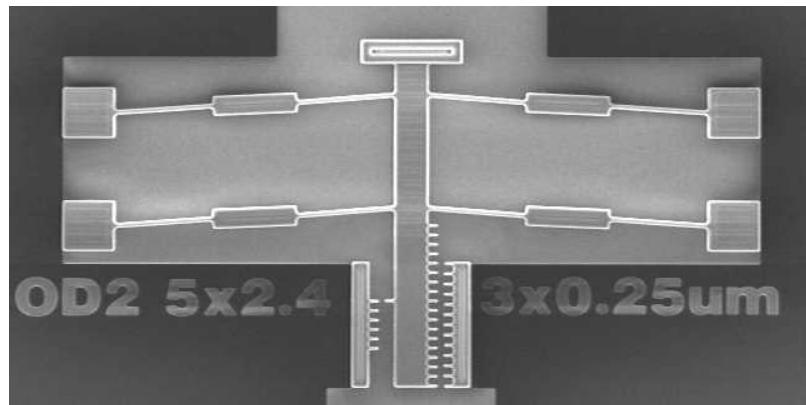


Design Space Exploration for Robust Global Optimization of Micro Bistable Mechanisms

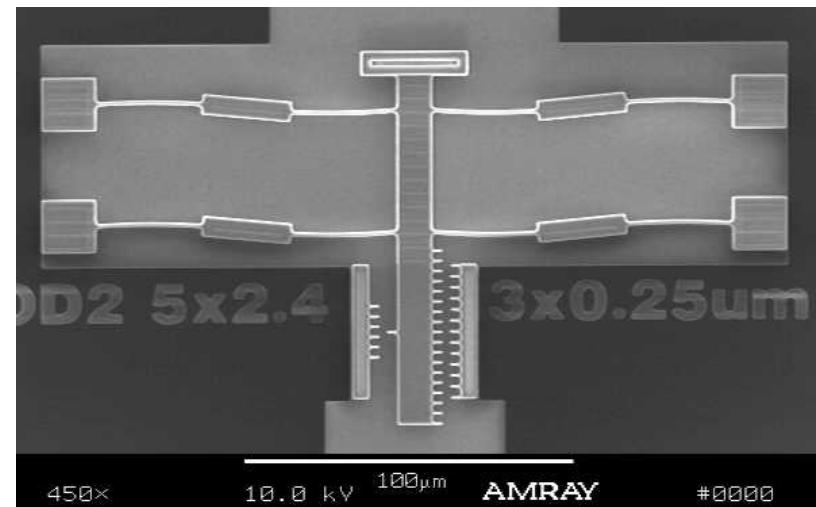
Jonathan W. Wittwer
Michael S. Baker
Brian M. Adams

