

# Stress Waves Propagating through Bolted Joints

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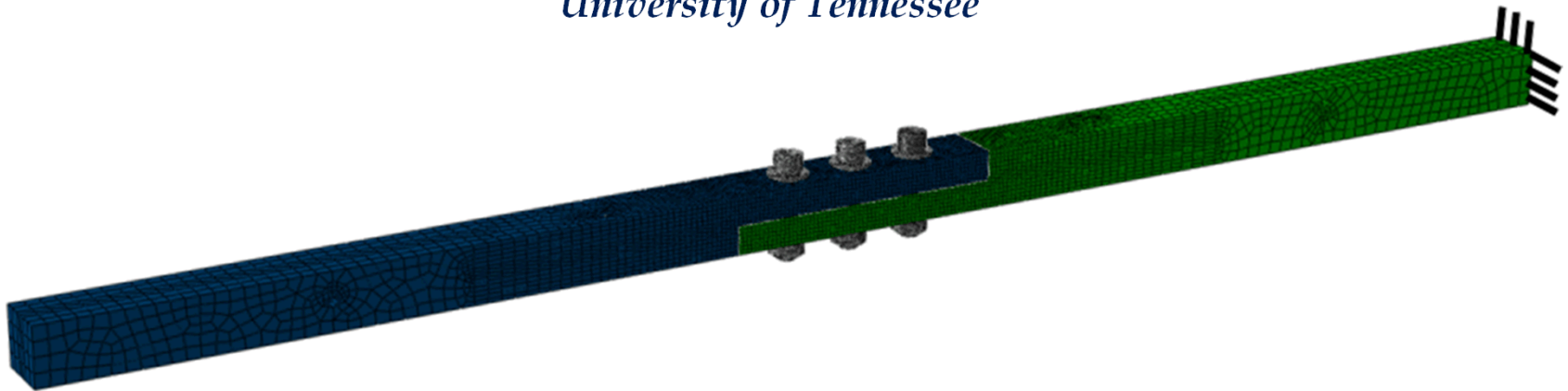
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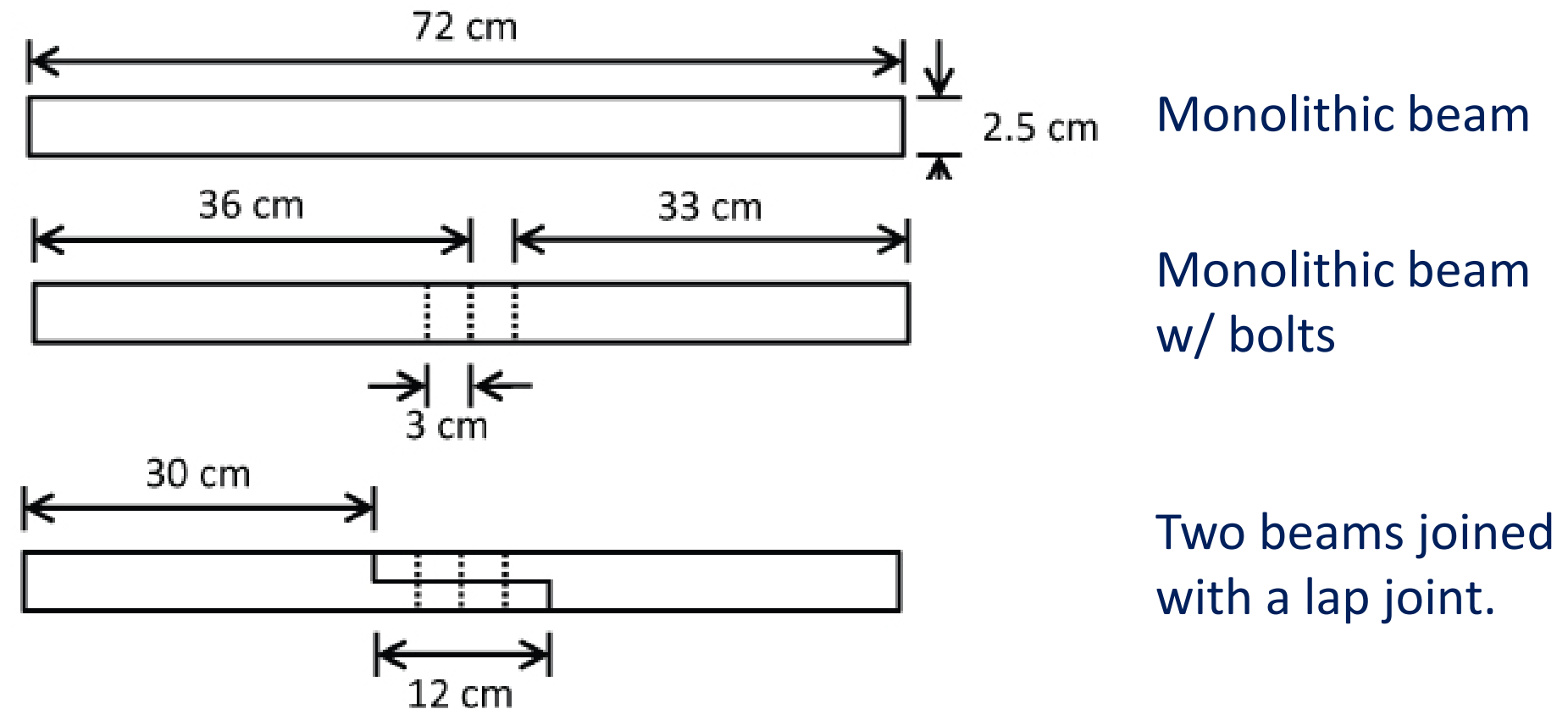
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# Introduction

