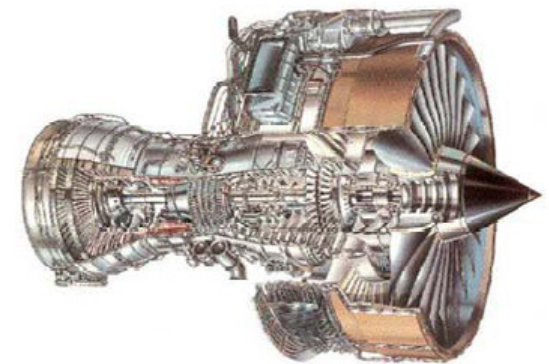
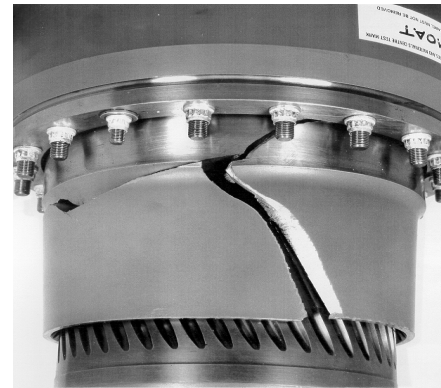
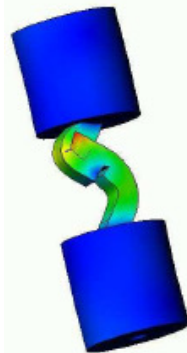
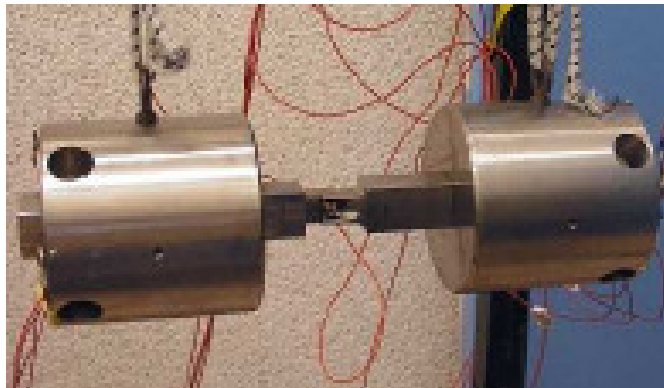


*Exceptional service in the national interest*



## Project 4: Potential of analytical and experimental model reduction techniques

Johann Gross (University of Stuttgart),  
Merten Tiedemann (Audi AG)

Matt Brake (Sandia National Labs)  
Randy Mayes (Sandia National Labs)



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# Project Introduction

- Given a real industrial Structure, reduction techniques are necessary to enable large linear/nonlinear computations
- Different methods (analytical and experimental) shall be investigated concerning the feasibility and effectiveness to predict nonlinear dynamic responses
- **Main Tasks:**
  - Basic **validation** of the available linear FE models against test data
  - **Analytic model reduction** of components
  - **Experimental model reduction** of components
  - **Nonlinear modelling** with reduced models
  - **Model correlation** with experimental data

# Benchmark FE structure: overview

## Substructure 1: knuckle

Etype = tetra10

#elements = 81.9k

#nodes = 108k

E-mod = 70k [N/mm<sup>2</sup>]

Nu = 0.31 [-]

Density = 2.7e-9 [t/mm<sup>3</sup>]

