

Vision for validating the liquid hydrogen plume model at temperatures less than 80K

Ethan Hecht, Isaac Ekoto

Combustion Research Facility, Sandia National Laboratories, Livermore, CA

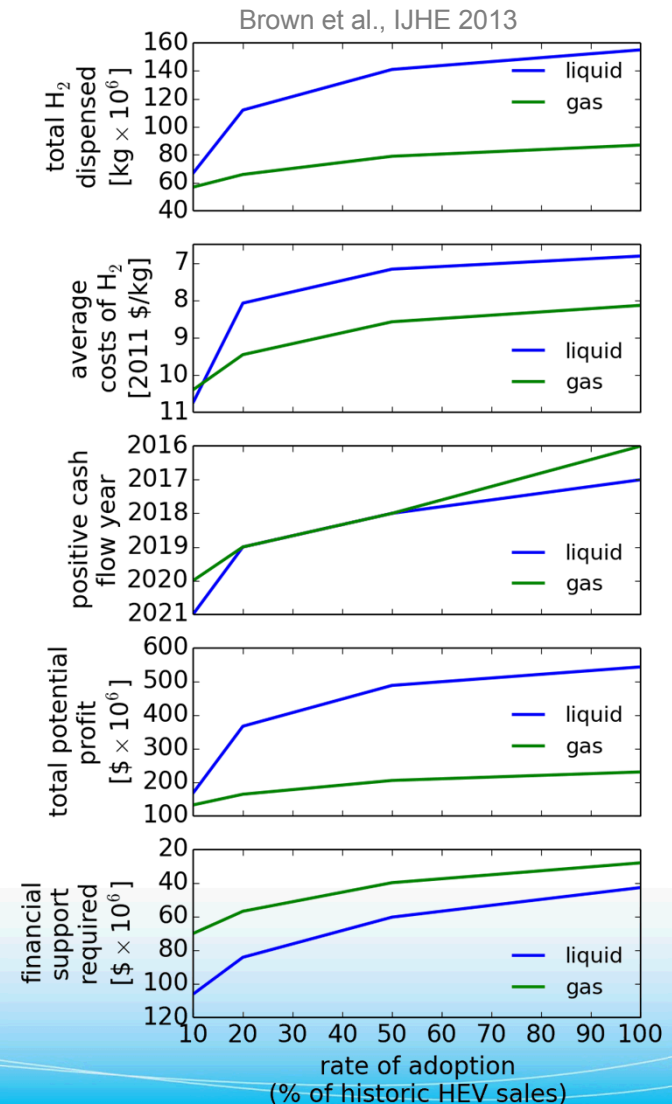
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Washington D.C., Nov. 11, 2014

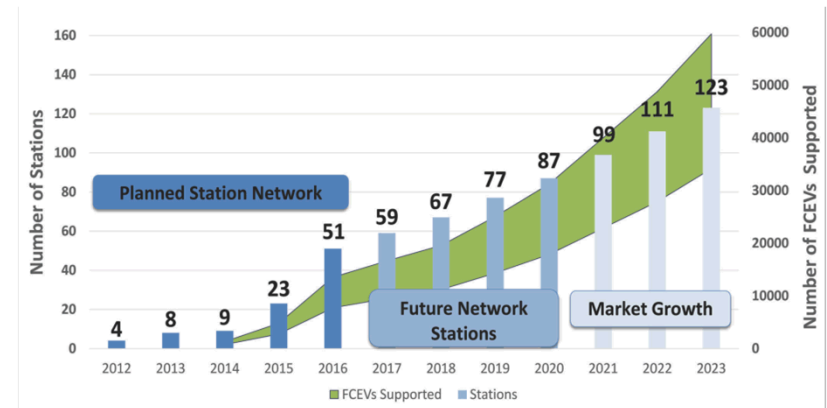
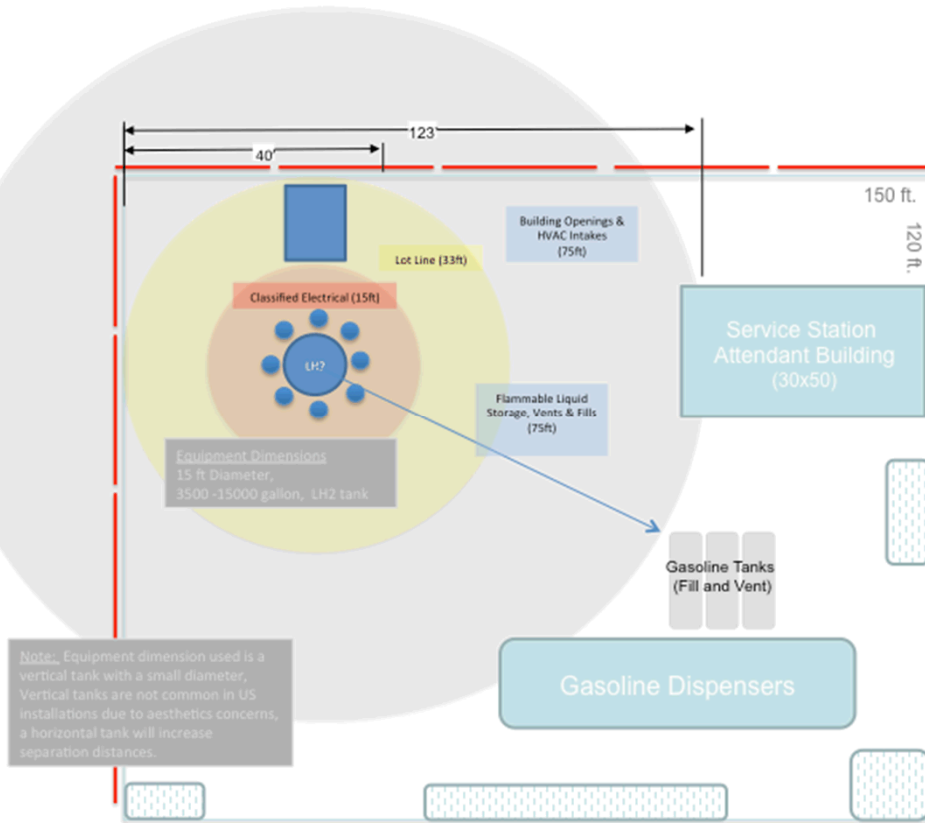
Liquid hydrogen stations have been found to be more economically favorable than gaseous stations