Sandia National Laboratories

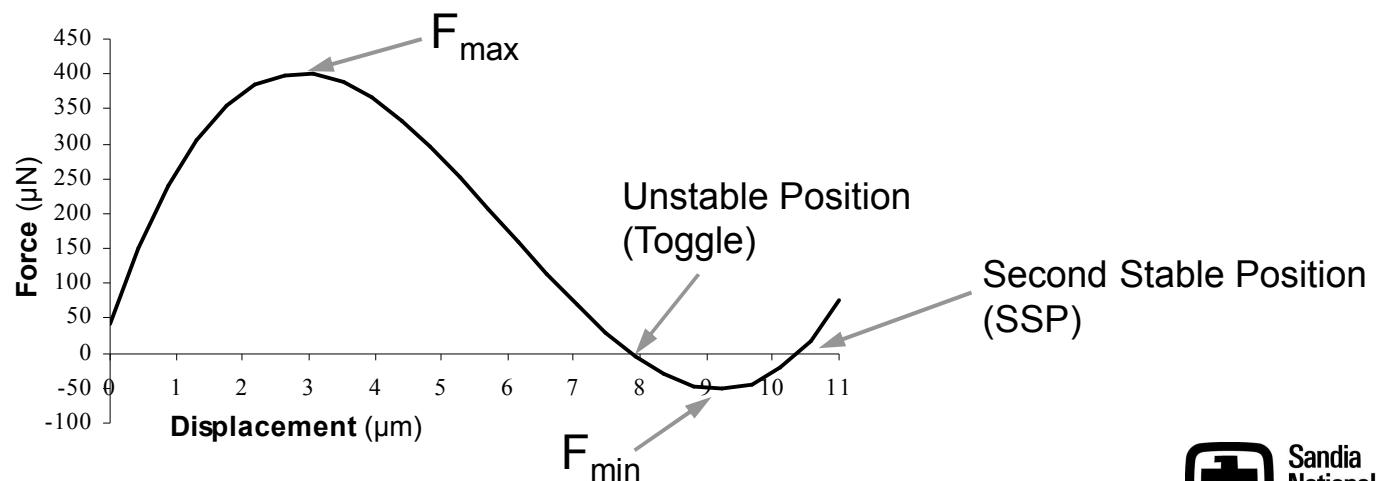
Bistable Micro Mechanism



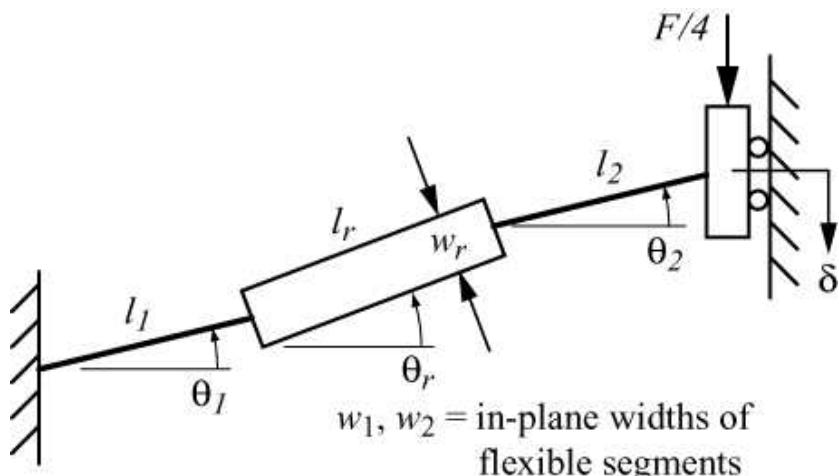
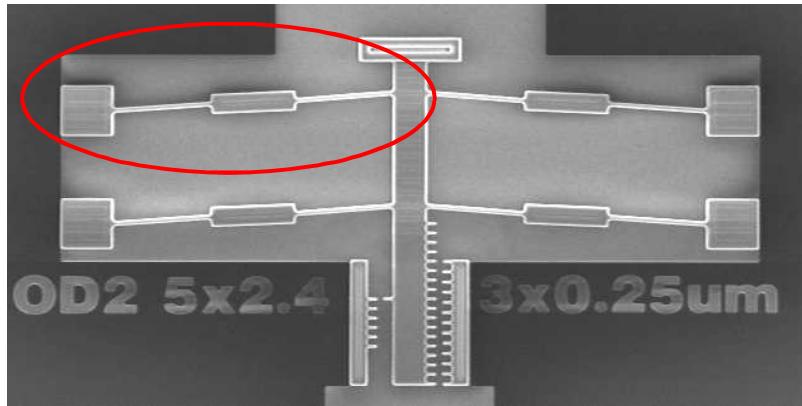
As Fabricated (First Stable Position)



Second Stable Position (SSP)



Model and Design Parameters



Quarter Model

Model Details

1. Matlab-Ansys

- 1D nonlinear, full model
- 2.25 minutes / eval

2. Dakota-ARIA

- 2D nonlinear, $\frac{1}{4}$ model
- 5.7 minutes / eval

Design Parameters

- Beam lengths: L_1, L_2, L_r
- Beam angles: $\theta_1, \theta_2, \theta_r$
- Beam widths

