



A comparison of fuel/air mixing behavior between PLIF experiments (SNL) and simulation (CSI)

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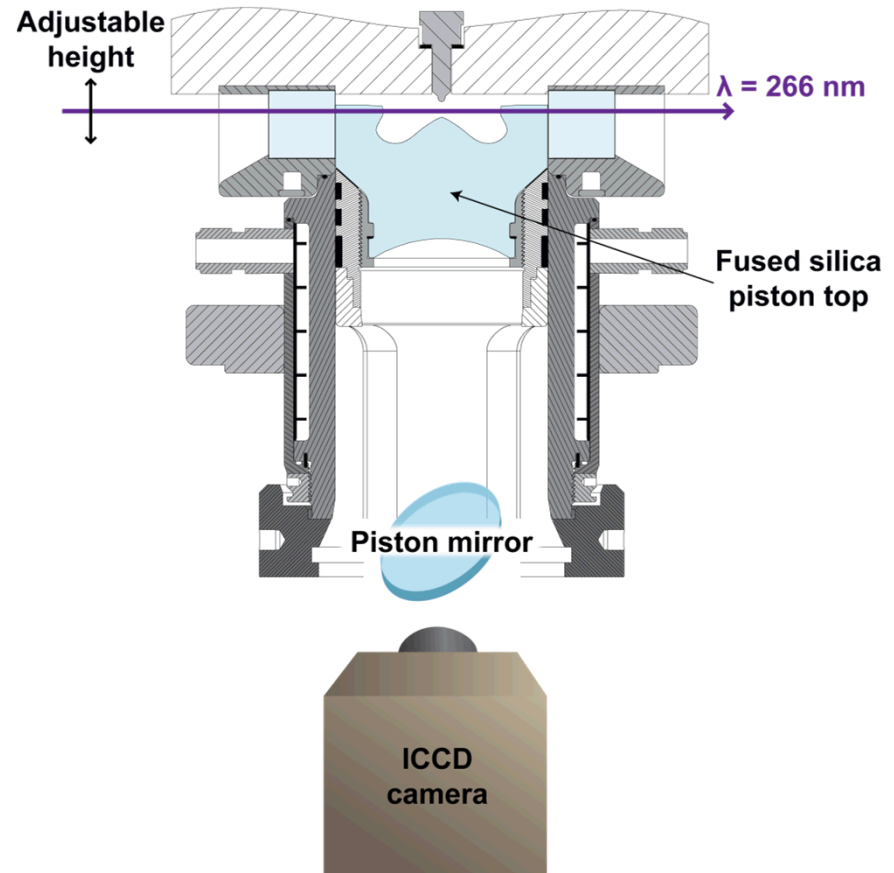
Processing PLIF and CFD data to make comparisons of the mixture formation process

- Setup
 - Engine operation
 - Experimental uncertainties
 - CFD models used
- Metrics for comparison
 - Spreading angle and penetration length quantification
 - Spray structure: deflection of spray by swirl
 - Fuel concentration gradients
- Comparisons
- Summary