## Substructure 2: carrier

Etype = tetra10

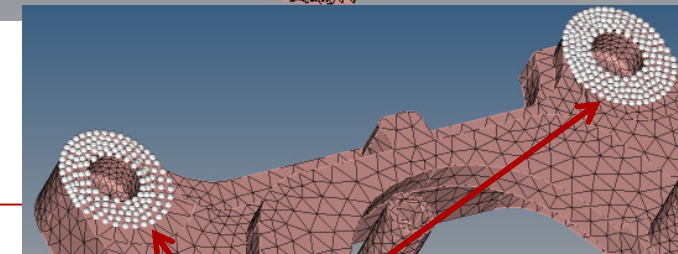
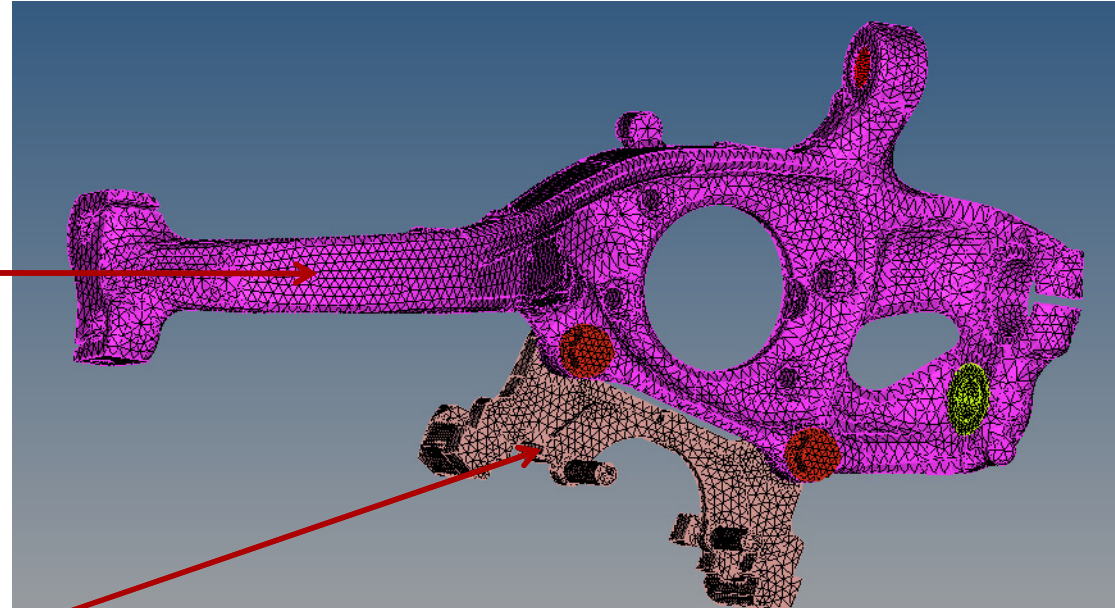
#elemets = 23.5k

#nodes = 41k

E-mod = 190k [N/mm<sup>2</sup>]

Nu = 0.26 [-]

Density = 7.3e-9 [t/mm<sup>3</sup>]

