

Time Domain Model Reduction of Linear Viscoelastic Finite Element Models

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service
in the
national
interest*



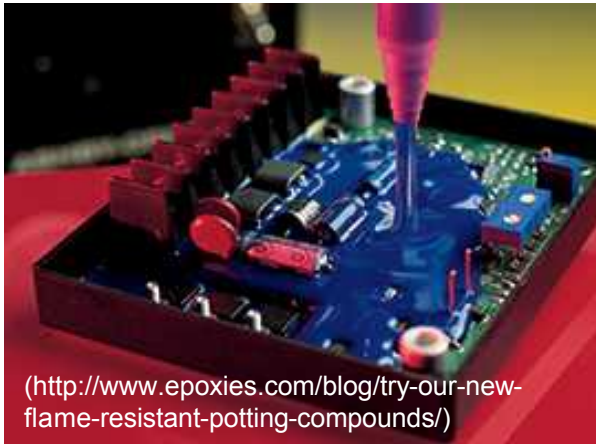
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What is our goal?

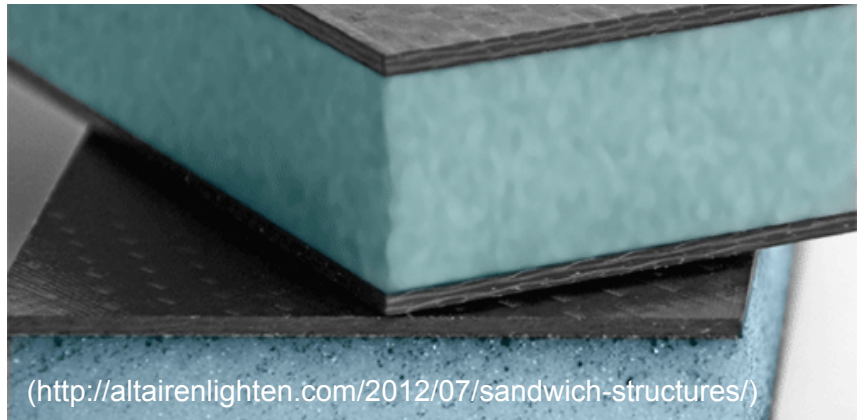
- Develop reduced order models (ROMs) of finite element models with linear viscoelastic material behavior for **time domain** structural dynamic simulations
- Reduce computational burden of repetitive numerical solutions while preserving the accuracy of the full order model
- Incorporate non-viscous damping into ROMs via material property data

Applications with Linear Viscoelastic Behavior

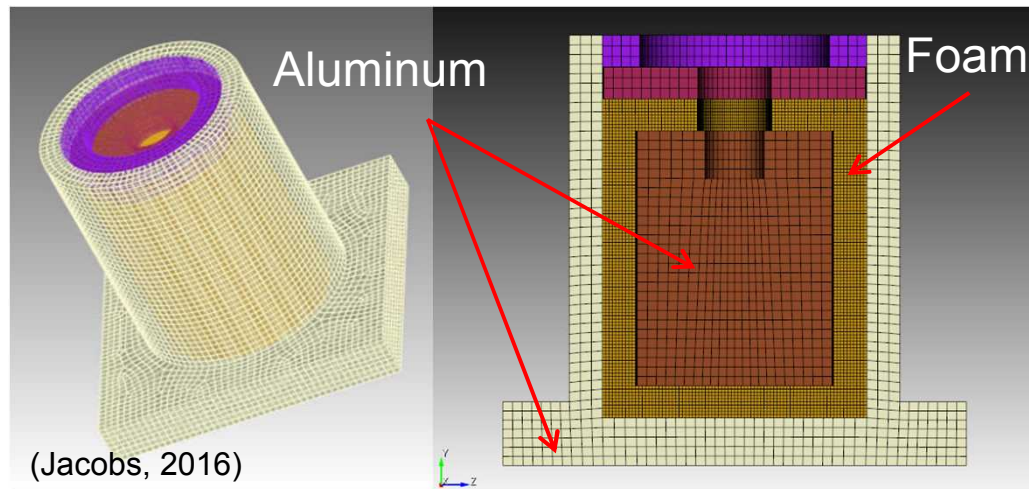
Encapsulation



Sandwich Structured Composites



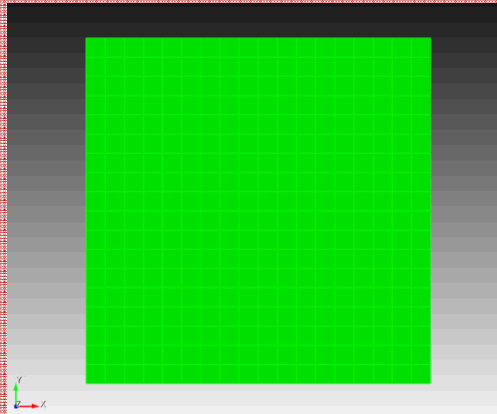
Ministack



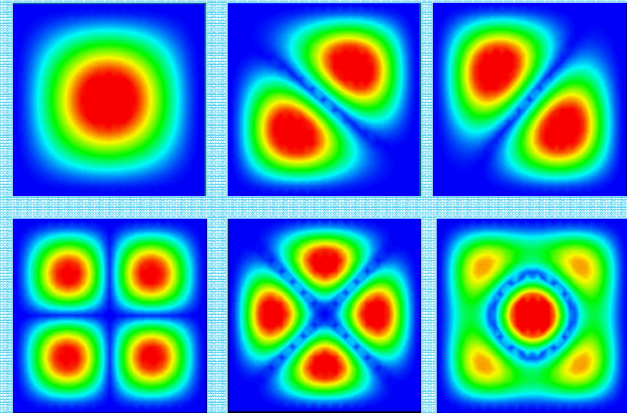
Reduced Order Modeling Approach

FE mesh in physical coordinates

$$\mathbf{M}\ddot{\mathbf{x}} + \mathbf{K}\mathbf{x} + \mathbf{f}_{matl}(\mathbf{x}, \dot{\mathbf{x}}) = \mathbf{f}(t)$$



Determine appropriate basis, or shape vectors based on the physical equations of motion



Solve reduced equations

$$\hat{\mathbf{M}}\ddot{\mathbf{q}} + \hat{\mathbf{K}}\mathbf{q} + \mathbf{f}_{matl}(\mathbf{q}, \dot{\mathbf{q}}) = \hat{\mathbf{f}}(t)$$

Project full equations of motion onto a small set of basis vectors

Related by transformation:

$$\mathbf{x}(t) = \mathbf{T}\mathbf{q}(t)$$

$$*\mathbf{q}(t) \ll \mathbf{x}(t)$$

Linear Viscoelasticity with Prony Series

- **Stress dependent upon time**

$$\sigma(t) = \int_0^t E(t - \tau) \frac{d\epsilon}{d\tau} d\tau$$

- **Prony series**

$$E(t) = E_\infty + (E_g - E_\infty)\zeta(t)$$

$$\zeta(t) = \sum_{i=1}^N E_i e^{-t/\tau_i}$$

Relaxation modulus: describes time- and history-dependent behavior!

⋮
(skipping detailed mathematics)

- **FEA equations of motion**

$$\mathbf{M}\ddot{\mathbf{x}} + \mathbf{K}_{v,K} \int_0^t \zeta_K(t - \tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_{v,G} \int_0^t \zeta_G(t - \tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_e \mathbf{x} = \mathbf{f}(t)$$

*Typically have many degrees-of-freedom!