→ Benchmark System: Brake-Reuß Beam

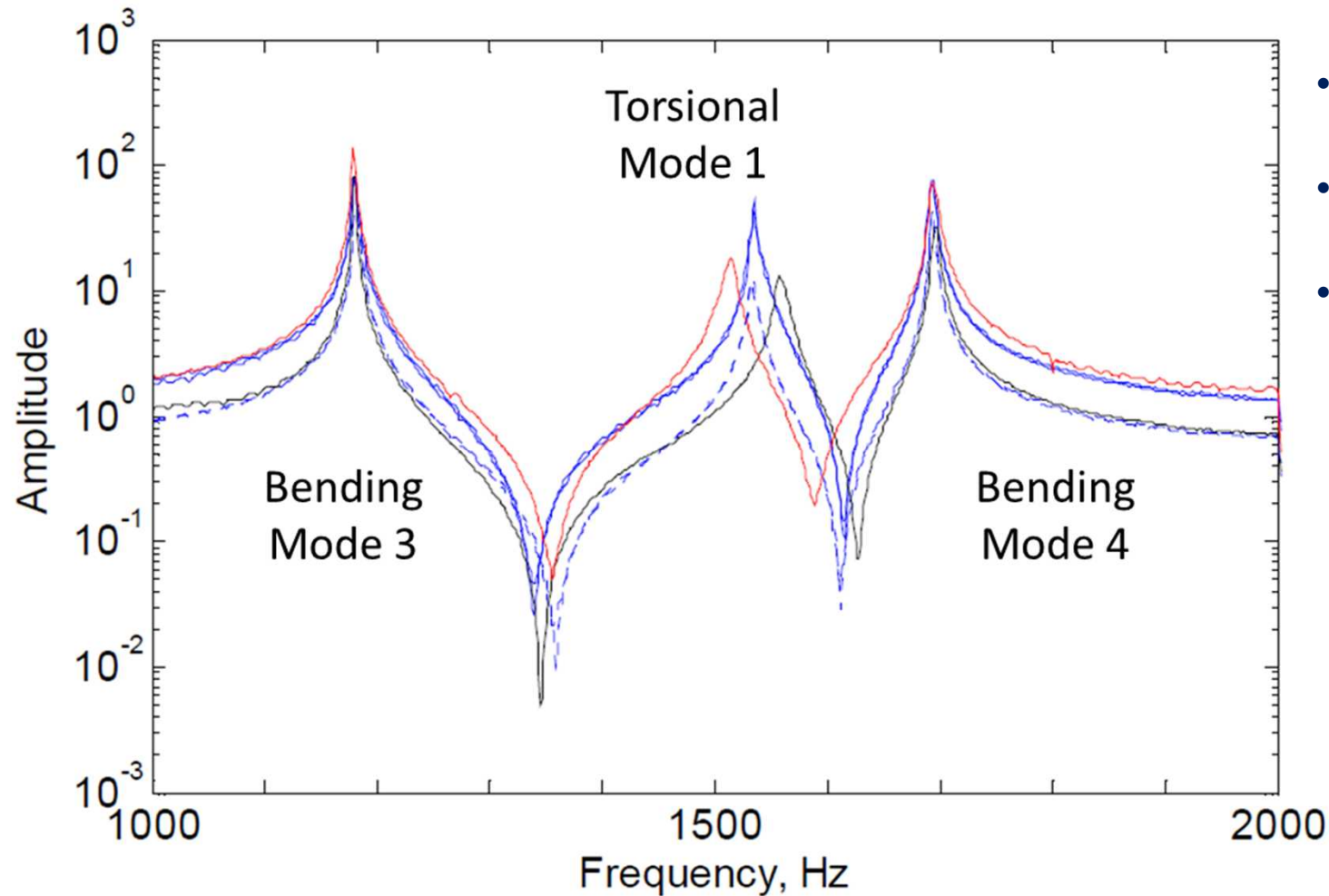


- Brake-Reuß beam: 72 cm long beam with a three bolt lap joint.
- Multiple versions to assess contribution of joint to dynamics.



# Introduction

## → Motivation for Current Research



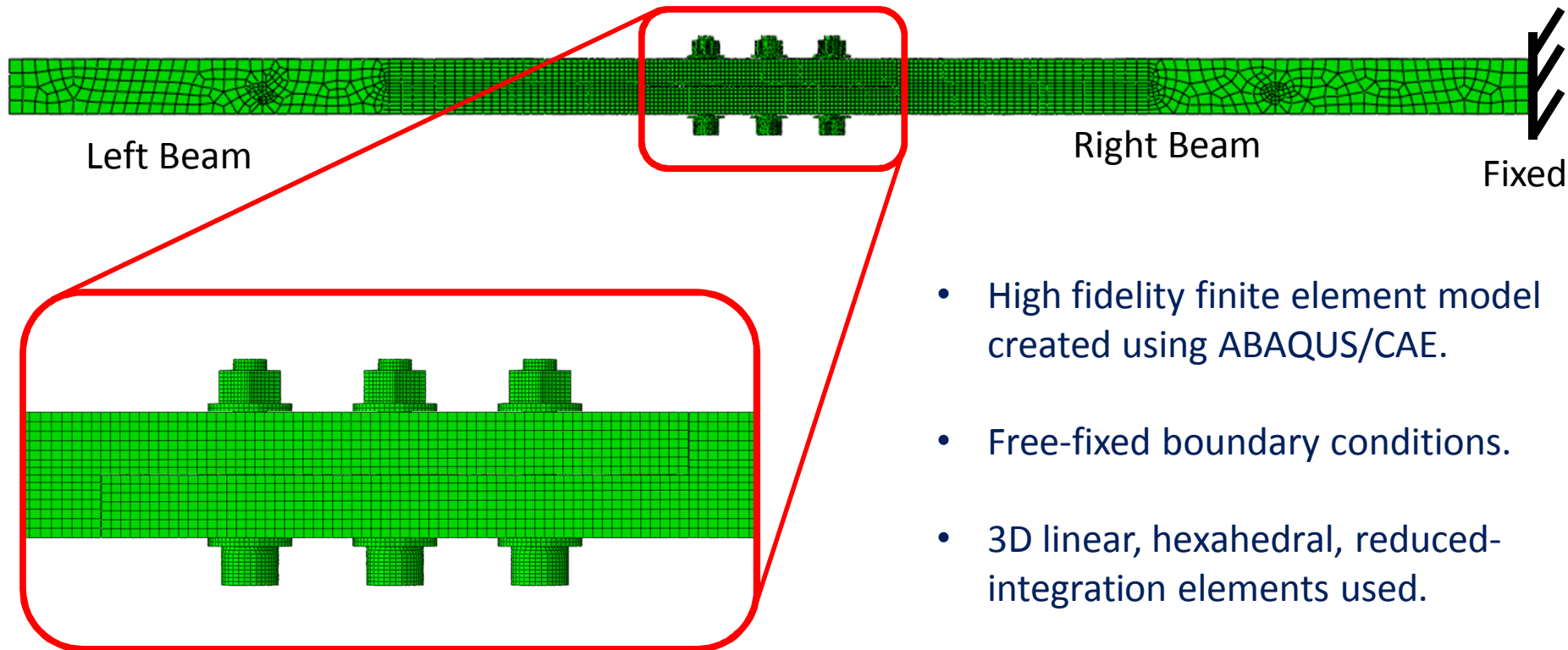
- 5 Nm torque
- Free-free BC
- Off-center shaker excitation

- Torsional modes very sensitive to preloads and tightening order and lack repeatability.



