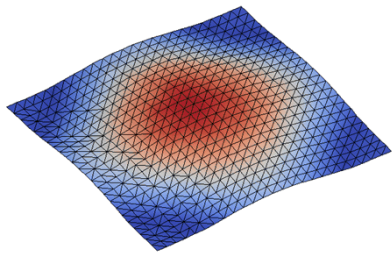
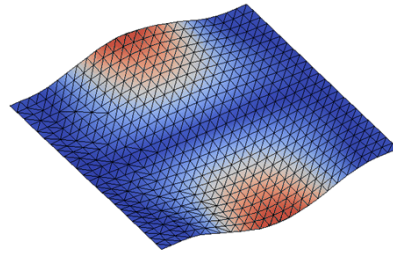


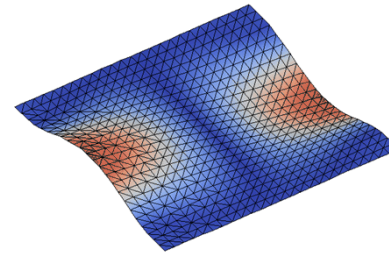
*Exceptional service in the national interest*



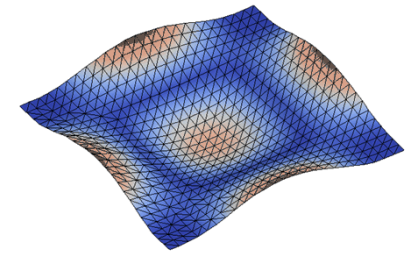
524.283203125 Hz



874.781677246 Hz



883.254394531 Hz



1405.89904785 Hz

# Stereo-DIC vs 3D Scanning LDV for Modal Testing

Which one do I choose?

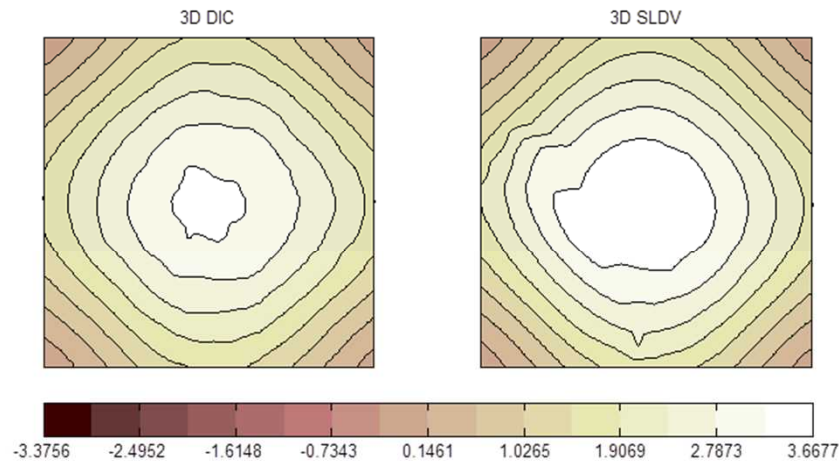
Phillip L. Reu, Daniel Rohe, Laura Jacobs and Dan Turner

1<sup>st</sup> Annual International DIC Society Meeting  
November 3-5, 2015, Columbia, SC

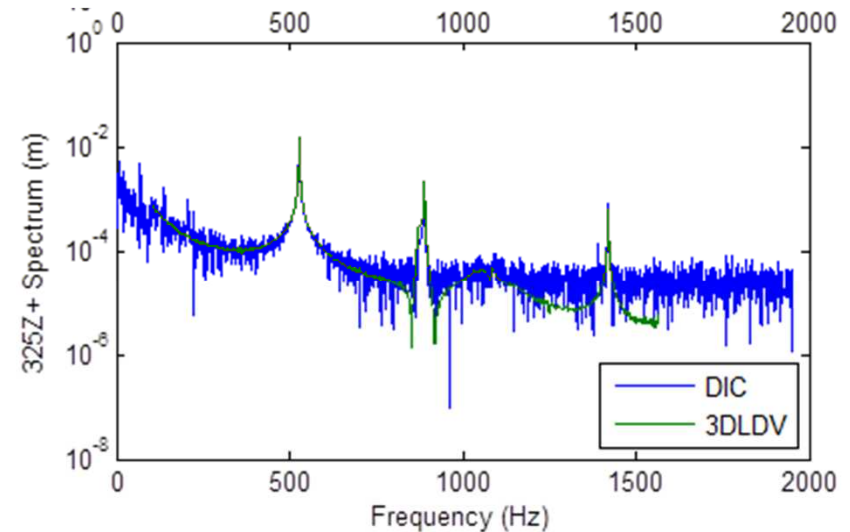


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# The goals of modal testing are determining the mode shape, frequency and damping.



Mode Shape  
Modal Frequency = 529Hz



Damping = 0.0004

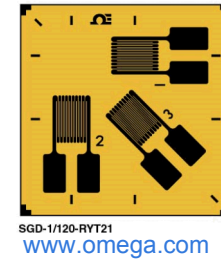
## Modal Testing Reveals

- Frequencies that the object will vibrate (resonance) at and are to be avoided.
- The damping of the structure – how quickly vibrations die out.
- Needed for design of systems and structures.
- Can be used for FE model validation.
- Strain hot-spots or places for failure in the system.

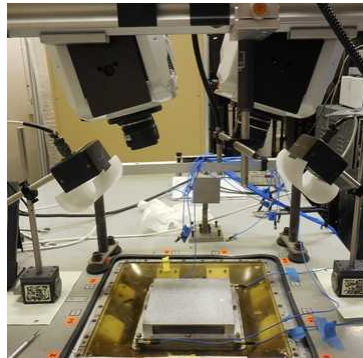
# There are a number of technologies that can be used for modal testing.



## Excitation Methods



## Single point measurement methods



## Non-contact and full-field methods

### Pros and Cons

- Strain gages are typically cheap and easily used.
- Accelerometers may load light structures.
- Single point measurements may miss the peak location.
- Full-field methods are expensive in cost and time (maybe).

# For many experiments full-field methods are preferable (if you have the money).

## Why full-field?

- Captures peak location.
- Allows visualization of the entire mode shape.
- Does not mass-load the structure.

