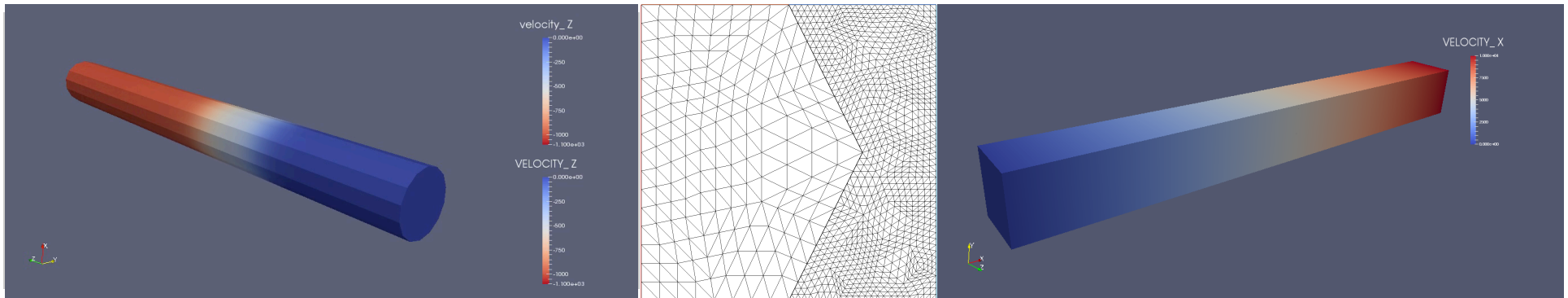


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# An explicit partitioned elastodynamics method based on Lagrange Multipliers

Pavel Bochev, Paul Kuberly and Kara Peterson



Sandia National Laboratories



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# First in a series of two talks focusing on:

## Problems with (physical or numerical) interfaces

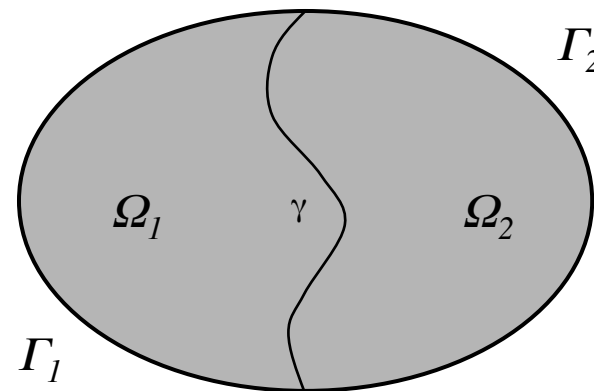
$$\ddot{u} - \nabla \cdot \sigma(u) = f \quad \text{in } \Omega \times T$$

$$u = 0 \quad \text{on } \Gamma \times T$$

$$u(x, 0) = u_0(x) \quad \text{in } \Omega \times T$$

$$\dot{u}(x, 0) = \dot{u}_0(x) \quad \text{in } \Omega \times T$$

$$\sigma(u) = \lambda(\nabla \cdot u)I + 2\mu \varepsilon(u)$$



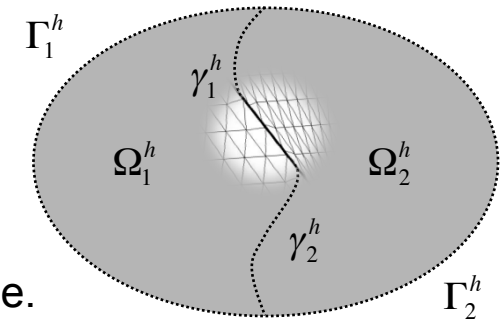
## Solutions satisfy the continuity/transmission conditions

$$u|_{\gamma^-} = u|_{\gamma^+} \quad \text{and} \quad \sigma(u^-) \cdot n_\gamma = \sigma(u^+) \cdot n_\gamma$$

# Today we will talk about

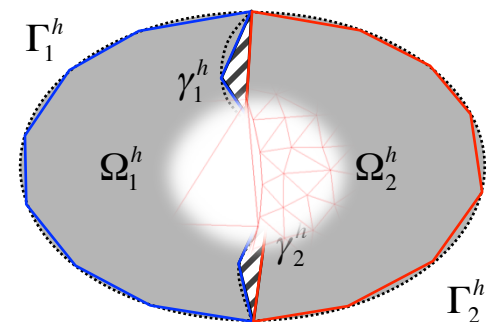
## A partitioned algorithm under “practical constraints”

- Interface is physical, e.g., material property.
- Mesh is **interface-fitted** but not necessarily matching.
- Each subdomain problem is solved independently by a different code.
- Information exchange between codes is limited to nodal masses and forces.
- Motivated by the FORTE coupling of Sandia’s Alegria and Sierra/SM codes.



## Thursday: An optimization-based, mesh-tying algorithm (P. Kuberry, 9:00am)

- Interface is physical or numerical, e.g., due to meshing
- **Separate meshing** creates 2 distinct, **non-coincident** versions of the same interface.
- Data transfer between non-coincident interfaces remains a tough challenge.
- Existing approaches typically involve complex mesh manipulations.



