

Friction and Adhesion in MEMS

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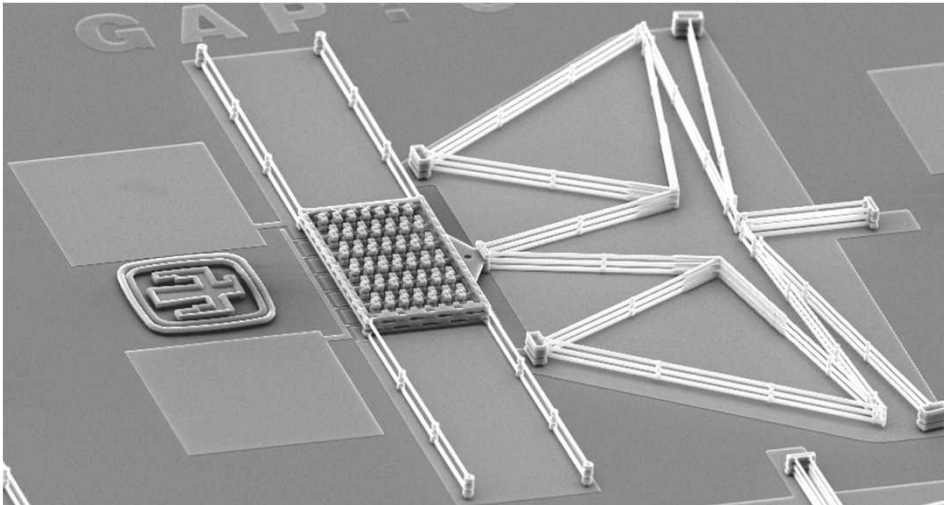
Acknowledgments

- **Sandia**
 - Alex Corwin
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 - Frank DelRio
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- **U. Wisconsin**
 - Rob Carpick

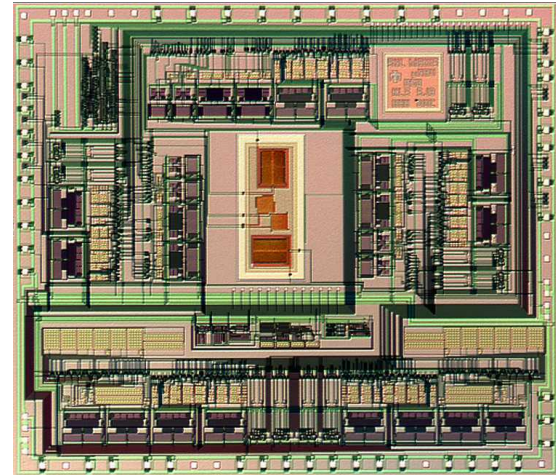
With polysilicon MEMS we can reliably accomplish electromechanical and optical functions

- thousands of devices simultaneously
- no assembly required
- hundreds of device concepts explored

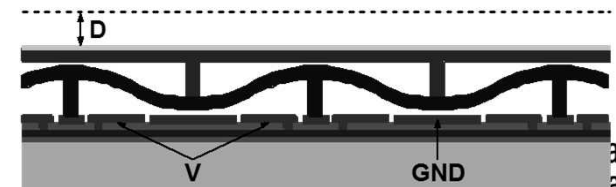
High performance comb drive
with mechanical amplifier



Integrated inertial sensor

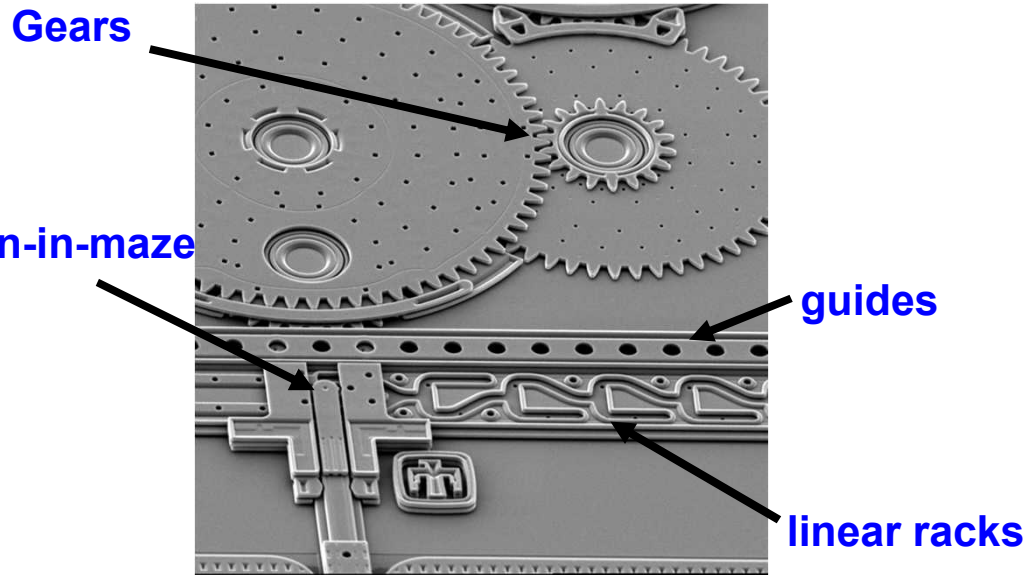


Polychromator :
programmable
diffraction grating

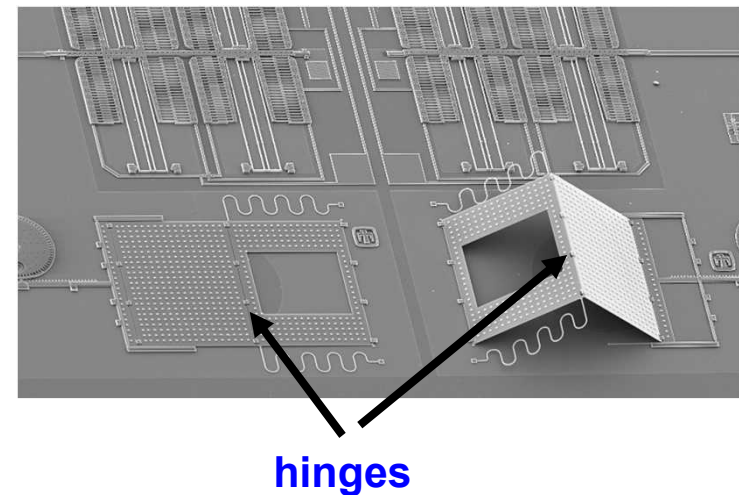


Allowing contact between MEMS surfaces significantly broadens the design space

Complex Mechanical Logic



Pop-up Mirrors



but ...

static friction can dominate the forces required

dynamic friction can dominate energy loss

adhesion, friction and wear become the most important failure mechanisms of contacting MEMS

MEMS – surface micromachining implementation

A series of structural and sacrificial layers are deposited

Ground plane layer (Poly 0)
4 structural levels
(Poly 1 - Poly 4)

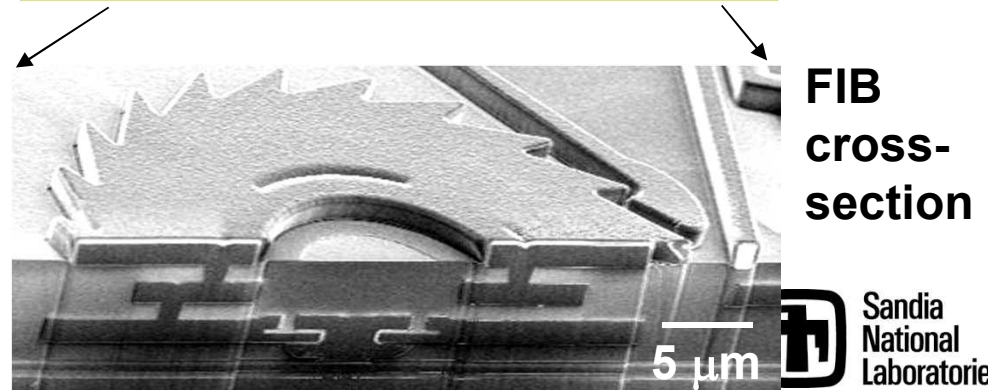
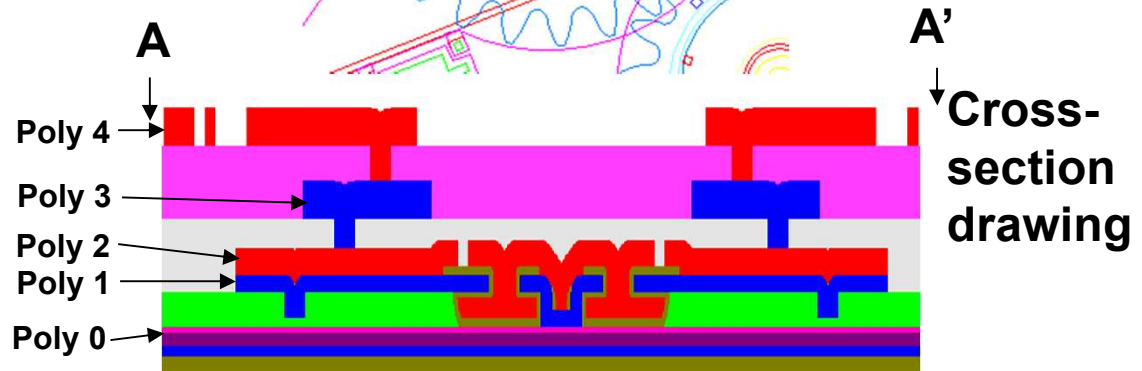
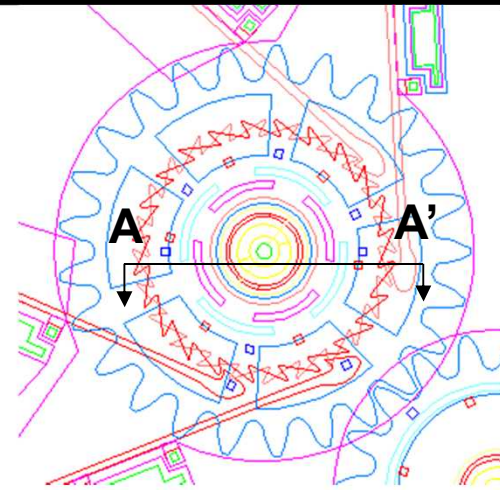
Chemical Mechanical
Planarization (CMP)