Real Eigenmode Basis

$$\mathbf{M}\ddot{\mathbf{x}} + \mathbf{K}_{v,K} \int_0^t \zeta_K(t-\tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_{v,G} \int_0^t \zeta_G(t-\tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_e \mathbf{x} = \mathbf{f}(t)$$

■ Real Eigenmodes with Static Correction

$$\left(\mathbf{K}_e - \omega_r^2 \mathbf{M} \right) \boldsymbol{\phi}_r = \mathbf{0} \quad \longrightarrow \quad \mathbf{T} = [\boldsymbol{\phi}_1 \quad \boldsymbol{\phi}_2 \quad \dots \quad \boldsymbol{\phi}_{N_R} \quad \mathbf{R}_s]$$

$\mathbf{M} + \mathbf{K}_e \int_0^t \zeta(t-\tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_e \mathbf{x} = \mathbf{f}(t)$

■ *Must satisfy three conditions to work

- All elements in the FEA model made of same viscoelastic material
- Kernel functions equal for shear and bulk relaxation moduli
 $\zeta_K(t) = \zeta_G(t)$
- It must hold that $\frac{G_g - G_\infty}{K_g - K_\infty} = \frac{G_\infty}{K_\infty}$

$$\therefore \quad \mathbf{M}\ddot{\mathbf{x}} + \eta \mathbf{K}_e \int_0^t \zeta(t-\tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_e \mathbf{x} = \mathbf{f}(t)$$

Linearized Complex Eigenmode Basis

$$\mathbf{M}\ddot{\mathbf{x}} + \mathbf{K}_{v,K} \int_0^t \zeta_K(t-\tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_{v,G} \int_0^t \zeta_G(t-\tau) \dot{\mathbf{x}}(\tau) d\tau + \mathbf{K}_e \mathbf{x} = \mathbf{f}(t)$$

■ Linearized Complex Eigenmodes

- Iterative approach that uses linearized quadratic eigensolver in Sierra/SD

Linearized quadratic eigenvalue problem: for each mode, iterate until $\text{Im}(\lambda_r) = \omega_0$

$$\left(\lambda_r^2 \mathbf{M} + \lambda_r \mathbf{K}_{v,K} \sum_{i=1}^{N_K} \frac{K_{coeff,i}}{\lambda_0 + 1/\tau_{K,i}} + \lambda_r \mathbf{K}_{v,G} \sum_{i=1}^{N_G} \frac{G_{coeff,i}}{\lambda_0 + 1/\tau_{G,i}} + \mathbf{K}_e \right) \mathbf{u}_r(\lambda_0) = \mathbf{0} \quad \text{with } \lambda_0 = i\omega_0$$

Quasi-static Correction

$$\mathbf{R}_{qs} = \left(i\omega \mathbf{K}_{v,K} \sum_{i=1}^{N_K} \frac{K_{coeff,i}}{i\omega + 1/\tau_{K,i}} + i\omega \mathbf{K}_{v,G} \sum_{i=1}^{N_G} \frac{G_{coeff,i}}{i\omega + 1/\tau_{G,i}} + \mathbf{K}_e \right)^{-1} \mathbf{b} \quad \text{with } \omega \gg 0$$



$$\mathbf{T} = [\text{Re}(\mathbf{u}_1 \quad \mathbf{u}_2 \quad \dots \quad \mathbf{u}_{N_R} \quad \mathbf{R}_{qs}) \quad \text{Im}(\mathbf{u}_1 \quad \mathbf{u}_2 \quad \dots \quad \mathbf{u}_{N_R} \quad \mathbf{R}_{qs})]$$

- Conditionally stable, second order accurate HHT method [1]

$$\mathbf{q}_{n+1} = \mathbf{q}_n + \Delta t_n \dot{\mathbf{q}}_n + \frac{\Delta t_n^2}{2} \ddot{\mathbf{q}}_n$$

$$\ddot{\mathbf{q}}_{n+1} = -\hat{\mathbf{M}}^{-1}[\mathbf{f}_{visco}(\mathbf{q}_{n+1}) + \mathbf{f}_e(\mathbf{q}_{n+1}) - \mathbf{f}_{ext}(t_{n+1})]$$

$$\dot{\mathbf{q}}_{n+1} = \dot{\mathbf{q}}_n + \frac{\Delta t_n}{2}(\ddot{\mathbf{q}}_n + \ddot{\mathbf{q}}_{n+1})$$

- Viscoelastic force recursively updated through incremental steps in time [2]

$$\mathbf{f}_{visco}(\mathbf{q}_{n+1}) = \sum_{i=1}^{N_K} \left[e^{-\frac{\Delta t_n}{\tau_{K,i}}} \mathbf{h}_i(t_n) + \frac{\tau_{K,i} K_{coeff,i} (1 - e^{-\frac{\Delta t_n}{\tau_{K,i}}})}{\Delta t_n} (\mathbf{q}_{n+1} - \mathbf{q}_n) \right]$$

$$\mathbf{h}_i(t_{n+1}) = e^{-\frac{\Delta t}{\tau_{K,i}}} \mathbf{h}_i(t_n) + \frac{\tau_{K,i} K_{coeff,i} (1 - e^{-\frac{\Delta t}{\tau_{K,i}}})}{\Delta t} (\mathbf{q}_{n+1} - \mathbf{q}_n)$$

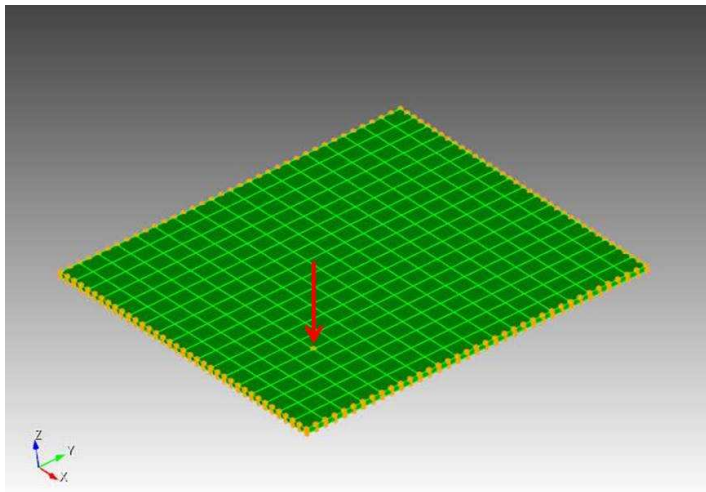
History state variable!

[1] H. M. Hilber, T. J. R. Hughes, and R. L. Taylor, "Improved Numerical Dissipation for Time Integration Algorithms in Structural Dynamics," *Earthquake Engineering & Structural Dynamics*, vol. 5, pp. 283-292, 1977.

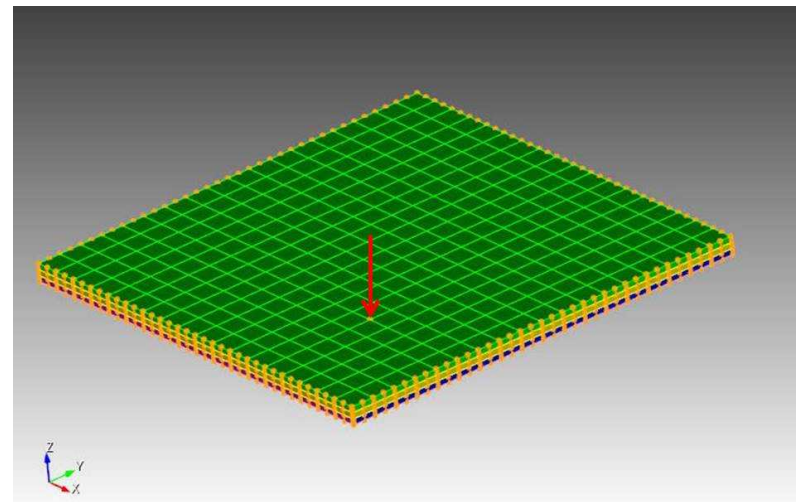
[2] J. C. Simo and T. J. R. Hughes, *Computational inelasticity*. New York: Springer, 1998.

