



Progress Report: Bowl Geometry Impact on Mixture Formation

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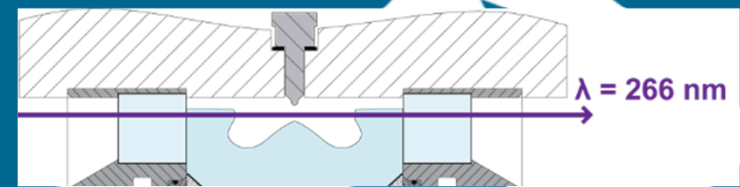
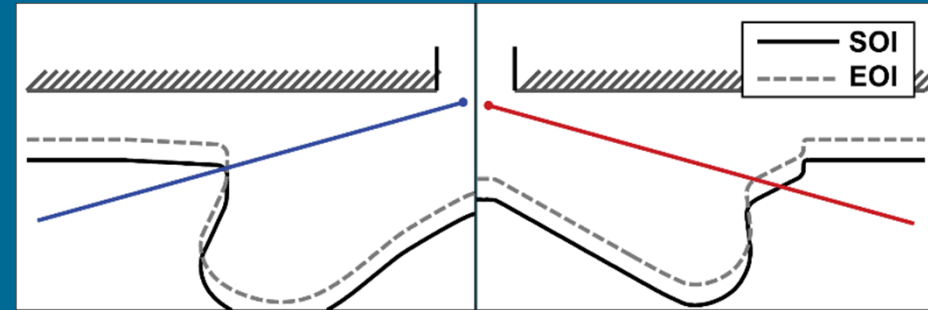
Sandia National Laboratories

October 19, 2016

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Outline

- State of knowledge and open questions
 - Early injection PCCI (LTC): mixture formation
 - Conventional diesel combustion: late-cycle mixing
- Experiments and analysis techniques
 - Fuel tracer PLIF
 - High speed liquid scattering imaging (in progress)
- Results
 - Three injection timings; three spray targeting cases
 - Differences in mixture formation processes; implications for combustion efficiency
- Ongoing experimental studies



State of knowledge: performance and emissions

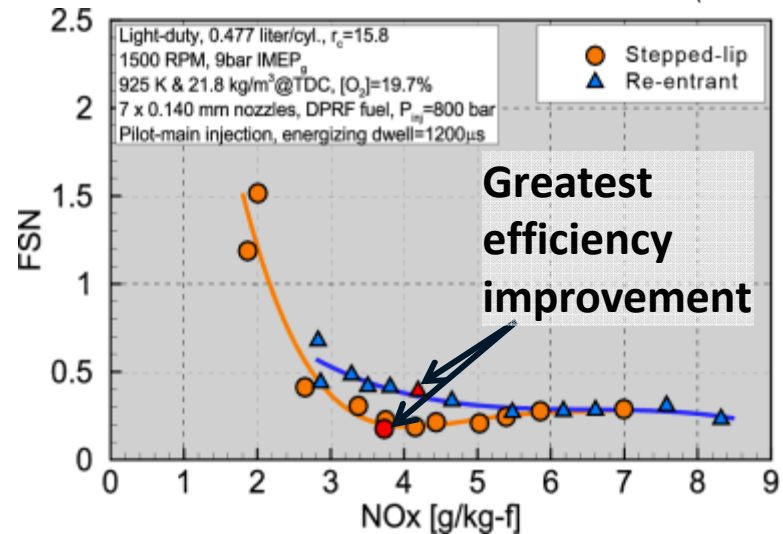
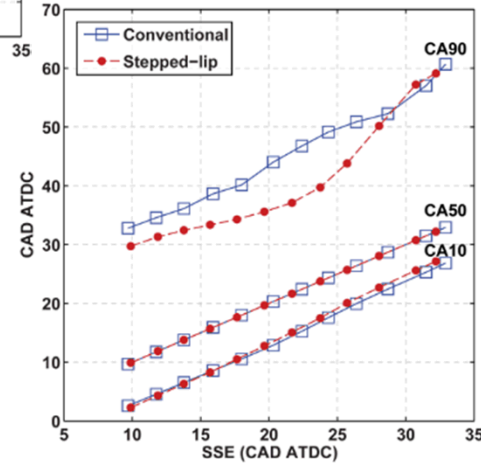
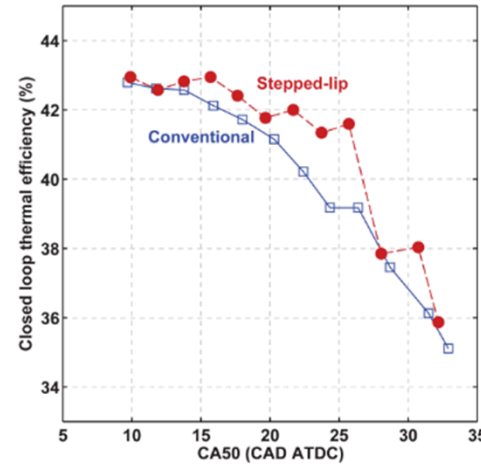
- **Conventional diesel combustion with a single pilot injection (CDC)**

- The stepped-lip piston yields higher thermal efficiency for some injection timings
- Efficiency gains appear to be most closely associated with faster late-cycle heat release (analysis ongoing)

- Stepped-lip efficiency gains coincide with reduced soot and NOx emissions

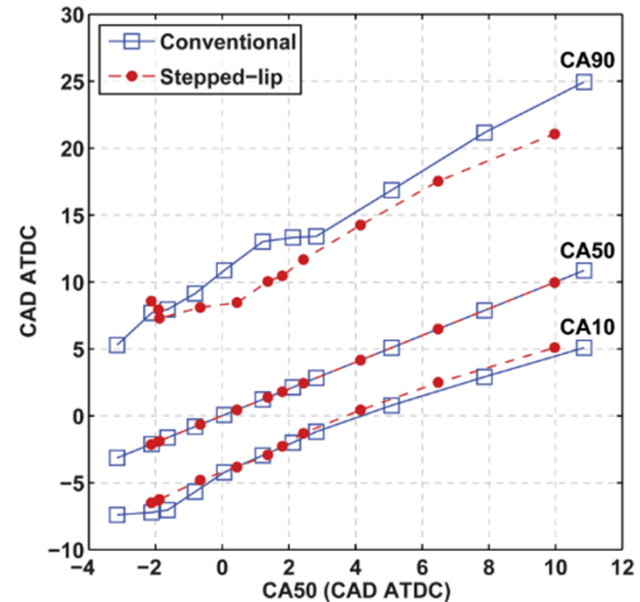
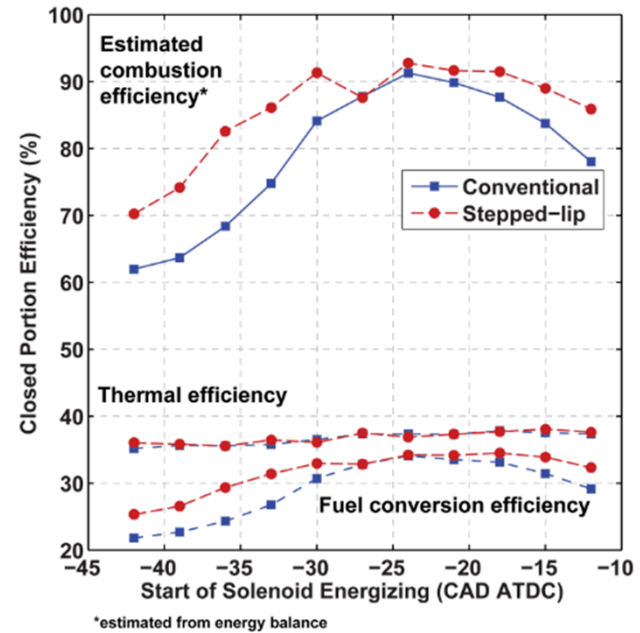
- **Open questions**

- What mechanisms govern these improvements in efficiency and emissions?
- Can these mechanisms be utilized to increase thermal efficiency while reducing soot and NOx emissions?



State of knowledge: performance and emissions

- **Early injection, PPCI combustion (LTC)**
 - Peak fuel conversion efficiency is slightly higher with the stepped-lip geometry
 - Efficiency gains with the stepped lip piston are most closely associated with improvements in combustion efficiency
 - Mixture formation with the stepped-lip bowl is less sensitive to injection timing
 - Late-cycle heat release appears to be slightly enhanced with the stepped-lip bowl
 - For this prediction, only premixed combustion was considered. The thermal efficiency improvement is minimal.



State of knowledge: bowl geometry impact on in-cylinder motored flow

- **Conventional re-entrant bowl geometry**

- Interactions between swirl and squish create a toroidal vortex in the bowl

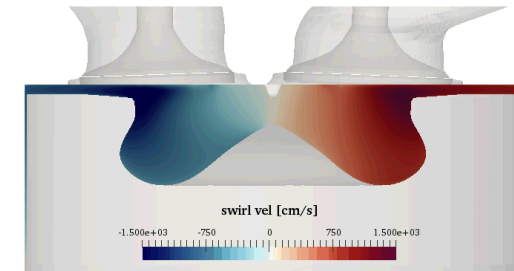
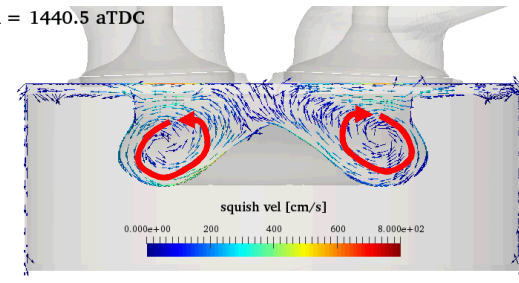
- **Stepped-lip bowl geometry**

- Expected squish / reverse squish velocities are half as fast as for the conventional bowl
- Reverse squish flow during the early expansion stroke is closer to ideal for the stepped-lip bowl – believed to be an inertial effect
- Higher degree of azimuthal asymmetry in near-TDC flow structure for the stepped-lip bowl compared to the conventional bowl

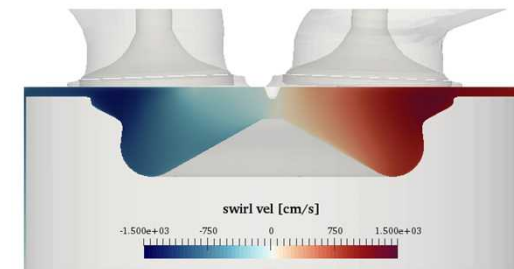
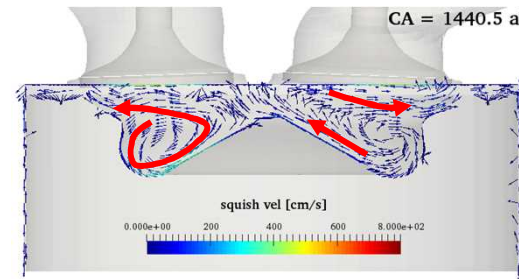
- **Conclusions**

How do different bowl geometries impact the flow behavior? How do different bowl geometries impact the flow structure?

