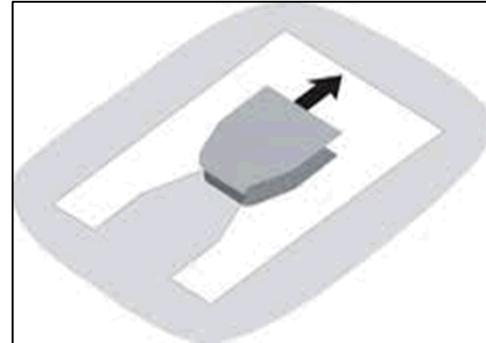


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# Using Tearing as a Method to Calculate Adhesion

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# Presentation Overview

- Introduction and Background
- Need Statement
- Theoretical Development
- Method of Approach
  - Preliminary Experiments
  - Test Set up

# Introduction and Background

- Adhesion is a property that can be seen nearly everywhere in daily life



# Introduction and Background

- Adhesion also has a large number of applications for the military

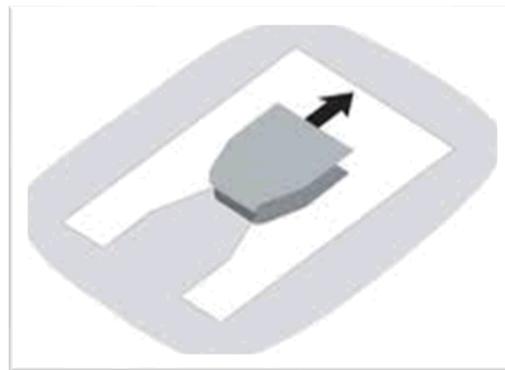
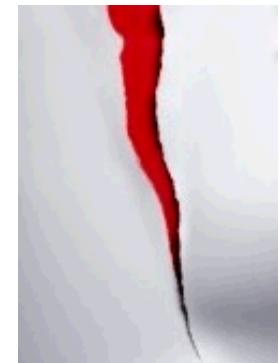


- Corrosion Inhibitors
- Radar Absorbing
- UV Protection

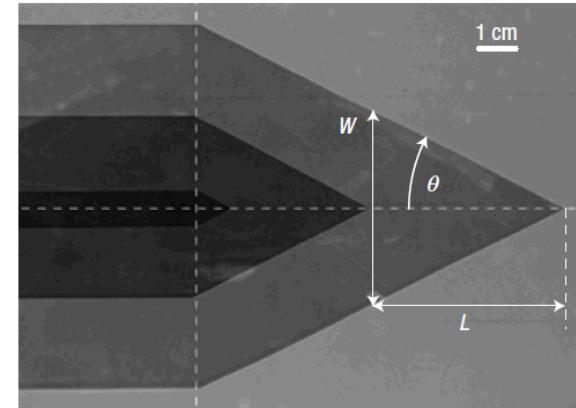
to name a few ....

# Introduction and Background

- When a material is torn, the tear will naturally take a converging path
- This path leads to the lowest energy state
- This natural tendency can be seen in everyday life



Hamm et al., 2008



Hamm et al., 2008

# Need Statement

- A method to quantify the adhesive strength between a polymer coating and a given substrate is needed.
- Additionally, a robust experimental approach that can be used to evaluate a variety of substrates is needed.

# Theory

## Energy of the System

Total Energy      Elastic Energy      Fracture Energy      Adhesion Energy

$$U = U_E + 2\gamma ts + \tau A$$

Equilibrium of a half-strip requires:

$$F = \tau \frac{W}{2} + \gamma t \cos \theta$$

$$(\frac{\partial U_E}{\partial W})_{x,\ell} = \gamma t \sin \theta$$

The flexural strain energy,  $U_E$ , is given as:

$$U_E = \frac{4BW}{h}, \text{ where } B = \frac{Et^3}{12(1-v^2)}$$

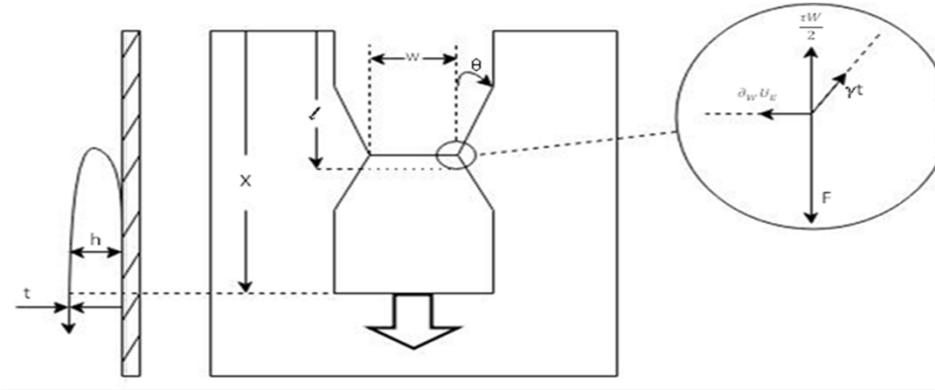
$$\text{Then, } (\frac{\partial U_E}{\partial W})_{x,\ell} = \frac{4B}{h}$$

So,

$$\sin \theta = \frac{4B}{\gamma th}$$

The Force,  $F$ , in the force balance equation can also be described using the Work Theorem as:

$$F = (\partial_x U_E)_{W,\ell}$$



Therefore, taking  $F = (\partial_x U_E)_{W,\ell}$  gives:

$$F = \frac{\partial}{\partial x} \left( \frac{4BW}{h} \right) = -\frac{4BW}{h^2} \frac{\partial h}{\partial x}$$

$$\text{Letting } \eta^2 = -\frac{\partial h}{\partial x}$$

We can then say:

$$F = 4\eta^2 \frac{BW}{h^2}$$

For equilibrium:

$$4\eta^2 \frac{BW}{h^2} = \tau \frac{W}{2} + \gamma t \cos \theta$$

For large values of  $W$ ,  $\gamma t \cos \theta$  is negligible.

Solving for  $h$  then gives

$$h = 2\eta \sqrt{\frac{2B}{\tau}}$$

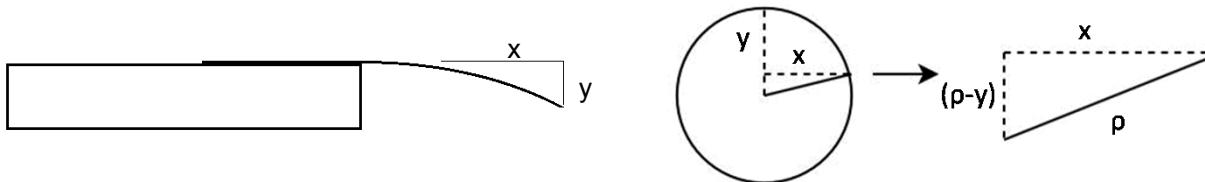
$$\text{So, } \sin \theta = \frac{\sqrt{2B\tau}}{\eta\gamma t} \quad \text{or} \quad \tau = \frac{(\eta\gamma t \sin \theta)^2}{2B}$$

# Method of Approach

- Before starting the tearing experiment, the bending stiffness ( $B$ ), fracture force ( $\gamma t$ ), and the parameter  $\eta$  must be determined
- Bending stiffness,  $B$ , can be found by using a simple experiment based on elastic beam theory
- Fracture force,  $\gamma t$ , can be found by using ASTM 1938
- $\eta$  must be found graphically

$$\tau = \frac{(\eta \gamma t \sin \theta)^2}{2B}$$

# Finding Bending Stiffness



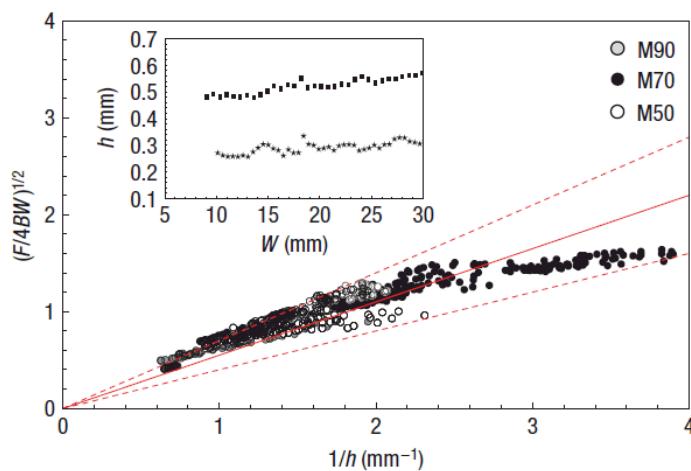
- The mass per unit length must be found and then the moment,  $M$ , calculated
- The radius of curvature,  $\rho$ , can be found using the Pythagorean theorem where  $x$  and  $y$  are known
- Utilizing the follow equation, the Bending stiffness can be found