Application to Plate Model in Sierra/SD

Case 1: Viscoelastic Plate



Case 2: Viscoelastic Sandwich Plate

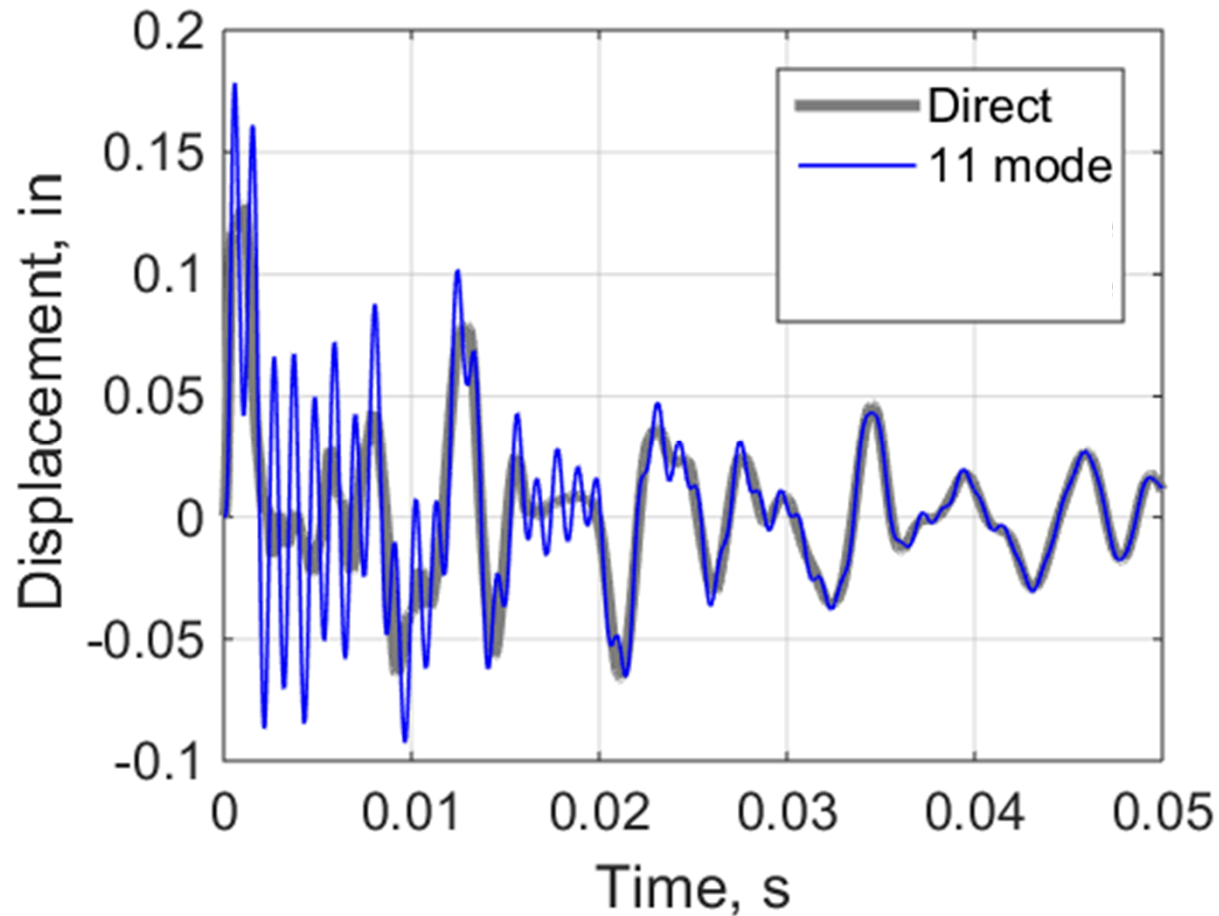


- PMDI 22 foam with 20 exp. terms
- Shear, bulk relaxation have same kernel function
- Rubbery & glassy moduli hold specific ratio:

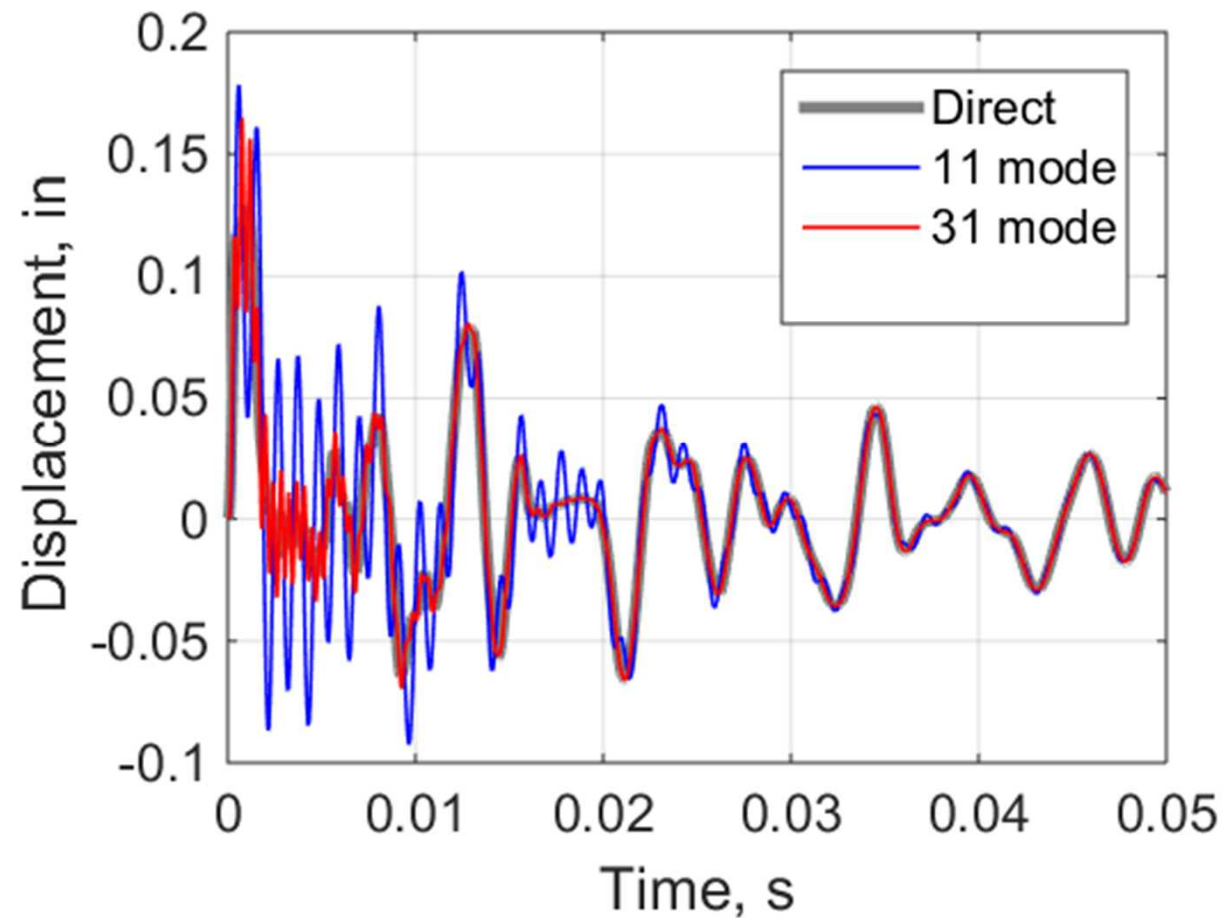
$$\frac{G_g - G_\infty}{K_g - K_\infty} = \frac{G_\infty}{K_\infty}$$

- PMDI 22 foam sandwiched between two AL 6061-T6 plates
- **Criterion for real modes not met**