$$L_1 = L_2$$

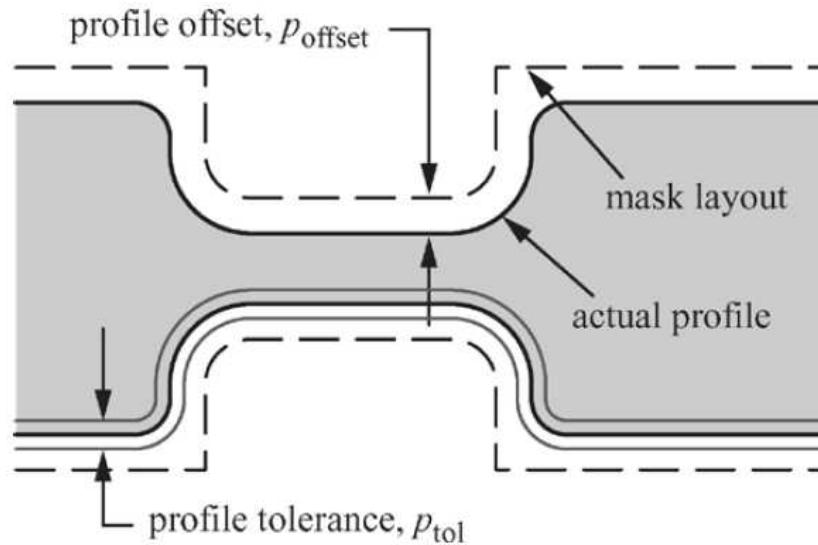
$$\theta_1 = \theta_2$$

$$w_1 = w_2$$



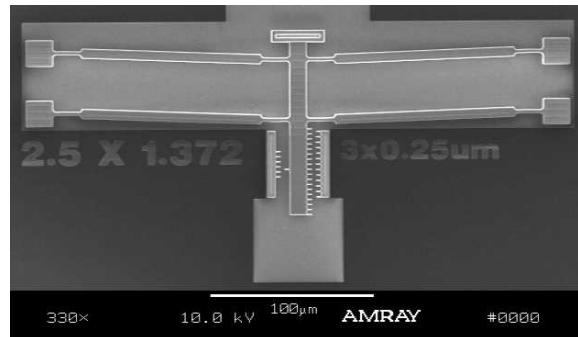
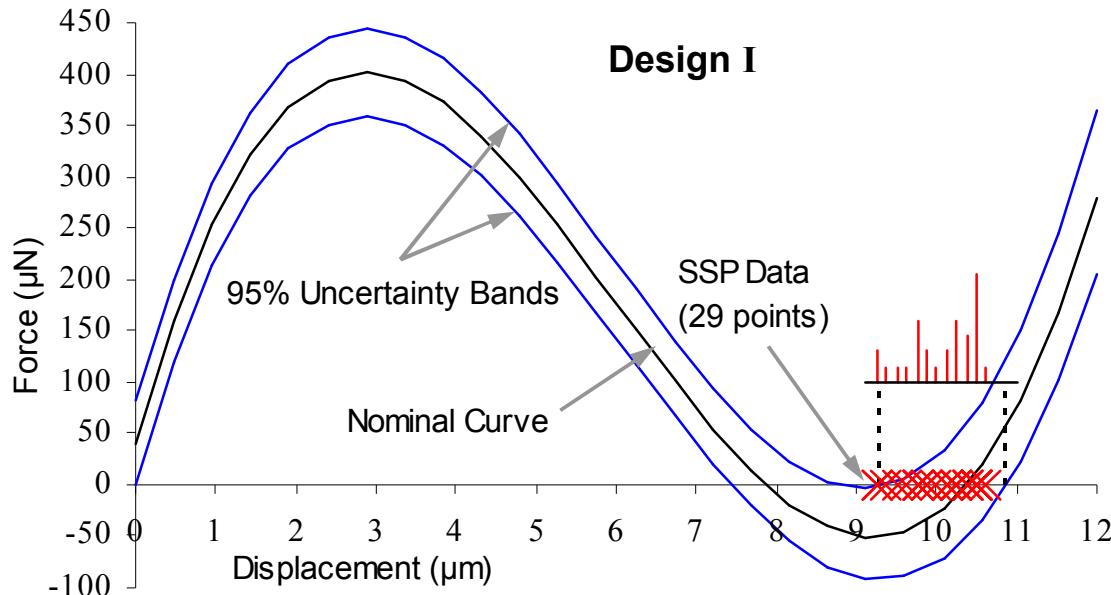
Uncertainties

1. Residual Stress: $-10 \pm 6.4 \text{ MPa}$ (compressive)
2. Edge Bias (due to Lithographic variations)
 - Results in a profile offset
 - Nominal: $0.1 \mu\text{m}$ per edge
 - Variation across a wafer: $0.02 - 0.18 \mu\text{m}$

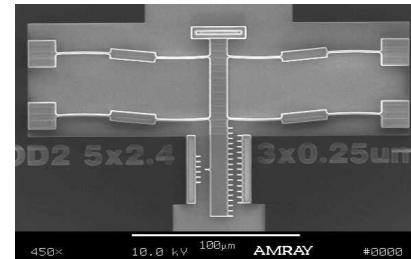
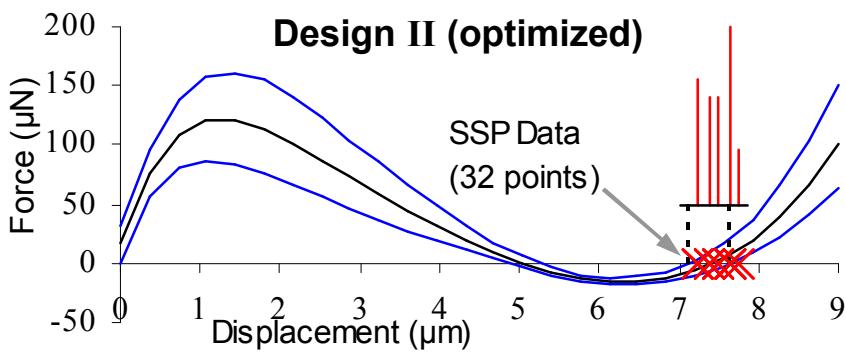