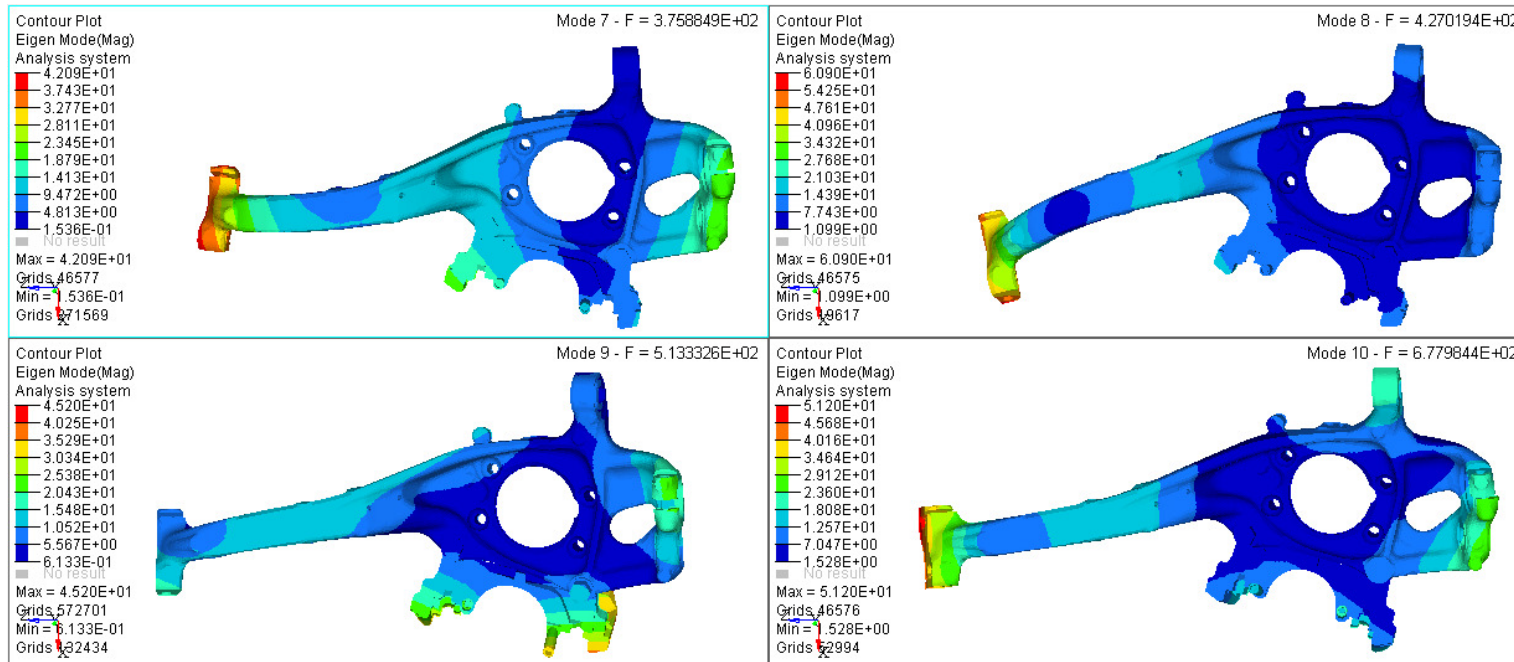


Interface k2c:

Conforming mesh at interface  
#interface nodes = 476

# Validation of linear Structure (I)

- Analytical results: Eigenshapes and eigenfrequencies (linear)