1 μm design rule

Create freestanding thin film
structures by “release”
process

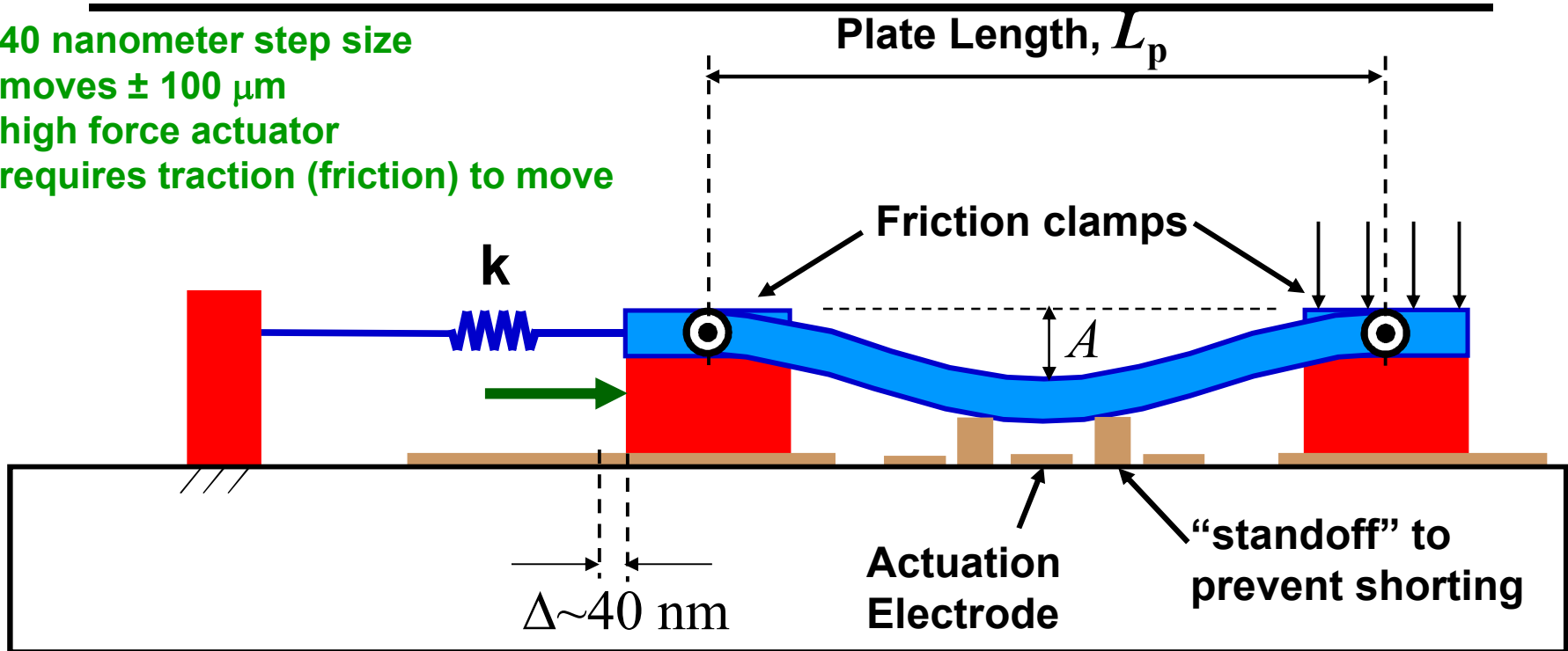
Sniegowski & de Boer,
Annu. Rev. Mater. Sci.
(2000)

Design



Friction can be good: We developed a high-performance friction-based actuator

- 40 nanometer step size
- moves $\pm 100 \mu\text{m}$
- high force actuator
- requires traction (friction) to move

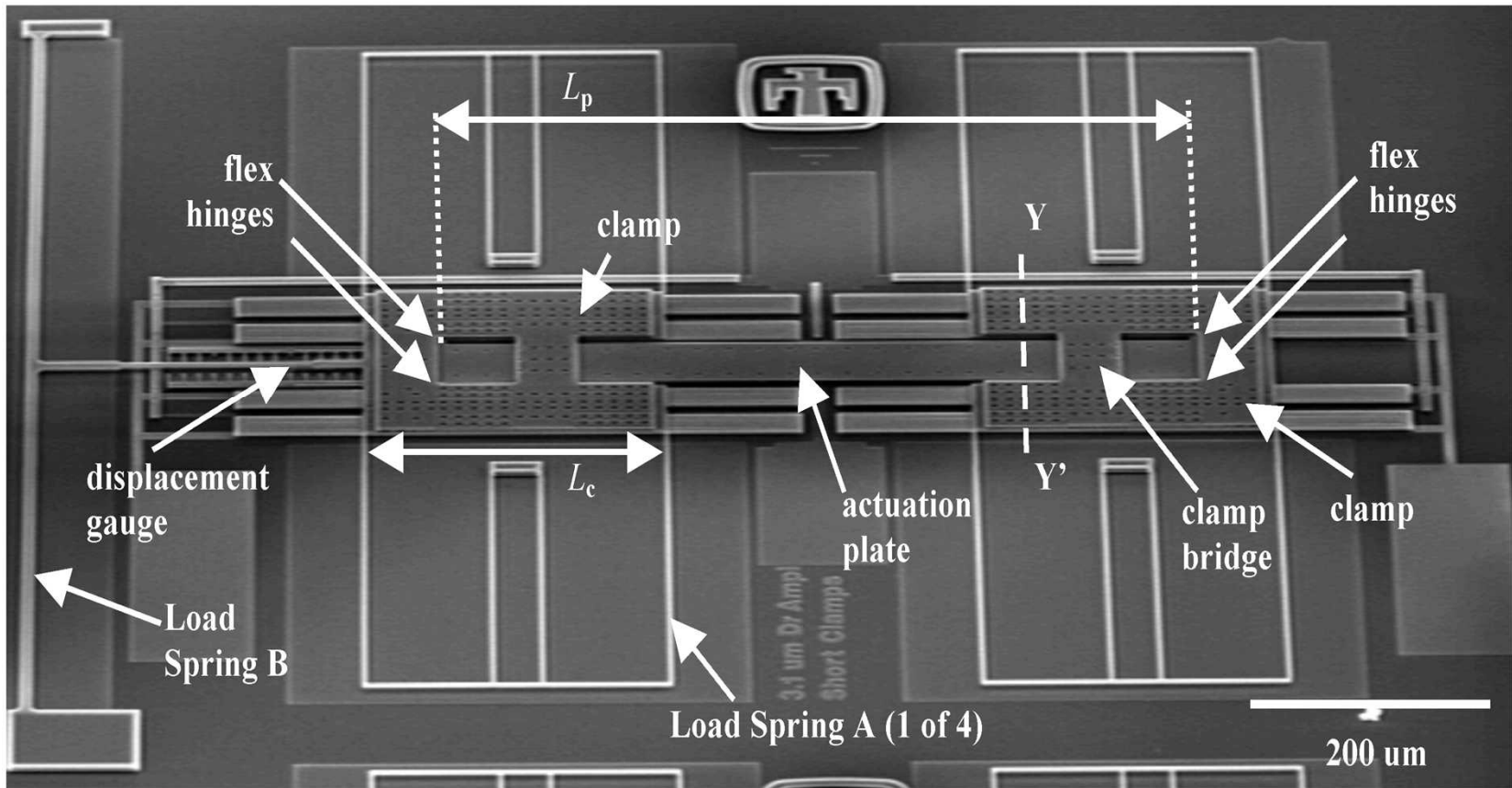


$$F_{\text{max}} \sim 2 \left(\frac{A}{L_p} \right)^2 \approx 1 \text{ mN}$$

**large tangential
force range**

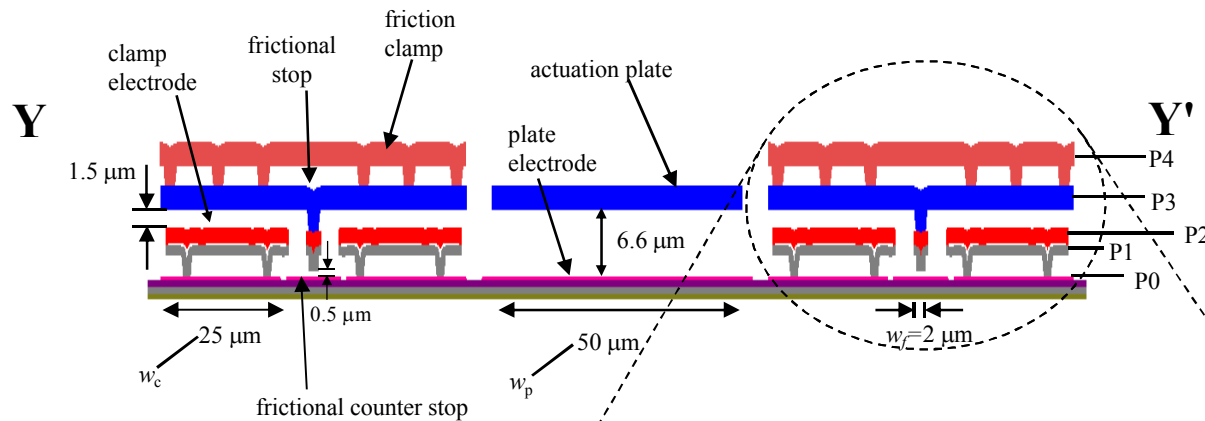


Nanotractor - SEM

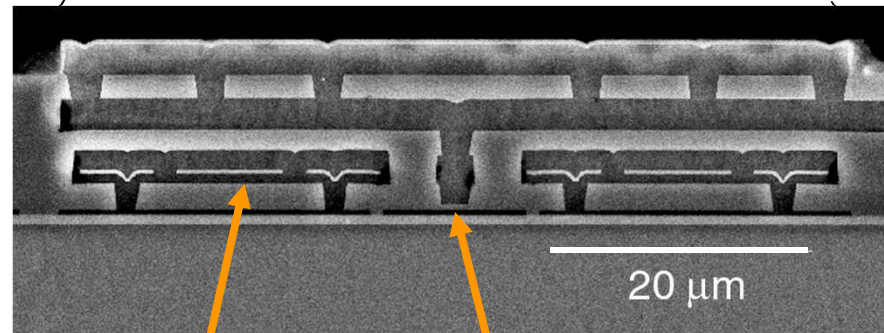


M. P. de Boer, D. L. Luck, W. R. Ashurst et al.
“High Performance Surface-Micromachined Inchworm Actuator”
J. MicroElectroMechanical Systems, Feb. 2004

Nanotractor – clamp cross section



**SEM cross-section
(before release)**



Normal force is applied electrostatically and borne mechanically

We can apply normal force from $1 \mu\text{N}$ to 10 mN with this arrangement

large normal force range

MEMS monolayer coupling agent

FOTAS (tridecafluoro-1,1,2,2-tetrahydrodecyltris(dimethylamino)silane)

