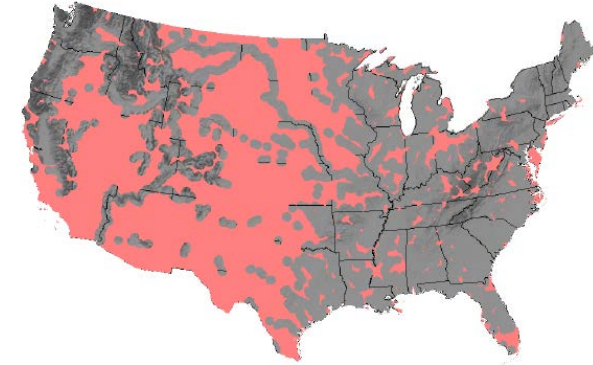
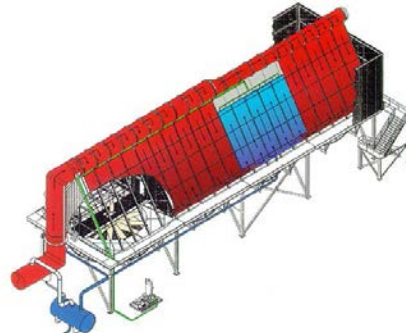


*Exceptional service in the national interest*



# DRY-COOLED SUPERCRITICAL CO<sub>2</sub> POWER FOR ADVANCED NUCLEAR REACTORS

**T. M. Conboy, \*M. D. Carlson, G. E. Rochau**

June 16-20, 2014, Dusseldorf, Germany

ASME Turbo Expo 2014

**GT2014-25079**



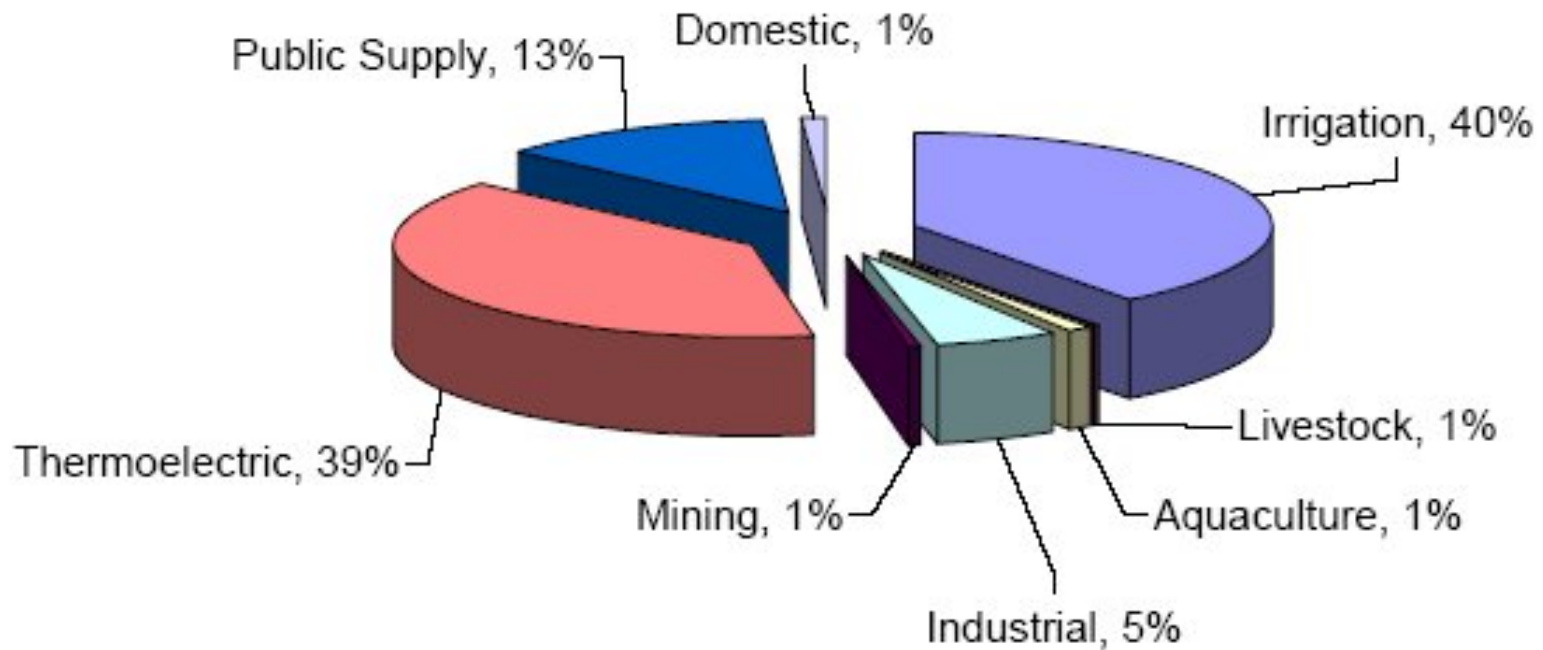
Sandia National Laboratories is a multi-program laboratory managed and operated by Sandia Corporation, a wholly owned subsidiary of Lockheed Martin Corporation, for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-AC04-94AL85000.

# Overview – Dry-Cooled SCO<sub>2</sub> SMRs

- The water-energy nexus is a critical challenge for power plants
  - Current nuclear, coal, and concentrating solar plants require cooling
  - 39% of U.S. freshwater withdrawal is used for cooling power plants
  - Supply is decreasing while demand is increasing
- Supercritical CO<sub>2</sub> (s-CO<sub>2</sub>) power cycles offer many advantages
  - Proposed as an alternative to steam and organic Rankine systems
  - Offer high efficiency, compact turbomachinery, fluid compatibility
- s-CO<sub>2</sub> power cycles are well suited for use of dry-air cooling
  - Maintain good efficiency at high heat sink temp
  - Allow for closer average temperature differences
  - Less capital in building, operating cooling towers.
- **Benefits of dry-cooled s-CO<sub>2</sub> cycle could aid the rise of next-generation nuclear systems, and could redraw the map for siting of new nuclear**

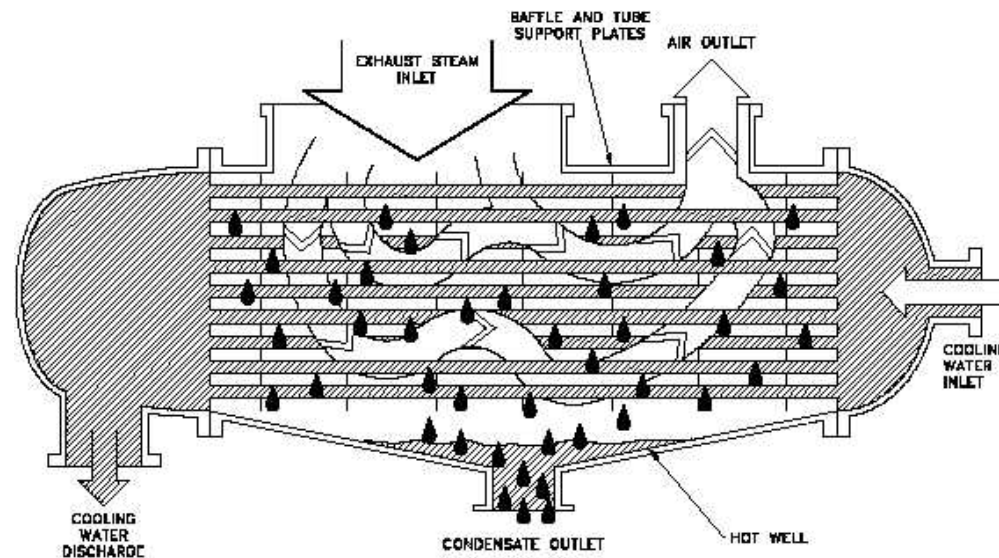
# Introduction – U.S. Water Use

**39% of U.S. withdrawal is for power plant cooling**



# Nuclear Steam Condensers

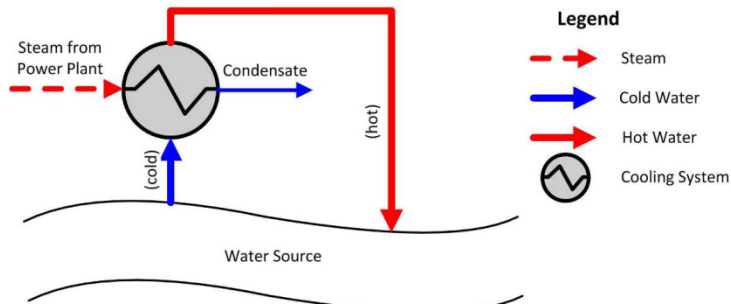
- Most U.S. electricity is generated by 33% efficient steam turbines, with the remaining ~67% of the thermal source energy rejected to the environment
- Steam condensers are a critical component for safety and performance
- Currently shell & tube, with tube-fin, wet, dry, and hybrid options possible
- Operate at sub-atmospheric turbine back pressure
- Performance can be hampered by the build-up of non-condensable gasses



