

# Dish Stirling High Performance Thermal Storage

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CSP SunShot SUMMIT 2016: HEAT TRANSPORT AND STORAGE

## Introduction & Background

### Concept

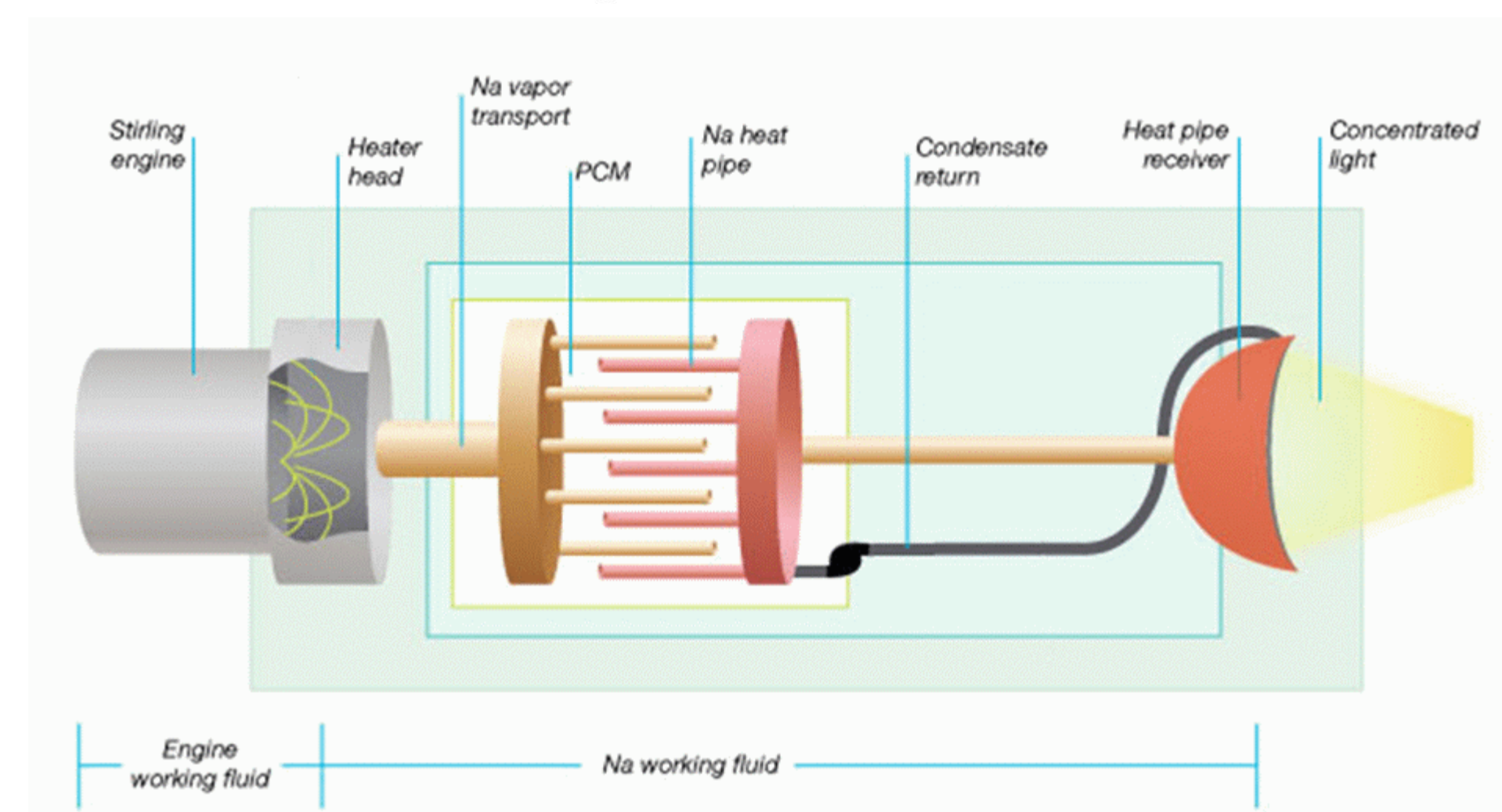
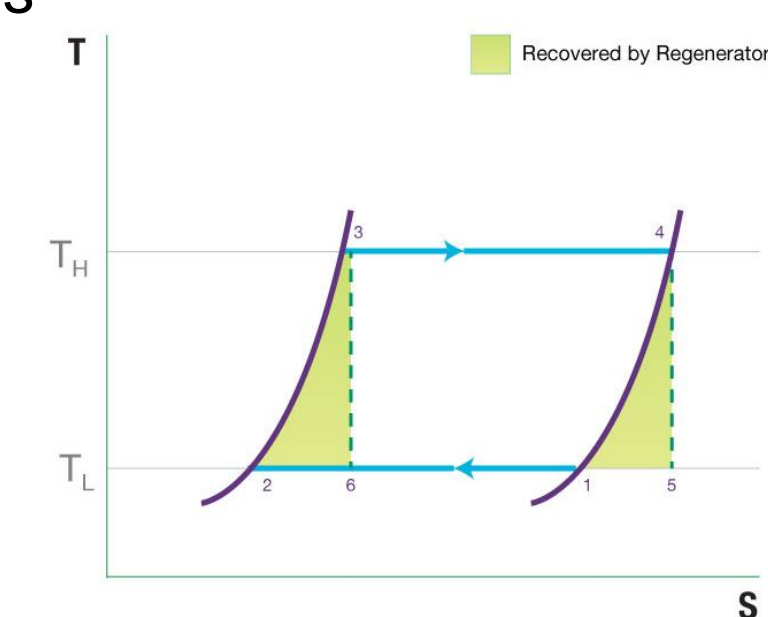
Enhancing high-performance dish-Stirling systems with up to 6 ours of thermal energy storage has the potential to increase performance, improve capacity, and enhance interest, making dish-Stirling systems a leading candidate to meet SunShot goals

