

Chapter 11: Application of UQ and V&V to Experiments and Simulations of Heated Pipes Pressurized to Failure

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Vicente Romero, Frank Dempsey, Bonnie Antoun
Sandia National Laboratories*
Albuquerque, NM

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**MSS Special Session on Release of JANNAF Volume:
*Simulation Credibility—Advances in V&V and UQ***

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Elements of this VVUQ Case Study



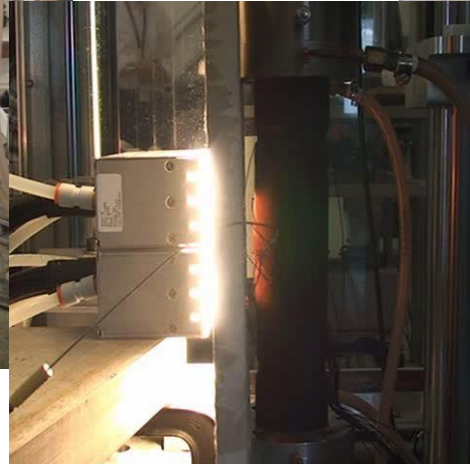
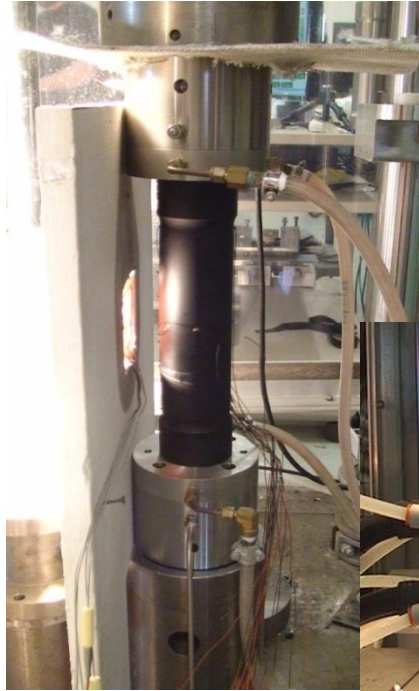
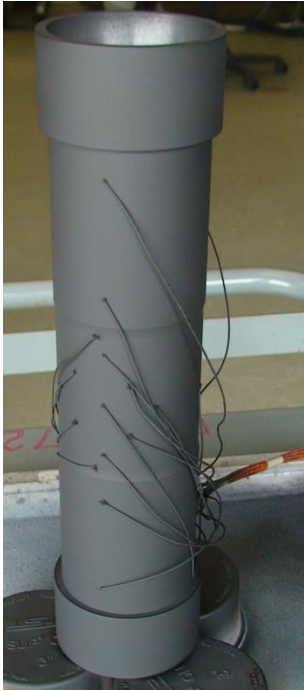
- **Material-strength experimental characterization and associated constitutive-model calibration**
- **Model-assisted design and analysis of validation experiments to characterize and reduce uncertainty**
- **Mesh and solver discretization studies (simulation verification) to control and characterize solution error and uncertainty**
- **Model validation comparison of experimental and simulation results with uncertainties and interpretation of the results**

Challenging/Advanced VVUQ Aspects of this Application Problem

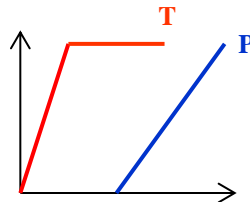
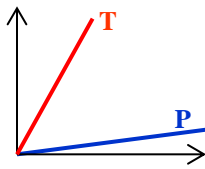
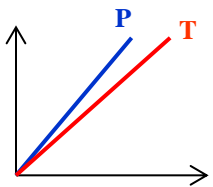


- **Material variability is characterized by discrete random functions** (temperature dependent stress-strain curves)
- **Very limited (sparse) experimental data**
 - at material testing/characterization level
 - at pipe-test validation level
- **Boundary condition reconstruction uncertainties from spatially sparse sensor data**
- **Normalization of pipe experimental responses for:**
 - measured input-condition differences among multiple replicate tests
 - random and systematic uncertainties in measurements and processing of experimental inputs & outputs
- **Very high computational cost → very limited # of model runs for UQ, and significant discretization errors/uncertainties to manage, quantify, and account for**

“Pipe Bomb” Validation Experiments



Ramp temperature and pressure independently to failure



Material Characterization:

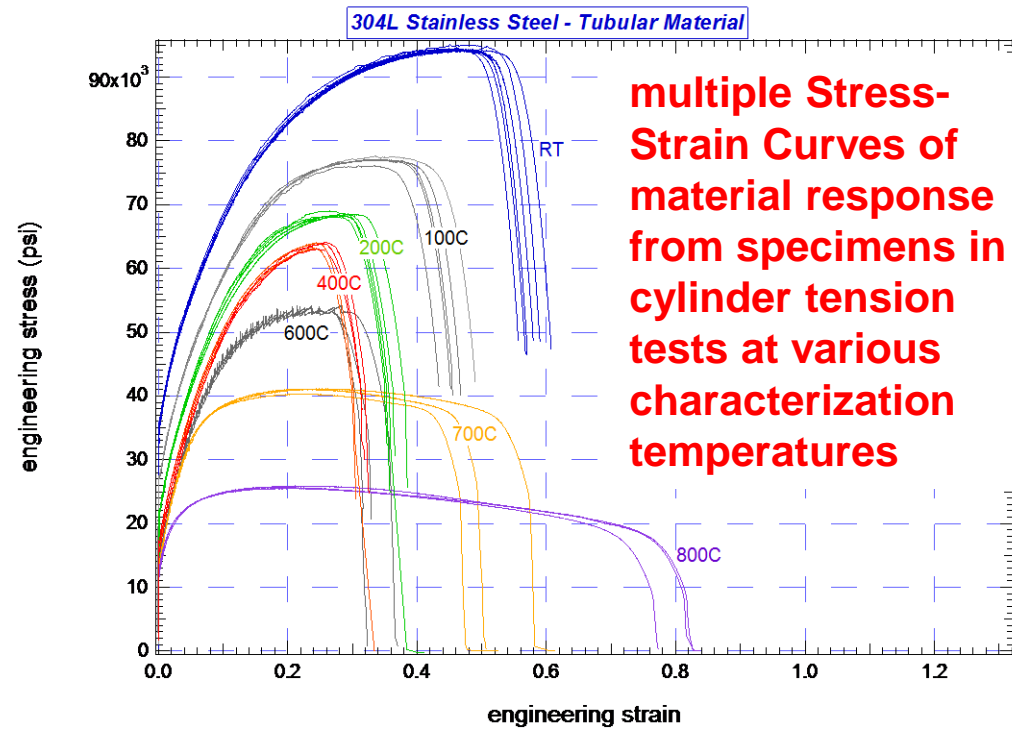
Aleatory and Epistemic uncertainties from
Sparse samples of Discrete Random Functions



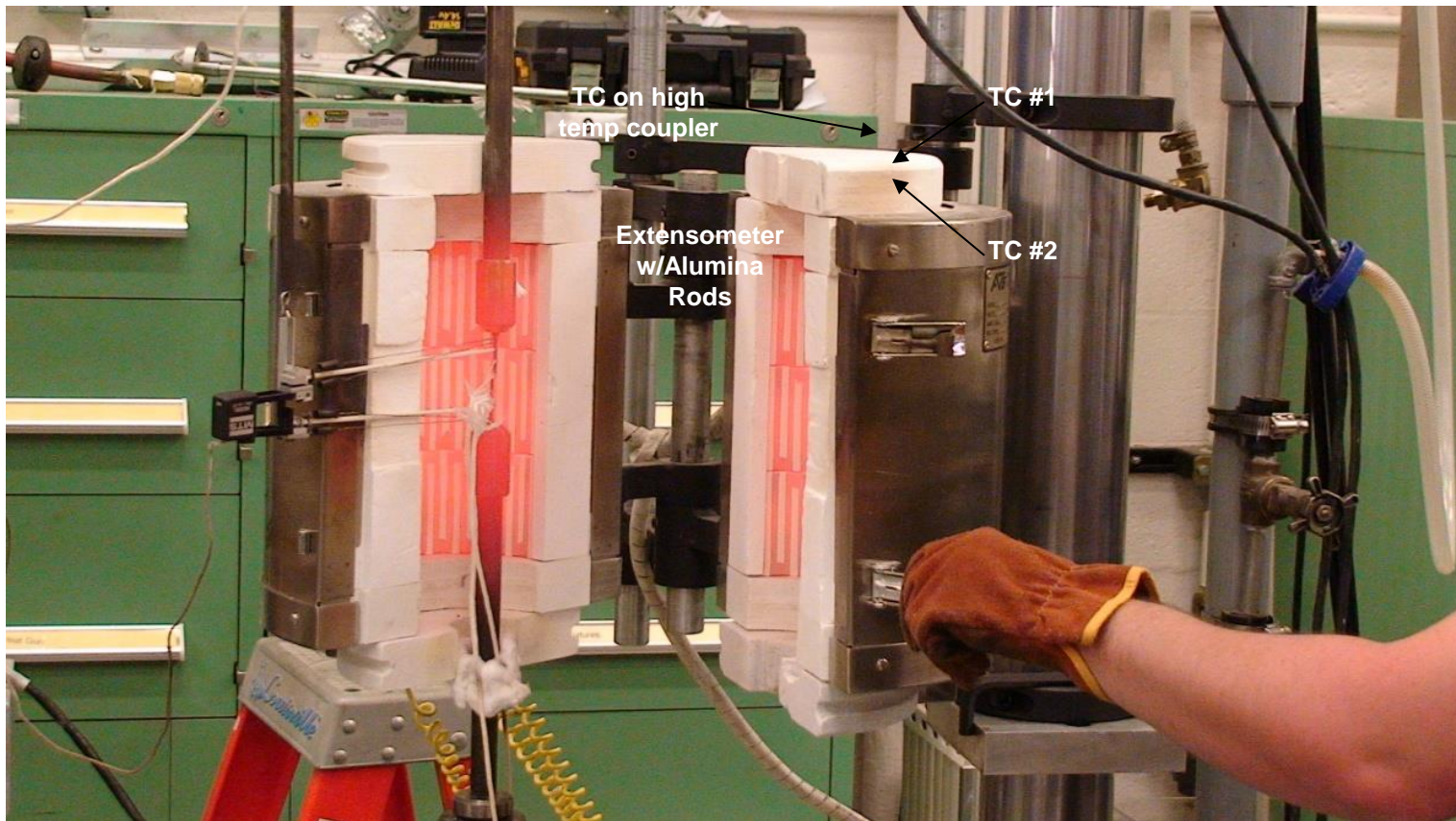
- **Example: multiple Stress-Strain Curves of material variability in calibration of constitutive model**
- **Similar issues in e.g. electronics modeling** – calibration to experimental response curves of electronics yields discrete parameter sets considered non-interpolable in-between.



