

Responses of Structures to SDoF vs MDoF Vibration Testing

IMAC XXXIV

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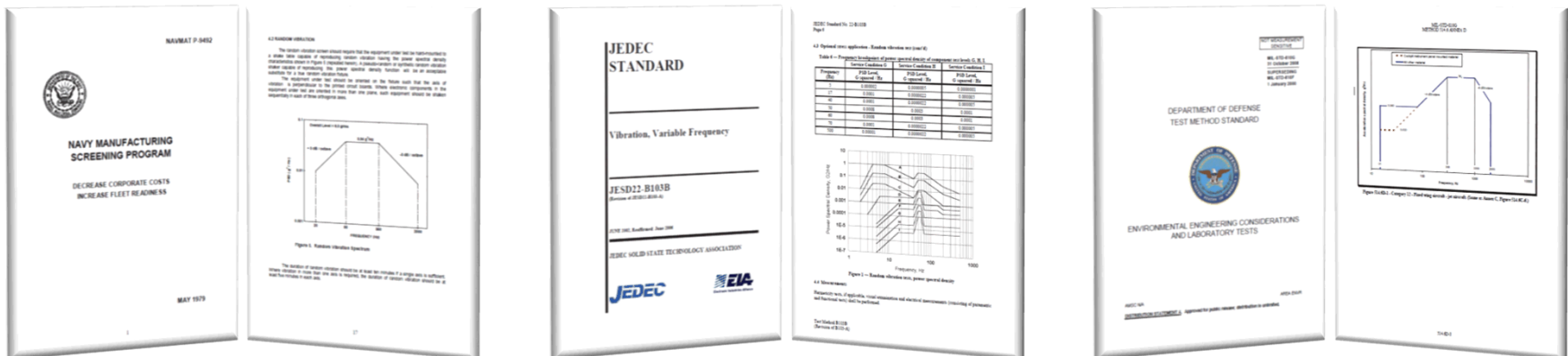
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Outline

- Motivations & Objective
- Equipment
- Test Article
- Sensor Selection & Configuration
- Test Sequence
 - Axes
 - Levels
 - Coherence and Phase
- Overview of Test Results
 - Energy Levels
 - Peak Accelerations
 - Modal Contributions

Motivations

- Sequential single axis testing has been firmly established as the preferred test method for environmental vibration characterization and analysis
 - MIL STD 810G: U.S. Department of Defense Environmental Test Standard
 - NAVMAT P-9492: U.S. Navy Manufacturing Screening Test Standards
 - JESD22-B103B: JEDEC Environmental Test Standards for Microelectronics




Motivations

- Unfortunately, vibrations in real world environments are 3-dimensional and these vibrations can result in different failure modes and component lifecycles [1-5]
- Recent developments in electrodynamic shaker capabilities have enabled reliable and controllable simultaneous multi-axis testing [6,7]
- Multi-axis control makes possible true single axis testing by allowing control of off axis and rotational vibrations that may be present in uniaxial shakers [8]
- Model validation and system identification via single axis test results cannot account for off axis affects [9]

Objective

- Through a collaboration of experimental and modeling work conducted on a given test article investigate:
 - The relationship between single axis and multi-axis vibration testing
 - What is the difference in response when off-axis energy is controlled vs when it is not?
 - What is the influence of the boundary conditions of the different types of tests on the response?
 - What's the effect of multi-axis inputs on a test article's modal response?
 - How much off-axis energy is input for the different test methods?
 - The benefit of single axis testing conducted on equipment capable of mitigating off-axis (and rotational) vibration

Test Equipment – Multi-axis

- Shaker System: Team Corporation Tensor™ 900 
 - Simultaneous or sequential excitation of X, Y, and/or Z axes
 - Complete control of rotations around all axes

Specifications

Table First Frequency	5,000 Hz
Test Frequency Range	10 - 5,000 Hz
Max Payload	9 lbs
Max Displacement	0.5 in
Max Acceleration (w/max payload)	10 g



- Controller Software: Spectral Dynamics JAGUAR Shaker Control and Analysis System

- Multi-Input and Multi-Output Control
- Input and Output Transformation for 6DOF Control



- Data Acquisition: National Instruments™ LabVIEW and NI PXIe-4496 Data Acquisition Modules



Test Equipment – Single Axis

- Shaker System: Unholtz-Dickie T2000
 - Sequential excitation of X, Y, and Z axes
 - To test different axes an adapter cube is required