$$M = B \frac{\partial^2 y}{\partial x^2}$$

- In this case  $\frac{\partial^2 y}{\partial x^2} = \frac{1}{\rho}$

# Finding $\eta$

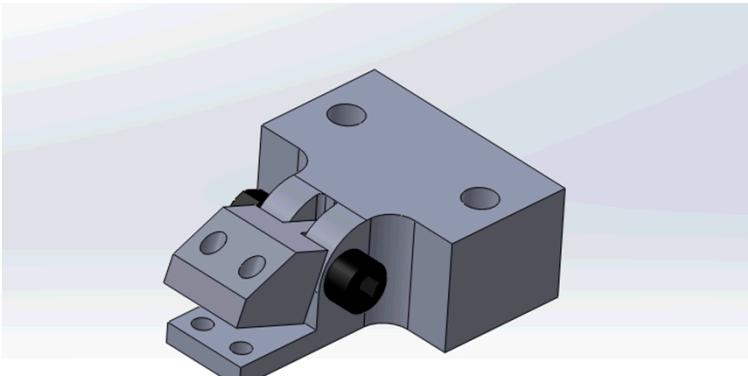
- Under the assumption that  $\eta$  is a constant and  $h$  is a variable,  $F = 4\eta^2 \frac{BW}{h^2}$  can be rearranged  $\sqrt{\frac{F}{4BW}} = \frac{1}{h}\eta$  and plotted
- Once plotted, the slope of the line can be extracted and will give the value of  $\eta$  (show typical plot)



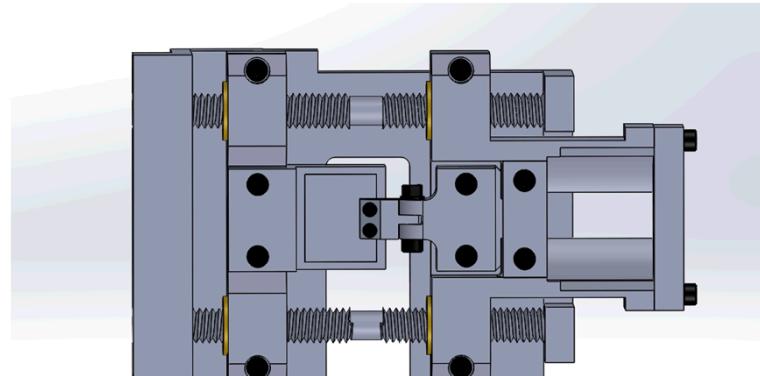
Hamm et al., 2008

- The equation  $\sin \theta = \frac{4B}{\gamma th}$  implies that  $h$ , is a constant. However, in reality, there are small variations in  $h$  which allow us to assume this relation is correct, but these variations are measured and used to help find  $\eta$