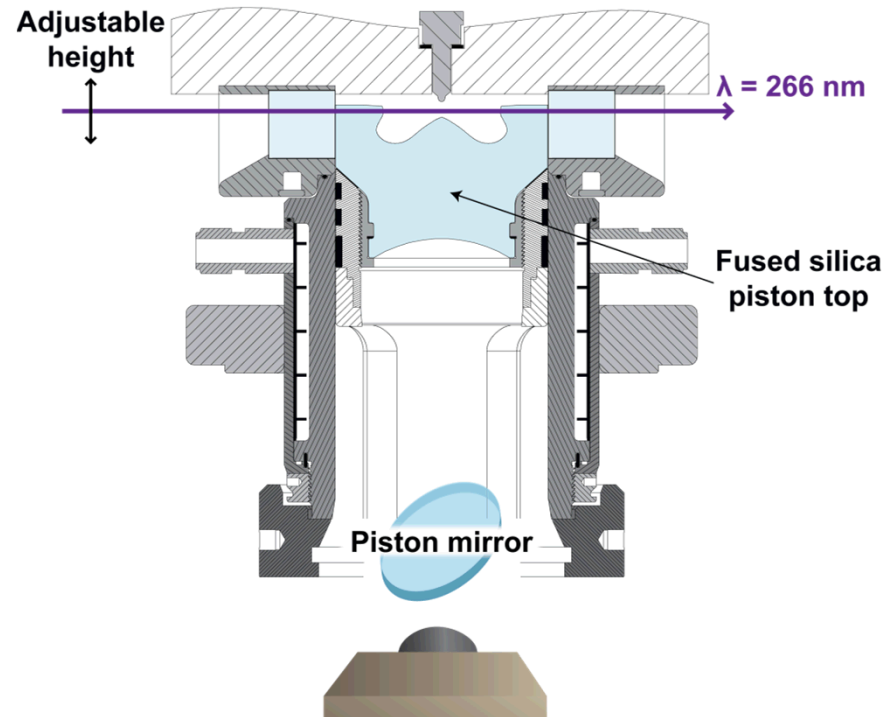
Setup PLIF Experiments in the SNL light-duty single-cylinder optical engine

- Combustion chamber
 - Bowl volume: 0.028 L
 - Squish height: 1.35 mm
 - Geometric compression ratio: 15.8
 - Flat-top piston: 100% wetted surfaces
- Engine operating conditions
 - 1500 rpm, 10 bar IMEPg, Fuel = 2
 - Ignition timing = 10 deg BTDC, SFE 27, 100 BTDC
 - Non-combusting: 100% N₂
- Fuel: 42 wt% n-pentane + 30 wt% i-pentane + 15 wt% n-hexane + 13 wt% 1-methylnaphthalene



Setup PLIF Experiments in the SNL light-duty single-cylinder optical engine

- Illumination source
 - Nd:YAG, 4th harmonic (266 nm)
 - ~ 30 mJ/pulse (relative intensity measured for each pulse)
 - Laser sheet thickness = 1 mm
- Imaging
 - PI-MAX camera (gate = 15 ns)
 - 100 mm lens (f/16)
 - 295 long pass filter



Bore x stroke	82 mm x 90.4 mm
Compression ratio	15.8:1
Valves	4
Injector type	Solenoid
Holes	7 x 139 μm
ks	1.5/86
Included Angle	149°

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SB1

There are many other sources of uncertainty, such as signal trapping, background reflections of fluorescence, fluorescence from laser light reflected off of chamber surfaces, camera nonlinearity at high intensities, etc.

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Setup: CFD spray models

- Blob injection model
 - Initial drop sizes equal to effective nozzle diameter
- Rebound model
 - No wall film formation
- O'Rourke Turbulent dispersion model
- Dynamic drop drag model
- KH-RT modified breakup model
- NTC collision model
- Post model for collision outcomes
- Frossling correlation to model spray evaporation

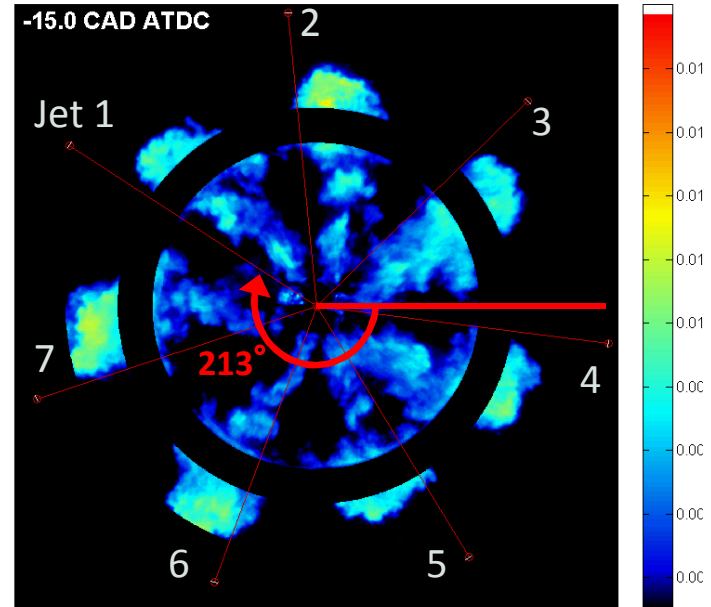


Injector clocking

- 7 holes with equal azimuthal distribution
- Jet 1: axis oriented 213° from horizontal axes
 - Angle estimated from initial jet penetration