### Why dish-Stirling?

- Demonstrated over 31% sun-to-grid, 26% annual
- High temperature, high concentration systems
- Highest efficiency thermodynamic cycle
- 6¢-8¢/kWh attainable with engineering and supply chain

### Latent heat transport and storage

- Isothermal input to engine
- Best match to isothermal transport, isothermal storage
- High exergy efficiency
- Isothermal transport has additional demonstrated system performance improvements
- 10-20% system performance boost
- Independent optimization of receiver, storage, engine
- Heat pipe is a “thermal transformer”
- First- and second-law improvements over existing systems



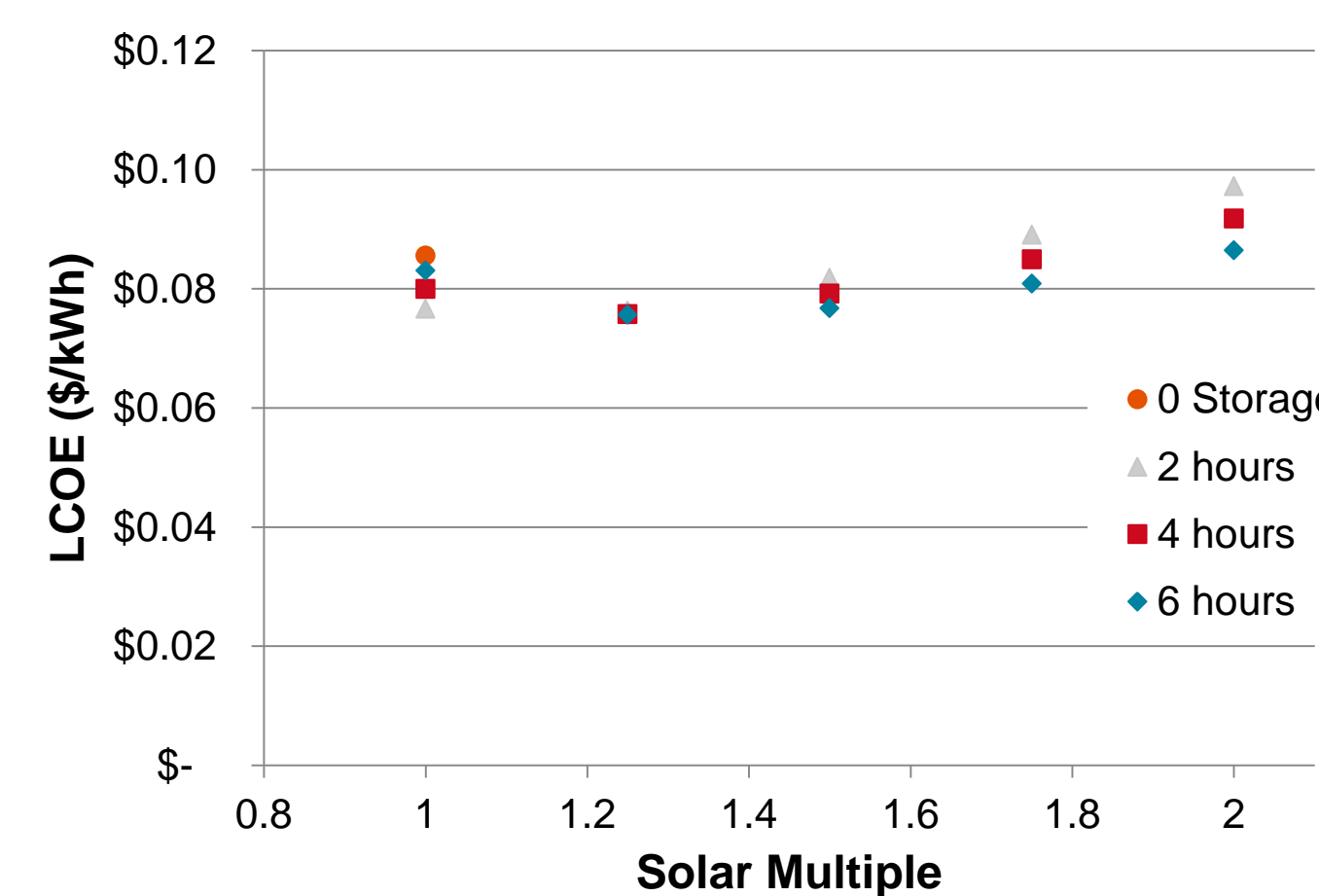
## System Level Model

### Field-level model

- Dish-to-dish shading
- Annual meteorological data (15-minute)

### Storage accumulator model

- Thermal input from met data
- Thermal output when engine running
- Shed energy when full (lost)
- Measured data with heat pipe receiver