cylinder
Tension-test
specimens

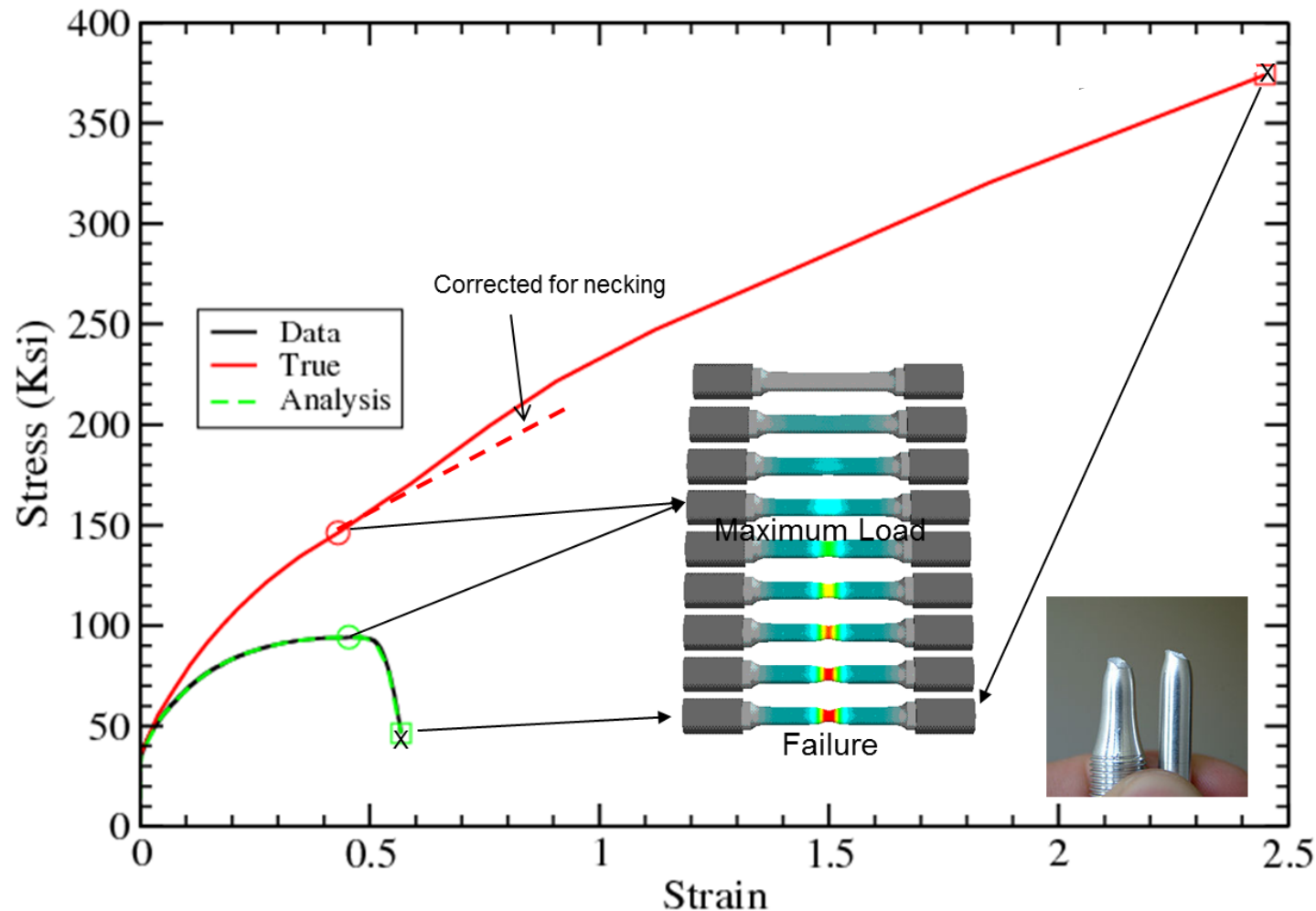


Cylinder Material Specimen Tension Test at 800C



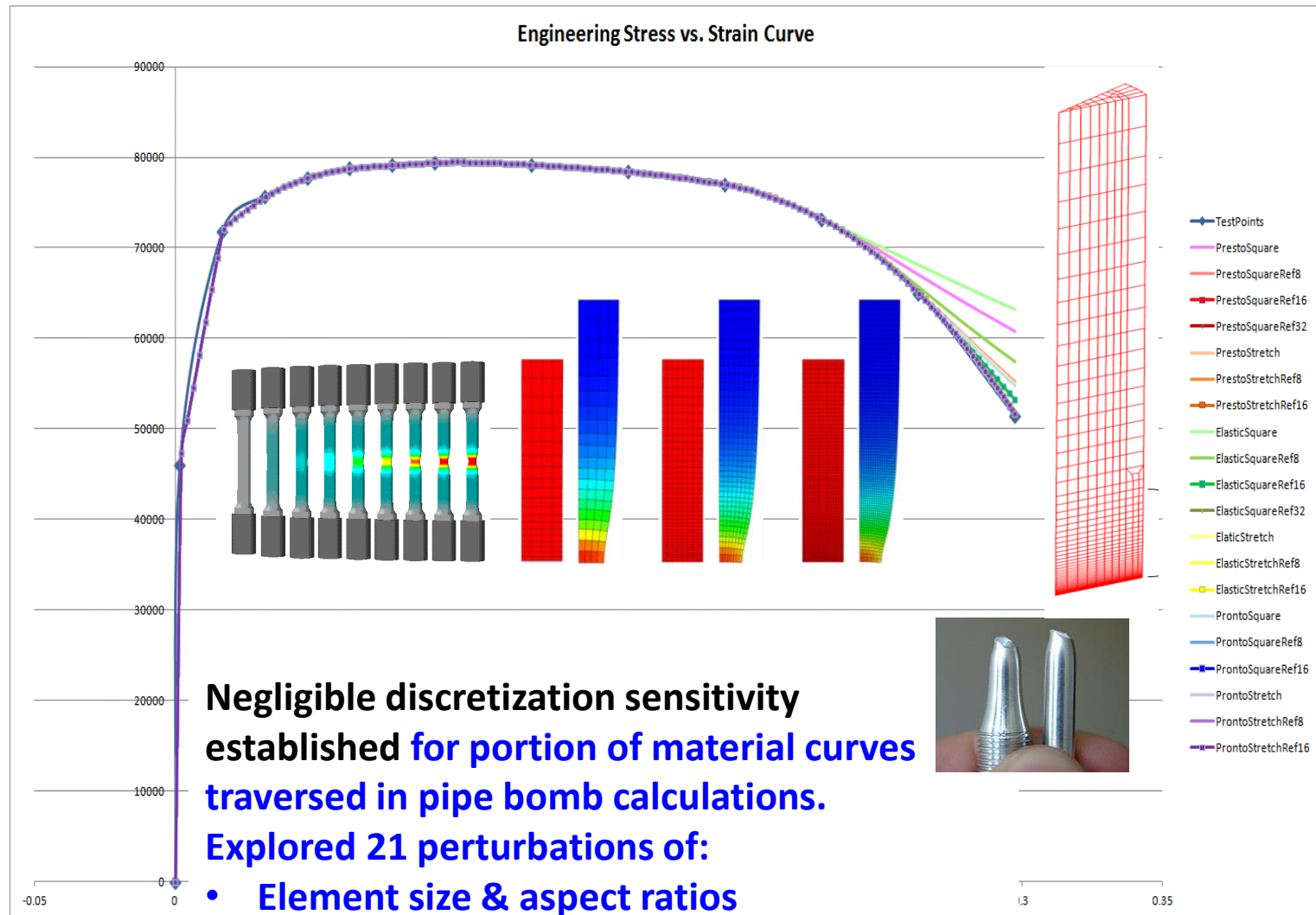
Inversion Procedure to extract Cauchy-Stress/Logarithmic-Strain from Experimental Stress-Strain Curves

Quasi-Static Thermal-Elastic-Plastic Stainless Steel Constitutive Model



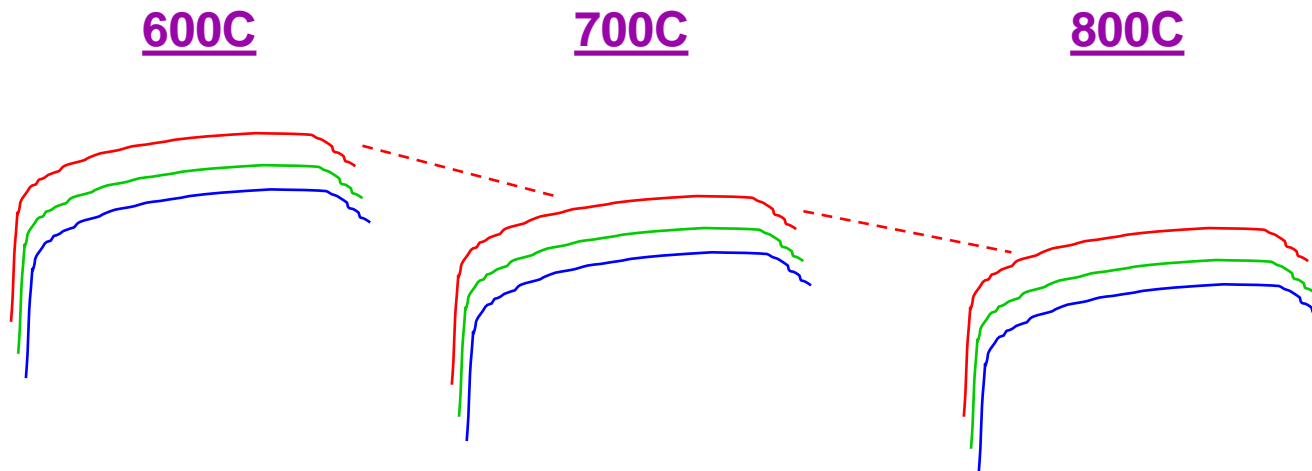
Mesh and Solver Effects

in modeled necking/failure in material characterization tests



Correlation of Like-Ranked S-S Curves across Characterization Temperatures

- Assume material strength (as indicated by model-predicted failure pressure) is strongly correlated across temperatures
- Enables interpolation to other temperatures for spatial and/or temporal variations of material temperatures