# Introduction

## → Modeling Approach



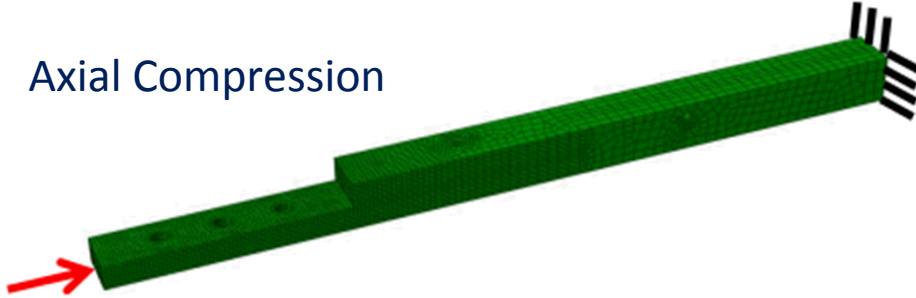
- High fidelity finite element model created using ABAQUS/CAE.
- Free-fixed boundary conditions.
- 3D linear, hexahedral, reduced-integration elements used.
- Nuts, bolts, and washers modeled separately from beams, but connected using node-tie constraints.
- Surface-to-surface contact pair with standard Coulomb dry friction model,  $\mu = 0.3$ .
- Bolt load of 4 kN applied to each bolt.