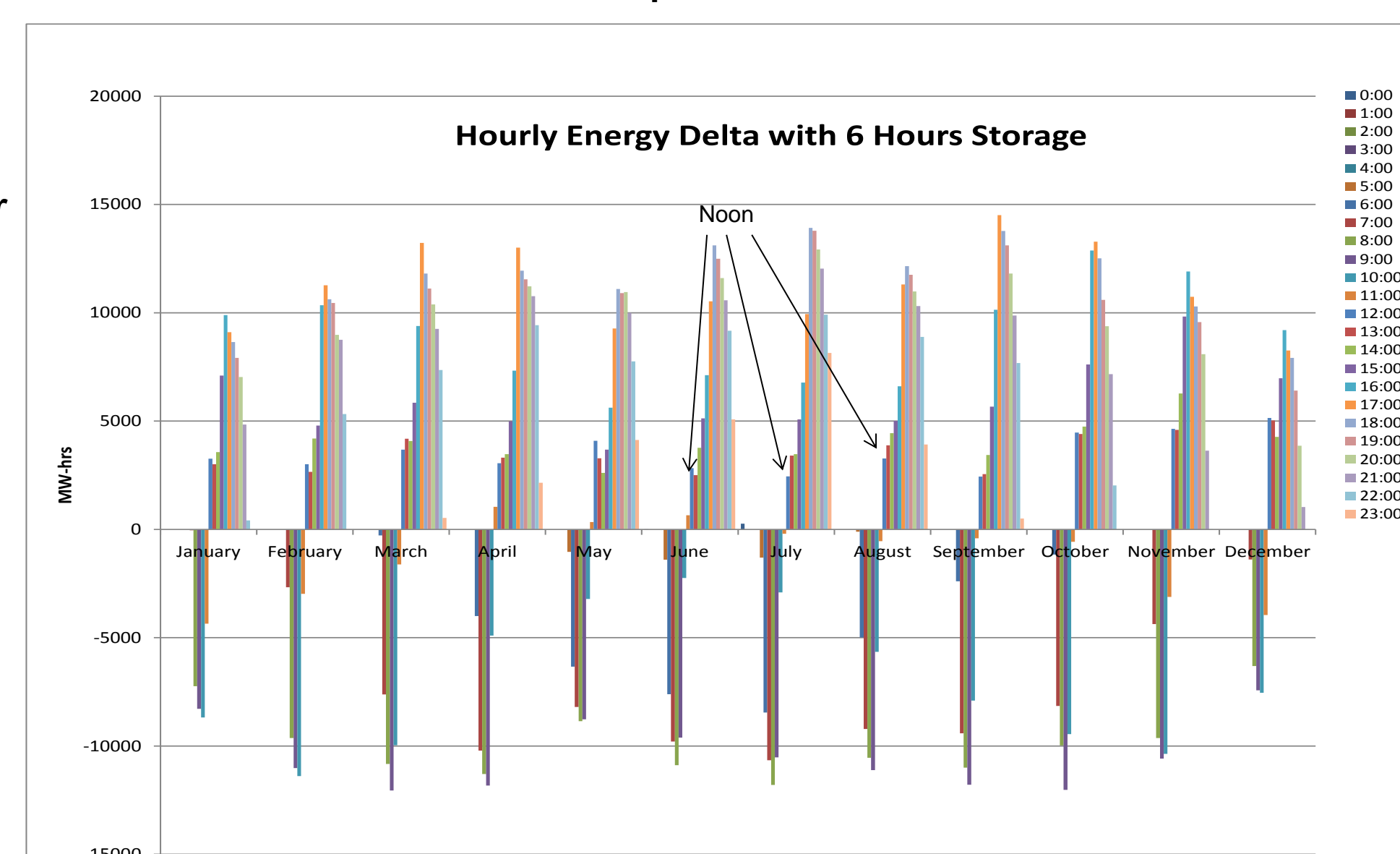


### Financial model

- Calculate LCOE based on 7.42% FCR
- Calculate “profit” based on SCE TOD
- Adjust dish and spacing proportional to solar multiple
- Fixed and variable cost of storage
  - \$3k/dish fixed
  - \$20/kWh<sub>th</sub> variable with storage size
- System cost set to \$2/W

### Model inputs exercised

- Size of storage
- Solar multiple
- Control modes



### Clear financial benefit

- About 1¢/kWh LCOE
- 2 ¢/kWh profit, due to TOD mapping

### Clear optimum in Solar Multiple at 1.25 for cases studied

- Greater storage improves LCOE to a point