Exhaust

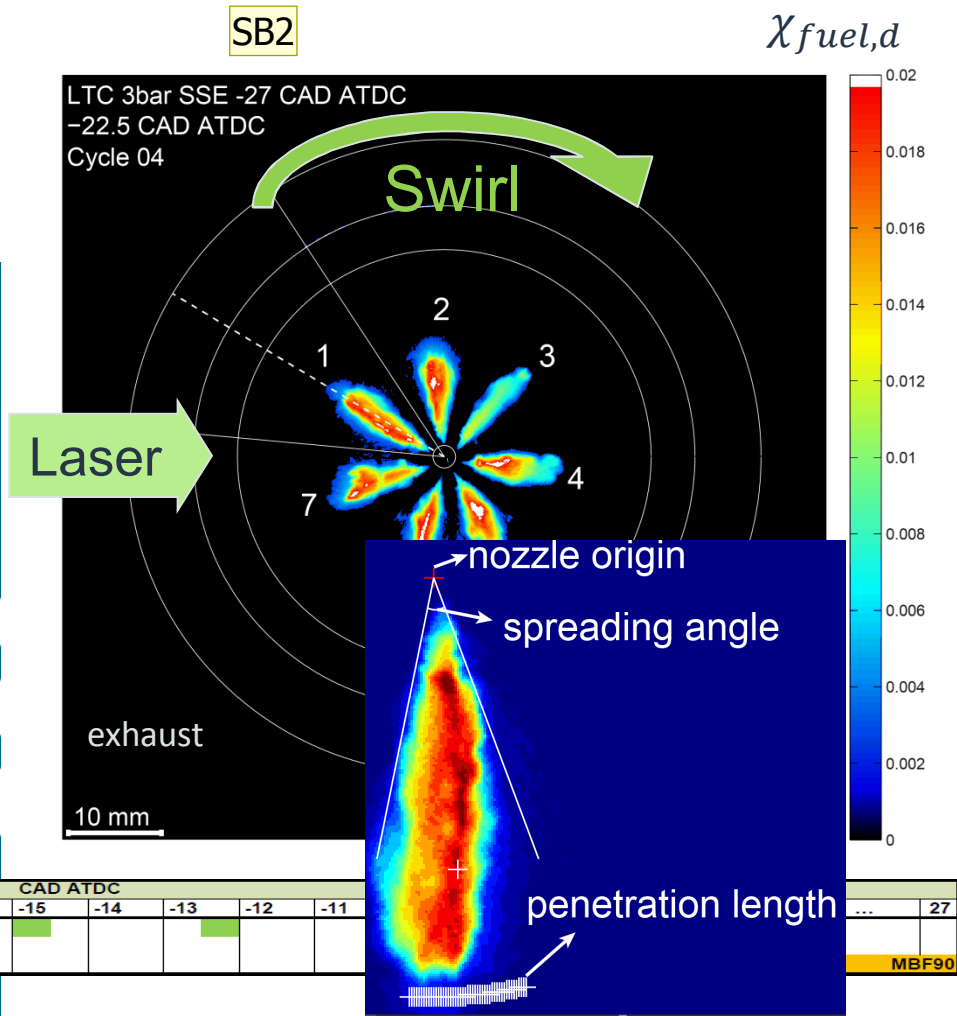
Intake



Spreading angle and penetration length quantification

- Background subtraction:
 - For -22.5 and -20° aTDC, $\chi_{fuel,d} < 0.0025$ was removed
 - For $CAD \geq -17.5^\circ$ aTDC, $\chi_{fuel,d} < 0.0010$ was removed
- Spreading angle and penetration length calculation using ECM parameters, SAE Paper 960034).
 - Since measurements are not taken on the jet axis, the spreading angle is the projection of the spray onto the image plane.