CA = 1440.5 aTDC

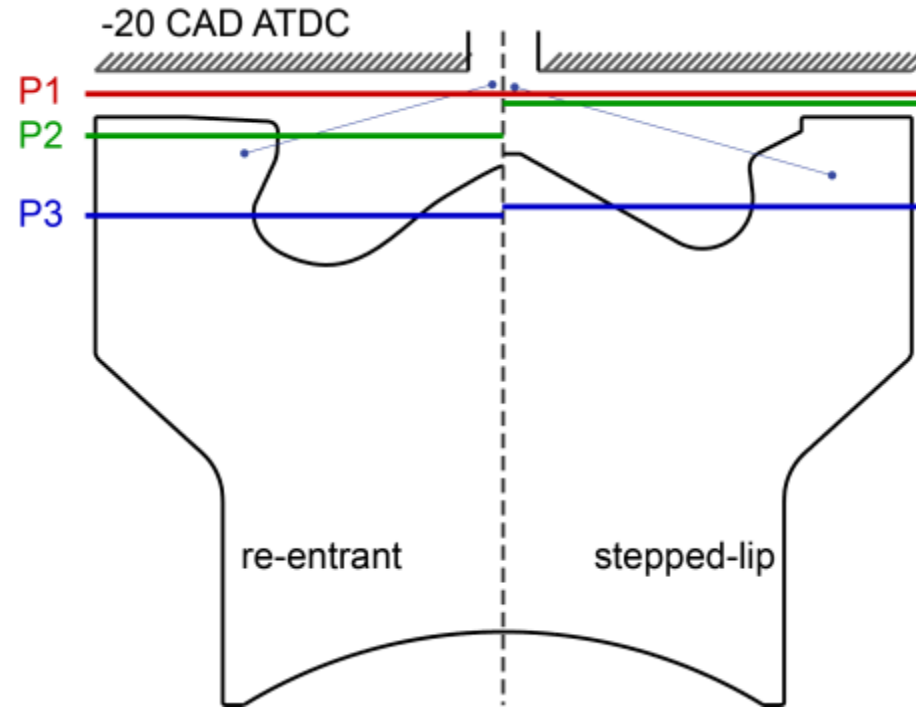


CA = 1440.5 aTDC



PLIF imaging plane locations

- **Plane 1 (position relative to piston and engine changes with crank angle)**
 - Halfway between piston top and cylinder head
- **Plane 2 (moves with piston)**
 - Conventional piston: passes through bowl rim
 - Stepped-lip piston: 1.4 mm above the piston top to minimize reflection
 - Passing a laser sheet through the step region is not possible
- **Plane 3 (moves with piston)**
 - Conventional piston: 10 mm below piston top
 - Stepped-lip piston: 10 mm below piston top
- **Distortion correction via a semi-automated, ray tracing based routine**
- **Iterative computation of fuel concentration**



Sample PLIF result images

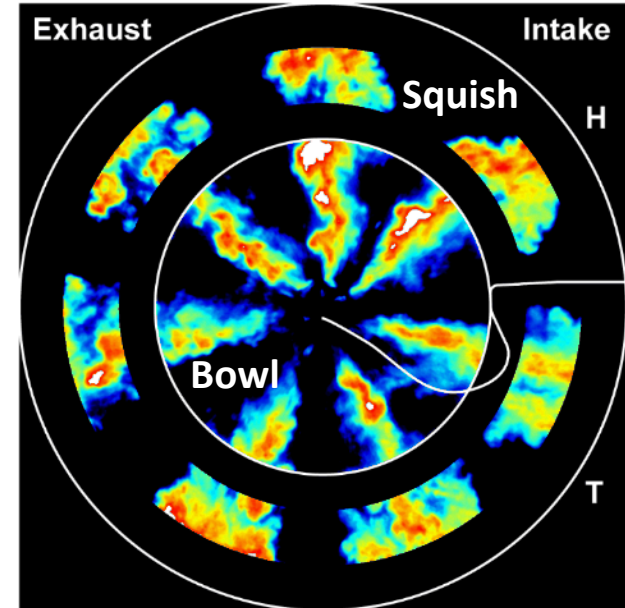
- **Conventional piston**

- The region above the bowl is fully visible
- Only a portion of the squish region is visible

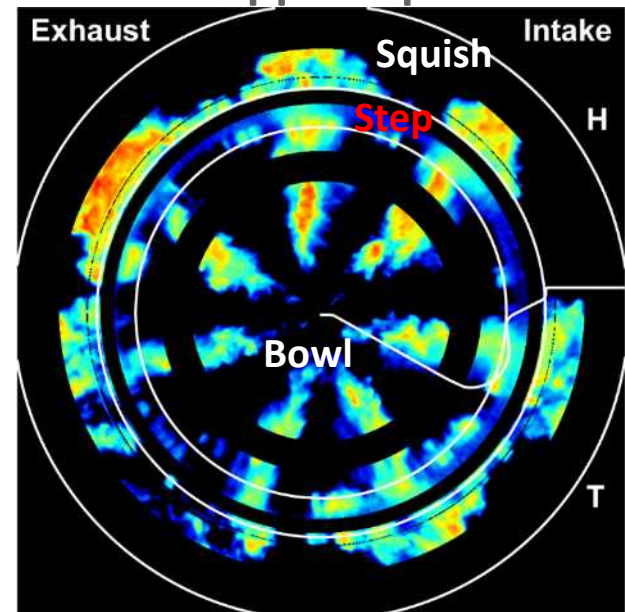
- **Stepped-lip piston: more complex distortion correction than with the conventional bowl**

- Dramatic change in bowl profile radius of curvature at bottom lip severely degrades image quality
- The region in the bowl above the bowl is split into two regions (shown at right)
- Full view of bowl above the bowl is not possible (text at right)
- A limited amount of information is available in the bowl region

Conventional

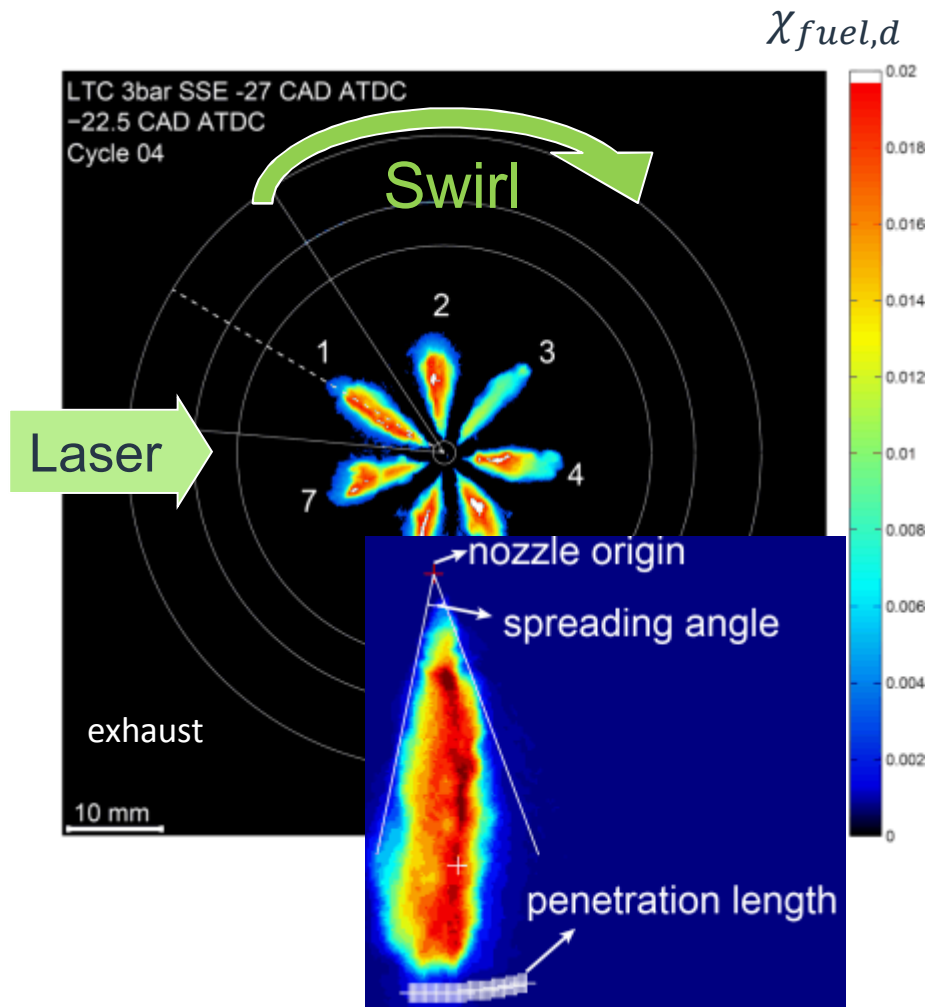


Stepped-lip



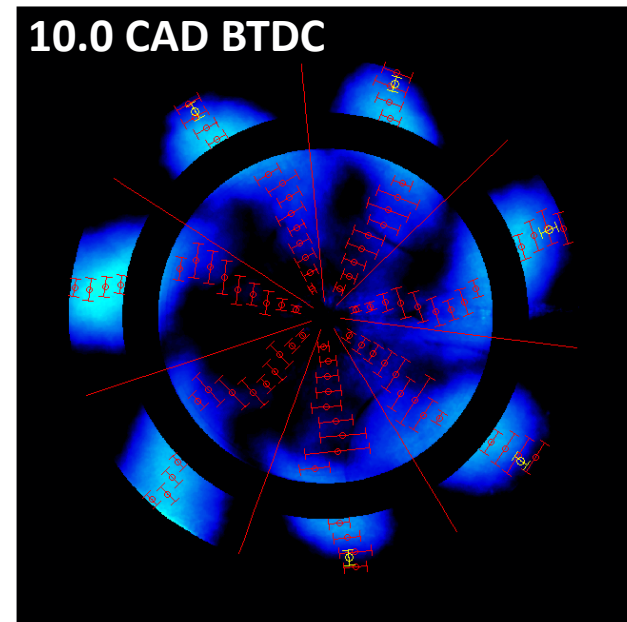
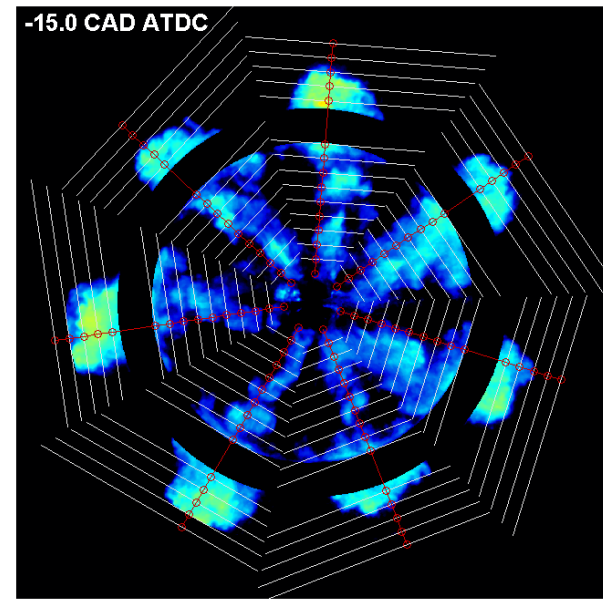
Spreading angle and penetration length quantification

- Background subtraction:
 - Background reflection problems are mitigated by subtracting a constant from each image
- Spreading angle and penetration length calculation using ECN script (Siebers, SAE Paper 960034).
 - Since measurements were not taken perpendicular to the spray axis, calculated “spreading angle” is projection on the field-of-view
- Calculations performed for spray volume fraction over cycles ($\chi_{fuel,d}$)



Spray structure characterization

- Fuel concentration-weighted centroid evaluated along multiple profiles (shown as white lines) across each jet
- Ensemble average provides simplified representation of jet structure, insight into how jets are deflected by swirl
- Angular deviation from the axes can be plotted as a function of radius for different crank angles
- This analysis can be applied to CRF as an apples-to-apples comparison



Experimental setup: high speed fuel injection imaging (in progress)