## Experimental Setup

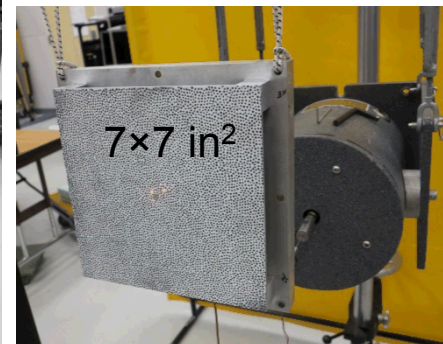
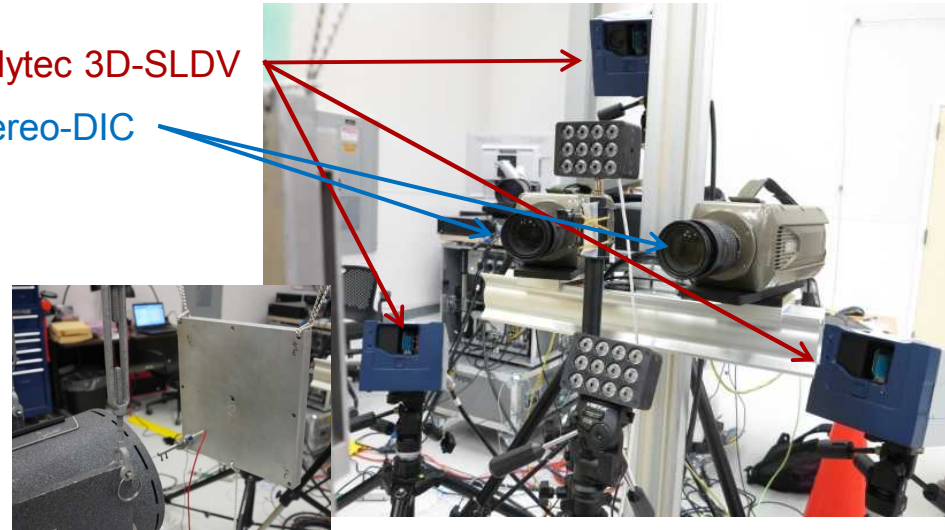
- Polytec PSV-500 3D-Scanning LDV system.
- Vic3D and Phantom 611 Cameras (800×800)
- 3906.25 Hz (200  $\mu$ s exposure) to match LDV
- MB-50 Shaker on a shaker stand (Pseudo-Random)
- Speckle painted surface (not ideal for LDV)
- Retro surface (not possible for DIC)

## What we investigated

- Time to measurement (setup)
- Cost (not really)
- Measurement time
- Analysis time/data point
- Noise floor/resolution
- Ease of use (subjective)

Polytec 3D-SLDV

Stereo-DIC



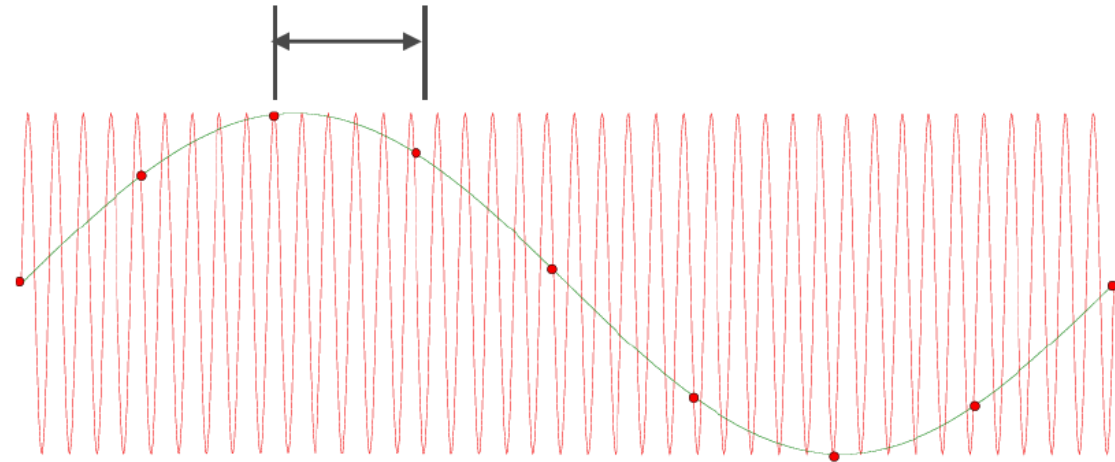


# Other possibilities for harmonic motion. But not for random or impact results.



[www.correlatedsolutions.com](http://www.correlatedsolutions.com)

Camera inter-frame time.



## Strobe Applications

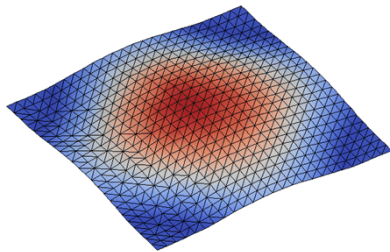
- Harmonic motion only
- Sine drive
- Allows high-resolution cameras
- Exposure controlled by strobe
- Slip-sync with DAQ hardware

# Time to solution: When do you want to wait?

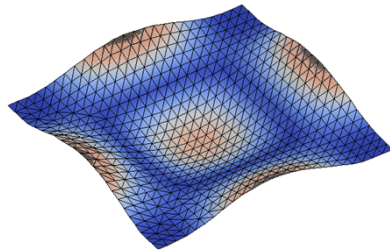
## During the test or after.

### LDV solution time

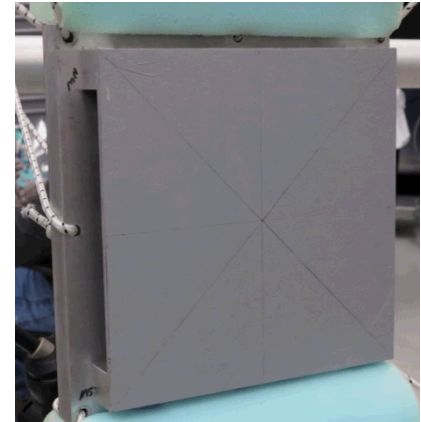
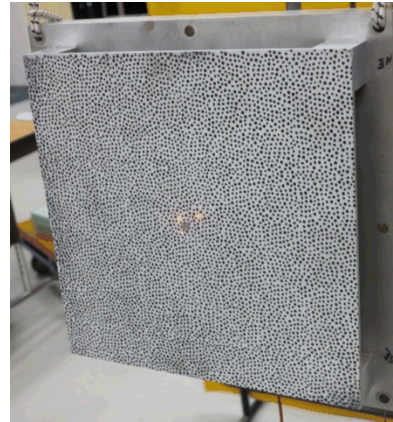
- Scanning for **545** points took 45 minutes
- For 715 points (to match DIC) it would be 60 minutes.
- Data storage is minimal if only the frequency data maintained.