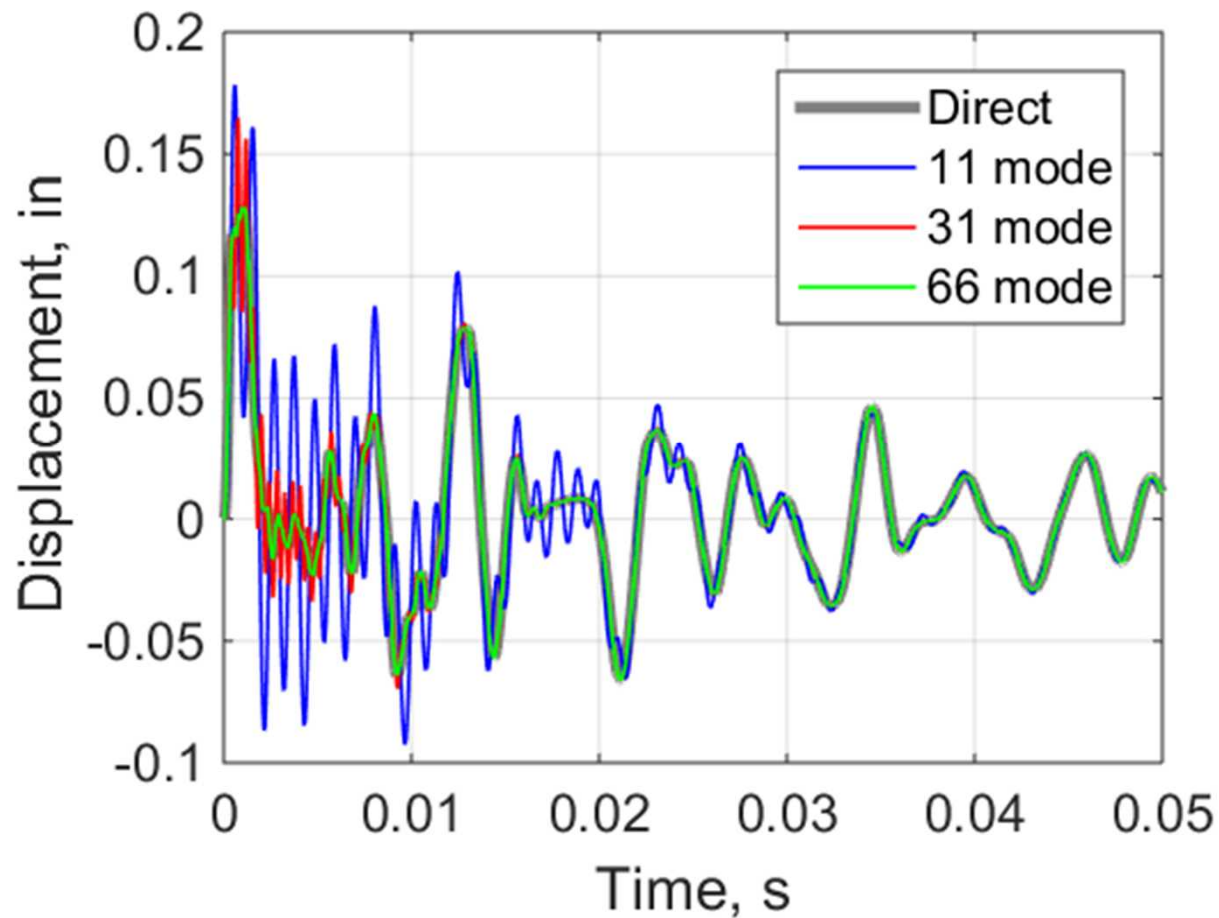
Case 1: Viscoelastic Plate with Real Modes



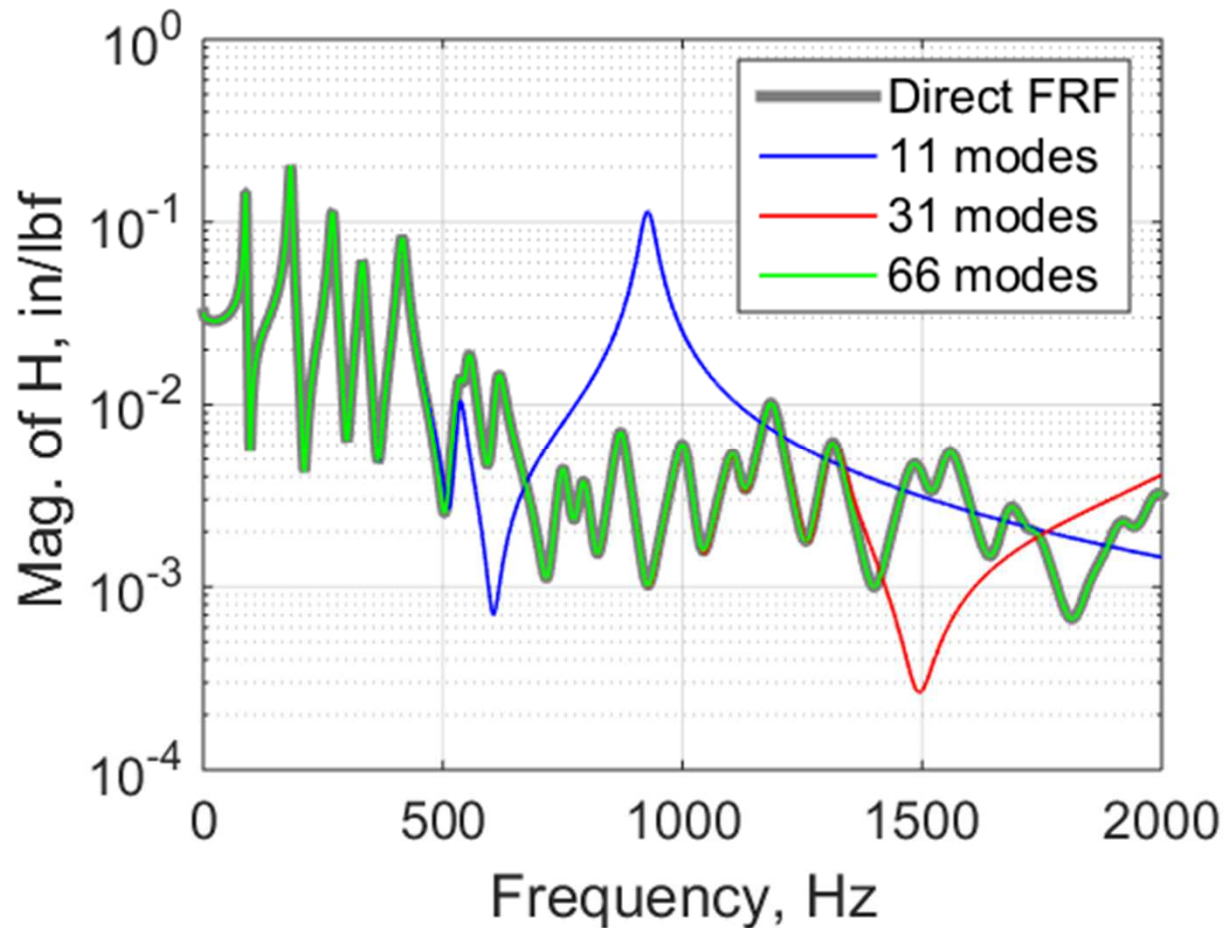
Case 1: Viscoelastic Plate with Real Modes