# Original “welded” interface coupling in Forte: Sandia National Laboratories

## “Welded” interface coupling

- Mass and forces **swapped** at nodes
- **Minimally intrusive** (black-box) coupling, but...
- Requires matching interface nodes
- One code may require **finer mesh**:  
⇒ Forces **excessive mesh refinement!**

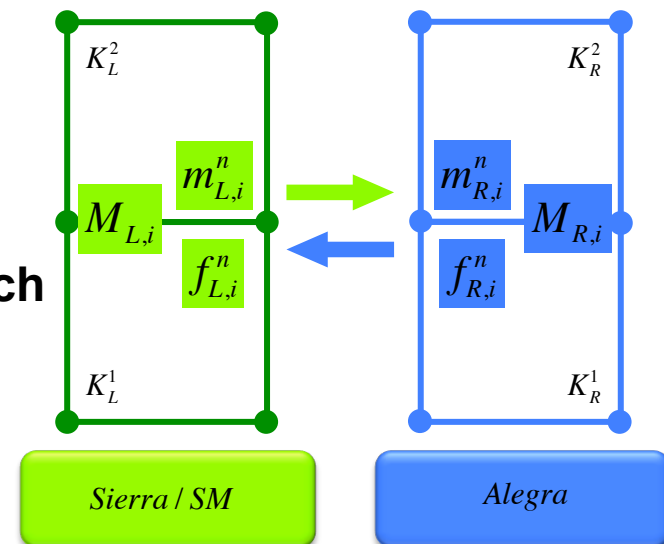
**Goal: develop a new partitioned algorithm, which**

- Defaults to “welded” interface on **matching** grids.
- Handles interfaces with **non-matching** grids.
- Has **linear** consistency
- Is **second-order** accurate

## Discrete subdomain equations

$$M_L u_L^{n+1} = \vec{m}_L^n + \vec{f}_L^n \quad M_R u_R^{n+1} = \vec{m}_R^n + \vec{f}_R^n$$

### Mass-force exchange



**Completed equation @ interface node:**

$$(M_L + M_R)u_{L/R}^{n+1} = (\vec{m}_L^n + \vec{m}_R^n) + (\vec{f}_L^n + \vec{f}_R^n)$$

# What does this swap mean mathematically?

## Let's reverse-engineer the “welded” interface coupling

$$(M_L + M_R)u_L^{n+1} = (\vec{m}_L^n + \vec{m}_R^n) + (\vec{f}_L^n + \vec{f}_R^n)$$

$$(M_L + M_R)u_R^{n+1} = (\vec{m}_L^n + \vec{m}_R^n) + (\vec{f}_L^n + \vec{f}_R^n)$$

$$u_L^{n+1} = u_R^{n+1}$$

This is a pair of identical “completed” equations at an interface node

Because they are the same, they imply continuity of the nodal displacement!

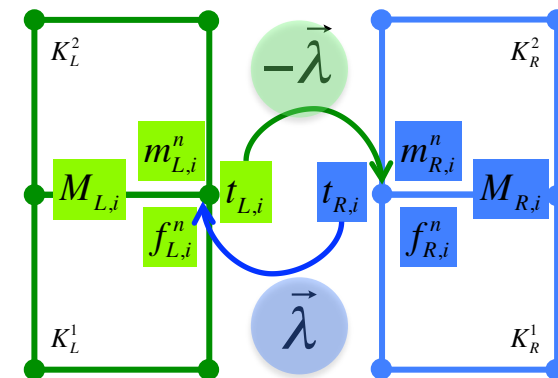
$$M_L u_L^{n+1} = (\vec{m}_L^n + \vec{f}_L^n) + (\vec{m}_R^n + \vec{f}_R^n - M_R u_L^{n+1})$$

$$M_R u_R^{n+1} = (\vec{m}_L^n + \vec{f}_L^n) + (\vec{m}_R^n + \vec{f}_R^n - M_L u_R^{n+1})$$

$$u_L^{n+1} = u_R^{n+1}$$

Let's group all terms from the “other” side

This is the contact force at the node!



It's beginning to look a lot like....