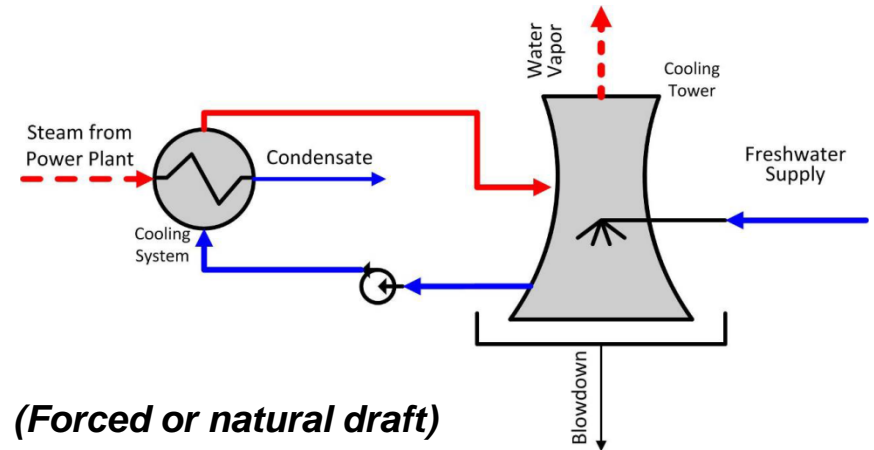
# Heat Sink Options:

*Each has advantages and disadvantages*

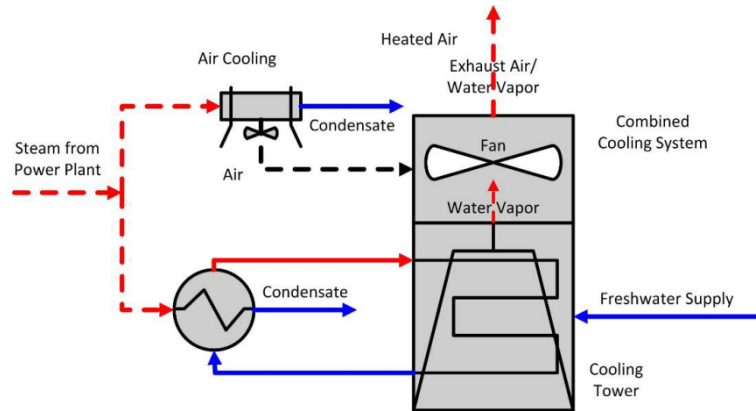
### Once-through, wet



### Indirect wet cooling tower

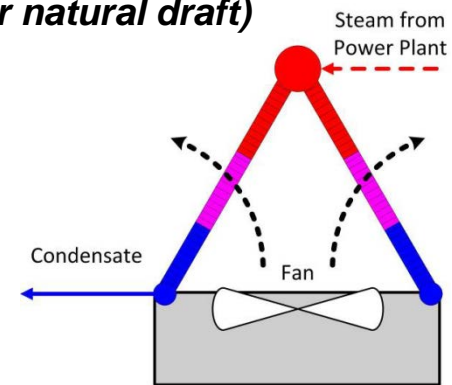


*(Forced or natural draft)*



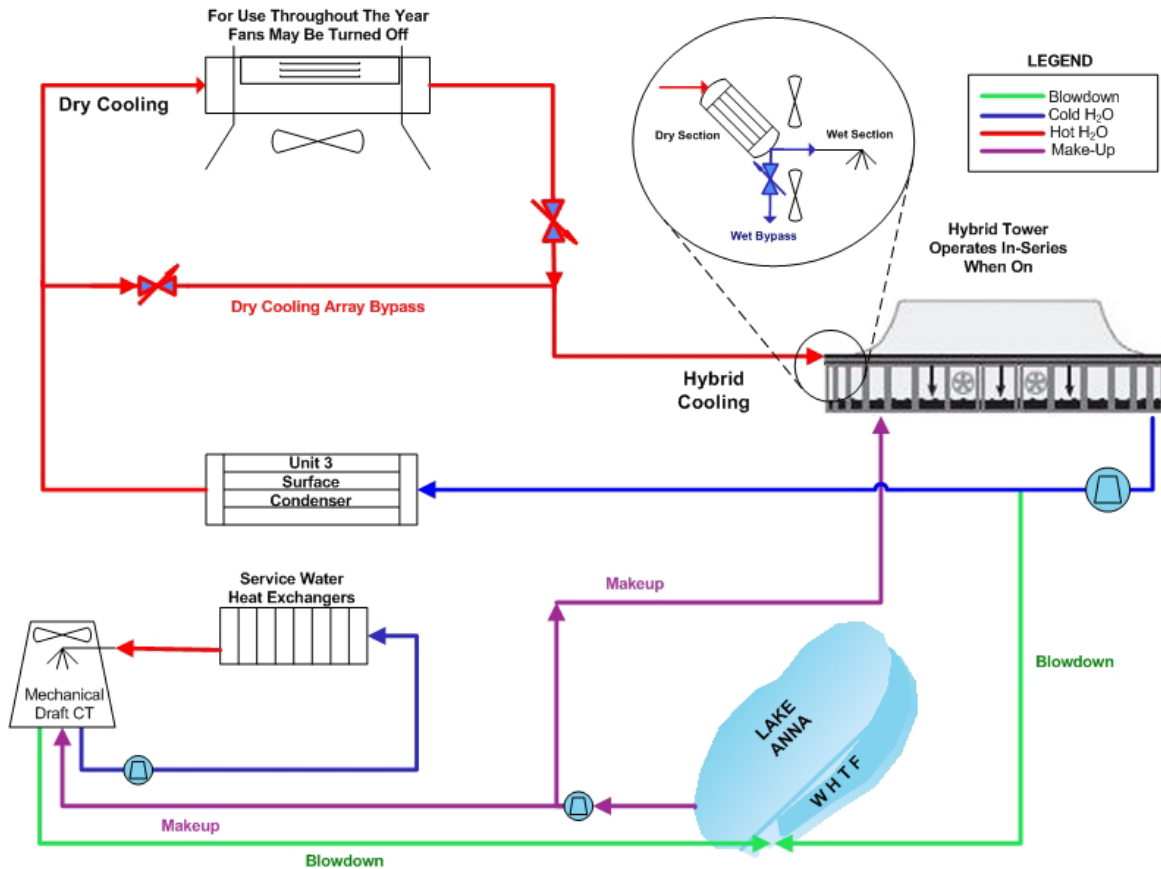
### Hybrid cooling tower

*(Forced or natural draft)*



### *Dry cooling tower*

# Cooling in the news at nuclear plants



– Oyster Creek (NJ) agrees to shut down a decade early rather than build cooling towers at cost of \$1-2B

– Legal action to force cooling towers at Millstone (CT), Salem 1 and 2 (NJ), and Indian Pt (NY)

– North Anna (VA) new build proposes use of hybrid cooling after water concerns

– etc

North Anna Unit 3 (proposed BWR)  
-COL submitted

# Nuclear siting

## Factors considered in nuclear plant siting:

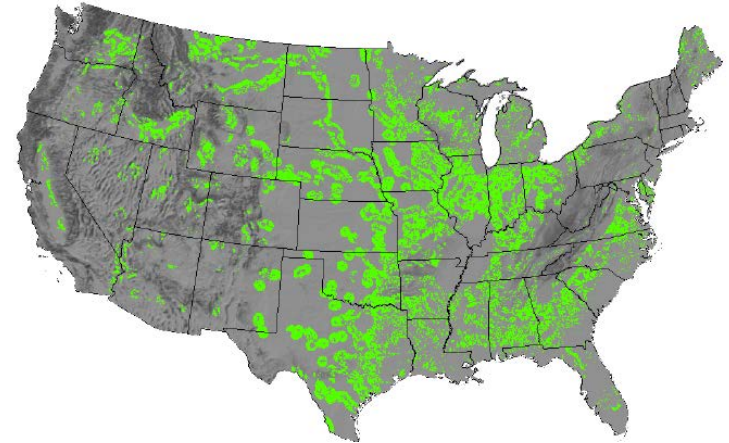
- Local population density
- Proximity to faults, protected lands, slopes, flood terrain
- Wide open free space around the site (500+ acres)

### Cooling water

- 50,000 gpm for 300MWe SMR
- 200,000 gpm for 1000+ MWe

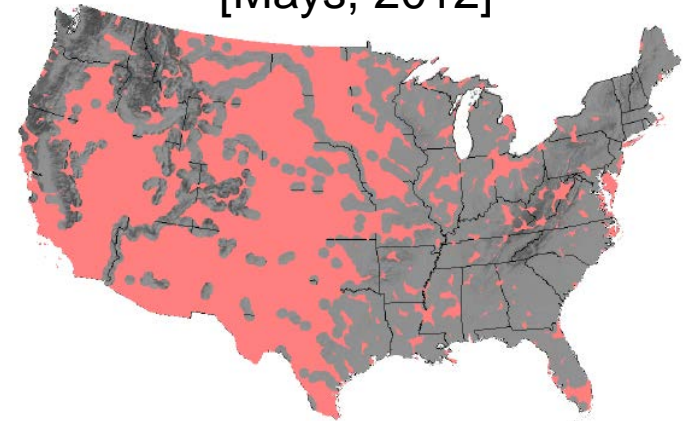
Study at ORNL in 2012 considered all of these factors and more, scanned spatial map of US for potential sites

Much of Western US is excluded from consideration because of cooling water requirements



Green: areas of sufficient water

[Mays, 2012]