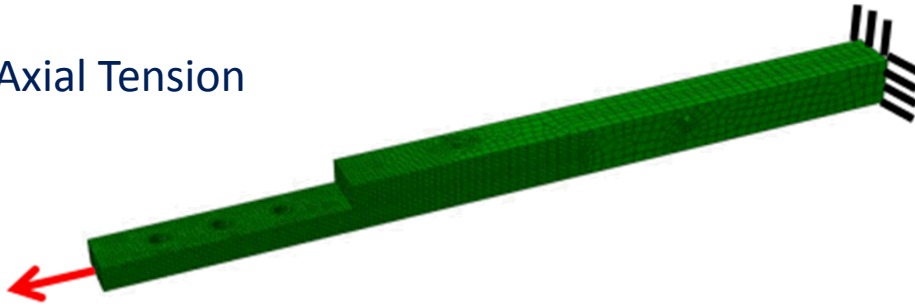
# Simulation Phase I

## → Residual Stress Application

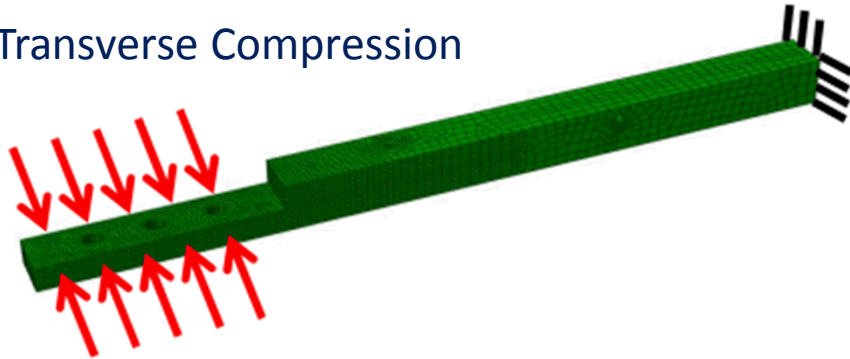
Axial Compression



Axial Tension



Transverse Compression

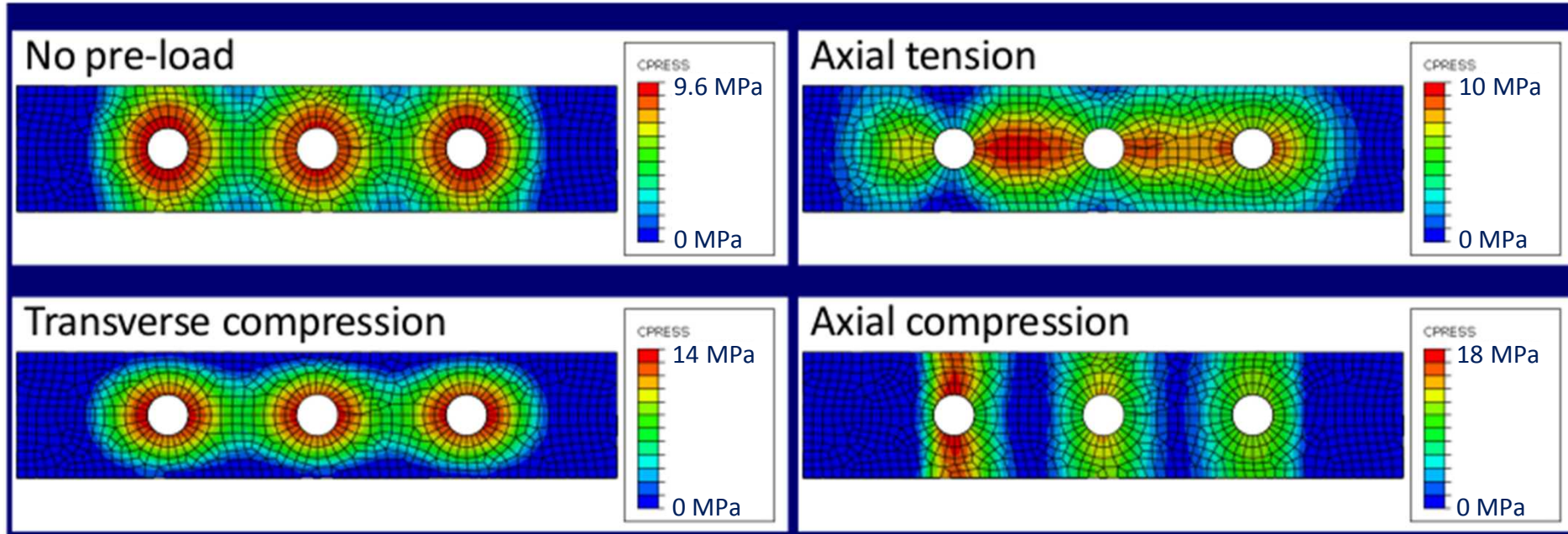


- Three types of residual stresses investigated
- Residual stresses installed by
  1. Apply pre-load to right beam
  2. Apply bolt load
  3. Release pre-load
- All pre-loads applied using specified displacements.



# Simulation Phase I

→ Results: Residual Stress Effect on Contact Pressure

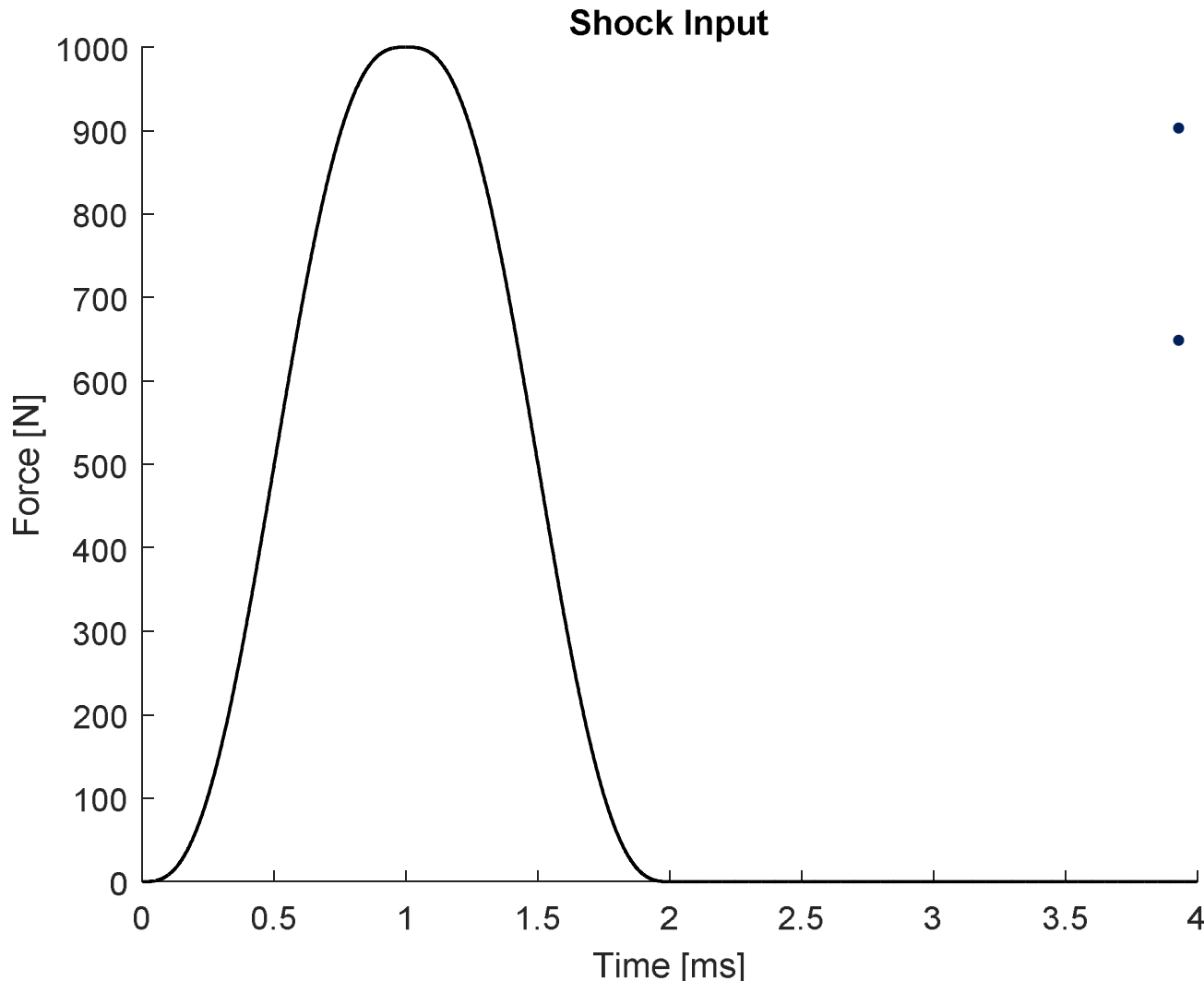


- Pre-load significantly modifies the contact's behavior.
- Axial tension and compression have opposite effects.
- Transverse compression has least effect.
- Axial compressions results in separation bands.