$$M_L u_L^{n+1} = (\vec{m}_L^n + \vec{f}_L^n) + \vec{\lambda}$$

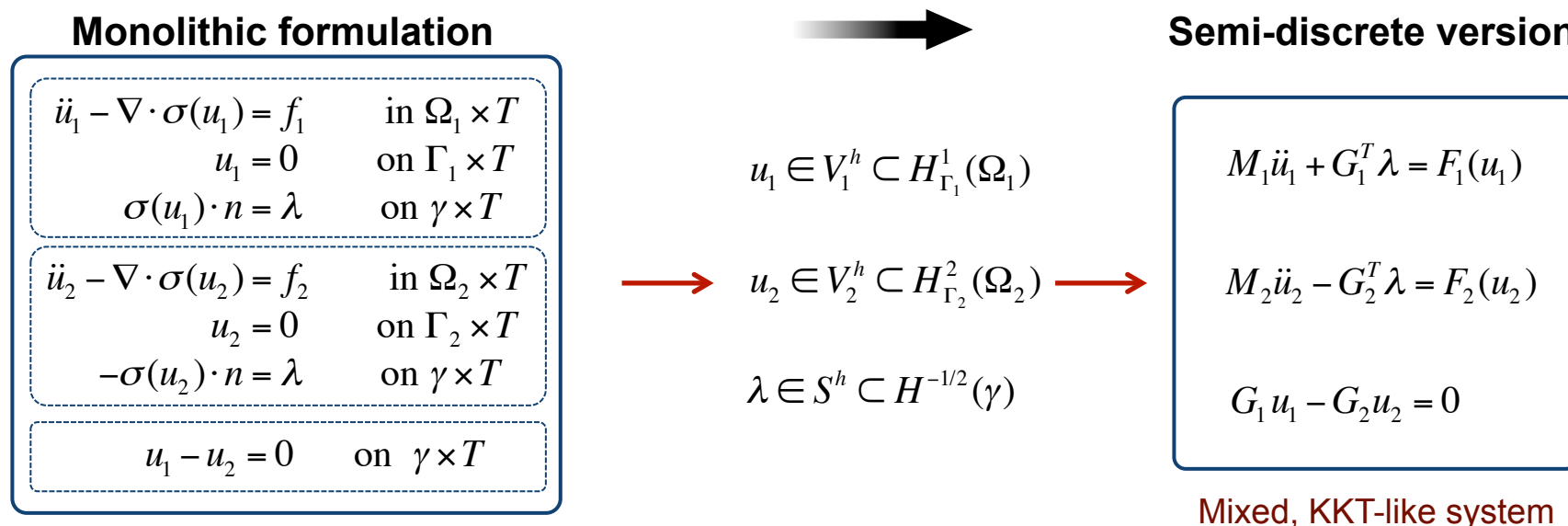
$$M_R u_R^{n+1} = (\vec{m}_L^n + \vec{f}_L^n) - \vec{\lambda}$$

$$u_L^{n+1} = u_R^{n+1}$$

**A mixed Lagrange multiplier formulation!**

# Let's start from a monolithic formulation

- Write the problem as a system of two subdomain equations with **mixed boundary conditions**.
- The **Neumann** boundary condition involves an **unknown contact force**  $\lambda$ .
- Close the system by adding the **displacement continuity condition**:



- System of **3 equations for 3 unknowns**: subdomain displacements and contact force.
- **Contact force continuity**  $\sigma(u_1) \cdot n_1 + \sigma(u_2) \cdot n_2 = 0$  on  $\gamma \times T$  **subsumed** in the equations.
- **Displacement continuity**  $u_1 - u_2 = 0$  on  $\gamma \times T$  **enforced explicitly**
- For problems with an energy principle  $t$  can be identified with a **Lagrange multiplier**.

# There's one problem though...

## Lagrange multipliers are not the most natural setting for partitioned schemes

- Result in Index-2 DAE that are more difficult to solve
- Not compatible with explicit time integration

$$M_1 \ddot{u}_1 + G_1^T \lambda = F_1(u_1)$$

$$M_2 \ddot{u}_2 - G_2^T \lambda = F_2(u_2)$$

$$G_1 u_1 - G_2 u_2 = 0$$

## Partial solution, *Carpenter et al, IJNME, 1991*

- References the multiplier one time increment ahead (forward increment LM method).
- Resulting method still not purely explicit

## A (simple) solution: **switch constraints from displacement to acceleration**

$$u_1 = u_2 \Big|_\gamma$$

$$\ddot{u}_1 = \ddot{u}_2 \Big|_\gamma$$

**index 2 DAE**

**index 1 DAE**

implies the original  
constraint under  
suitable assumptions

$$\begin{bmatrix} M_{1,\sigma} & 0 & G_1^T & 0 & 0 \\ 0 & M_{2,\sigma} & -G_2^T & 0 & 0 \\ G_1 & -G_2 & 0 & 0 & 0 \\ \hline 0 & 0 & 0 & M_{1,0} & 0 \\ 0 & 0 & 0 & 0 & M_{2,0} \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \\ \ddot{u}_{1,0} \\ \ddot{u}_{2,0} \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \\ f_{1,0} \\ f_{2,0} \end{bmatrix}$$

- Caution: works for **transmission problems** but may not work for **contact problems**!

# The master formulation on matching grids

The master monolithic (mixed) problem

$$\begin{bmatrix} M_{1,\sigma} & 0 & G_1^T & 0 & 0 \\ 0 & M_{2,\sigma} & -G_2^T & 0 & 0 \\ G_1 & -G_2 & 0 & 0 & 0 \\ \hline 0 & 0 & 0 & M_{1,0} & 0 \\ 0 & 0 & 0 & 0 & M_{2,0} \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \\ \ddot{u}_{1,0} \\ \ddot{u}_{2,0} \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \\ f_{1,0} \\ f_{2,0} \end{bmatrix}$$

Option A: (1,1) Schur

eliminate  
interface  
DOF

$$\begin{bmatrix} M_{1,\sigma} & 0 & \tilde{M}_{1,\sigma} \\ 0 & M_{2,\sigma} & -\tilde{M}_{2,\sigma} \\ \hline \tilde{M}_{1,\sigma} & -\tilde{M}_{2,\sigma} & 0 \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \end{bmatrix}$$