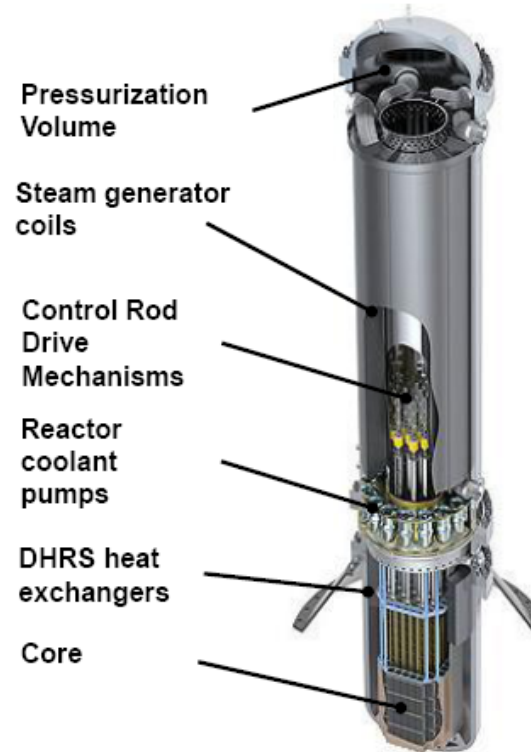


Pink: areas excluded from nuclear siting

# Dry cooling in nuclear?

## Generation mPower

- Developed by Babcock & Wilcox
  - 125 MWe capacity
  - Integral PWR configuration
  - Forced circulation of primary coolant
  - Standard 17x17 pin fuel assemblies with 4.5-yr refueling cycle
  - Once-through straight tube steam generator
  - 3.6-m-dia by 22-m-tall reactor vessel
- advertises 31% efficiency in dry cooling [WNA, 2013]
  - Still requires large volume of water on site for 7 day emergency cooling



# Current dry-cooled steam plants

- 1% of installed steam plants are dry
- Because of limitations, its only economic when water is prohibitively expensive
- Even then, gas turbines or solar PV can be used

- **Higgins Generating station (NV)**

- *550MWe combined cycle*
- *Forced mechanical dry cooling*

- **Kendal Power Station (South Africa)**

- *6 x 686MWe coal plant*
- *Natural draft dry cooling plant*

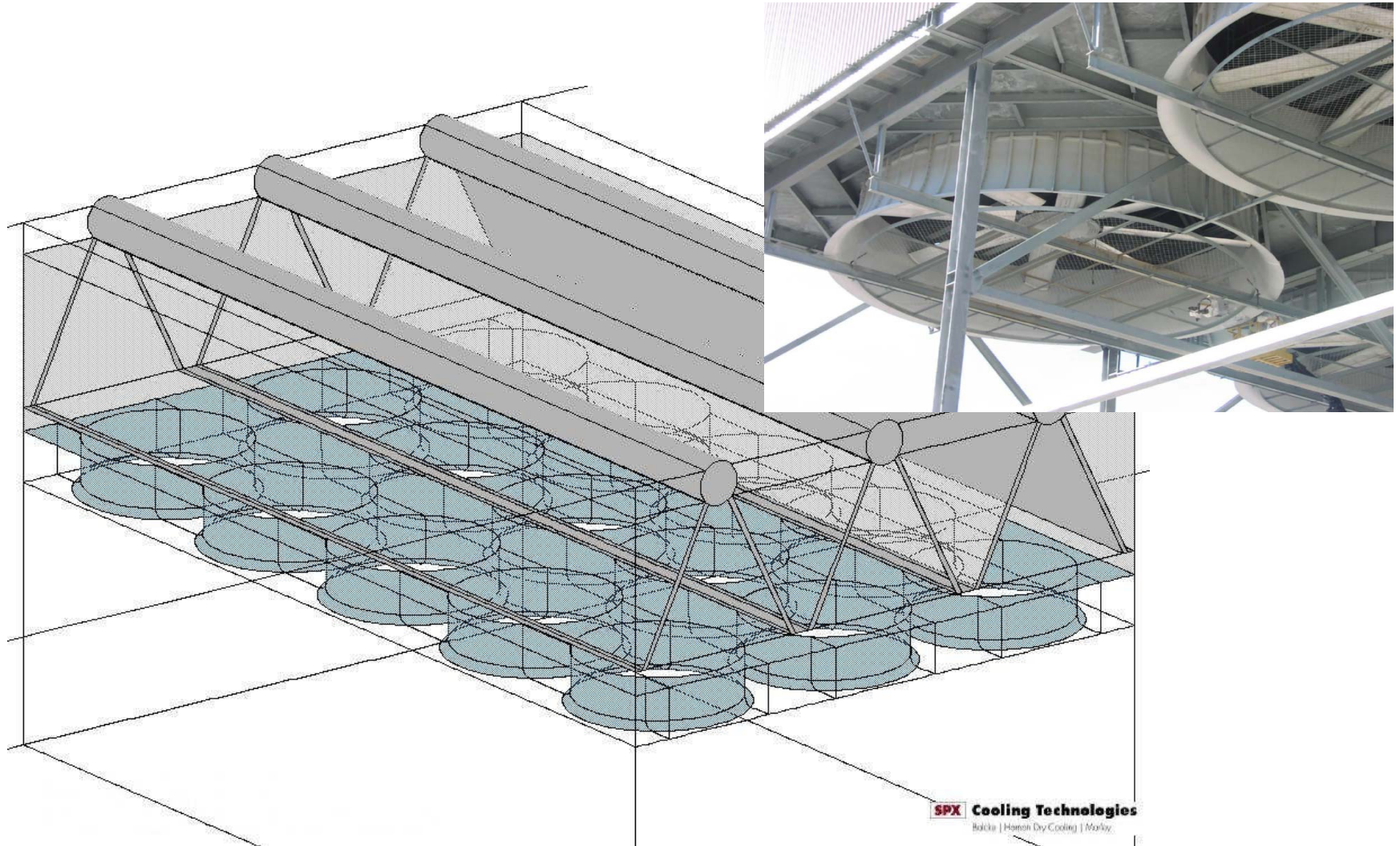
*Generally operate at backpressure of 14-17kPa ( $T_{sat}=55.8C$ ) instead of 6kPa ( $T_{sat}=36.2C$ )*

- **Ivanpah (under construction)**

- *372MWe CSP (concentrating solar)*
- *Forced mechanical dry cooling*
- *Solar drives much of dry cooling technology today*

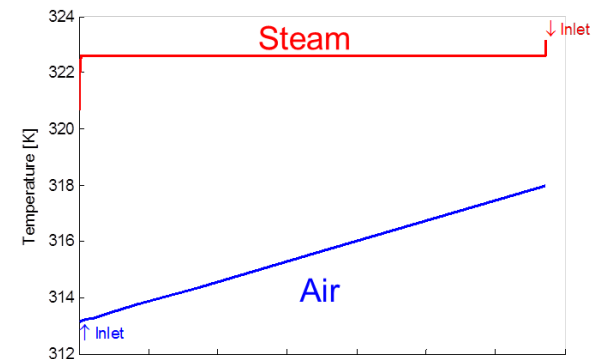


# Cooling fans at Higgins Station

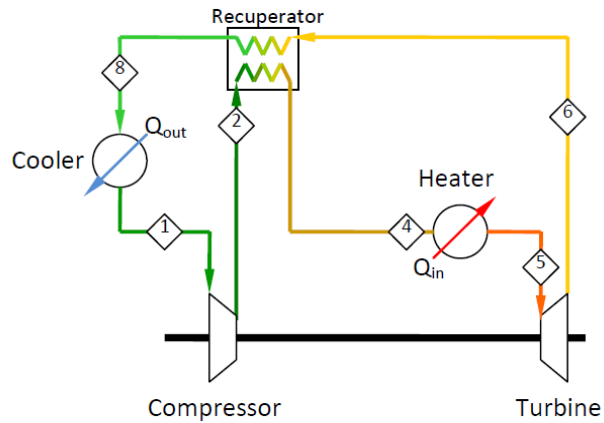


# Challenges for dry-cooled condensers

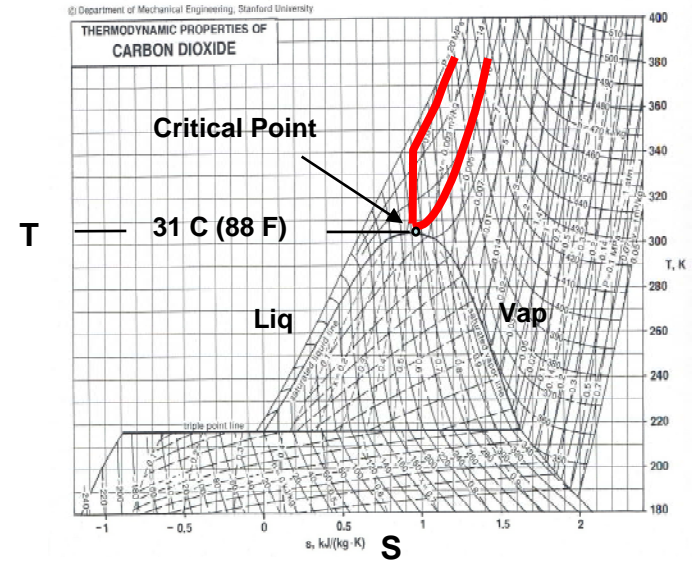
- **Dry heat rejection is attractive in theory**, use on commercial steam
- is rare despite obvious advantages
- **Difficulties include:**
  - *Limited by constant temp of condensing fluid*
  - *Poor heat flow profile match, exergy loss*
  - *Bounded by dry-bulb temp, instead of wet-bulb*
  - *3-5x as expensive to build [NEI]*
  - *10x as expensive to operate [NEI]*
  - *10% efficiency penalty when using dry cooling*
  - *Water is needed anyway for emergency cooling (nuclear)*
- These issues are **fundamental physical limits**
- Much of ongoing research is looking at novel hx surfaces such as conducting polymers, metal foams, nanostructures [See FOAs: NSF/EPRI 2013, ARPA-E 2012]. Relevant on large scale?
- **Use of alternative working fluid is relatively unexplored solution**