- Better amortization of equipment costs
- Too much storage cannot be consistently used

### Total energy increase

- Greater collection area (solar multiple)
- Higher efficiency (always at design point of engine)

### Summer afternoon critical to profit

## Phase Change Material (PCM)

### PCM Development and Selection

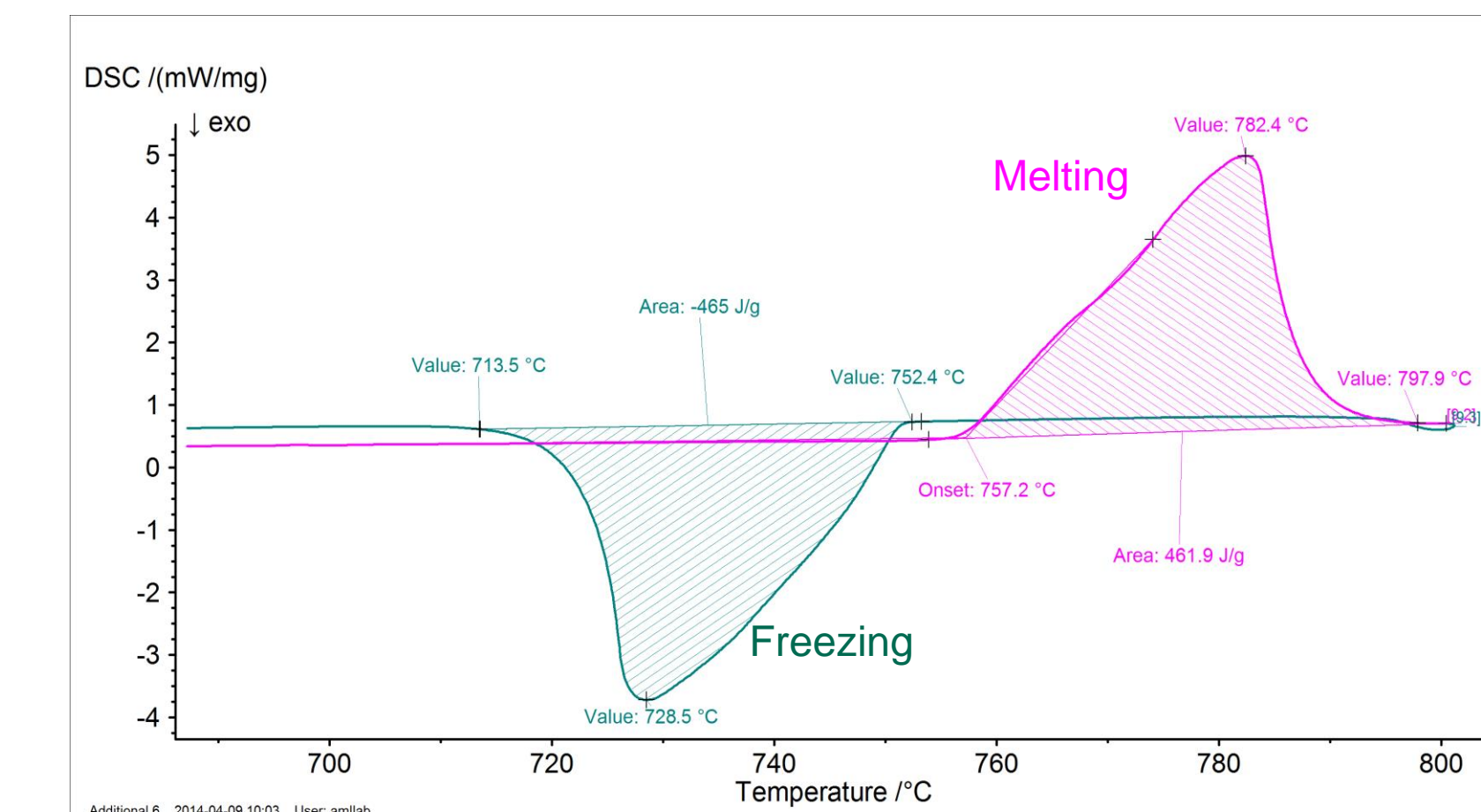
- Melting point goal 750-800°C
- Down Selected to metallic PCM's
  - High conductivity
  - High storage capacity
  - 2-D numeric modeling
- Two potential PCM's identified
  - Literature
  - FactSage modeling
  - Phase diagrams
- Ternary CuMgSi Selected