Calculations were performed for the spray plume from individual frames (2)



	-26	-25	-24	-23	-22	-21	-20	-19	-18	-17	-16	-15	-14	-13	-12	-11	...	27
Images																		
Injection		SOI																
Combustion																EOI		

MBF90

Slide 8

SB2

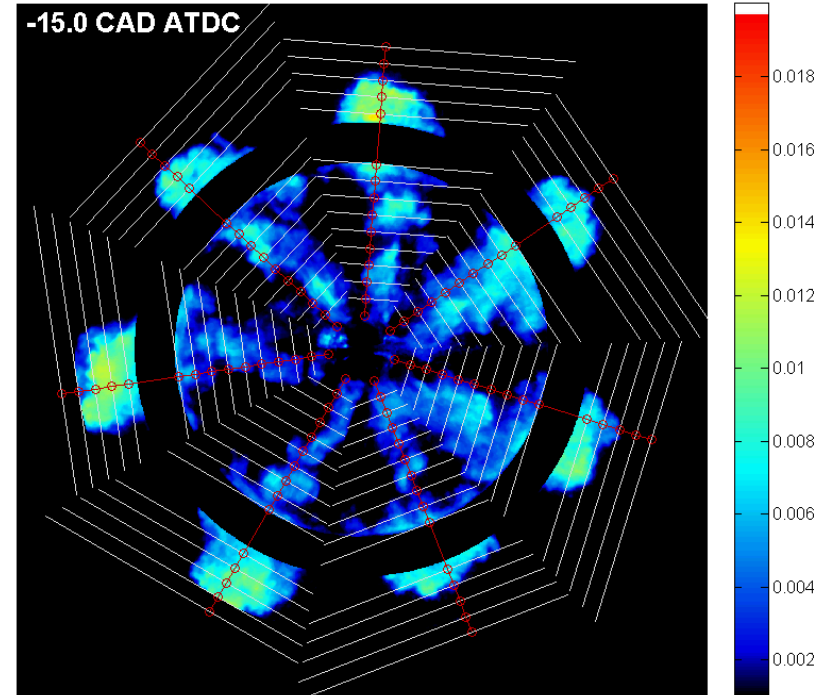
When publishing this work, it will be important to cite work that demonstrates how fluorescence-based and schlieren measurements compare, as Siebers developed his methodology for schlieren images.

I believe Ethan's paper shows a comparison of these things.

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Spray structure characterization

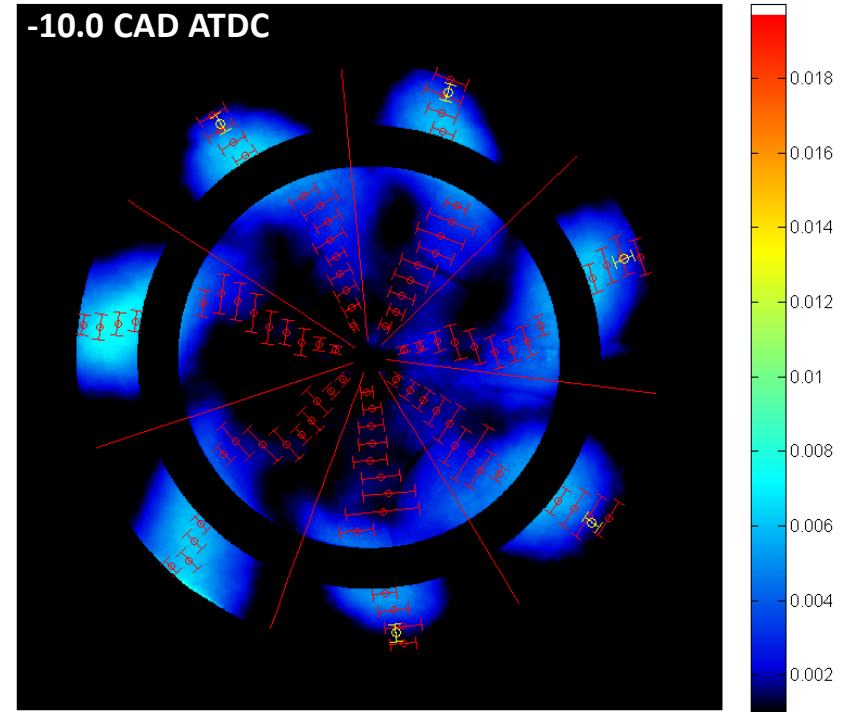
- Fuel concentration-weighted centroid evaluated along multiple profiles (shown as white lines) across each jet
- Ensemble average provides a simplified representation of jet structure. Insight into how jets are affected by swirl
- The procedure is used to extract experimental data