# Validation of linear Structure (II)

- Experimental results: Eigenshapes and eigenfrequencies

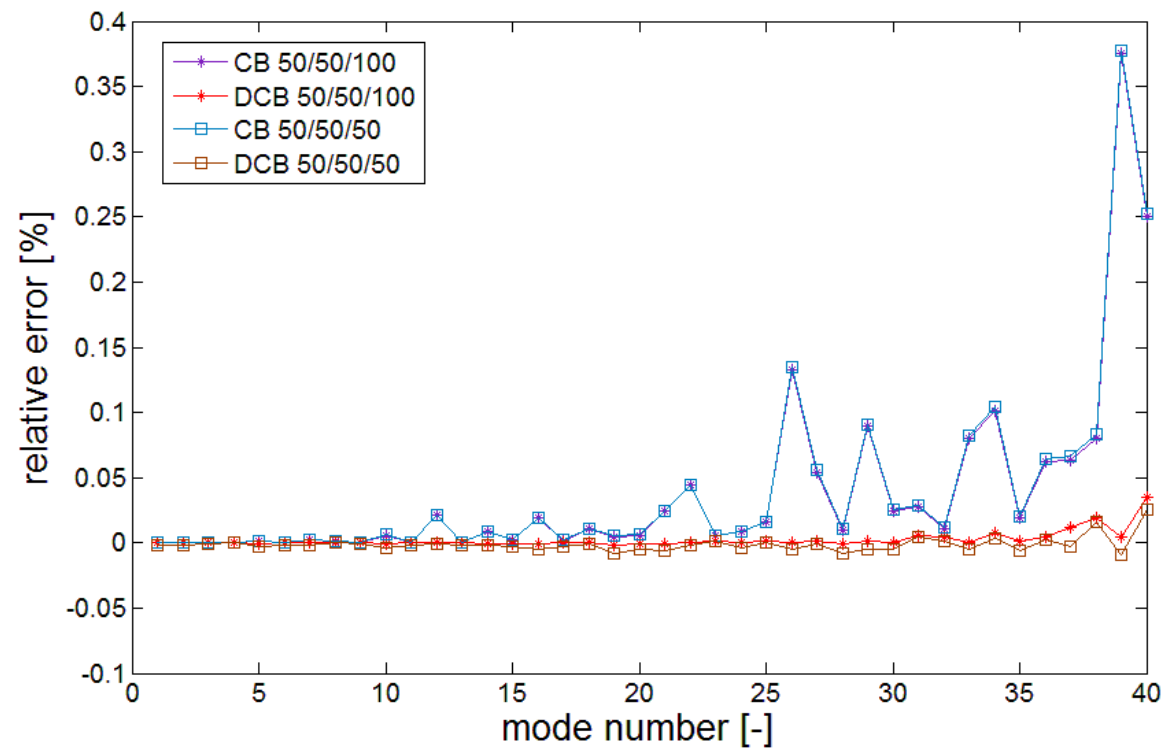


# Model reduction techniques

- Two methods were implemented during the Summer School:
  - **Classical Craig-Bampton** (ref. Craig&Chang)
    - $X = [\Phi_{\text{fixed\_interface}} \ \Psi_c]^*[\eta; u];$
  - **Dual Craig-Bampton** (ref. Rixen)
    - $X = [-G_{\text{res}}^*B^T \ \Phi_{\text{free\_interface}} \ R]^*[\lambda; \eta; \alpha]$
- → linear Interface modes