As compared to gaseous stations, liquid storage stations have:

- Larger storage capacity
- Lower costs for product
- Similar positive cash flow year
- Higher potential profit (although more investment is required)



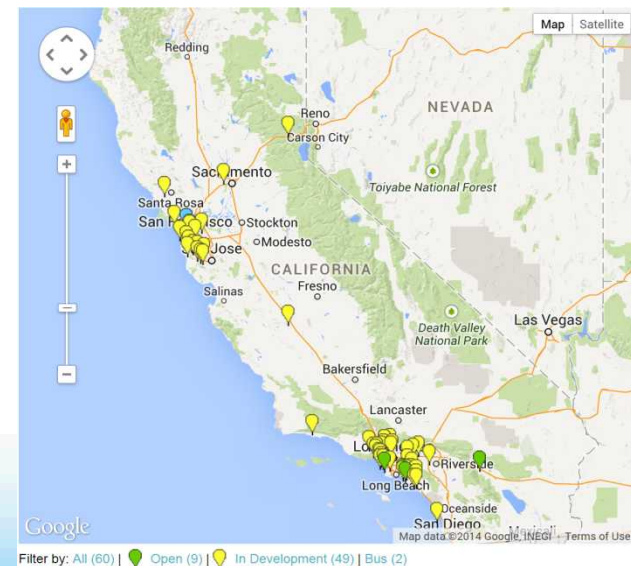
Standoff distances in NFPA 2 for liquid hydrogen stations are often prohibitively large



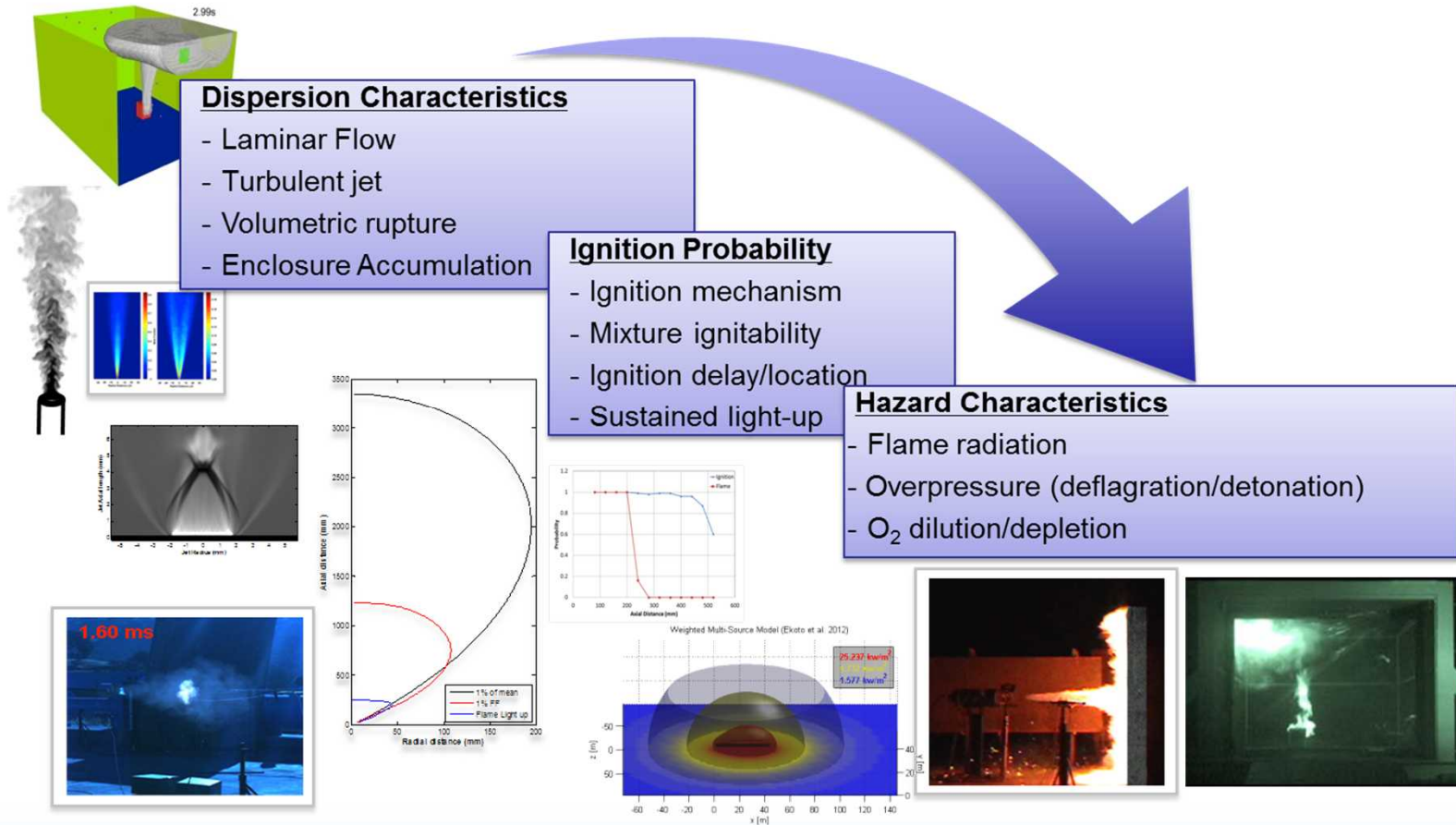
A California Road Map: The Commercialization of Hydrogen Fuel Cell Vehicles, CalFCP, July 2014

70 stations surveyed (of 343 sites), none met the NFPA 2 Ch. 6 separation distance requirements.