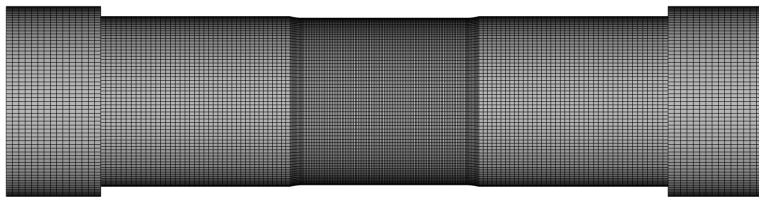
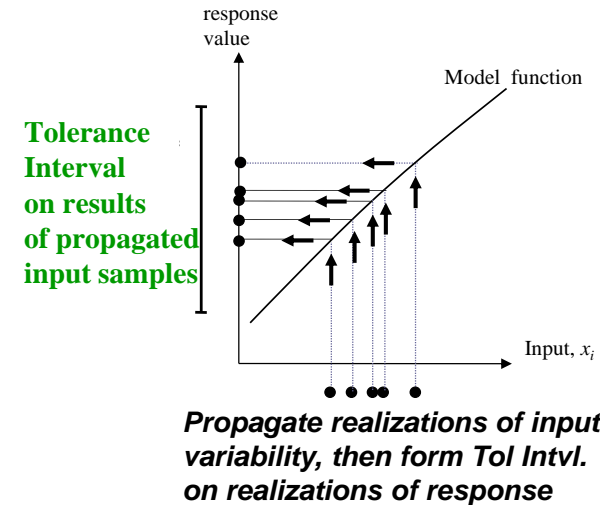
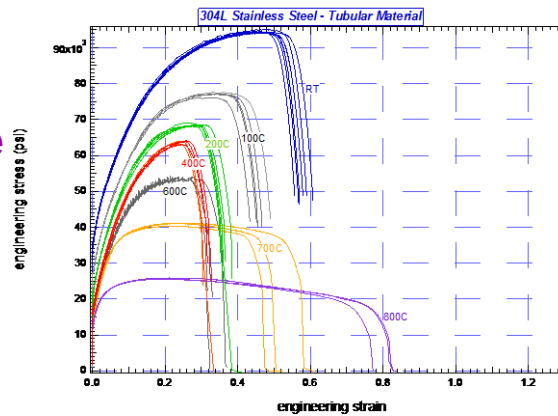
Red curves = high strength (**HS**) σ - ϵ curve **set** over temperatures

Green curves = medium strength (**MS**) **set** over temperatures

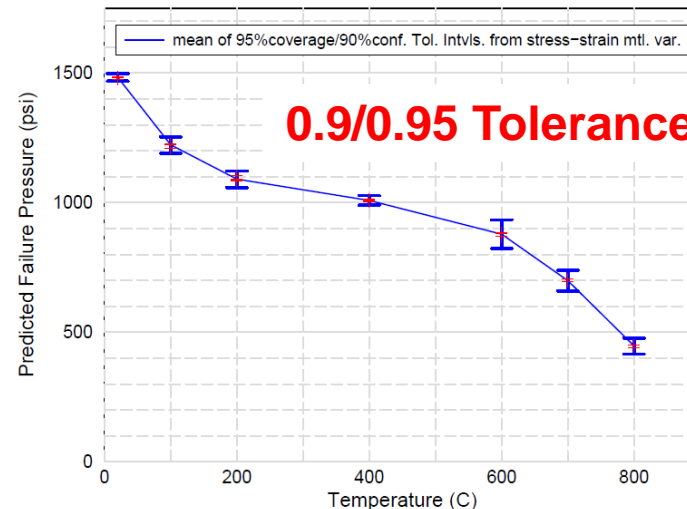
Blue curves = low strength (**LS**) **set** over temperatures

Predicted Variability of Failure Pressures due to Variability of Material Stress-Strain Curves

- Model is run for the individual Stress-Strain Curves at a test temperature
- 0.9/0.95 Tolerance Intervals are formed on the set of calculated failure pressures at that temperature

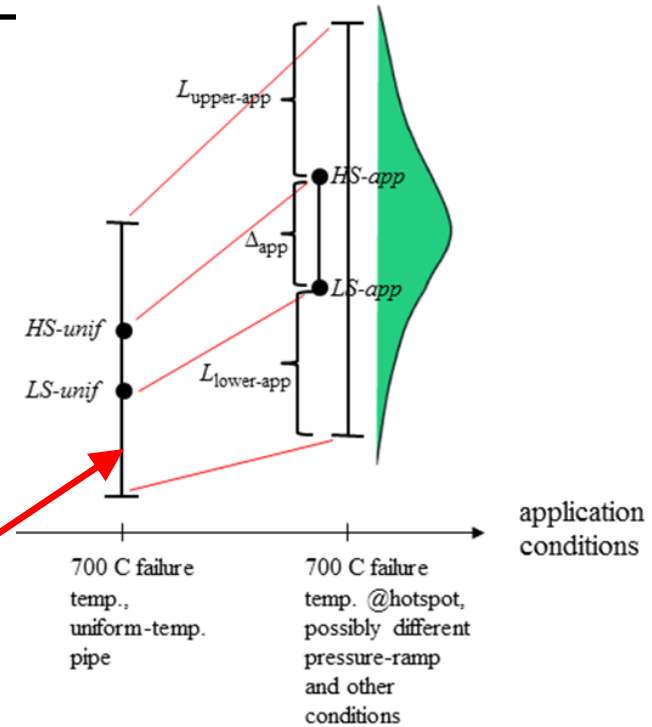
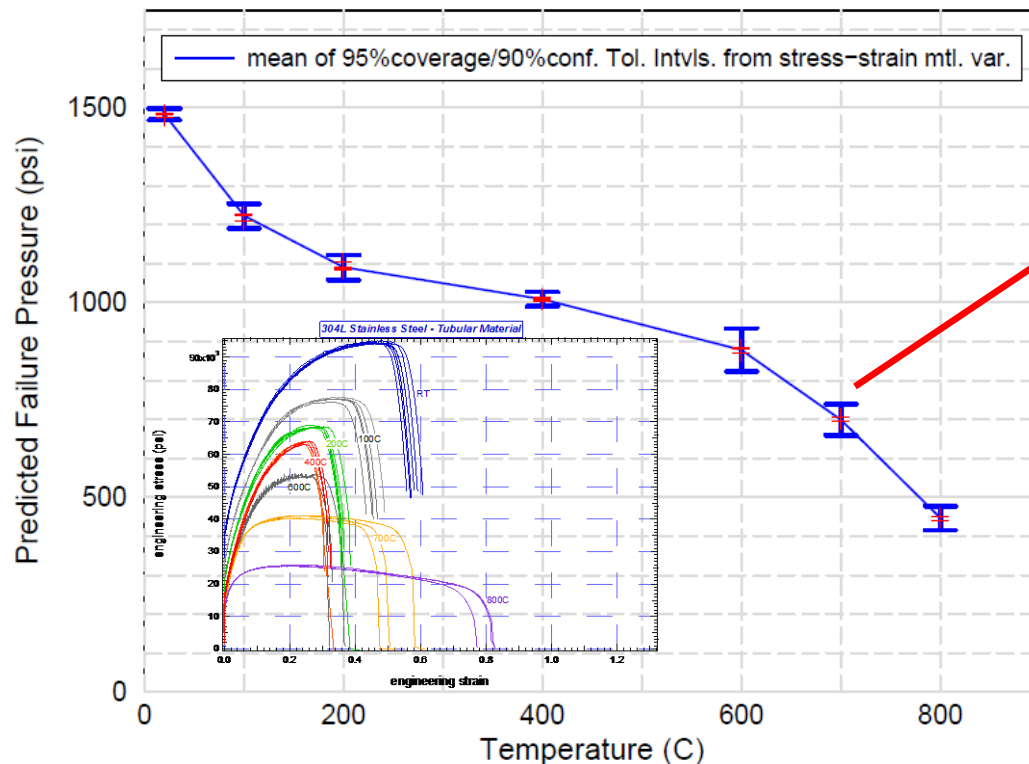


Pressure ramped uniform-temperature
HEX finite element model of
quasi-static solid mechanics



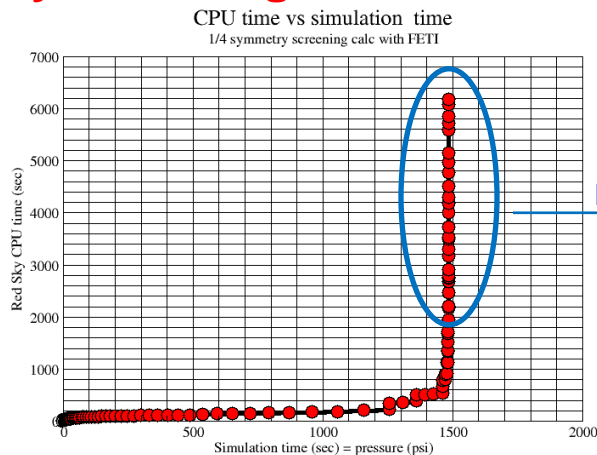
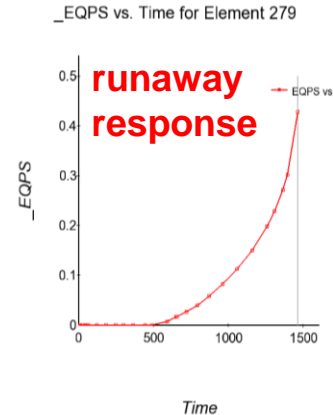
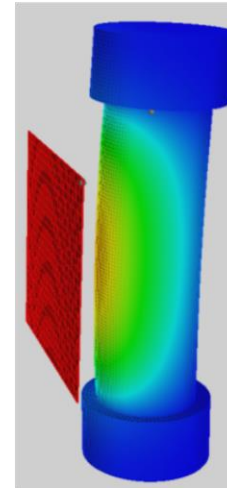
Economical Parameterization of TIs by **High** & **Low** Strength Material Curves

- Tolerance Intervals are constructed from multiple stress-strain curves
- But TIs can be parameterized by 2 s-s curves for only 2 Val./UQ sims. w/ full-geom. model