• Engine operation

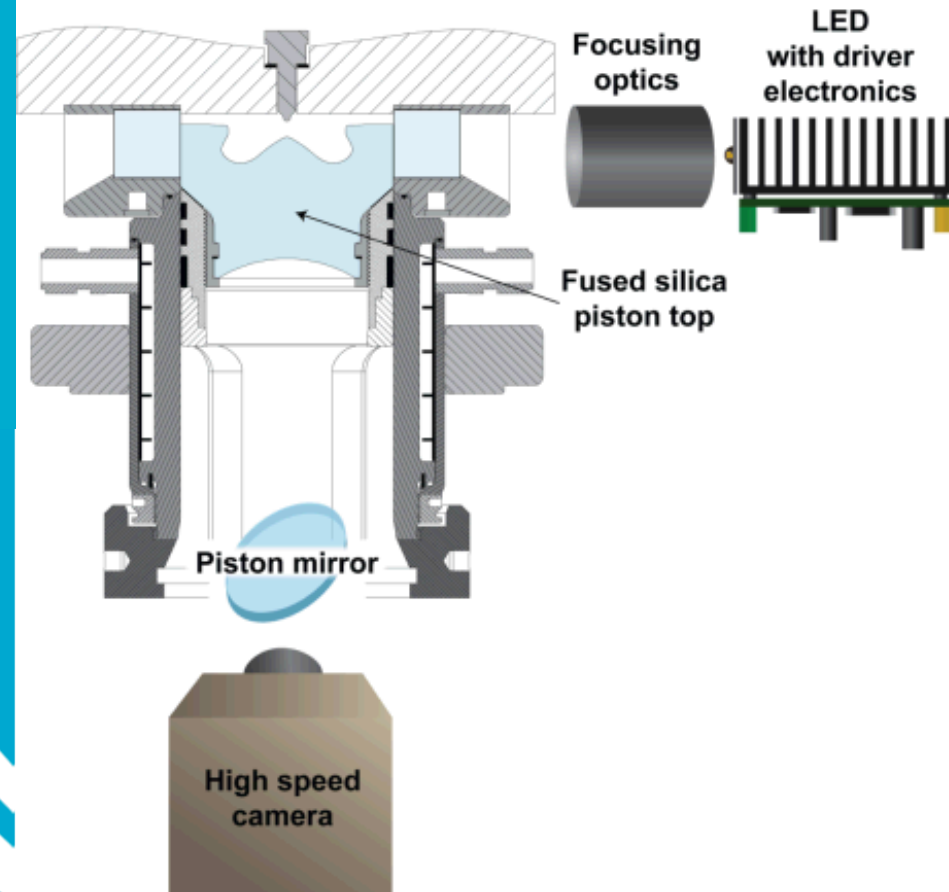
- 1500 rpm; motored operation (0% O₂)
- Fuel: 42 vol% n-hexadecane + 58 vol% heptamethylnonane
- Swirl ratio: 2.2
- Skip-fired (1 fired : 4 skipped)

• Elastic scattering Illumination source

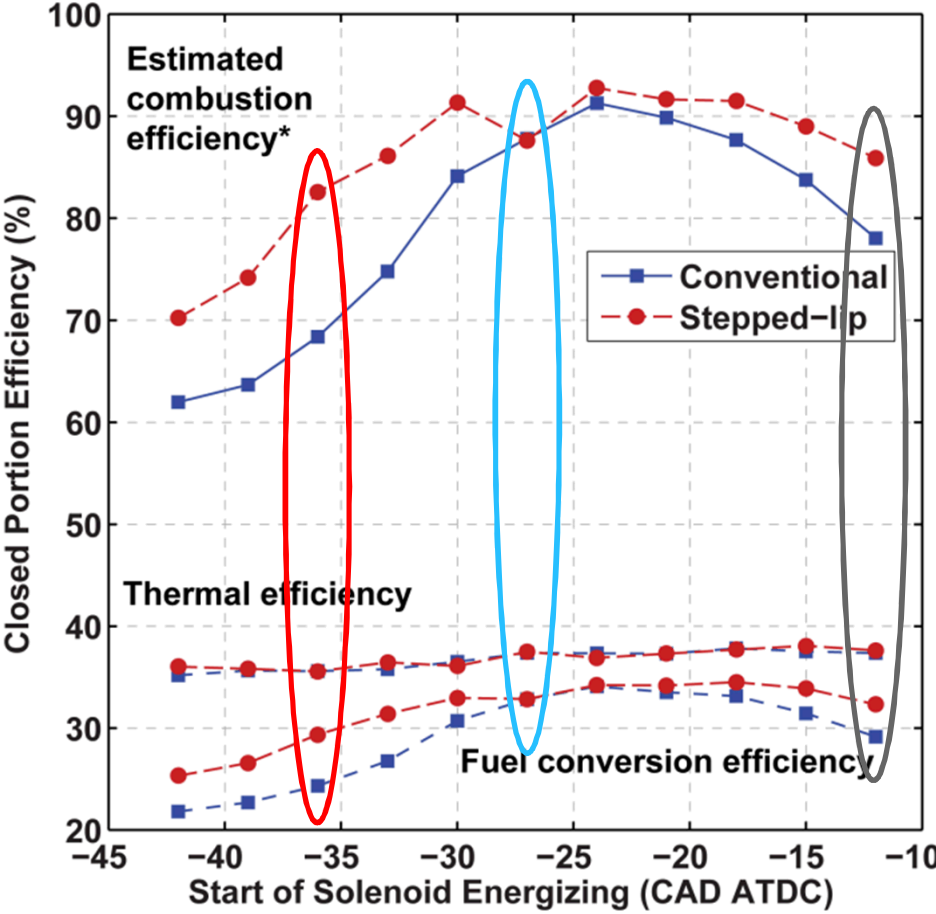
- Illumination through side window
- High-energy pulsed InGaN die LED, 3μs pulses
- Non-uniform illumination

• Imaging through piston

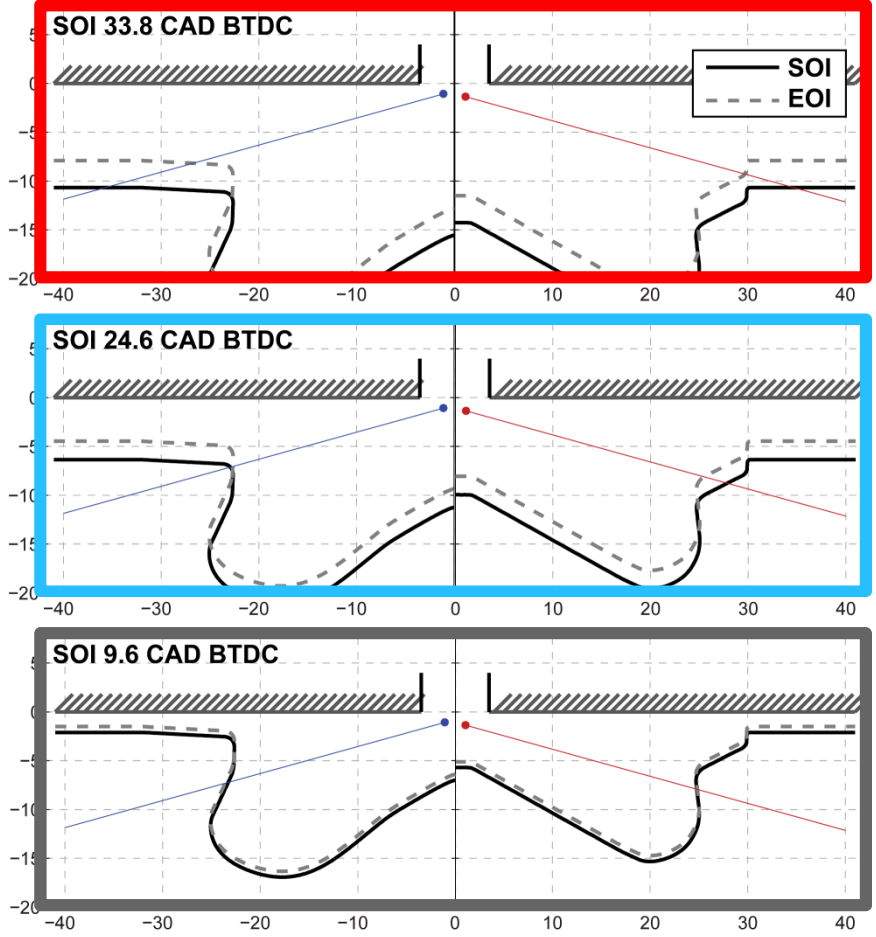
- Phosphor-coated monochrome CMOS camera
- Frame rate: 40 kfps
- Resolution: 225 CAD resolution (500 μm)
- Image size: 1000 pixels



Early injection PCCI – spray targeting for three injection timings of interest

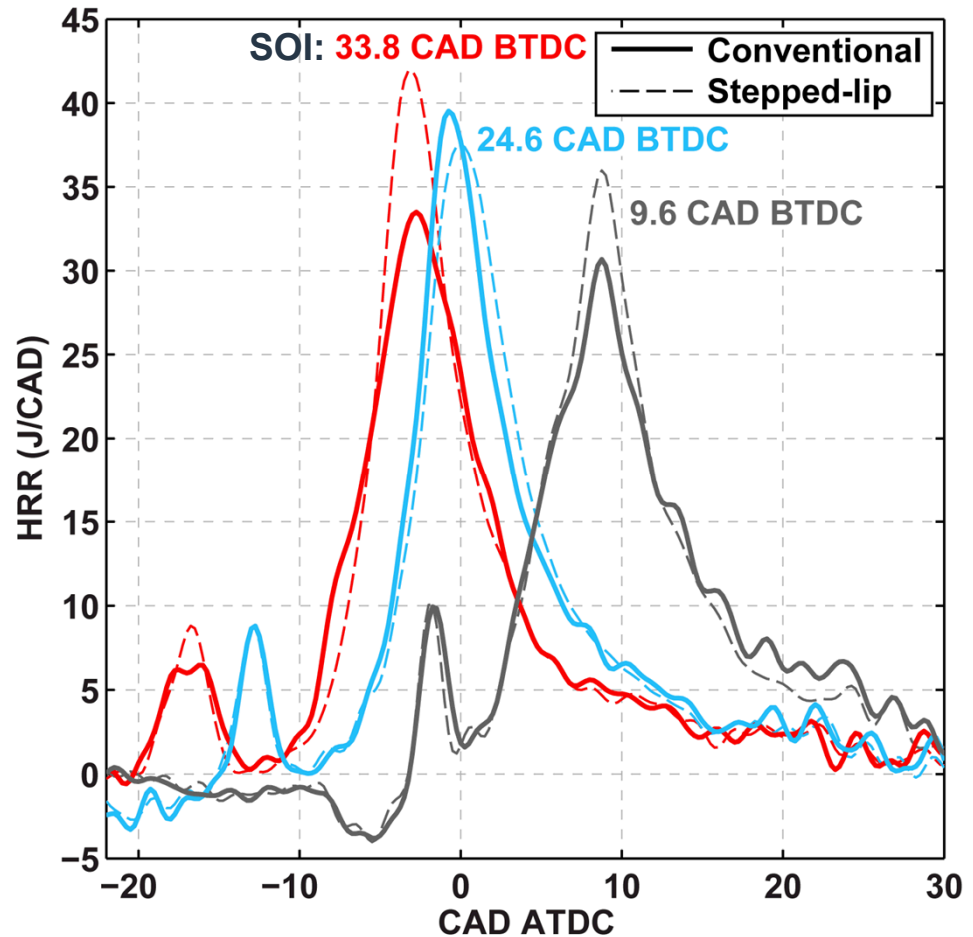


*estimated from energy balance



Heat release comparison: PCCI operation

- **Early SOI: 33.8 CAD BTDC**
 - Most prominent differences during second stage (high temperature) heat release
 - Can differences in the mixture fraction distribution be determined at the start of heat release that could explain the more robust combustion?
- **Intermediate SOI: 24.6 CAD BTDC**
 - Similar heat release profiles, but heat release is slightly advanced with the conventional piston bowl
 - How do spray targeting effects impact the mixture formation process?
- **Late SOI: 9.6 CAD BTDC**
 - Differences are observed during the second stage heat release and in the later stage of combustion (15-25 CAD ATDC)



Mixture formation processes compared: SOI 33.8 CAD BTDC (Plane 1)