Characterization of jet deflection

- Ensemble averaged image shown with red lines indicating jet axes
- Red bars show ensemble averaged fuel concentration-weighted centroids for each jet
- Jet deflection increases with injection angle; detection capability (for early injection) diminishes
- Swirl intensity more intense in the bowl than the region
- Jet size each jet by its angular deviation from the jet axis and radius of radius of each imaged cross-section

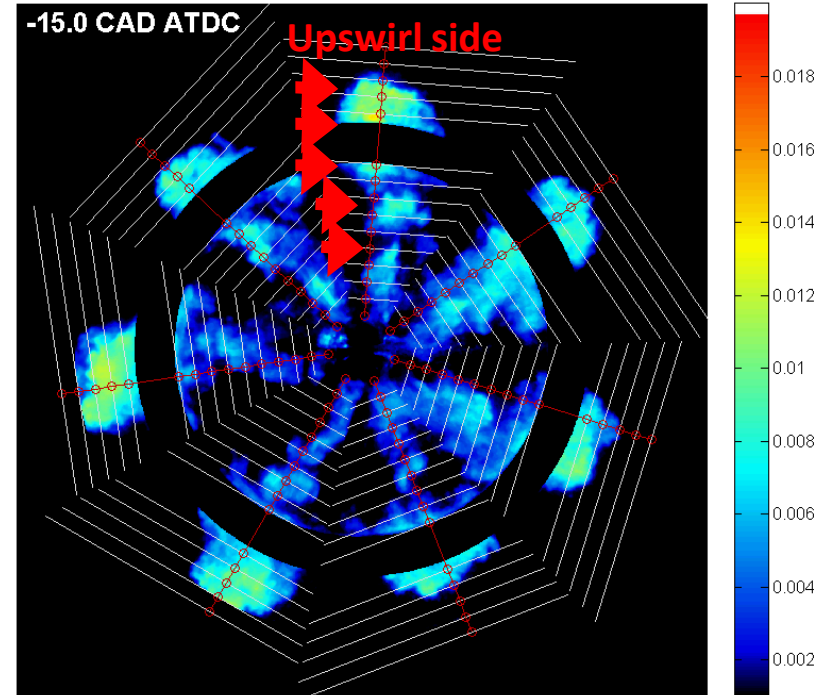
Fuel mole frac.



Characterizing fuel concentration gradients

- Fuel concentration data have been collected along the white profiles shown on the right
- For each cycle, jet, and nozzle position, the gradient on the upswirl side of the jet is estimated
 - see right for definition of upswirl