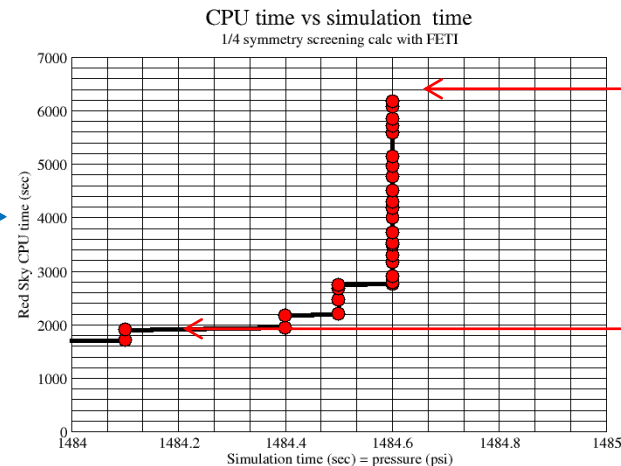


Simulation Difficulty: *creep up to a physical instability point*

- Pipe wall failure is indicated when the quasi-static calculations reach a physical instability point
 - when the internal pressure exceeds the material's resisting force no static equilibrium is attainable and no inertia terms to stabilize the calculation through breakup
- large sensitivity to mesh and solver settings
- excessive run times
- highly distorting elements



Magnified



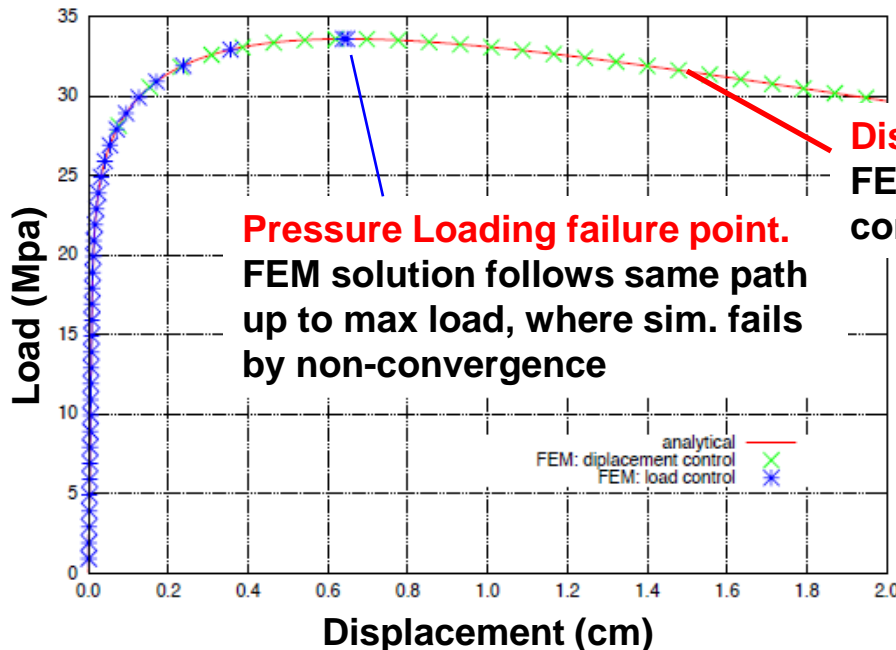
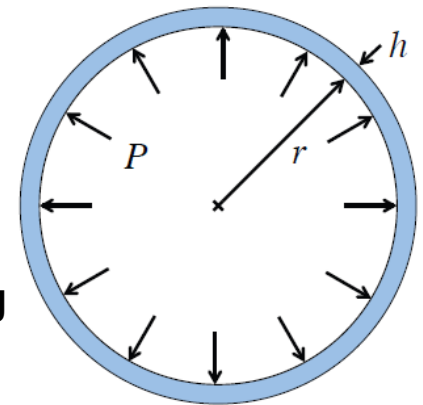
weeks

days

Calculation Instability signifying Structural Failure is Confirmed by 2D Test Problem with Analytic Solution



- Test Problem w/same Code & Constit. Model
 - Ring internally loaded to failure (plain strain)
 - Two types of loading:
 - *displacement controlled* – radial displacement loading
 - *load controlled* – internal pressurization



Displacement Controlled
FEM and Analytical solutions
continue past max load

Work performed by
Bill Scherzinger,
Sandia Labs

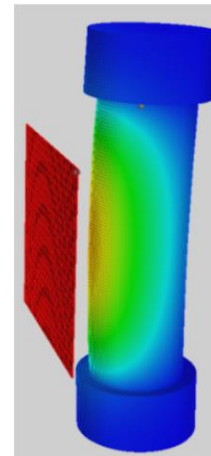
Solver Accuracy and Speed Assessment for Accurate Curve “Strength” Rankings

Test & temperature cases	CG 10^{-6} Failure psi (CPU time*)	FETI-CG 10^{-4} Failure psi (CPU time*)	FETI-CG 10^{-5} Failure psi (CPU time*)	FETI-CG 10^{-6} Failure psi (CPU time*)
try26-700C	704.0 (40.30)	702.0 (20.3)	703.8 (5.87)	703.7 (5.24)
try27-700C	704.9 (40.29)	704.1 (19.1)	704.2 (5.28)	704.2 (6.21)
try3-20C	1485.9 (21.1)	1490.70 (12.1)	1484.5 (7.8)	1484.5 (9.78)
try6-20C	1486.3 (15.2)	1487.20 (4.6)	1485.0 (2.9)	1485.0 (4.39)
try5-20C	1486.4 (16.0)	1492.60 (41.3)	1485.2 (20.7)	1485.2 (8.26)

* CPU times reported in Adagio output file via global output variable *cpu_time*. CG and FETI sims. were run on 192 processors of Red Sky

- Various hourglass treatments also investigated
- verified to not have significant effect on predicted failure pressures

- Results effectively unchanged when solver tolerance is changed from 10^{-5} to 10^{-6} (for 4tt mesh).
- CPU time not \gg for 10^{-6}
- Use 10^{-6} for production calcs.



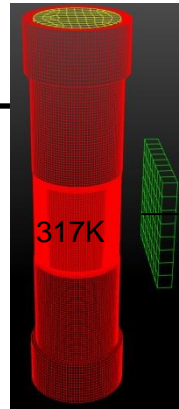
Models used for UQ



4tt

High-Low materials study
Isothermal - $\frac{1}{8}$ symmetry

4tt



4tt

Coupled Self Check mapping
PB# 1 Nearby problem
Used $\frac{1}{4}$ symmetry



1tt

32K

2tt

276K

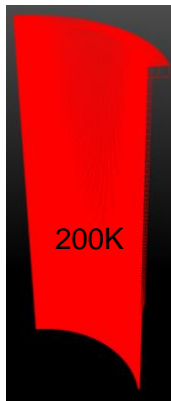
2.2M

7.5M

4tt

6tt

Mesh convergence
 $\frac{1}{4}$ symmetry



4tt

Solver parameters study
Isothermal - $\frac{1}{8}$ symmetry



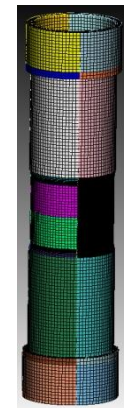
No contact

1tt-170K
2tt-1.5M
4tt-11.6M



contact

1tt-83K
2tt-570K
4tt-4M



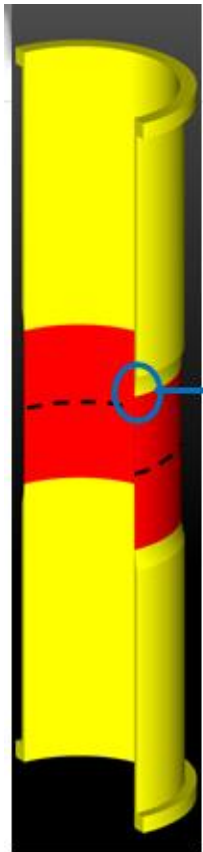
More contact

1tt-42K
2tt-285K
4tt-2M

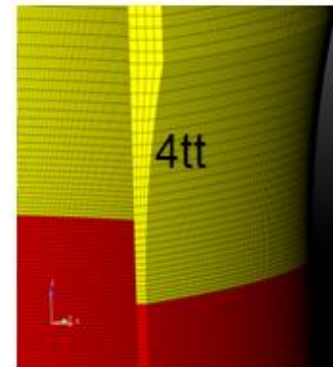
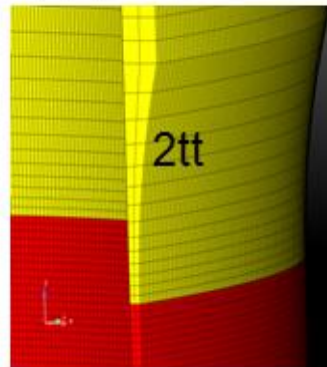
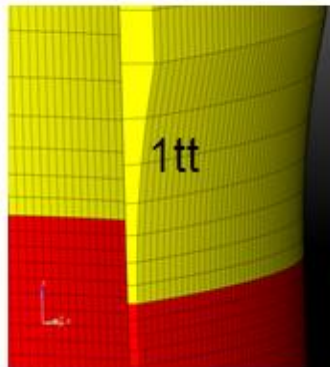
Validation to Experiments – Full symmetry