Specifications

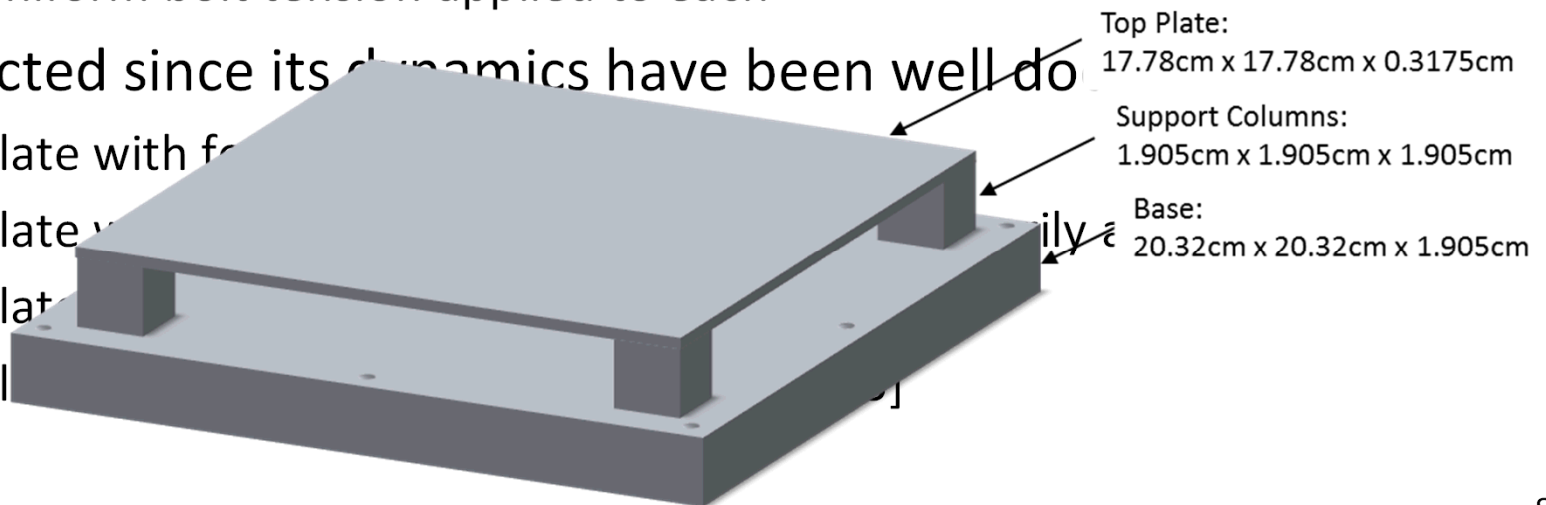
Test Frequency Range	10 – 3,000 Hz
Max Displacement	1.0 in
Force Rating - Random	107 kN-rms
Force Rating - Sine	111 kN-peak

- Controller Software: Spectral Dynamics JAGUAR Shaker Control and Analysis System
- Data Acquisition: National Instruments™ LabVIEW and NI PXIe-4496 Data Acquisition Modules

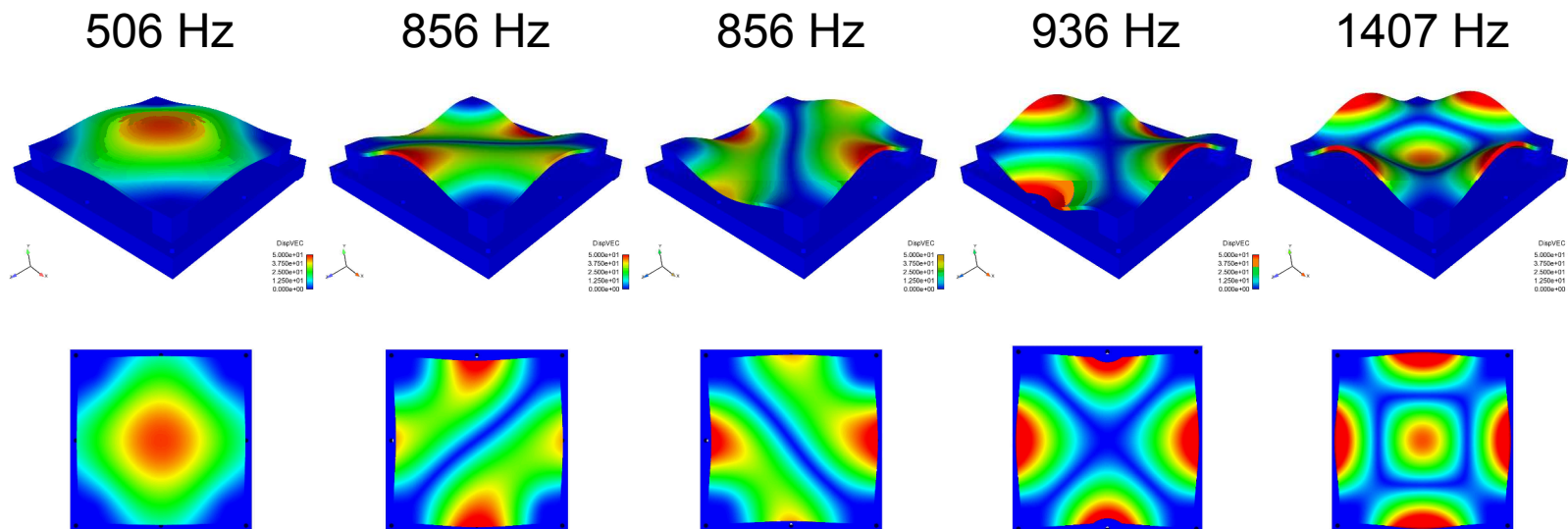


Test Article

- A column supported square aluminum 6061 plate
 - Base, columns, and top plate made of a continuous piece of material
 - Both homogenous and isotropic
 - Eliminates added dynamics due to support boundaries
- Evenly spaced mounting holes
 - Eight (8) circumferential and one (1) central
 - Uniform bolt tension applied to each
- Selected since its dynamics have been well do
 - Plate with f
 - Plate v
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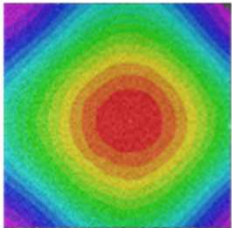
- Finite Element Model
 - Used to predict dominant mode shapes and frequency components
 - Confirms that support columns are not dynamically active in frequency range of interest
- Primary Mode Shapes Identified



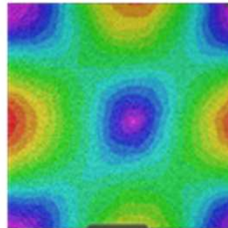
Experimental Test Article Mode Shapes

- Digital Image Correlation used to extract and verify mode shapes
- The mode shapes and their frequencies agree with those predicted by the model