### PCM Compatibility

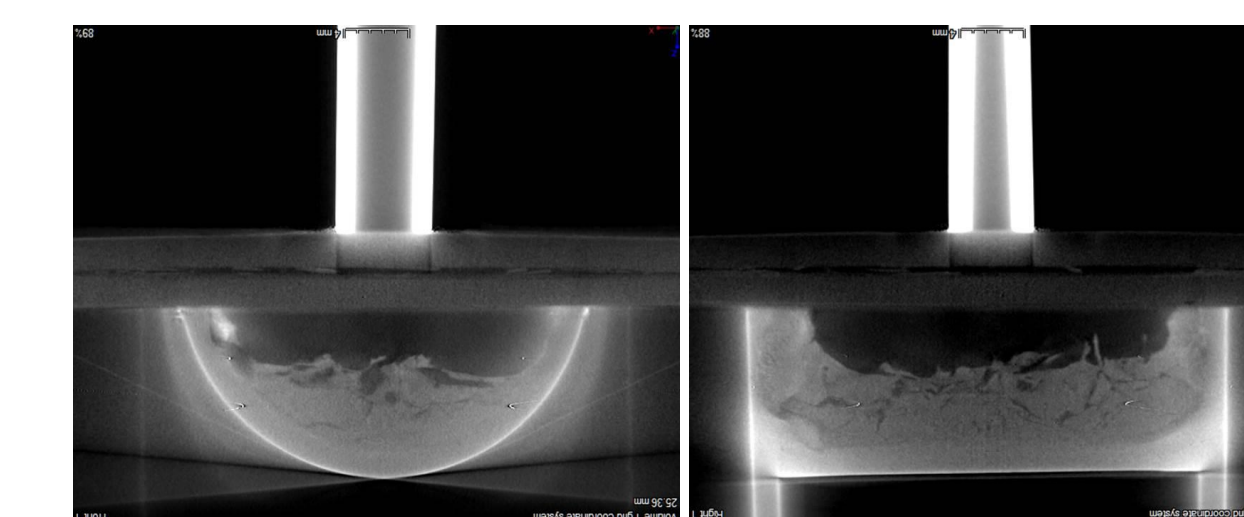
- Short-term testing demonstrated acute attack
  - 30% loss of containment in 150 hours
  - 3 containment alloys tested
- Protective coating development
  - Complex geometry limits options
  - Initial focus on solution coatings
  - Later focus on commercial thermal spray
  - J-tube test avoided weld and sealing issues
- Approach
  - Identify potential candidates by Gibbs Free Energy
  - Powdered XRD after crucible exposure
  - Sample boat coating exposure
    - Sectioning
    - X-Ray CT scans
- Results
  - MgAl<sub>2</sub>O<sub>4</sub> and Al<sub>2</sub>O<sub>3</sub> promising
  - Inconsistent vendor-to-vendor results