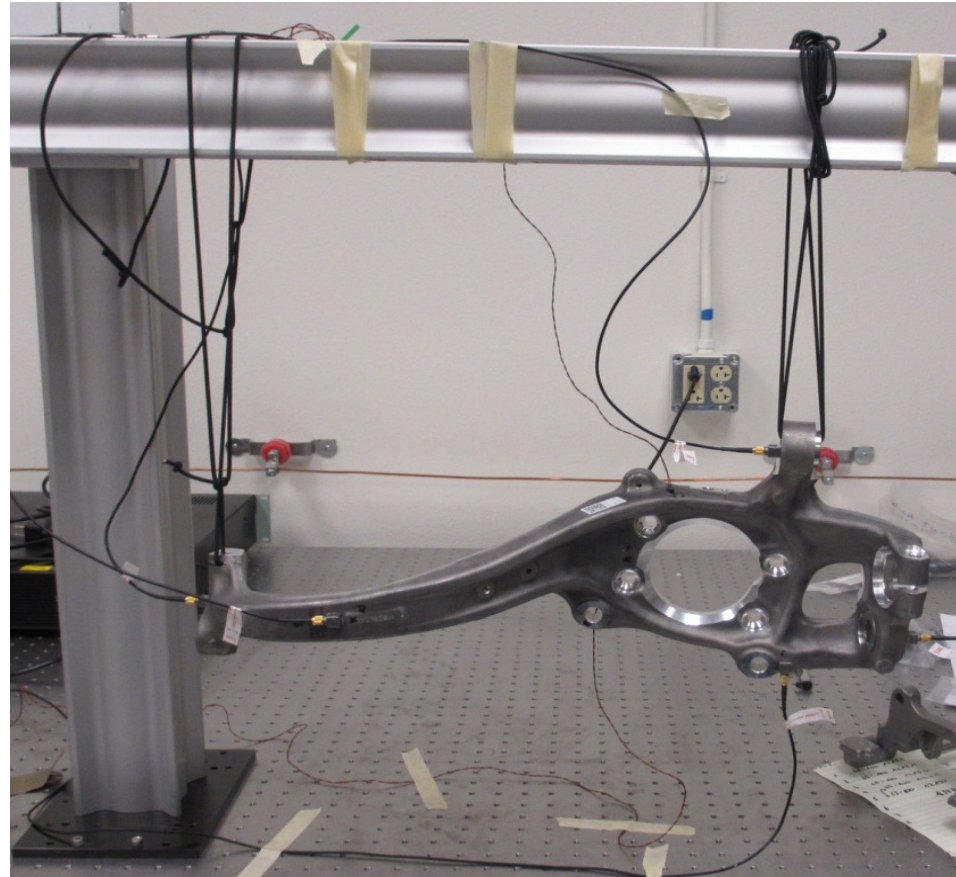
# Analytical reduction of Components

- Performance CB vs. DCB



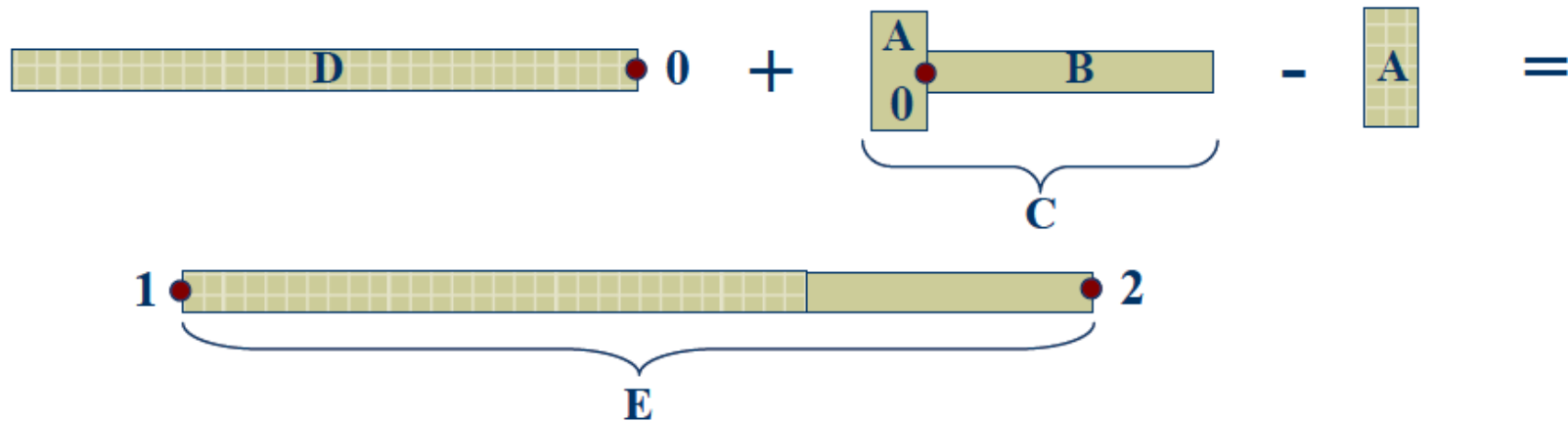


# Experimental model





# Transmission Simulator Approach



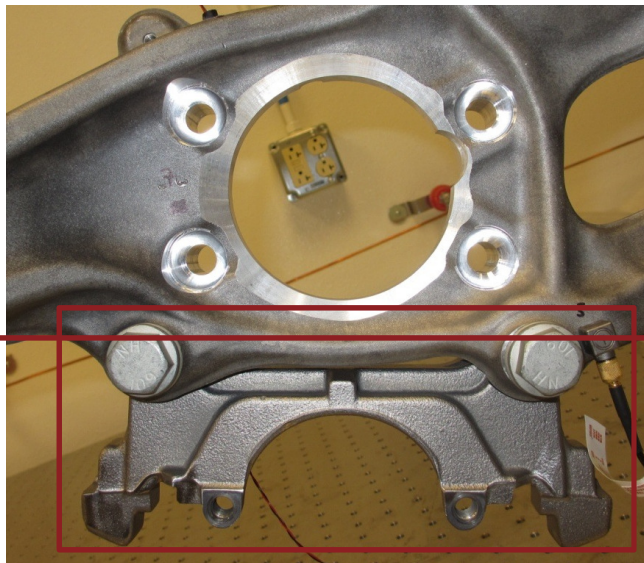
- Measure  $C_{\text{Exp}} = A_{\text{Exp}} + B_{\text{exp}}$
- Subtraction:  $C_{\text{Exp}} - A_{\text{FE}} = B_{\text{Exp}}$
- Couple  $B_{\text{exp}}$  to  $D_{\text{FE}}$  or  $D_{\text{exp}}$  to obtain **E**
- Measure  $E_{\text{truth}}$
- Compare results