VFR

Option B: (2,2) Schur

eliminate  
internal  
DOF

$$\begin{bmatrix} M_{1,\sigma} & 0 & \tilde{M}_{1,\sigma} \\ 0 & M_{2,\sigma} & -\tilde{M}_{2,\sigma} \\ \hline \tilde{M}_{1,\sigma} & -\tilde{M}_{2,\sigma} & 0 \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \end{bmatrix}$$

$$\tilde{M}_{*,\sigma}(M_{1,\sigma}^{-1} + M_{2,\sigma}^{-1})\tilde{M}_{*,\sigma}\lambda = \tilde{M}_{*,\sigma}(M_{2,\sigma}^{-1}f_{2,\sigma} - M_{1,\sigma}^{-1}f_{1,\sigma})$$

MFR

$$\begin{aligned} (M_{1,\sigma}^i + M_{2,\sigma}^i)\ddot{u}_{1,\sigma}^i &= f_{1,\sigma}^i + f_{2,\sigma}^i \\ (M_{1,\sigma}^i + M_{2,\sigma}^i)\ddot{u}_{2,\sigma}^i &= f_{1,\sigma}^i + f_{2,\sigma}^i \end{aligned}$$

Identical Partitioned systems for matching nodes!



# The master formulation on non-matching grids

$$\begin{bmatrix} M_{1,\sigma} & 0 & G_1^T & 0 & 0 \\ 0 & M_{2,\sigma} & -G_2^T & 0 & 0 \\ G_1 & -G_2 & 0 & 0 & 0 \\ \hline 0 & 0 & 0 & M_{1,0} & 0 \\ 0 & 0 & 0 & 0 & M_{2,0} \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \\ \dot{u}_{1,0} \\ \dot{u}_{2,0} \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \\ f_{1,0} \\ f_{2,0} \end{bmatrix}$$

## Option A (VFR)

$$\begin{bmatrix} M_{1,\sigma} & 0 & G_1^T \\ 0 & M_{2,\sigma} & -\tilde{M}_{2,\sigma} \\ G_1 & -\tilde{M}_{2,\sigma} & 0 \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} M_{2,\sigma} & 0 & -G_2^T \\ 0 & M_{1,\sigma} & \tilde{M}_{1,\sigma} \\ G_2 & -\tilde{M}_{1,\sigma} & 0 \end{bmatrix} \begin{bmatrix} \ddot{u}_{2,\sigma} \\ \ddot{u}_{1,\sigma} \\ \lambda \end{bmatrix} = \begin{bmatrix} f_{2,\sigma} \\ f_{1,\sigma} \\ 0 \end{bmatrix}$$

- Requires 2 separate master systems for each side
- LM collocated with displacement on the opposite side
- LM spaces simple to construct
- Results in a generalized “mass-force exchange” between the subdomains

## Option B (MFR)

$$\begin{bmatrix} M_{1,\sigma} & 0 & G_1^T \\ 0 & M_{2,\sigma} & -G_2^T \\ G_1 & -G_2 & 0 \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \end{bmatrix}$$

$$(G_1 M_{1,\sigma}^{-1} G_1^T + G_2 M_{2,\sigma}^{-1} G_2^T) \lambda = G_2 M_{2,\sigma}^{-1} f_{2,\sigma} - G_1 M_{1,\sigma}^{-1} f_{1,\sigma}$$

- Leads to a system for interface DOFs
- Requires preconditioning
- Index 1 enables explicit treatment of the LM
- Requires common mesh refinement for LM
- Potentially more accurate...
- Related to **Dual Schur Complement** systems in DD, FETI, hybrid methods,...

# The two options in a nutshell

Options A and B are simply two different ways of expressing the contact force:

(A): **Indirect** representation: Generate



$$\begin{array}{c} \text{Red box } M \end{array} + \begin{array}{c} \text{Blue box } \Delta M \end{array} = \begin{array}{c} \text{Red box } f \end{array} + \begin{array}{c} \text{Blue box } \Delta f \end{array}$$

This is a **primal Schur** complement approach in which we express the contact force indirectly in terms of mass/force updates to the interface eqs.

(B) **Direct** representation: Generate



$$\begin{array}{c} \text{Red box } M \end{array} = \begin{array}{c} \text{Red box } f \end{array} + \begin{array}{c} \text{Green box } \Delta f \end{array}$$

This is a **dual Schur** complement approach in which we solve an equation for the LM to obtain the contact force on the interface

Most approaches to partitioned algorithms are direct

# Option A gives the desired generalization of FORTE

Consider the system for subdomain 1 (LM collocated with interface on subdomain 2)

$$\begin{bmatrix} \boxed{M_{1,\sigma}} & 0 & \tilde{M}_{1,\sigma} \overleftarrow{\Pi} & 0 & 0 \\ 0 & M_{2,\sigma} & -\tilde{M}_2 & 0 & 0 \\ \tilde{M}_{2,\sigma} \overrightarrow{\Pi} & -\tilde{M}_{2,\sigma} & 0 & 0 & 0 \\ \hline 0 & 0 & 0 & M_{1,0} & 0 \\ 0 & 0 & 0 & 0 & M_{2,0} \end{bmatrix} \begin{bmatrix} \ddot{u}_{1,\sigma} \\ \ddot{u}_{2,\sigma} \\ \lambda \\ \ddot{u}_{1,0} \\ \ddot{u}_{2,0} \end{bmatrix} = \begin{bmatrix} f_{1,\sigma} \\ f_{2,\sigma} \\ 0 \\ f_{1,0} \\ f_{2,0} \end{bmatrix}$$