### PCM Fabrication

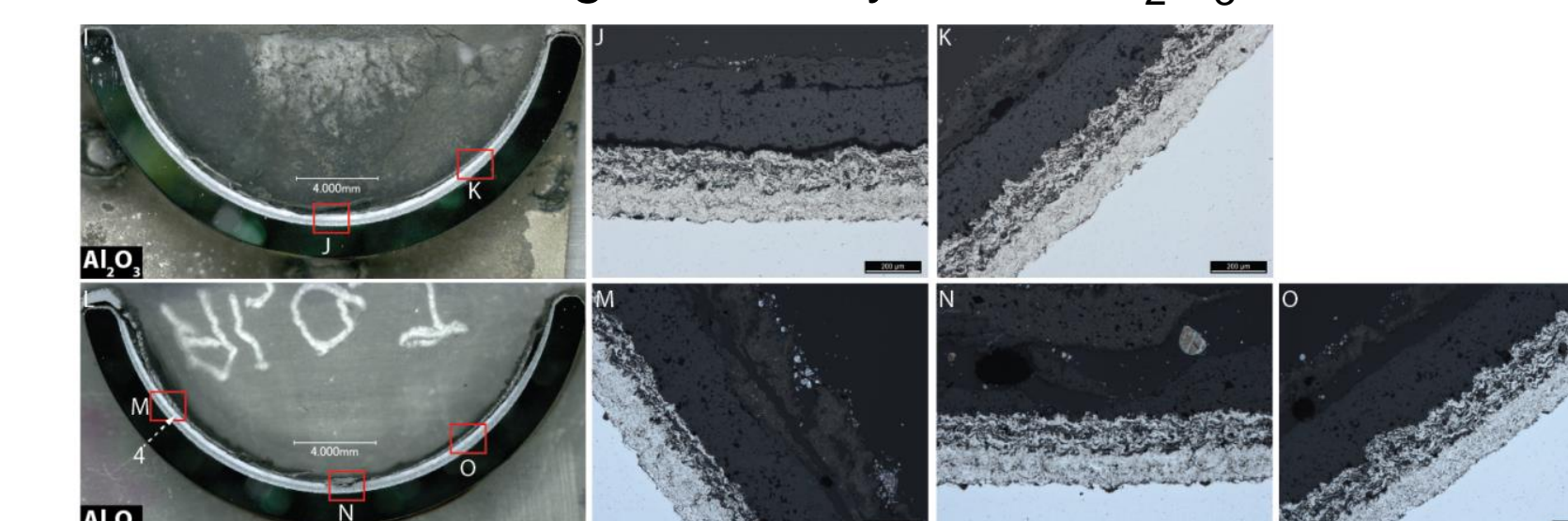
- Literature compositions varied
- Large variations of melt points of constituents and volatility made manufacturing technique elusive
- Successfully fabricated 3 literature compositions
  - Confirmed Birchenal
  - Confirmed high heat capacity
- Detailed exploration around terminal eutectic
  - 462 J/g
  - 757°C Onset of Melt
  - 10°C/min ramp rate in DSC