# Simulation Phase II

→ Shock Input

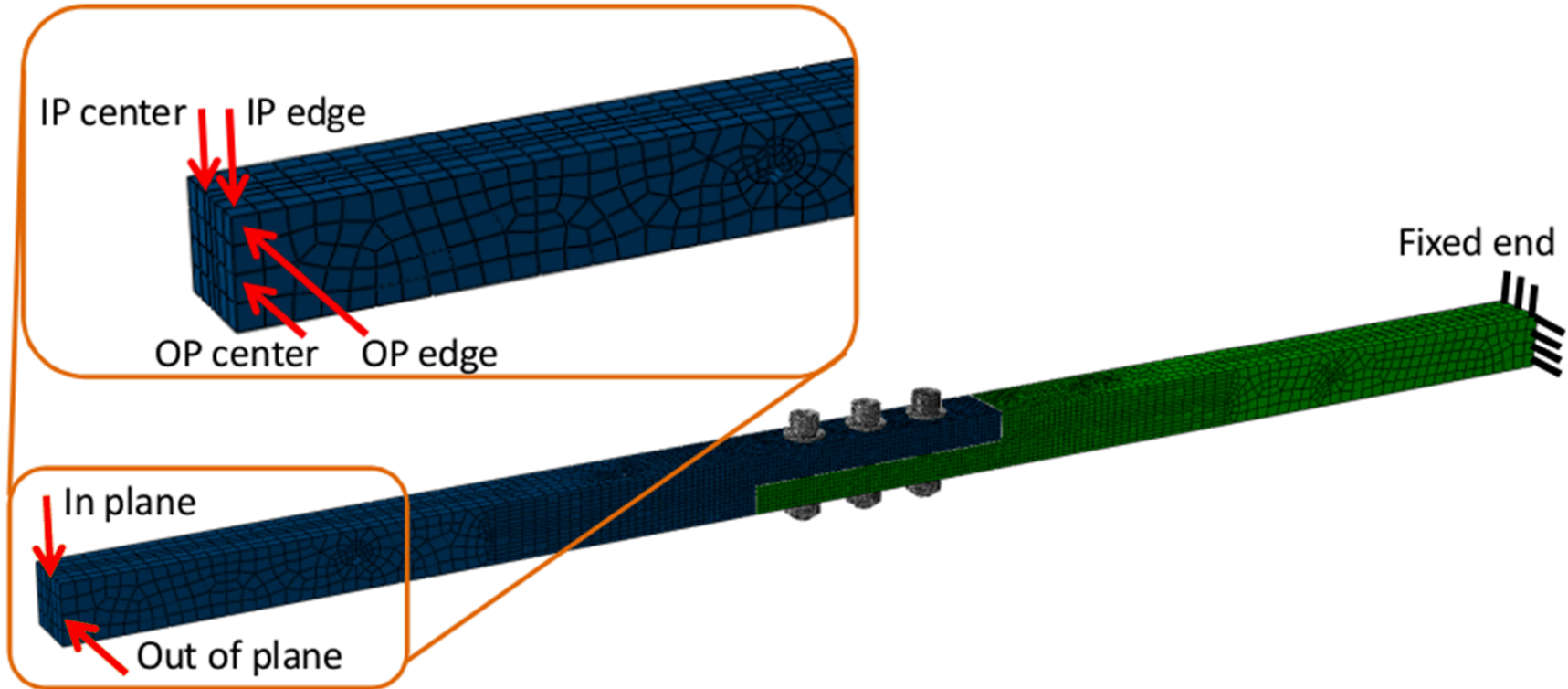


- Shock inputs applied using ABAQUS/CAE Smooth Step amplitude command.
- Work done kept constant by varying amplitude and duration of shock.



# Simulation Phase II

→ Shock Input Locations



- In plane and out of plane loadings considered – beam is stiffest in plane.
- Center and edge shocks considered to compare bending and torsional effects.