524.283203125 Hz



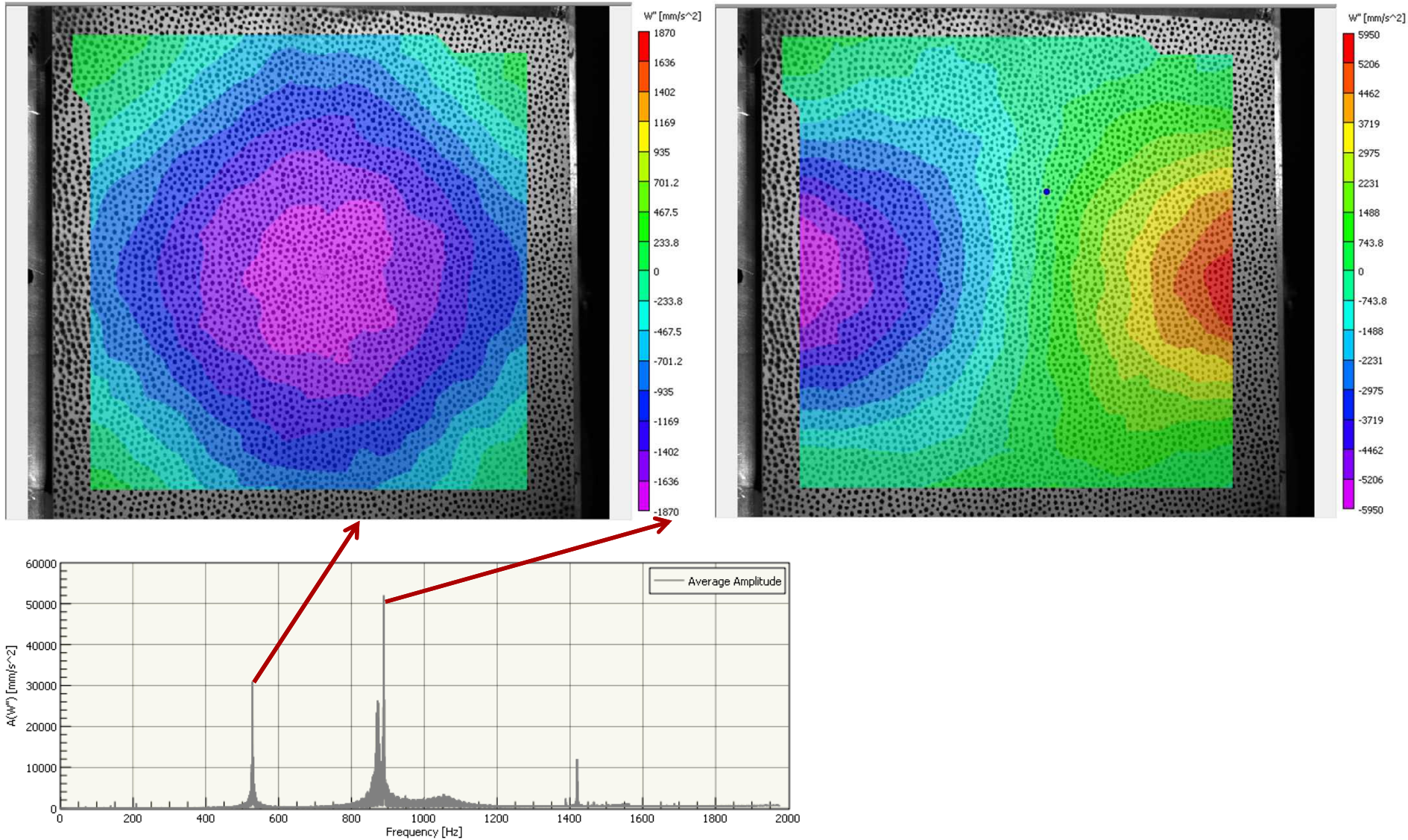
1405.89904785 Hz



### DIC solution time

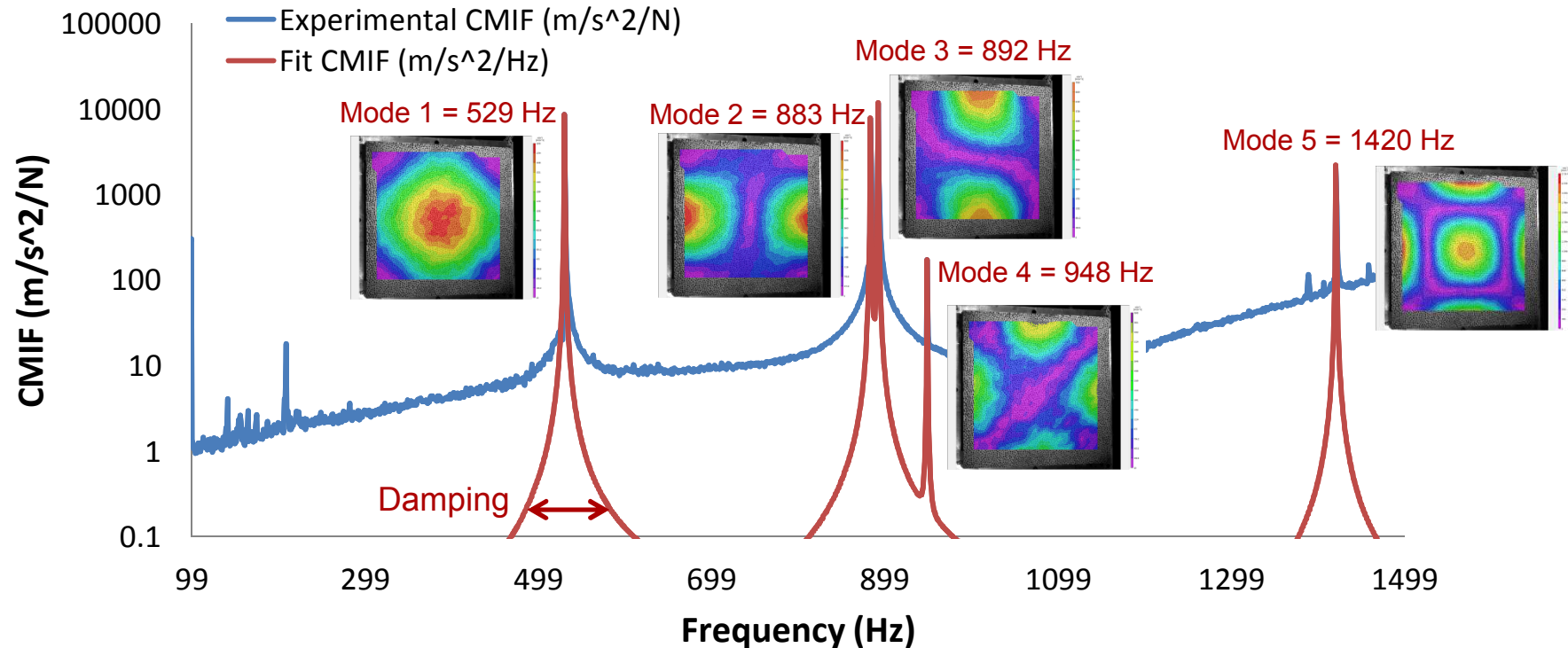
- Data collection is 6 seconds ( $800 \times 800$  pixels<sup>2</sup> or 640,000 points)
- Download: 23,723 8-bit images from two cameras  $\approx 20$  minutes
- Data Storage is 16-Gbytes of data ( $\times 2$ )
- **715** Data points solved in 1.5 hours
- Subset = 51, Step = 25, Strain Window = 15, NSSD, 4-Tap, No filtering