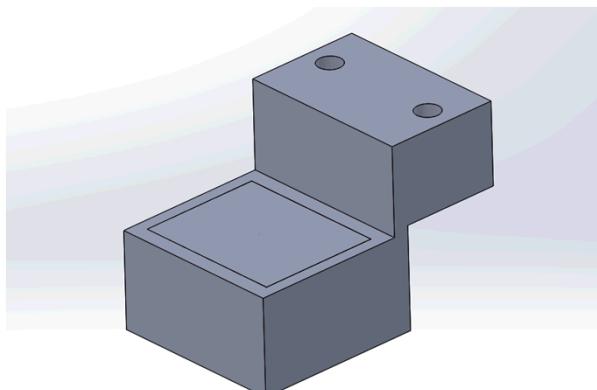
# Test Set up



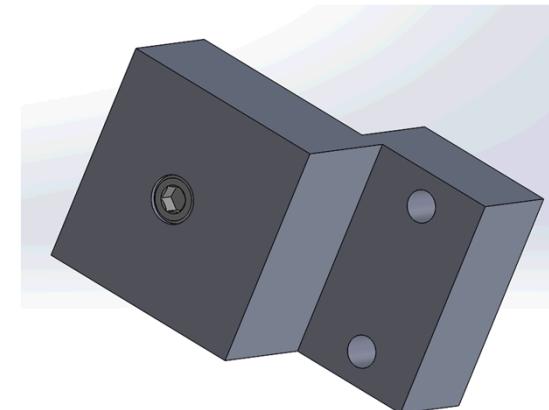
Initial Clamp Design



Top View of Test Set up

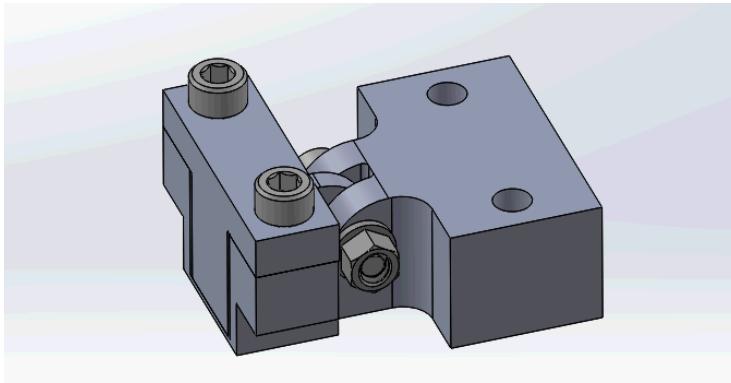


Substrate frame with Substrate Plate

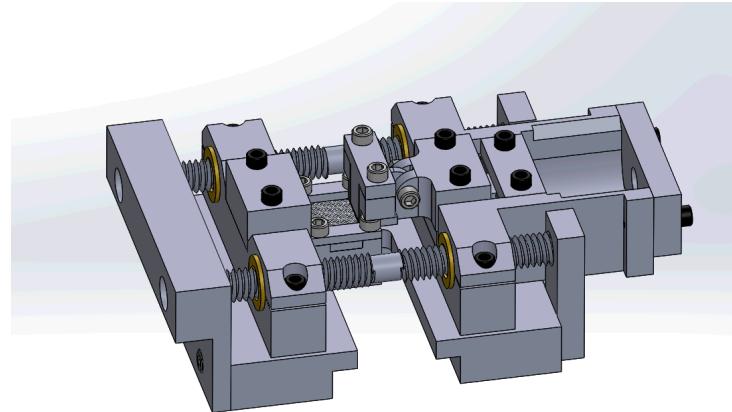


Bottom of Substrate frame

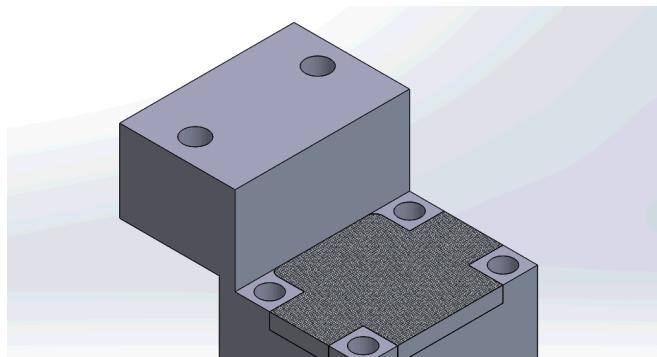
# Test Set up



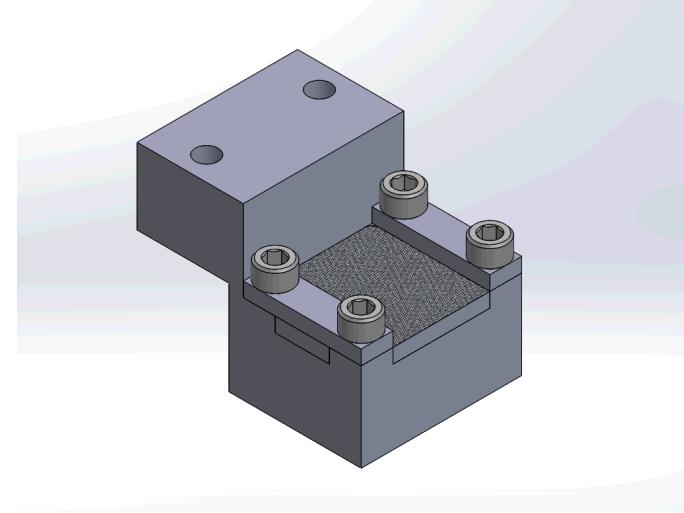
Updated Clamp design



Updated Test Set up



Modified Substrate Frame



Modified Substrate Frame with top plate

# Summary

- Adhesion was shown to be a characteristic needed for a variety of critical applications
- The adhesion strength was shown to be governed by elastic and fracture properties
- A delineation between the material properties and geometric parameters was made
- A conceptual design was created in order to quantify the adhesion strength for a variety of different substrates