Taking the (1,1) Schur complement gives a system for  $u_1$

$$\rightarrow (M_{1,\sigma} + \boxed{\tilde{M}_{1,\sigma} \overleftarrow{P} \tilde{M}_{2,\sigma}^{-1} M_{2,\sigma} \overrightarrow{P}}) \ddot{u}_{1,\sigma} = f_{1,\sigma} + \boxed{\tilde{M}_{1,\sigma} \overleftarrow{P} \tilde{M}_{2,\sigma}^{-1} f_{2,\sigma}}$$

and analogously (using a second mixed problem) for  $u_2$

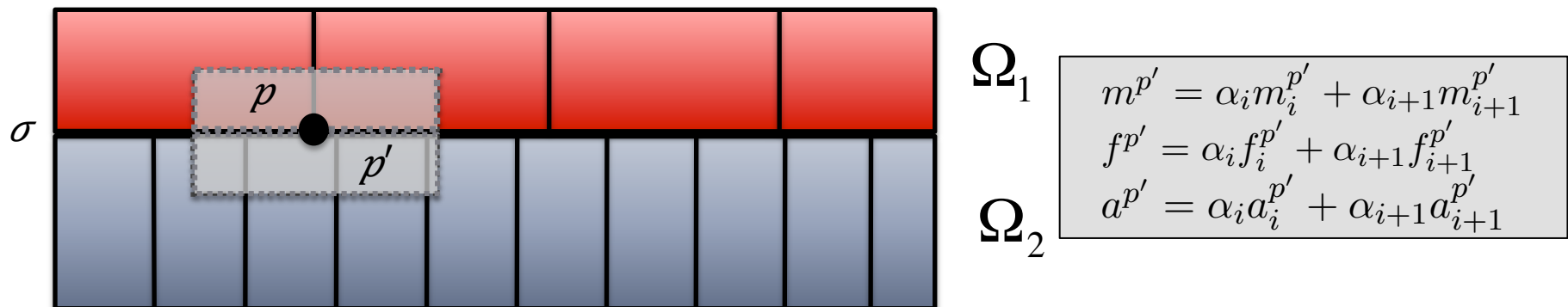
$$(\boxed{M_{2,\sigma} + \tilde{M}_{2,\sigma} \overrightarrow{P} \tilde{M}_{1,\sigma}^{-1} M_{1,\sigma} \overleftarrow{P}}) \ddot{u}_{2,\sigma} = f_{2,\sigma} + \boxed{\tilde{M}_{2,\sigma} \overrightarrow{P} \tilde{M}_{1,\sigma}^{-1} f_{1,\sigma}}$$

generalization of  
Forte's mass-  
force exchange

mass  $\leftarrow$  exchange  $\Rightarrow$  force

# Comparison with a Slide Lines method

- We examine connections between Option A, i.e., generalized Forte coupling and slide lines for tied contact applications, Kuchařík et al. , *Comp. & Fluids*, 83 2013.



- The slide line method is derived by considering virtual cells straddling the interface and then writing out contact conditions at  $p=p'$ :
  - Continuity of accelerations & continuity of the contact force
  - This leads to the following auxiliary system:

$$\begin{aligned} m^p \ddot{u}^p &= f^p \\ m^{p'} \frac{a^p}{a^{p'}} \ddot{u}^{p'} &= f^{p'} \frac{a^p}{a^{p'}} \end{aligned} \quad \longrightarrow \quad (m^p + m^{p'} \frac{a^p}{a^{p'}}) \ddot{u}^p = f^p + f^{p'} \frac{a^p}{a^{p'}}$$

The slide line equation at interface node

# Comparison with a Slide Lines method

## Slide lines (Kuchařík et al.)

$$\left( m^p + \frac{(\alpha_i m_i^{p'} + \alpha_{i+1} m_{i+1}^{p'}) a^p}{\alpha_i a_i^{p'} + \alpha_{i+1} a_{i+1}^{p'}} \right) \ddot{u}^p = f^p + \frac{(\alpha_i f_i^{p'} + \alpha_{i+1} f_{i+1}^{p'}) a^p}{\alpha_i a_i^{p'} + \alpha_{i+1} a_{i+1}^{p'}}$$

Interpolate then scale vs. scale then interpolate

$$\left( m^p + \left( \alpha_i \frac{m_i^{p'}}{a_i^{p'}} + \alpha_{i+1} \frac{m_{i+1}^{p'}}{a_{i+1}^{p'}} \right) a^p \right) \ddot{u}^p = f^p + \left( \alpha_i \frac{f_i^{p'}}{a_i^{p'}} + \alpha_{i+1} \frac{f_{i+1}^{p'}}{a_{i+1}^{p'}} \right) a^p$$