Ternary PCM Heat of Melting Characterization



Coating Boat x-ray CT for Al<sub>2</sub>O<sub>3</sub>



Sectioning of Al<sub>2</sub>O<sub>3</sub> showing no degradation after 500 hours



Air furnace and closed boat assembly

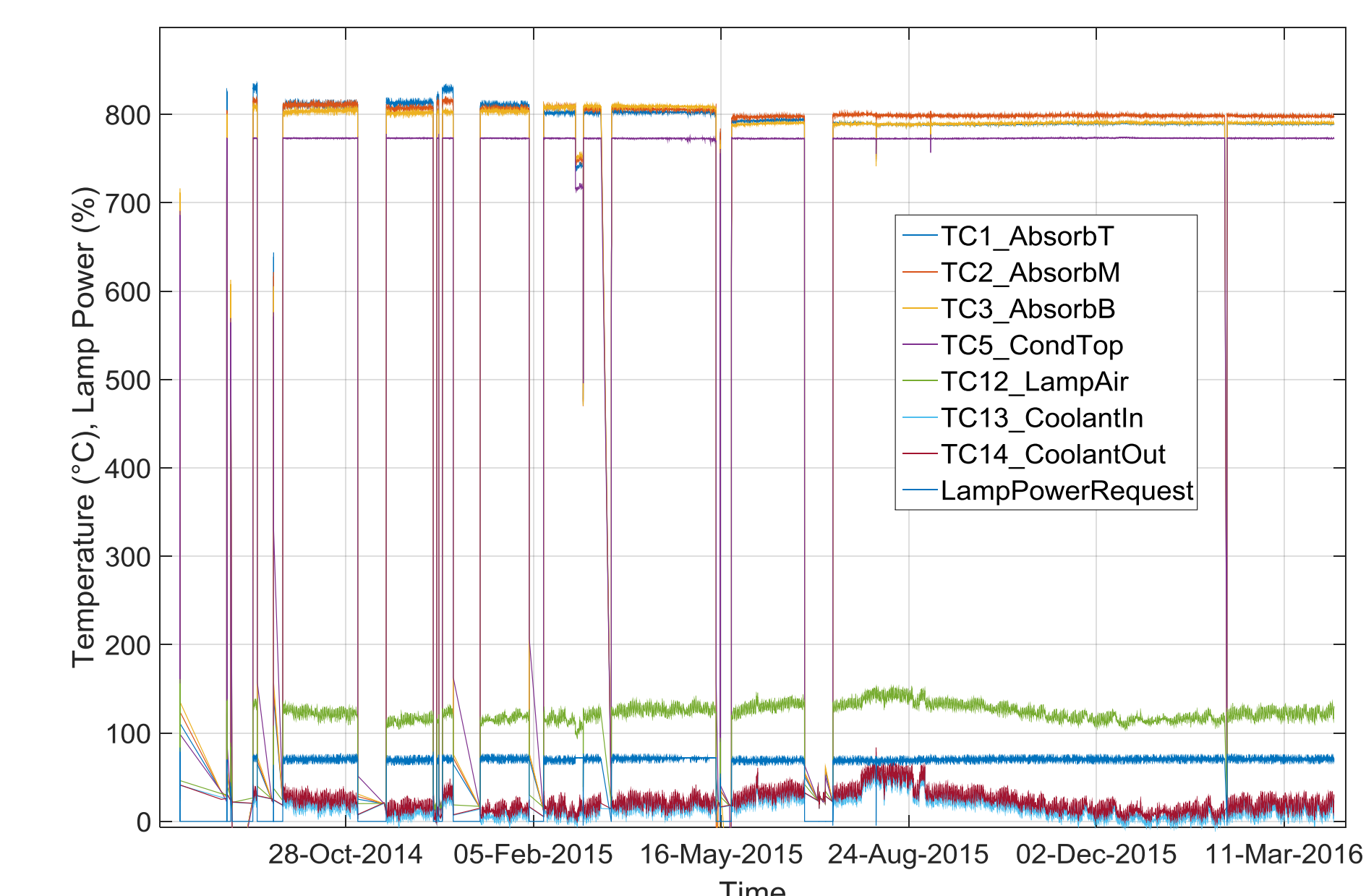
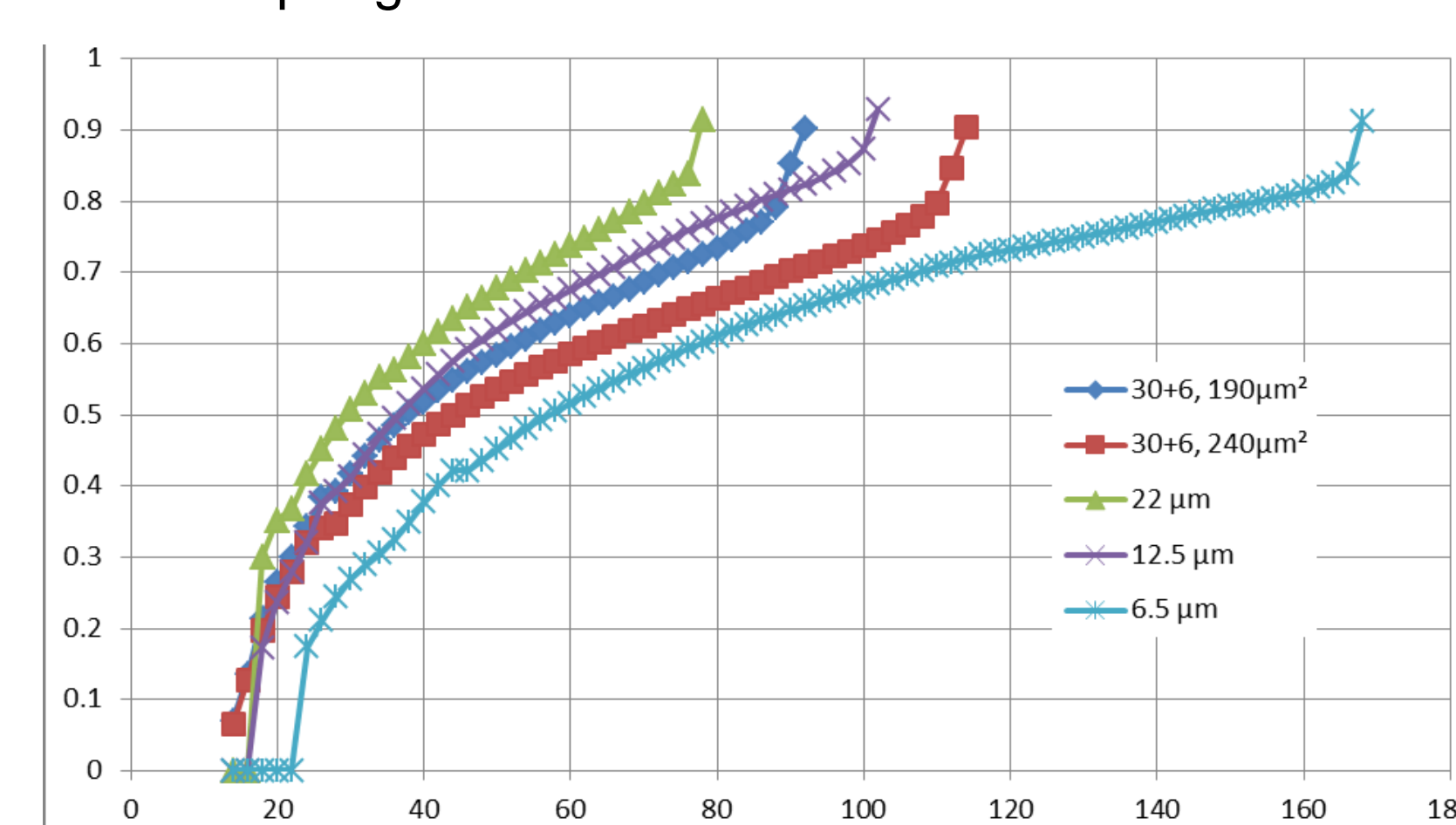
## Advanced Heat Pipe Receiver