- The design is ready to move forward to the manufacturing stage for proof of concept

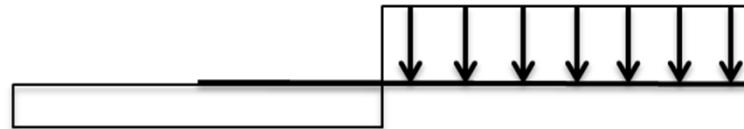
# Acknowledgements

- Tim Briggs- A big thank you to Tim for helping me break down and understand the math and theory behind the nature of tearing
- Staci Dorsey- Thank you to Staci for setting up the intership and giving me this opportunity

# Questions?

# References

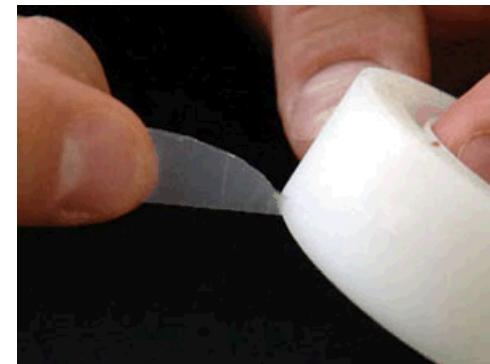
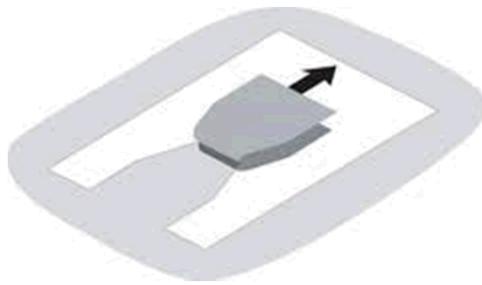
1. Hamm, E., Reis, P., LeBlanc, M., Roman, B. & Cerdá, E. Tearing as a test for mechanical characterization of thin adhesive films. *Nature Materials* **7**, 386-390 (2008).
2. ASTM standards
3. Books and other papers



# Theory

- When a film is peeled from a substrate, the tear will naturally take a converging path
- The first step in finding adhesion from this natural path is to mathematically describe it
- To start, an equation to characterize the energy of the system is created