# Application

A



B



A

C

- A: Transmission simulator
- B: Brake knuckle
- C: Assembly trans. sim. + knuckle
- D: Brake carrier
- E: Assembly carrier + knuckle

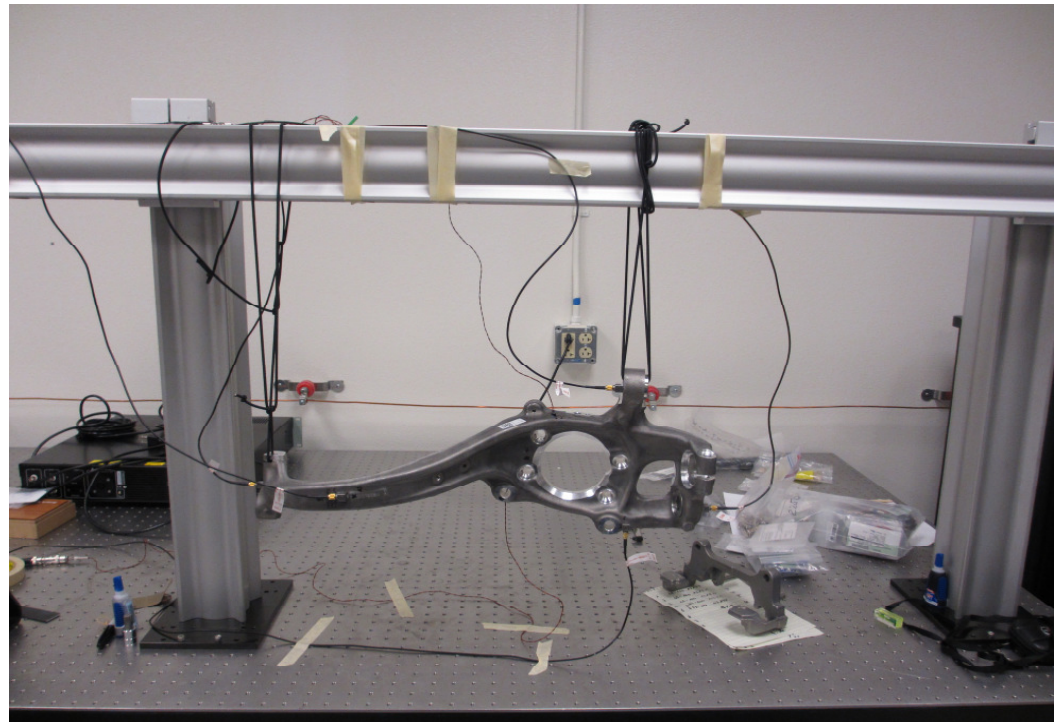
# Testing Approach

- Excitation Method: Impact Hammer (PCB 06C05)
- Measurement Method: Accelerometers (PCB 352A24s, Endev 65-100s)
- DAQ and Collection Software: NX-Ideas
- FEM Model/accelerometer locations calculated from model supplied by Audi
- Hit in line of accelerometers, either directly on with a cap or as close as possible
- Free-Free boundary
- Collected FRF Coherence/Auto Spectra to 2000Hz