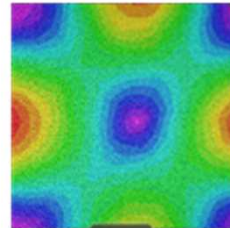
Mode 1
512.5 Hz



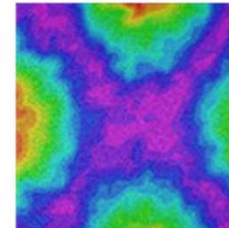
Mode 2a
874.5 Hz



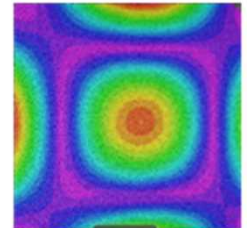
Mode 2b
874.5 Hz



Mode 3
950.1 Hz



Mode 4
1417.9 Hz



Sensor Configuration – Multi-axis

■ Control Accelerometers,

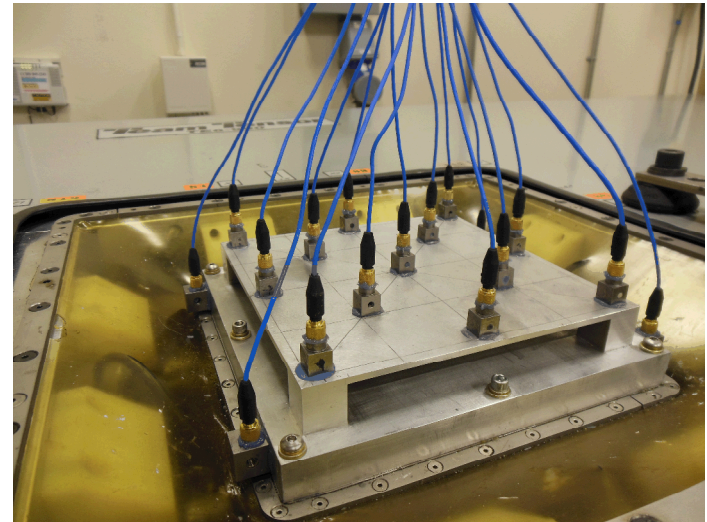
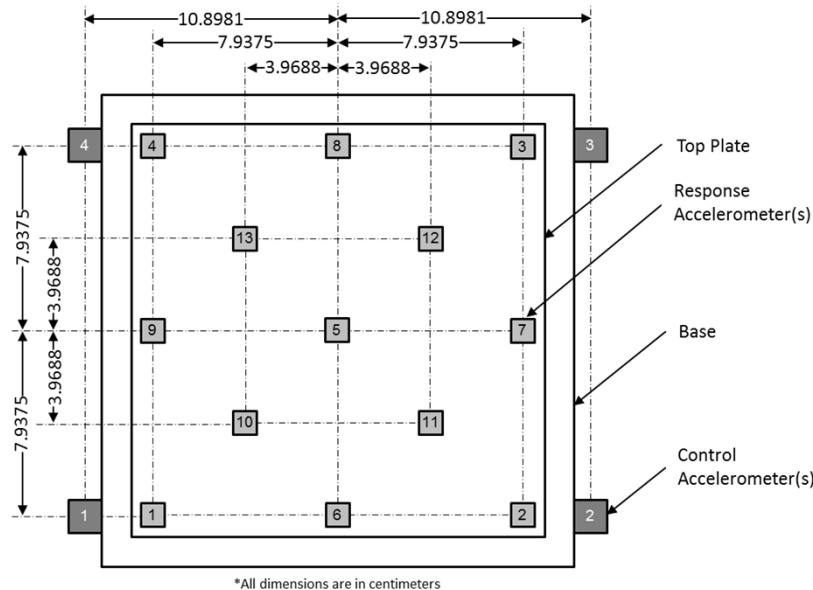
■ PCB 356A15