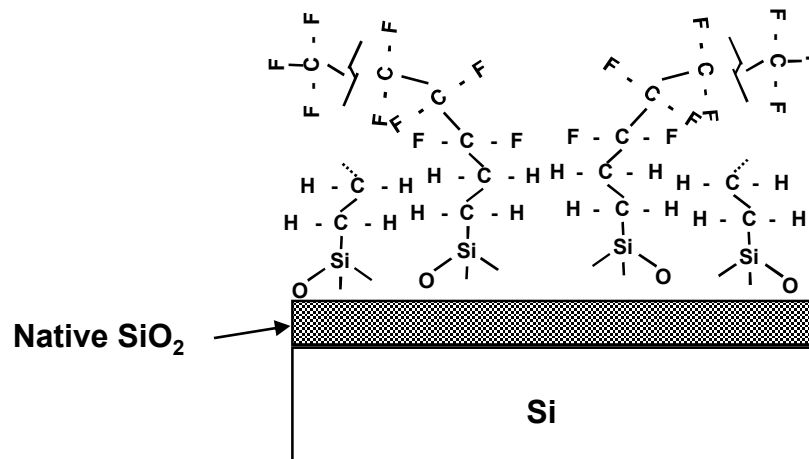
vapor deposition

8 carbon chain

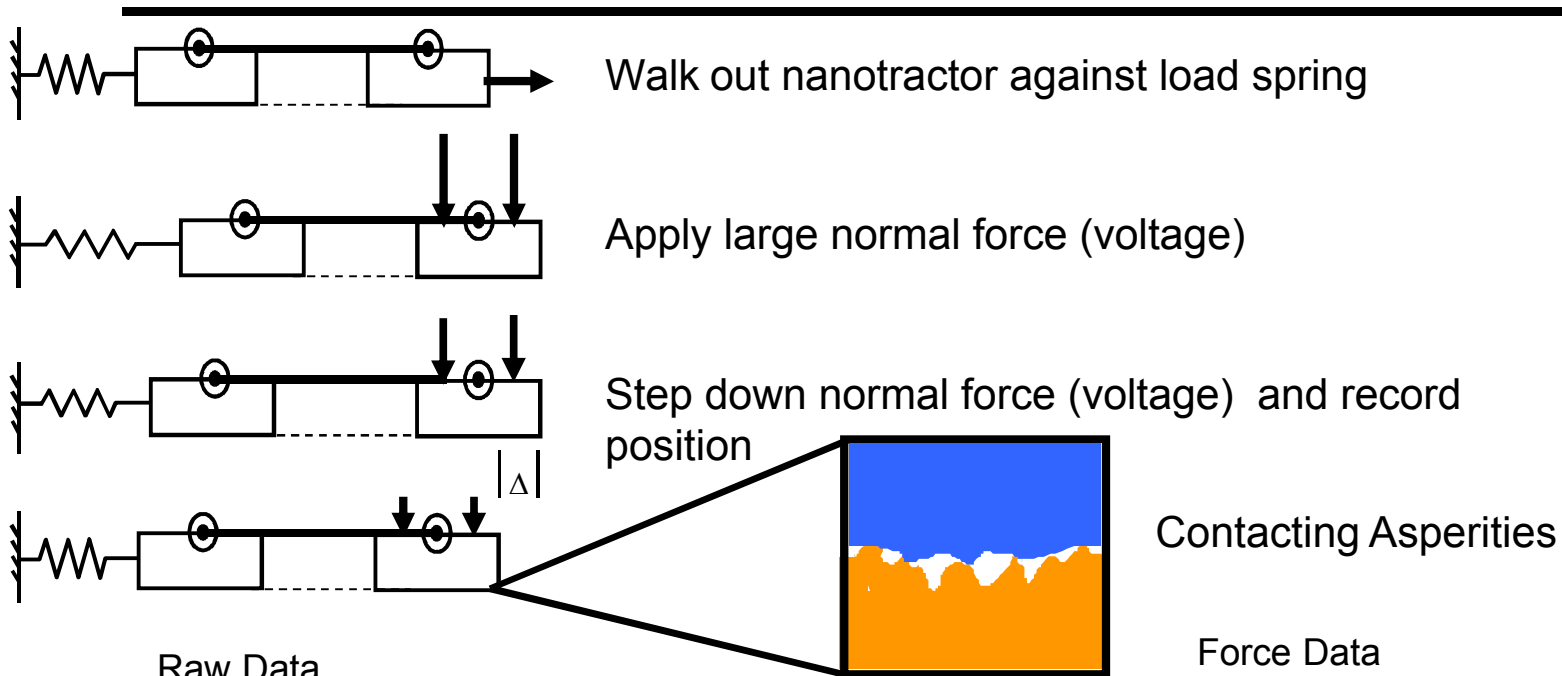
van der Waals forces not strong enough to self assemble (tangled)

contact angle $\sim 110^\circ$

FOTAS 8-carbon
fluorinated chain
(disordered, tangled)

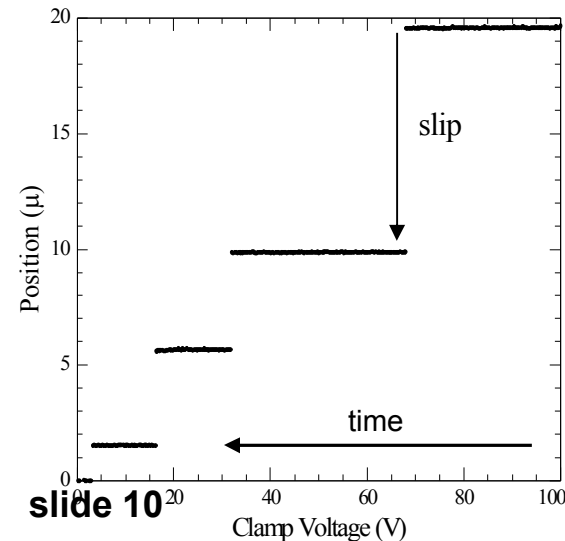


Static friction measurement

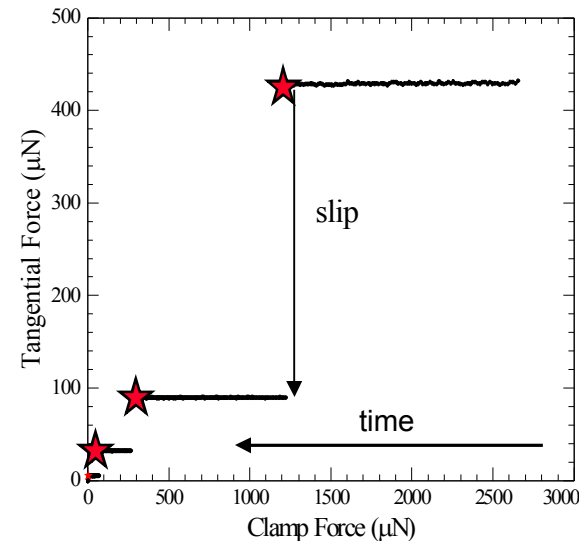


Raw Data

Force Data



Convert clamp voltage to clamp force and position to tangential force

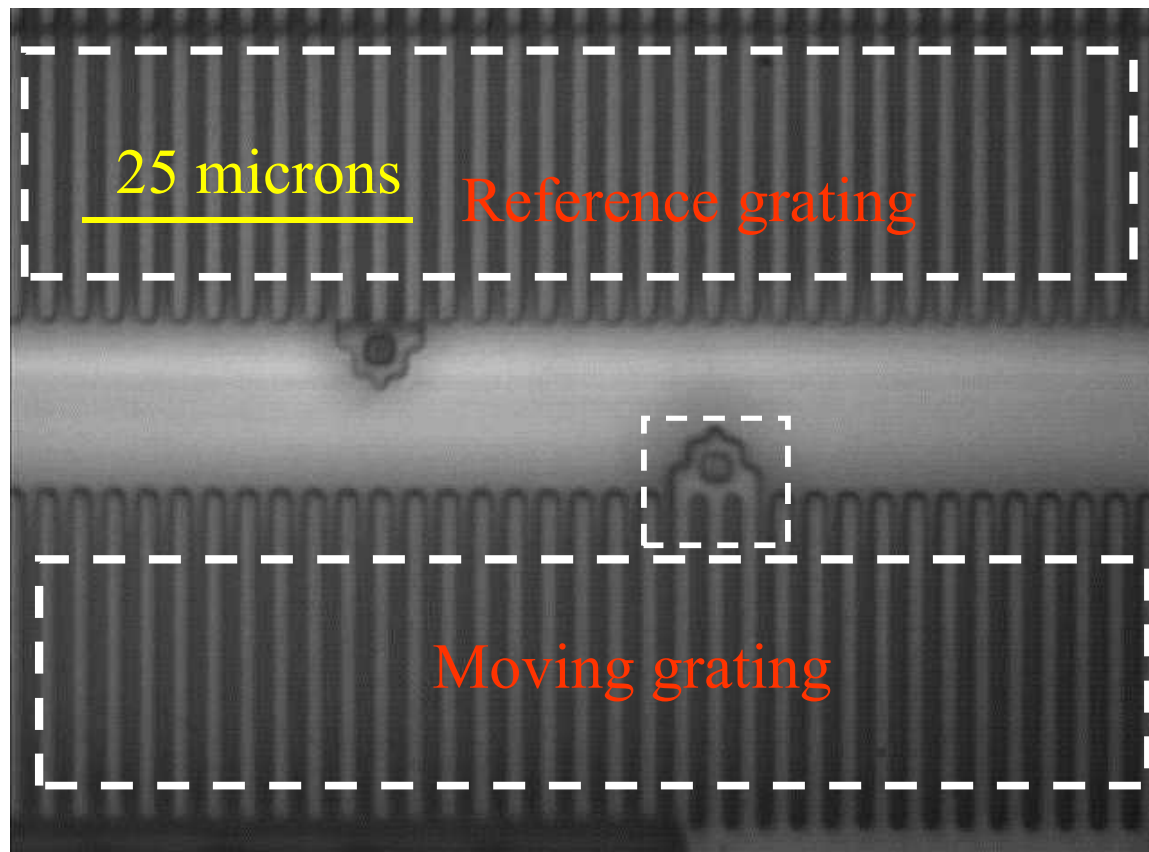


Identify equilibrium points right before jumps where

$$F_{Tan} = \mu_s F_N$$

Highly resolved slip measurements: we measure position to 1 nm

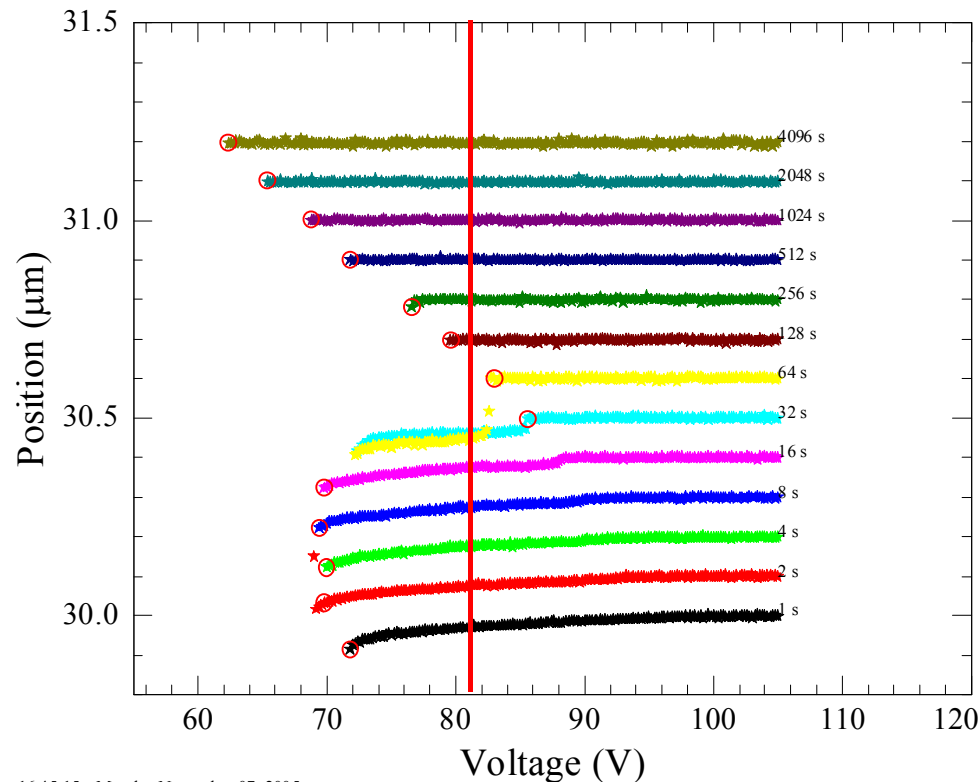
Use periodic grating and measure relative phase to 1 part in 2500