Fuel concentration gradients are computed based on the location of the jet head



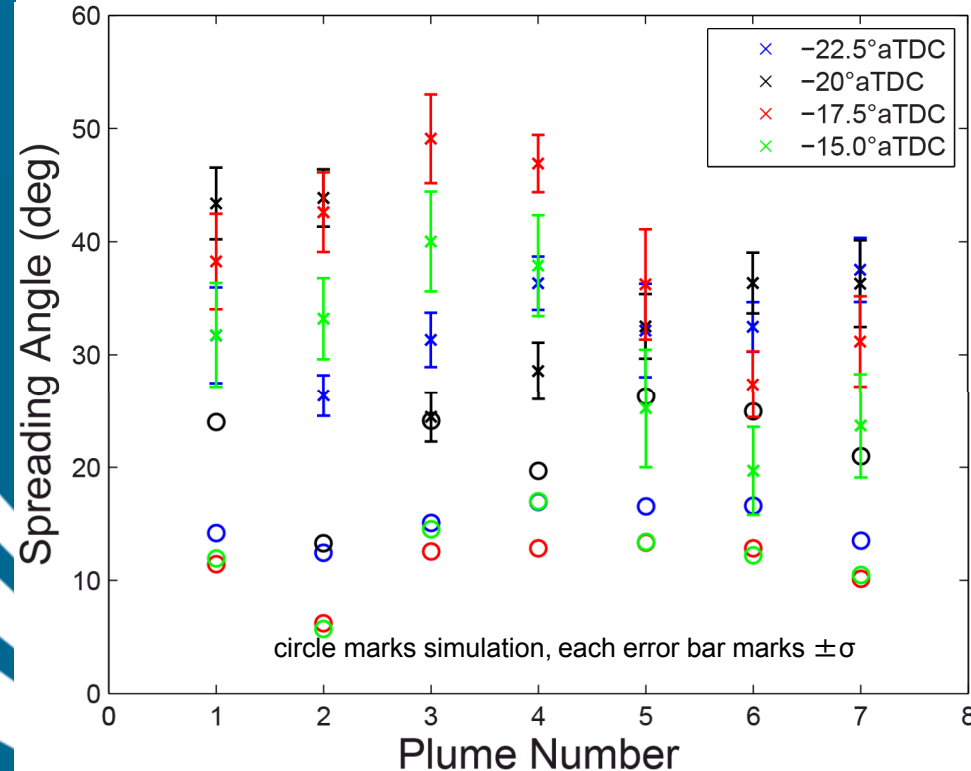
Results



Spreading angle comparison

- Simulated spreading angles are typically smaller than experimentally determined angles
- The experimentally determined spreading angle changes systematically with crank angle
 - Can this be replicated in the simulation?
- Potential improvements
 - Interrogation depth (shear thickness) for output
 - Undersampling to finite