Single cycles

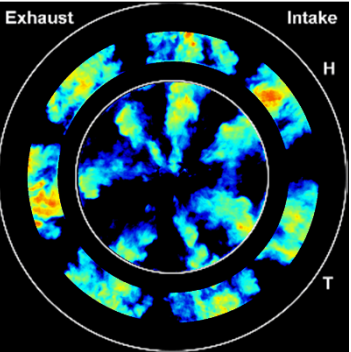
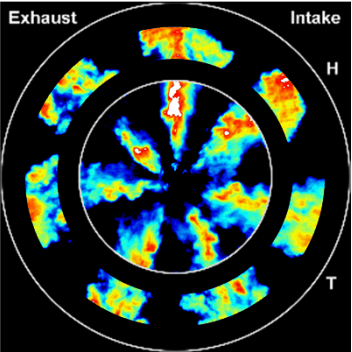
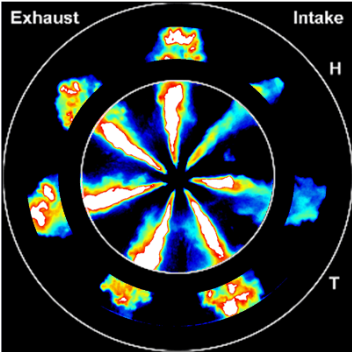
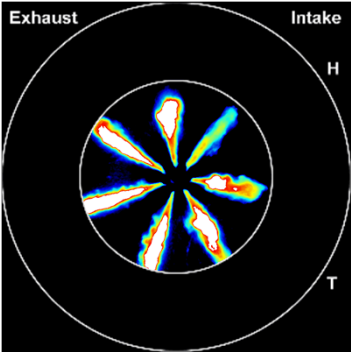
30 CAD BTDC

27.5 CAD BTDC

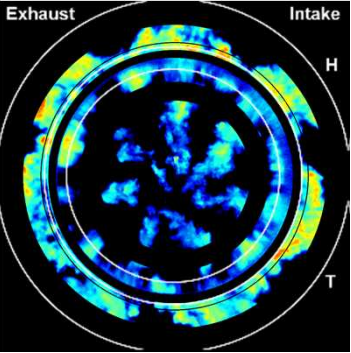
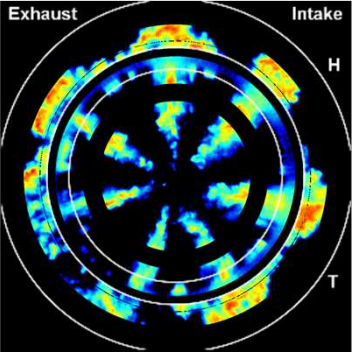
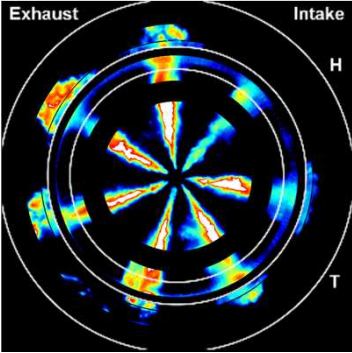
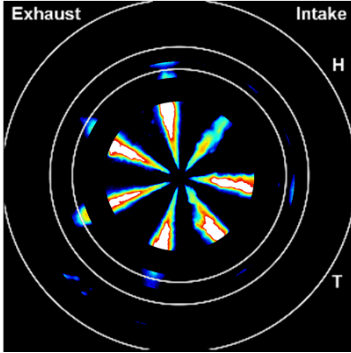
25 CAD BTDC

22.5 CAD BTDC

Conventional



Stepped-lip

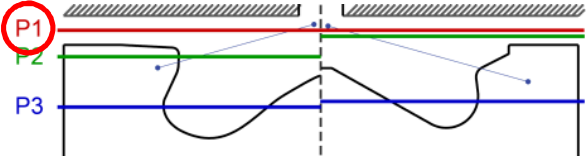


Fuel reaches bowl rim

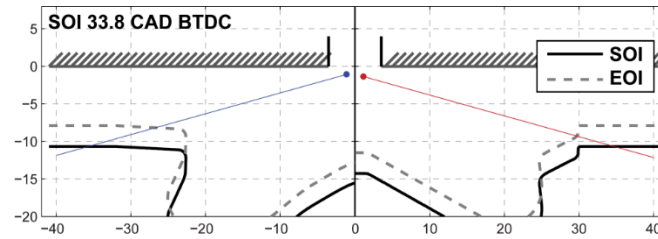
SL: Interaction with step

SL: Jets begin to interact with one another

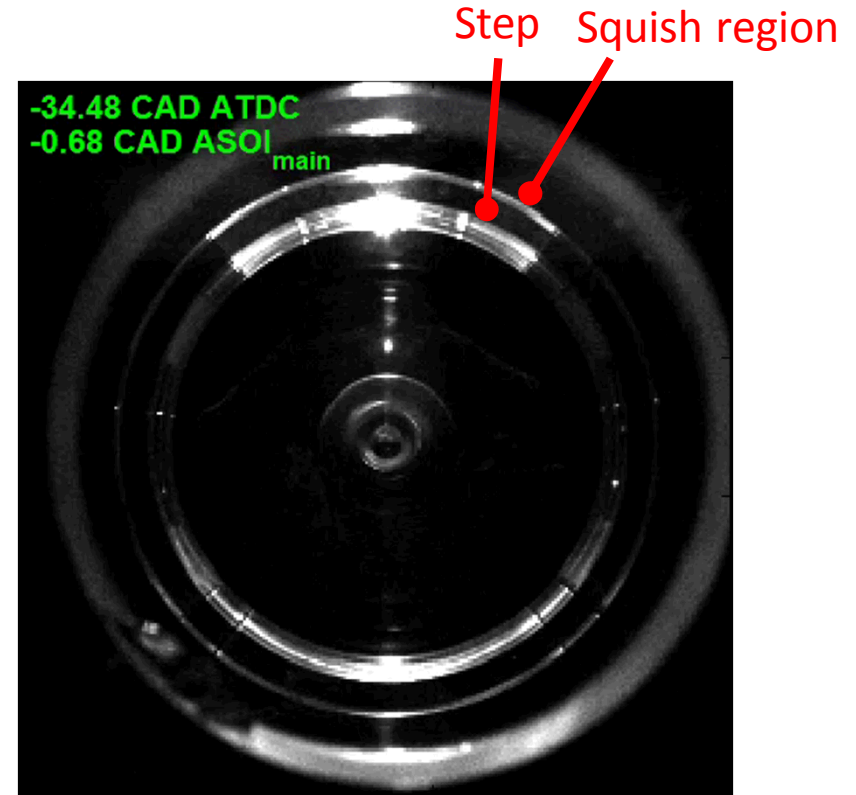
Conventional: jets not merged



Interaction between spray and piston step: SOI 33.8 CAD BTDC



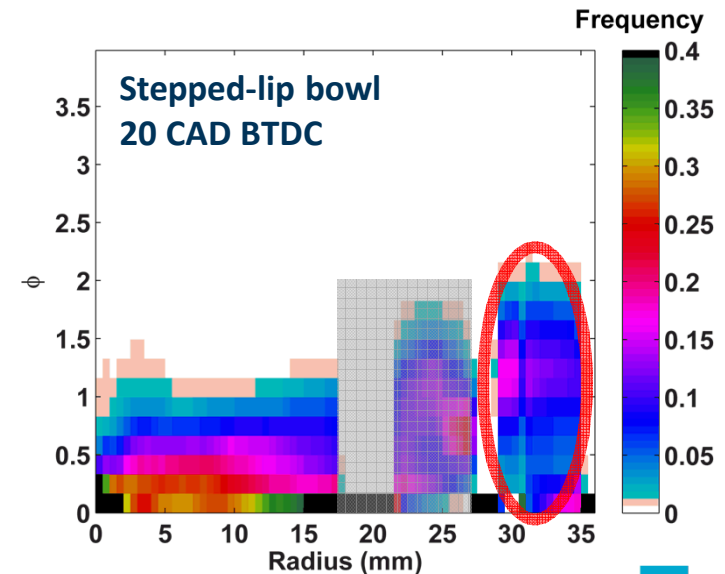
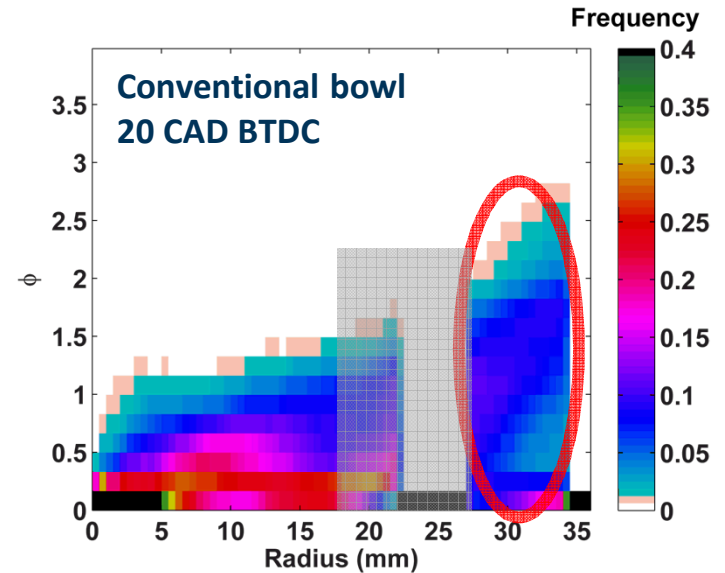
Conventional piston video coming soon...



Single cycle video; 40 kfps

Mixture distribution at start of low-temperature heat release; SOI 33.8 CAD BTDC

- Fuel-air equivalence ratios are binned as a function of radius for each of 50 measured cycles
- Equivalence ratio distributions are shown as a ϕ -R heat map at the start of low-temperature heat release
- Comparisons are not possible for radii between ~ 17 - 26 mm
- Differences near a radius of 30 mm
 - Narrower and richer fuel/air ratio distribution with the stepped-lip bowl
 - Interaction between the fuel jets and the step may act to reduce overmixing and thereby sources of lean CO emissions
- Differences in the bowl are more difficult to interpret
 - Leanest equivalence ratios are most susceptible to background interference



Interaction between spray and piston step: SOI 33.8 CAD BTDC

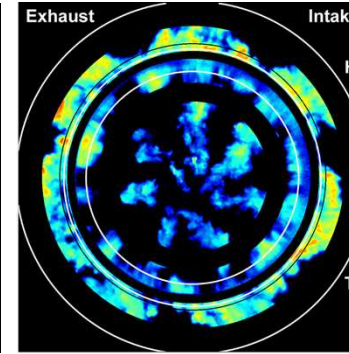
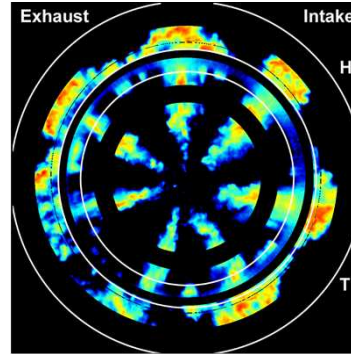
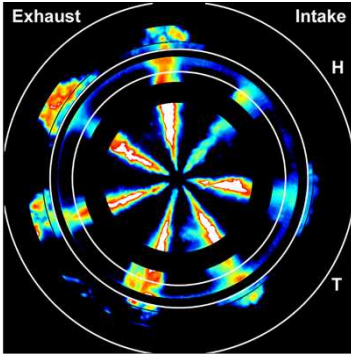
Single cycles

27.5 CAD BTDC

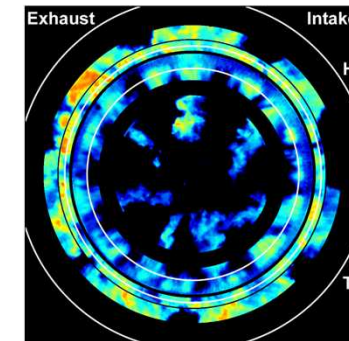
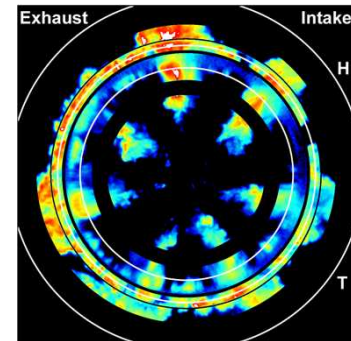
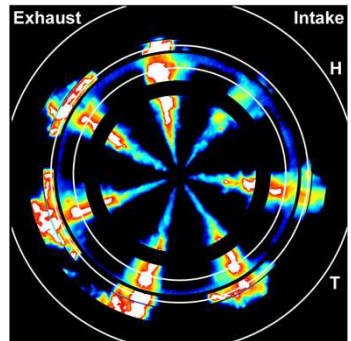
25.0 CAD BTDC