Pipe Bomb Calculation Verification

Mesh Refinement Studies



1/4 Pipe model = top 1/2 of half-pipe shown

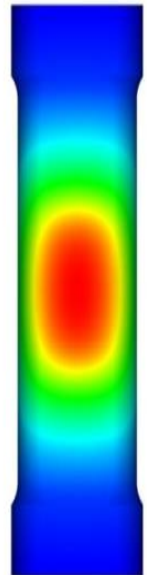
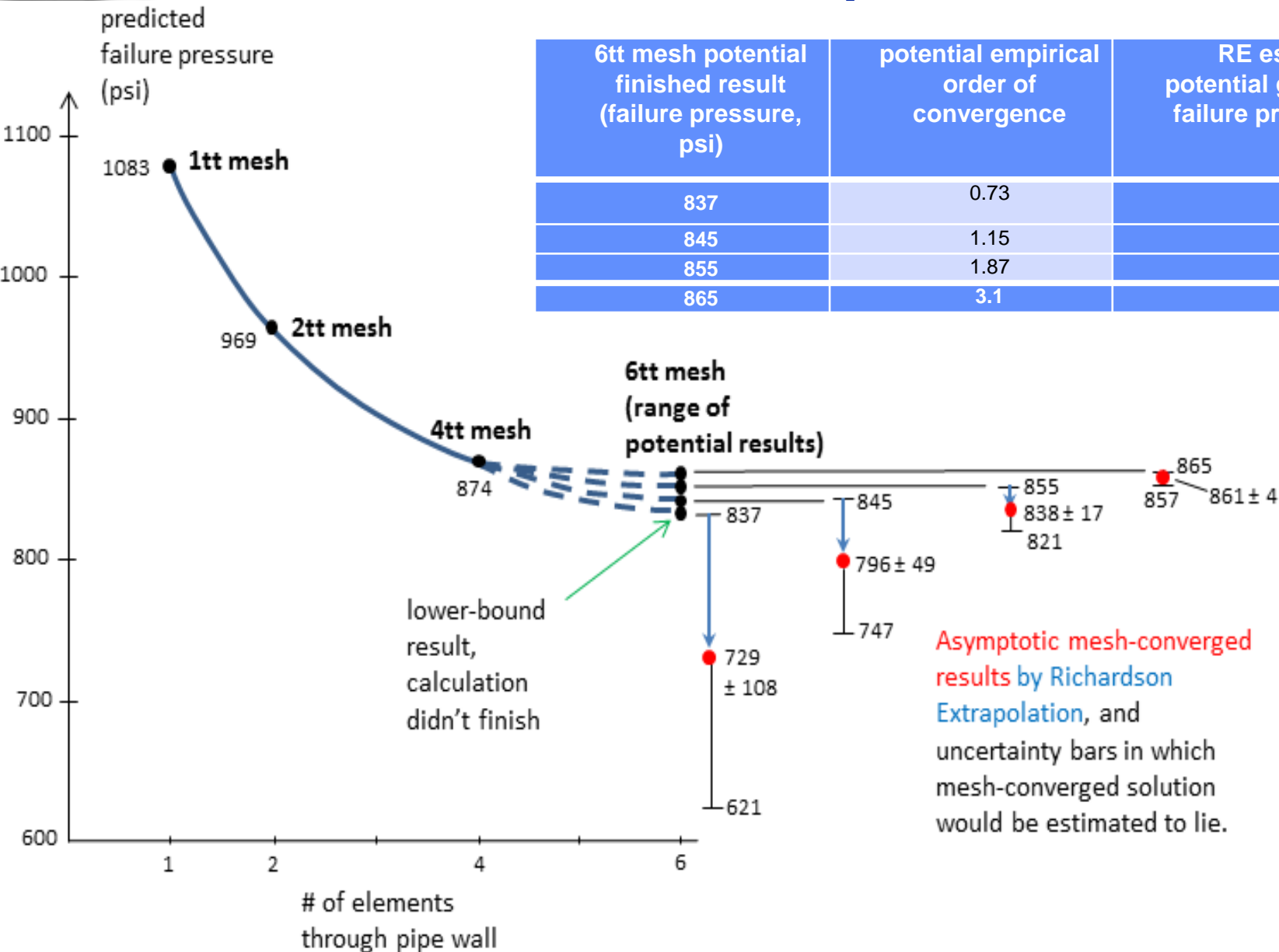


Geometrically similar meshes

Number of Elements thru thickness of wall	1	2	4	6
# Elements (1/4 model)	32,368	276,080	2,173,600	7,458,912
Pressure at Fail (psi)	1069	955	850	819.1* (*didn't finish, 36 days on 400cpu's)

Calculation Verification

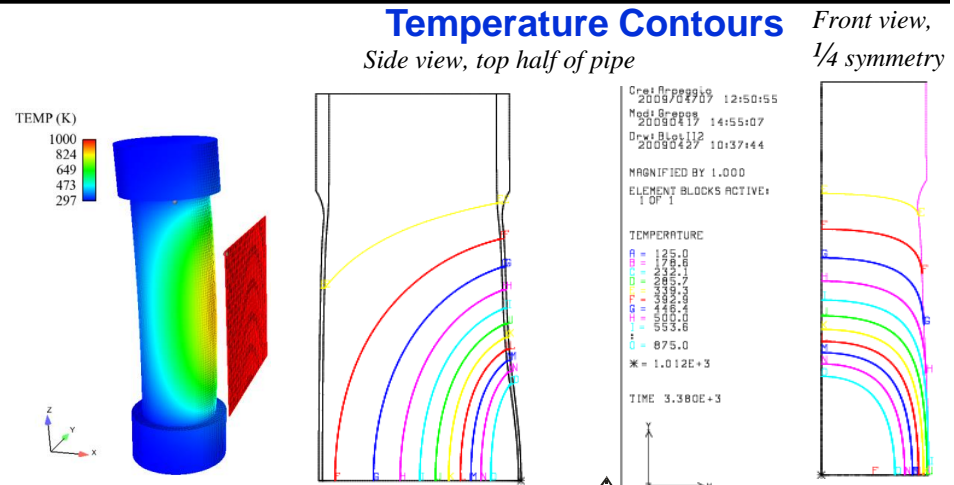
Mesh Study Results



Coupled Thermo-Mechanical modeling to Design Experiments & Thermocouple Locations to Reconstruct Temperature Field

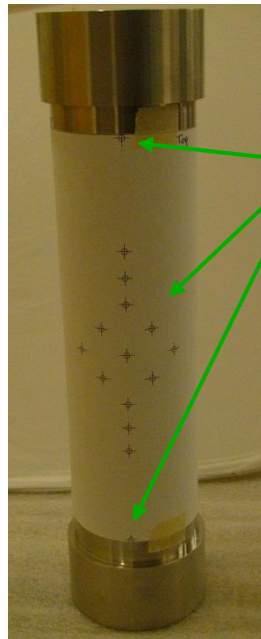
Model

- Pipe radiatively heated by plate
- Convection neglected
- Viewfactors change as pipe bulges toward plate at hot spot



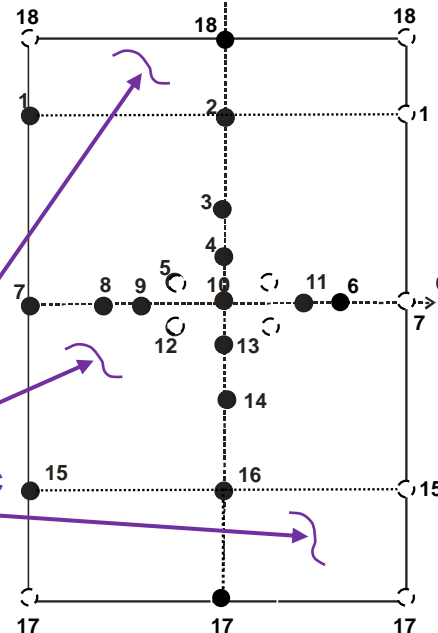
Experiment Design Quantities

- Size & location of plate relative to pipe
- # of thermocouples and locations to adequately reproduce temperature field on pipe surface
- in conjunction with design of interpolation method



**Thermocouples
(23 total, front
& back)**

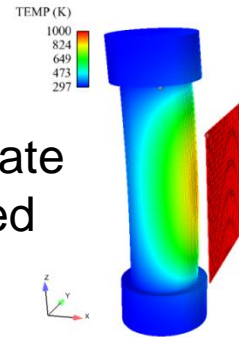
**8 Linear to Cubic
interpolation
patches (C^0
continuous)**



“Nearby Problem” to Quantify Error in Temperature Field obtained from TC Interpolation

Nearby Problem:

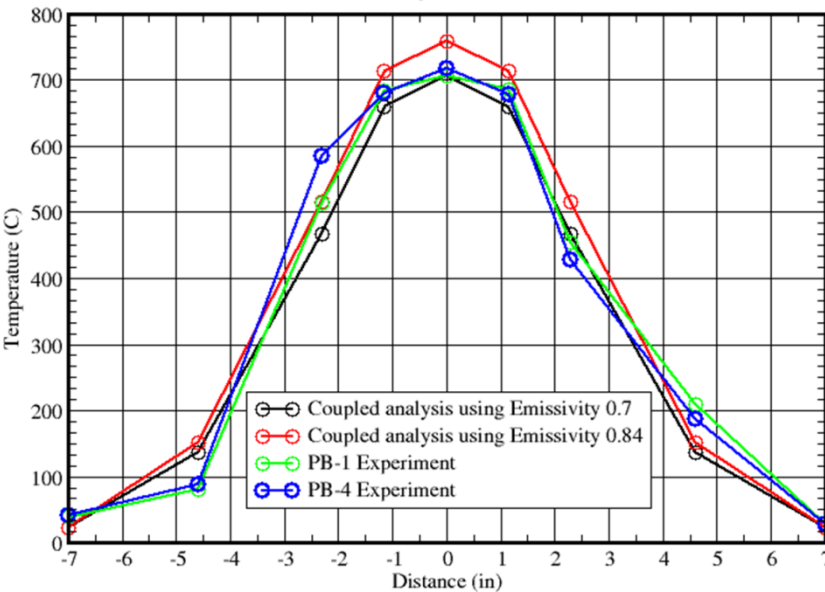
- Pipe irradiated by plate
- Convection neglected



Run Matrix

Simulation case	Time @failure (sec.)	Temperature @failure (C)	Pressure @failure (psi)	Tear Param. @failure
Coupled-high-0.84	2277.1	759.0	809.9	6.23
Interp-high-0.84	2332.8	758.9	857.9	7.27
Coupled-low-0.84	2255.9	759.0	791.6	5.40
Interp-low-0.84	2299.3	759.0	829.0	7.46
Coupled-high-0.7	2386.0	706.9	903.7	3.9
Interp-high-0.7	2400.9	706.9	916.2	4.43
Coupled-low-0.7	2350.4	706.8	873.0	3.50
Interp-low-0.7	2353.6	706.8	875.8	3.86

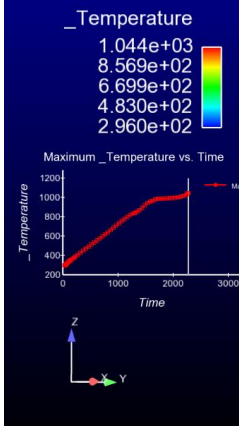
Axial temperature distribution
Pipe Bomb



Coupled Thermo-Mechanical sim.