***As hold time increases, static friction increases.
For short hold times we get small-scale sliding.***

H:/friction_tests/waits/newfotos/r5277/L495/c3/d3/r14

Overlay of all Position versus Voltage data



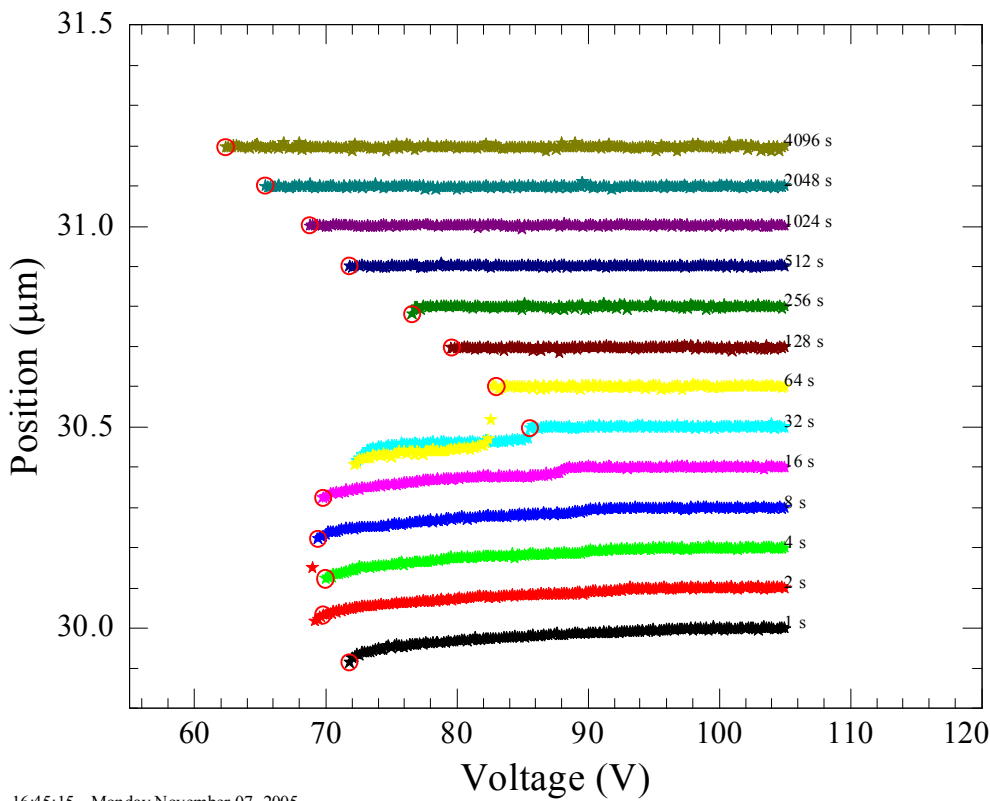
16:45:15 - Monday November 07, 2005

Jumps that occur above this critical voltage V_c (lower apparent friction) end up with PSTD, while those released with voltage below it (higher apparent static friction) end up jumping

At longer hold times, we obtain a typical u_s time-dependence

H:/friction_tests/waits/newfotas/r5277/L495/c3/d3/r14

Overlay of all Position versus Voltage data

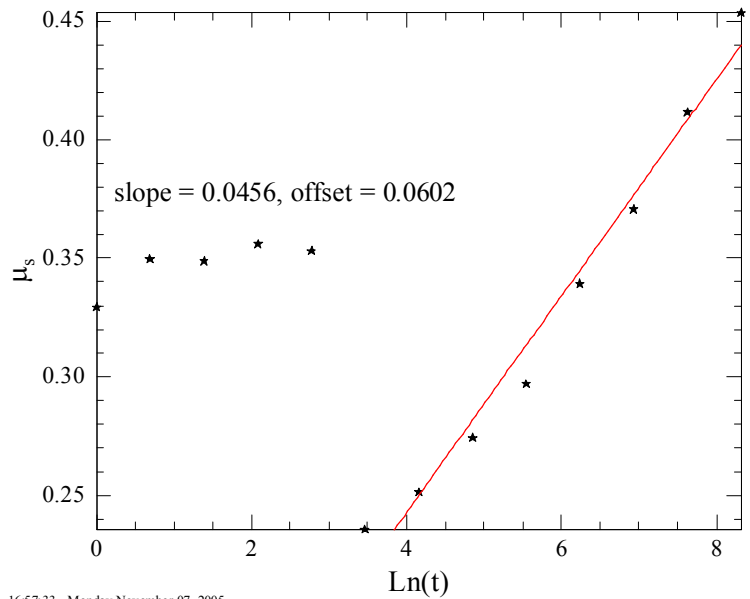


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From the plot on the left, we should only consider times of 32 seconds or longer as having well defined jumps

H:/friction_tests/waits/newfotas/r5277/L495/c3/d3/r14

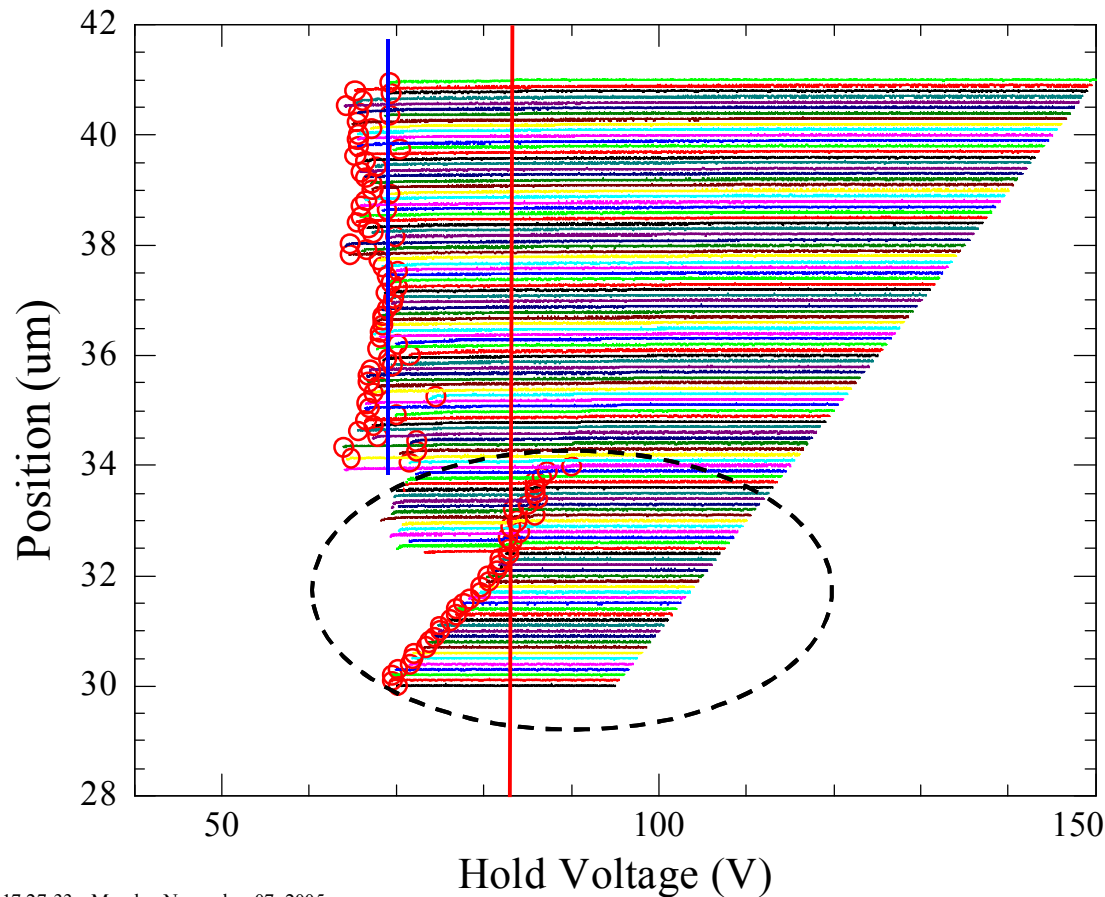
μ_s as a function of $\ln(t)$ for $V_{start} = 105$



16:57:33 - Monday November 07, 2005

When we hold the clamp with a larger normal force, we experience a correspondingly lower friction force

H:/friction_tests/waits/newfotas/r5277/L495/c3/D3/r26

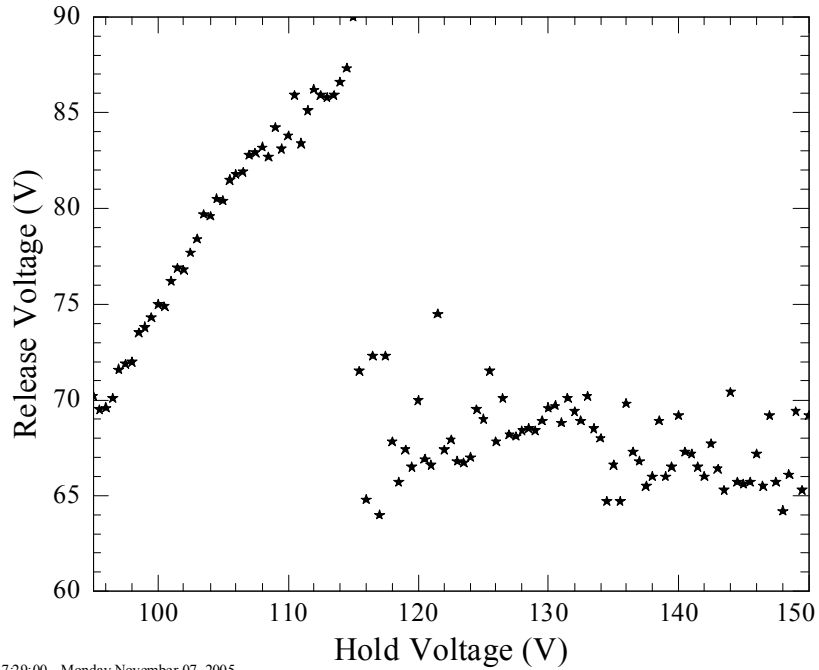


(64 sec hold)

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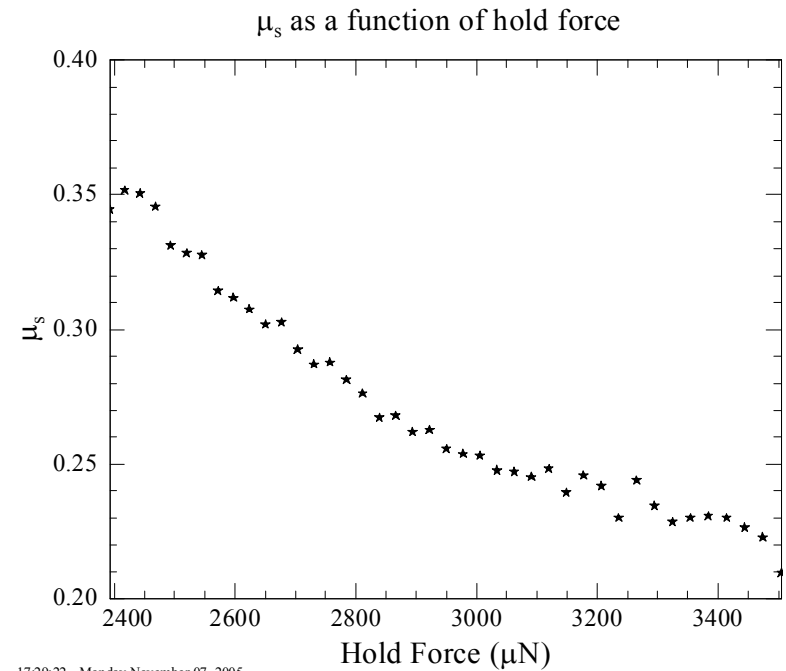
u_s decreases with hold force