22.5 CAD BTDC

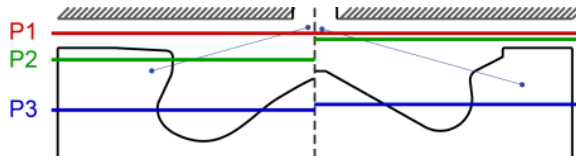
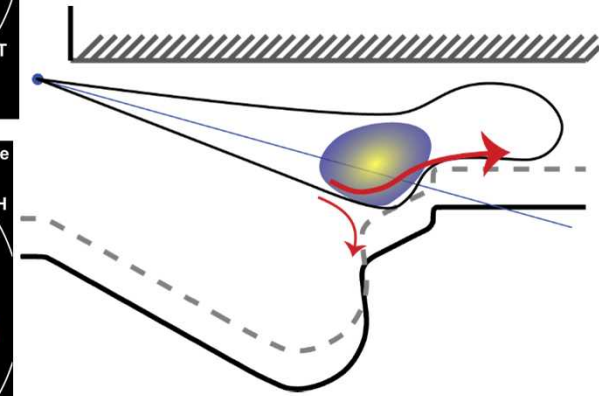
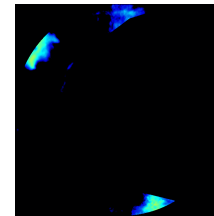
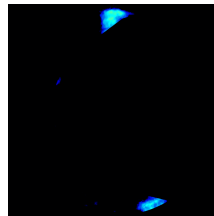
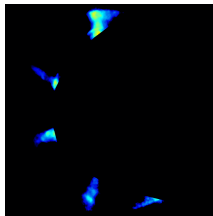
Plane 1



Plane 2



Plane 3



Interaction between spray and piston step: SOI 33.8 CAD BTDC

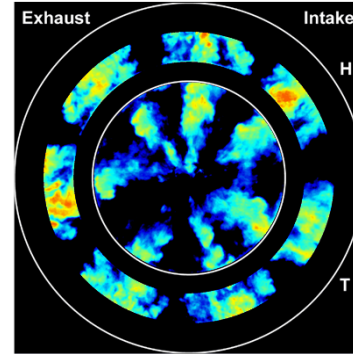
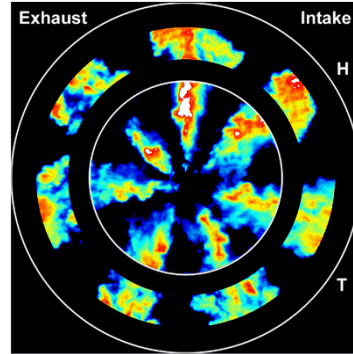
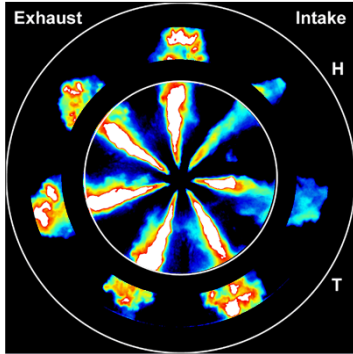
Single cycles

27.5 CAD BTDC

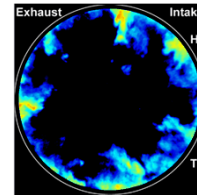
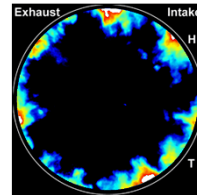
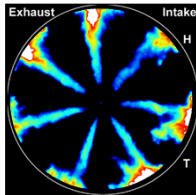
25.0 CAD BTDC

22.5 CAD BTDC

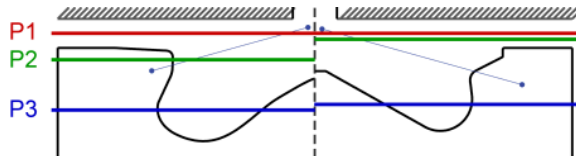
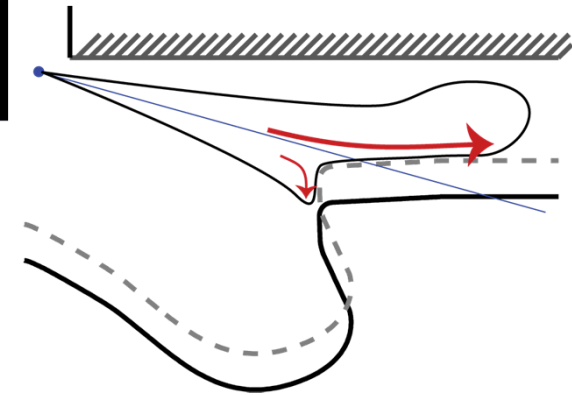
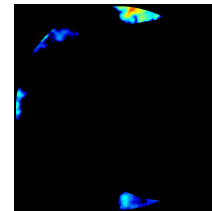
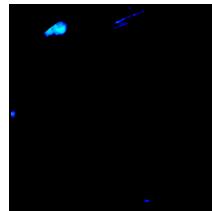
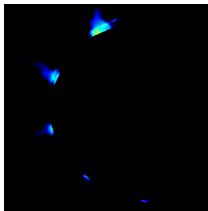
Plane 1



Plane 2



Plane 3



Recap: early injection PCCI (SOI 33.8 CAD BTDC)

- Spray targeting considerations: fuel jets impact primarily on squish regions of pistons and are deflected towards the liner
 - Significant penetration into the piston bowls is not expected
 - Stepped-lip piston: some interaction with the step is expected
- Jets interact with the step of the stepped-lip piston
 - Interactions between liquid fuel and the step appear stronger on the exhaust side
 - Fuel impinges on the step and is deflected tangentially; radial penetration is impeded; jet heads merge with the stepped-lip piston but not with the conventional bowl
 - Start of low-temperature heat release: richer conditions exist above the stepped-lip bowl near a radius of 30 mm than above the conventional bowl
 - This mechanism may hinder over mixing and improve combustion efficiency



Mixture formation processes compared: SOI 24.6 CAD BTDC (Plane 1)

Single cycles

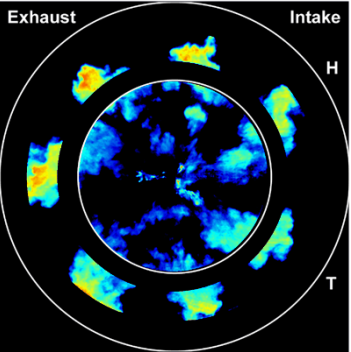
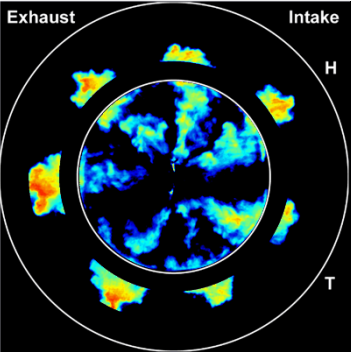
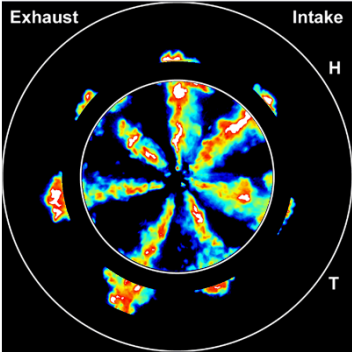
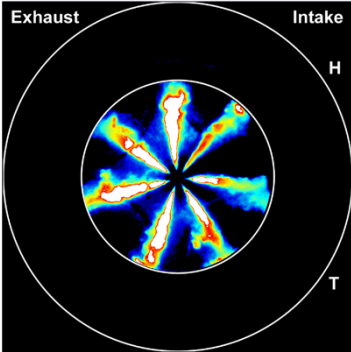
20 CAD BTDC

17.5 CAD BTDC

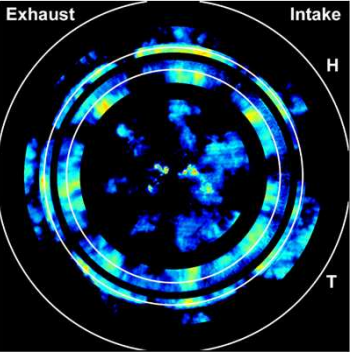
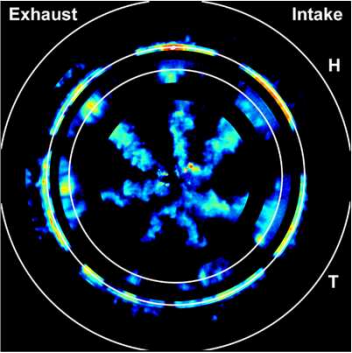
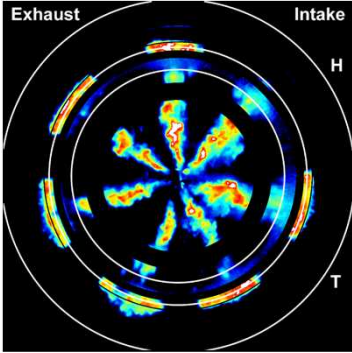
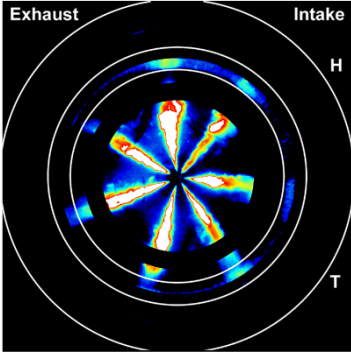
15 CAD BTDC