- Triaxial ICP Accelerometer
- Nominal Sensitivity: 100 mV/g
- Weight: 0.37 oz

■ Response Accelerometers

■ PCB 356A33

- Triaxial ICP Accelerometer
- Nominal Sensitivity: 10 mV/g



Sensor Configuration – Single Axis

■ Control Accelerometers,

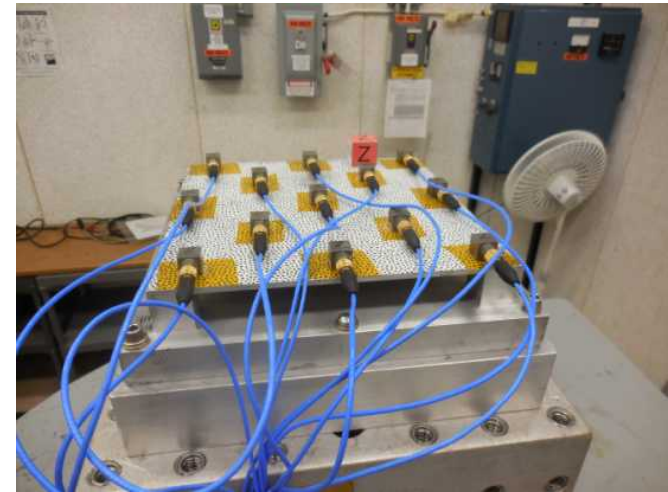
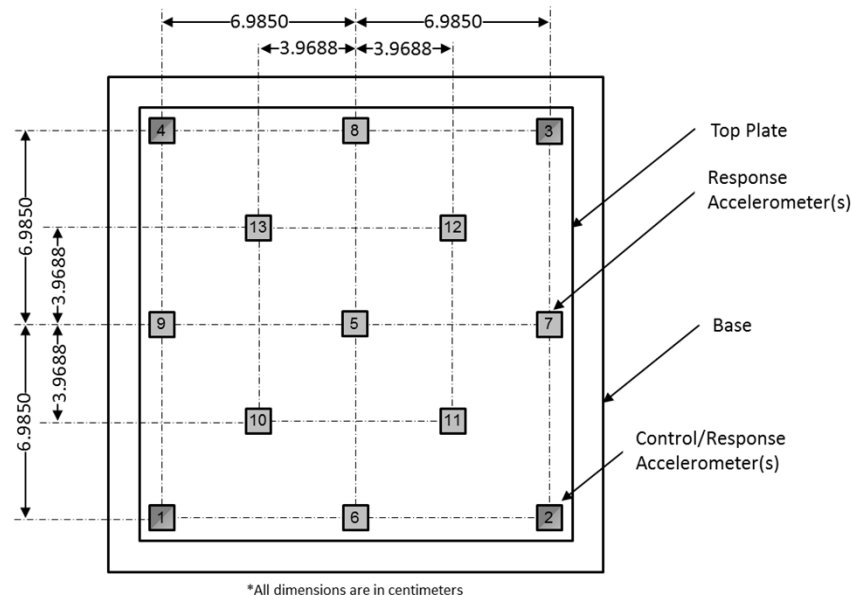
■ PCB 356A33

- Triaxial ICP Accelerometer
- Nominal Sensitivity: 10 mV/g

■ Response Accelerometers

■ PCB 356A33

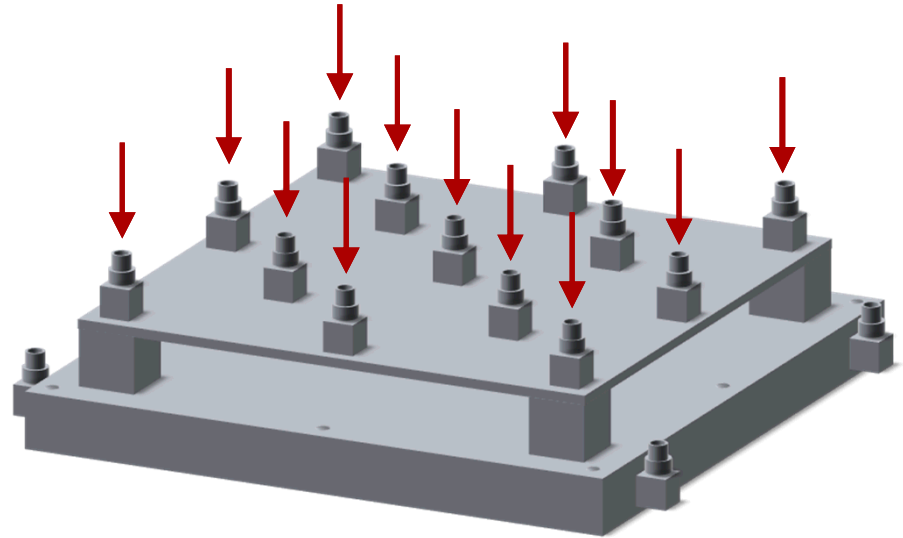
- Triaxial ICP Accelerometer
- Nominal Sensitivity: 10 mV/g



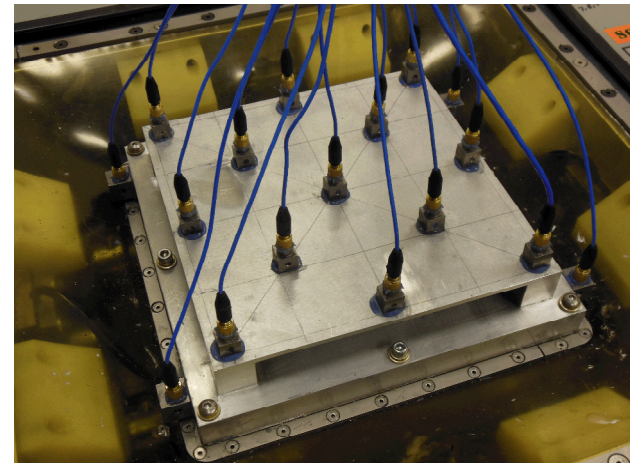
Sensor Selection & Configuration

■ Reference Sensor Placement:

- Center of plate
- Each corner
- Mid-span of each side
- Along each diagonal placed evenly between the center and corner accelerometers



- The high number of accelerometers caused a mass loading affect on the top plate
- Their positions were selected to mitigate mode shape distortion
- FEA data confirms that modes are preserved although all frequencies were shifted lower