Robust Optimization



$$F_{\min} = -51 \pm 47 \mu\text{N}$$



$$F_{\min} = -16 \pm 3 \mu\text{N}$$

SSP data courtesy of Mike Baker, Sandia National Laboratories
SEM courtesy of Jeremy Walraven, Sandia National Laboratories



Motivation for Design Space Exploration

- We frequently design new bistable mechanisms
 - Different Forces and Displacements
 - Different Fabrication Processes
- Past design optimization has shown there to be many local minima
- Random starting locations leads to a great deal of wasted analysis time
- Perhaps the design space can be reduced

Design Space Exploration (6 Variables)

- **Grid Sampling**
 - Too many points, $10^6 = 1,000,000$ points
- **Latin Hypercube Sampling**
 - Can choose the number of points
 - More effective space-filling than Monte Carlo simulation

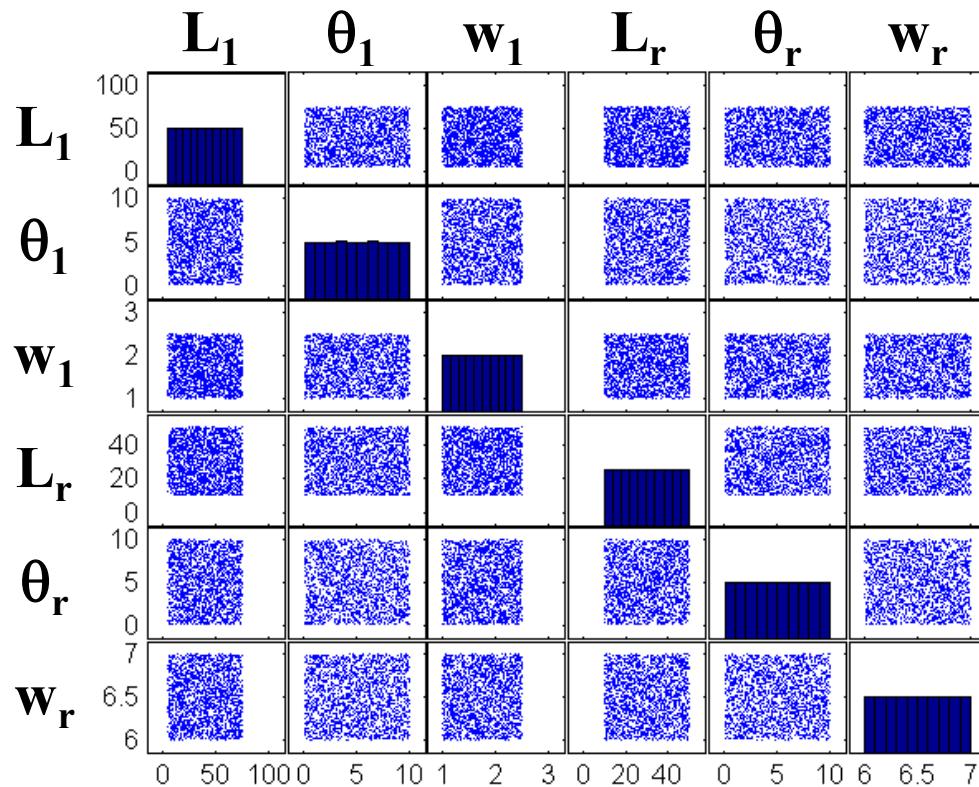


$$X = \begin{bmatrix} L_1 & \theta_1 & w_1 & L_r & \theta_r & w_r \\ 21.71 & 4.43 & 1.12 & 47.67 & 3.97 & 6.39 \\ 13.81 & 1.15 & 1.51 & 19.09 & 4.72 & 6.97 \\ 6.06 & 3.54 & 1.89 & 45.89 & 1.52 & 6.99 \\ \dots & \dots & \dots & \dots & \dots & \dots \end{bmatrix} \xrightarrow{Y=} \begin{bmatrix} S_{\max} & F_{\max} & F_{\min} \\ 1308.6 & 156.8 & \dots & -58.5 & \dots \\ 1034.3 & 182.5 & \dots & 64.8 & \dots \\ 2094.6 & 516.1 & \dots & 581.5 & \dots \\ \dots & \dots & \dots & \dots & \dots \end{bmatrix}$$