Harris, SAND-2014-3416



Previous modeling of releases from gaseous hydrogen storage have informed the fire code

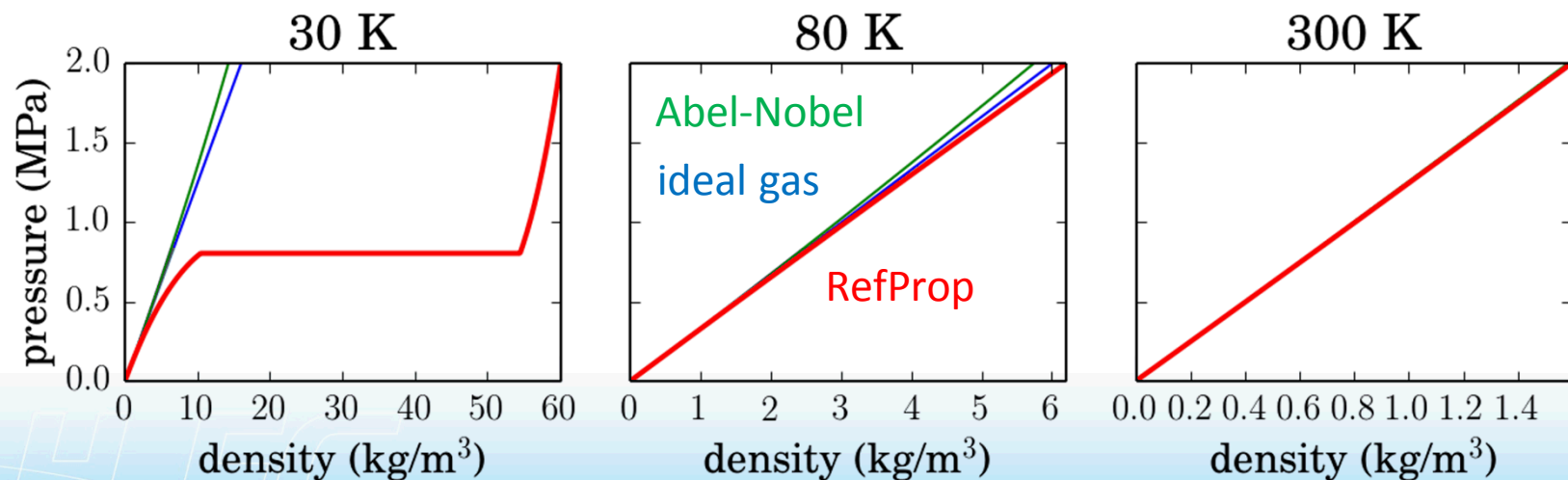


Risk requires a **Release**, then **Ignition**, forming a **Hazard**, causing **Harm**

- We **quantify** each of these events using models
- **Purple** events quantified with statistical models, **Red** with reduced-order behavior models

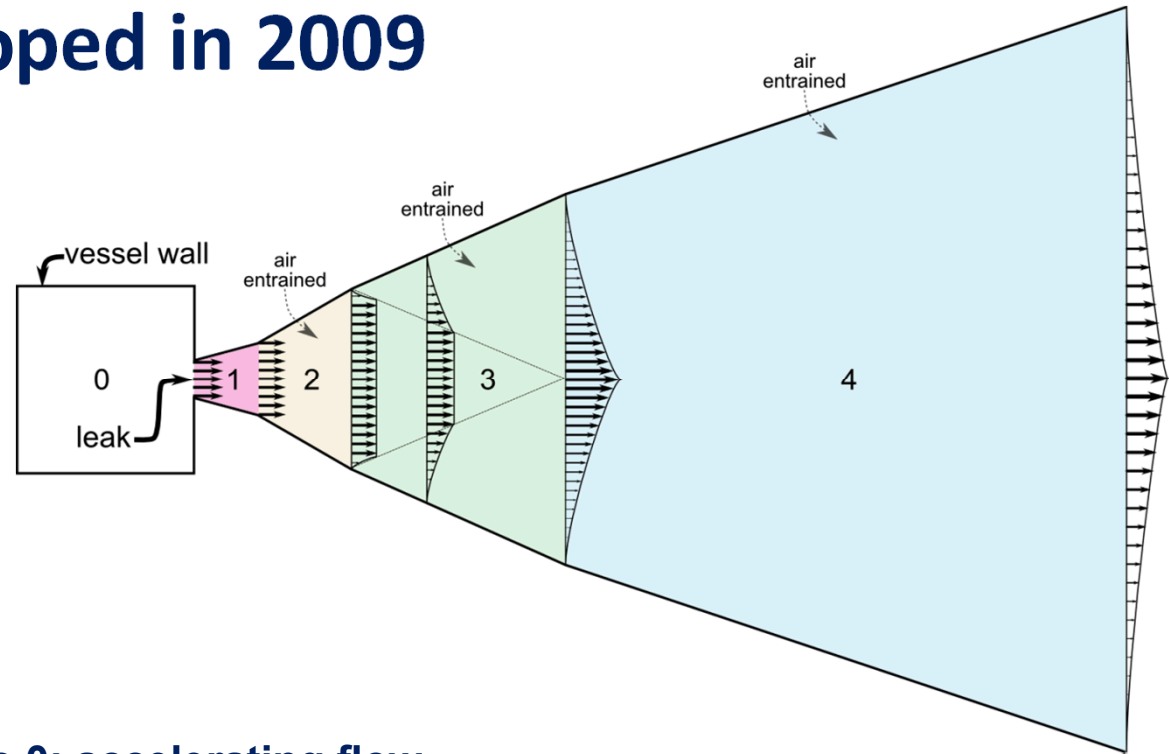
Current network flow model (NETFLOW) must be updated for use near saturation conditions

- Models 1-D flow networks (e.g. piping, valves, tanks) by solving conservation and state modeling equations with local corrections for wall friction, heat transfer, and pressure loss
- Conventional state equations invalid near saturation conditions
- Important to capture phase-change behavior
- Must model compressible and incompressible flows