# Native FFT analysis in Vic3D





# Modal fitting analysis approach.



## Modal Fitting Software

- SMAC<sup>†</sup> – Sandia's modal identification software
- There was some challenge with the large number of data points from the DIC software.
- CMIF is the Complex Mode Indicator Function.

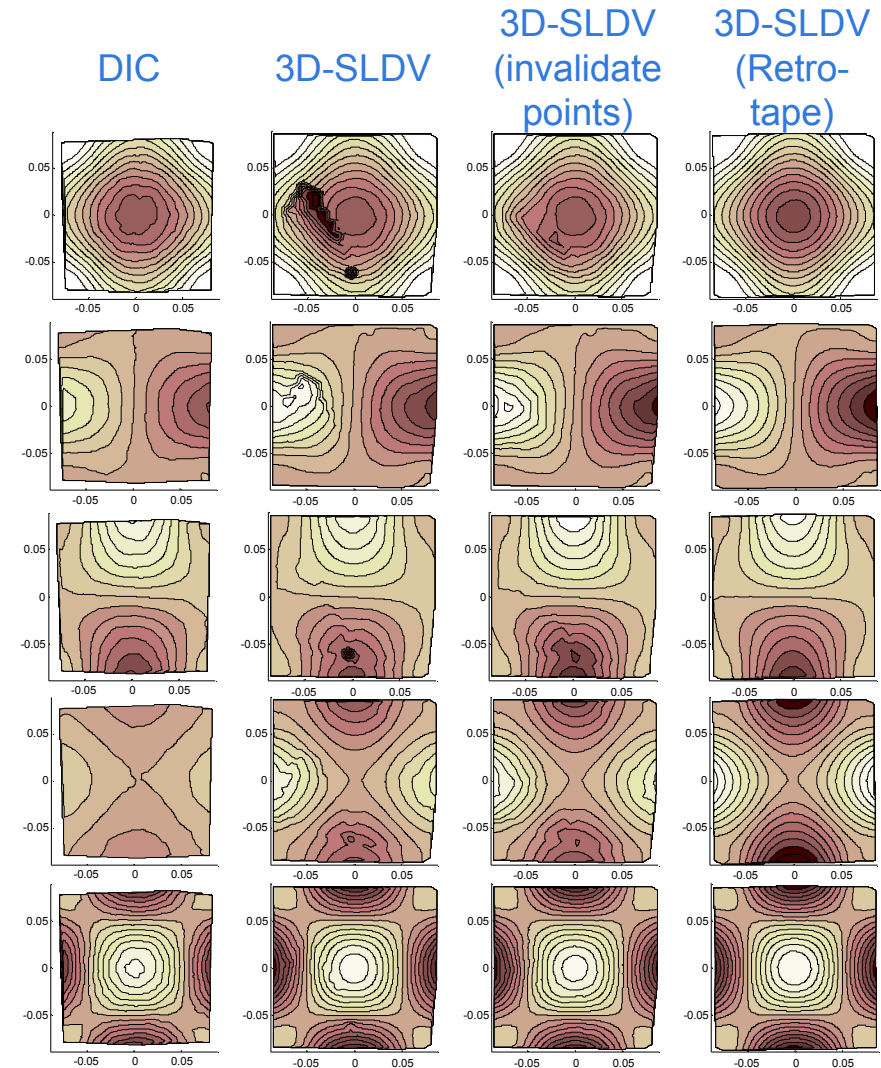


# Fitting results and comparisons. Show some mode shapes, damping and frequencies.

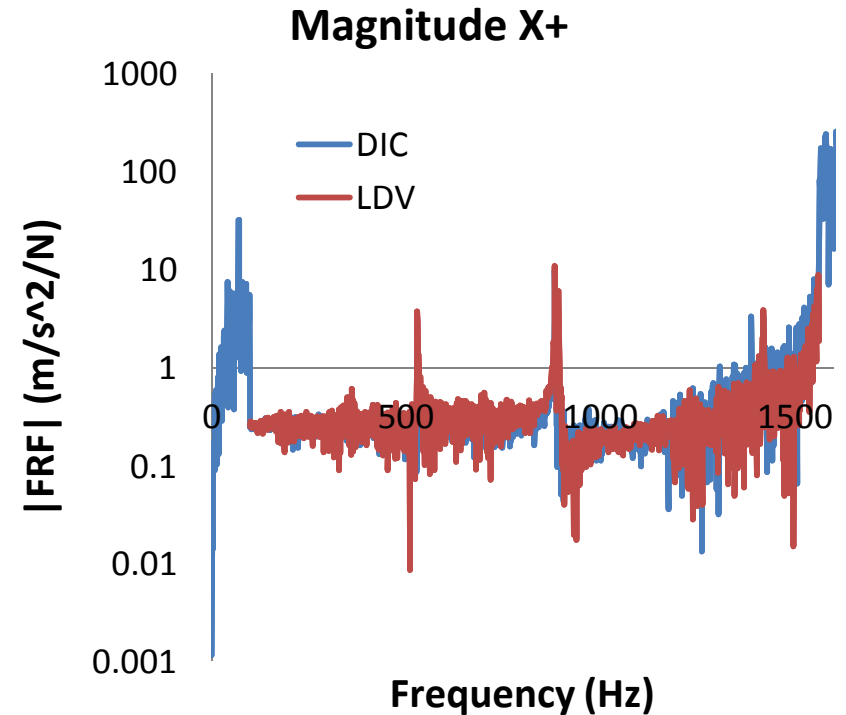
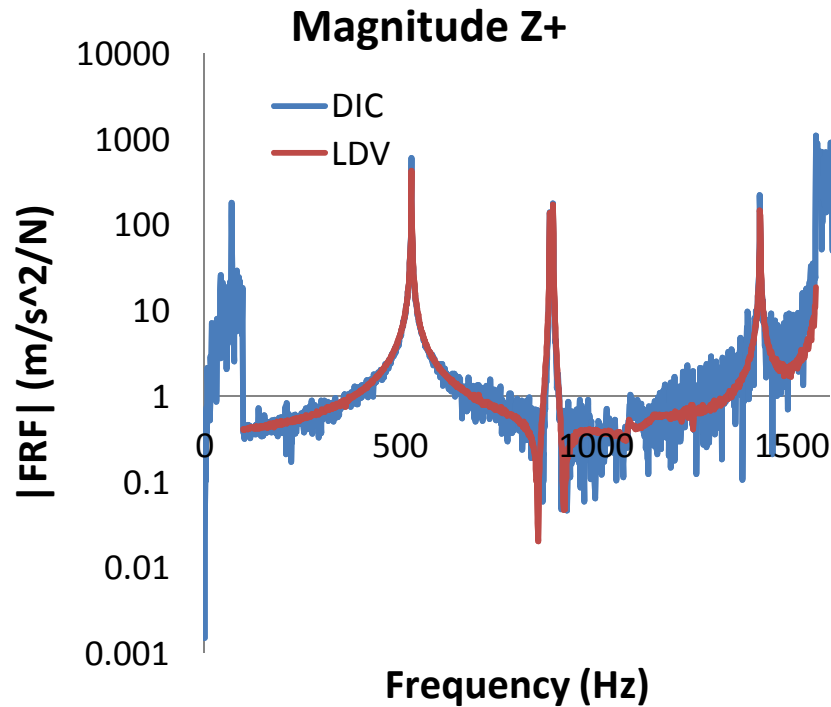
Mode	Frequency		Damping	
	DIC	LDV	DIC	LDV
1	529.4	529.5	0.042	0.042
2	883.0	883.1	0.022	0.024
3	891.7	891.7	0.015	0.026
4	948.3	948.4	0.019	0.019
5	1420.1	1420.4	0.015	0.020