12.5 CAD BTDC

Conventional



Stepped-lip

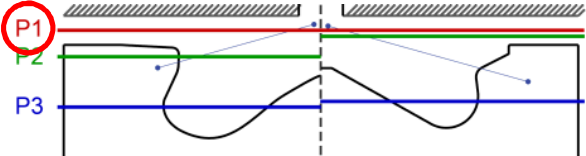


Fuel reaches bowl rim

SL: no penetration into squish region

Conventional: slow penetration into squish region

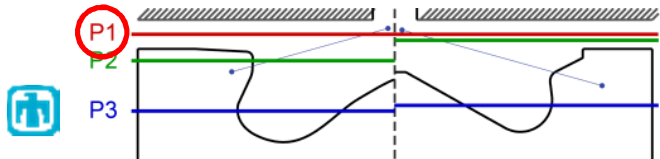
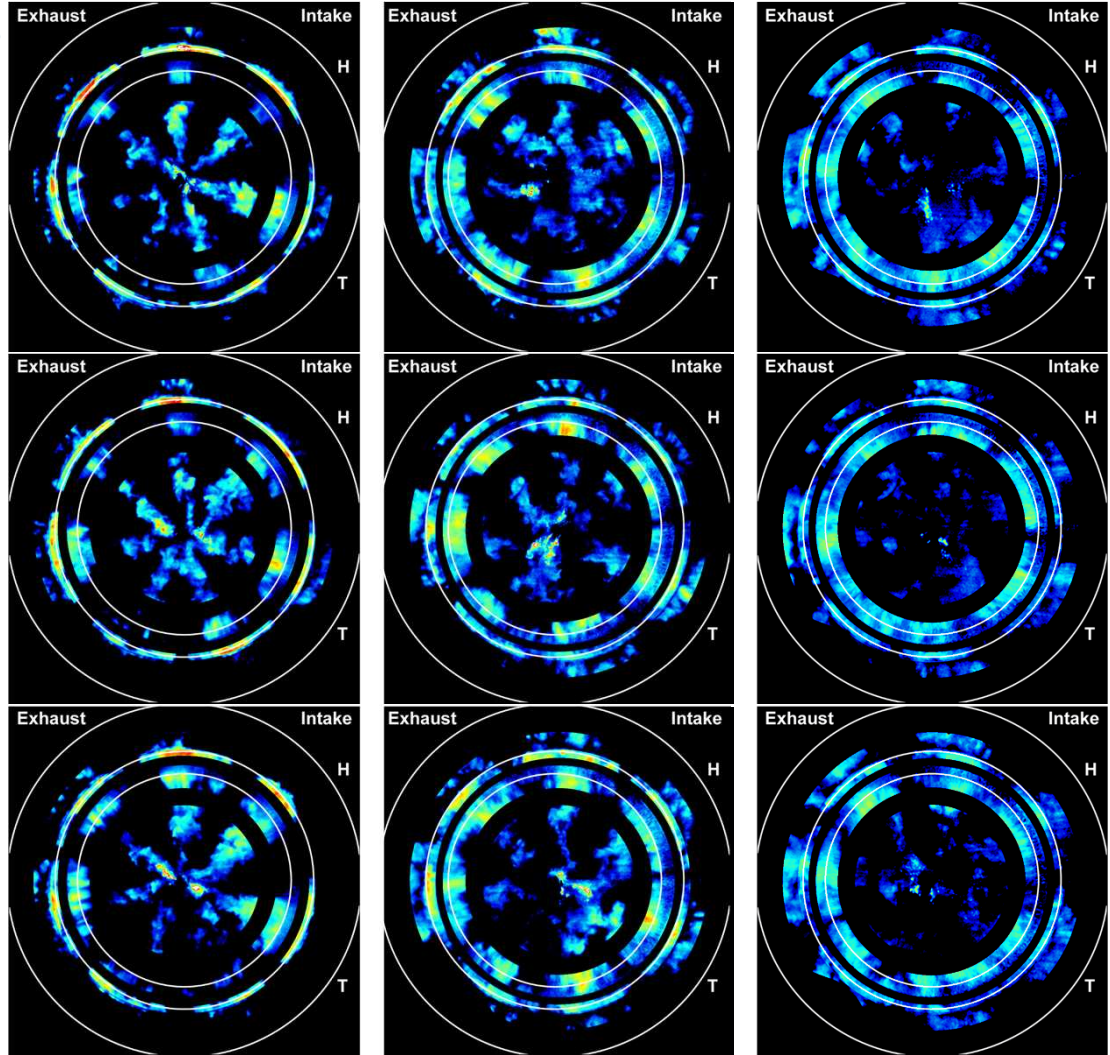
SL: fuel in outer portion of squish region



Interaction between spray and piston step: SOI 24.6 CAD BTDC

Single cycles
 15.0 CAD BTDC 12.5 CAD BTDC 10.0 CAD BTDC

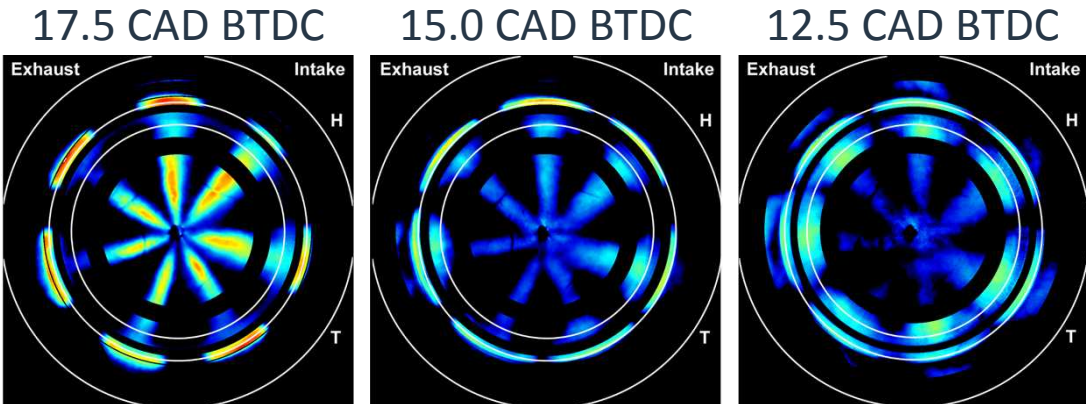
- 15.5 CAD BTDC: fuel first observed at outer edge of visible area
- 12.5 CAD BTDC: fuel in outer squish regions appears to have moved inward
- 10.0 CAD BTDC: inward progression has continued



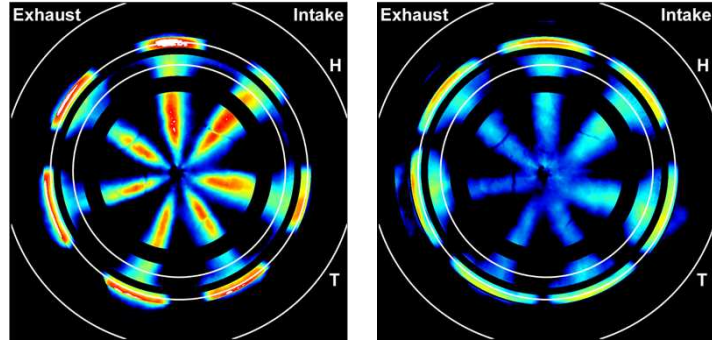
Mixture formation with the stepped-lip bowl: SOI 24.6 CAD BTDC

Ensemble average

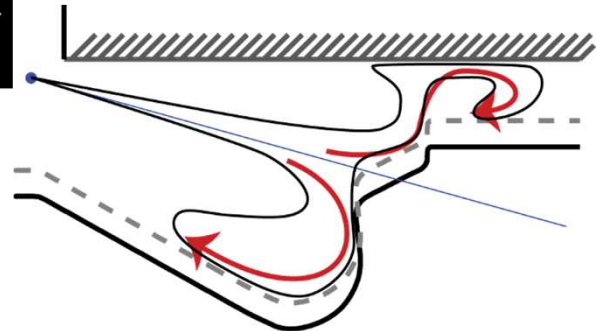
Plane 1



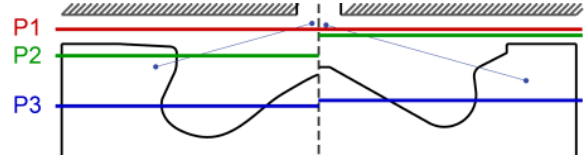
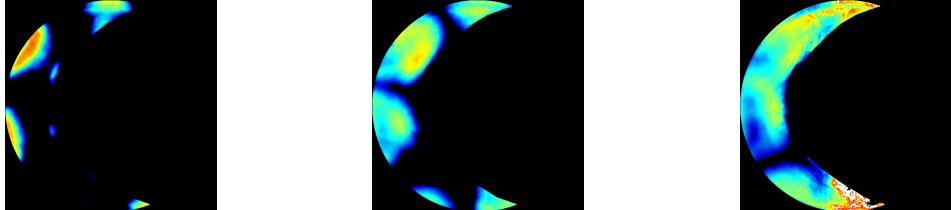
Plane 2



Data not available

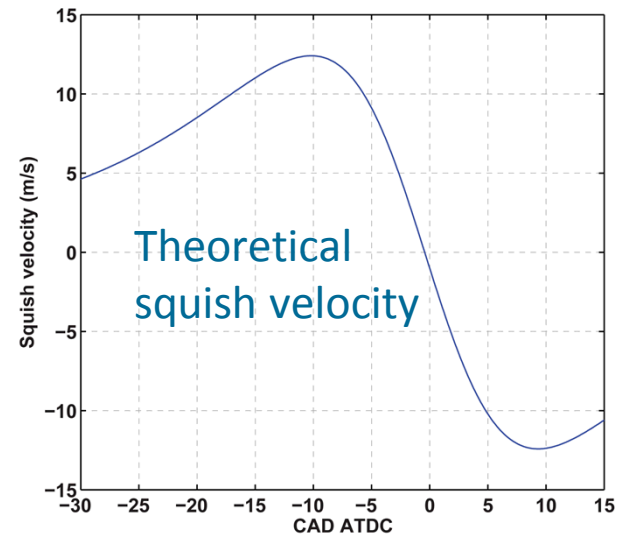
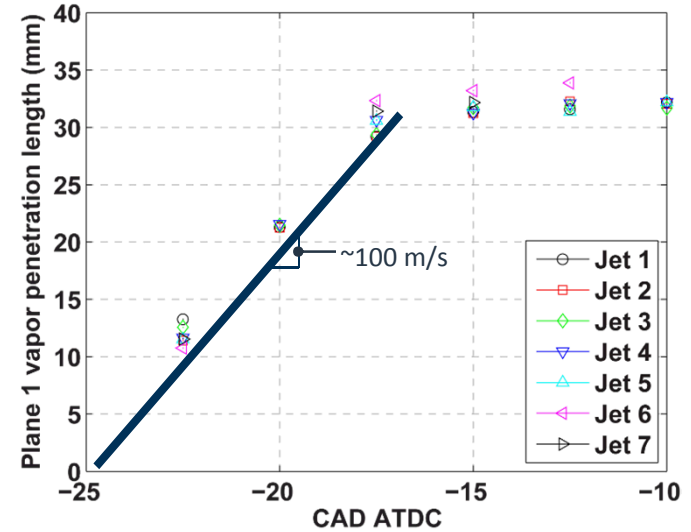


Plane 3



Vapor penetration: conventional bowl, SOI 24.6 CAD BTDC

- Vapor penetration levels off after 17.5 CAD BTDC
- Squish velocity and increasing density impede the outward penetration of the sprays
 - This may limit over mixing and maintain high combustion efficiency

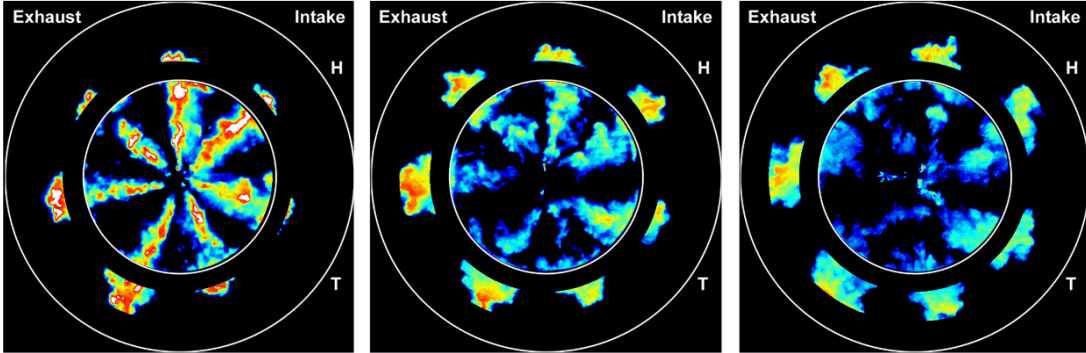


Mixture formation with the conventional bowl: SOI 24.6 CAD BTDC

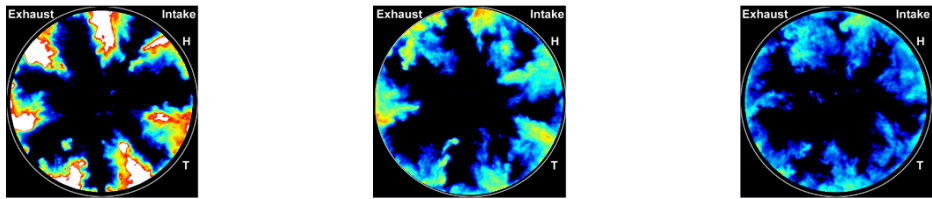
Single cycles

17.5 CAD BTDC 15.0 CAD BTDC 12.5 CAD BTDC

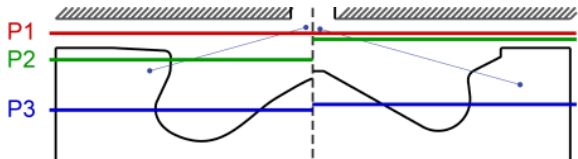
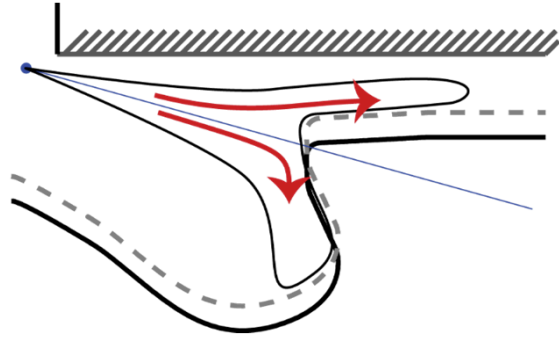
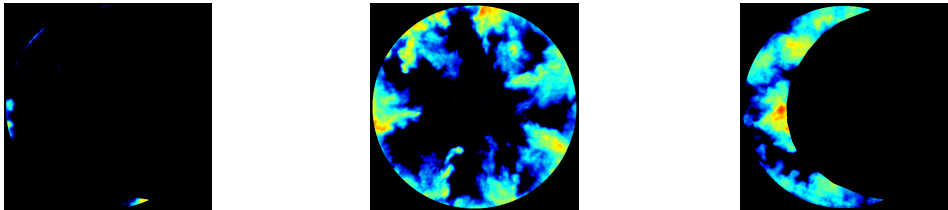
Plane 1