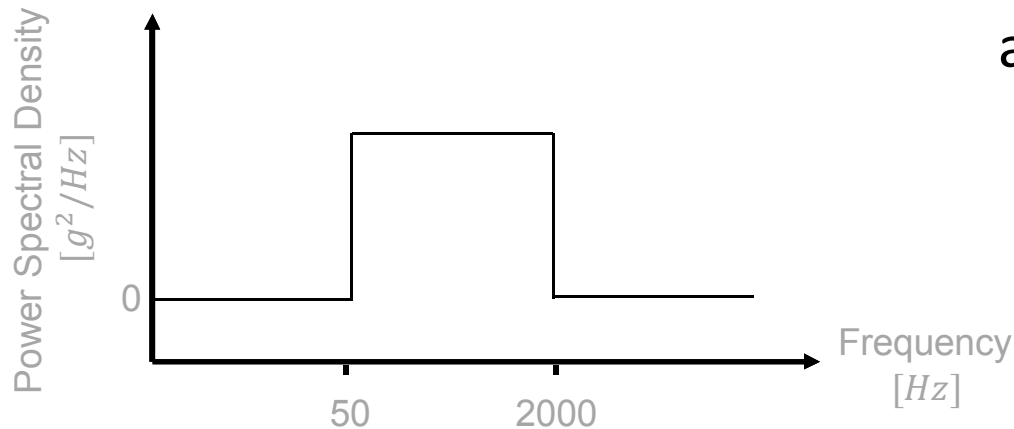


Test Sequence

➤ Control Signal

- Axes
- Input Level
- Cross-Axis Coherence & Phase

- Band-limited white noise (20Hz – 2kHz)
- Causes simultaneous excitation of all frequencies within range
- All frequency dependence in stimulated responses can be attributed to plate dynamics



Test Sequence

- Control Signal

- Axes

- Input Level

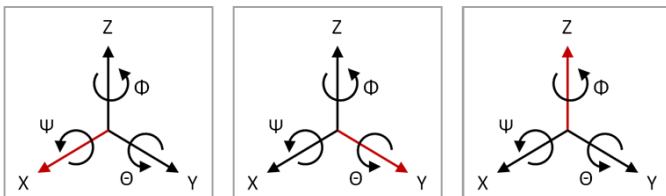
- Cross-Axis
Coherence & Phase

- Single and Multi-Axis

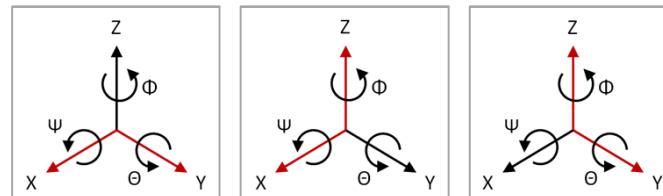
- Uniaxial: One translational axis at a time (X, Y, Z)
- Biaxial: Two translational axes at a time (XY,XZ,YZ)
- Triaxial: All three axes simultaneously (XYZ)

- All axes always controlled, but not all to full test levels

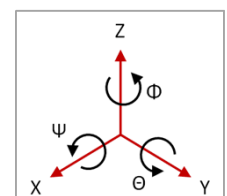
Uniaxial



Biaxial



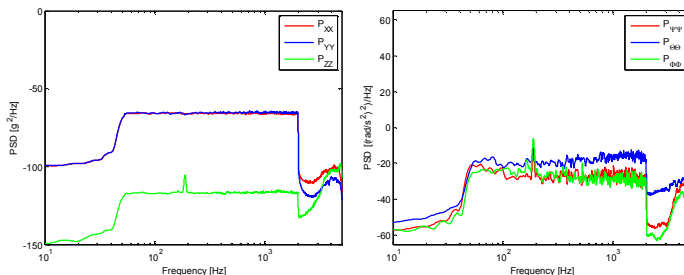
Triaxial



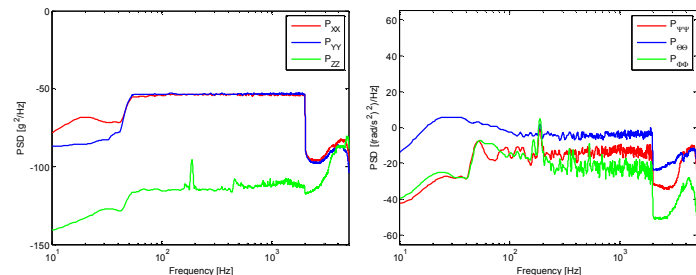
Test Sequence

- Control Signal
- Axes
- Input Level
- Cross-Axis Coherence & Phase
- Input Acceleration Levels:
low ($1g_{rms}$) & high ($2g_{rms}$)
- Same acceleration level for all applicable axes
- All other DOFs controlled to low level

X & Y Translation Controlled (@ 1g)
Z Translation & All Rotations Minimized



X & Y Translation Controlled (@ 2g)
Z Translation & All Rotations Minimized

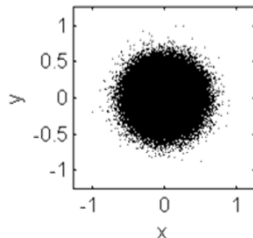


Test Sequence

- Control Signal
 - Axes
 - Input Level
 - Cross-Axis Coherence & Phase
- Zero phase between all axes
 - Coherence is measure of relationship between two signals
 - Levels: low (~ 0), medium (0.50), and high (~ 1)
 - Coherence with and between all other DOFs set at zero

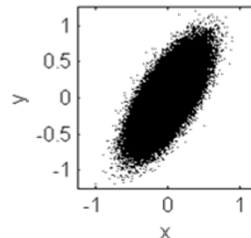
X,Y Unrelated

$$C_{XY} \cong 0$$



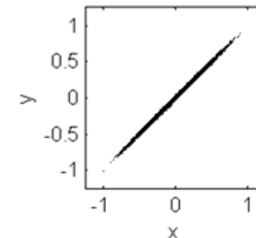
X,Y Partially Related

$$C_{XY} \cong 0.5$$



X,Y Highly Related

$$C_{XY} \cong 1$$



Off Axis Acceleration Levels

System	Test	Level	Acceleration		
			Transverse X (g)	Longitudinal Y (g)	VerticalZ (g)
1D	X	1g	-	*	*
		2g	-	7.01%	36.54%
	Z	1g	6.95%	9.10%	-
		2g	6.35%	8.44%	-
6D	X	1g	-	6.17%	11.64%
		2g	-	3.24%	13.24%
	Y	1g	6.03%	-	8.18%
		2g	3.13%	-	5.78%
	Z	1g	6.08%	6.46%	-
		2g	3.29%	3.38%	-
*Data Not Available					