## Generalized Forte

Thanks to M. Shashkov (LANL) for pointing out Slide Lines reference

# Properties

## Equivalence to a monolithic explicit solution for matching grid interfaces

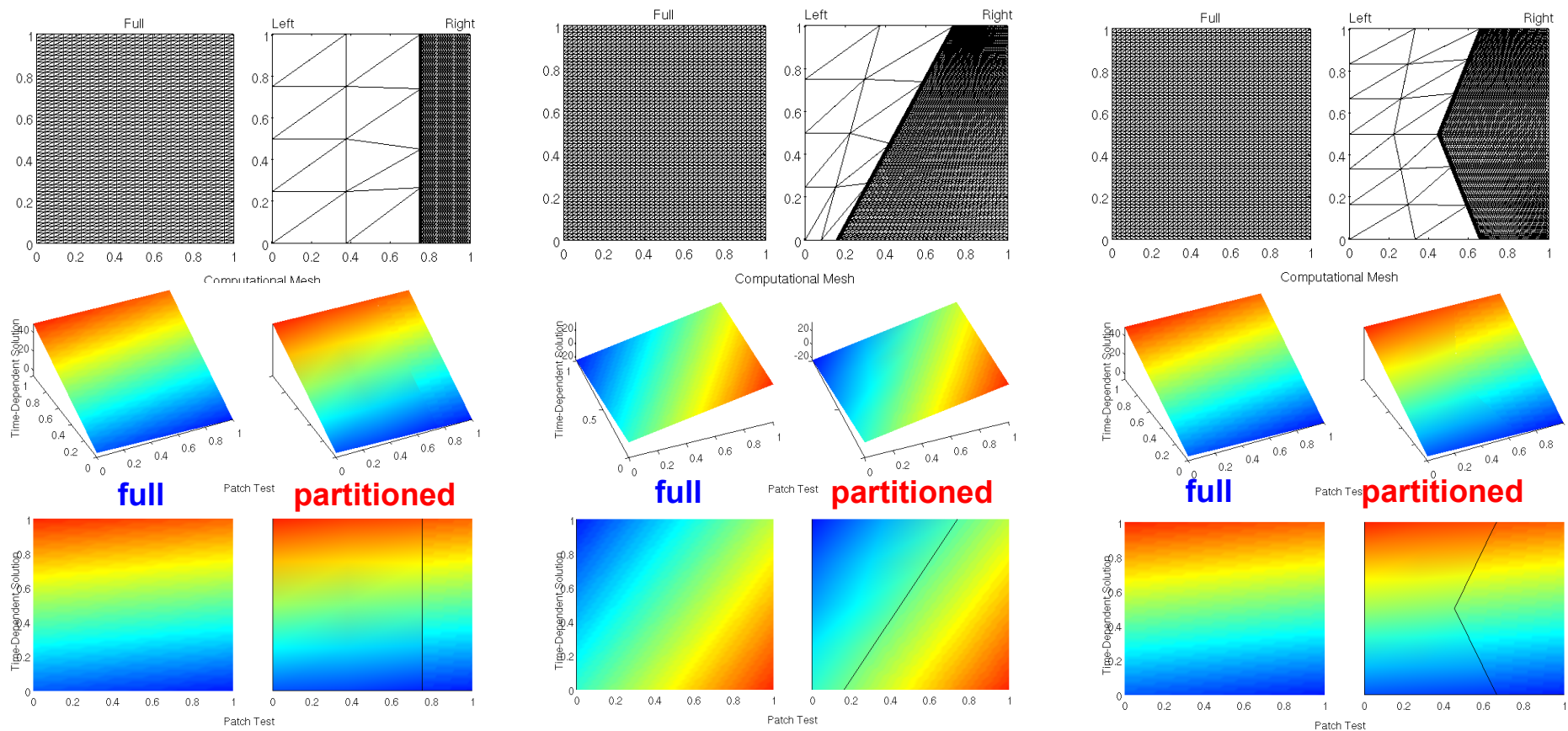
For interfaces with matching nodes there holds  $M_{1\sigma} = M_{2\sigma}$  and  $\tilde{P} = \vec{P} = I$

$$\begin{aligned} (M_1 + M_{1\sigma} \tilde{P} M_{2\sigma}^{-1} M_2 \tilde{P}) \ddot{u}_1 &= F_1 + M_{1\sigma} \tilde{P} M_{2\sigma}^{-1} F_2 & (M_2 + M_{2\sigma} \vec{P} M_{1\sigma}^{-1} M_1 \vec{P}) \ddot{u}_2 &= F_2 + M_{2\sigma} \vec{P} M_{1\sigma}^{-1} F_1 \\ \downarrow & & \downarrow & \\ (M_1 + M_2) \ddot{u}_1 &= F_1 + F_2 & (M_2 + M_1) \ddot{u}_2 &= F_2 + F_1 \end{aligned}$$

Interface	Vertical	Slanted
Mesh $\Omega_1$	24x20	24x20
Mesh $\Omega_2$	24x20	24x20
$L_2$ error $\Omega_1$	3.38E-17	9.43E-17
$L_2$ error $\Omega_2$	1.07E-15	1.05E-15
$L_2$ error $\Omega_1$	2.49E-15	7.74E-15
$L_2$ error $\Omega_2$	9.92E-14	1.23E-13