Plane 2

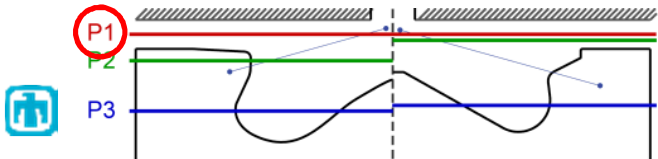
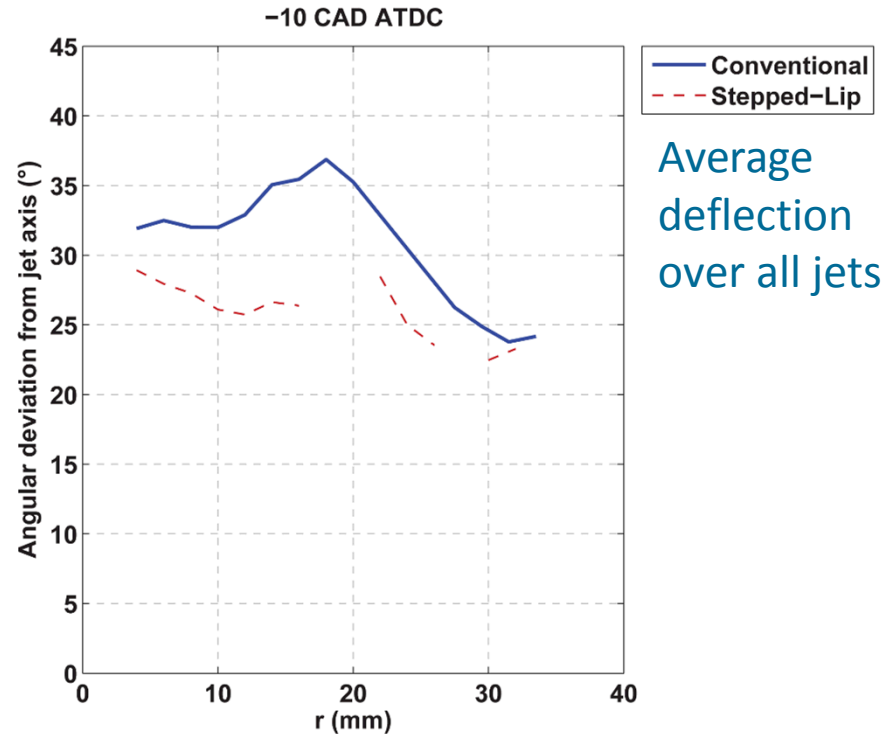


Plane 3



Spray-swirl interactions: SOI 24.6 CAD BTDC

- Average jet deflection by swirl is larger with the conventional piston after 17.5 CAD BTDC
- Swirl amplification is known to be greater with the conventional piston
- Angular momentum is redistributed differently by the squish flow for each piston geometry
- Impact on mixing and combustion is not yet well understood



Recap: intermediate injection PCCI (SOI 24.6 CAD BTDC)

- Spray targeting considerations: fuel jets impact primarily on bowl rim (conventional bowl) or step region (stepped-lip bowl)
- Conventional bowl
 - Fuel jets penetrate into the squish region, but progress is likely hindered by squish flow; jets are also directed downward into the bowl
 - The jets are more affected by swirl flow than for the stepped-lip bowl
 - More fuel is present in the bowl than for the early injection case
- Stepped-lip bowl
 - Penetration into squish region is impeded as fuel is redirected upward at the step's outer rim
 - Mixture moving inward from the outer portion of the squish region is consistent with a toroidal vortex that forms as a result of interactions with the piston bowl, cylinder head, and possibly the bore wall
 - This is evidence of a mixing mechanism that does not exist with the conventional piston bowl
 - Jets do not merge above the bowl rim as with the early injection case
 - More fuel is present in the bowl than for the early injection case
- Despite differences in mixture preparation, both pistons perform similarly at this condition



Mixture formation processes compared: SOI 9.6 CAD BTDC (Plane 1)

Single cycles

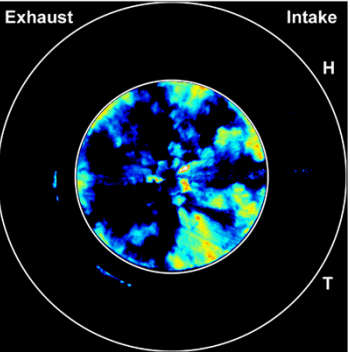
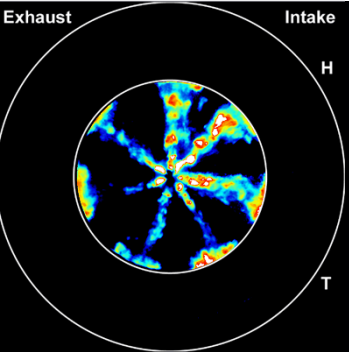
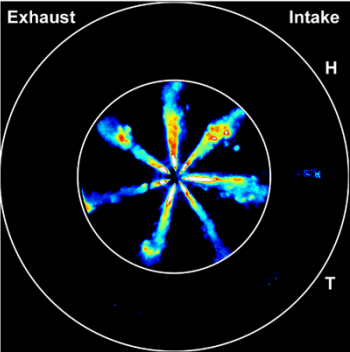
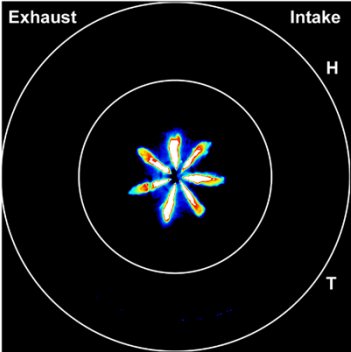
7.5 CAD BTDC

5.0 CAD BTDC

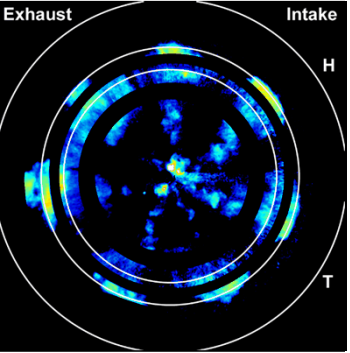
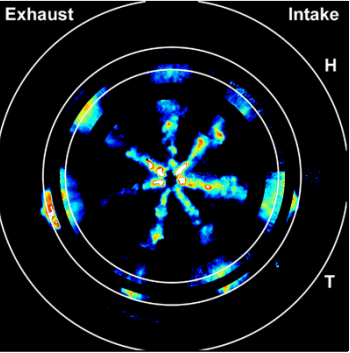
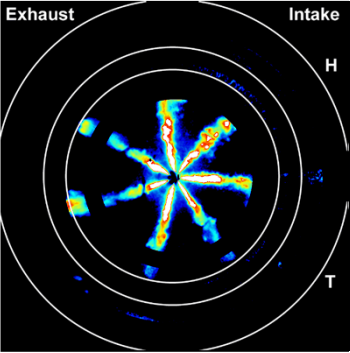
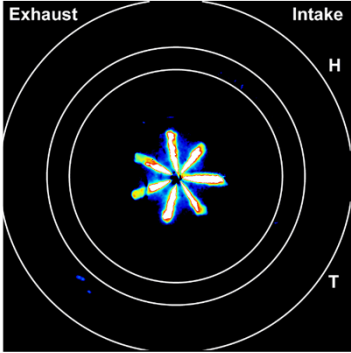
2.5 CAD BTDC

TDC

Conventional



Stepped-lip

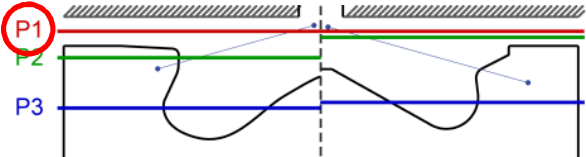


Shortly after SOI

Jets reach bowl rims

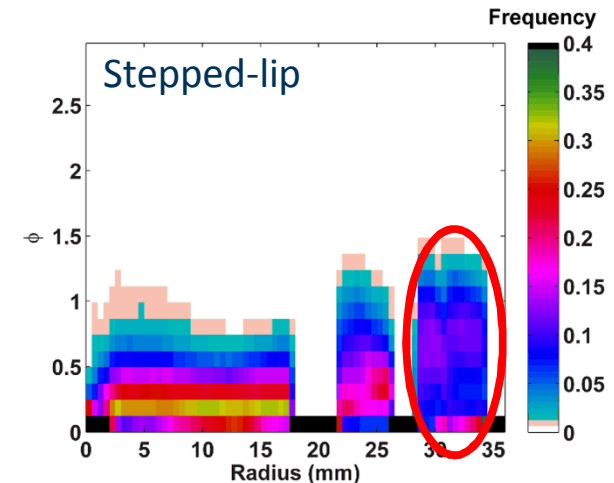
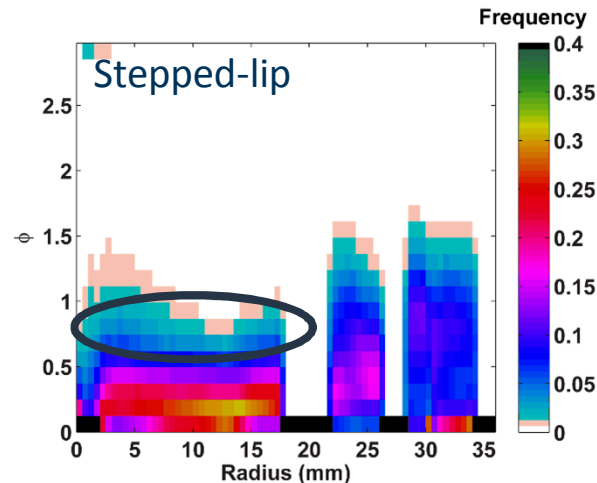
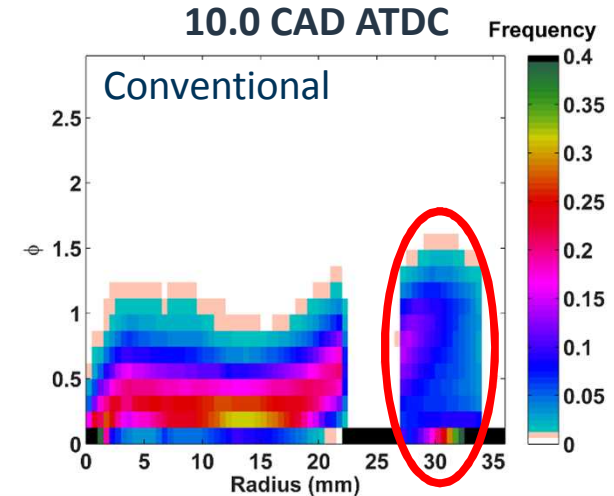
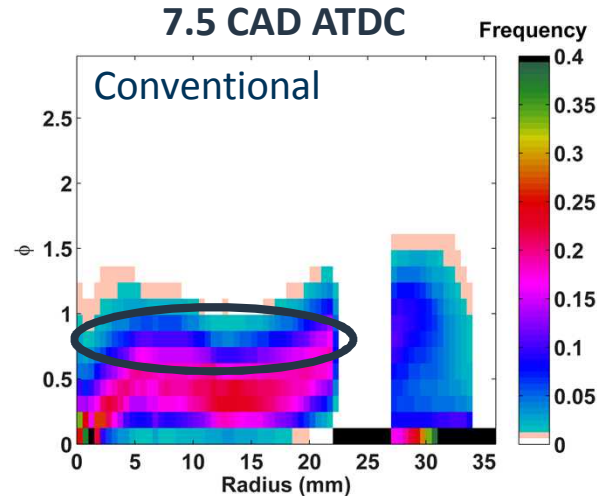
SL: small amounts of fuel above the step