# What is a SCO2 Brayton Cycle?

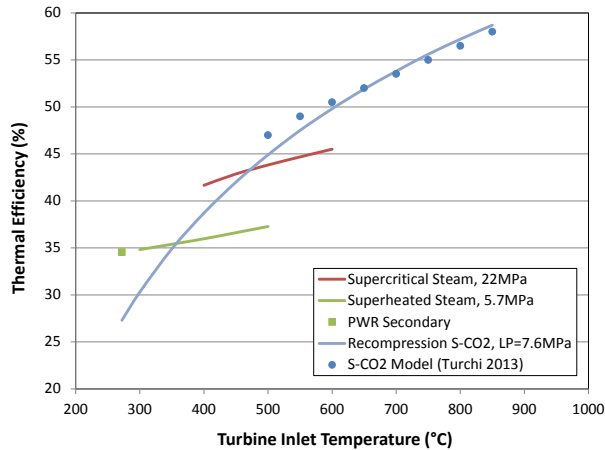


**Liquid like Densities with CO<sub>2</sub>**  
**Very Small Systems,**  
**High Efficiency due to Low Pumping Power**

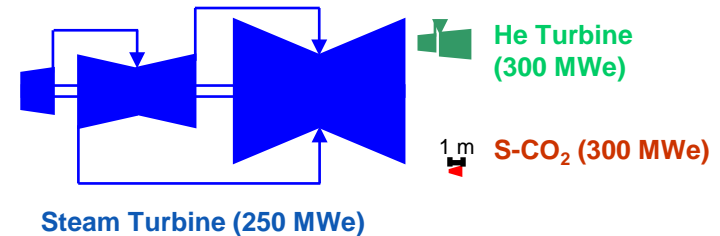


**Rejects Heat**  
**Above Critical Point**  
**High Efficiency Non-Ideal Gas**  
**Sufficiently High for Dry Cooling**

**Critical Point**  
**88 F / 31 C**  
**1070 psia / 7.3 MPa**



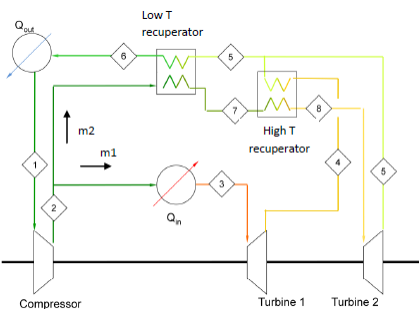
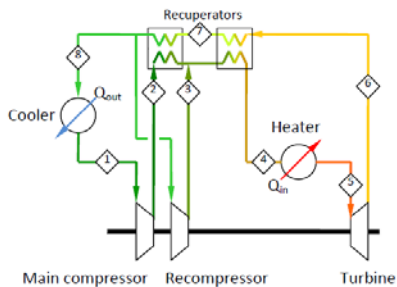
**High Efficiency at Lower Temp**  
**(Due to Non-Ideal Gas Props)**



**High Density Means Very Small Power Conversion System**  
**Non-Ideal Gas Means Higher Efficiency at Moderate Temperature**

# Nuclear Applications of s-CO<sub>2</sub> Power

- *s-CO<sub>2</sub> Brayton cycle first rose to prominence in 1960s*
- *Decades later, revived by Gen-IV Forum and work at MIT (2002-2006)*
- *Earlier work served as theoretical basis for ongoing demo projects at SNL (2007-present)*
- *Many studies have been published on use of s-CO<sub>2</sub> in different reactor types*
- *Cycle should be chosen to match  $\Delta T_{rise}$  with desired  $\Delta T_{rise}$  across core*
  - *Liquid metal reactors  $\Delta T \sim 150\text{C}$ , gas reactors  $\Delta T \sim 400\text{-}500^\circ\text{C}$*



| Reactor type | Coolant          | Neutron spectrum | Peak Temp. | Optimum S-CO <sub>2</sub> Cycle |
|--------------|------------------|------------------|------------|---------------------------------|
| SFR          | Na               | fast             | 550°C      | Recompression                   |
| LFR          | Pb-Bi            | fast             | 800°C      | Recompression                   |
| GFR-indirect | He               | fast             | 850°C      | Cascaded/ Series                |
| GFR-direct   | CO <sub>2</sub>  | fast             | 650°C      | Recompression                   |
| LWR          | H <sub>2</sub> O | thermal          | 300°C      | Simple Cycle                    |
| VHTR         | He               | fast             | 1000°C     | Cascaded/ Series                |
| MSR          | fluoride salt    | fast/thermal     | 1000°C     | Recompression                   |

[Moisseytsev, 2009, etc]

[Moisseytsev, 2008, etc]

[Wright, 2010]

[Parma, 2011; Handwerk 2007]

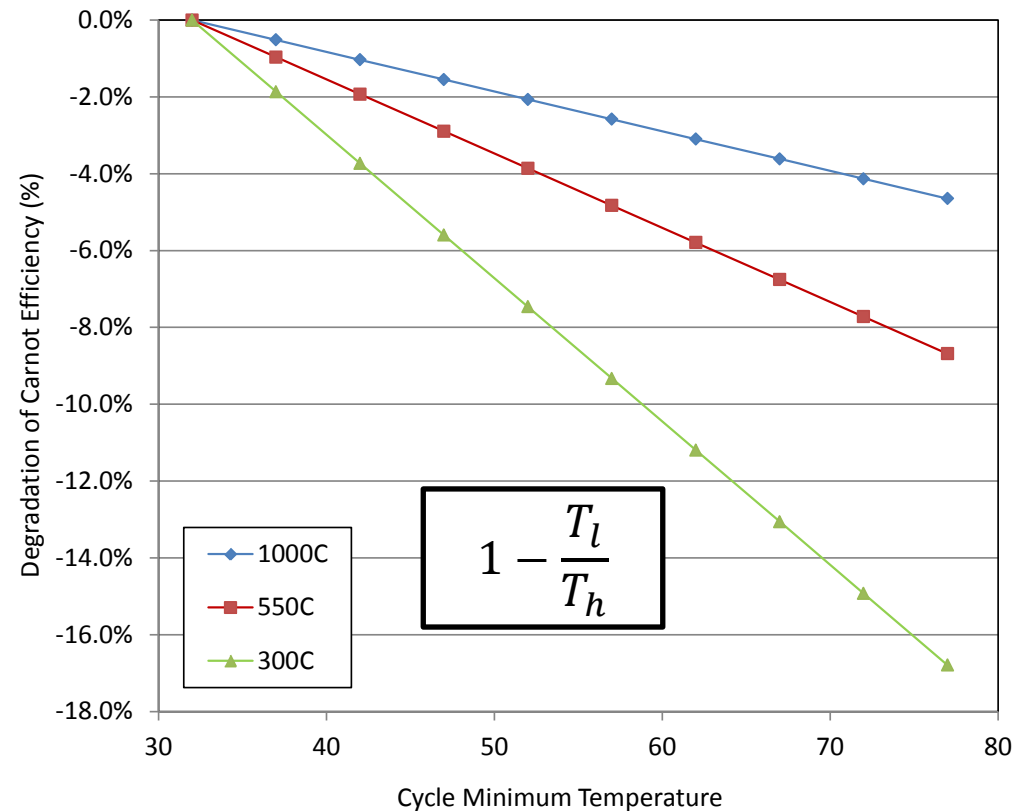
[Yoon, 2011; Kao, 2011]

[Wright, 2010]

[Forsberg, 2012]

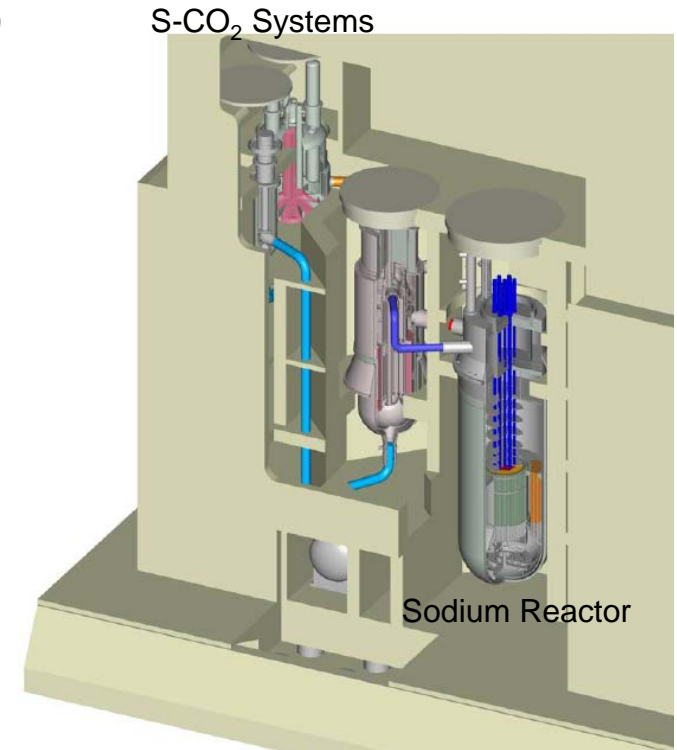
# Dry Cooling Power Cycle Efficiency

- *Power cycle lower limit is hostage to ambient air conditions*
- *May result in elevated compressor inlet conditions*
  - *negatively impacts cycle efficiency*
- *Certainly introduces increased variability (day/night, seasonal, etc)*
- *Effect is reduced for high temperature cycles*