A conceptual model for liquid H₂ releases was originally developed in 2009

- **Steady-state**
- **1-dimensional (along streamline coordinate)**



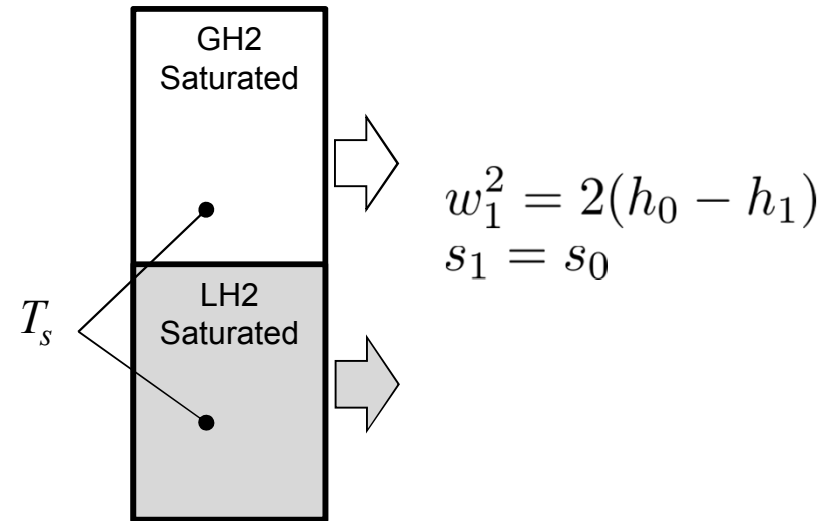
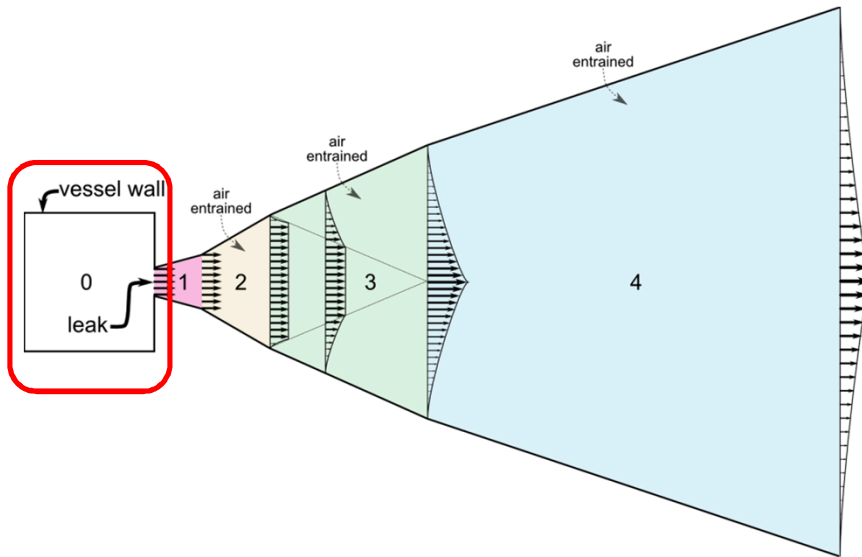
- **Zone 0: accelerating flow**
- **Zone 1: underexpanded jet**
- **Zone 2: initial entrainment and heating**
- **Zone 3: flow establishment**
- **Zone 4: self-similar, established flow**

Winters, SAND Report 2009-0035
 Winters & Houf, IJHE, 2011
 Houf & Winters, IJHE, 2013
 Ekoto et al., SAND2014-18776

Accelerating flow (leak) develops from saturated storage conditions

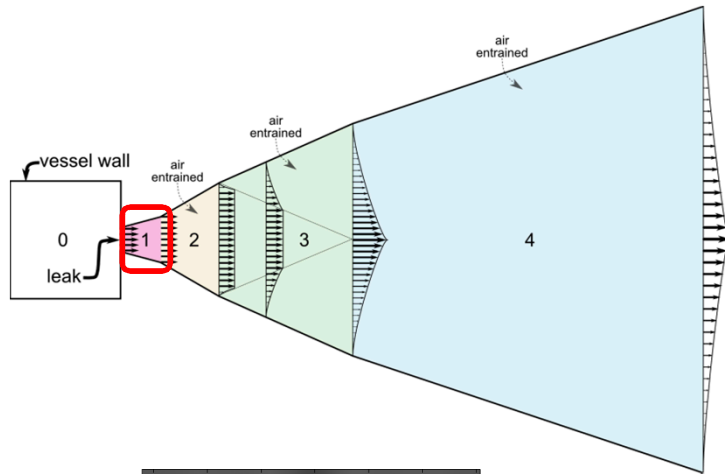
- conserved energy with isentropic expansion

Ekoto et al., SAND2014-18776



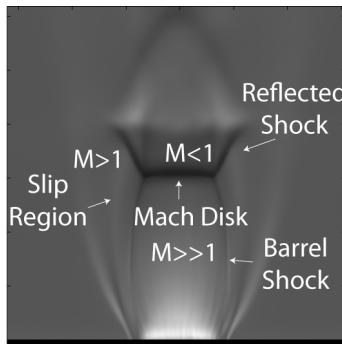
- conditions at zone 0 capture by network flow model (requires development)
- hydrogen is stored as a pure substance
- multi-phase components have equal velocities

Pseudo source models are used to account for choked flow behavior in Zone 1 (if applicable)



Several source models have been developed to predict the mass weighted effective diameter, (i.e., the critical scaling parameter): $d^* \equiv$

$$d_{eff} \sqrt{\rho_{eff} / \rho_{amb}}$$



Ruggles & Ekoto, *IJHE*, 2012

| Source Model | d^* [mm] |
|-------------------------|--------------|
| Birch et al. (1984) | 0.947 |
| Ewan & Moodie (1986) | 0.993 |
| Birch et al. (1987) | 0.790 |
| Yuceil & Otugen (2002) | 0.790 |
| Harstad & Bellan (2006) | 1.440 |
| Molkov (2008) | 0.993 |
| SNL Data (2011) | 0.867 |

Neglects Mach Disk (i.e., fully supersonic)

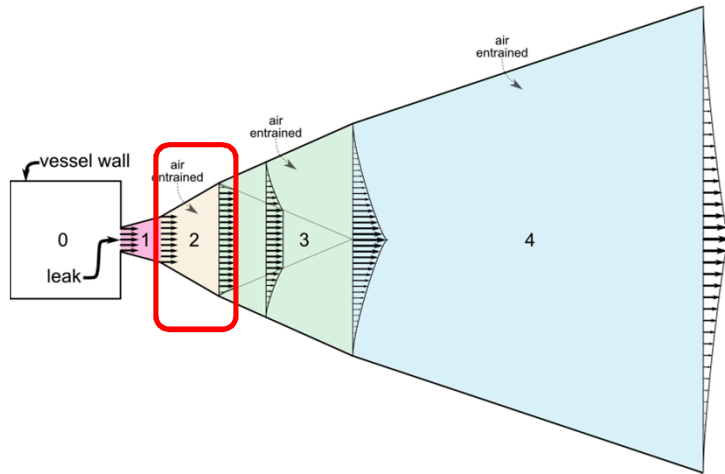
All flow through Mach disk (i.e., fully subsonic)