## Notes on the Results

- DIC = 715 Points, LDV = 545 Points
- Scale identical for each group of results
- DIC plots are from the speckled surface.
- 3D-SLDV are taken on the speckle surface.
- 3D-SLDV (invalidate points) removes “bad” data points from the analysis.
- 3D-SLDV (retro-tape) covered the speckle pattern with a retro tape to improve signal quality.



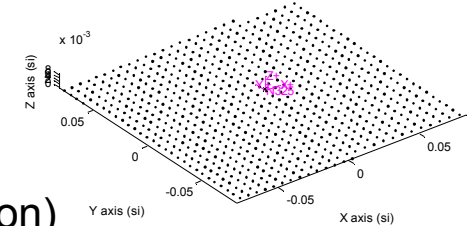
# LDV has a better noise floor than DIC in Z and comparable in X and Y.



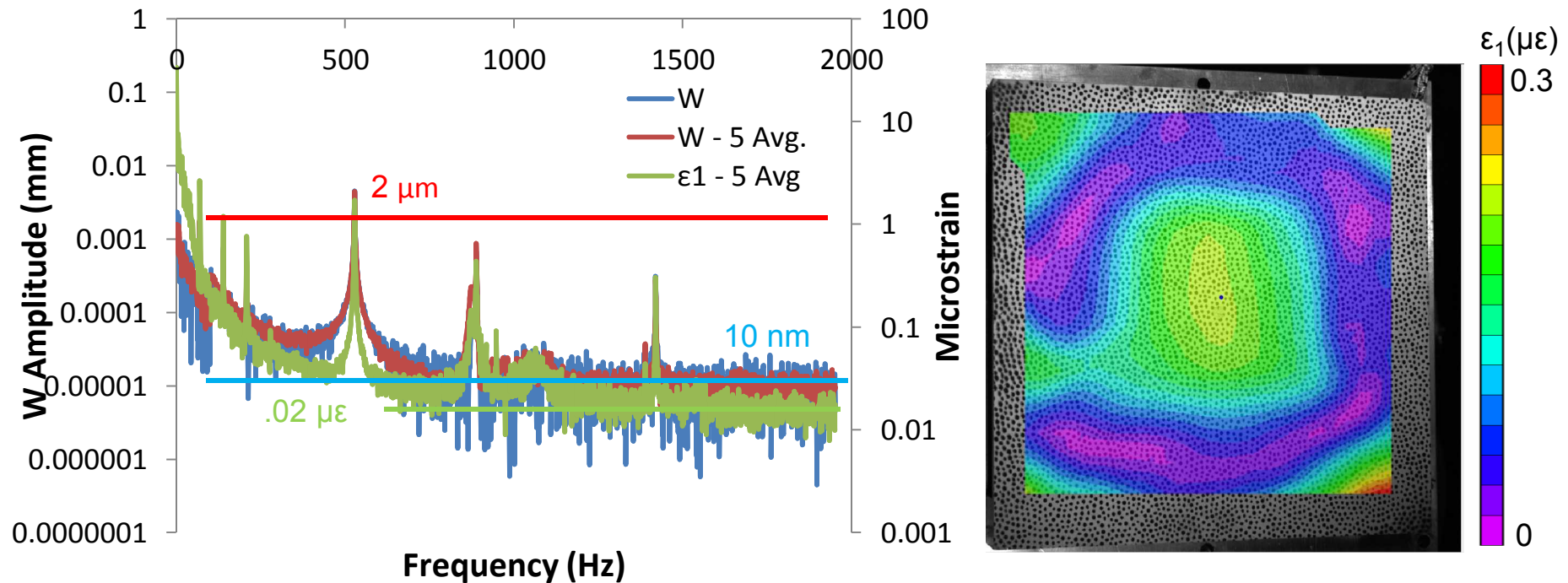
## Notes on the comparison

- Data at point near the middle of the plate.
- LDV is more sensitive out-of-plane, DIC is less sensitive.
- Not much in-plane motion at center (not important for this comparison)