# Case Study: SFR at 550C turbine inlet temperature

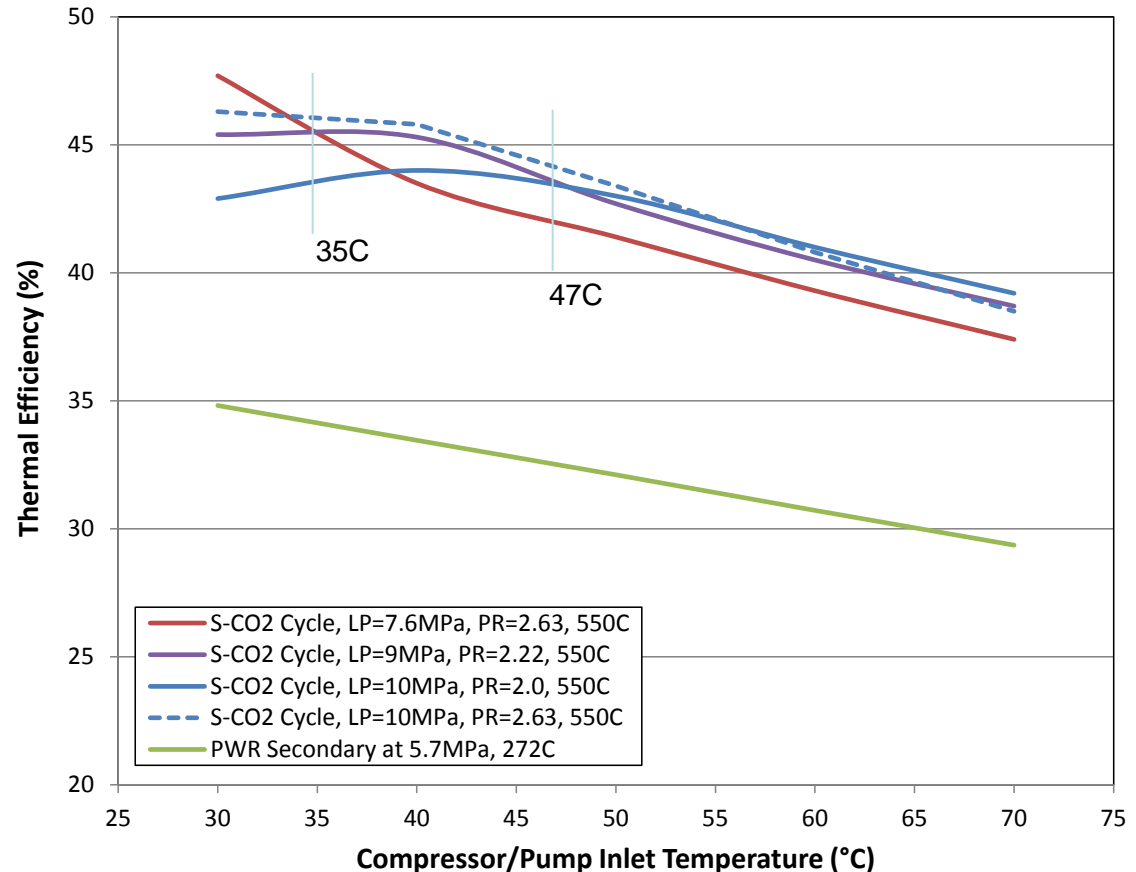
- *Intermediate temperature SFR at 550°C chosen as example for further study*
- *Currently baseline concept for DOE-NE ARC Program*
- *SFRs have operated 550°C+ (Phenix, etc)*
- *Metal corrosion with CO<sub>2</sub> well understood (AGRs, lab-scale tests)*
- *Hot enough to reach 48% thermal efficiency, Cool enough that materials risks are minimal*
- *Design condition 150°C temperature rise [Sienicki 2007]*
- *Use of SS316 and other standard materials*
- *Na-CO<sub>2</sub> coupling removes Na-steam interaction, eliminate intermediate Na system?*



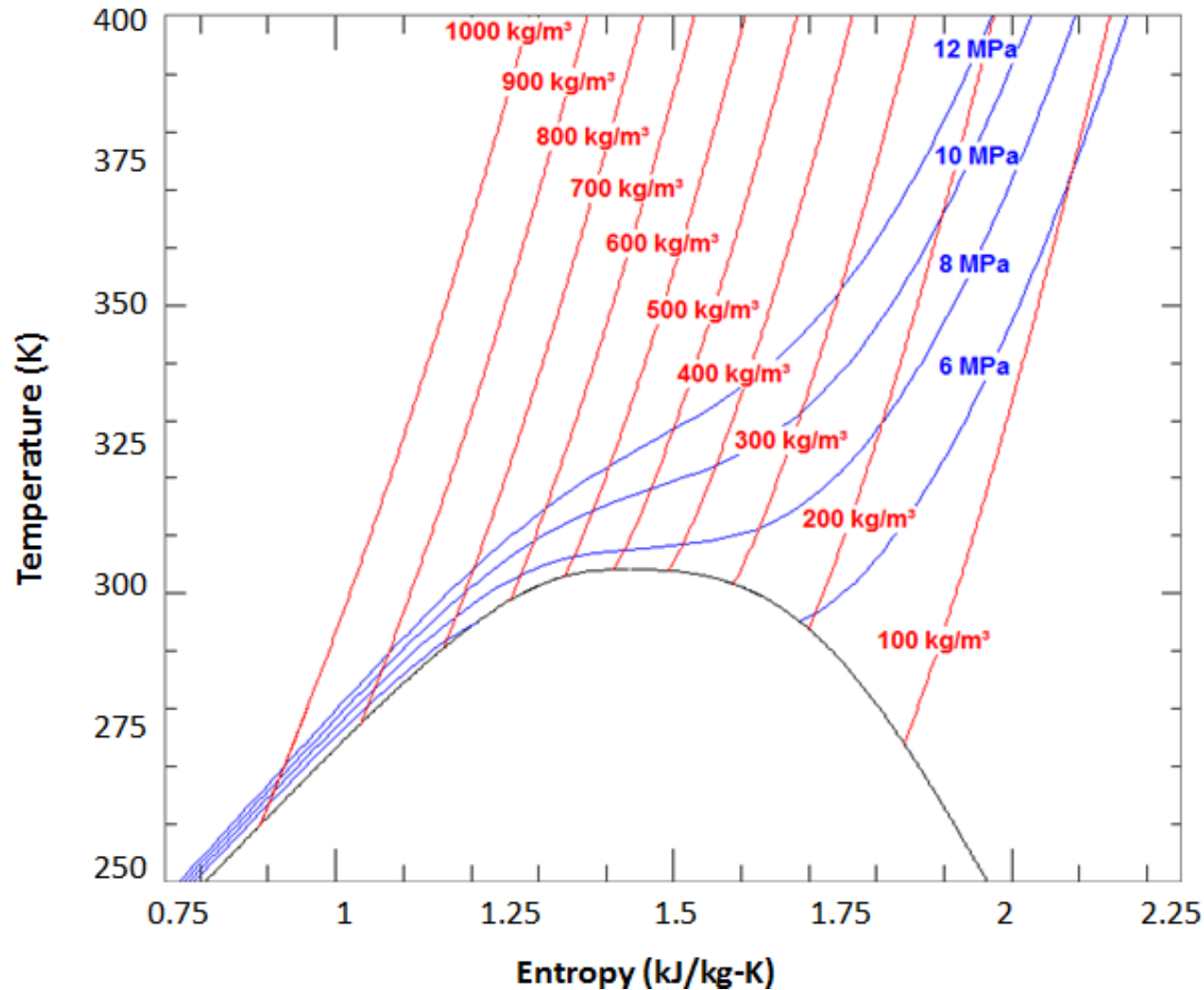
[ANL, 2005]

# S-CO<sub>2</sub> cycle efficiency vs. compressor inlet temp

- *Simulates temperature rise due to change in ambient conditions, ie dry cooling*
- *Near critical point, efficiency is very sensitive to change*
- *Results show that increasing pressure helps, even if max pressure is fixed*