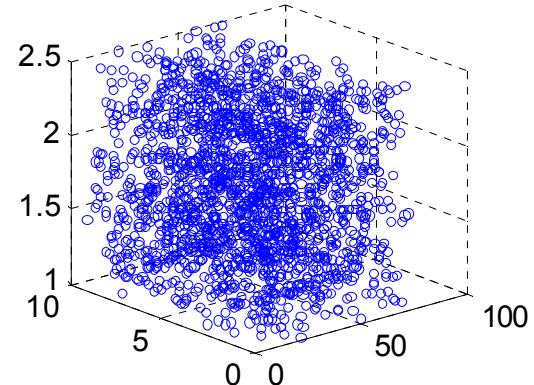
Iterator: Evaluates 1 row at a time, or parallelized

Design Space Visualization – Plot Matrix

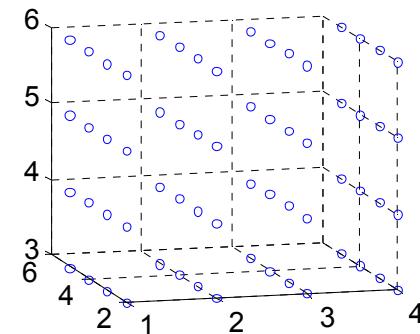
- **plotmatrix(X,X)**
 - Creates a matrix of 2D scatter plots
 - Histograms for each variable along the diagonal



3D Projection of LHS Design
2000 points



3D Projection of Grid Design
4096 points (4^6)



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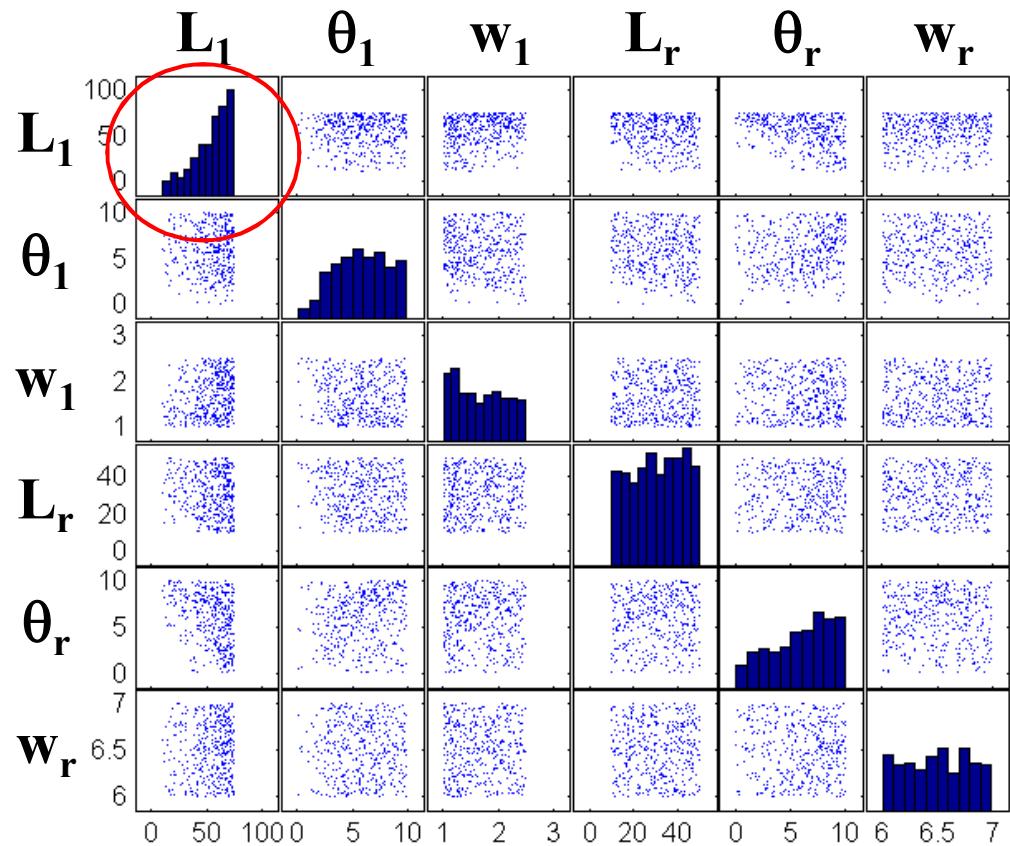
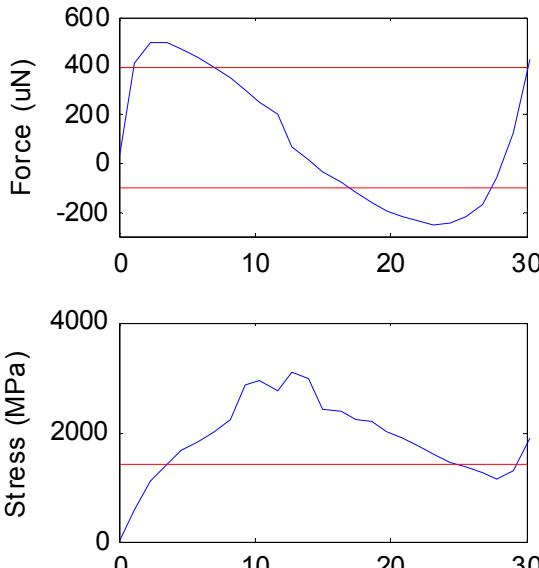
Simulation Run Times

