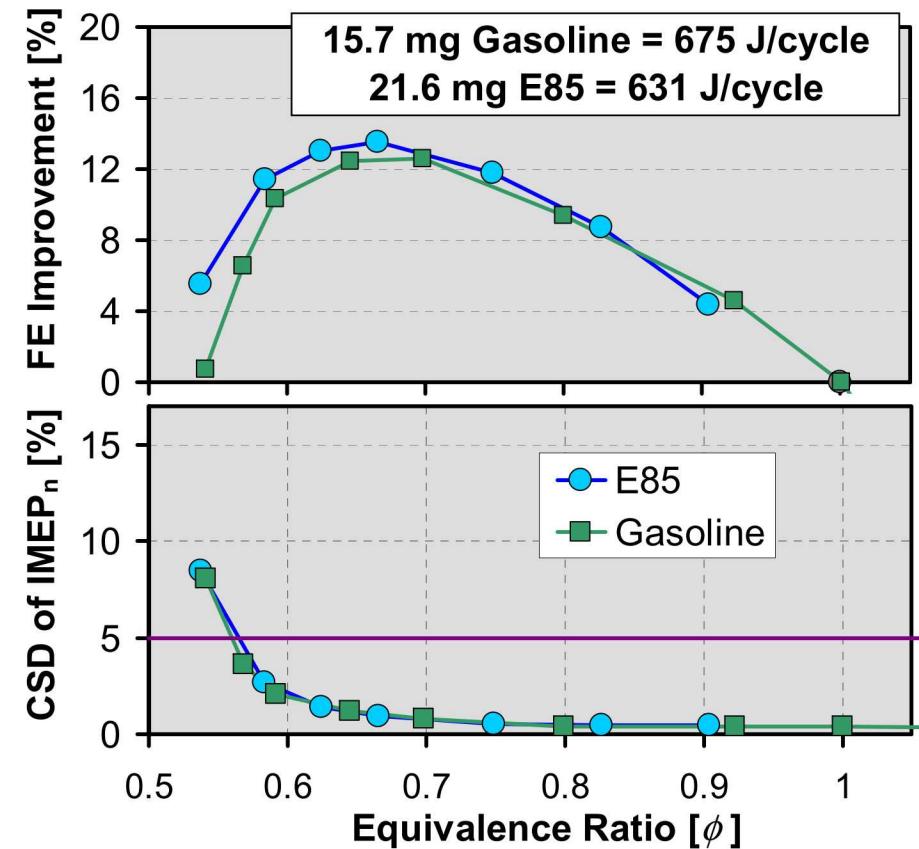
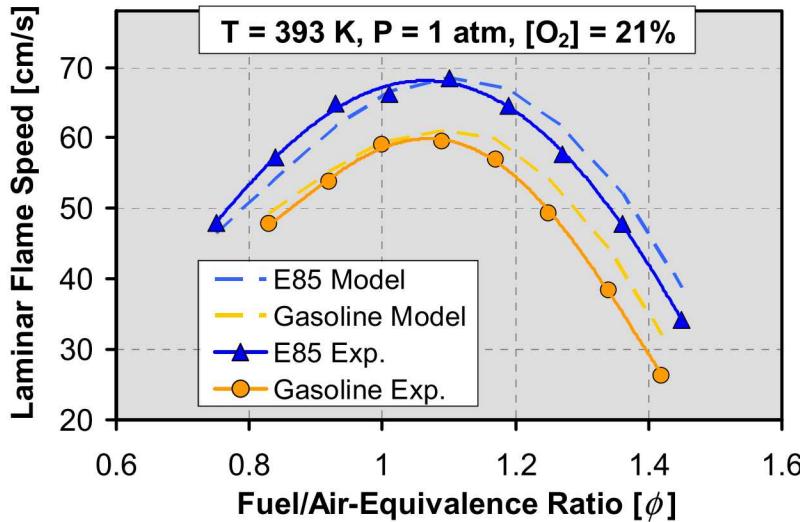
- Contributes to cycle-to-cycle variability for lean operation.
  - Fragile flame kernel.
  - Long induction time from spark to CA10.

# $\phi$ - Sweeps, Regular Spark, E85 and Gasoline



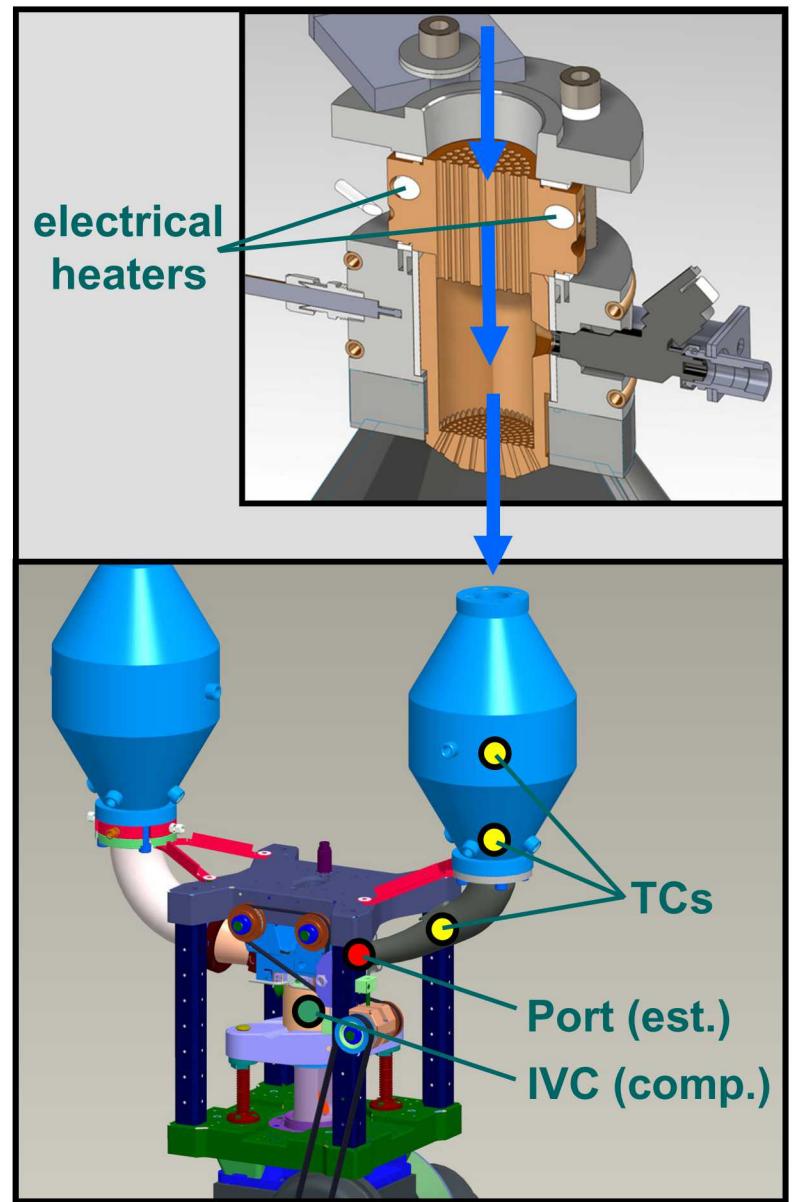
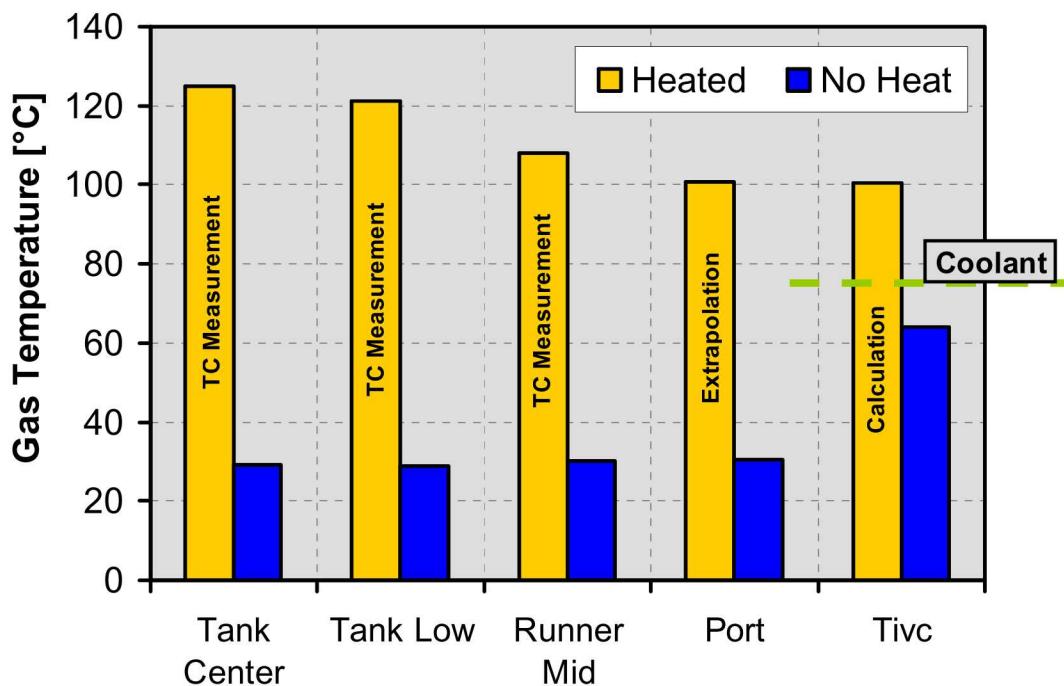
- $T_{in} \approx 30^\circ\text{C}$ . Identical lean instability limits for gasoline and E85.
- Flame measurements and modeling suggest stronger E85 flame.
- Recent modeling by Marco Mehl at LLNL shows that flame speeds of early flames for gasoline and E85 become nearly identically low for  $\phi < 0.6$ .
- Also possible that stronger DI vaporization cooling effect of E85 counteracts a slightly stronger E85 flame.
- FE gains are similar for gasoline and E85.