H:/friction_tests/waits/newfotos/r5277/L495/c3/D3/r26



17:29:00 - Monday November 07, 2005

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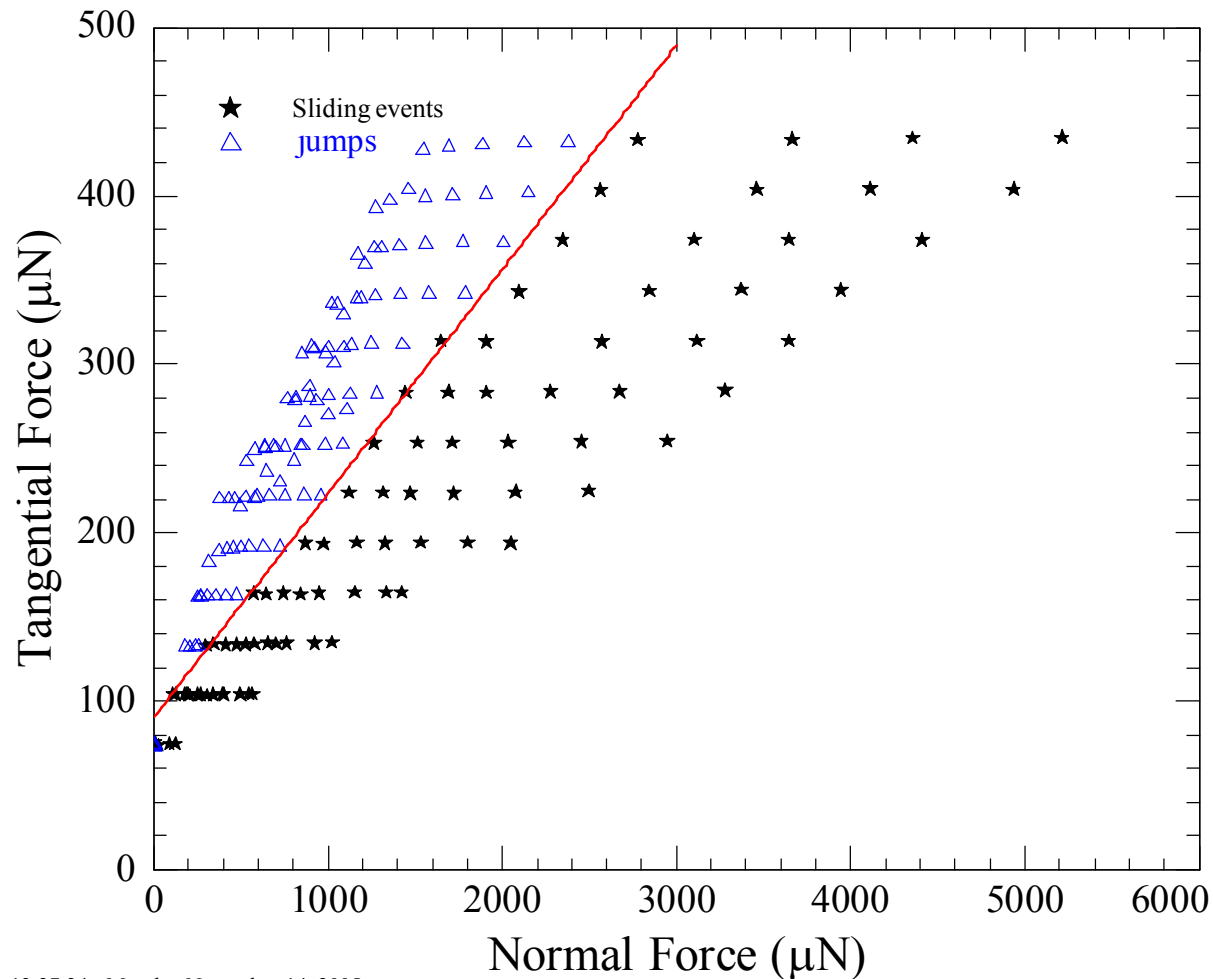


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We observe a bifurcation in the jumps versus sliding events

C:/friction_tests/nforce_manypos/newfotas/r5277/L495/c5/D3/r5

Phase diagram of Jumps and Sliding Events



Summary of friction data

The nanotractor can be used to study friction of surface-micromachined interfaces

With FOTAS coating:

$$u_s(t) = \alpha + \beta \ln(t)$$

u_s decreases with increasing hold voltage (counter-intuitive)

If motion begins above (below) a certain critical voltage, sliding (a jump) is observed.

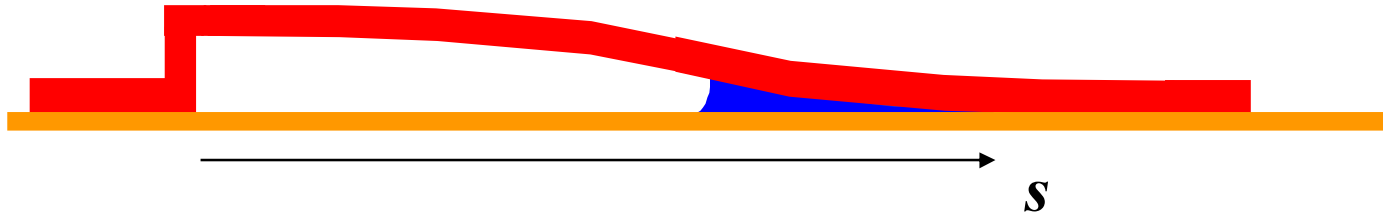
Phase space can be mapped out – this behavior occurs independent of tangential load

Adhesion (e.g., “stiction”) is a big problem in micromachining

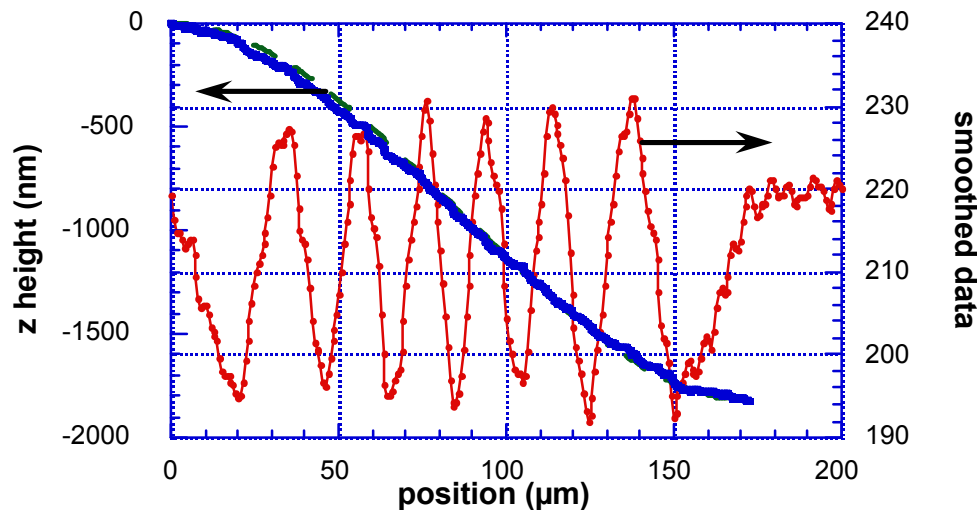
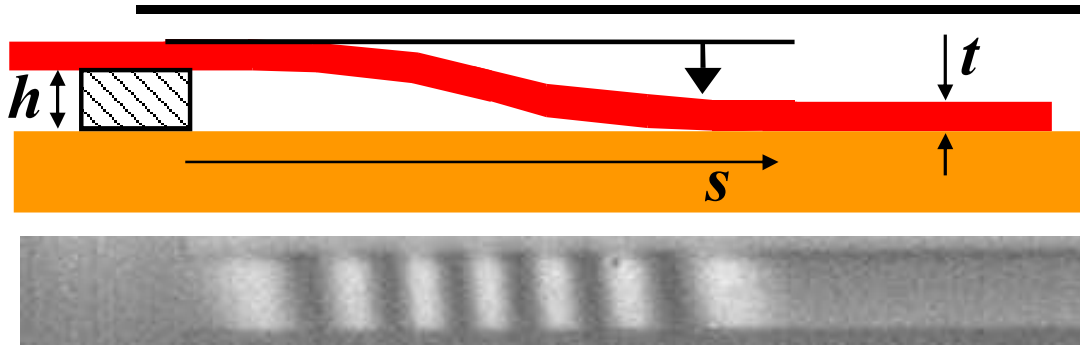
Initially free beam, but still in water



Drying leads to “stiction”



We can use cantilevers to quantify the adhesion, Γ

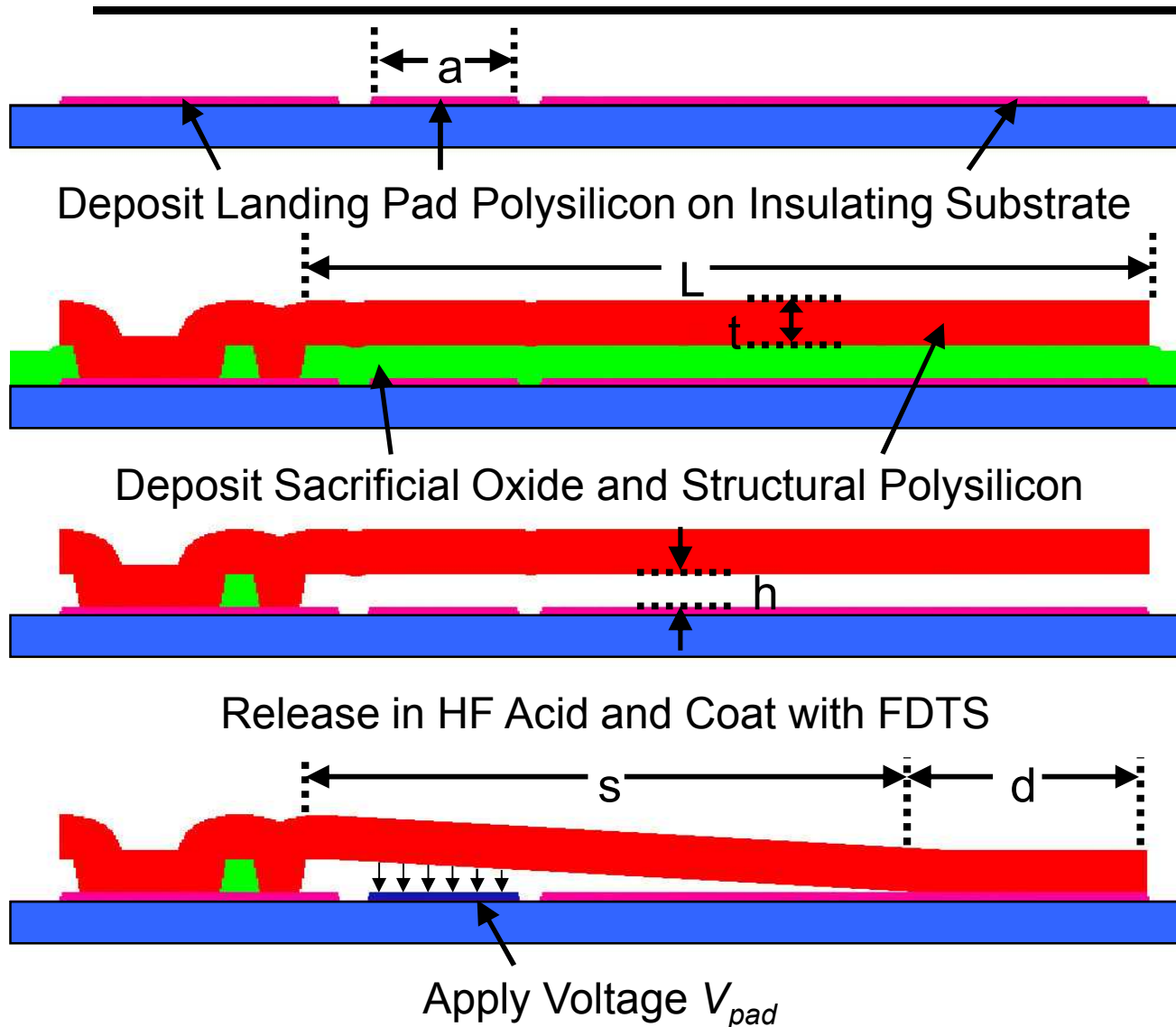


Capillary adhesion can be avoided by critical point drying or by applying monolayer coatings