Near start of high-temperature heat release; little mixture in squish region

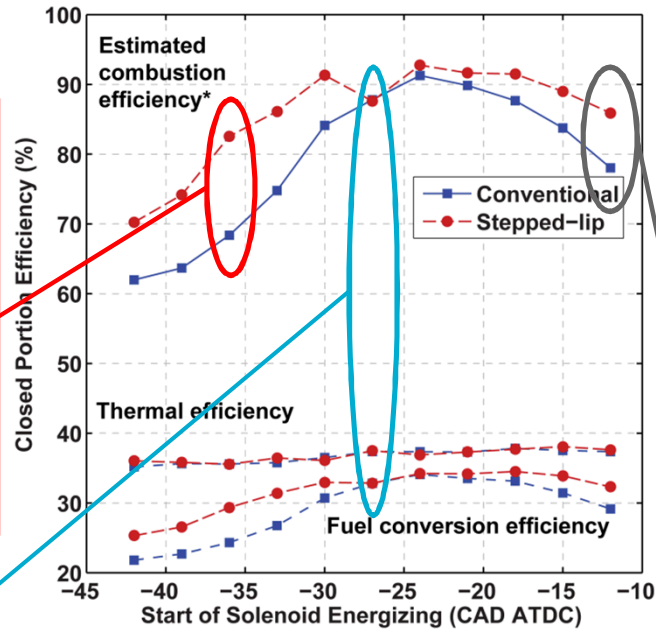


Behavior after TDC: SOI 9.6 CAD BTDC (Plane 1)

- Mixtures above the conventional bowl are richer than above the stepped-lip bowl (black ellipses)
- Richer mixture penetrates further into the squish region with the stepped-lip bowl (red ellipses)
 - Counterintuitive based on how we expect the reverse squish flow to behave
- Richer mixtures in the squish region could better promote CO oxidation and complete combustion with the stepped-lip bowl
- Further research is needed



Summary: theories about mixture formation and combustion efficiency for early-injection PCCI operation



- Piston step acts to spread jets tangentially
- Richer, narrower fuel/air mixture distribution near the stepped-lip bowl rim
- Likely cause of improved combustion efficiency

- Richer mixtures observed in the squish region with the stepped-lip bowl after TDC
- This could promote complete combustion, but the mechanism is unclear

- Conventional bowl: strong squish flow impedes fuel jet penetration into squish region
- Stepped-lip bowl: formation of a toroidal vortex: a mixing mechanism not observed with the conventional bowl
- Comparable performance despite mixture formation differences

*estimated from energy balance



Ongoing studies

- High-speed Mie scattering: characterizing liquid lengths, understanding interactions with piston geometry
- Conventional diesel combustion: desire to understand late-cycle turbulent mixing behavior
- High-speed natural luminosity imaging
 - Where and when is rich, sooty combustion occurring?
 - Can we see the difference in flow between the combustion regions?
 - Combustion imaging velocimetry
- Comparison of D

-7.83 CAD ATDC
-6.30 CAD ASOI_{main}

Thank you for your attention

Questions?

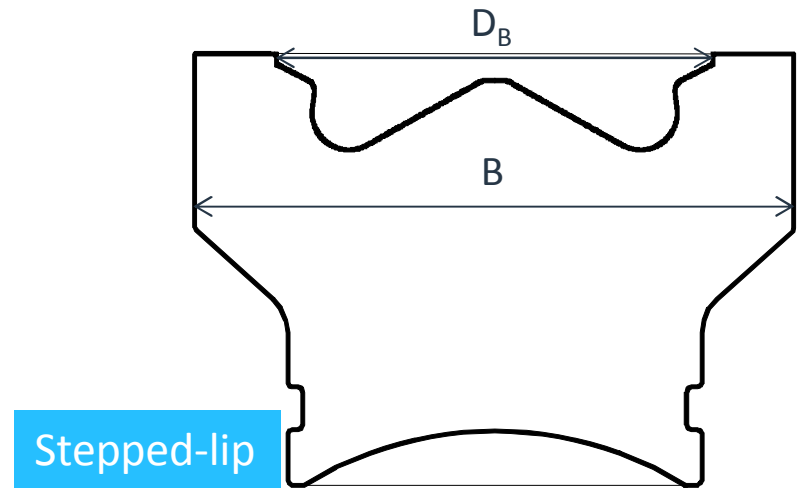
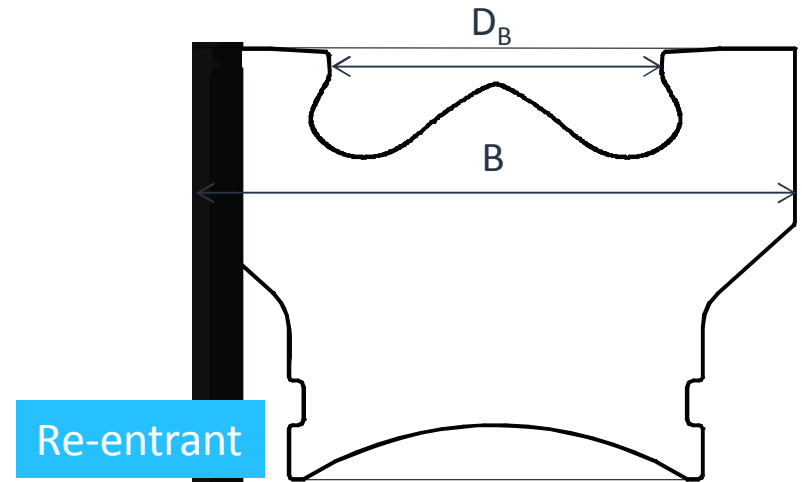


Backup slides



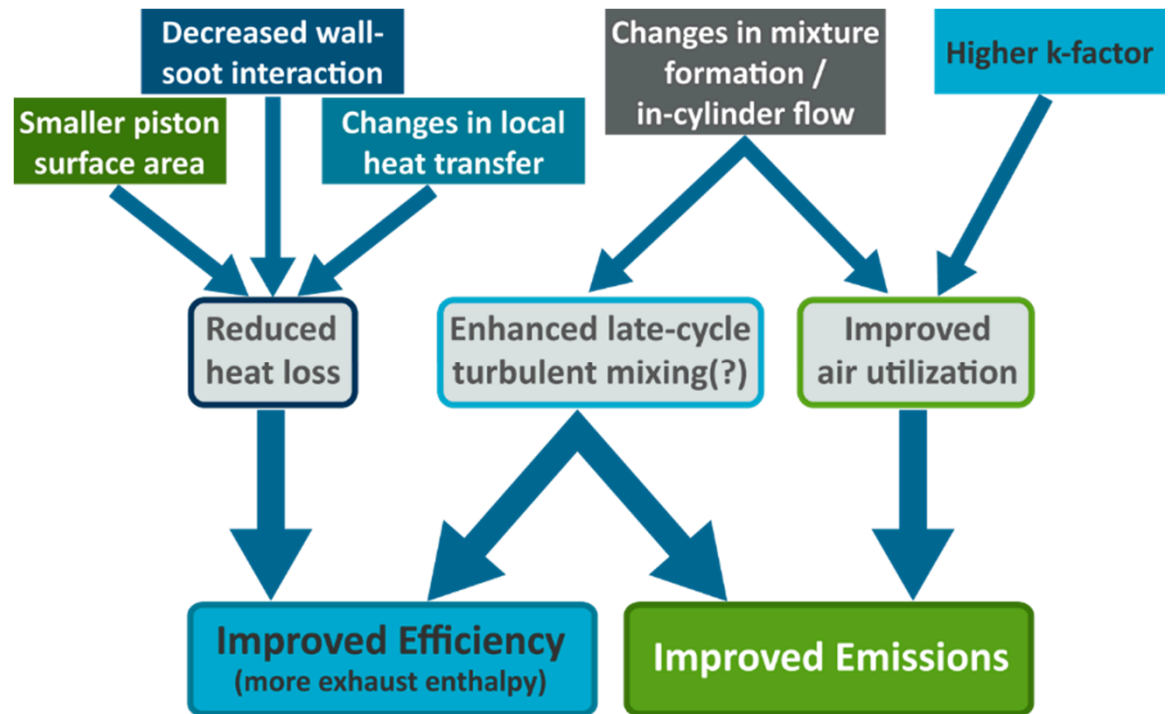
SNL optical piston bowl geometries

- Two fused silica pistons with identical:
 - Bowl volume = 0.028 L
 - Squish height = 1.35 mm
 - Compression ratio = 15.8:1
 - No valve cut-outs
- Different bowl geometries result in different bowl-to-bore ratios:
 - Re-entrant: $D_B/B = 0.55$
 - Stepped-lip: $D_B/B = 0.73$
- Surface area of stepped-lip bowl is ~10% less than for the conventional bowl
- Injector timing is set to be used to reduce emissions from OHC pistons



Understanding efficiency and emissions advantages with stepped-lip pistons

- The mechanisms responsible for these advantages are not well documented
- Limitations and drawbacks of stepped-lip pistons are not well understood
 - Spray targeting sensitivity
 - High speeds/loads
 - Others?
- Ultimate goal
 - Understand how piston geometry can be optimized to improve efficiency and reduce emissions



RE vs. SL Flow comparison: $R_s=2.2$, $CR=16.1$, $CA = -40$ aTDC

Y=0

reEntrant

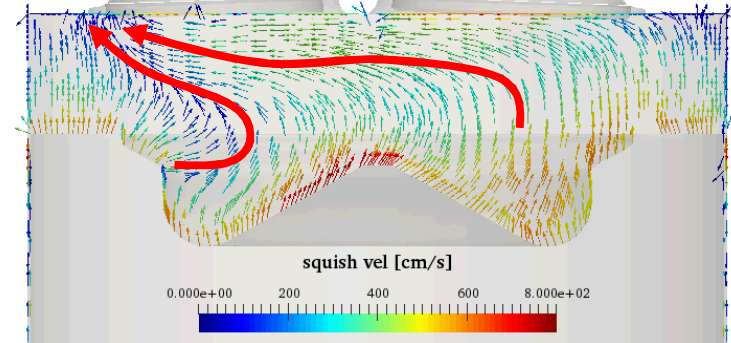
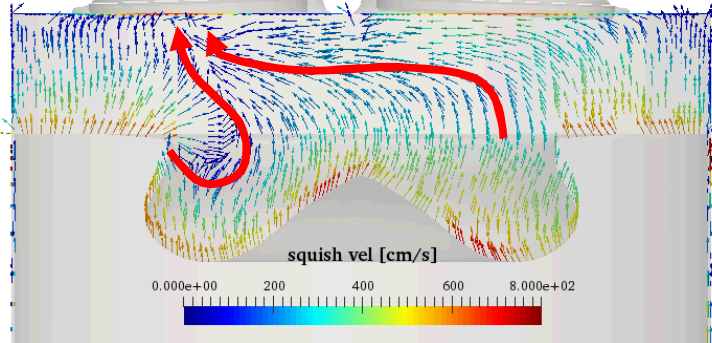
steppedLip

exhaust intake

exhaust intake

CA = 1400.6 aTDC

CA = 1400.6 aTDC



squish motion appears in reEntrant, not yet in steppedLip

First stub of squish behavior appears at the step's rim