Coupled experiment calculation
High strength, Nominal Emissivity
with TC outputs for self check (right)

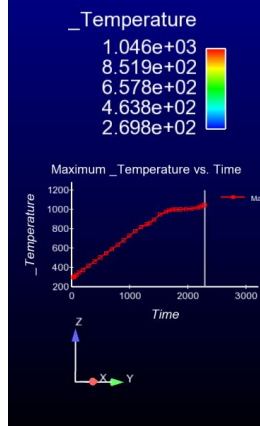
Time = 2273



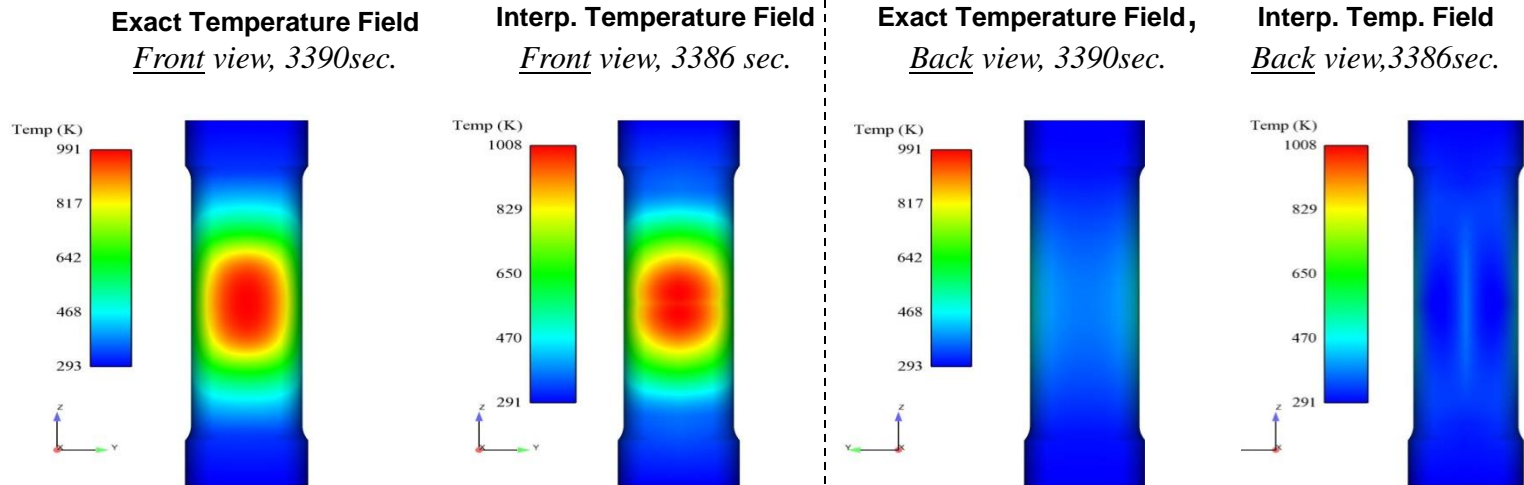
“Self-Check” sim. w/TC interp. temperatures

TC interpolation check calculation
Using coupled TC outputs (left)

Time = 2290



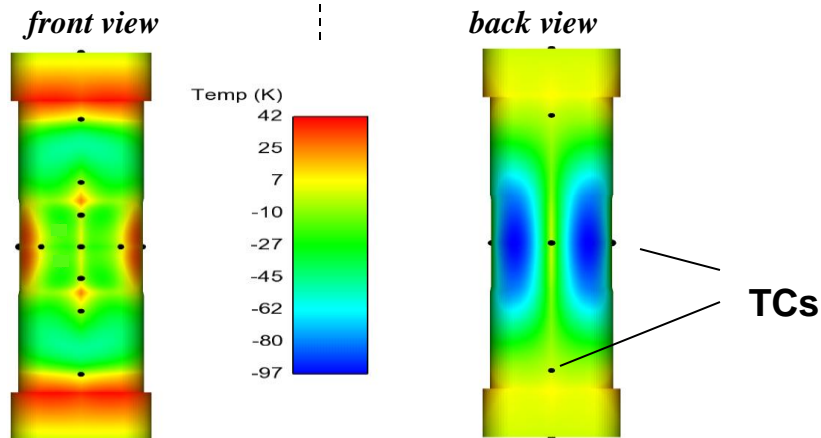
Temperature Field Reconstruction Error due to Spatially Sparse TC data



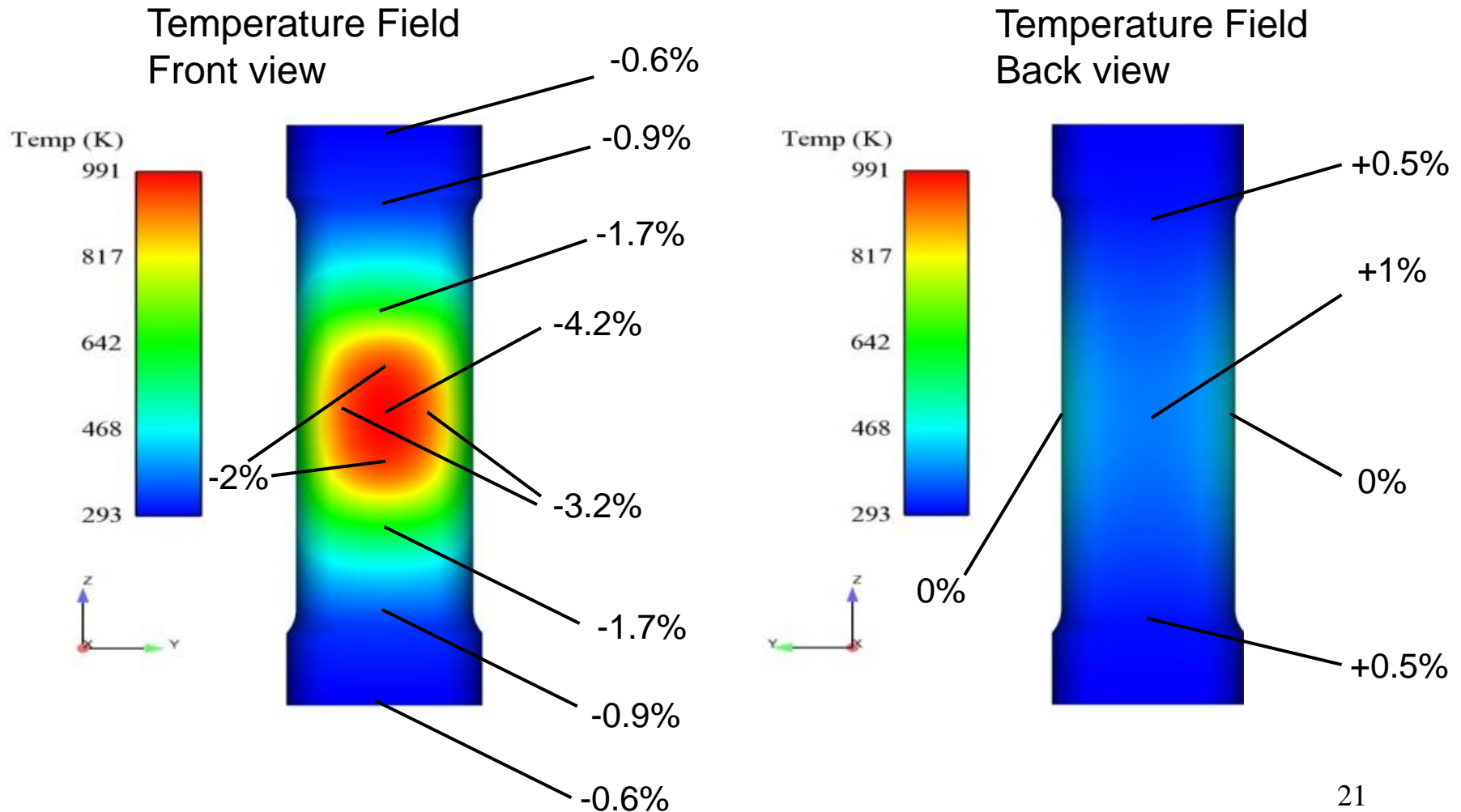
Difference (error) Plots



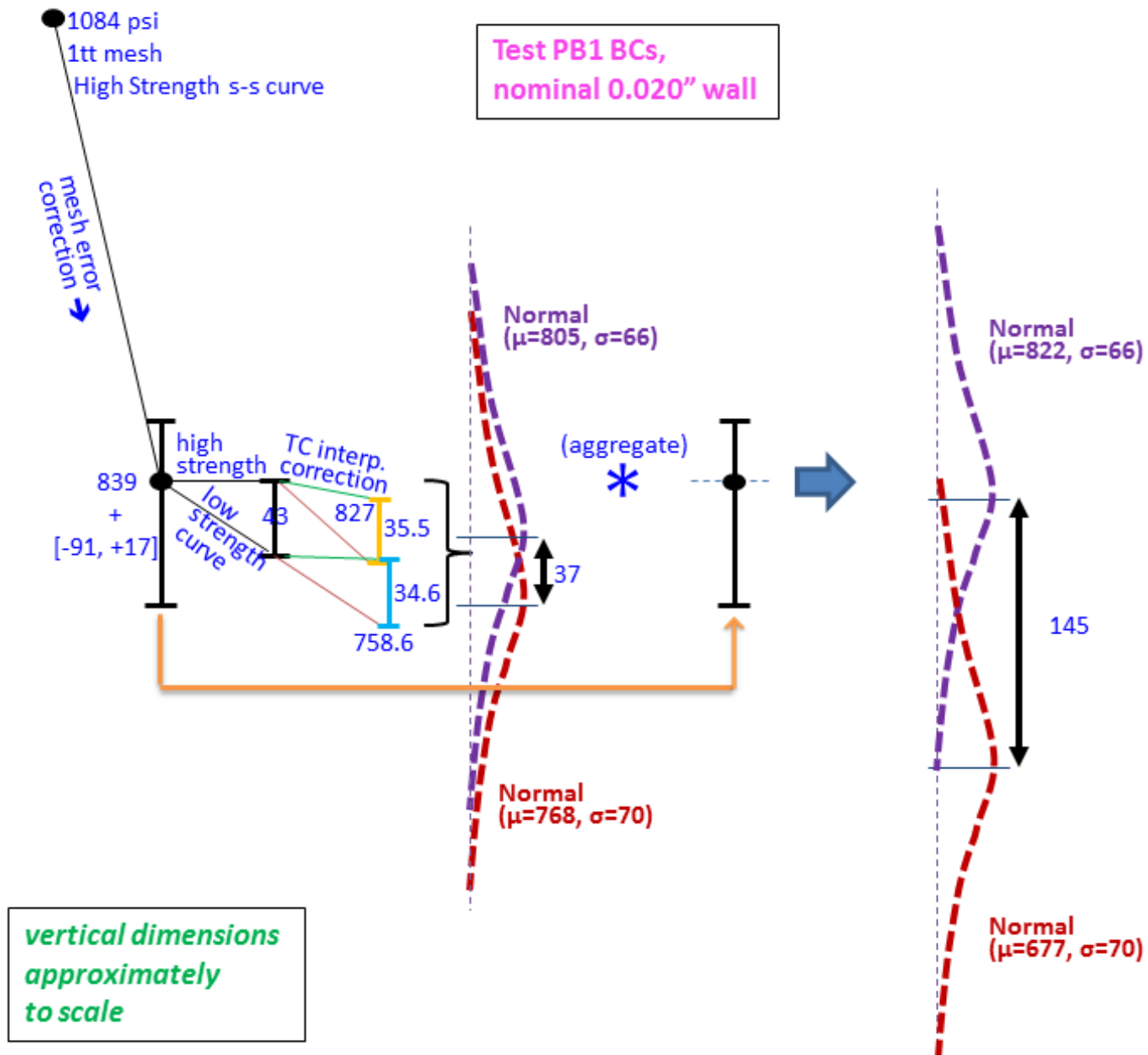
- temperature interpolation error is characterized and corrected for validation predictions
- a ~4% (35 PSI) error in predicted failure pressure if not corrected for interp. error