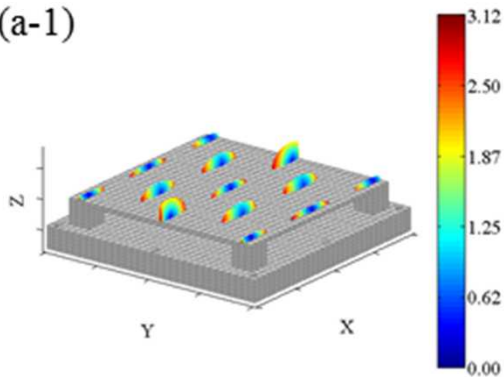
RMS Accelerometer Responses

X Excitation

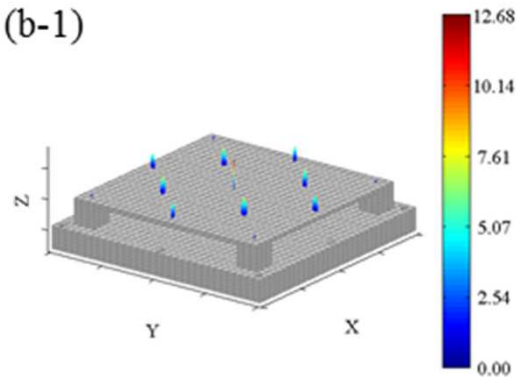
Z Excitation

X&Z Excitation

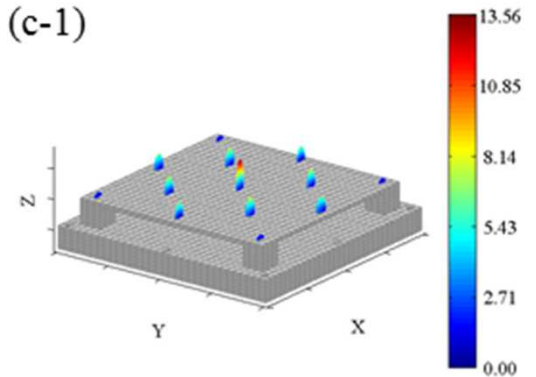
(a-1)



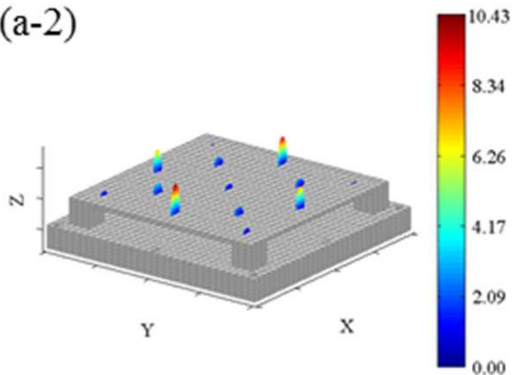
(b-1)



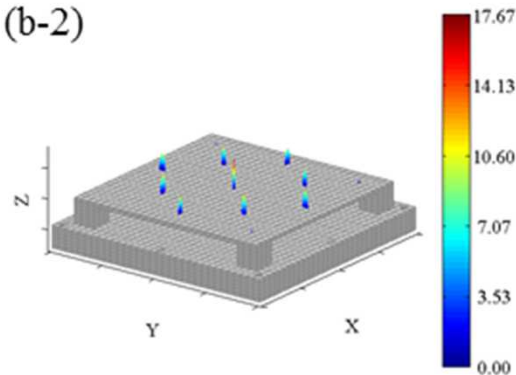
(c-1)



(a-2)



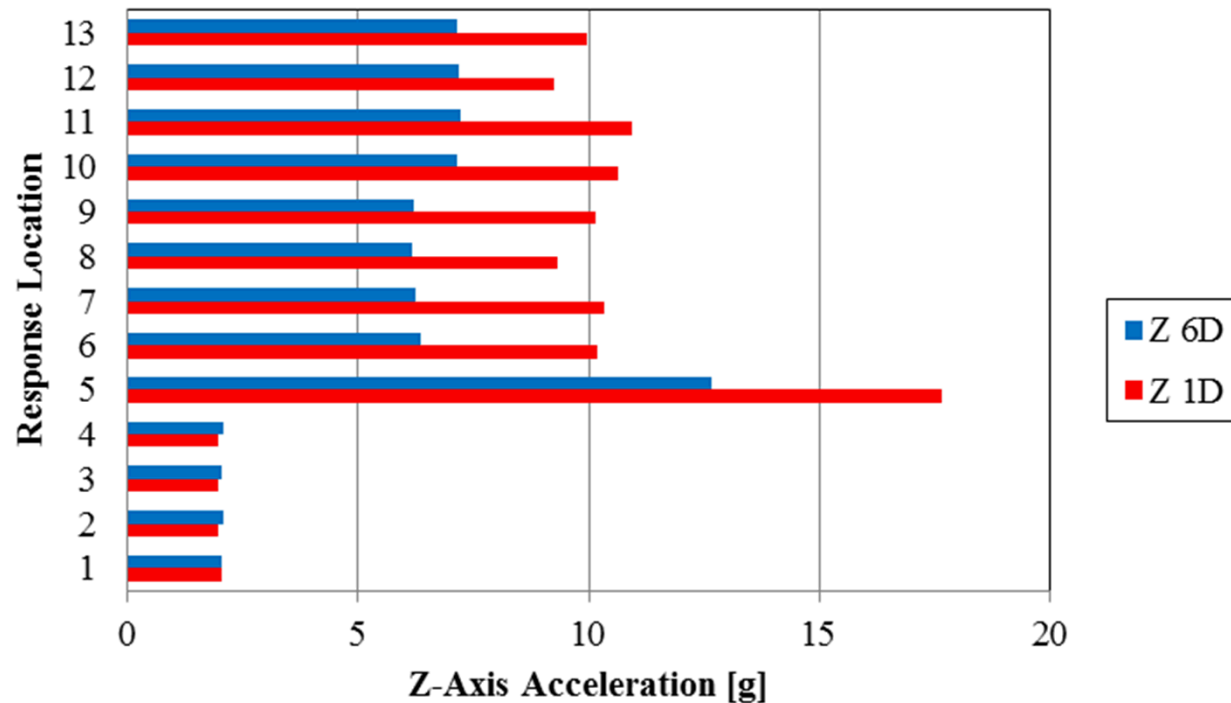
(b-2)