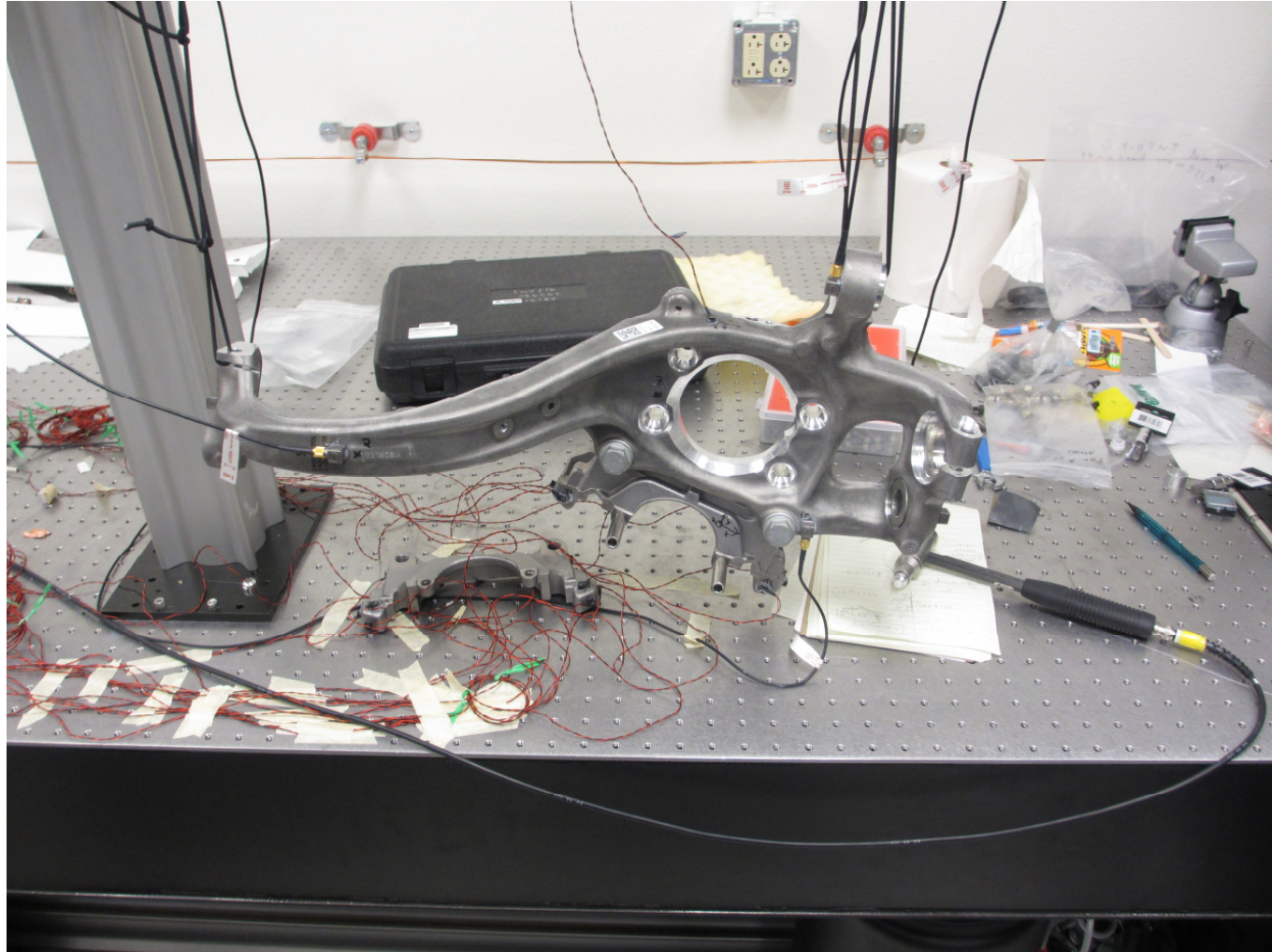
# Bare Knuckle

- Typically used 1.5-2x the accelerometers “theoretically” needed
- This figured to 5 Triaxial and 2 Uniaxial for the Knuckle alone