Zhao, Egolfopoulos, Mehl & Pitz

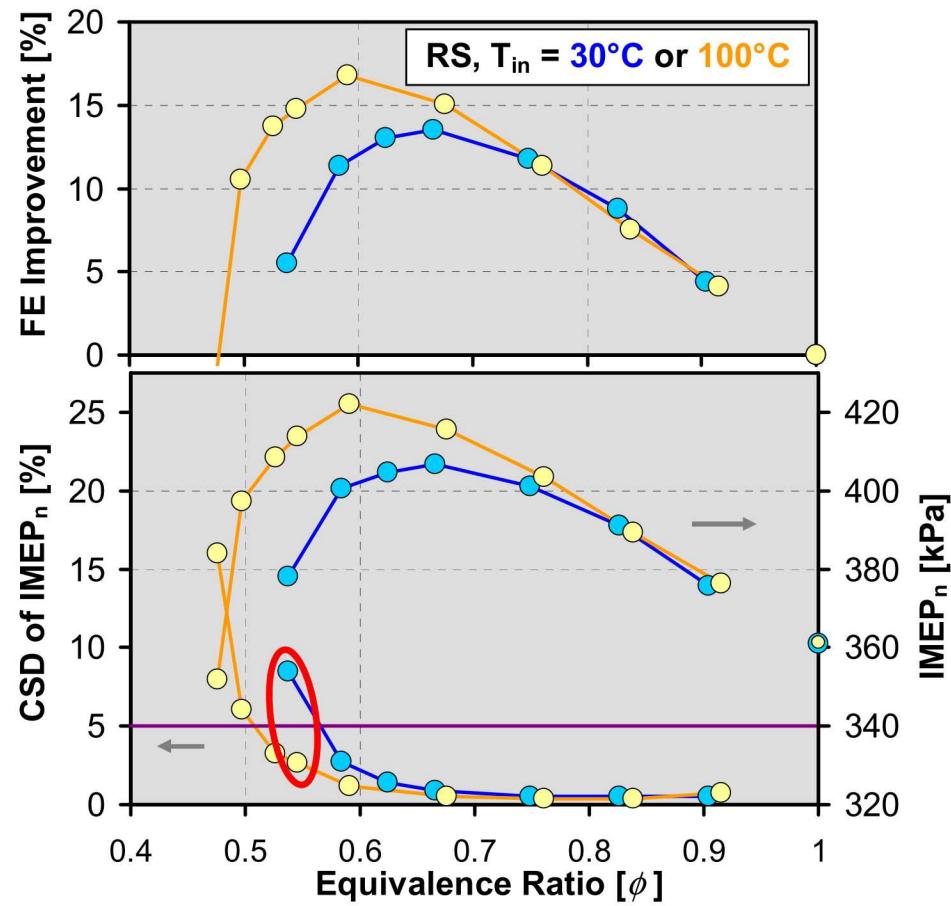
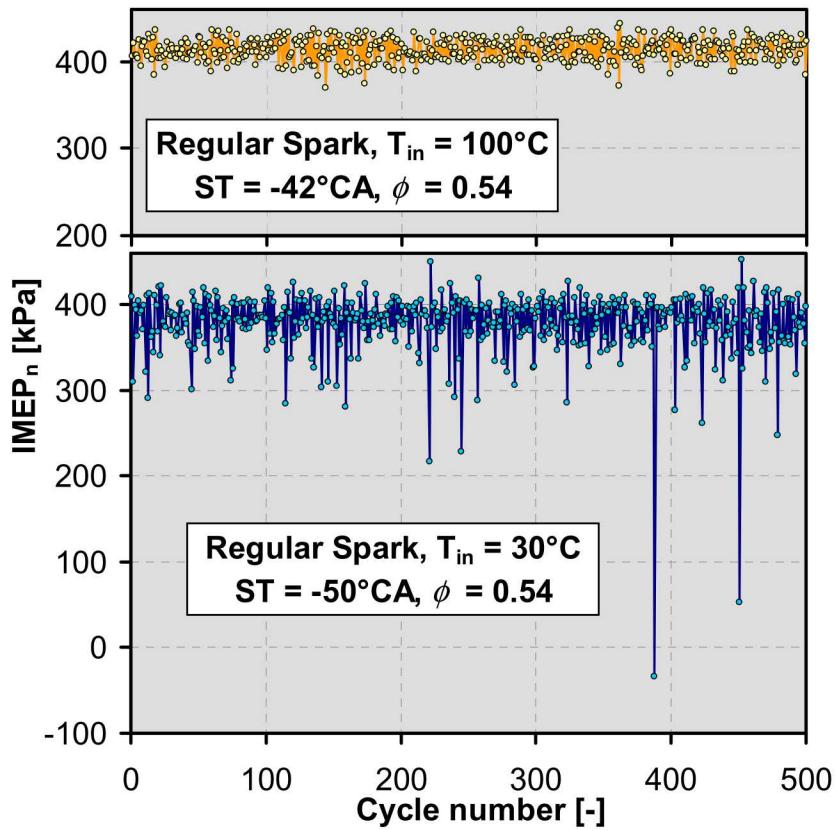


# Intake Air Heating

- Electrical intake air heater.
- Three thermocouples  $\Rightarrow$
- Estimate port temp. Rises 70 K.
- Also, compute  $T_{IVC}$ . Rises only  $\approx 40$  K.
  - Heat-transfer during intake stroke.
  - Mixing with residuals ( $\approx 5\%$ ).

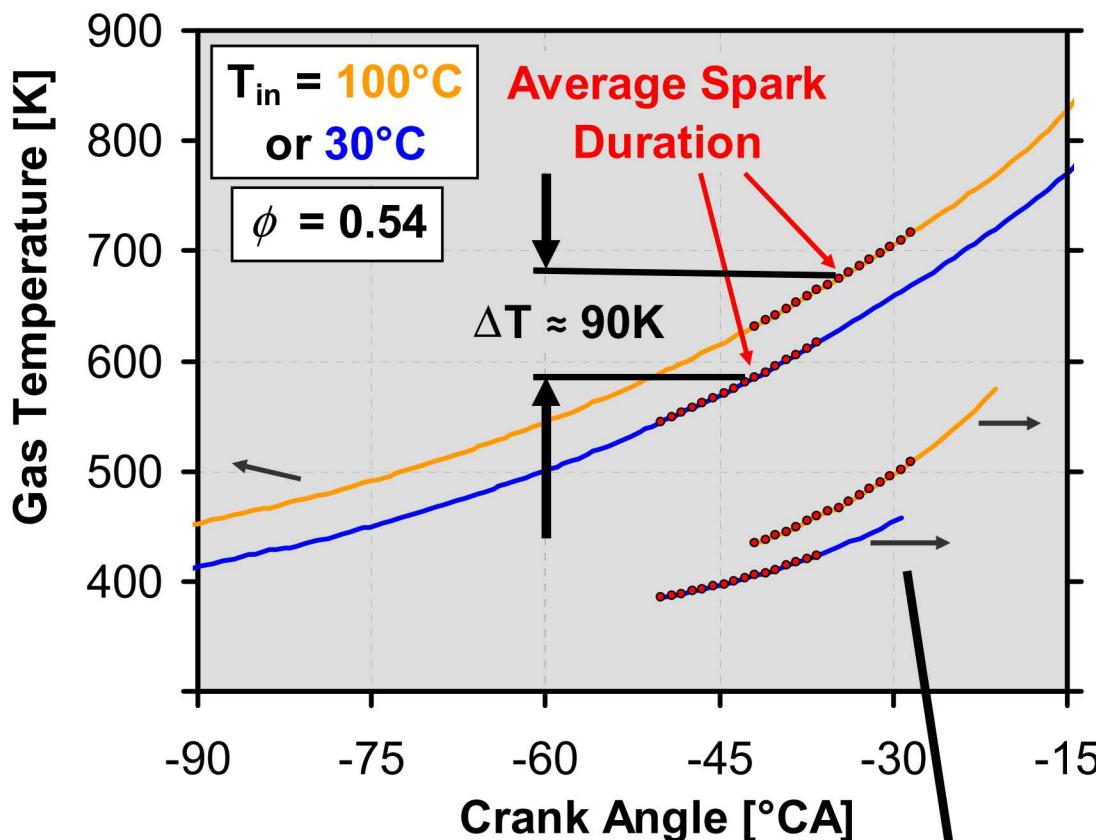


- Higher  $T_{in}$  improves lean operation.
  - Higher FE gain of 17%.
  - Leaner stability limit.
- Remarkably strong effect of 40K.



# E85, Combined ST & $T_{in}$ Effects

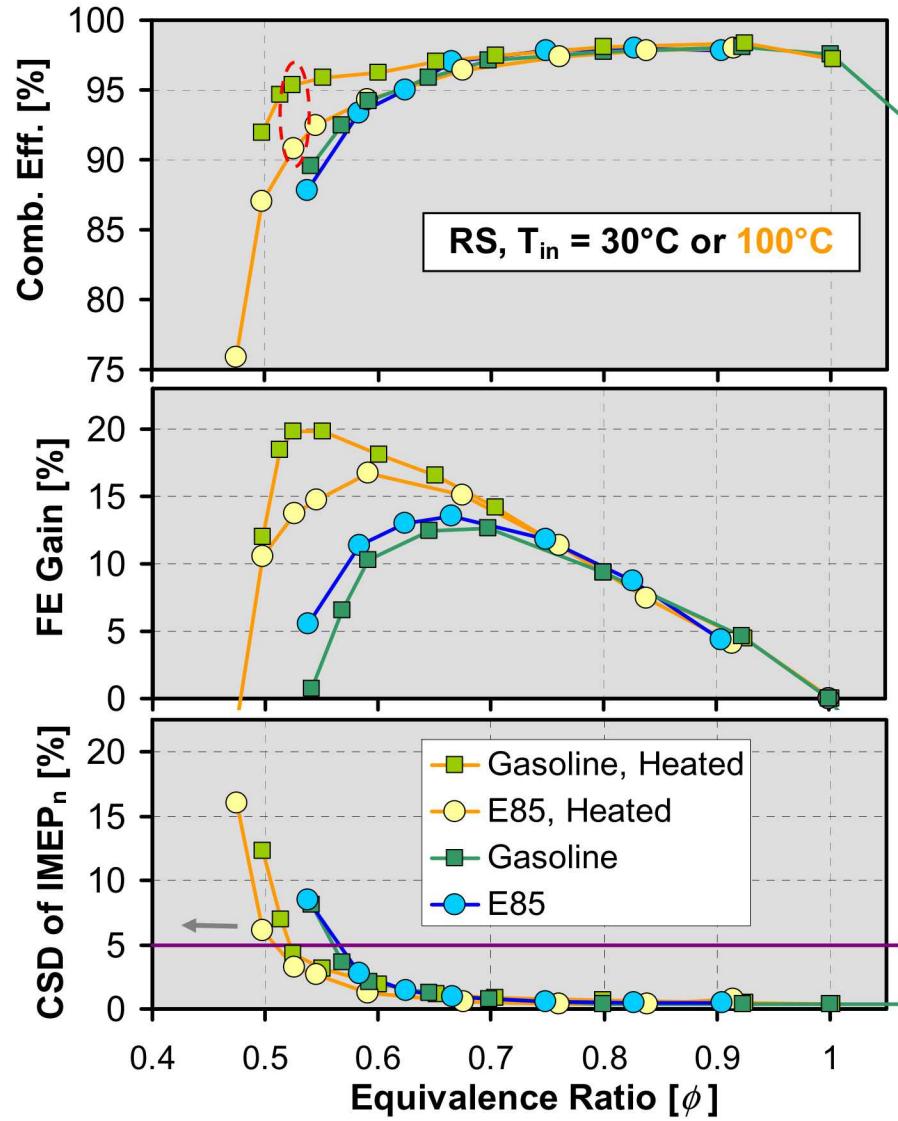
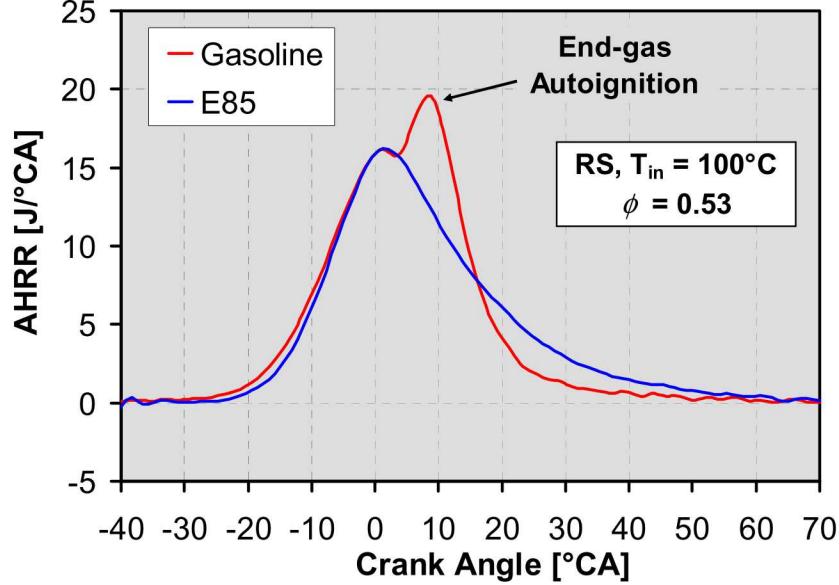
- Higher  $T_{in}$  allows later spark  $\Rightarrow$  Enhances effect of higher  $T_{in}$ .
- $\Delta T$  during spark event is  $\approx 90K$ .
- Large relative increase of  $S_L$ .