- Response of structure agrees well with expected response for multi-axis shaker (1)
- When conducting test on single axis shaker, uncontrolled off axis energy results dominated by Z axis.

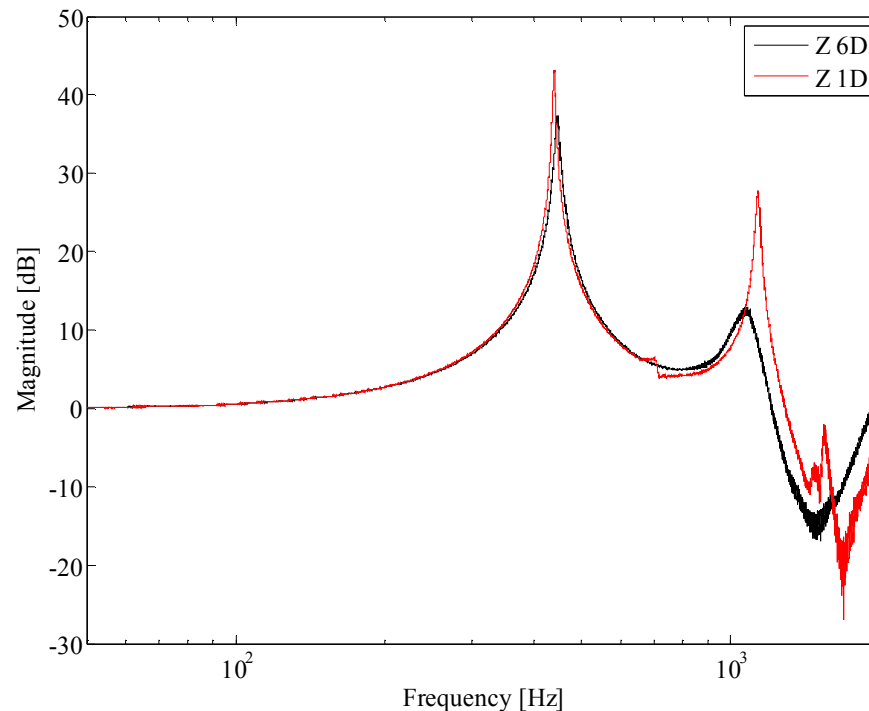
RMS Accelerometer Responses (2)

- Controlled the vertical acceleration to the same level on both shakers, the RMS levels of the plate response on the systems differed substantially.



Response at the Center of the Plate

- Response influenced by two critical modes.
 - On multi-axis shaker, they are at 446Hz and 1073Hz
 - On uniaxial shaker, they are at 440Hz and 1137Hz
 - Larger response occurs at both resonances on uniaxial shaker system.

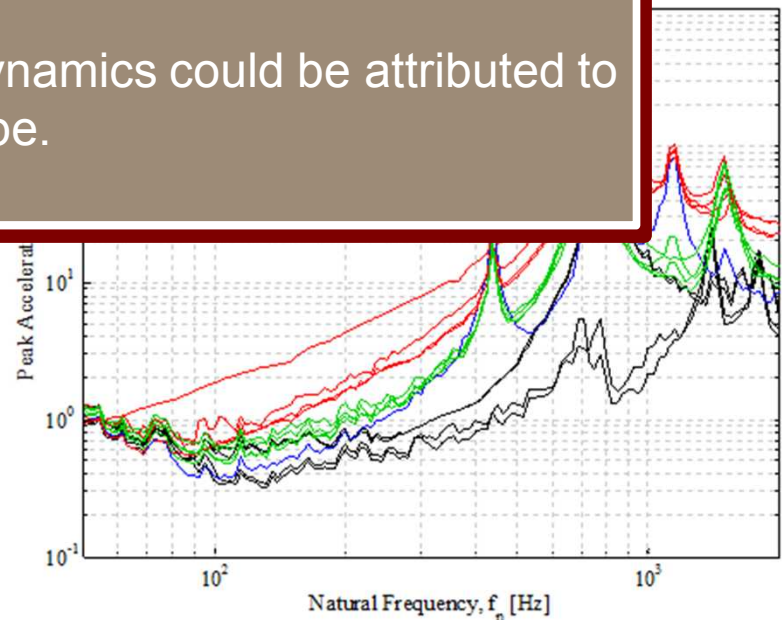
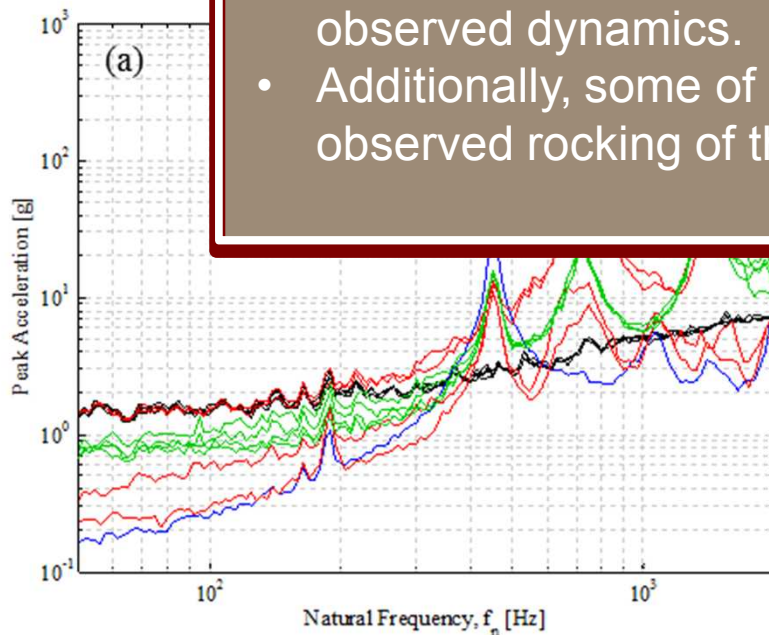