Bias Correction of TC Temperatures for Contact Resistance and Fin Effects

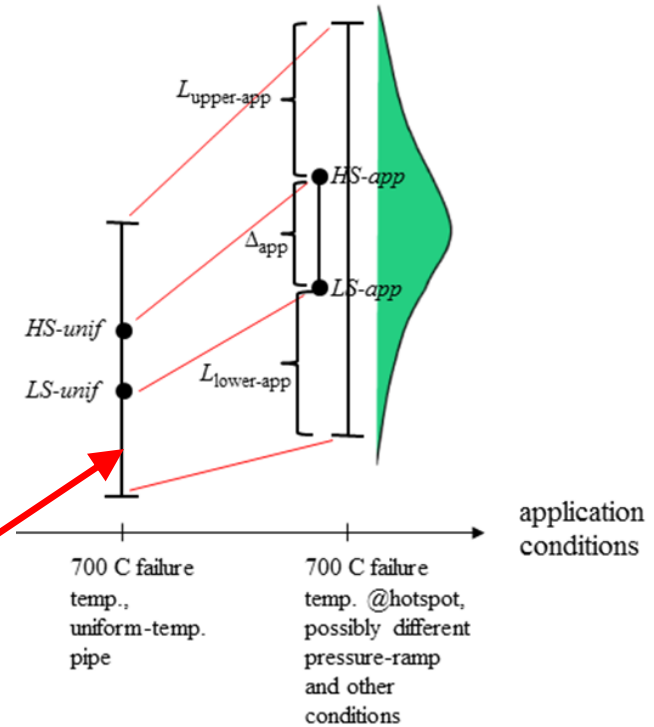
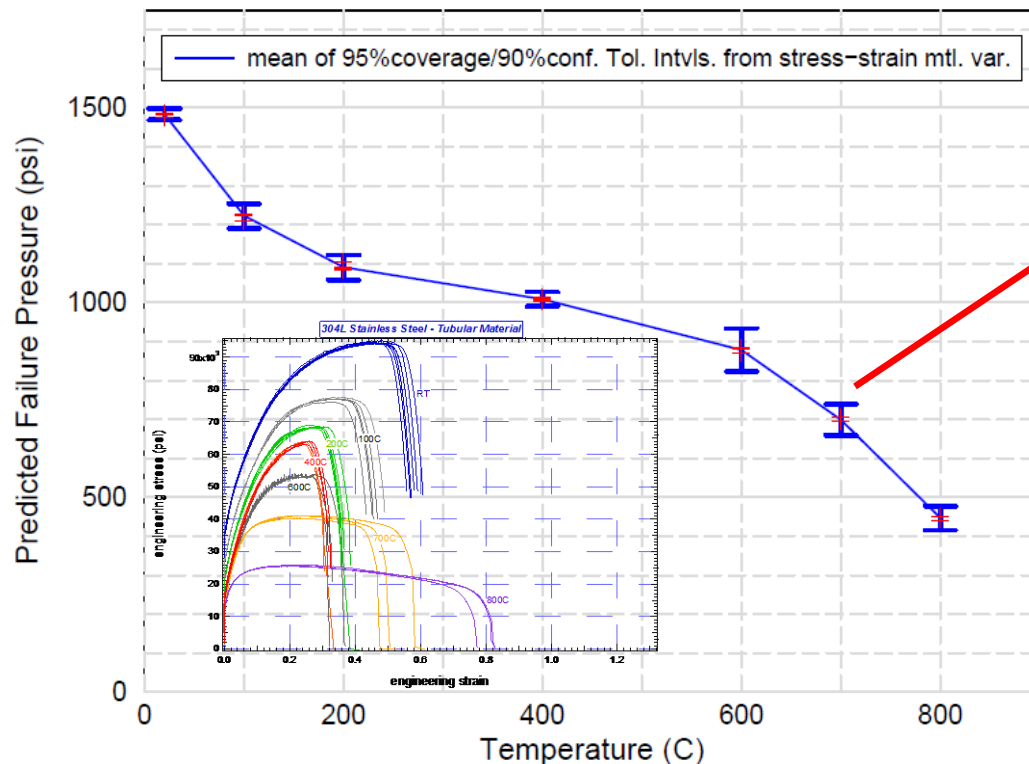


Simulation UQ Roll Up



Economical Parameterization of TIs by **High** & **Low** Strength Material Curves

- Tolerance Intervals are constructed from multiple stress-strain curves
- But TIs can be parameterized by 2 s-s curves for only 2 Val./UQ sims. w/ full-geom. model



Processing of Experimental Failure Pressures



Normalize Experimental Results
to the same Reference Conditions Input to Model
for “Apples to Apples” results comparisons

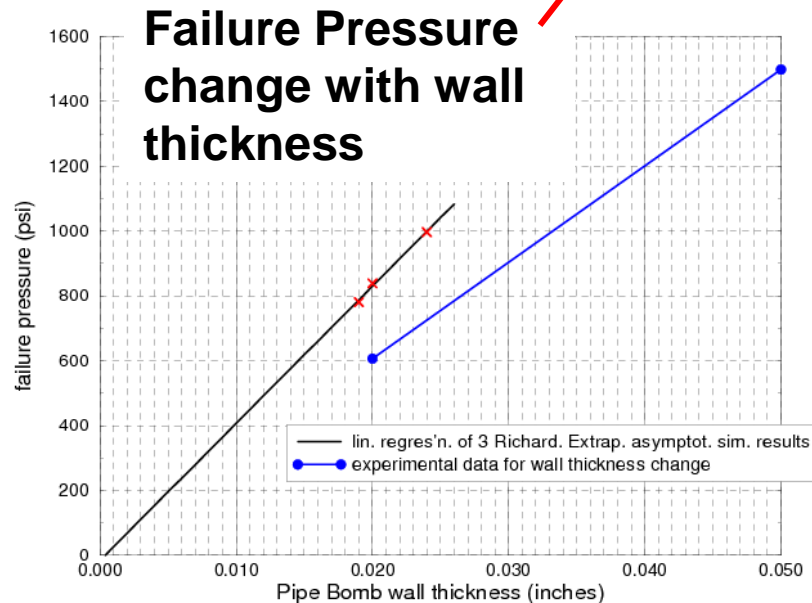
Taylor Series approach:

Example Nominal wall
thickness = 0.02”

$$P_{failPB4}(\vec{x}_{PB1ref}) \approx P_{failPB4}(\vec{x}_{PB4}) + \sum \frac{\partial (P_{failPB4})}{\partial (x_i)} \cdot (x_{i_PB1ref} - x_{i_PB4})$$

Normalized
Failure Pressure
in Test PB4

Measured
Failure Pressure
in Test PB4



wall thickness
uncertainty

Part No.	Wall Thickness
1	.019/.022"
2	.020/.022"
4	.020/.024"
5	.020/.022"
7	.019/.022"
8	.020/.021"
9	.019/.0225"
10	.019/.021"

Spreadsheet Processing of Experimental Results & Uncertainties

“Linear+” propagation method samples uncertain sensitivities $\partial(P_{fail})/\partial X_i$ as well as the uncertain input variables X_i

Systematic uncertainties correlated with uncers. in same columns of the spreadsheets of the other 3 experiments

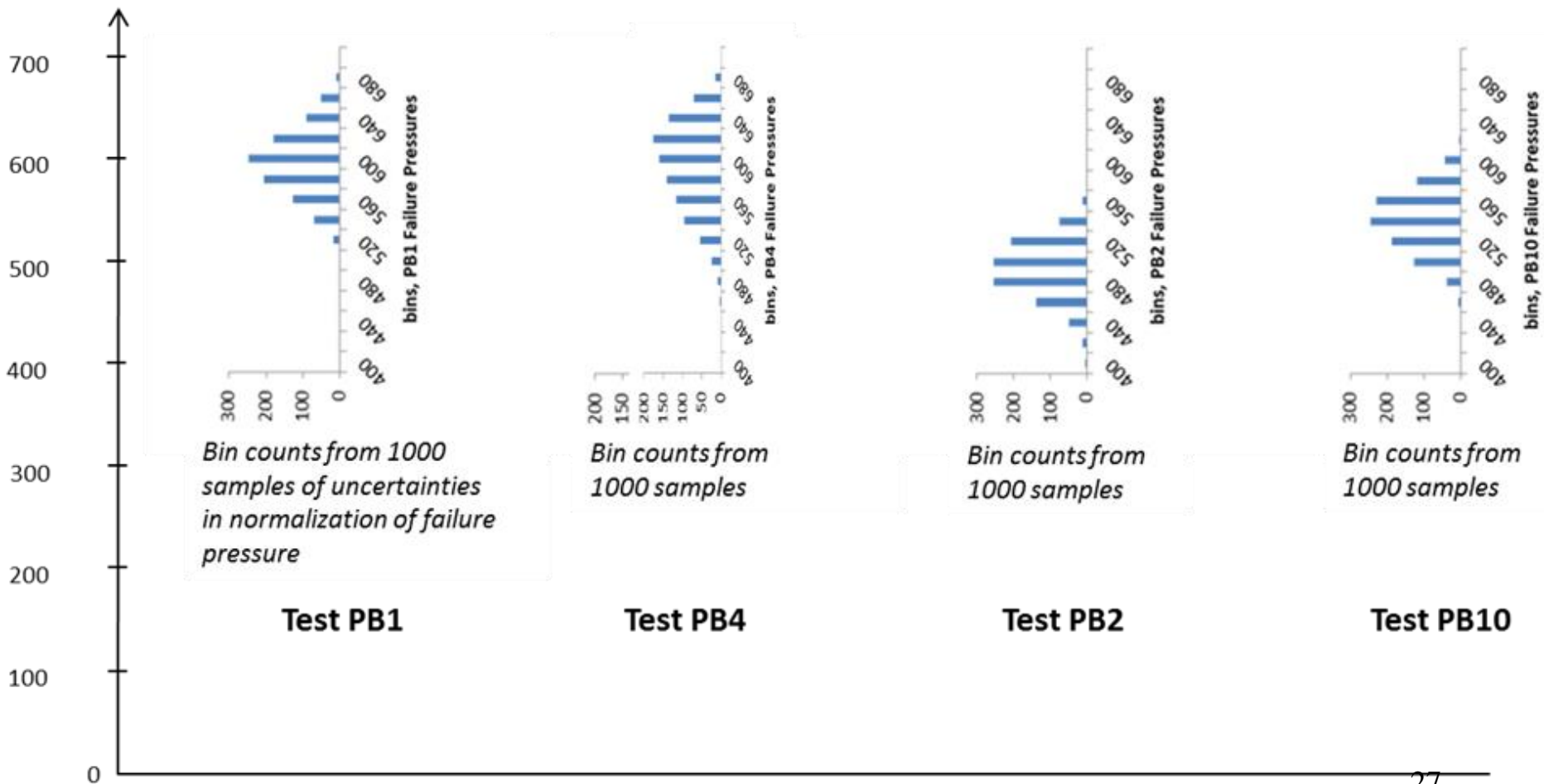
- Account for random and systematic errors/uncertainties in measurement/processing/inference of experimental input and output quantities