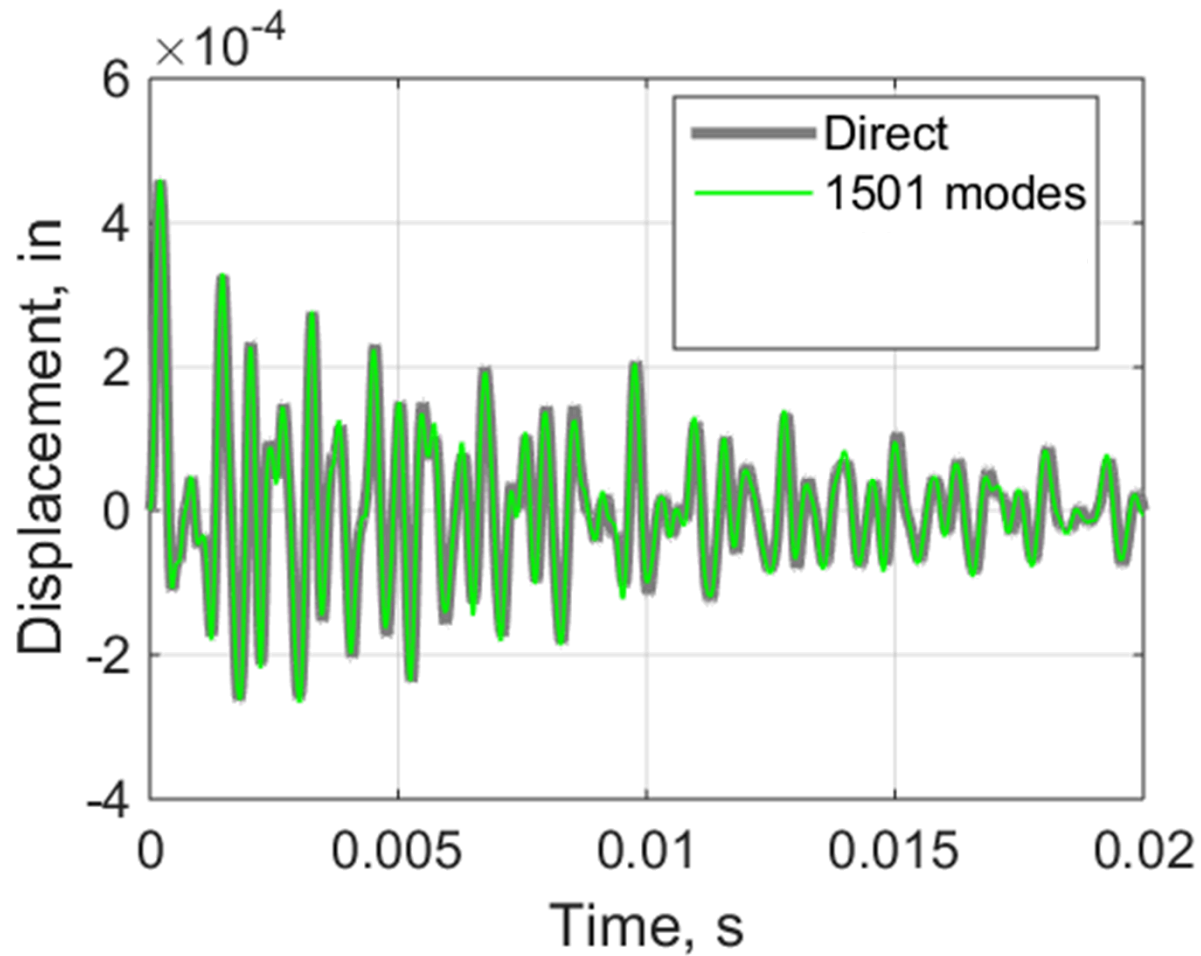
Case 1: Viscoelastic Plate with Real Modes



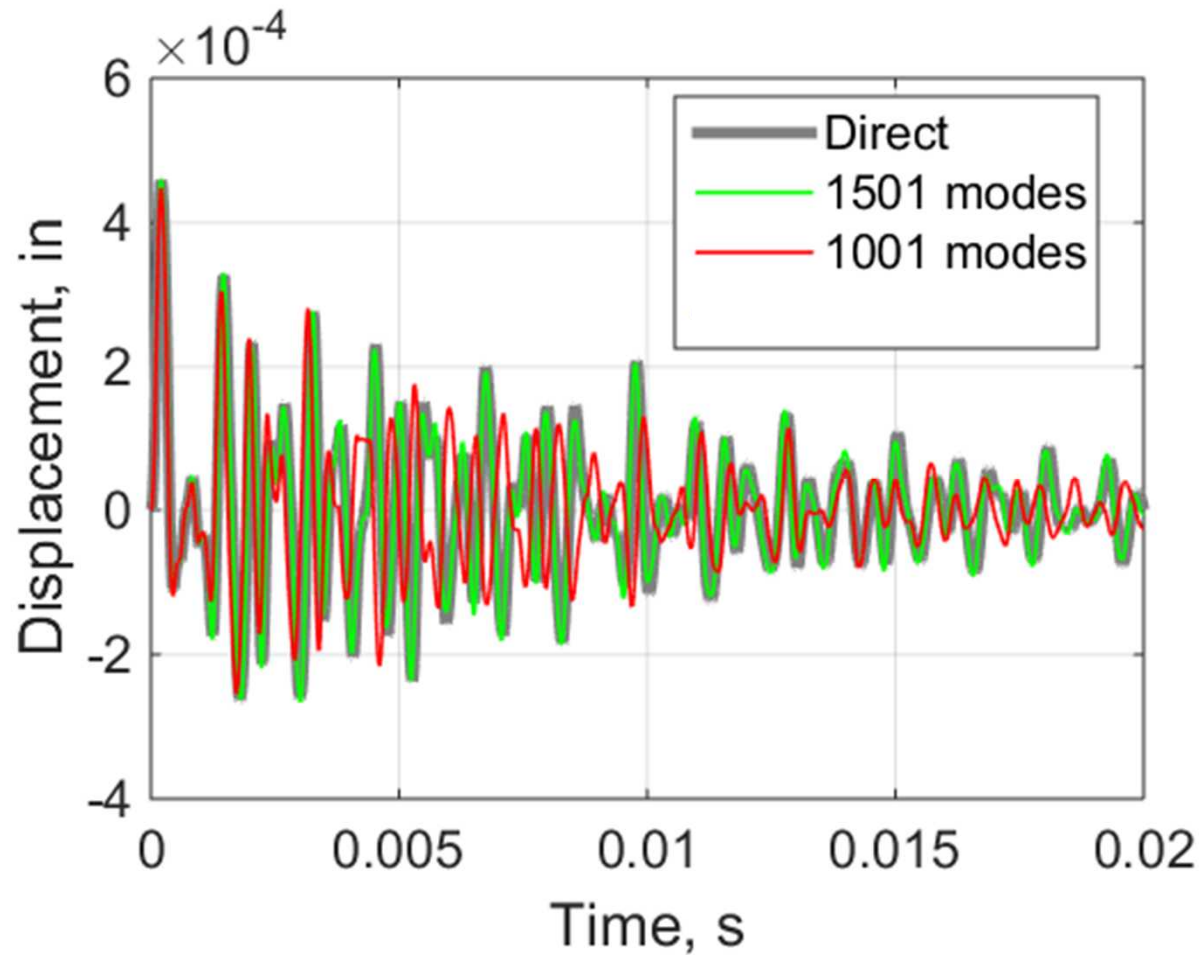
Case 1: Viscoelastic Plate with Real Modes