# Properties

## Recovery of linear displacements (patch test)

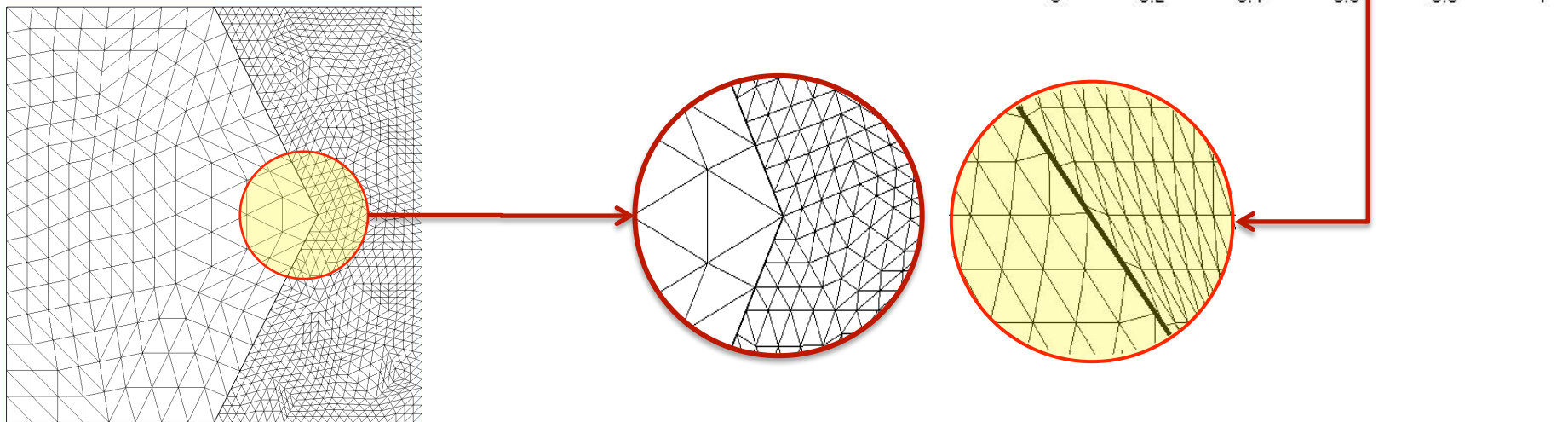


**VFR approach recovers linear solution to machine precision on interfaces with non-matching grids.**

# Properties

## 2<sup>nd</sup> order accuracy on non-matching interfaces

	Error		Rate
Mesh $\Omega_1$	28x40	56x80	
Mesh $\Omega_2$	52x40	104x80	
$L_2$ error $\Omega_1$	2.07E-03	5.15E-04	2.01
$L_2$ error $\Omega_2$	3.79E-03	9.58E-04	1.99
$H^1$ error $\Omega_1$	2.78E-01	1.39E-01	1.00
$H^1$ error $\Omega_2$	8.22E-01	4.12E-01	1.00



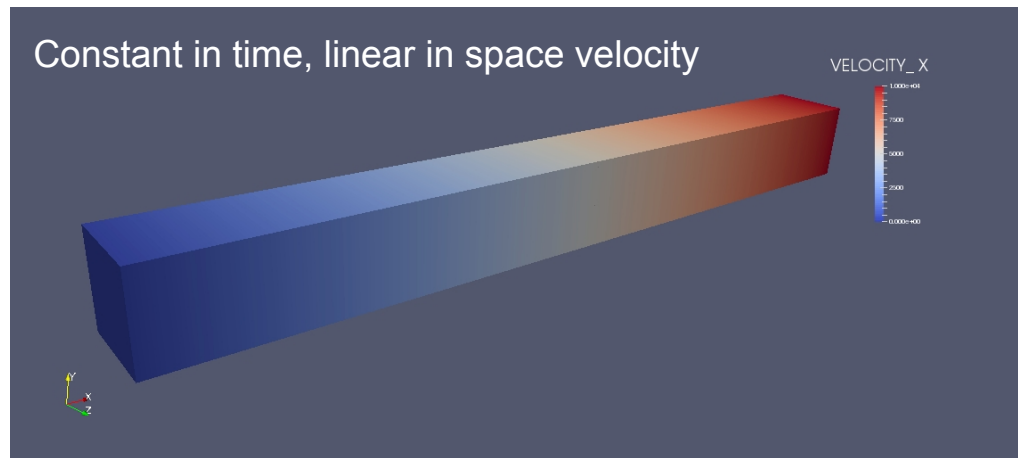


# Implementation in production codes

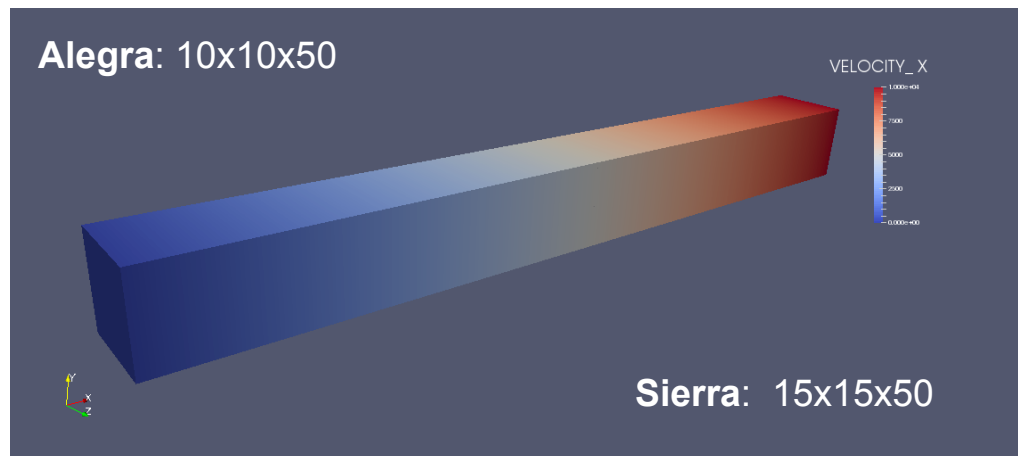
**VFR coupling has been deployed in Sandia's Forte software.**