- Matlab – Ansys
 - 2000 Evaluations
 - 1 Processor @ 2.25 minutes / eval ~ 75 hours
 - Ran over the weekend, and stopped it when I came into work (ran 1733 Evals in 65 hours)
 - 411 Failures
- DAKOTA – Aria
 - 10,000 Evaluations
 - 100 Processors @ 5.75 minutes / eval ~ 10 hours
 - Actually ran in two stages (6 nodes then 50 nodes) for a total wall time of about 27 hours
 - 793 Failures
- *All subsequent slides show the Matlab-Ansys results*



Failures

- Only obvious trend is that most failures occurred for the longer beams (L_1)
- Main cause for failure was non-convergence due to buckling
- Failures scattered throughout the design space



We don't want buckling, so failure to converge is a convenient filter

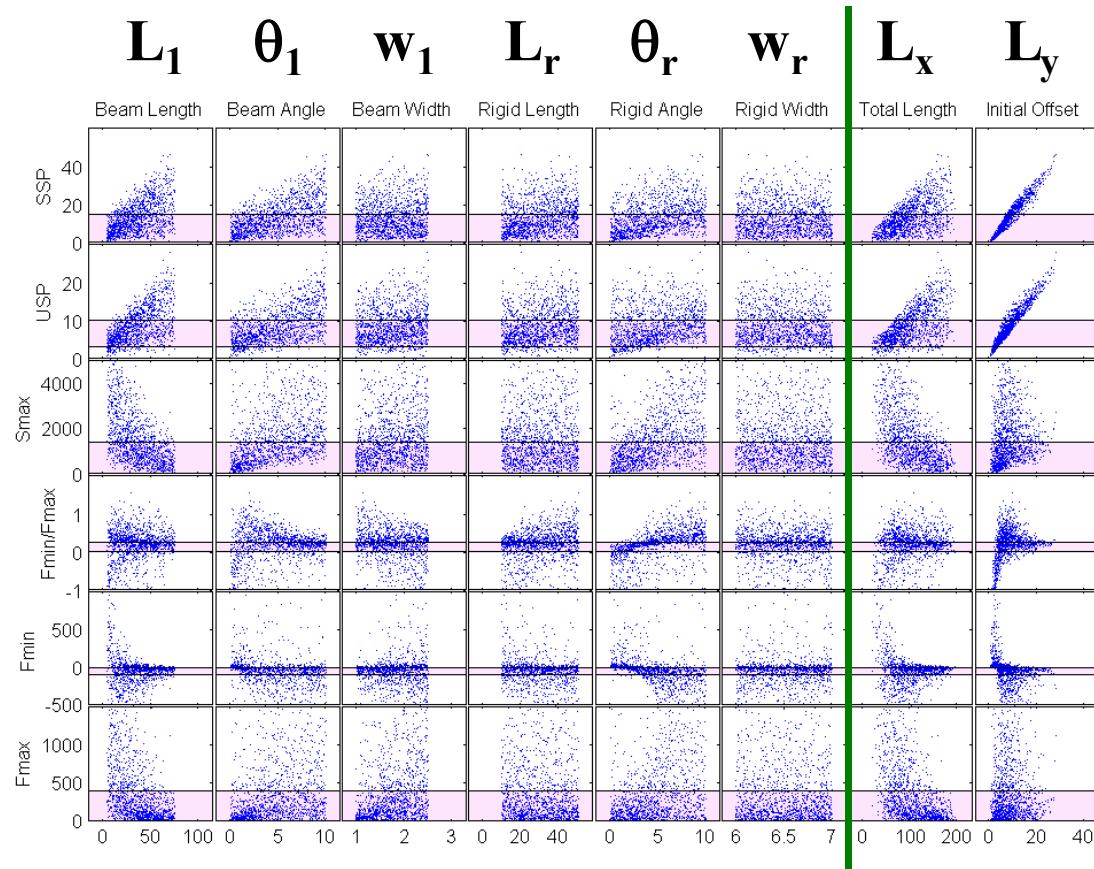


Find Set of Starting Points for Optimization

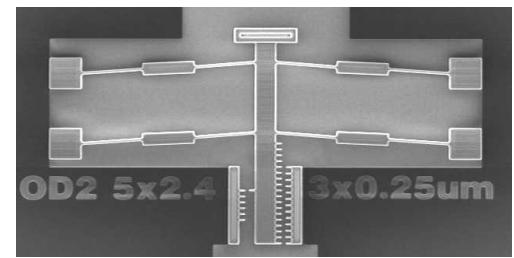