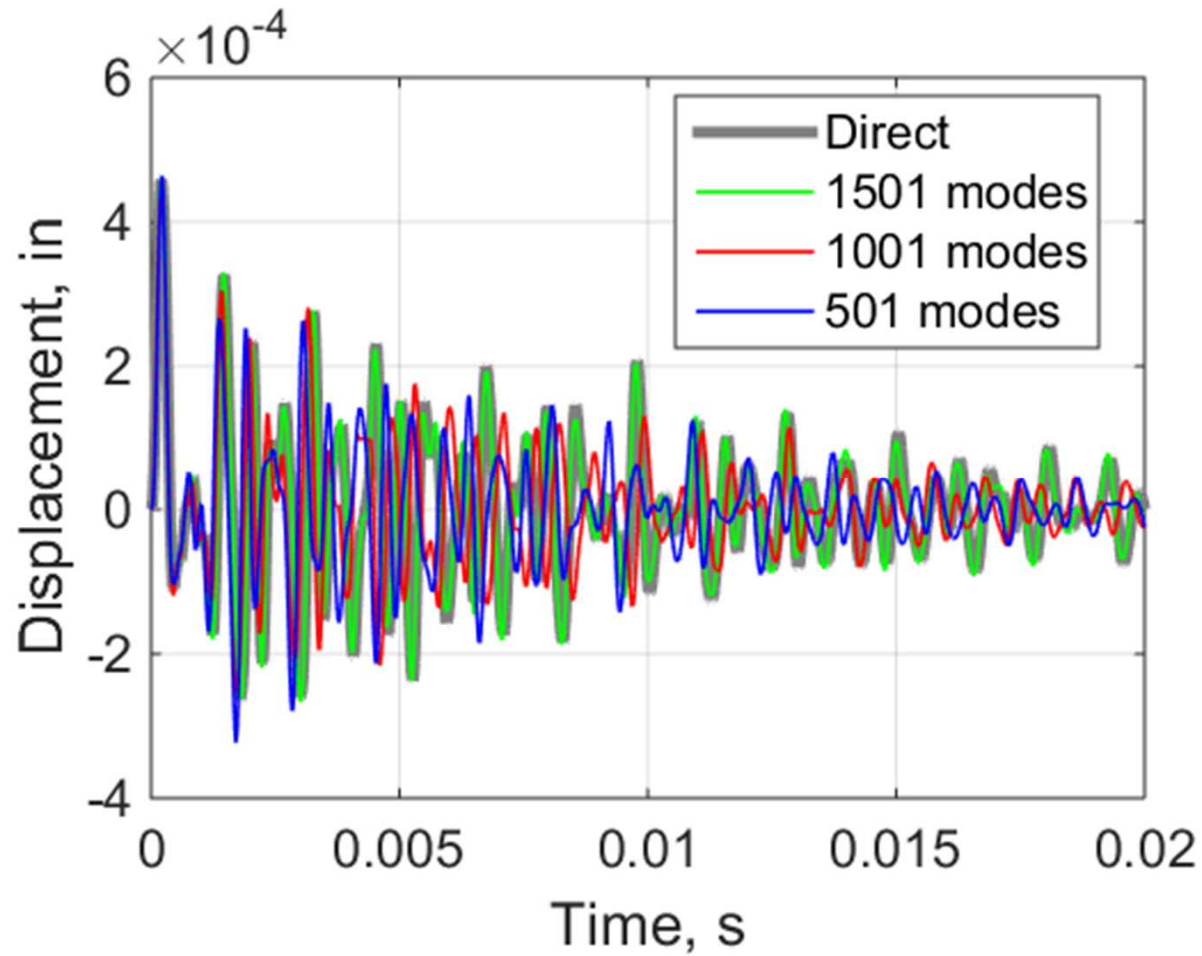
Case 2: Viscoelastic Sandwich Plate with Real Modes



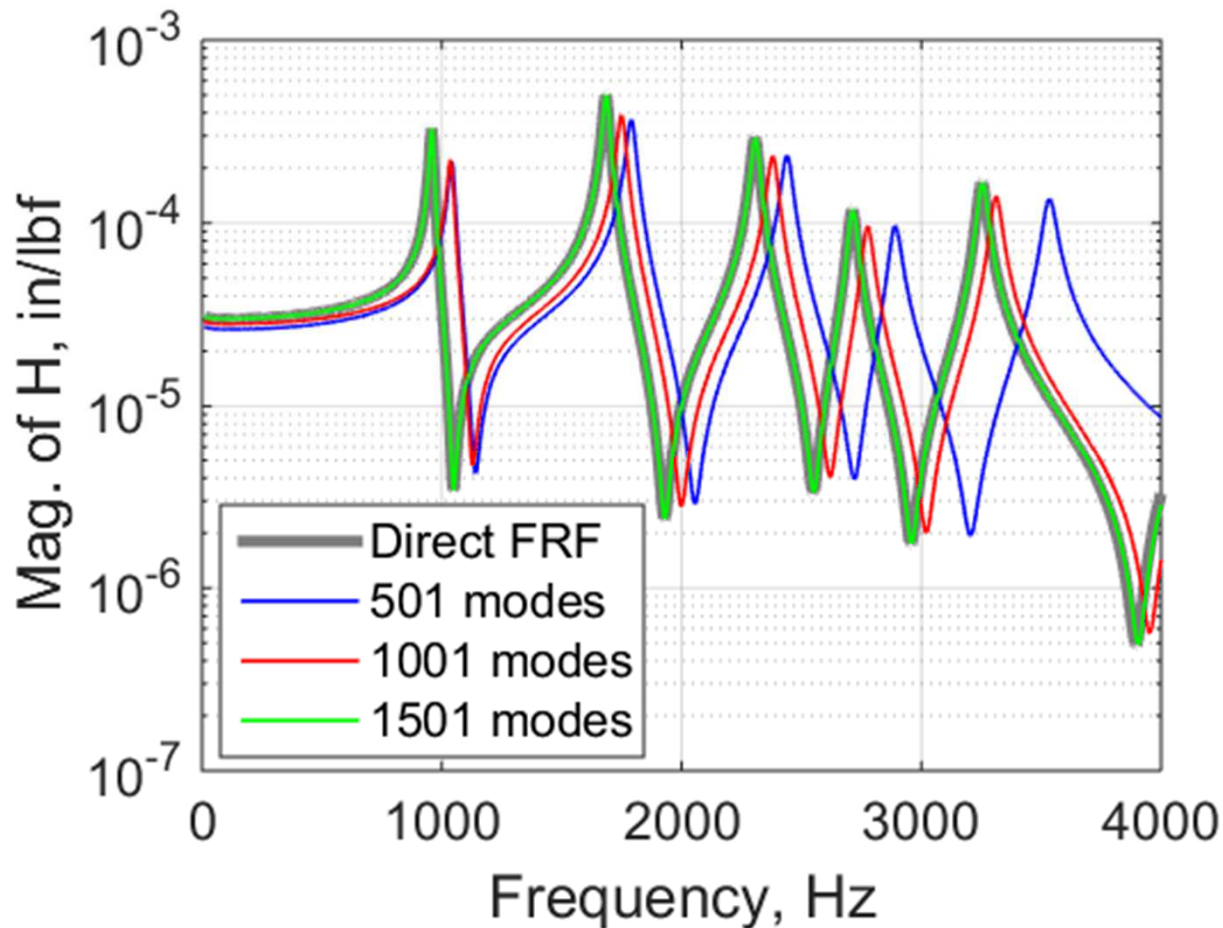
Case 2: Viscoelastic Sandwich Plate with Real Modes