DIC & SLDV Point Plotted Above



# The DIC noise floor is significantly better in the frequency domain than the time domain.

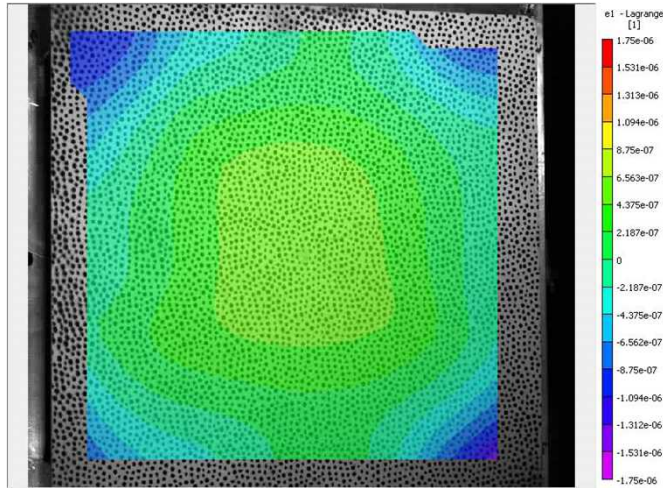


## Notes on Resolution

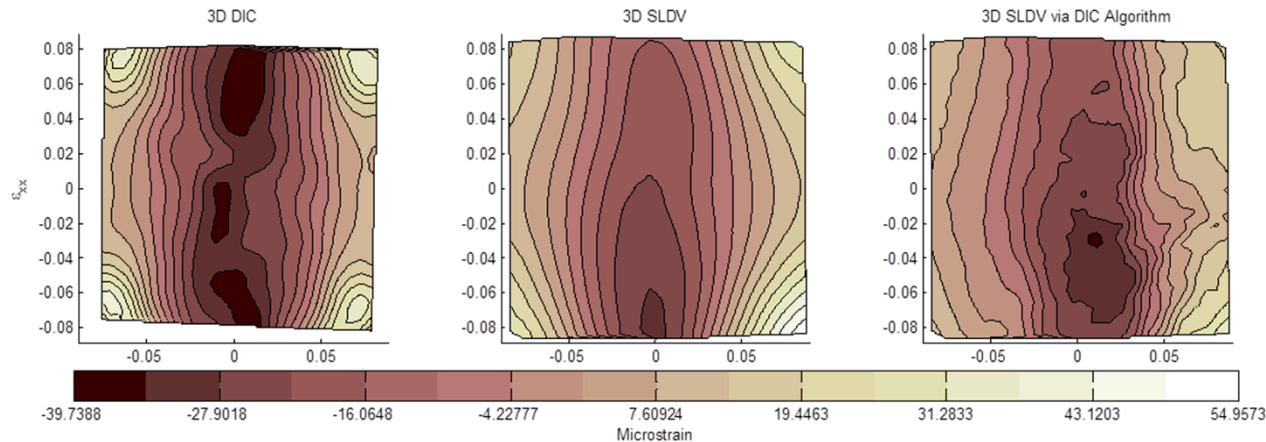
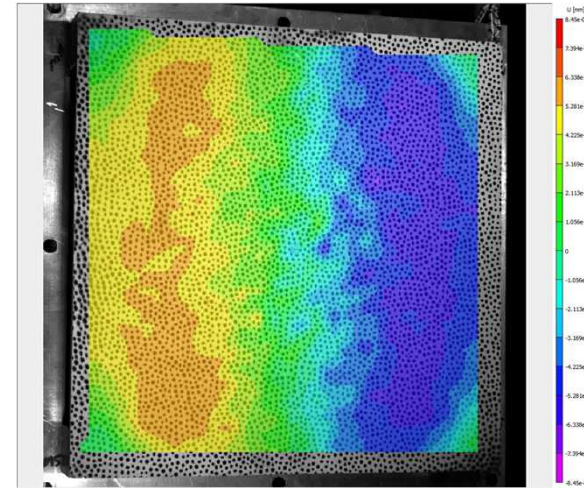
- Typical DIC resolution for a 7 inch FOV would be 2  $\mu\text{m}$  or 200  $\mu\epsilon$ .
- The noise is distributed across all the frequencies – therefore lower at any given frequency band.
- Displacement resolution in the frequency domain is much better than that in the time domain.
- FFT averaging improves the noise even further (i.e. more images are better!), i.e. you can trade frequency resolution for noise reduction (i.e. more averaging).

# Strain calculations: Should we do this in the modal fit space? Is it the same?

$$\epsilon_1 = \pm 1.75 \mu\epsilon$$



$$U = \pm 84.5 \text{ nm}$$



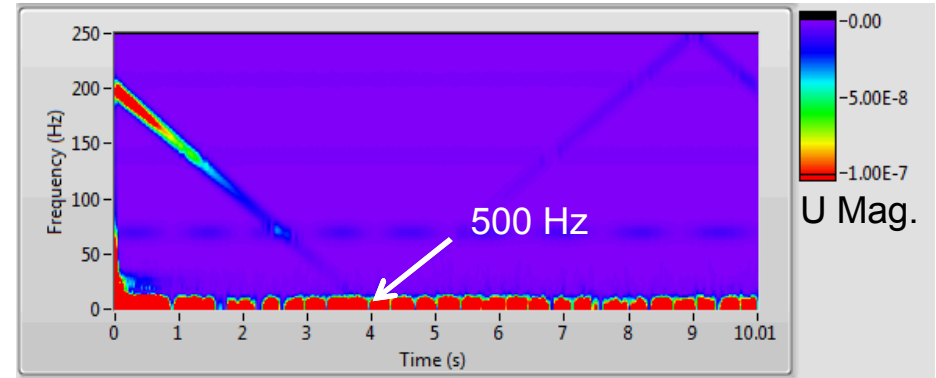
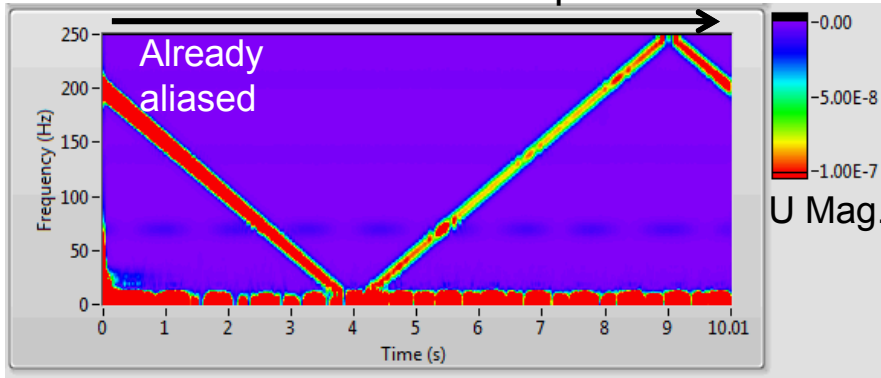


# Temporal<sup>†</sup> anti-aliasing. It can be a big deal.

Frame Rate = 500 Hz: Nyquist = 250 Hz

<sup>†</sup>Spatial aliasing is another completely different and important topic.