$$U = U_E + 2\gamma ts + \tau A \quad (1)$$

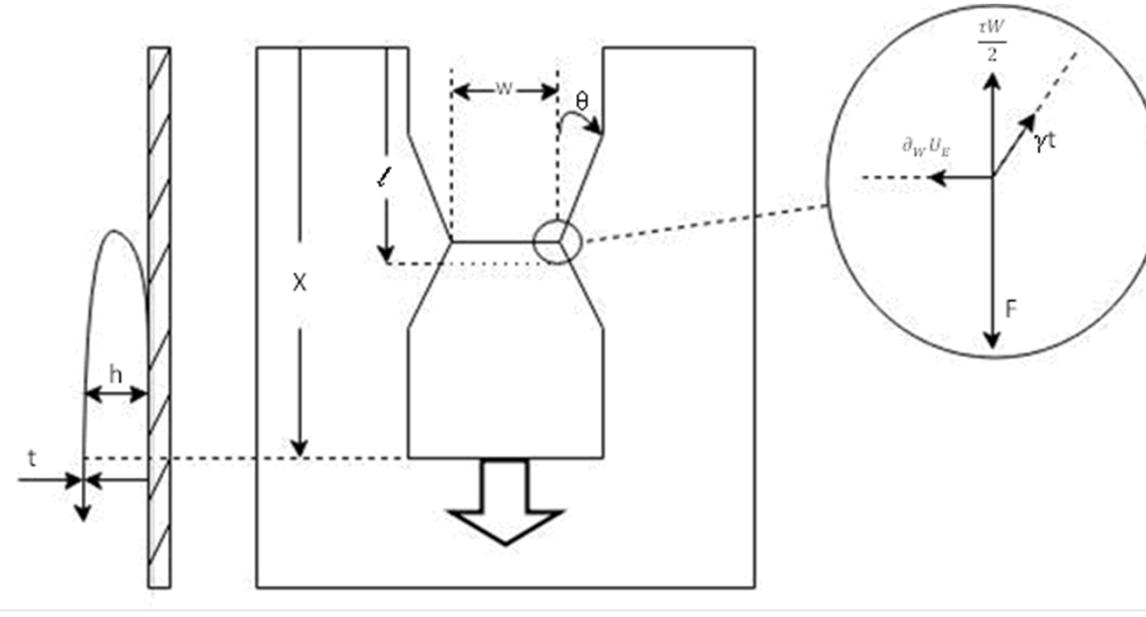


# Theory

- From the energy equation, a balance of forces can be derived for  $\frac{1}{2}$  of the strip and yields:

$$F = \tau \frac{W}{2} + \gamma t \cos \theta \quad (2)$$

$$\left(\frac{\partial U_E}{\partial W}\right)_{x,\ell} = \gamma t \sin \theta \quad (3)$$



# Theory

- The classical elasitca of Euler yields that the elastic energy available is  $U_E = \frac{4BW}{h}$ , where  $B$  is the Bending Stiffness
- Plugging the equation for elastic energy into  $(\frac{\partial U_E}{\partial W})_{x,\ell} = \gamma t \sin \theta$ , equation (3) now becomes

$$\sin \theta = \frac{4B}{\gamma t h} \quad (4)$$

$$U = U_E + 2\gamma ts + \tau A \quad (1)$$

$$F = \tau \frac{W}{2} + \gamma t \cos \theta \quad (2)$$

$$(\frac{\partial U_E}{\partial W})_{x,\ell} = \gamma t \sin \theta \quad (3)$$

- Because  $\sin \theta$  is a constant, this new equation implies that the height,  $h$ , is constant.

# Theory

- The force in equation (2) can also be described as  

$$F = (\partial_x U_E)_{W,\ell}$$
- To make the partial derivative of the elastic energy work in the above equation, a new parameter,  $\eta^2$ , is defined.  

$$\eta^2 = -\partial_x h$$
- By using  $\eta^2$  and the equation for elastic energy, the force in equation (2) can now be written as  $F = 4\eta^2 \frac{BW}{h^2}$
- Substituting in the above equation for the force in equation (2) yields

$$4\eta^2 \frac{BW}{h^2} = \tau \frac{W}{2} + \gamma t \cos \theta \quad (5)$$

$$U = U_E + 2\gamma ts + \tau A \quad (1)$$

$$F = \tau \frac{W}{2} + \gamma t \cos \theta \quad (2)$$

$$(\frac{\partial U_E}{\partial W})_{x,\ell} = \gamma t \sin \theta \quad (3)$$

$$\sin \theta = \frac{4B}{\gamma th} \quad (4)$$

# Theory

- It was found that for large values of  $W$  that the last term in equation (5) is negligible
- This allows us to solve for the height,  $h$ , through the elimination of the last term in equation (5) and gives us that  $h = 2\eta\sqrt{\frac{2B}{\tau}}$
- By plugging the above equation for height into equation (4), we get the final equation needed to find adhesion and yields

$$\sin \theta = \frac{\sqrt{2B\tau}}{\eta\gamma t} \quad (6)$$

- A quick rearrangement of equation (6) gives the more usable form of

$$\tau = \frac{(\eta\gamma t \sin \theta)^2}{2B}$$

$$U = U_E + 2\gamma ts + \tau A \quad (1)$$

$$\sin \theta = \frac{4B}{\gamma t h} \quad (4)$$

$$4\eta^2 \frac{BW}{h^2} = \tau \frac{W}{2} + \gamma t \cos \theta \quad (5)$$