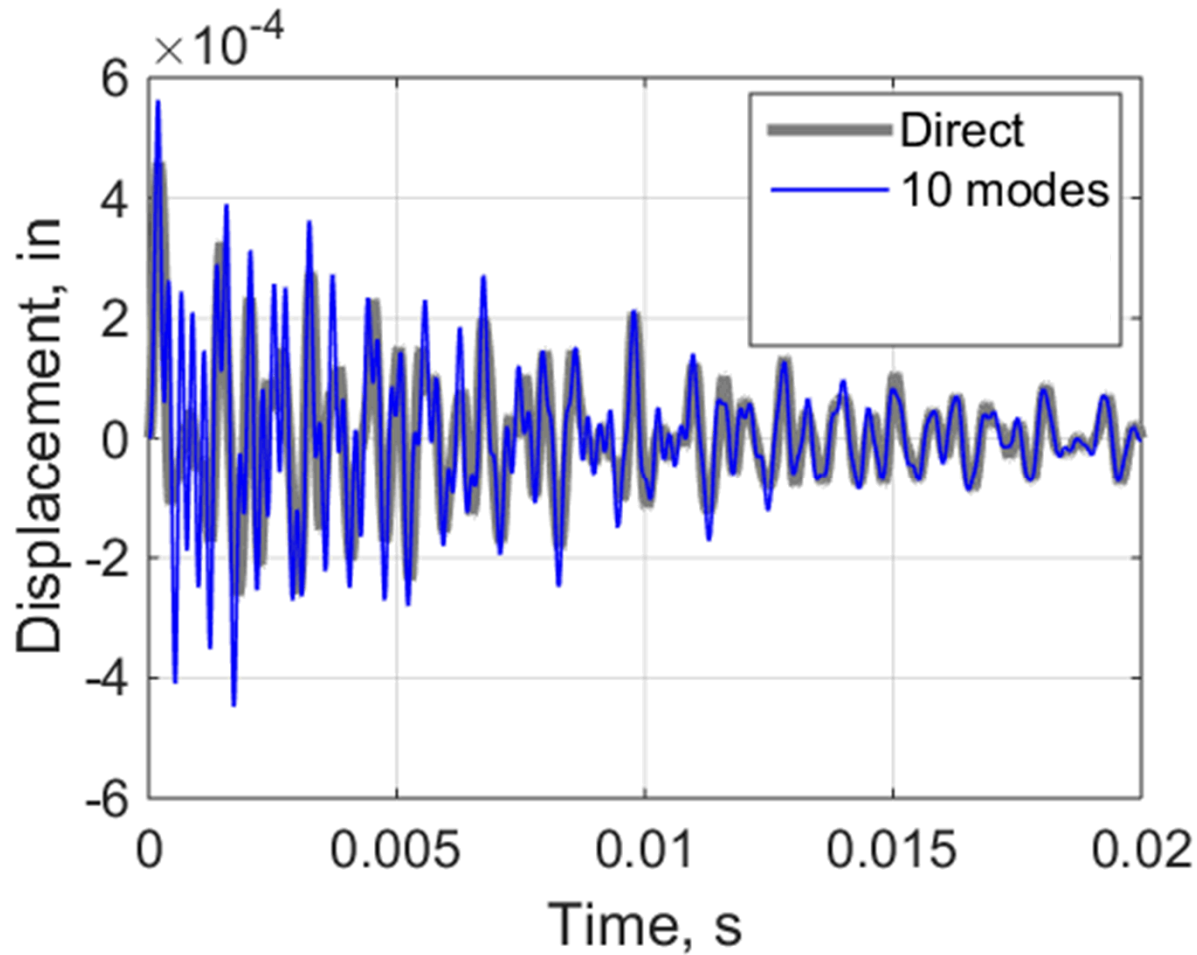
Case 2: Viscoelastic Sandwich Plate with Real Modes



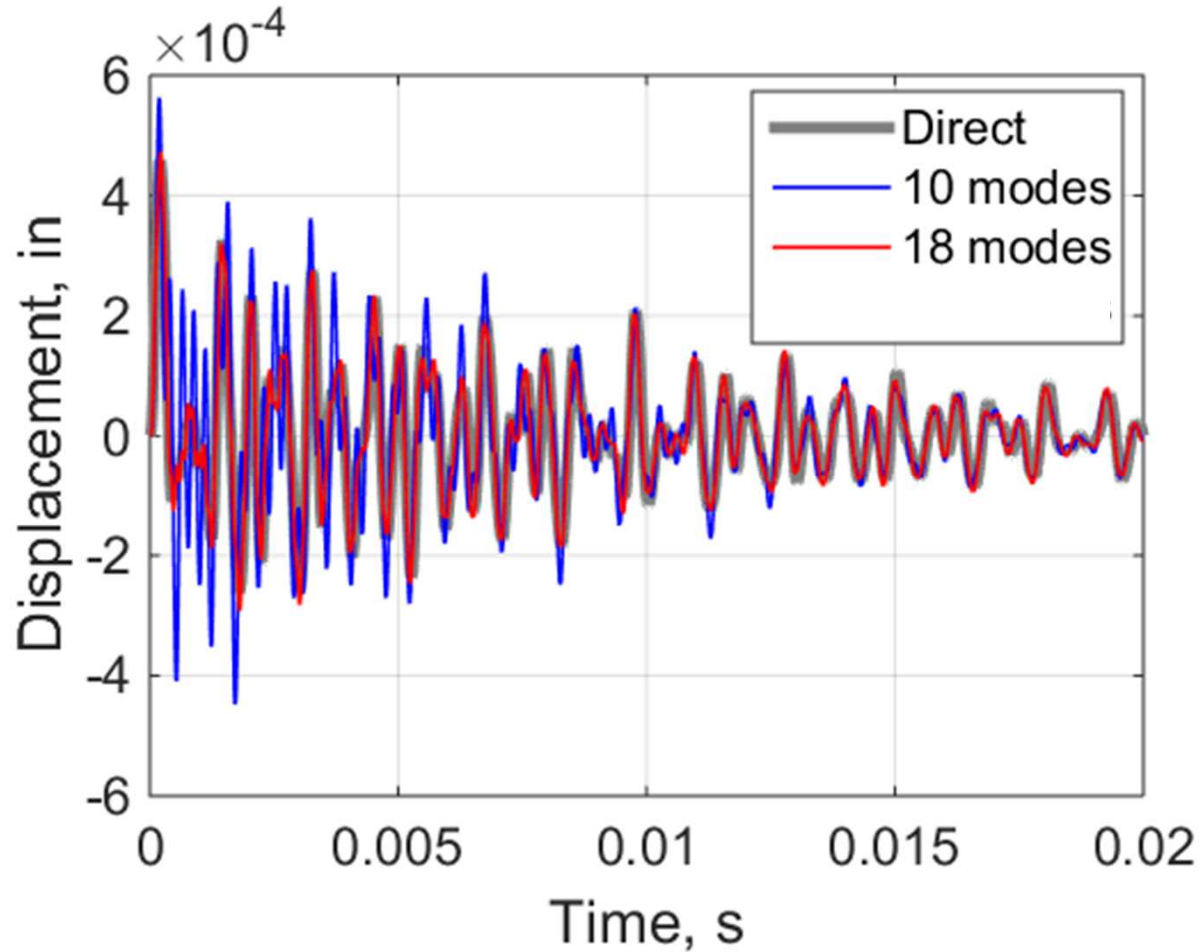
Case 2: Viscoelastic Sandwich Plate with Real Modes