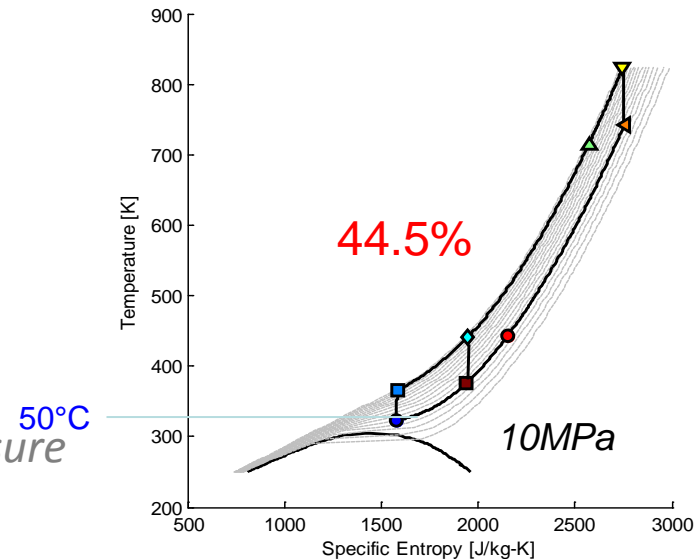
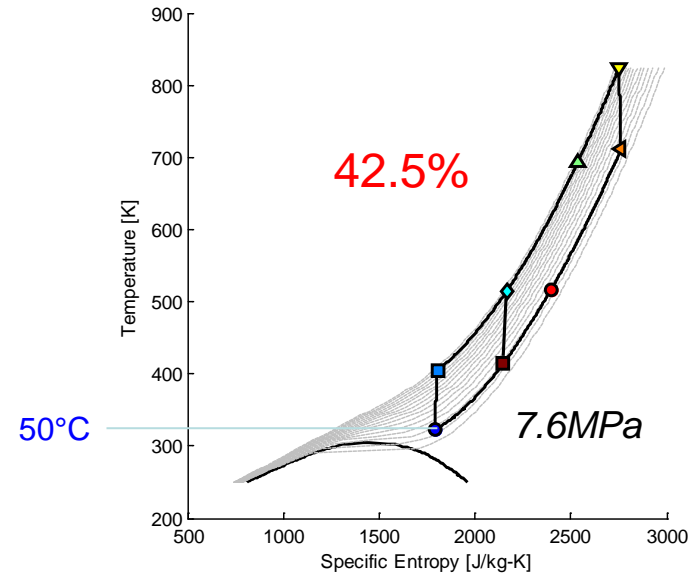
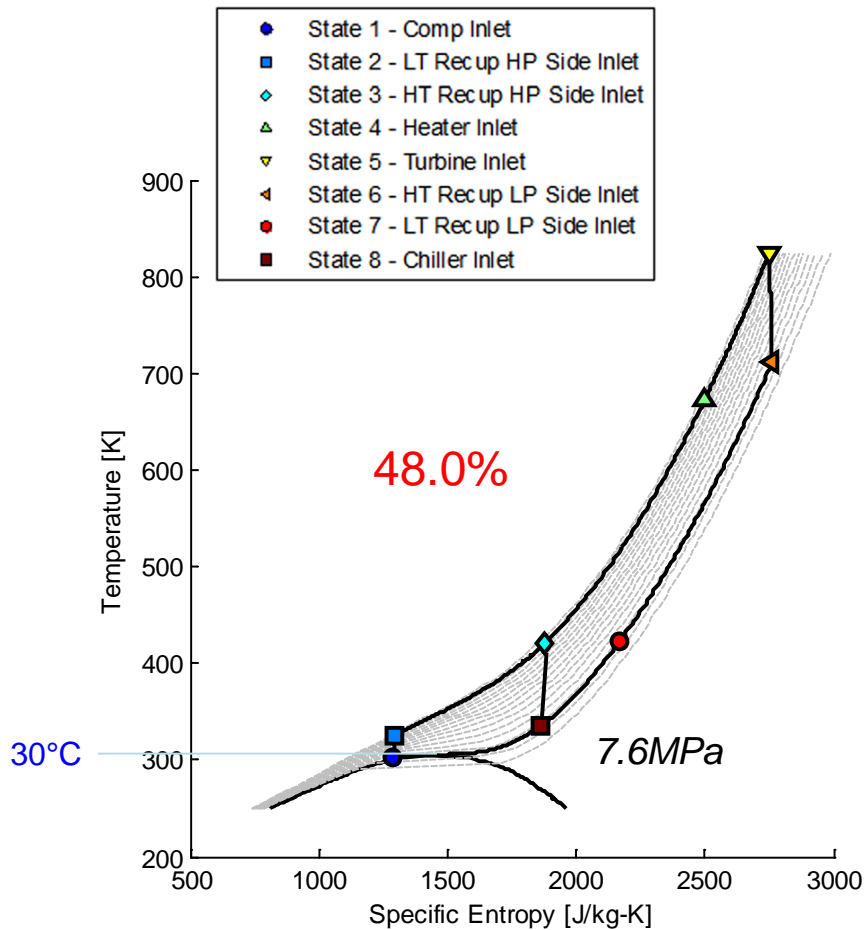


# CO<sub>2</sub> Equation of State



- *s-CO<sub>2</sub> cycle relies on low compressibility, high density of fluid during compression for reduced pumping power*
- *Density increases for a constant temperature by increasing pressure*
- *Analogous chart can be drawn to show CO<sub>2</sub> compressibility*
- *As temp rises, CO<sub>2</sub> conditions are adjusted for min. work*

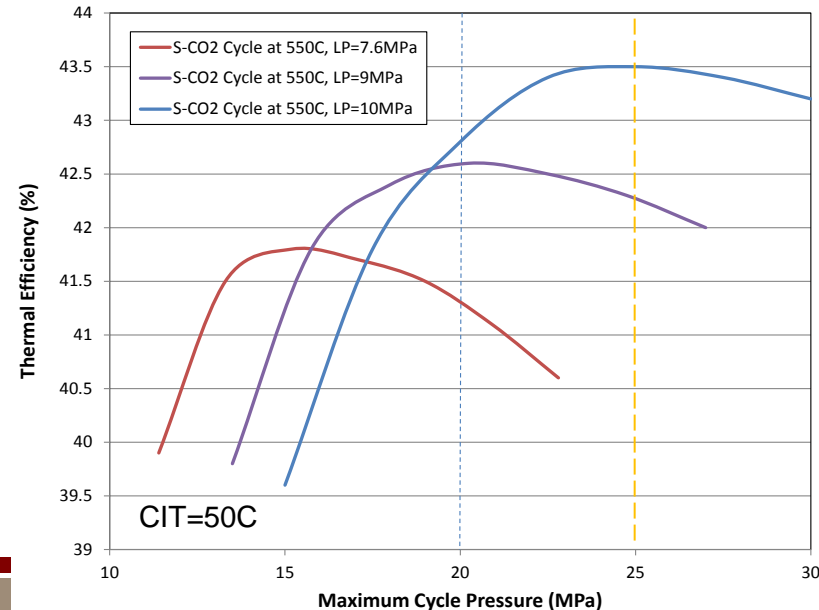
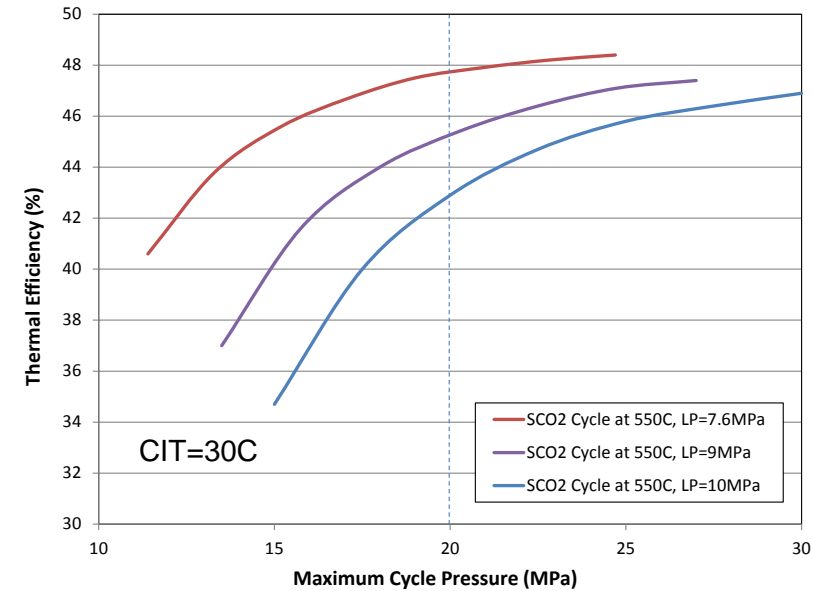
# s-CO<sub>2</sub> cycle optimization



Unlike steam system, cycle min temp and pressure can be controlled independently, as ambient conditions change