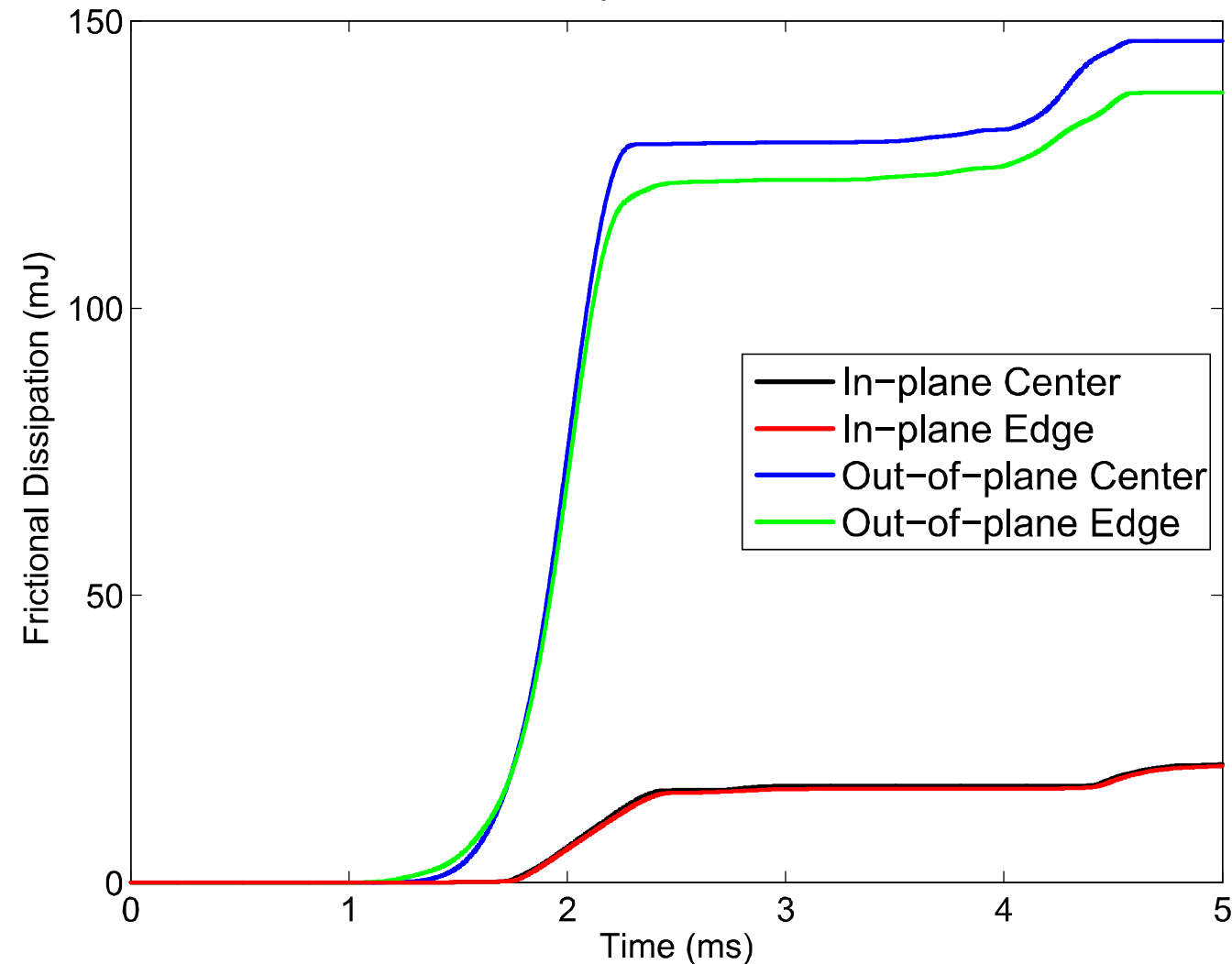




# Simulation Phase II

→ Results: Shock Location – Low Amplitude Shock

1kN Impact, 2ms Pulse



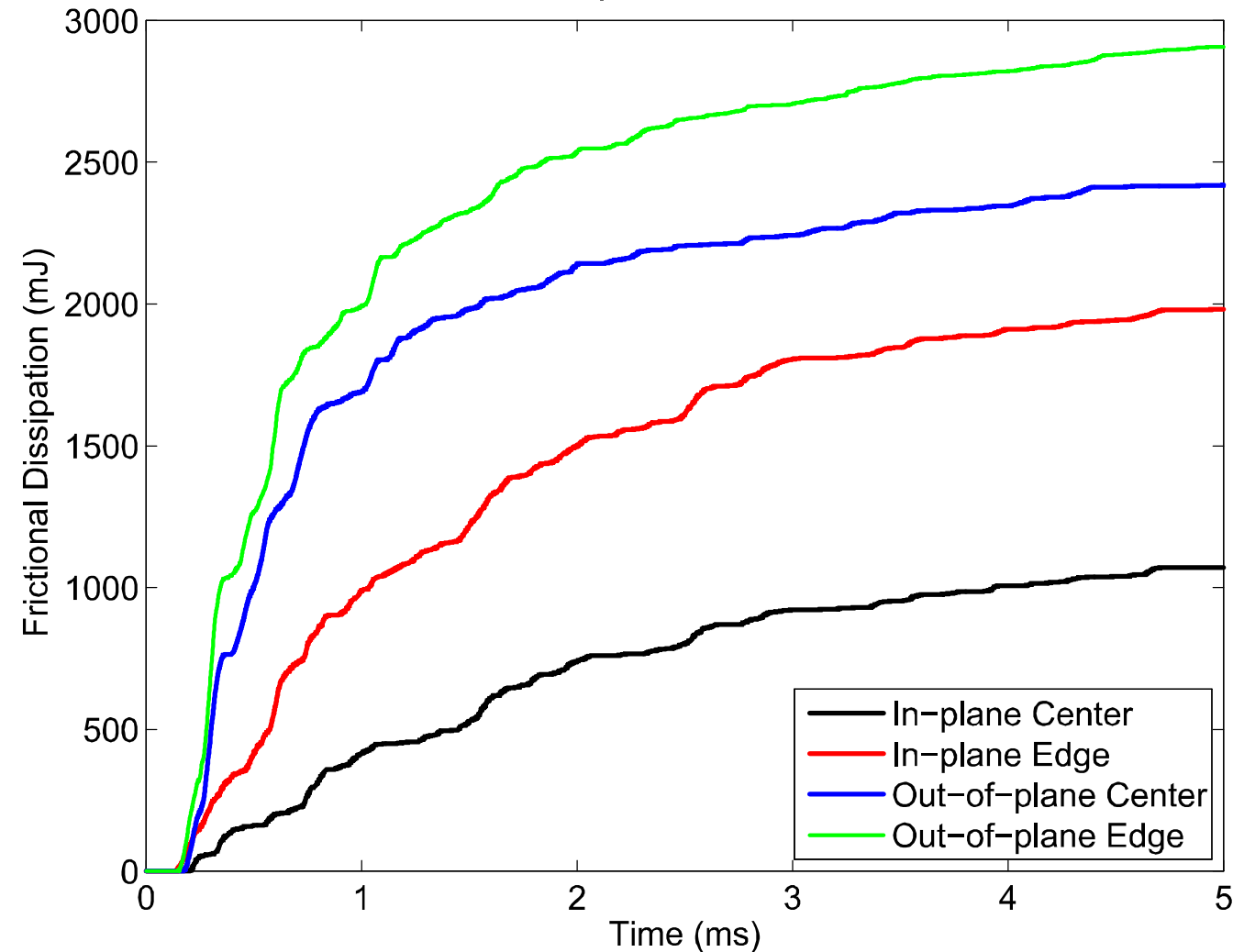
- Out-of-plane impacts result in higher frictional dissipation.
- Small difference between center and edge impacts.
- Torsional effects minimal