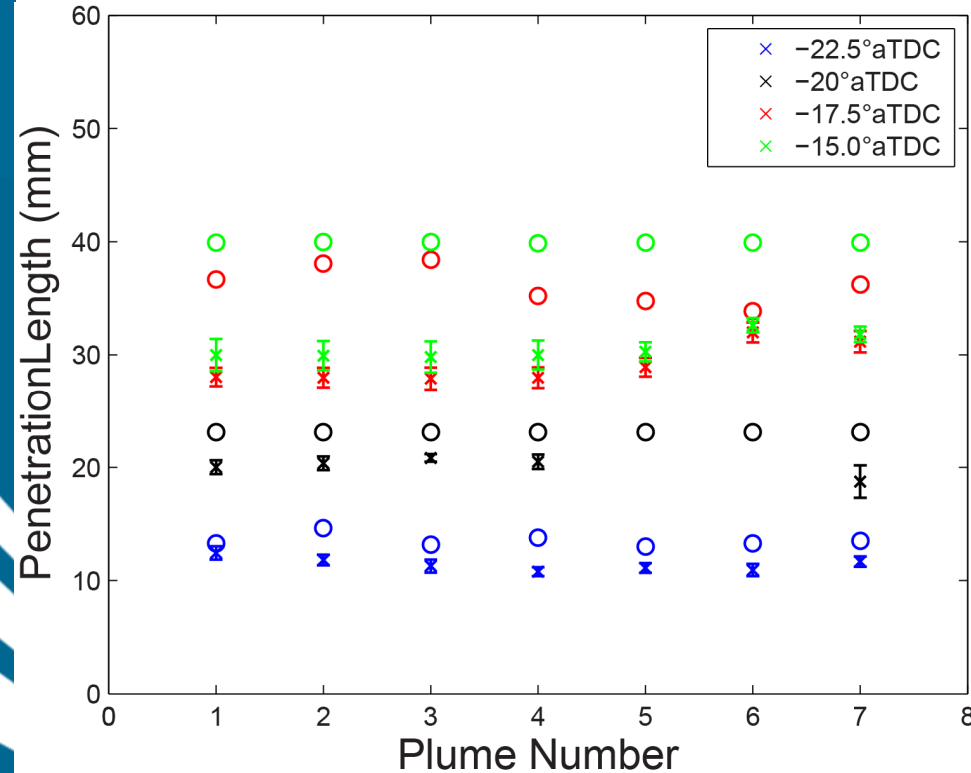
Cone angle initialized as 30deg



Penetration length comparison

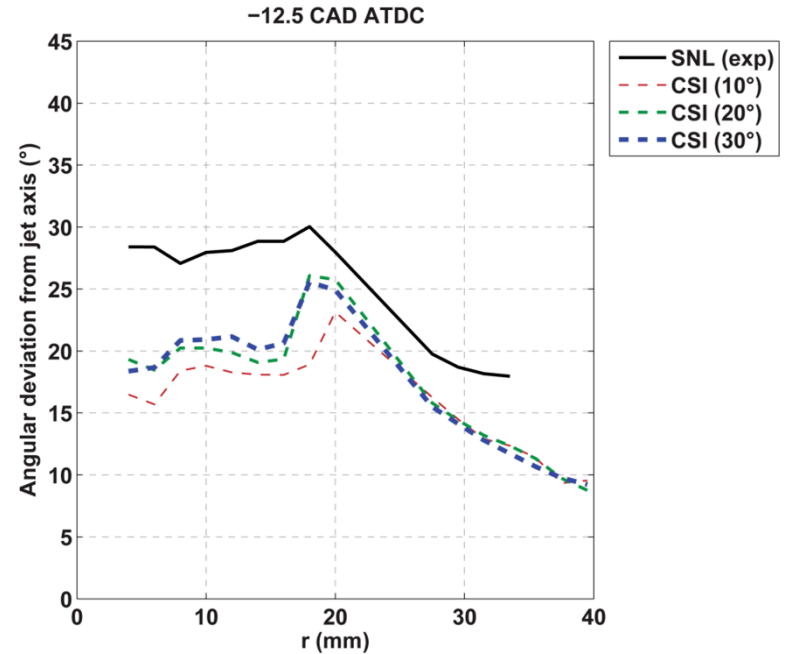
- Simulation over predicts spray penetration regardless of spray angle
 - Larger spreading angles may provide better results
- Over prediction is relatively small early in the injection event (-2.5 aTDC) but increases with time
 - At 4 aTDC, simulated spray reaches both nozzle exit (experiments)
 - After EOI, higher spray penetration is observed in experiments
 - Simulation shows symmetric flow
 - Hole diameter

Cone angle initialized as 10deg



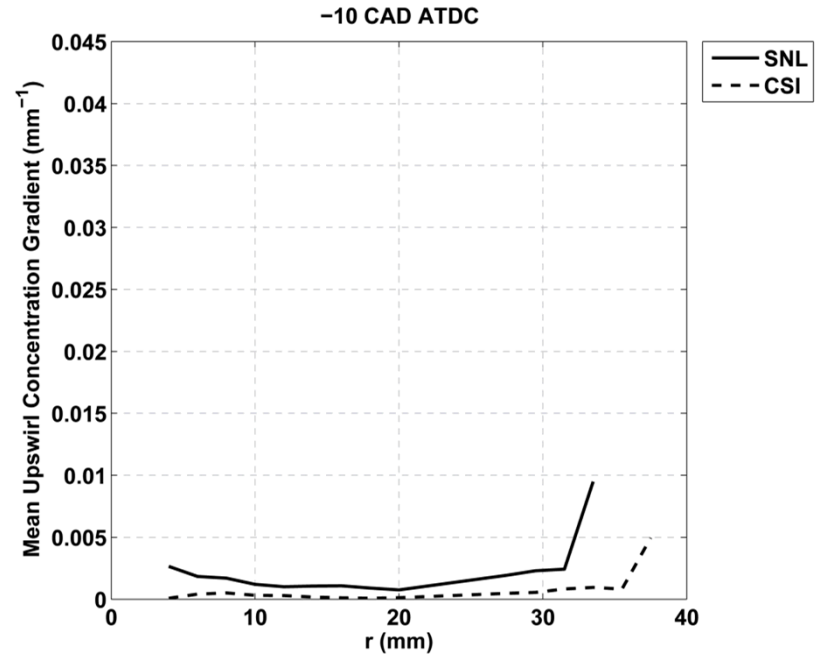
Spray structure: jet-averaged angular deviation from jet axes