Courtesy of M. Mehl at LLNL

# $T_{in}$ Effect Gasoline

- Higher  $T_{in}$  benefits gasoline variability slightly less than E85.
- For FE gain, increased  $T_{in}$  benefits gasoline more: +20%.
- Maintain CE >95% down to  $\phi = 0.53$ .
- Gasoline is relatively reactive.
  - Repeatable endgas autoignition.



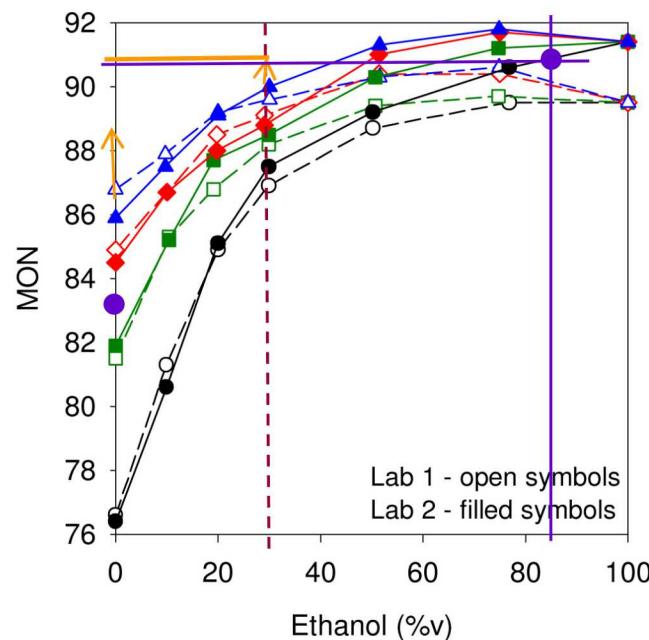
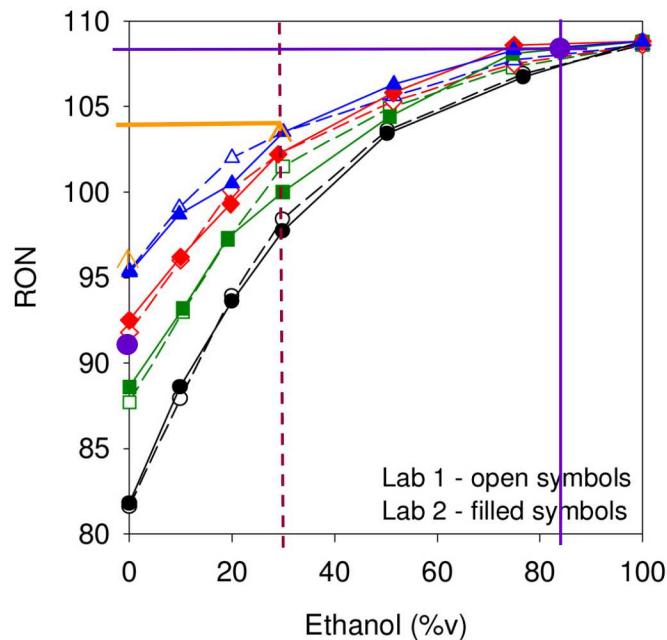
# Octane Numbers

- High-octane certification gasoline.
  - Used for all gasoline data here.
- **E30** was blended of anhydrous ethanol and certification gasoline.
  - 30% / 70% blend ratio  $\Rightarrow$  E30.
- **E85** was blended from another lower-octane gasoline.
- Estimate RON & MON from SAE Paper 2012-01-1274, Anderson *et al.*

RD3-87



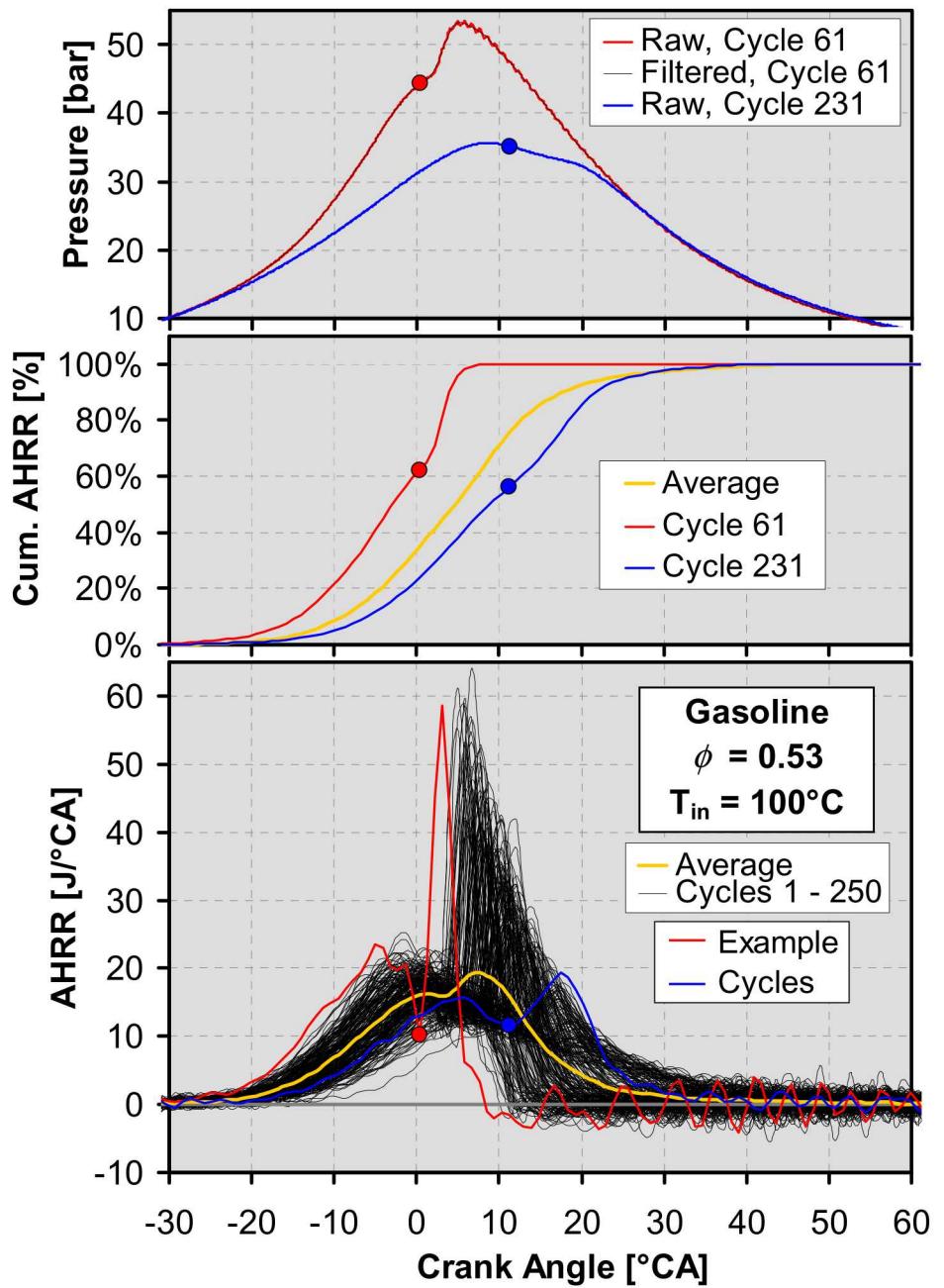
	<u>Gasoline</u>	$\Rightarrow$ <b>E30</b>	<b>E85</b>
AKI (R+M)/2	92.7	97.5	$\approx$ 99.5
RON	96.6	104	$\approx$ 108
MON	88.7	91	$\approx$ 91
A/F Stoich.	14.6	12.8	9.8