# Simulation Phase II

→ Results: Shock Location – High Amplitude Shock

20kN Impact, 0.1ms Pulse



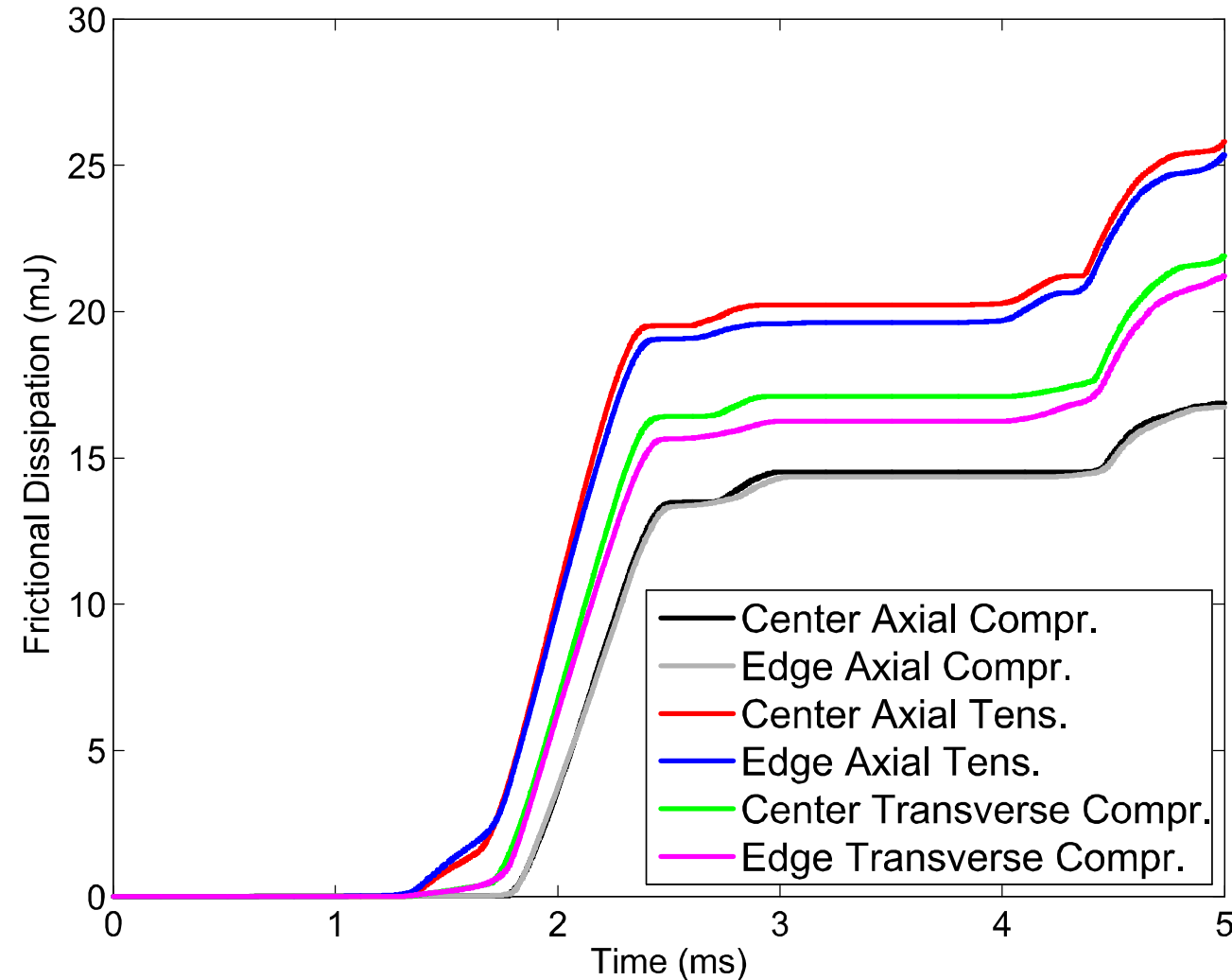
- Out-of-plane impacts result in higher frictional dissipation.
- Large difference between center and edge impacts.
- Torsional effects more prominent.



# Simulation Phase II

→ Results: Residual Stress – Low Amplitude Shock

In-Plane 1kN Impact, 2ms Pulse



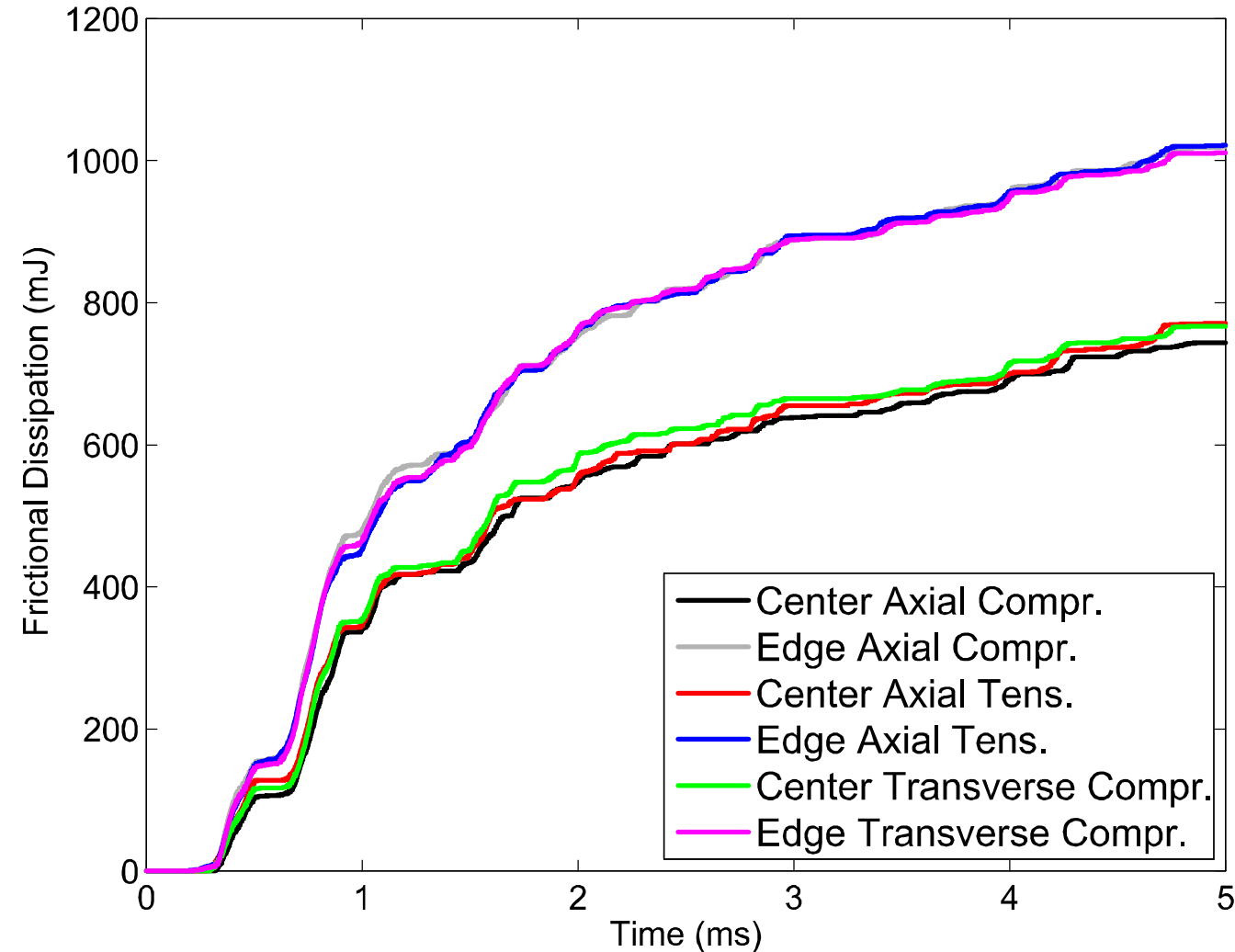
- Residual stress has some effect.
- Small difference between center and edge impacts.
- Torsional effects minimal.