Reality is that fluid is split between the slip and Mach disk regions

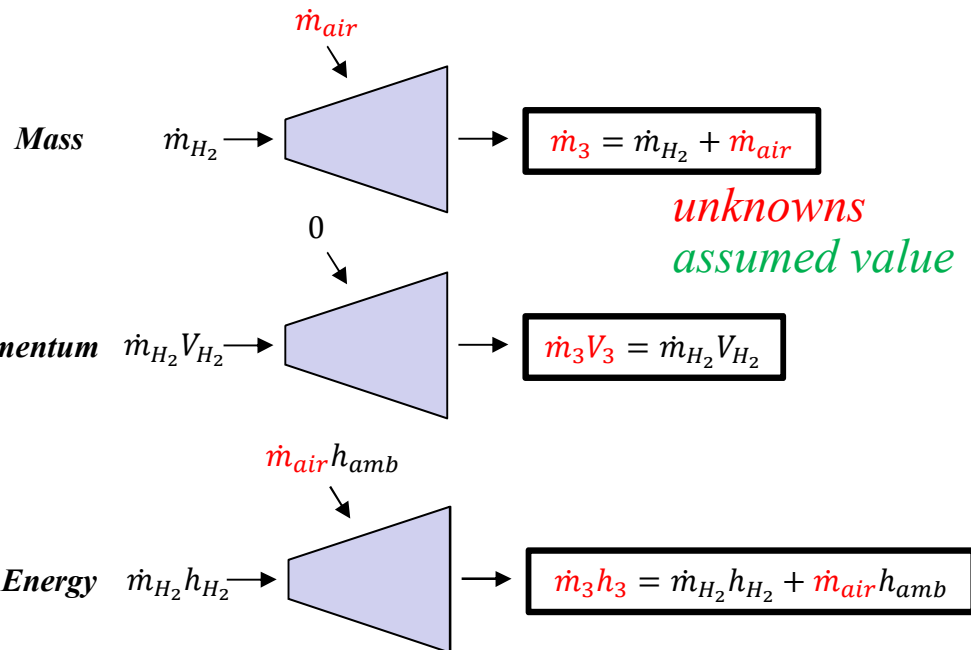
*All models updated w/ Able-Noble EOS

Ongoing work to develop validated two-zone source model that accounts for the fluid split ratio between the slip region & Mach disk regions

Plug flow assumption invoked for Zone 2 as the jet begins to warm



Winters, SAND Report 2009-0035



State modeling by NIST H₂ EOS:

$$h_3 = f(Y_{H_2,3}, p_{amb}, T_3)$$

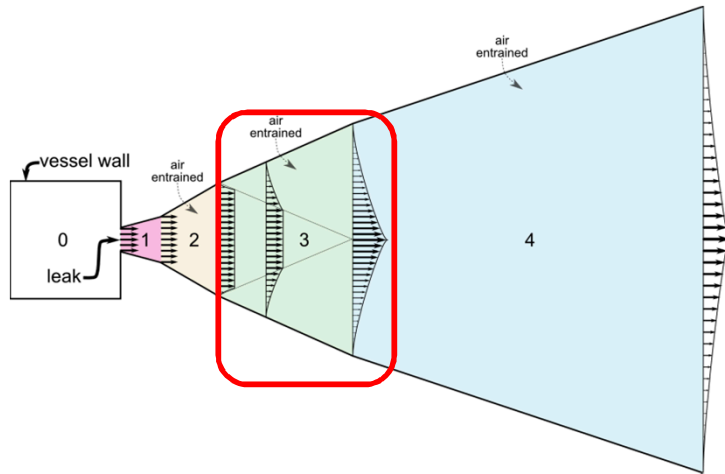
Species conservation used to close system of equations:

$$\dot{m}_{air} = \dot{m}_{H_2} \frac{1 - Y_{H_2,3}}{Y_{H_2,3}}$$

Turbulent jet entrainment rate used to estimate zone length:

$$E_{mom} \equiv \frac{1}{\rho_{amb}} \frac{d\dot{m}}{dS} \approx \frac{1}{\rho_{amb}} \frac{\dot{m}_{air}}{S_3} \Rightarrow S_3 = \frac{\dot{m}_{air}}{E_{mom} \rho_{amb}}, \text{ where } E_{mom} = \alpha_m \left(\frac{\pi D_{H_2}^2 \rho_{H_2} V_{H_2}^2}{4 \rho_{amb}} \right)^{\frac{1}{2}}$$

Flow develops to the assumed self-similar profile in Zone 3



unknowns
assumed value

$$V_{CL,4} = V_3$$

Winters, SAND Report 2009-0035

Mass

$$\rho_3 \frac{D_3^2}{4} = B_4^2 \left[\rho_{amb} - \frac{\lambda^2}{\lambda^2 + 1} (\rho_{amb} - \rho_{CL,4}) \right]$$

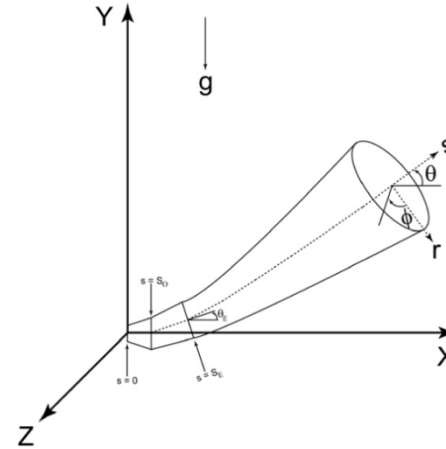
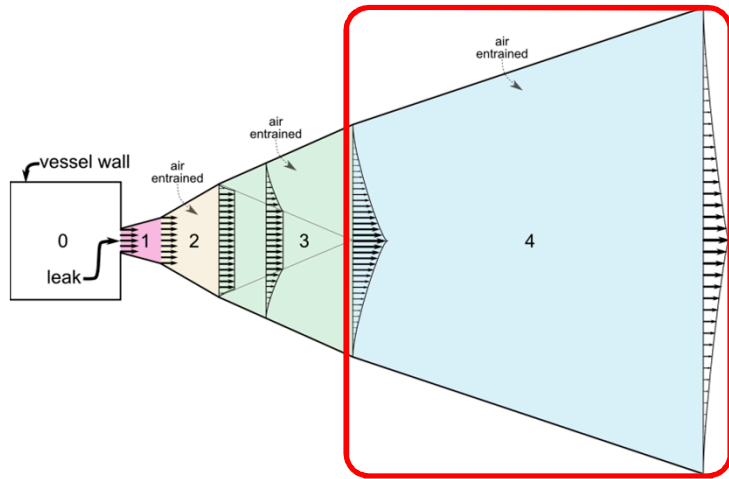
Momentum

$$\underbrace{(\rho_{amb} - \rho_3) \frac{D_3^2}{4}}_{S_3} = B_4^2 \underbrace{\left[\frac{\rho_{amb}}{2} - \frac{\lambda^2}{2\lambda^2 + 1} (\rho_{amb} - \rho_{CL,4}) \right]}_{S_4}$$