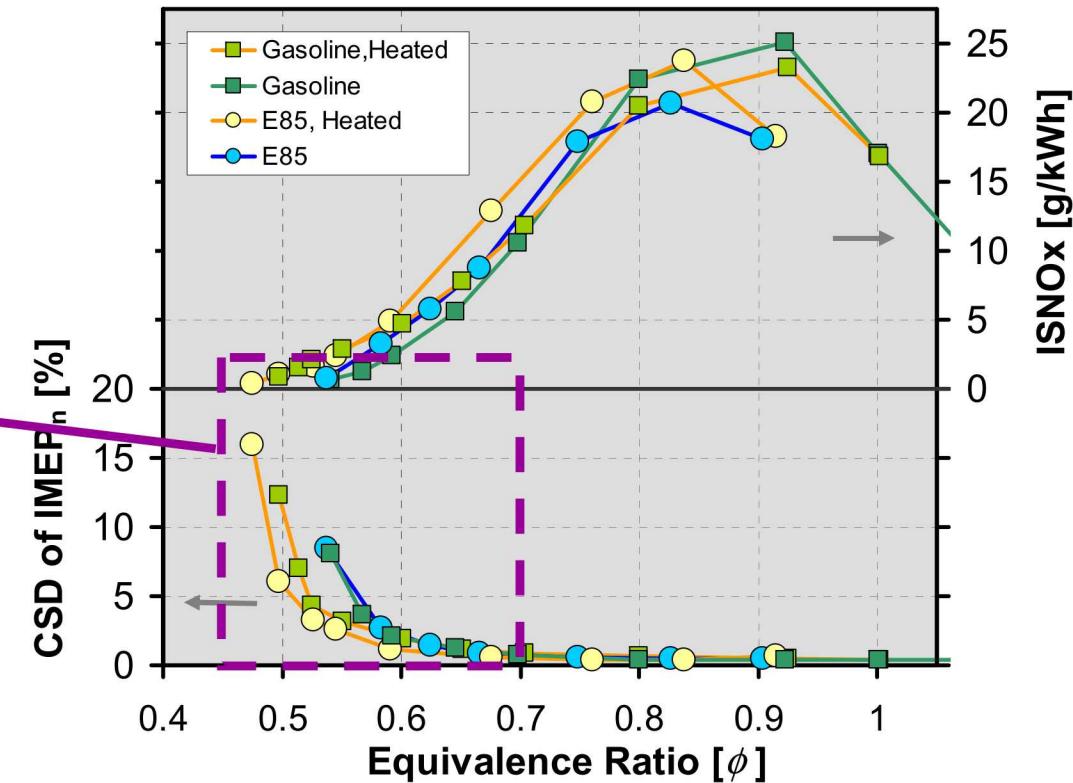
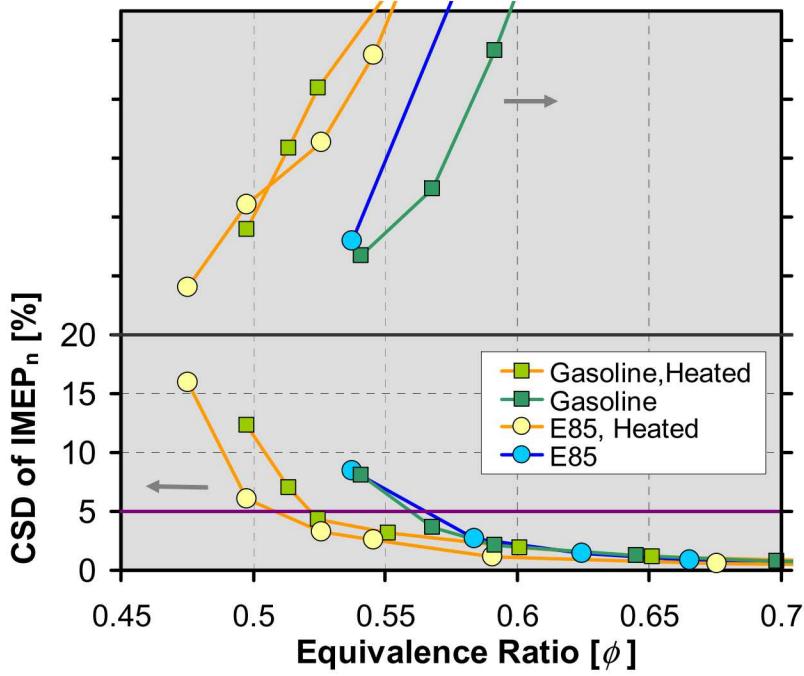
# AHRR Lean Heated Gasoline



- Even with MON = 88.7, gasoline is sufficiently reactive.
- Deflagration  $\Rightarrow$  end-gas autoignition.
- Like SACI, but coming from SI side.
- Deflagration accounts for  $\approx 60\%$  of combustion event.
- Pressure ripple is mild even for the most advanced cycles.
- More repeatable deflagration should stabilize endgas autoignition.
- Here, ST =  $-53^{\circ}\text{CA}$ .
  - Long inflammation period.
- Later spark with an “enhanced” ignition system should reduce variability.



- NO<sub>x</sub> trends and levels are comparable for gasoline and E85.
- Only beyond the lean stability limit of 5% are we reaching less than 1 g/kWh.
- 1 g/kWh = 4 g/kg gasoline.
- Lean NO<sub>x</sub> aftertreatment may be challenging.
- Revert to dilute  $\phi = 1$  with TWC?

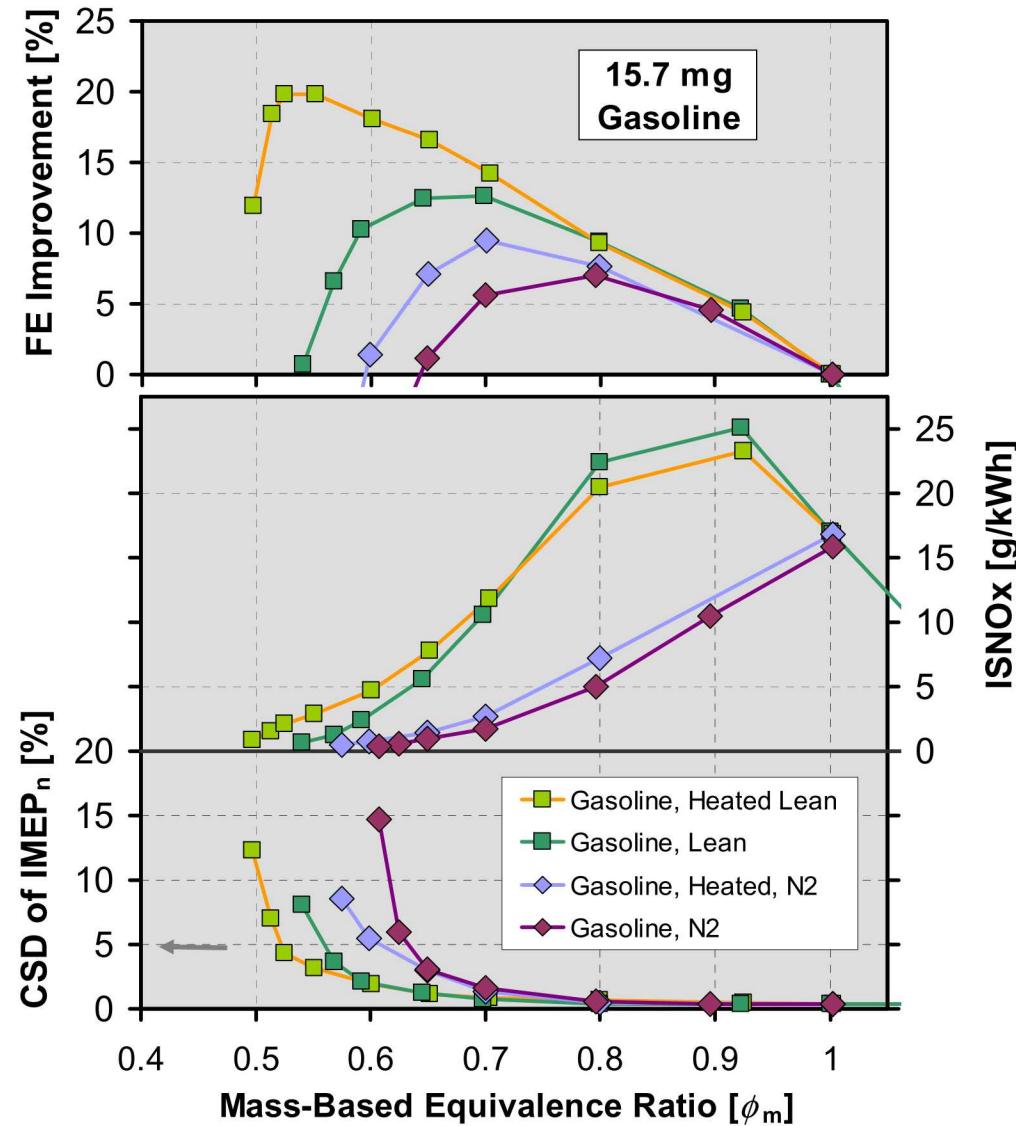


# Dilute Gasoline Operation

- Dilute  $\phi = 1$  operation leads to lower  $\text{NO}_x$ , and a three-way catalyst (TWC) can be used.
- Dilute limits are much worse than for lean operation.
- Less than 10% FE improvement for heated dilute operation.
- What is the problem?

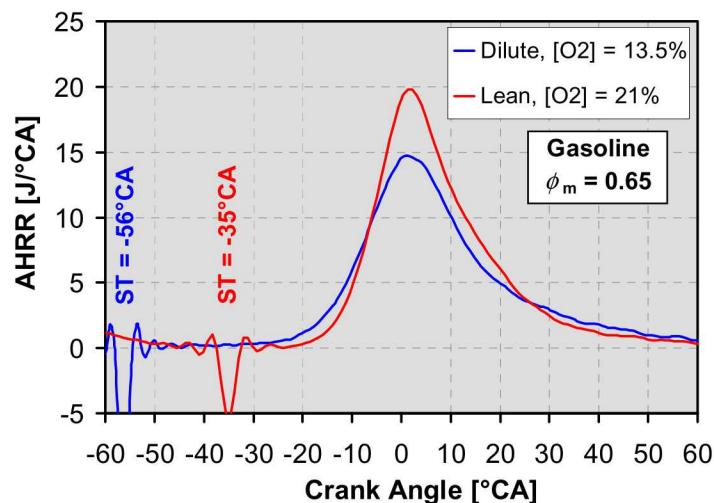
$$\phi_m \equiv \frac{(F/C)_{Actual}}{(F/A)_{Stoichiometric}}$$

- $\phi_m$  is a measure of chemical energy per reactant mass.
- F = Fuel mass, A = Air mass.
- C = Gas Charge Mass (in this case Air +  $\text{N}_2$ ).