Shock Response Spectra (SRS)

- SRS graphically portrays the peak acceleration that would be experienced by a series of single degree of freedom system with

- The c
frequ

- Additional modal contributions on uniaxial test likely due to boundary conditions created by the cube.
- Out of axis deformation of cube could contribute to observed dynamics.
- Additionally, some of the dynamics could be attributed to observed rocking of the cube.



Conclusions

- Single axis testing on uniaxial shaker results in larger peak energy levels than those conducted on multi-axial shakers
 - Comparing results to multiaxial tests, it is not due contributions of off-axis translational excitation
 - Tests were not conducted with added rotations, which may have contributed to this effect.
- Any test with a z-axis stimuli would be dominated by a response in the z-axis, even the x-axis test on the uniaxial shaker.
- The peak acceleration levels found by the SRS demonstrated that the primary modes were invariant to the addition of lateral excitation.
- The lateral response had additional modes due to the dynamics of the mounting cube,
- For model validation, the uncontrolled effects of the testing system must be well documented.

References

- [1] R.M. French, R. Handy, and H.L Cooper, "A comparison of simultaneous and sequential single-axis durability testing," *Experimental Techniques*, vol. 30, no. 5, pp. 32-37, Sept./Oct. 2006.
- [2] D.L. Gregory, F. Bitsie, and D.O. Smallwood, "Comparison of the response of a simple structure to single axis and multiple axis random vibration inputs," in *Proc. 80th Shock and Vibration Symp.*, San Diego, CA, Oct. 2009.
- [3] E. Habtour et al., "Improved reliability testing with multiaxial electrodynamics vibration," in *Proc. 2010 Reliability and Maintainability Symp.*, San Jose, CA, pp. 1-6, Jan. 2010.
- [4] B. Aytekin and H.N. Ozguven, "Vibration analysis of a simply supported PCB with a component - an analytical approach," in *Proc. 10th Electronics Packaging Technology Conf.*, Singapore, pp. 1178-1183, Dec. 2008.
- [5] W.E Whiteman, "Multiaxial versus uniaxial vibration testing: a research plan for comparison," in *19th Int. Modal Analysis Conf.*, Orlando, FL, Feb. 2001.
- [6] D.O. Smallwood and D.L. Gregory, "Evaluation of a 6-DOF electrodynamic shaker system," in *Proc. 79th Shock and Vibration Symp.*, Orlando, FL, Oct. 2008.
- [7] D.J. Osterholt, N.C. Yoder, and D. Linehan, "Advances in six degree of freedom vibration testing," in *Proc. Experimental and Appl. Mechanics*, June 2007.

References

- [8] K.Y. Chang and A.M. Frydman, "Three-dimensional random vibration testing definition and simulation," in *Proc. 36th Annual Mtg. Institute of Environmental Sciences*, Mount Prospect, IL, pp. 129-139, 1990.
- [9] U. Fullekrug and M. Sinapius, "Structural dynamics identification by means of multi-axial base excitation - theory and application," in *Proc. European Conf. on Spacecraft Structures, Materials, and Mechanical Testing*, Braunschweig, Germany, pp. 503-509, Nov. 1998.
- [10] R.D. Blevins, "Plates," in *Formulas for Natural Frequency and Mode Shape*, Malabar, FL: Krieger, Jan. 2001, pp. 264-269.
- [11] D.J. Johns and R. Nataraja, "Vibration of a square plate symmetrically supported at four points," *J. of Sound and Vibration*, vol. 25, no. 1, pp. 75-82, Nov. 1972.
- [12] M.B. Felemban, "Vibration analysis of point and column supported mindlin plates," M.S. thesis, Mech. Eng., King Fahd Univ. of Petroleum & Minerals, Dharan, Saudi Arabia, 1989.
- [13] M. Petyt and W.H. Mirza, "Vibration of column-supported floor slabs," *J. of Sound and Vibration*, vol. 21, no. 3, pp. 355-364, Apr. 1972.
- [14] M.A. Underwood and T. Keller, "Applying coordinate transformations to multi-DOF shaker control," *Sound and Vibration*, vol. 40, no. 1, pp. 22-27, Jan. 2006.

Thank you for your attention!

QUESTIONS?