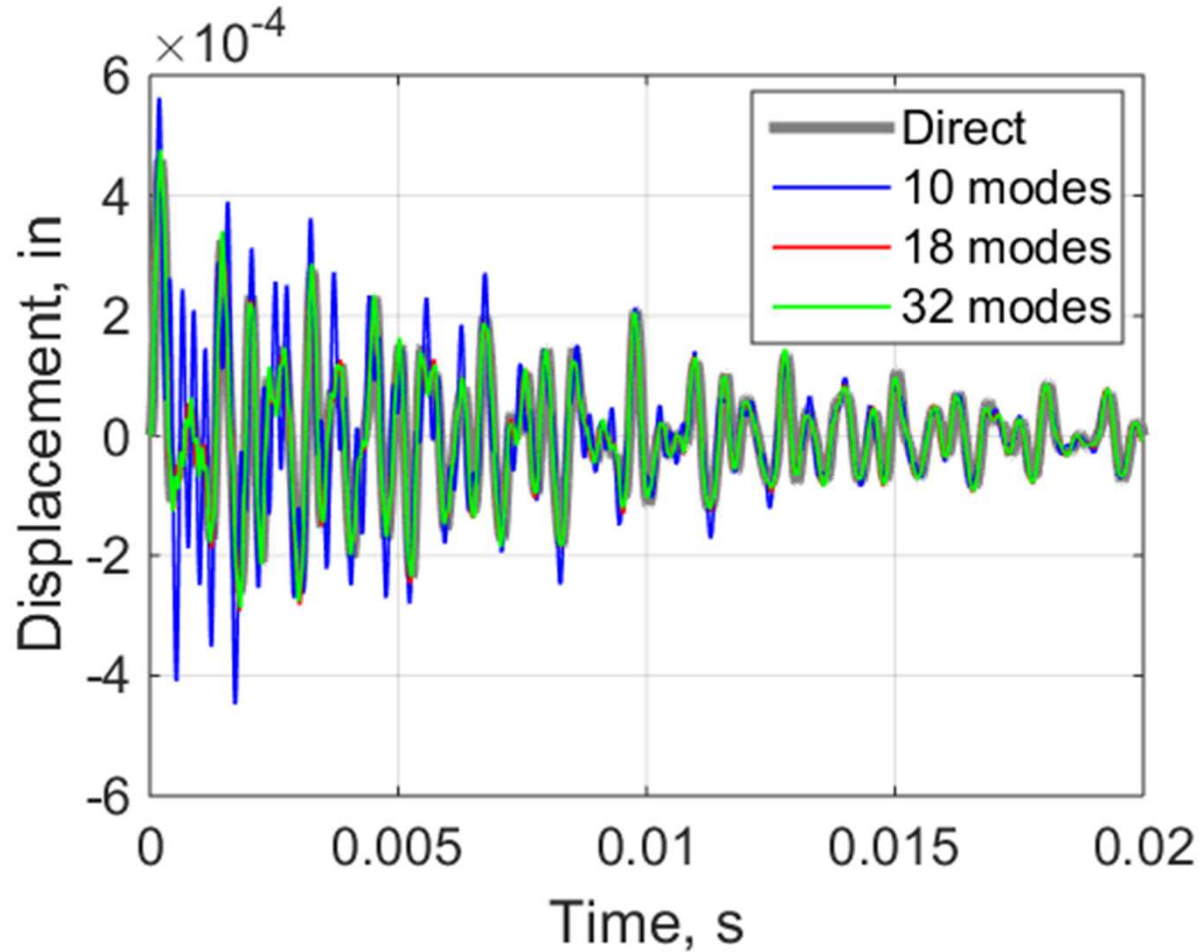
Case 2: Viscoelastic Sandwich Plate with Linearized Complex Modes



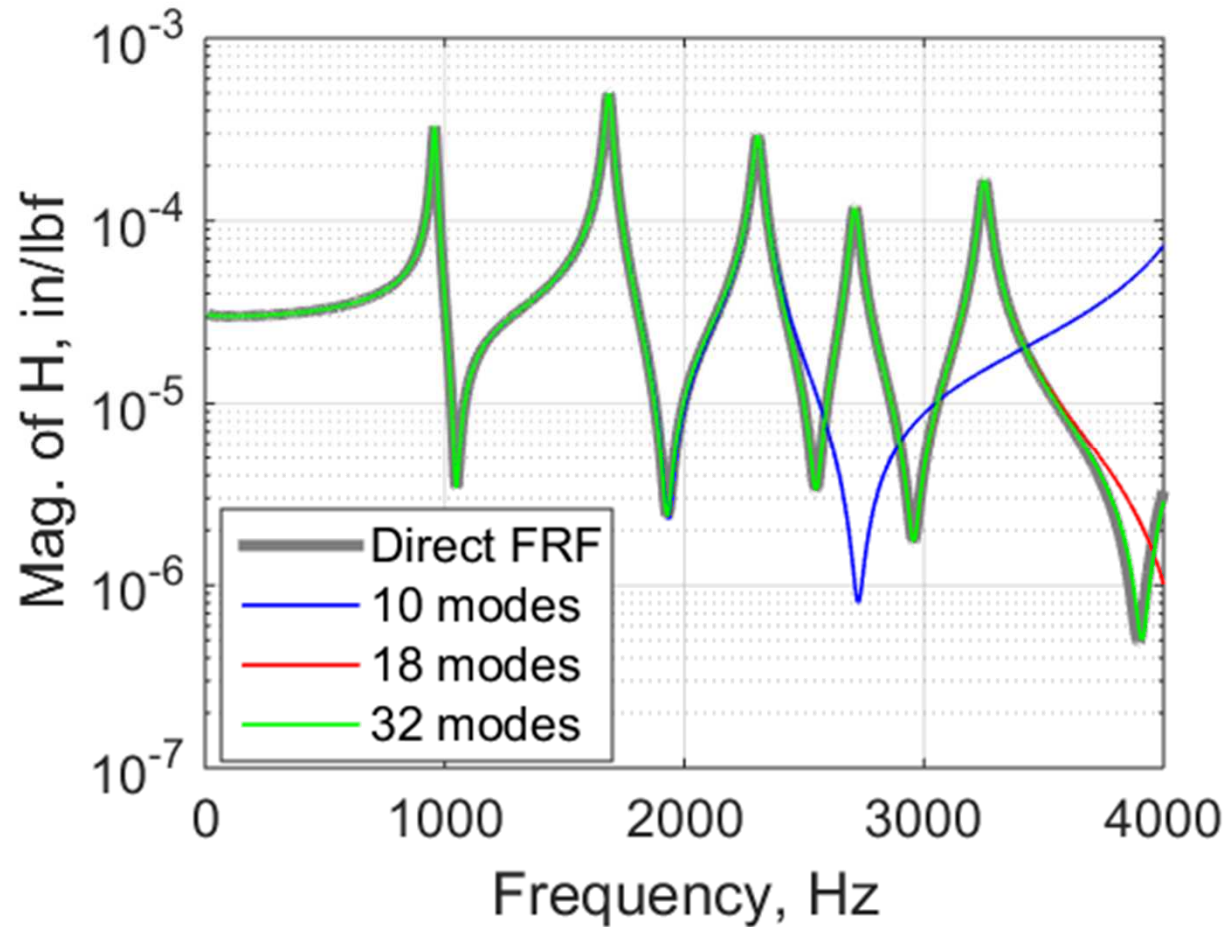
Case 2: Viscoelastic Sandwich Plate with Linearized Complex Modes



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Transient Solutions of Sandwich Plate

Model	Eigensolution	Solution Time	Total Cost
Full FEA model	n/a	~10 days* (dt = 1e-7)	~864,000 seconds (~14,400 min)
1501 mode ROM with real modes	229 seconds	2571 seconds (dt = 1e-6)	2800 seconds (~47 min)
32 mode ROM with linearized complex modes	4923 seconds	4.2 seconds (dt = 5e-6)	4927 seconds (~82 min)

*Estimate based on integration in Matlab on single processor

Conclusions

- Developed **two approaches to reduce the size** of a finite element model with linear viscoelastic material behavior
 - Real eigenmodes
 - Linearized complex eigenmodes
- Real eigenmodes perform well when the **three conditions are met**; linearized complex eigenmodes are efficient for the **general case**.
- Obtained reduced order models with **non-viscous damping** from Prony series representation of the viscoelastic material

Any Questions?

- This research was supported by the Laboratory Directed Research and Development program at Sandia National Laboratories, a multi-program laboratory managed and operated by Sandia Corporation, a wholly owned subsidiary of Lockheed Martin Corporation, for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-AC04-94AL85000.

- Contact Information
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 - Kevin Troyer: kltroye@sandia.gov

Extra Slides

	Real Modes	Complex Modes
Pro	<ul style="list-style-type: none">• Upfront cost of computing modes is significantly less	<ul style="list-style-type: none">• Accounts for viscoelastic forces• Produces accurate model with efficient solve times
Con	<ul style="list-style-type: none">• Inaccurate for models when special conditions not met• Requires many more modes to achieve acceptable accuracy	<ul style="list-style-type: none">• Expensive to calculate each mode• Potential convergence issues solving quadratic eigenvalue problem