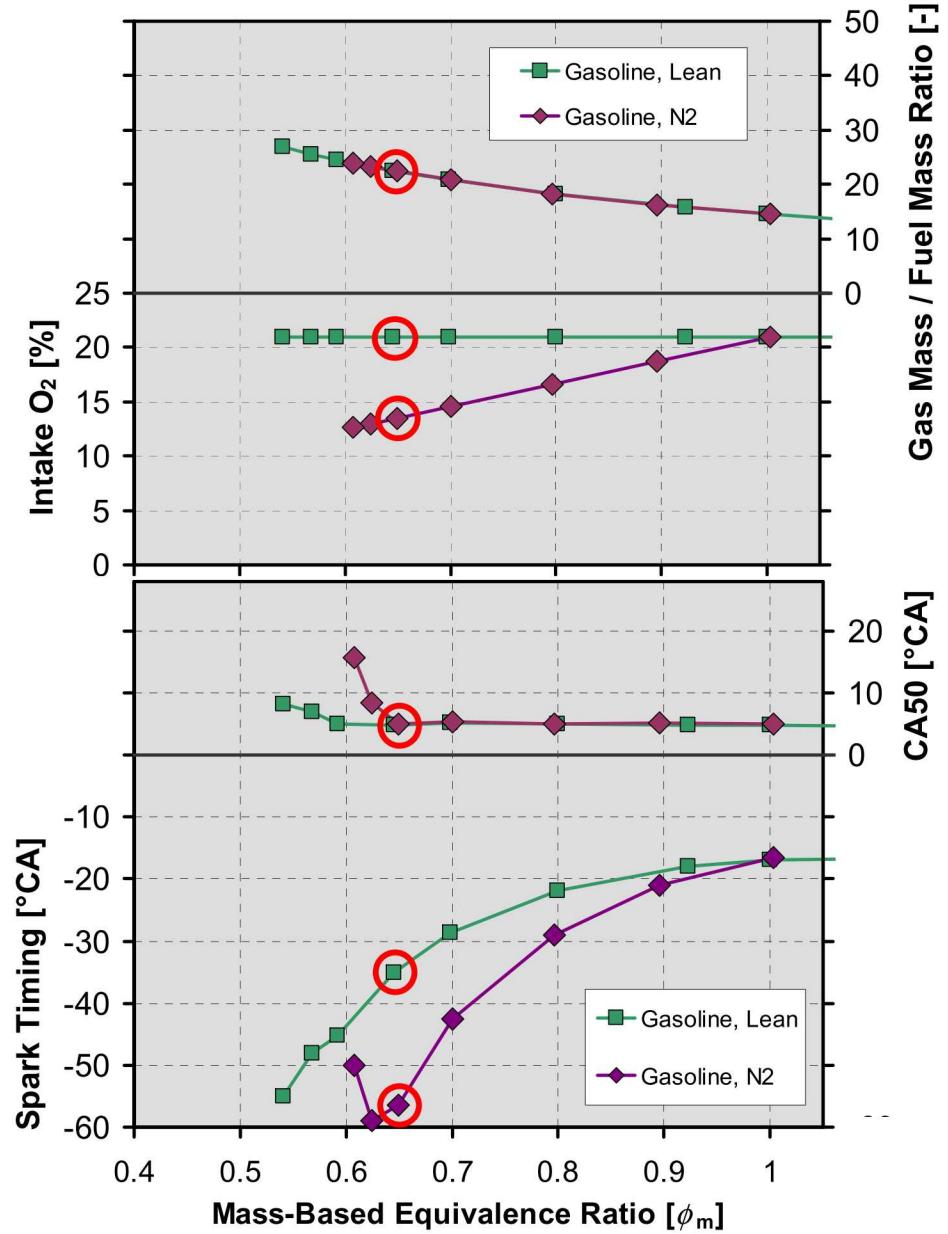
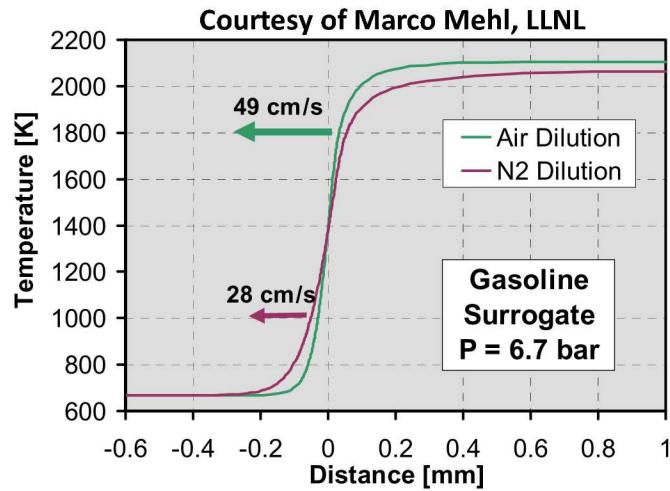


# AHRR for Dilute Operation

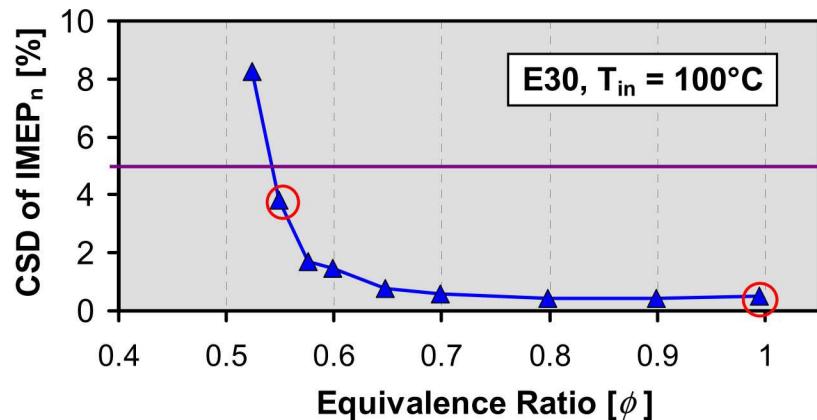
- Dilute  $\phi = 1$  operation leads to much slower spark-to-CA50.
- $[O_2]$  has huge effect on deflagration rates for early flame.



- Confirmed by CHEMKIN modeling.



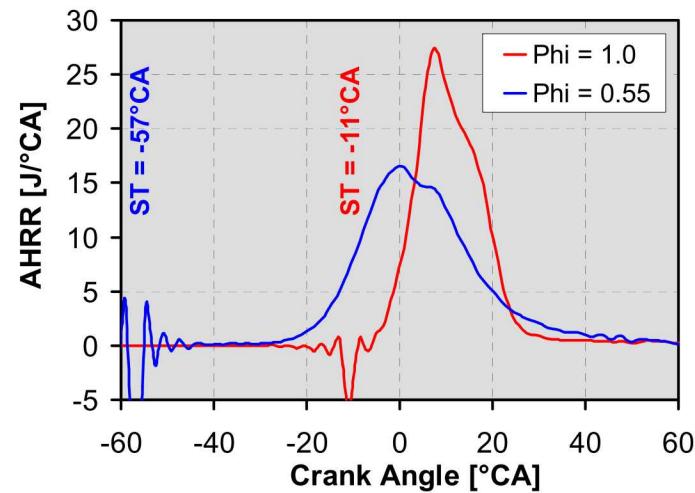
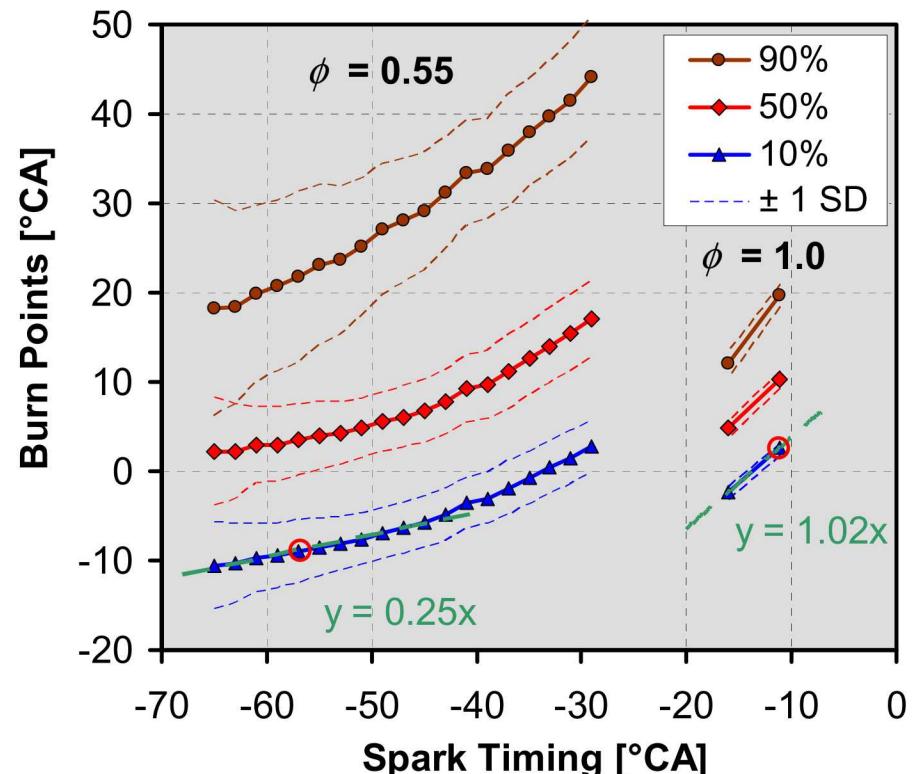
# Lean E30 – Combustion Control



- Stoichiometric combustion:
  - Spark provides very effective combustion-phasing control.
- Near lean-stability limit:
  - Spark has lost most of its

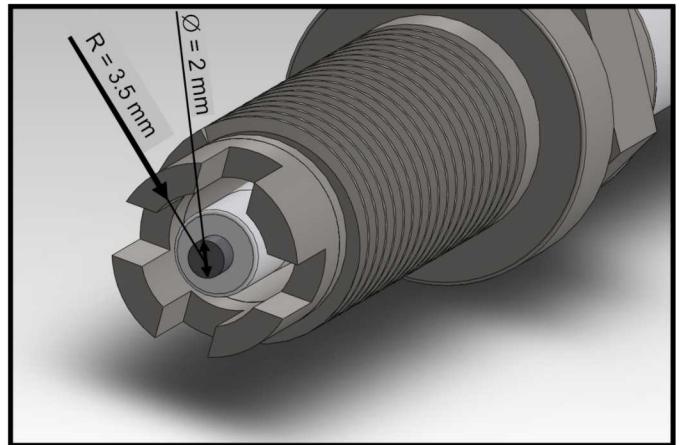
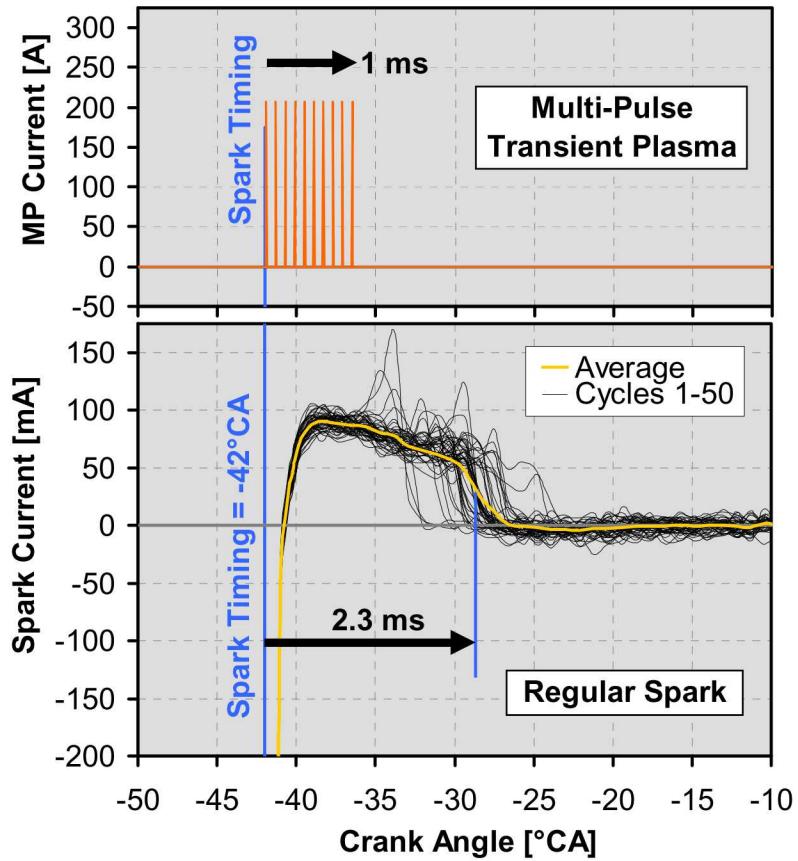
$$\text{control authority} = \frac{\Delta \text{CA10}}{\Delta \text{ST}}$$

- Slow early flame development.
- Use advanced ignition to restore control?