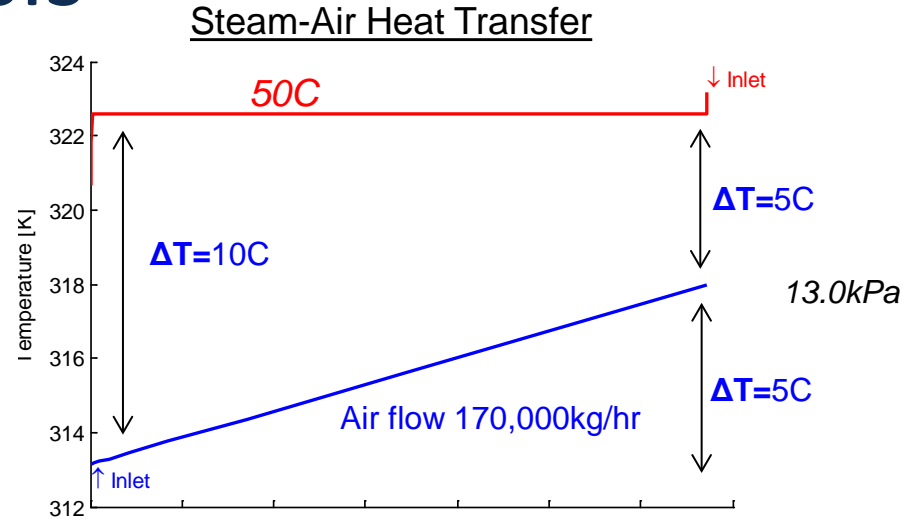
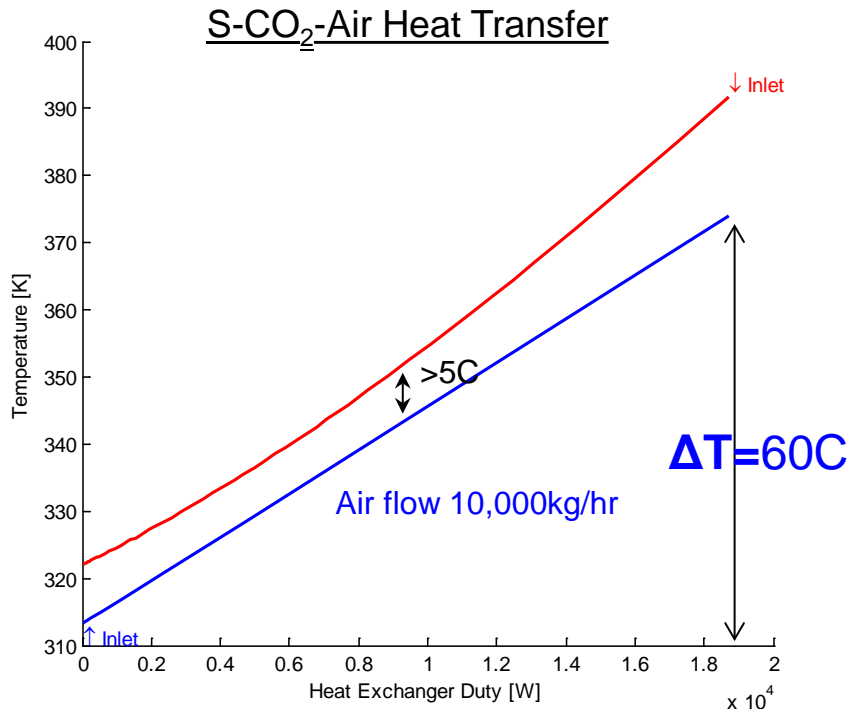
# s-CO<sub>2</sub> cycle optimization

- *Studies show that for a standard s-CO<sub>2</sub> case ( $T_{min}=30C$ ), pressure ratio should be increased as much as is feasible*
- *Baseline high pressure ~20MPa, max for present fossil systems ~25MPa*
- *For a warmer ‘dry-cooled’ s-CO<sub>2</sub> case ( $T_{min}=50C$ ), a distinct peak occurs, high min pressures show clear benefits*



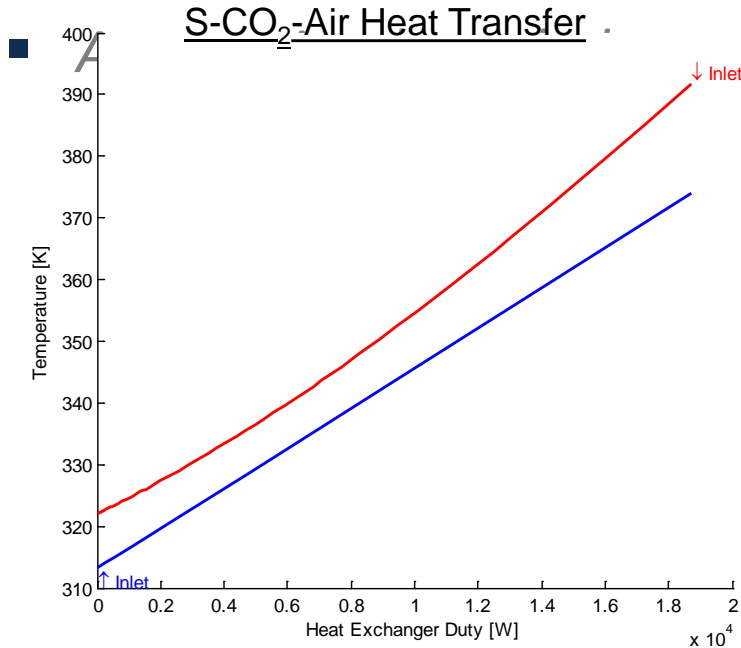
# Air-Cooled HX Analysis

- Assumes ambient air at 40C, no pinches <5C
  - Plots generated using SNL HX design tool [Carlson, 2013]

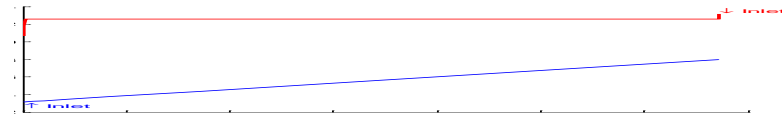


- Two cases:
  - To achieve same heat transfer
  - Air temp = 40C
  - CO<sub>2</sub> inlet temp = 50C
  - same UA value,
  - 17x air flow is needed**
- Not optimized; in practice steam pressure is increased

# Air-Cooled HX Analysis



Steam-Air Heat Transfer



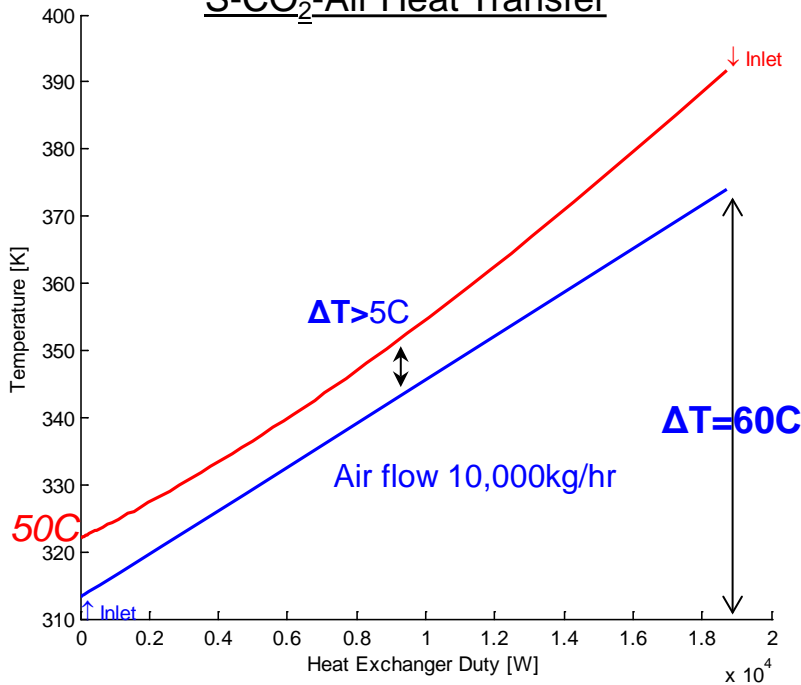
- **Two cases:**
  - achieve same heat transfer
  - Air temp = 40C
  - CO<sub>2</sub> inlet temp = 50C
  - same UA value,
  - **17x air flow is needed**

- **Not optimized; in practice steam pressure is increased at cost of efficiency**