- Normalize for:
 - nominal (measured) differences in experimental inputs in the replicate tests

realization j	B1	B2	B3	B4	B5	B6	B7	B8	B9	B10	B11	B12	B13
		$\Delta P_{fail_measPB4,j} = A2j, \text{ systematic betw. PB1 \& PB4}$	$w_{actPB4,j} = [0.02, 0.024] \text{ inches}$	$\Delta w_{PB4,j} = w_{nomPB4} - w_{actPB4,j} = 0.02'' - B3$	$\frac{\partial(P_{fail})}{\partial(w)} ; j = A5j, \text{ systematic betw. PB1 \& PB4}$	$= (B4*B5) ; \text{ psi}$	$\Delta T_{meas-TC/DAQ,j} = [-0.0025*711, +0.0025*711] \text{ C}$	$\Delta T_{meas-contact,j} = A8j, \text{ systematic betw. PB1 \& PB4}$	$\Delta T_{TC4location,j} = [-15, 15] \text{ C}$	$\frac{\partial(P_{fail})}{\partial(T@fail_point)} ; j = A10j, \text{ systematic betw. PB1 \& PB4}$	$= (B7 + B8 + B9) ; *B10j$	$[P_{fail_model}(x_{nomPB1}) - P_{fail_model}(x_{nomPB4})]$	$= 655 \text{ psi} + (B2 + B6 + B11 + B12) ; \text{ psi}$ $= P_{failPB4}(x_{nomPB1}) ;$
j	B2=A2, sys	B3=PB4wall	B4=Δ0.02"	B5=A5, sys	B6 = B4*B5	B7=ΔTC_DA	B8=A8, sys.	B9=ΔTC_loc	B10=A10, sy	B11=B10*(B7	B12=PfPB1-	B13=655+B2+	
1	7.76	0.02266	-0.00266	30701.34	-81.65	-1.39	0.52	12.84	-2.07	-24.83	-8.00	548.29	
2	-5.88	0.02327	-0.00327	37837.13	-123.76	0.91	1.18	-11.81	-1.87	18.17	-8.00	535.53	
3	-0.69	0.02195	-0.00195	26116.40	-51.05	0.07	1.61	-6.42	-1.83	8.66	-8.00	603.93	
999	-8.55	0.02399	-0.00399	37867.63	-150.97	0.67	0.10	-14.21	-2.15	28.95	-8.00	516.43	
1000	6.28	0.02353	-0.00353	29062.56	-102.57	-0.85	0.06	2.18	-1.99	-2.78	-8.00	547.94	
avg	0.00	0.02195	-0.00195	32361.71	-62.83	-0.05	0.91	-0.39	-1.94	-0.96	-8.00	583.21	
stdev	5.76				39.06					17.02		42.97	

Normalized Failure Pressures accounting for Experimental Uncertainties

Uncertainty of
normalized experimental
failure pressure (psi)



UQ Roll Up for Experiments

Uncertainty of 0.025 & 0.975 percentiles of Failure Pressure

- these %iles combine uncertainties in both mean & variance of response

assoc. w/ experimental factors in the tests:

- uncertainties in measurement and estimation of test conditions, responses, and normalization quantities*

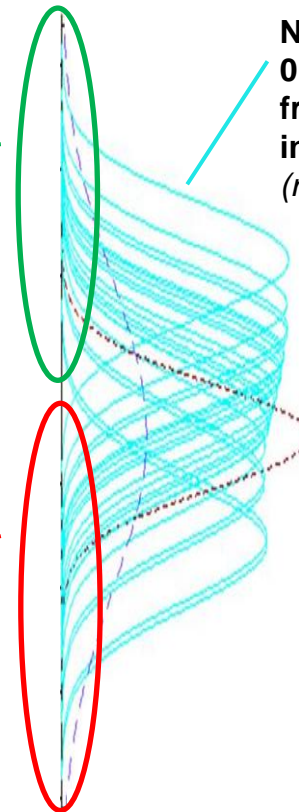
assoc. w/ # of tests

assoc. w/ response variability attributed to stochastic elements/ behaviors in systems tested

Uncertainty of 90% conf. upper bound on 0.975 percentile of experimental response (failure pressure)

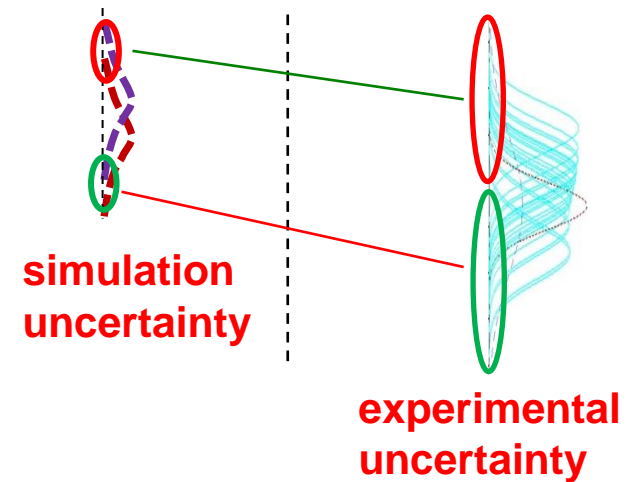
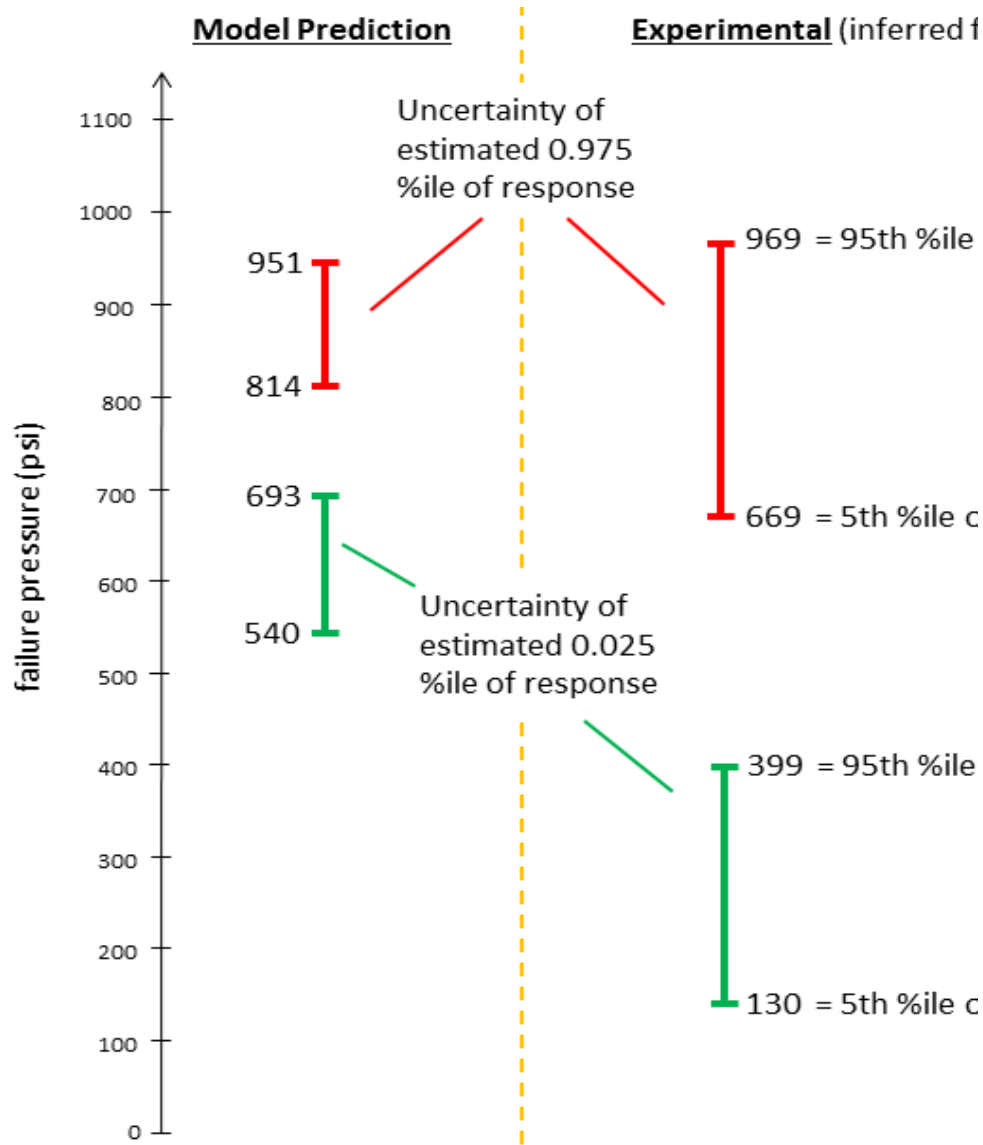
Uncertainty of 90% conf. lower bound on 0.025 percentile of experimental response (failure pressure)

Normal PDFs fit to 0.95/0.90 Tolerance Intervals from Small # of Tests involving uncertainty (notional PDFs for illustration)



Results of Percentile Comparisons:

Lower Percentile of Predicted Failure Pressure is NonConservative for Intended Model Use



Closing Remarks

- **The Real Space validation methodology is versatile and practical, geared for:**
 - expensive computational models (economical in # of simulations)
 - stochastic phenomena and models
 - multiple replicate experiments with random and systematic uncers.
 - few replicates (sparse experimental data)
 - appropriate representation and roll up of various types, sources, and representations of uncertainty
 - aleatory and epistemic
 - probabilistic, interval, and discrete variables and functions
- **Real Space Validation results are:**
 - relatively straightforward to interpret
 - especially relevant for assessing models/quantities to be used in the analysis of performance and safety margins for design and risk assessment