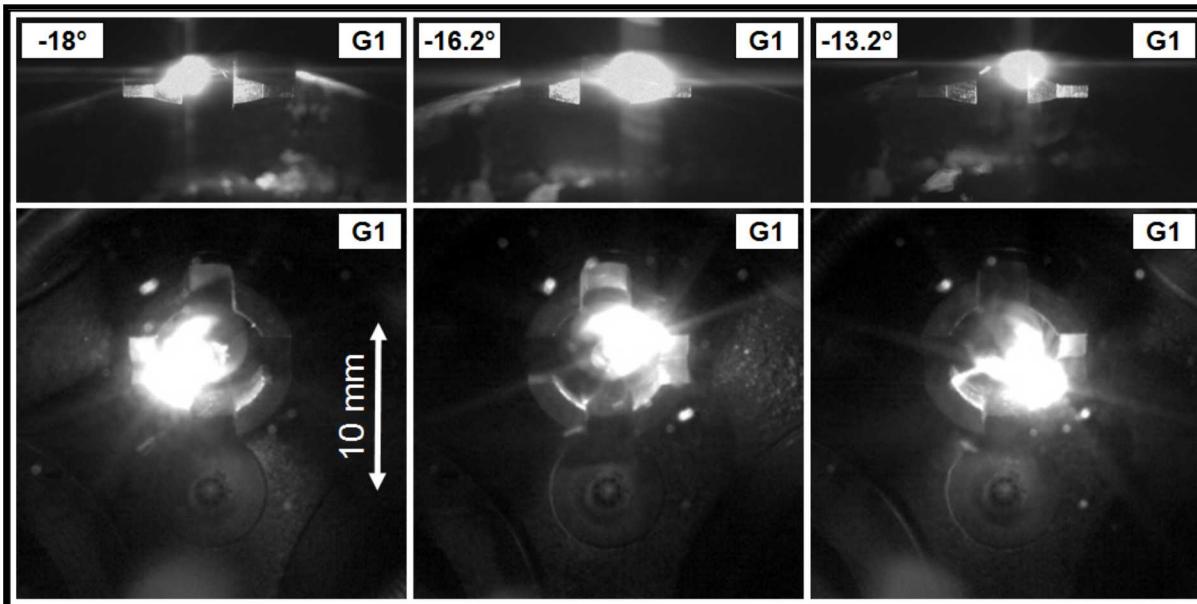
# Multi-pulse (MP) Transient Plasma

- Examine results from SAE Paper 2014-01-2615.
- Collaboration with Martin Gundersen,  
Dan Singleton & Jason Sanders.  USC
- Modified spark plug. 4 grounding arms.
- 10 ultra-short high-current pulses over 1 ms.

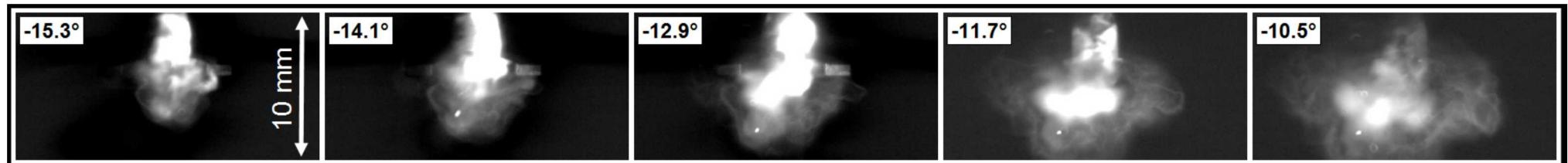


# MP Example at $\phi = 0.90$ , E85.

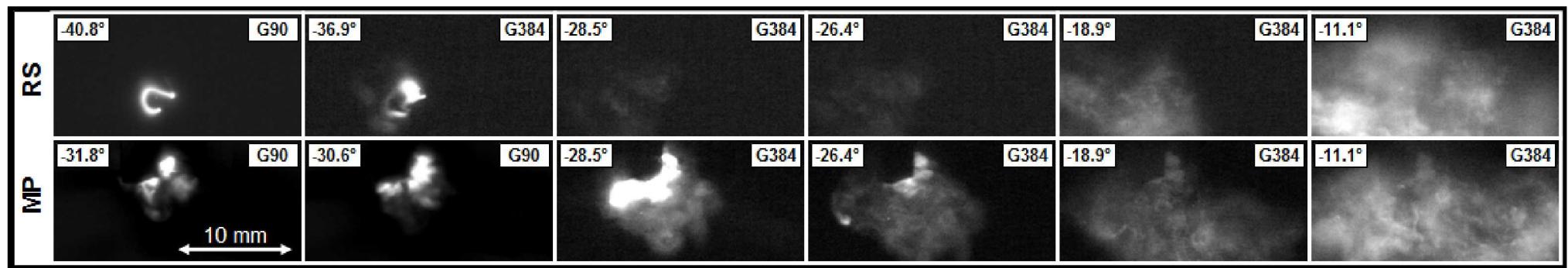
- Typically only one or two anodes show strikes.



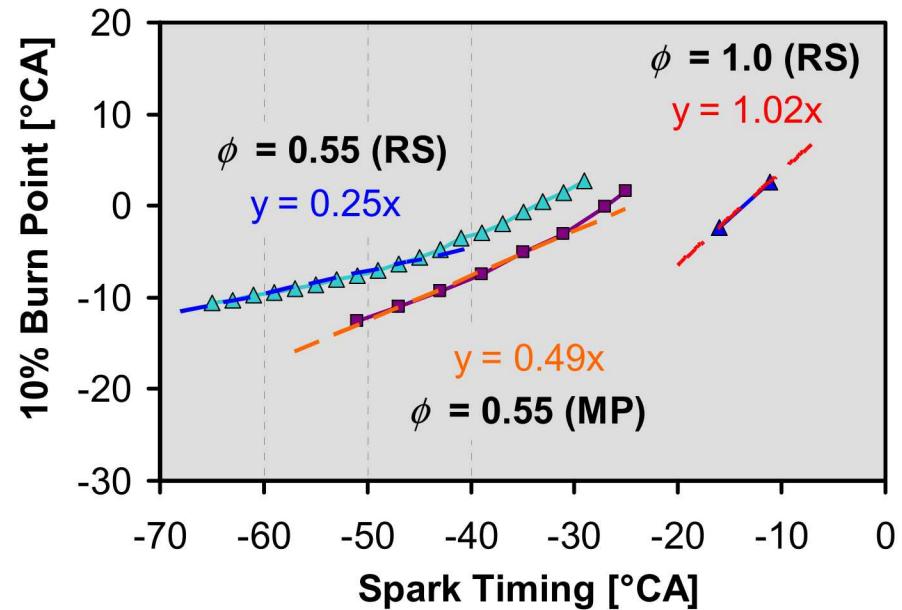
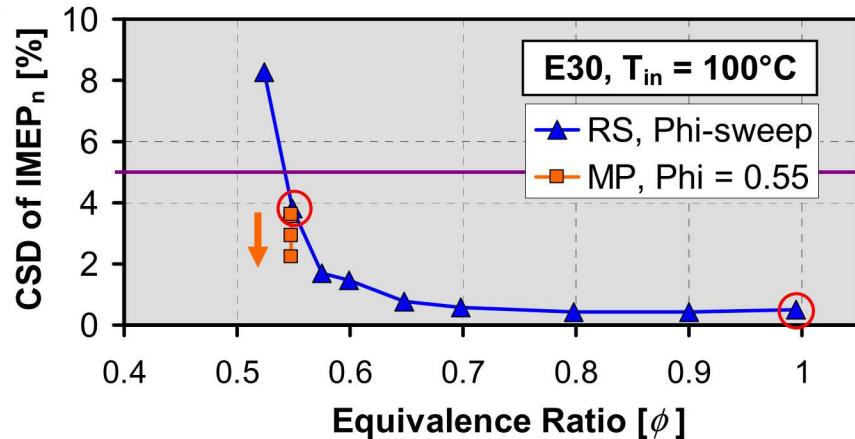
- Downward turbulent flame jet.



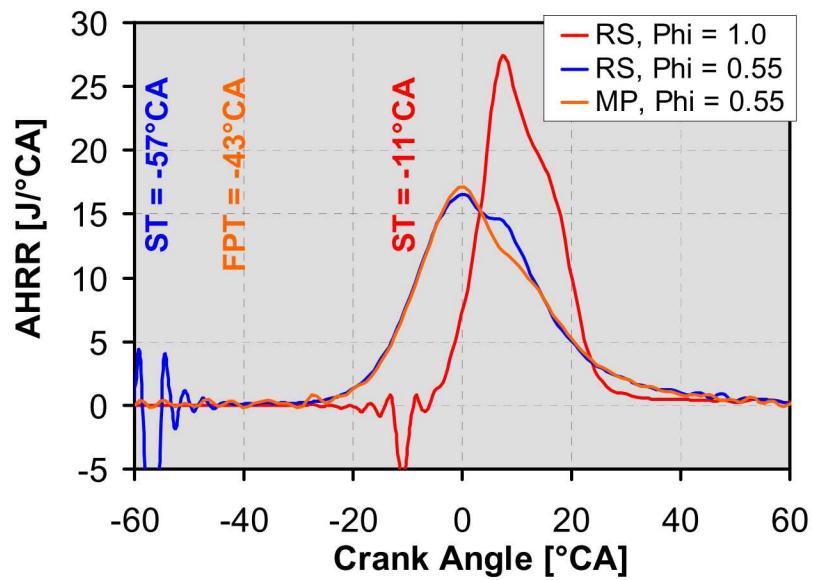
- MP “bypasses” slow and fragile early laminar flame period.