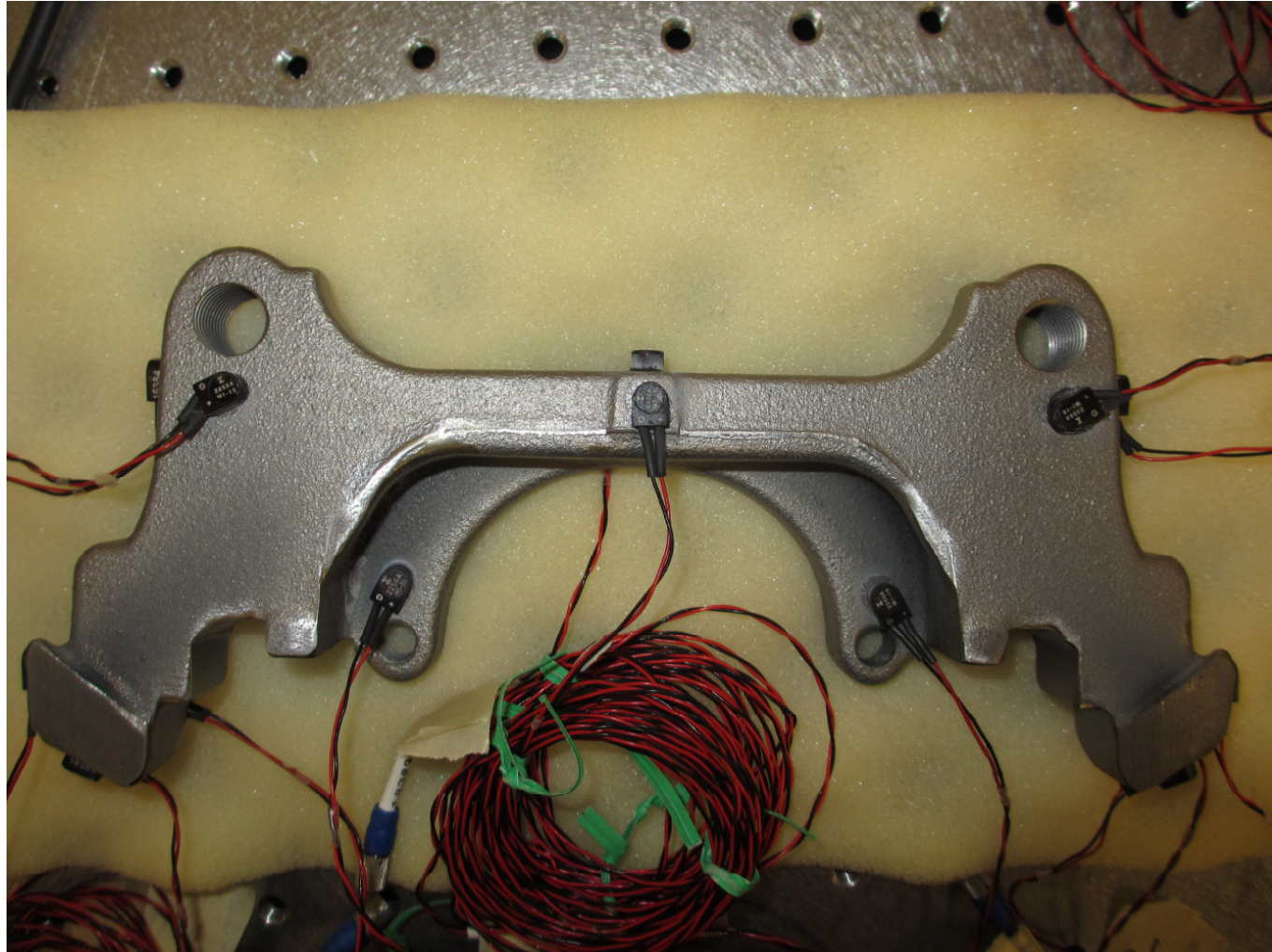




# Knuckle + Carrier (6 extra accels)



# Transmission Simulator





# TS+Knuckle