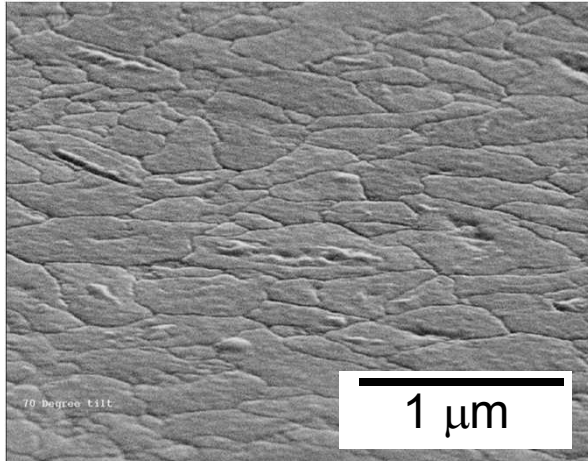
$$G = -\frac{dU_E}{wds} = \frac{3}{2}E \frac{h^2 t^3}{s^4} = \Gamma = 10 \frac{\text{mJ}}{\text{m}^2} \quad (\text{drying from water})$$

(de Boer and Michalske, Journal of Applied Physics, 1999)

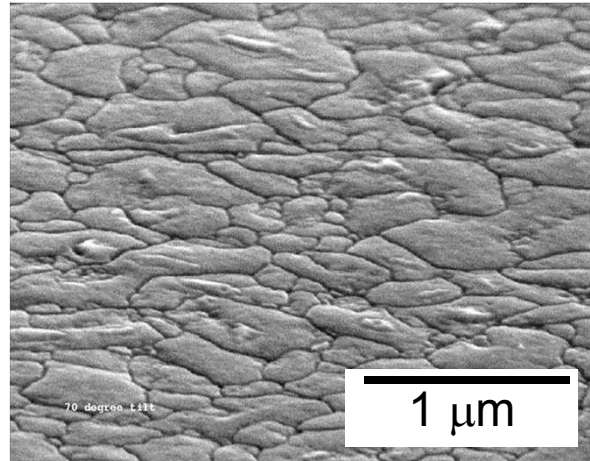
Microcantilever process and test flow



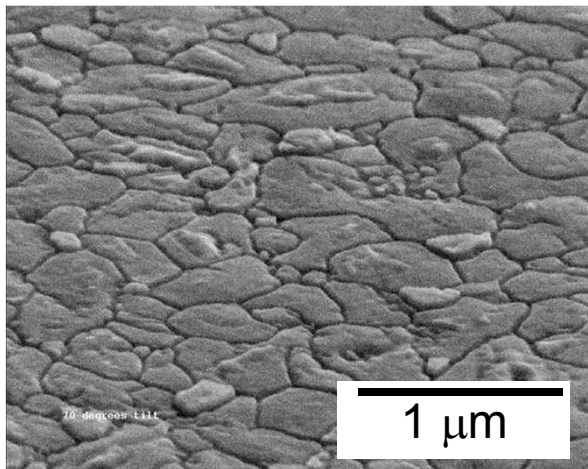
Oxidize the Poly 0 Surface to change surface roughness



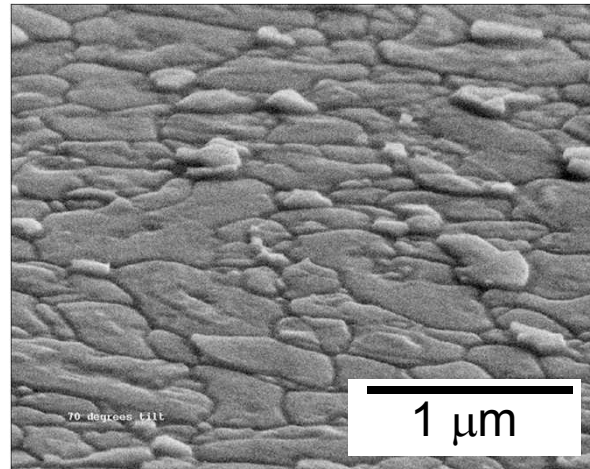
No oxidation, 2.6 nm rms



100 Å oxidation, 4.4 nm rms



300 Å oxidation, 5.6 nm rms

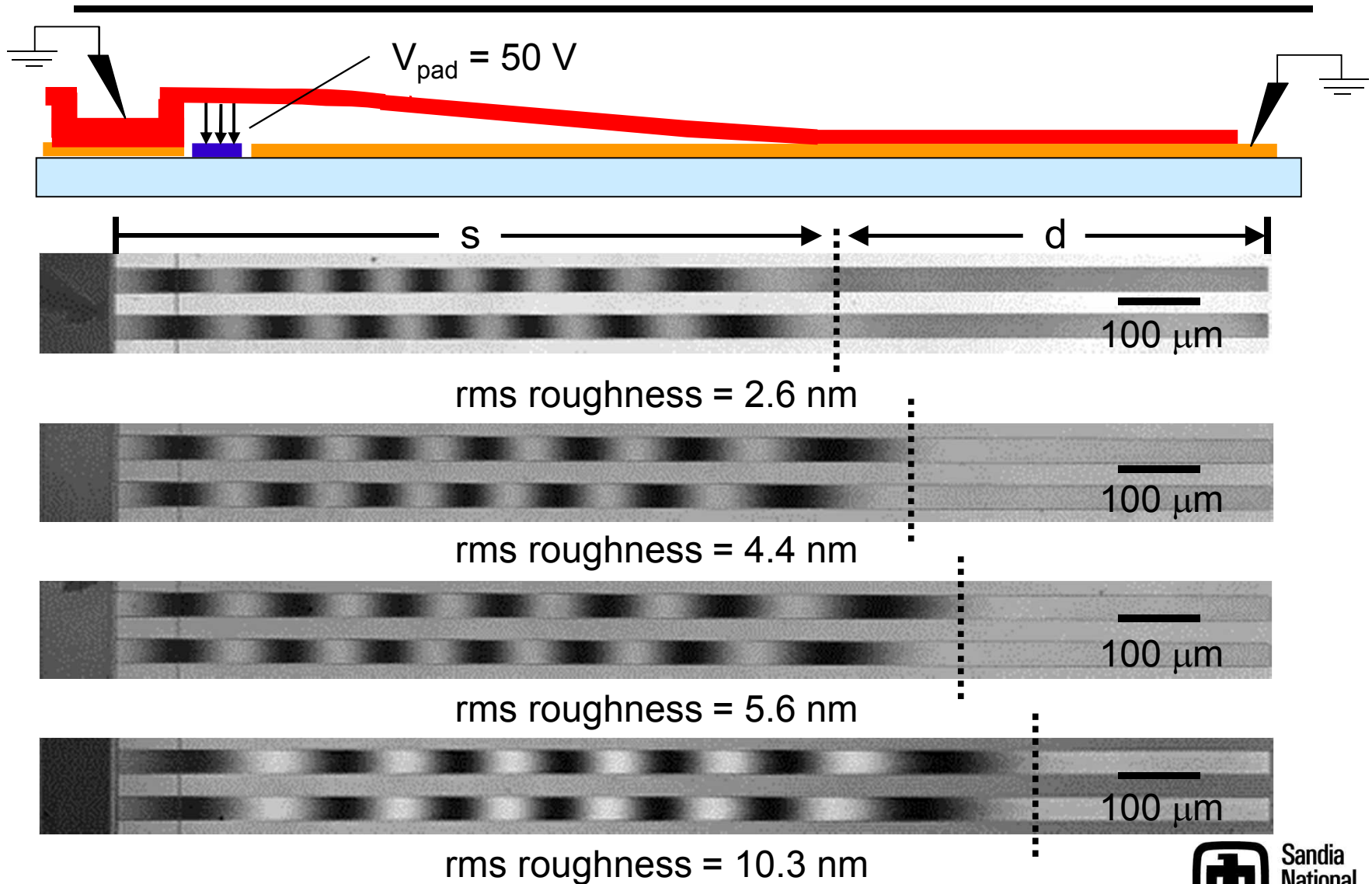


600 Å oxidation, 10.3 nm rms

Nanotexturing of the lower layer or polysilicon (P0) was accomplished via thermal oxidation in dry O₂ at 900° C for increasing times.

t (min)	tox (Å)	rms (nm)
0	--	2.6
20	100	4.4
136	300	5.6
400	600	10.3

Interferograms show qualitative relationship between surface roughness and crack length