# Lean E30 – Combustion Control with MP



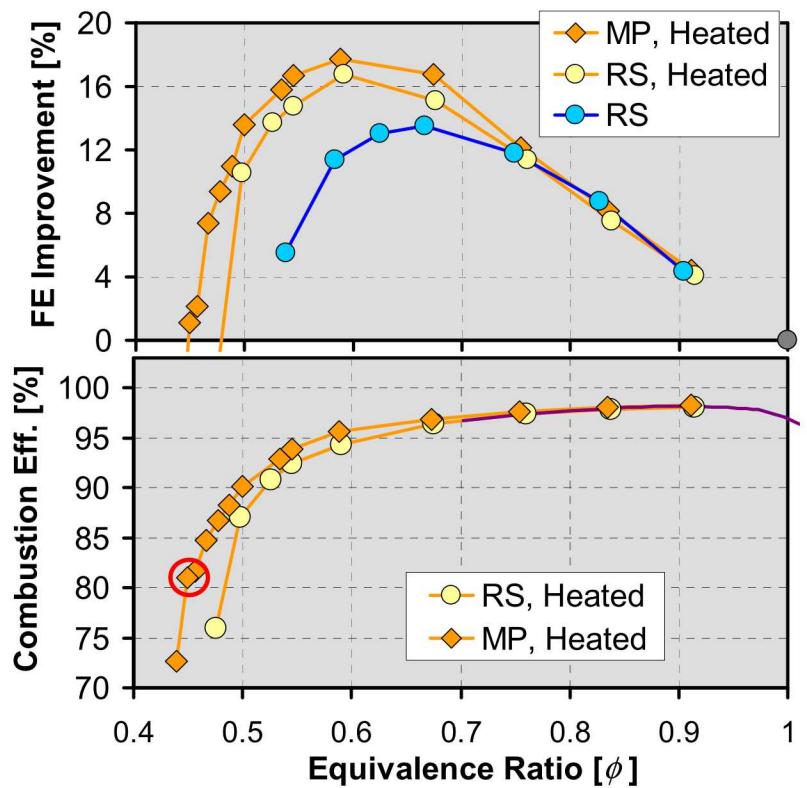
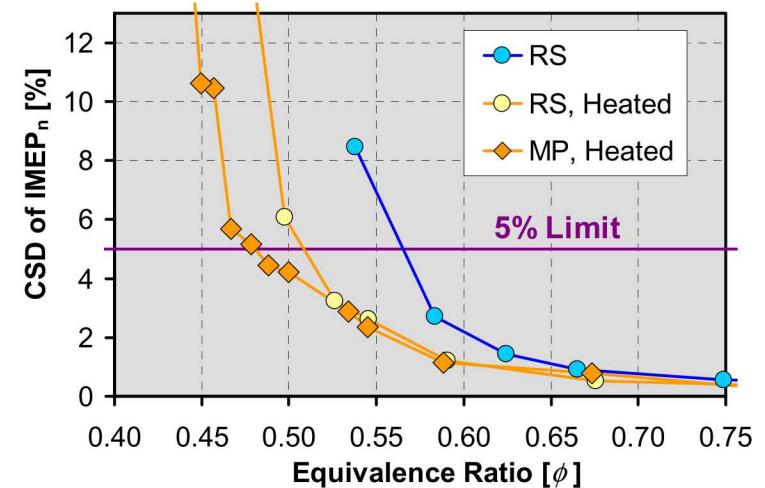
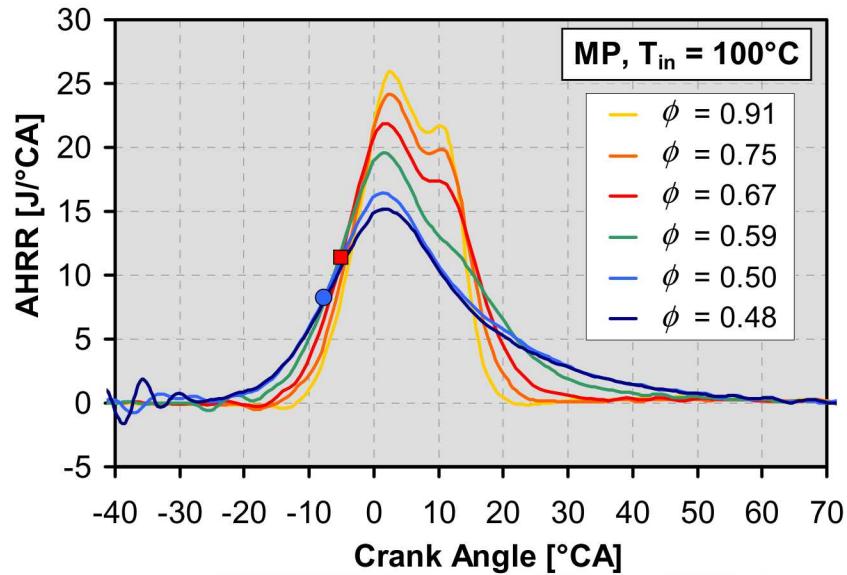
- Near lean-stability limit:
  - Spark has lost most of its authority.
  - Slow early flame development.
- 10-pulse transient plasma ignition doubles control authority.
  - Shorter ST – CA10 delay.
  - Use to lower variability.
  - Extend lean limit.



# E85: FE Gain Limited by Comb. In-Eff.



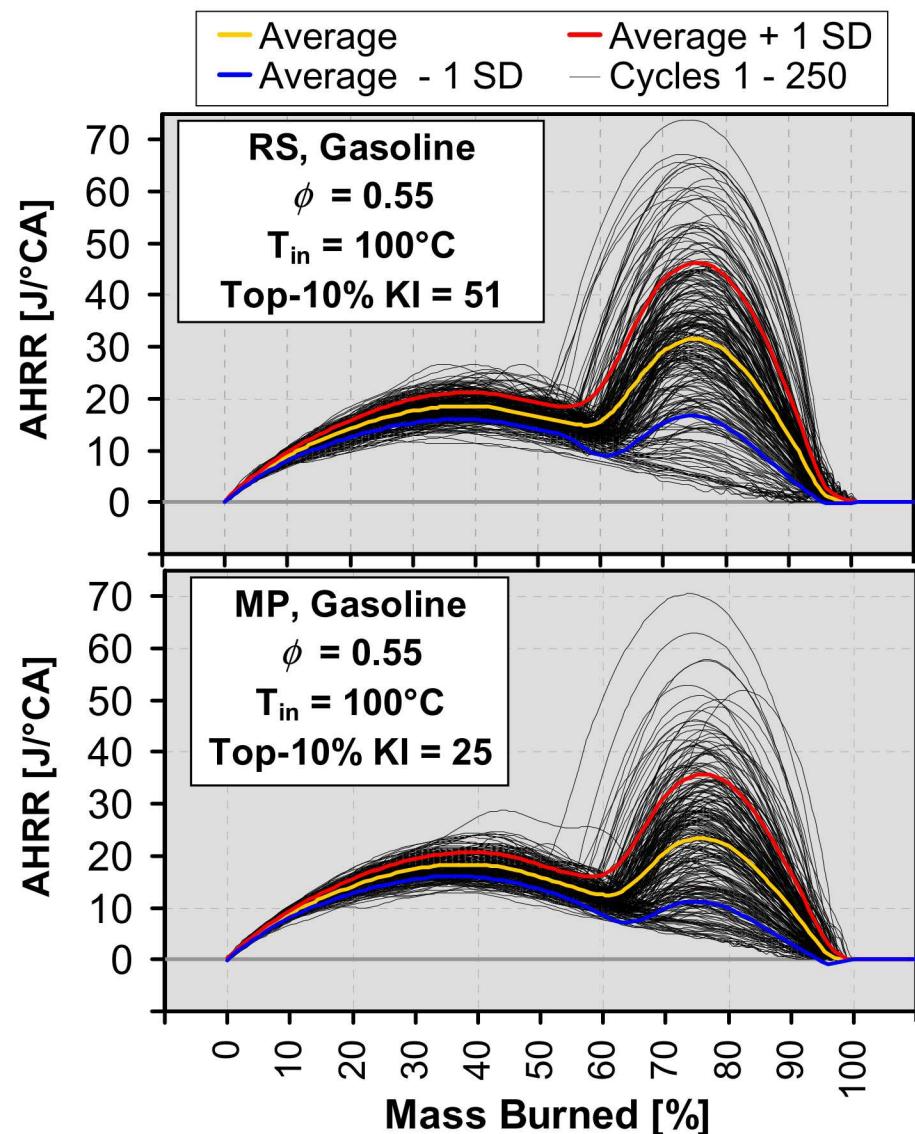
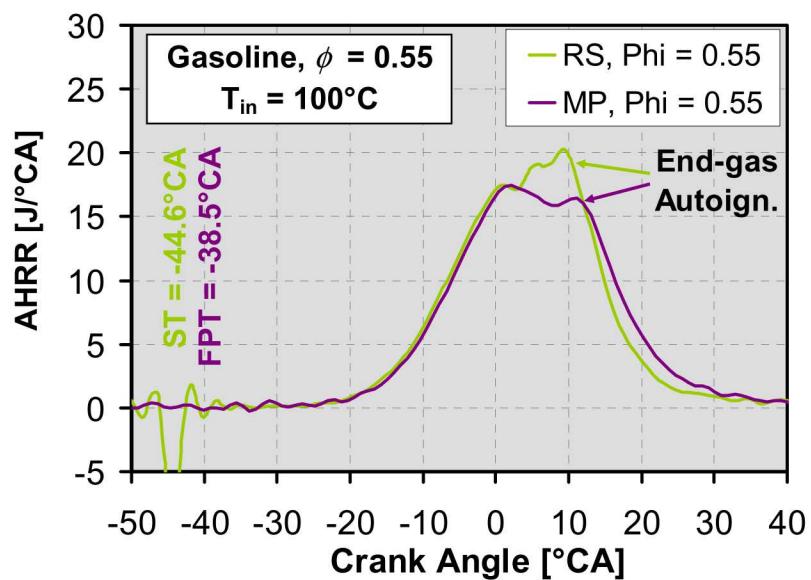
- Heated MP improves lean stability limit.
- $\approx 18\%$  FE improvement.
  - Only small benefit relative RS.
- Average  $\eta_{\text{comb}}$  suffers at stability limit.
- Slow and erratic burn out.
- Enabling step: Ensure complete and faster combustion for all cycles.
- Provoke end-gas autoignition.
- Challenging for high-octane E85.