S₃

S₄

Zone 4 modeled with previous SNL 1D integral jet/plume models that invoke self-similarity – FY08



Entrainment due to buoyancy & momentum

- F_{rL} : Jet Froude length
- α_b : Buoyancy entrainment coefficient
- α_m : Momentum entrainment coefficient
- g : Gravity constant

$$E_{buoy} = \frac{\alpha_b}{F_{rL}} (2\pi V_{CL} B) \sin \theta$$

$$E_{mom} = \alpha_m \left(\frac{\pi D^2}{4} \frac{\rho V^2}{\rho_{amb}} \right)^{\frac{1}{2}}$$

$$F_{rL} = \frac{V_{CL}^2 \rho_{exit}}{gB(\rho_{amb} - \rho_{CL})}$$

$$\text{Mass} \quad \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V r dr d\phi = \rho_{amb} E$$

$$\text{x-Mom} \quad \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V^2 \cos \theta r dr d\phi = 0$$

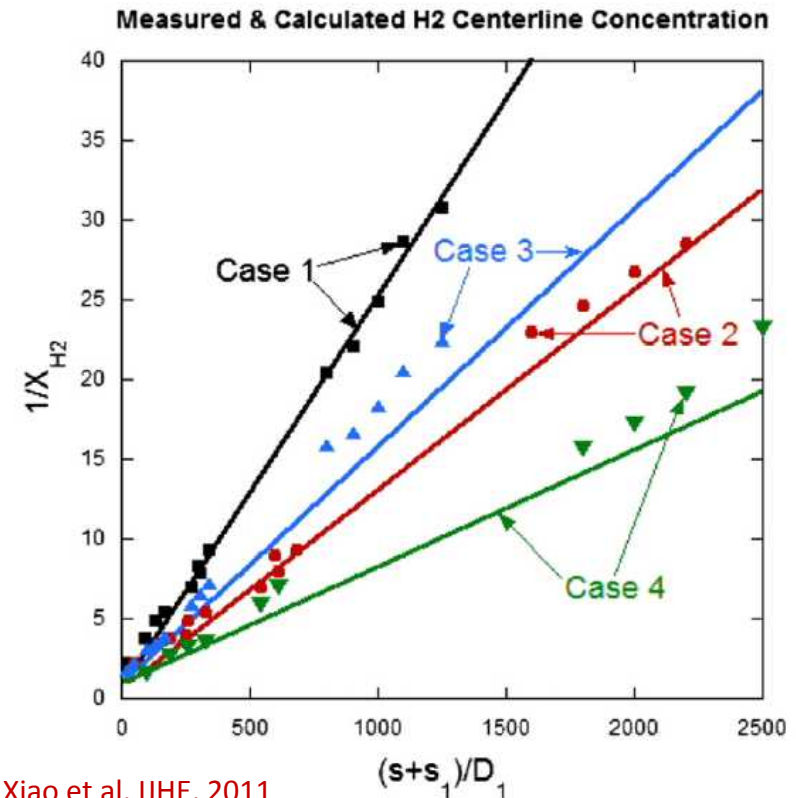
$$\text{y-Mom} \quad \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V^2 \sin \theta r dr d\phi = \int_0^{2\pi} \int_0^{\infty} (\rho_{amb} - \rho) g r dr d\phi$$

$$\text{Species} \quad \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V Y r dr d\phi = 0$$

$$\text{Energy} \quad \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V (h - h_{amb}) r dr d\phi = 0$$

Model results compare favorably to experiments from Karlsruhe Institute of Technology

| Case | Reservoir pressure [MPa] | Reservoir temperature [K] | Leak diameter [mm] |
|------|--------------------------|---------------------------|--------------------|
| 1 | 1.7 | 298 | 2 |
| 2 | 6.85 | 298 | 1 |
| 3 | 0.825 | 80 | 2 |
| 4 | 3.2 | 80 | 1 |

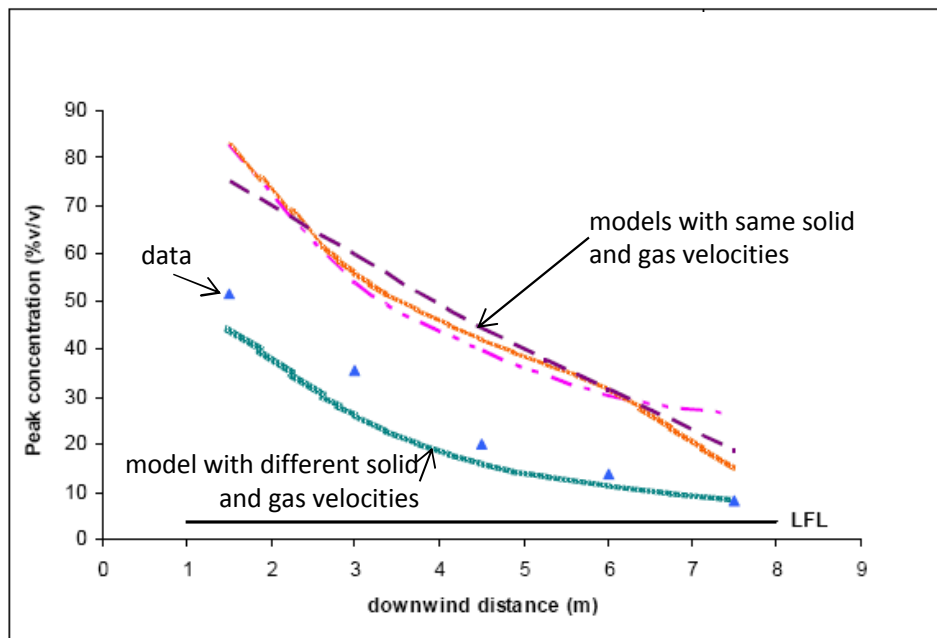


Xiao et al, IJHE, 2011
 Houf & Winters, IJHE, 2013

However, no well-controlled validation data is available at lower temperatures where multi-phase flows are expected (i.e., $T < 77$ K)

As moisture and air condense, multi-phase flows may have droplet/particle slip

Liquid and vapor phases have different velocities due to density differences — slip models have captured these effects in CFD simulations.