- Squish region: deviation under-predicted after 15 CAD ATDC
 - At 12.5 CAD BTDC, swirl-induced displacement of mixture in the squish region is under predicted
- Bowl: deviation under-predicted after 12.5 CAD ATDC
 - At 12.5 CAD BTDC, experimental data is less reliable due to cyclic variation as the crank angle increases



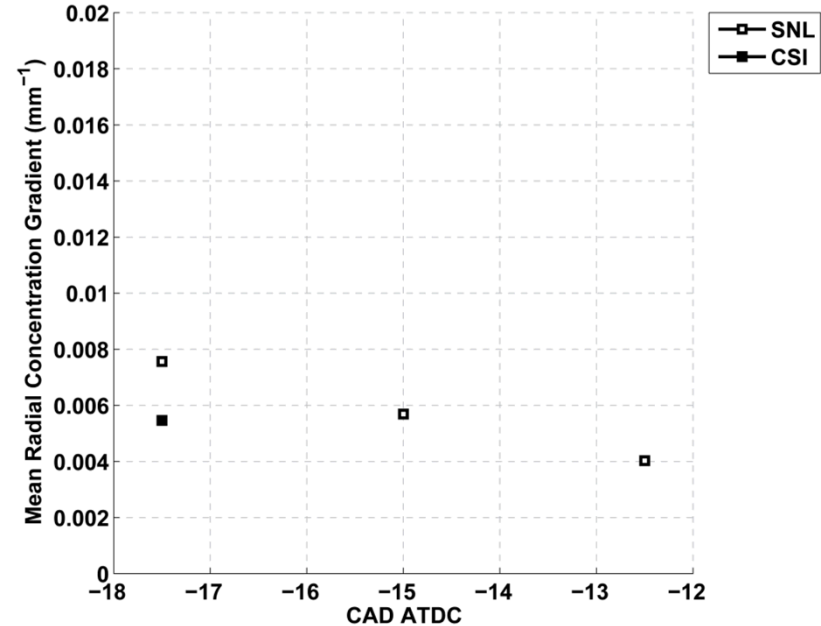
Upswirl fuel concentration gradients

- Experimentally-derived fuel concentration gradients are likely not reliable for earlier injection timings
 - Calibration for lean fuel ratios, not liquid fuel
 - Bright background reflections due to liquid fuel on concentration sensors
- Trend of concentration gradients for three injection timings at 7.5 CAD BTD
 - Gradients decrease with time
 - Some correlation of gradient magnitudes with injection timing
 - Gradients are likely related to the distribution of fuel concentration in the spray measurements



Radial gradients in jet head

- Experimental data are available / reliable for crank angles of 17.5, 15.0, and 12.5 CAD BTDC
- Simulated jets reach the wall by 15 CAD BTDC; radial gradient information is not available after that time
- Reasonable agreement (with cyclical variation) between given jets and CAD



Summary

- Simulated jet spreading angles are smaller than experimentally determined spreading angles
 - Experimentally determined spreading angles change with crank angle
- Simulated spray penetration is over predicted
 - Spray reaches the bore wall much sooner in the simulation than in the experiment (if the jets reach the wall at all in the experiment)
- Jet deflection by swirl is under predicted, especially after ~15 CAD BTDC
- Concentration gradients seem to match within reason
 - Match within range of cycle-to-cycle variability for a given jet