# Simulation Phase II

→ Results: Residual Stress – High Amplitude Shock

In-Plane 20kN Impact, 0.1ms Pulse

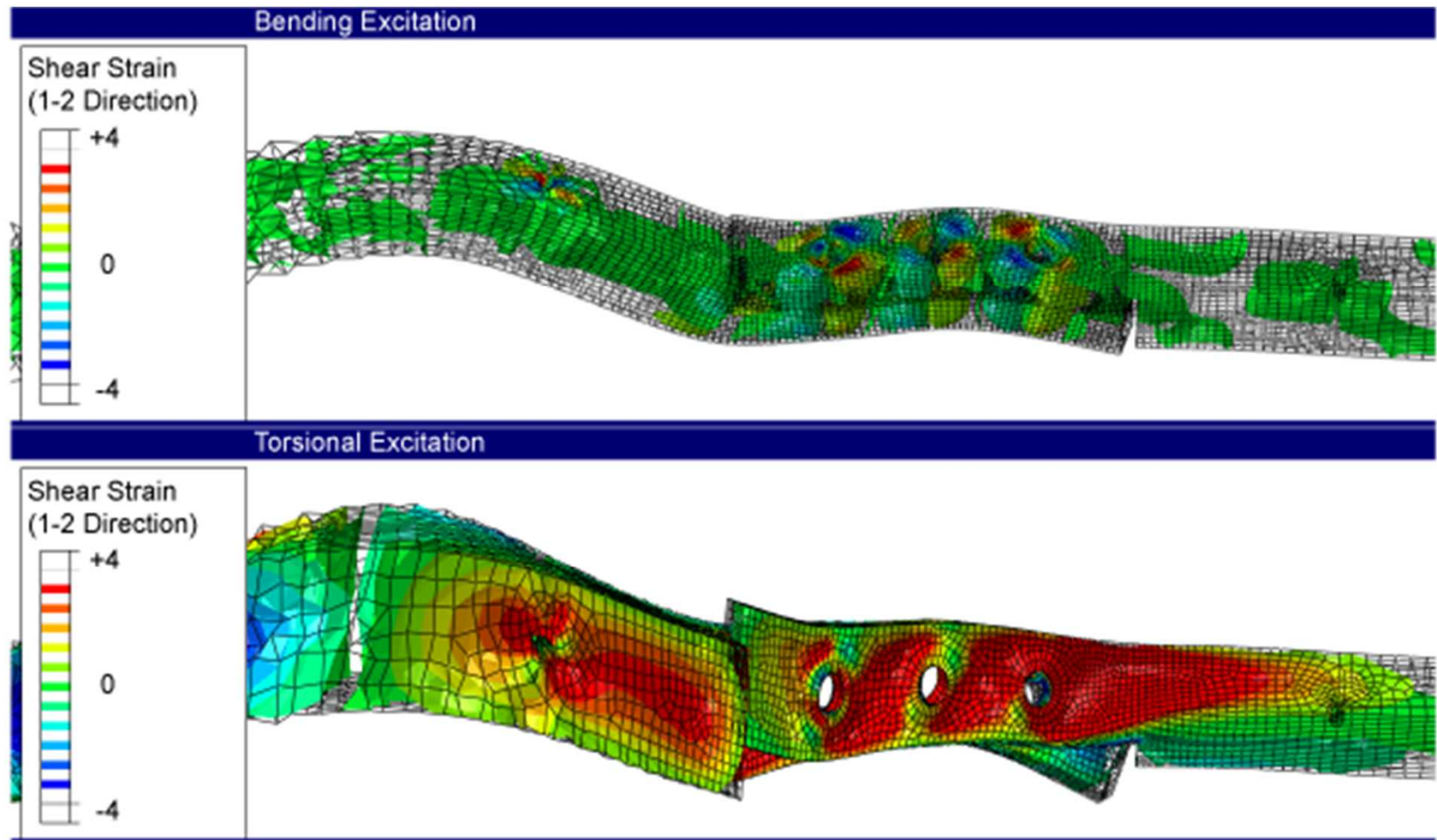


- Residual stress has minimal effect.
- Large difference between center and edge impacts.
- Torsional effects more prominent



# Simulation Phase III

→ Results: Bending vs. Torsional Excitation



- Large shear contributions under torsional excitation.



# Summary

- High fidelity finite element model used to simulate dynamic shock response of Brake-Reuß beam.
- Residual stresses have great effect on contact pressure – axial compression results in separation bands.
- Shock loading location varied to study in plane and out of plane bending and torsional effects. Work done kept constant by varying duration and amplitude.
- At low shock amplitudes,
  - Out of plane impacts result in higher dissipation
  - Torsional effects minimal
  - Residual stress effects more prominent
- At high shock amplitudes
  - Out of plane impacts result in higher dissipation
  - Torsional effects more prominent
  - Residual stress effect minimal



# References

1. “The Mechanics of Jointed Structures: Research on the Joint Challenges Defined at the International Workshops on the Mechanics of Jointed Structures,” Edited by M.R.W. Brake, *Springer*, 2016.
2. M. R. W. Brake, P. Reuss, C. W. Schwingshackl, L. Salles, M. E. Negus, D. E. Peebles, R. L. Mayes, J.-C. Bilbao-Ludena, M. S. Bonney, S. Catalfamo, C. Gastaldi, J. Gross, R. M. Lacayo, B. A. Robertson, S. Smith, C. Swacek, and M. Tiedemann, “The 2014 Sandia Nonlinear Mechanics and Dynamics Summer Research Institute,” 2015, *SAND2015-1876*, Sandia National Laboratories, Albuquerque, NM.
3. M. R. Brake, P. Reuss, D. J. Segalman, and L. Gaul, “Variability and Repeatability of Jointed Structures with Frictional Interfaces,” *IMAC XXXII A Conference and Exposition on Structural Dynamics*, Orlando, FL, February 2014.