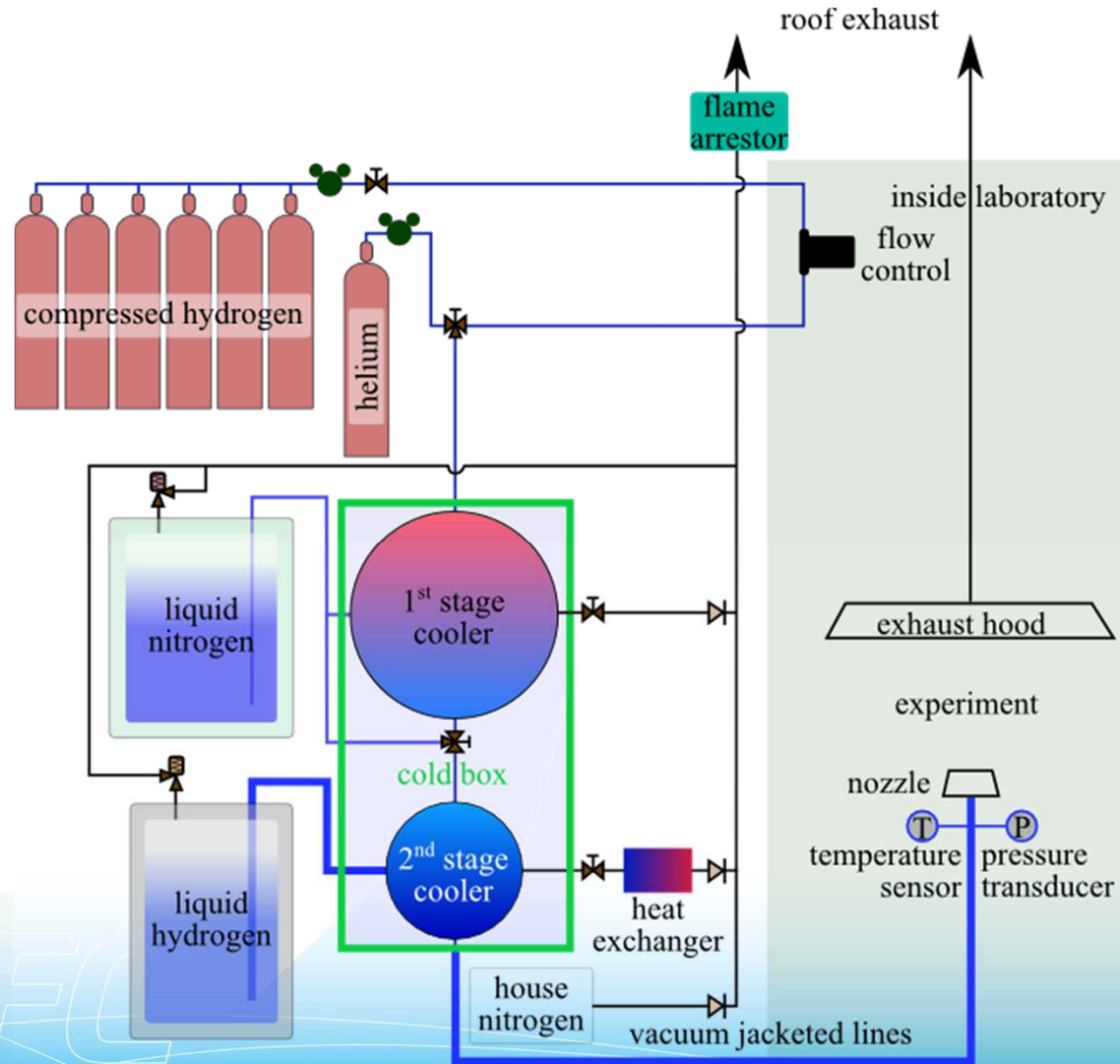
HSL Measurements: Sample probes
Hooker et al, ICHS, 2011

ADREA-HF CFD Simulations
Giannisi et al, ICHS, 2013

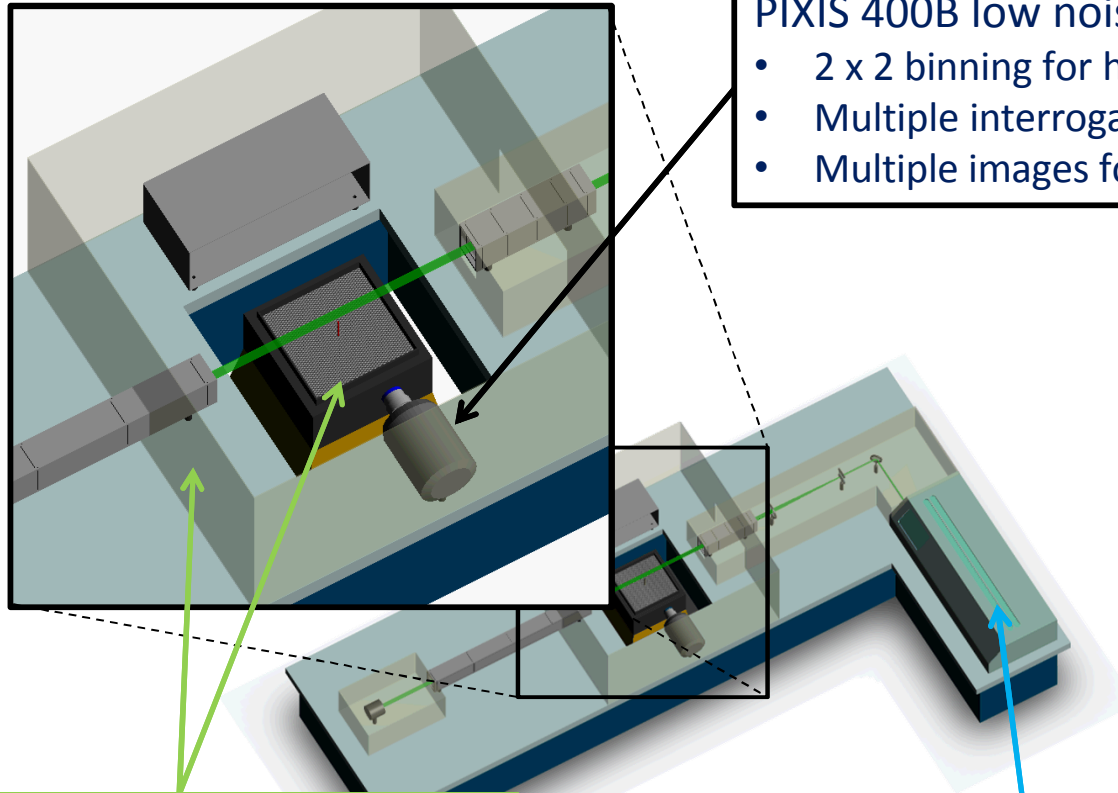
Substantial differences in model results suggest 2-phase effects cannot be neglected for LH2 releases