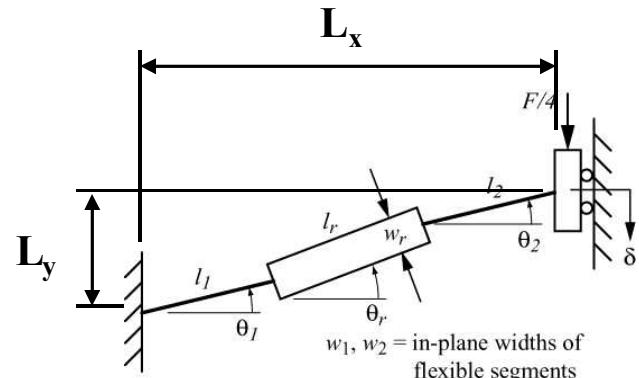
- Started with 1733 Designs, 411 of these were Failures
 - Filtered by Second Stable Position < 15 μm
 - 445 (445) Total Unqualified (Filtered in Sequence)
 - Filtered by Unstable Equilibrium Position > 3 μm
 - 82 (82)
 - Filtered by Maximum Stress < 1400 MPa
 - 593 (332)
 - Filtered by Bistability (Must be Bistable)
 - 318 (173)
 - Filtered by Force Ratio < 0.25 or $F_{\text{max}} > 8*F_{\text{min}}$
 - 571 (119)
 - Filtered by Maximum Force < 400 μN
 - 454 (5)
 - Filtered by Minimum Force > -100 μN
 - 425 (0)
- Ended with 156 Points to Feed to Optimization

Response Analysis and Constraints

Responses



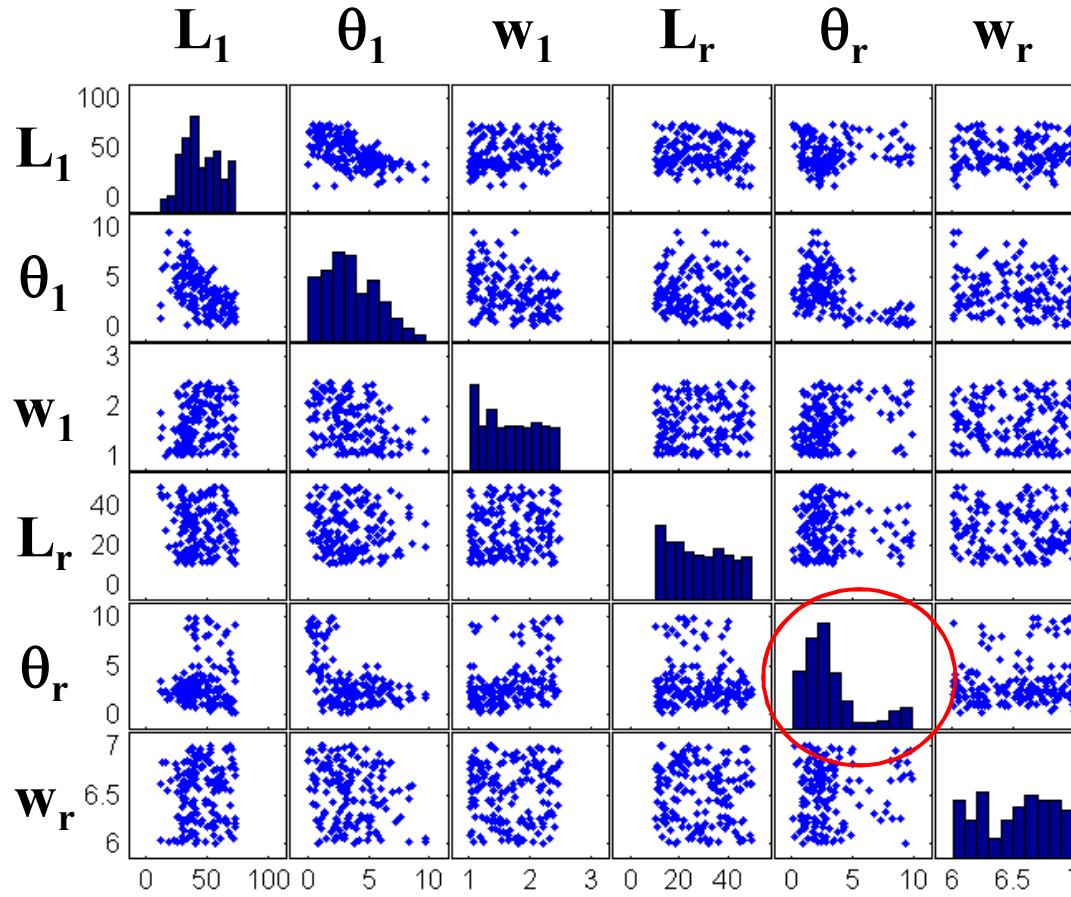
- Filtered out all points NOT in the pink region
- Can pre-filter (no need to analyze) designs with $Ly > 15 \mu\text{m}$