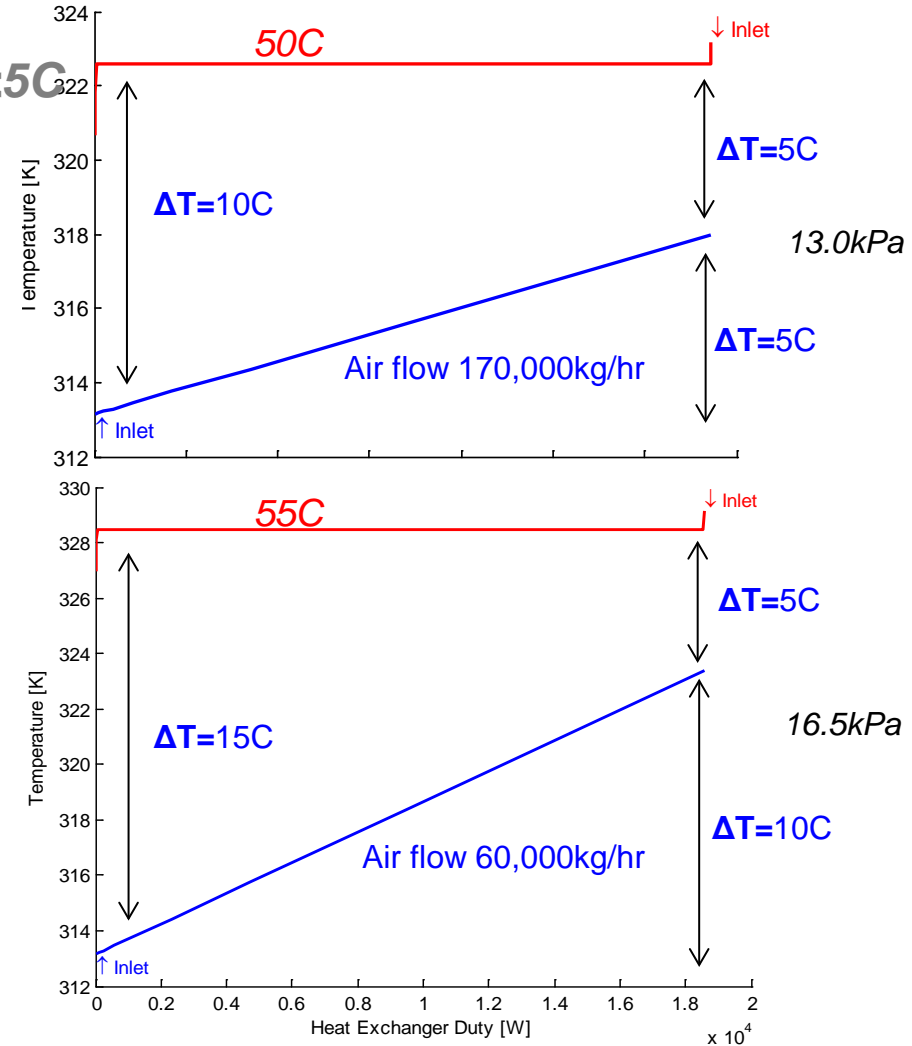
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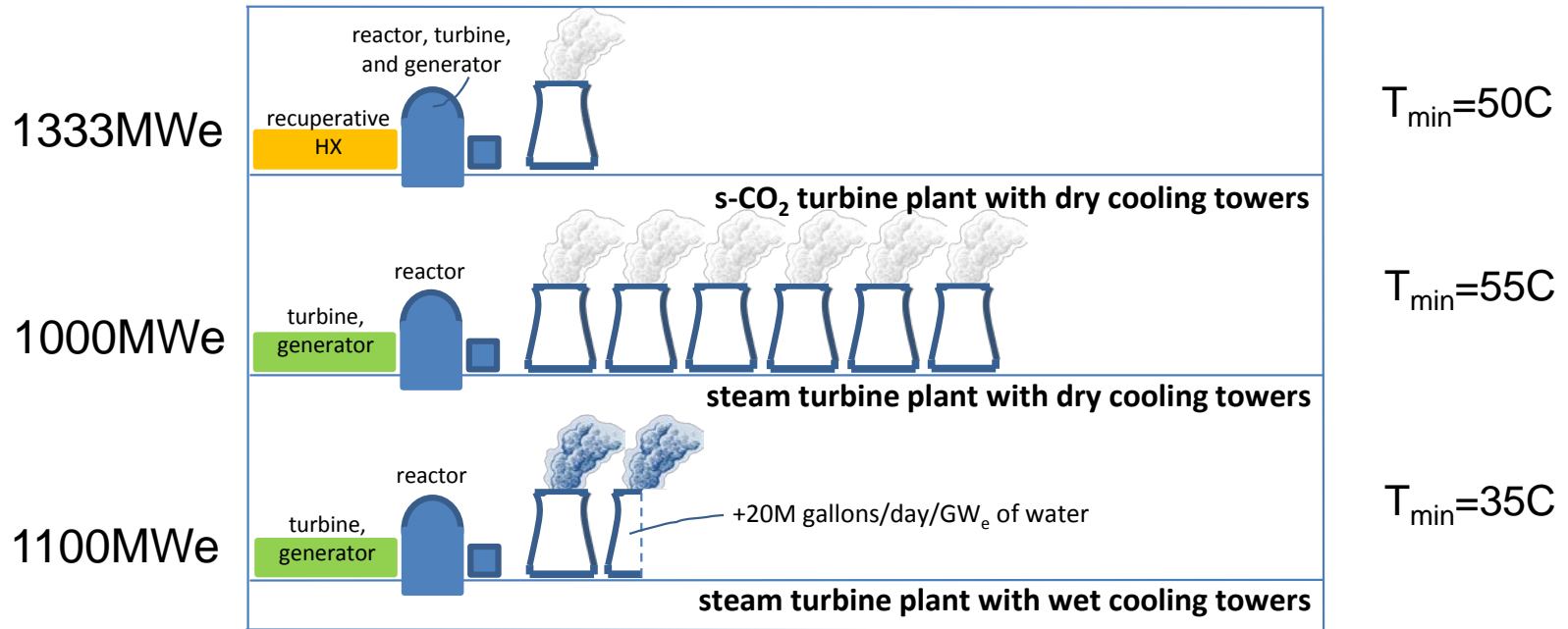
### S-CO<sub>2</sub>-Air Heat Transfer



### Steam-Air Heat Transfer

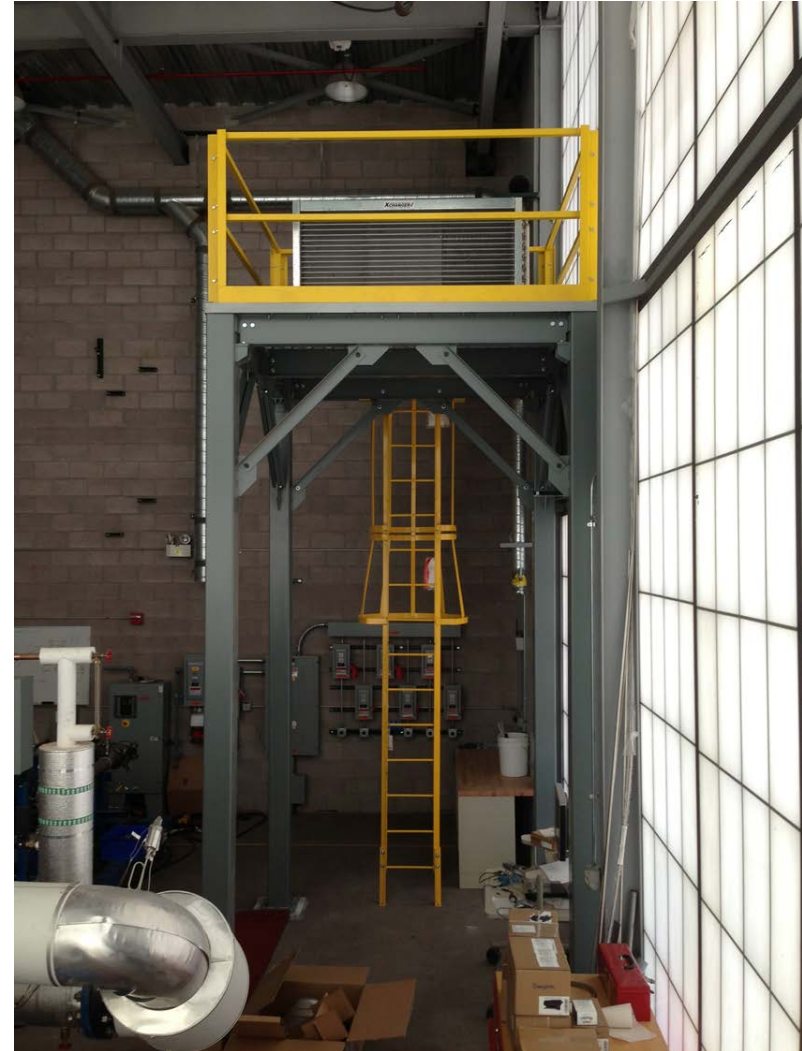


# s-CO<sub>2</sub> allows reduction in dry cooling costs



- ‘Artist’s conception’ of dry-cooled s-CO<sub>2</sub> plant as compared to steam
- Assuming dry natural draft units, s-CO<sub>2</sub> needs 1/6<sup>th</sup> cooling towers of steam-- therefore **cooling tower costs for dry-cooled s-CO<sub>2</sub> may be competitive with wet cooled steam plants, + higher thermal efficiency**

- **Loop has been designed and constructed, testing not yet begun**
  - Dry cooler operation, dry cooler heat exchanger profile temps
  - Passive decay heat removal
  - S-CO<sub>2</sub> inventory control operations
  - S-CO<sub>2</sub> based mixtures
  - Air-cooler surface coatings and advanced geometry for reduced size.
  - High pressure sight-glasses allow for two-phase visualization
  - S-CO<sub>2</sub> pumping power at alternate inlet conditions



# S-CO<sub>2</sub> natural circulation

- Increased reliance on passive emergency cooling using natural circulation is a major goal of next generation SMRs
- Natural circulation is driven by gravitational head resulting from elevation and density differences in a closed loop.
- Supercritical CO<sub>2</sub> is ideally suited for establishing natural circulation flow
  - Large density variations are achieved for only small elevation in temperature
  - This allows natural circulation to occur earlier in the emergency transient, provide increased flow throughout
  - Viscous-to-buoyancy forces of a given fluid are a good indicator of natural circulation potential (Grashof number);
  - At conditions of SCO<sub>2</sub> heat rejection system, the Grashof number is 4 orders of magnitude higher than water for a 10C temperature rise (1)

$$Gr = \frac{g \beta \rho^2 Lc^3 \Delta T}{\mu^2}$$

| Parameter        | Units             | CO <sub>2</sub> | Water   |
|------------------|-------------------|-----------------|---------|
| Bulk Temperature | K                 | 300             | 300     |
| Pressure         | MPa               | 7.69            | 7.69    |
| β                | 1/K               | 0.039           | 0.00037 |
| ρ                | kg/m <sup>3</sup> | 275.6           | 996.7   |
| μ                | Pa-s              | 2.21e-5         | 6.94e-4 |
| Gr               |                   | 5.9e14          | 7.5e10  |

\*Parameters were evaluated at conditions typical of an SCO<sub>2</sub> power heat rejection system.

(1) Parma, et al, (2011)

# Conclusions

- ***Dry cooling has not penetrated commercial steam plants because of economics, fundamental limits of air-cooled condensation***
- ***For LWRs, water needs extend beyond the cooling tower***
  
- ***S-CO<sub>2</sub> power cycles can achieve higher efficiency with elevated inlet temps, but this alone is not enough to make the case***
- ***For a number of reasons, s-CO<sub>2</sub> can be well matched with air-cooling, cooling tower costs for dry-cooled s-CO<sub>2</sub> appear to be similar to wet-cooled steam***
- ***More detailed analysis is needed***
- ***Experiments underway***
  
- ***Siting map should be reconsidered***
- ***Safety and regulatory issues***
- ***Application to many different reactor types could help fuel the rise of all Gen-IV systems***