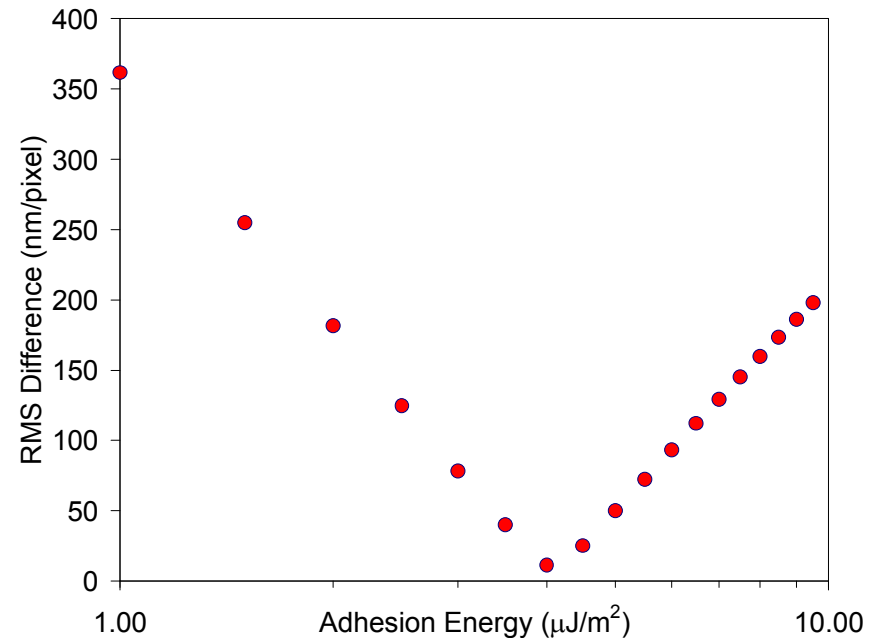
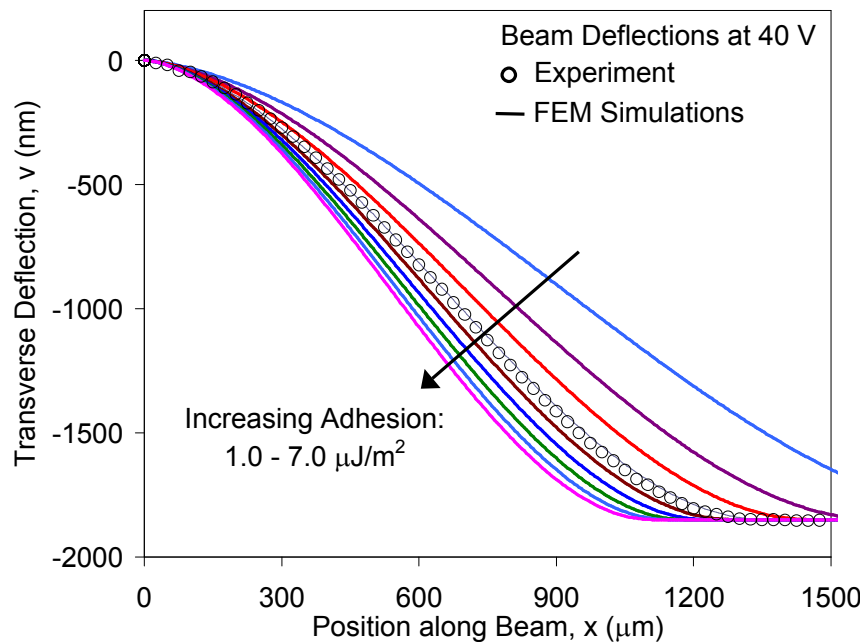


Adhesion measurement with applied voltage

Finite element analysis (ABAQUS) and user subroutines were used to find beam profiles with surface adhesion, electrostatic loading and initial stress gradient.

The only free parameter in the models is the adhesion Γ .

A least squares fit between the model and experiment was used to determine the value at each voltage.



(Knapp & de Boer, JMEMS, 2002)

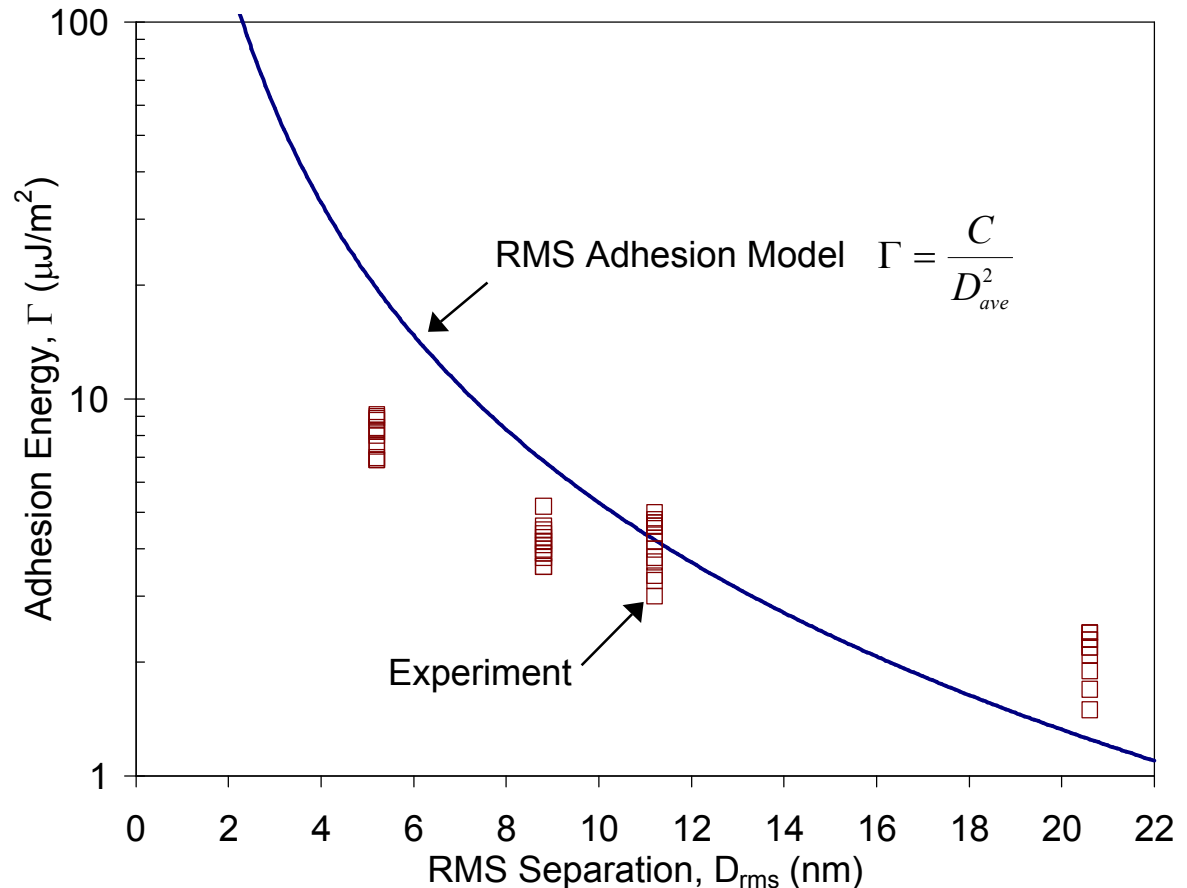
Experimental values of adhesion for each surface roughness

The measured values for adhesion loosely follow the approximation presented by Houston et al. (1996)

$$\Gamma = \frac{A}{12\pi D_{rms}^2}$$

These results raise the following questions:

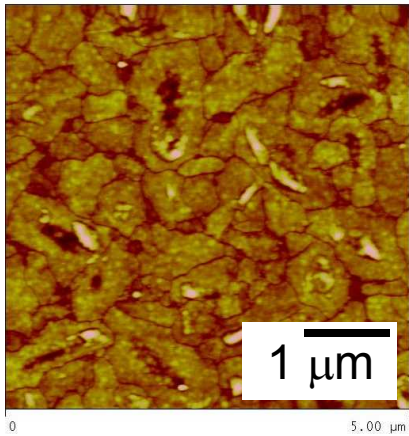
1. What is the best way to characterize the separation between the two surfaces?
2. Do we have another method to determine if these results are quantitatively correct?



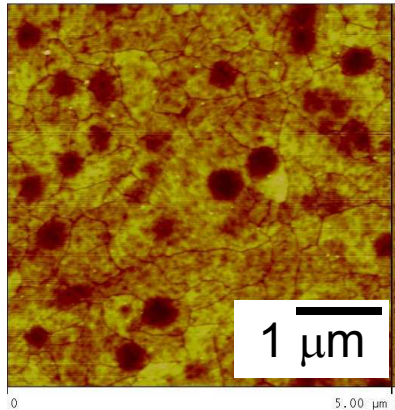
Atomic Force Microscopy Imaging with
Force Displacement Numerical Analysis

AFM topography data is analyzed using a numerical force-displacement routine

AFM Images

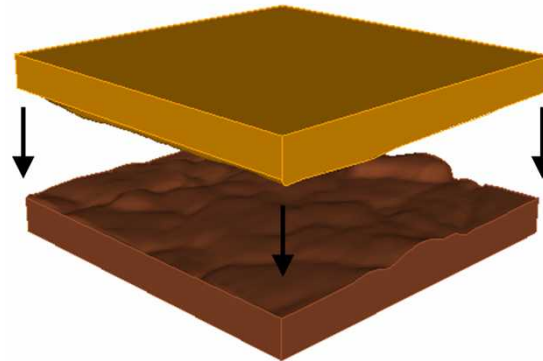


→
512 x 512
matrix with
surface
heights
entered into
force
displacement
routine
→



Numerical Force-Displacement Routine

1. Import AFM height data
2. Separate surfaces by initial displacement
3. Calculate separation for each pixel
4. Calculate force for each pixel
5. Find total force (sum)
6. Move surfaces towards each other
7. Repeat steps 3-6 to create attractive load-displacement curve



$$F_a = \frac{L_c^2}{N_{pixels}} \left[\sum_{all\ pixels} \frac{A g_f}{6\pi (d_{loc} + d_{co})^3} \right]$$

Anandarajah
and Chen 1995

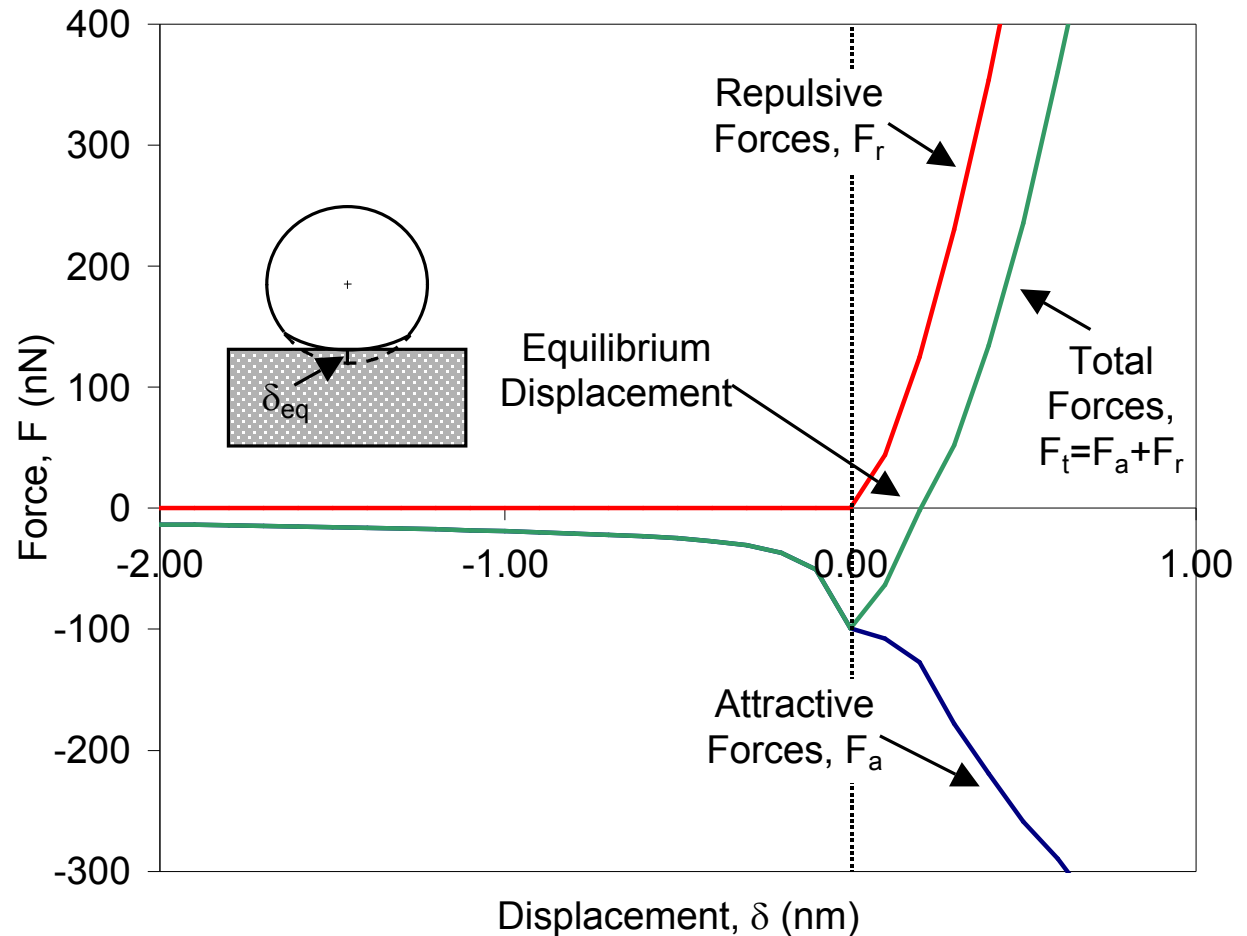
Calculate the total force-displacement curve using the AFM analysis and Hertzian mechanics

Attractive force-displacement curve based on AFM analysis

Repulsive force-displacement curve based on Hertzian mechanics

$$F_r = \frac{2}{3} \left(\frac{E}{1-\nu^2} \right) \sqrt{R\delta^3}$$

DMT Adhesion Model



Calculate adhesion energy by evaluating the area under the total force-displacement curve from the equilibrium displacement to infinity.