300 – 800 Hz Sweep



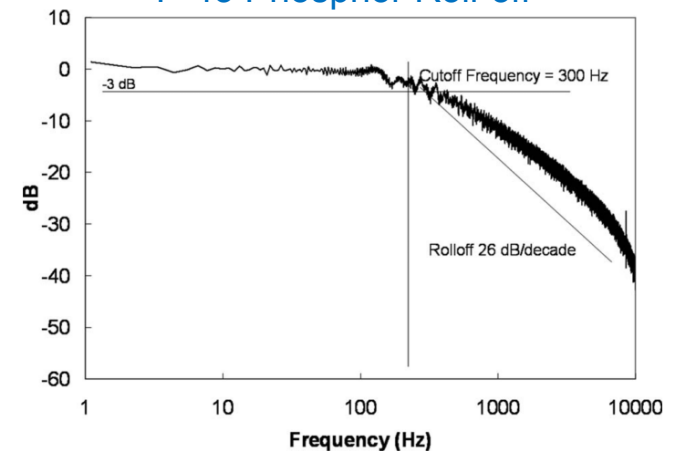
Exposure = 200  $\mu$ s or 5000 Hz

Exposure = 1900  $\mu$ s or 526 Hz

## Notes on aliasing









- With cameras – there are no good antialiasing filters. (Ideas do exist<sup>†</sup>)
- Long exposures provide some aliasing protection.
- Best solution: have a single point measurement (with antialiasing) to ensure the frequency content of the signal.
- With impact testing – the force profile rolls off gradually with frequency and will lead to aliasing issues.

## P-43 Phosphor Roll-off



<sup>†</sup>Reu, P. L. and B. D. Hansche (2008). "Optical temporal frequency low-pass filtering and heterodyning with a microchannel plate." Optical Engineering **47**(7).

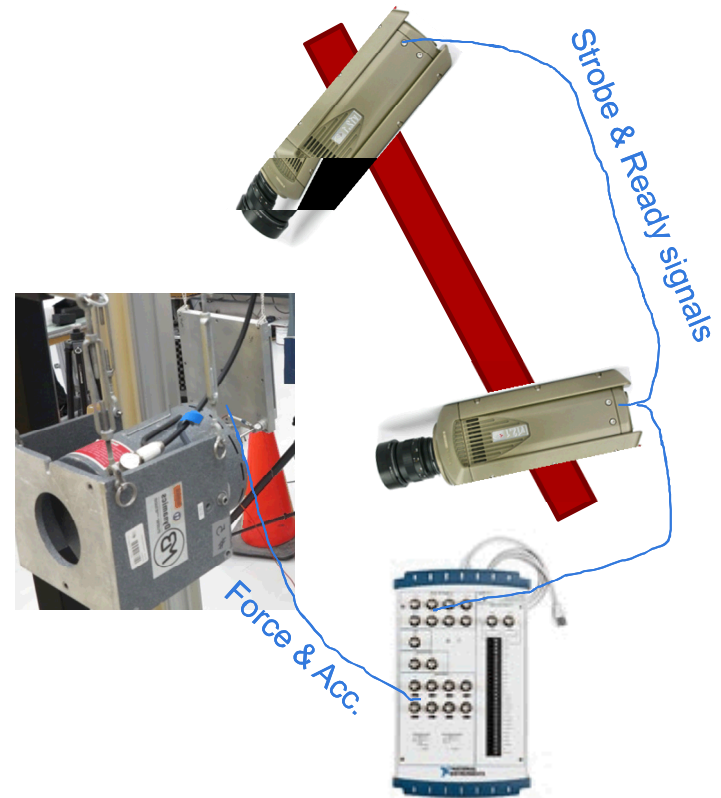
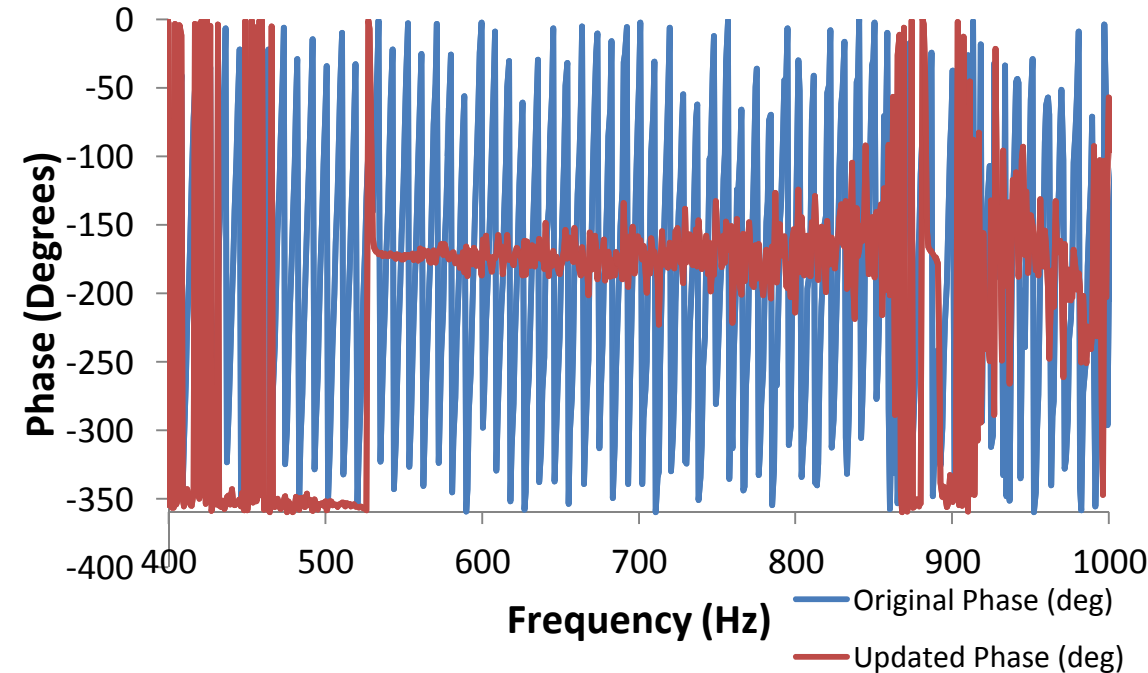
# What are the pros and cons of the two methods. When do we use one or the other.

Comparison Metric <sup>†</sup>	LDV	DIC
Cost	≈\$650k	≈\$350k 
Setup time	2 hours	2 hours
Acquisition Time	Hours	Seconds 
Analysis Time	Seconds 	Hours
Disp. Resolution	≈ picometers 	≈ nanometers
Strain resolution	?	5 microstrain 
Strain Calculation	Integrated – but researchy	Seamlessly Integrated 
Anti-aliasing	Included 	Not possible at the moment
Data volume	Small (Mbytes) includes only frequency data	Large (Gbytes) but includes time history
Software	Designed for modal analysis. 	In its infancy.