## Consistency test

Exact

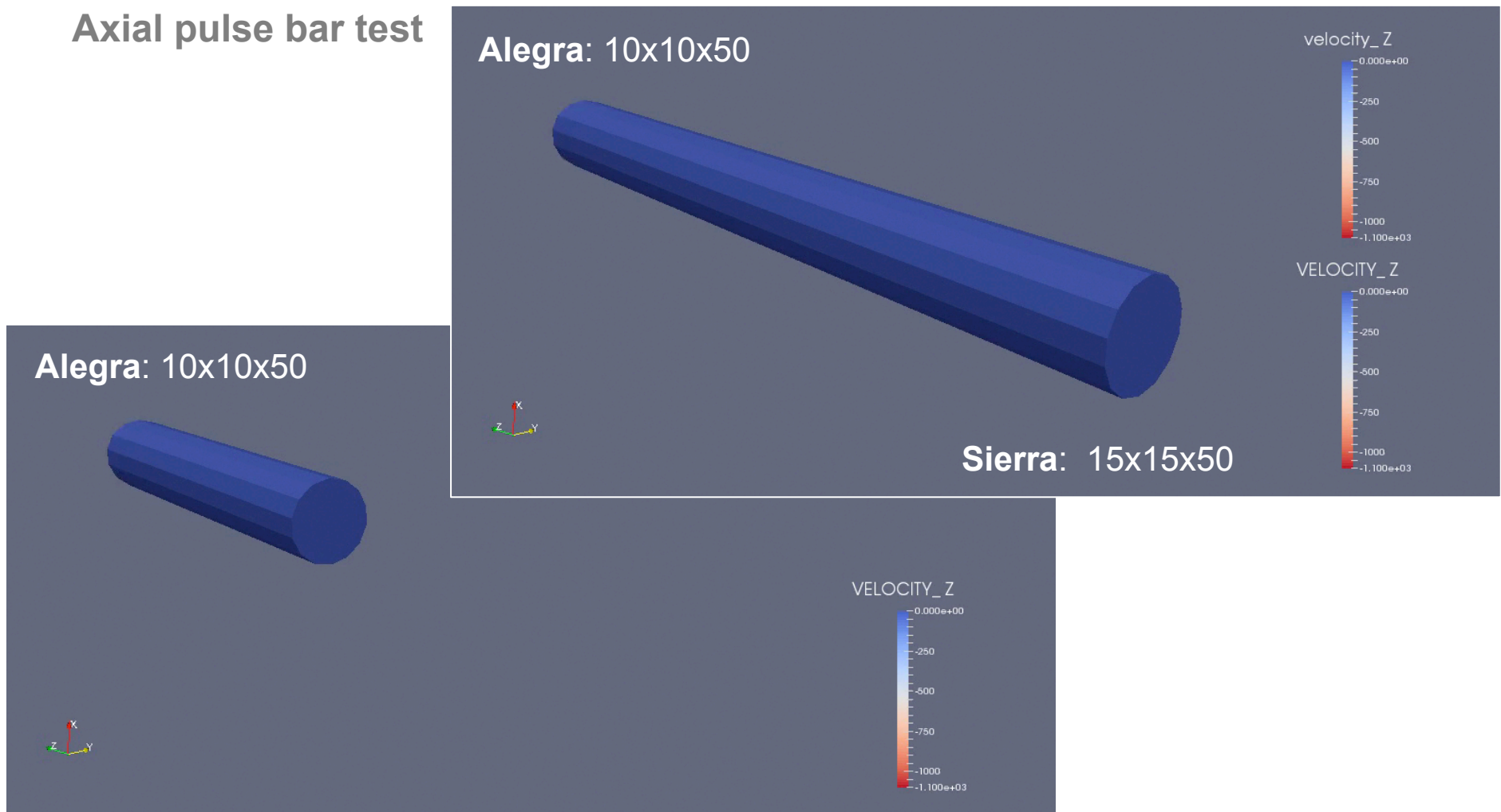


Forte+VFR



# Verification of Forte+VFR

## Axial pulse bar test



# Conclusions

- **Developed a general framework** that reduces a DAE 2 to a DAE 1 and is based on Lagrange multipliers.
- **Framework provides** a way to generate partitioned approaches that can be traced back to a well-posed system including both direct and indirect contact force representation methods.
- **Operator simplification** allows for a diagonal mass update to the indirect approach, avoiding complicated linear solves in an explicit approach.
- **Equivalence to a monolithic solution (proved)** in the case of matching nodes, **passing a patch test (proved)**, and **second order convergence** rates observed in numerical experiments.