### Advanced Wick Development

- Leverage felt wick concept for high performance through distributed pore sizes
- Enhance durability
  - Blended fiber sizes
  - Periodic support posts
- Permeability and pore distribution characterization
- Thermal loading

### Wick Model

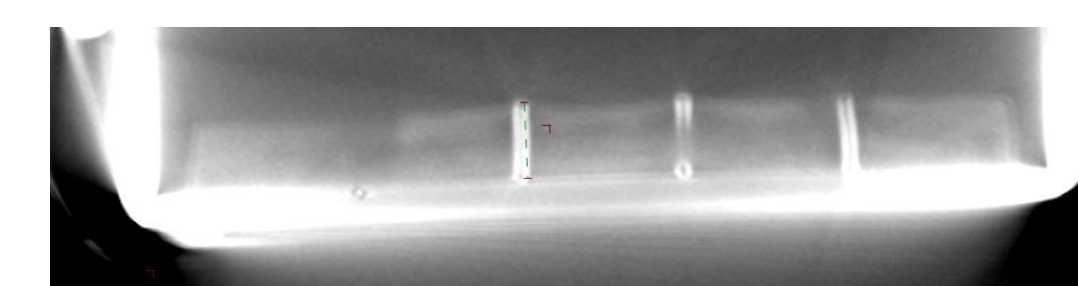
- Full scale receiver with realistic flux distribution
- 80kW throughput
- Correlate to bench scale 2.1kW model
- Permit vapor generation within wick



Bench-scale heat pipe durability test rig

### Durability Testing

- Completed over 12,000 hours
  - 1.5kW throughput
  - 800°C
- Periodic x-ray tomography confirms no wick collapse
- No change in operational characteristics



X-ray CT confirms no wick compression