Experiments had poor control of release and environmental boundary conditions, which are needed for suitable benchmark data

We plan to retrofit our lab to generate the necessary low temperature data for model validation



Optical diagnostics with carefully controlled boundary conditions will provide validation data



- PIXIS 400B low noise CCD Camera
- 2 x 2 binning for high signal-to-noise (~400:1)
 - Multiple interrogation regions to image full jet
 - Multiple images for converged statistics

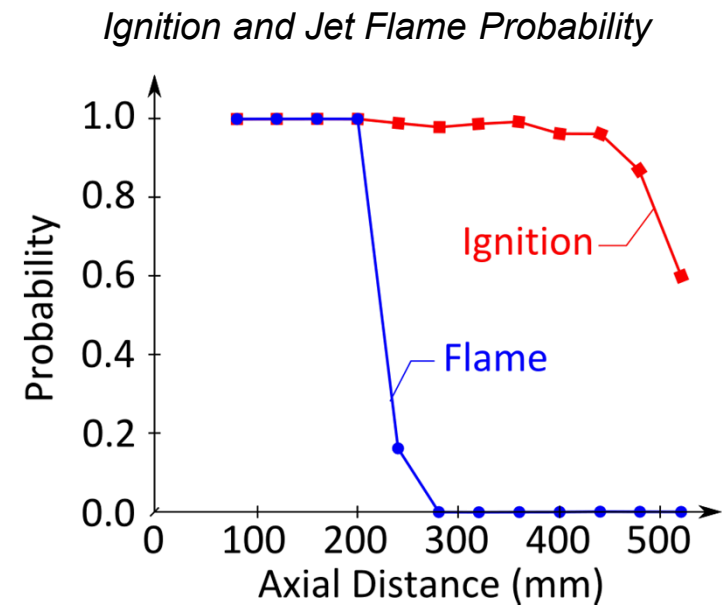
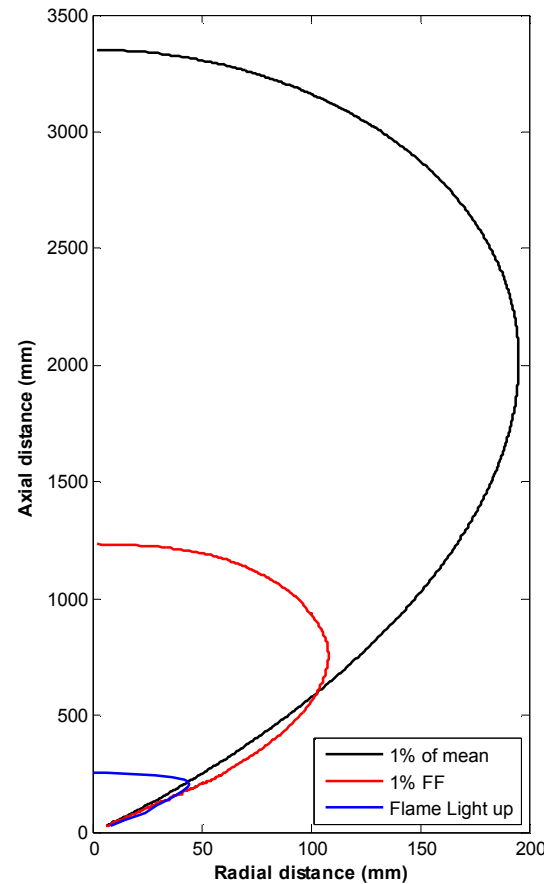
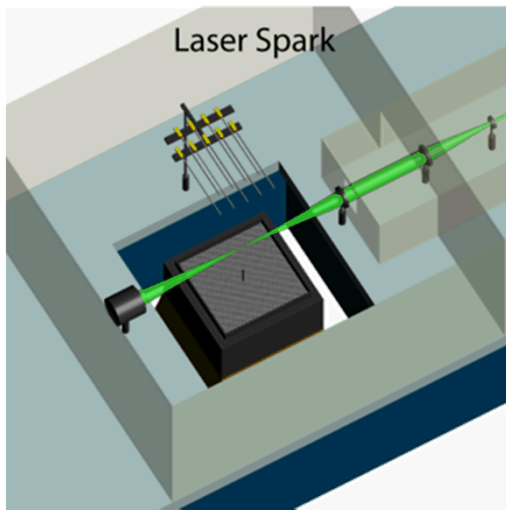
Air co-flow & barriers to minimize impact of room currents

Nd:YAG injection seeded laser (1 J/pulse @ 532 nm)



Opportunity for additional upstream measurements using complementary Raman diagnostics in an adjacent lab

Future work to verify and quantify ignition boundaries



Summary and conclusions

Experimental plans:

- update network flow model
- build out laboratory system
- planar laser Rayleigh scattering to measure jet spreading
- particle imaging velocimetry to measure velocity
- model validation and updating
- ignition quantification

Challenges for liquid H₂ reduced-order modeling:

- accurate state modeling
- pool spreading and evaporation
- humidity effects
- multiphase flow models, with velocity slip
- interactions with surfaces (e.g. barriers, ground)

Acknowledgements

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