# Improve End-gas Autoignition Control



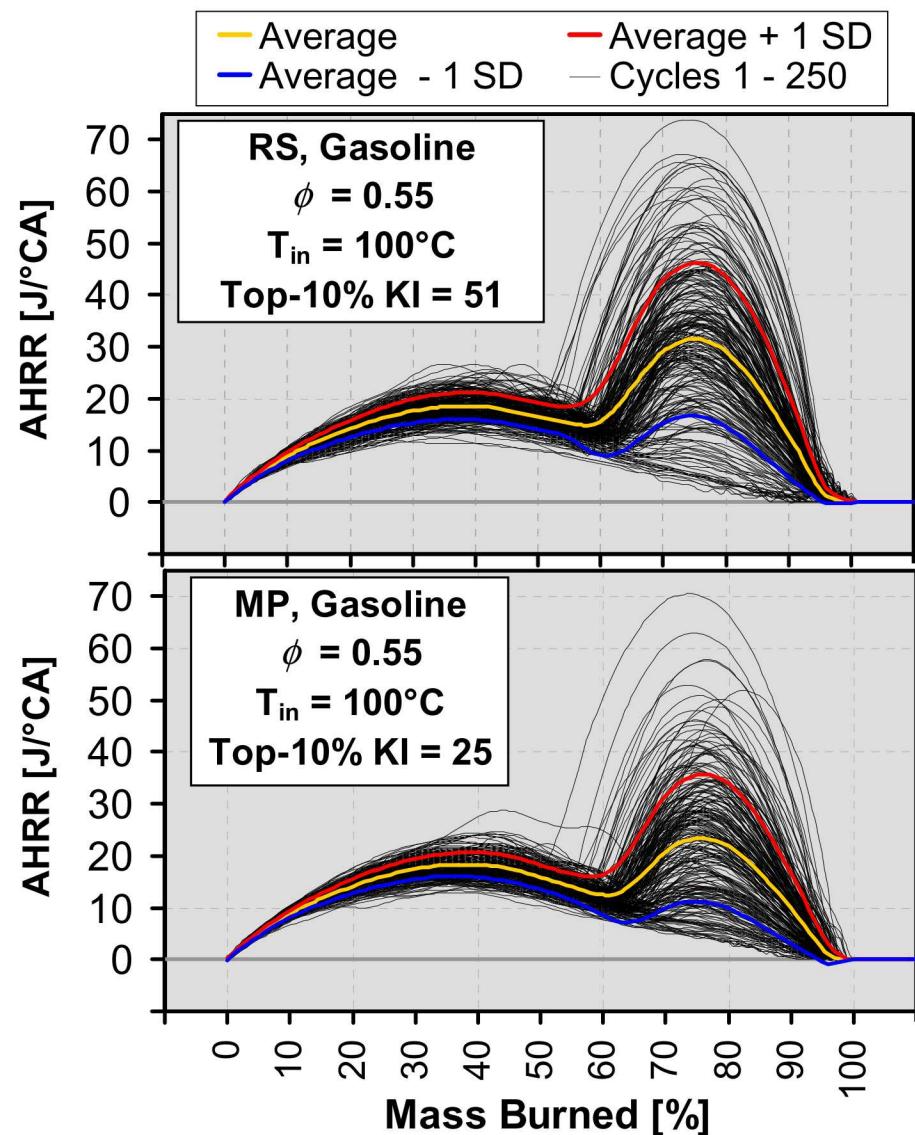
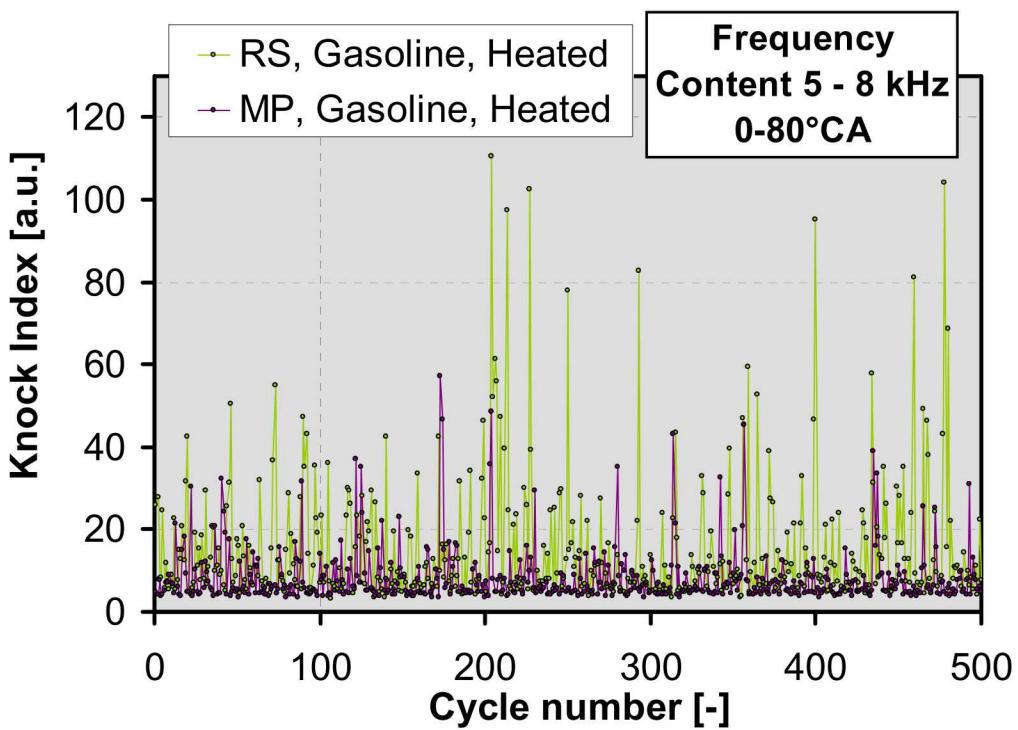
- Better control with MP allows later CA50 for ultra-lean SI gasoline operation.
- Average burn-point for end-gas autoignition shifts 58 to 62%  $\Rightarrow$ 
  - CAI portion reduced from 42 to 38%.
  - More repeatable combustion.
  - Fewer outlier cycles.



# Reduction of Knocking Cycles



- Fewer outlier cycles  $\Rightarrow$  Knock index (KI) for top-10% cycles cut in half.

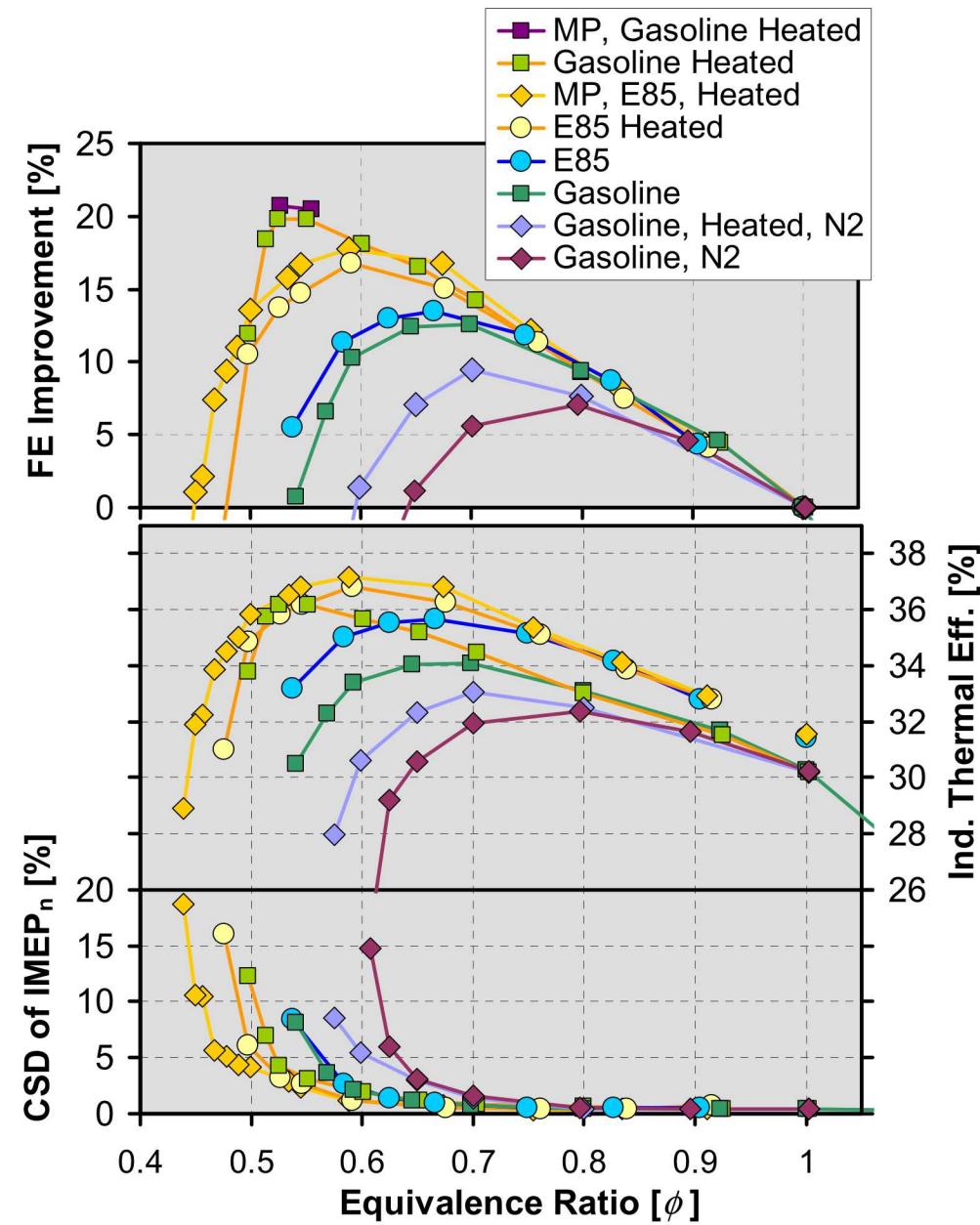


# Summary of Results at 1000 rpm

- MP Heated Gasoline leads to slightly larger FE improvement.
- Data sets sorted by 5% variability limit:

Operation	Max FE Impr.	Peak TE	Limit $[\phi_m]$
Heated E85, MP	17.7%	37.2%	0.480
Heated E85	16.8%	36.8%	0.508
Heated Gasoline	19.9%	36.2%	0.522
Gasoline	12.6%	34.1%	0.560
E85	13.5%	35.7%	0.565
Heated Gasoline, N <sub>2</sub>	9.4%	33.0%	0.608
Gasoline, N <sub>2</sub>	7.0%	32.4%	0.632

- E85 data sets have higher TE for  $\phi = 1$ .
- Peak TE are also higher.
  - Benefit of vaporization cooling.
  - $\downarrow$  comb. temps. &  $\gamma \uparrow$ .



# Conclusions

- Fuel-savings potential of dilute stoichiometric SI operation appears limited by very slow early flame development.
  - Need for highly enhanced ignition source to examine ultimate potential.
- Lean SI operation has substantial potential for efficiency gains, even with regular spark ignition.
- Fuel-economy gains are limited by combustion variability and inefficiencies.
- Stable combustion is promoted by:
  - Late spark timing for a given CA50.
  - High charge temperature at the time of spark.
  - Not relying on fragile early laminar flame.
- Multi-pulse transient plasma ignition with a semi-open spark geometry provides more stable lean combustion by creating a turbulent flame jet.
  - Bypassing phase with slow and fragile laminar flame.
- High combustion efficiency for ultra-lean operation requires end-gas autoignition.
  - Fuel reactivity becomes important.
- With better ignition-control authority for ultra-lean operation, end-gas autoignition becomes more repeatable.