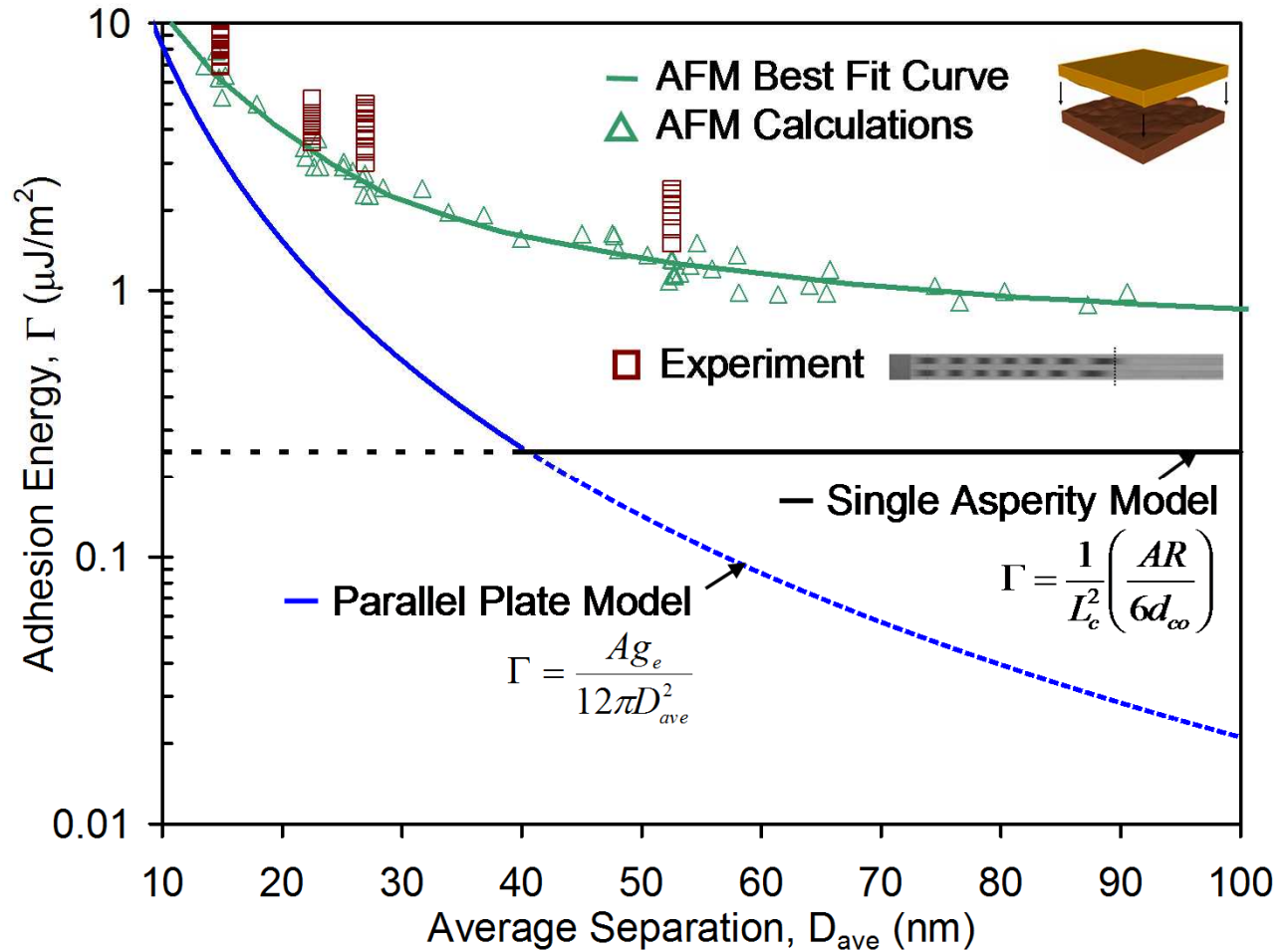
Predicted values of adhesion with AFM data

We placed the surfaces together in the following combinations for each roughness:

- Poly 0 and Poly 0
- Poly 0 and Poly 2

The average surface separation D_{ave} is calculated for each AFM pair according to

$$D_{ave} = \frac{1}{N_{pixels}} \left[\sum_{all\ pixels} d_{loc} \right]$$



Delrio, de Boer et al., Nat. Mat. (2005)

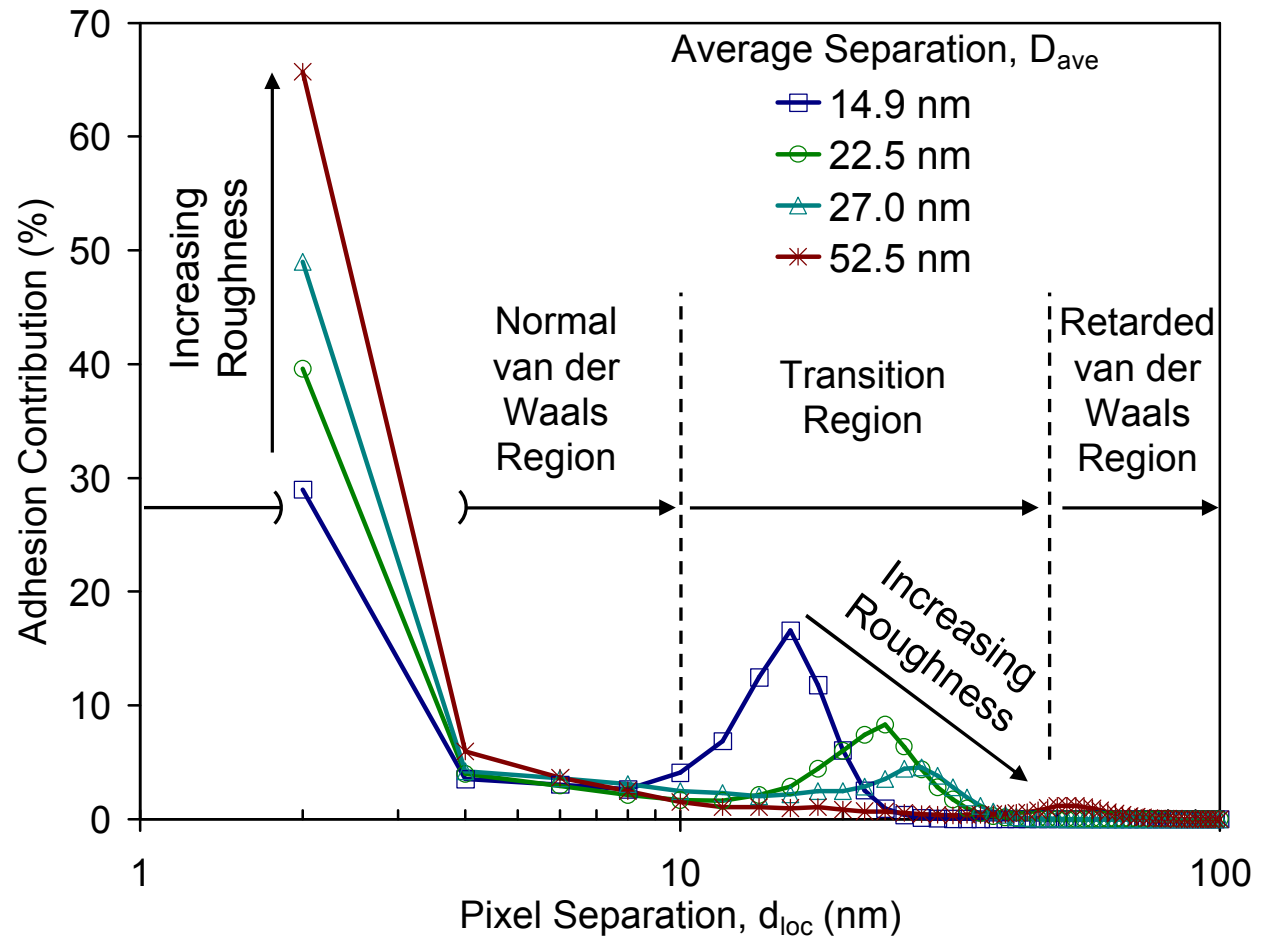
Histogram of adhesion contributions vs. pixel separation

Smoothest Surface

Adhesion contribution from both contacting asperities and non-contacting areas (combination of two extreme adhesion models).

Roughest Surface

Adhesion contribution mainly from contacting asperity (converging to Fuller-Tabor/Maugis model for single asperity).



Summary - dry adhesion in MEMS

Microcantilevers are used to measure adhesion in MEMS

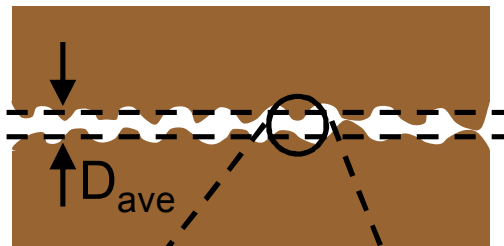
Adhesion is in the $\mu\text{J}/\text{m}^2$ range

For low surface roughness, adhesion dominated by retarded van der Waals forces

For higher surface roughnesses, adhesion dominated by non-retarded van der Waals forces

Two extreme models for adhesion

Smooth Surface



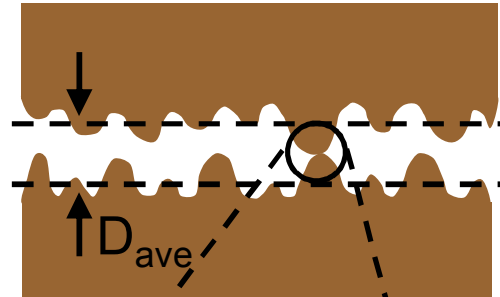
Parallel Plate Model

$$\Gamma = \frac{Ag_e}{12\pi D_{ave}^2}$$

Anandarajah and Chen 1995

The forces across non-contacting portions of the surfaces, whose area is far greater than the contacting area at the one asperity, will dominate the adhesion.

Rough Surface



Single Asperity Model

$$\Gamma = \frac{1}{L_c^2} \left(\frac{AR}{6d_{co}} \right)$$

Israelachvili 1992

A significant part of the area is too far apart to contribute to the adhesion; only the van der Waals forces near the single point of contact contribute.