## Questions

<sup>†</sup>The comparison is for this test setup – but should be broadly accurate in many other situations.

# The drive signal must be acquired synchronously with the images.



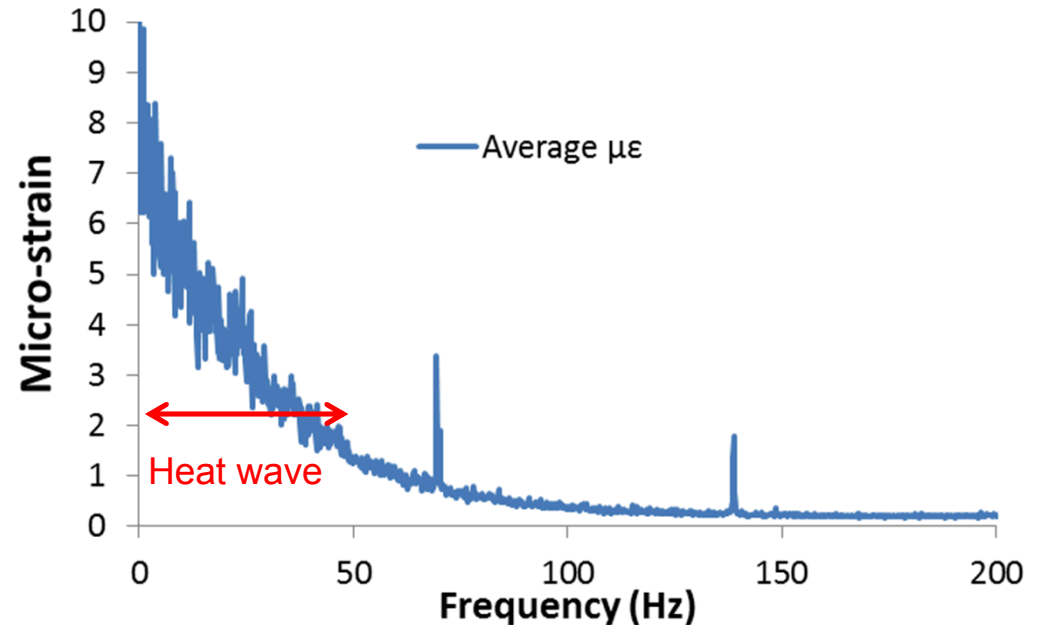
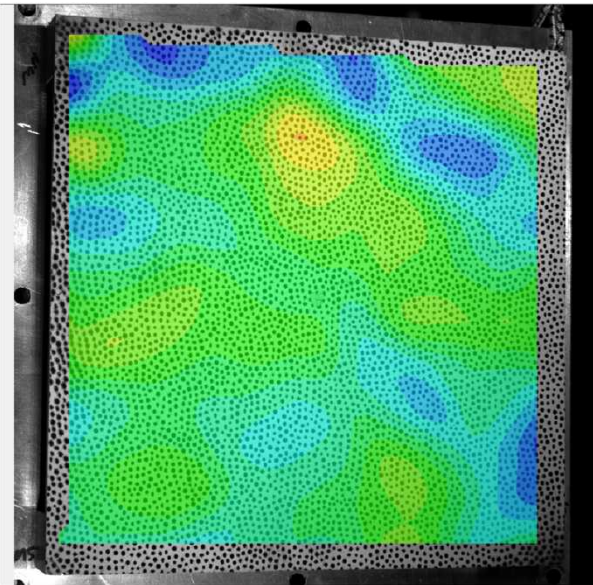
## High-Speed Camera Setup

- Must synchronize cameras with DAQ
- Measure input signals to system
- Shaker input control (sine or pseudorandom)

Phase shift in the camera DAQ was discovered and fixed by a simple linear phase shift. Not sure where this came from in our experiment.

# The influence of heat waves is in a specific frequency band.

$\epsilon_1$  at 8 Hz



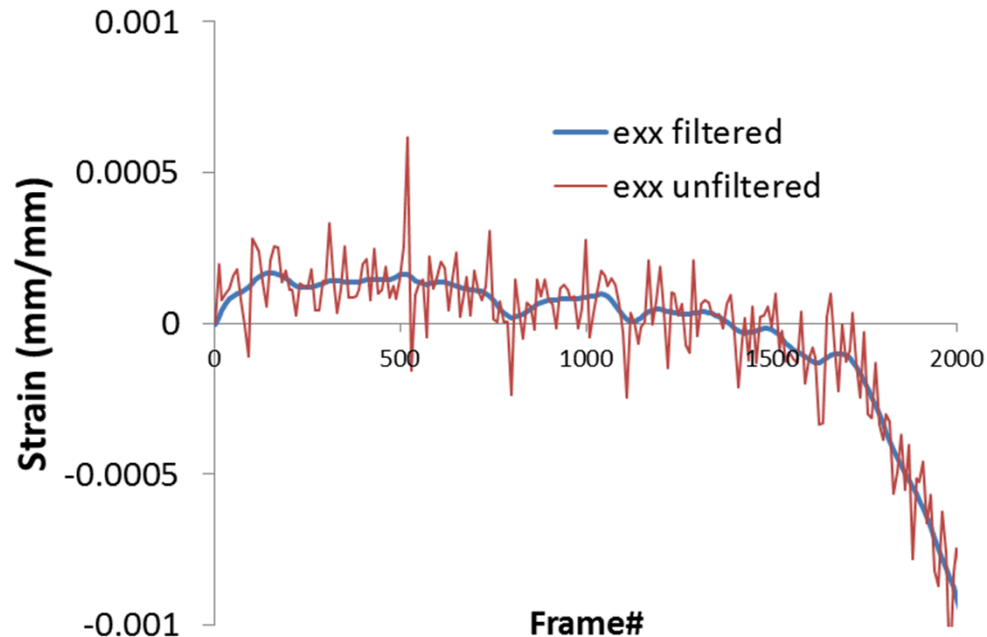
## Notes on heat waves

- The heat causes index of refraction changes and therefore “optical distortions”
- Contained in the 5 – 70 Hz band (for this experiment)
- For quasi-static testing – low pass filter the results using extra images (next slide)
- For dynamic testing – Use the frequency domain



# For quasi-static testing – temporal averaging (filtering) can remove heat waves.

Strain Measurement on a Heated Sample



## Notes

- At 5 frames/second the heat waves are highly aliased.
- A 0.1 Hz cut-off (Recursive 3<sup>rd</sup> order filter) was used
- Noise is greatly reduced.