Filtered Design Space (156 points)

Observations

- Some clustering, but very little except for the angle of the rigid segment
- All of the results to this point lead to the conclusion that the design space is highly nonlinear



Optimization

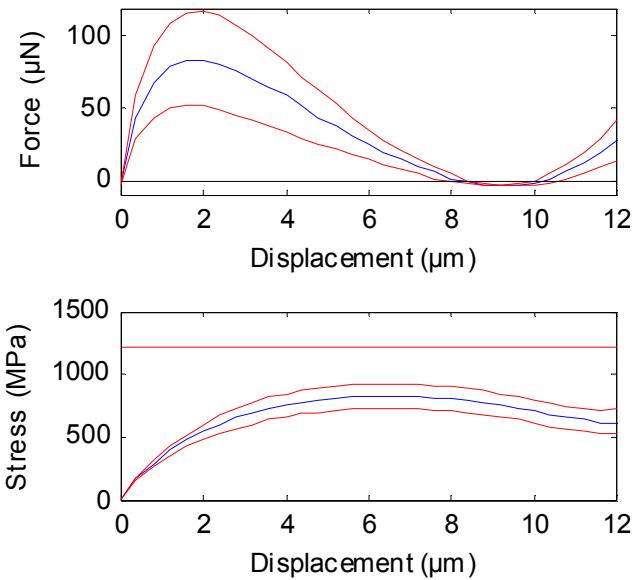
- Starting Points fed into a Robust Optimization routine
 - Nested Algorithm using SQP for Optimization where each function call runs an uncertainty analysis (MVFOSM)

- Objectives

- minimize uncertainty in F_{\min} , (σ/F_{\min}) weight: 3
- minimize the sensitivity to an off axis load weight: 1
- hit target by minimizing $(F_{\text{target}} - F_{\min})^2$ weight: 6

- Constraints

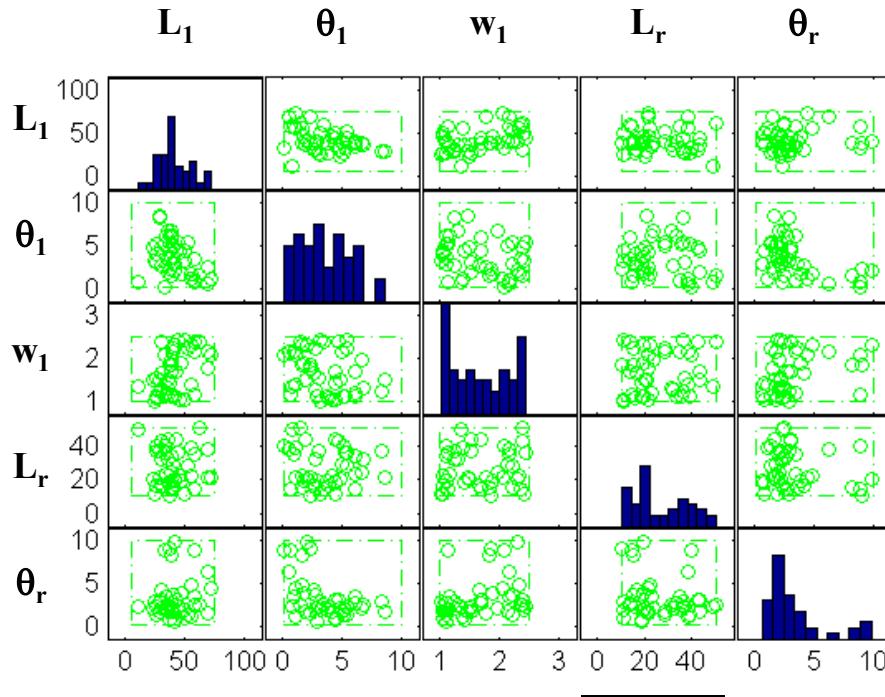
- Stress (S_{\max}) + $2\sigma < 1200$
- Force Ratio (F_{\min}/F_{\max}) + $2\sigma < 0.2$
- Second Stable Position + $2\sigma < 12$
- Toggle Position (USP) + $2\sigma > 3$



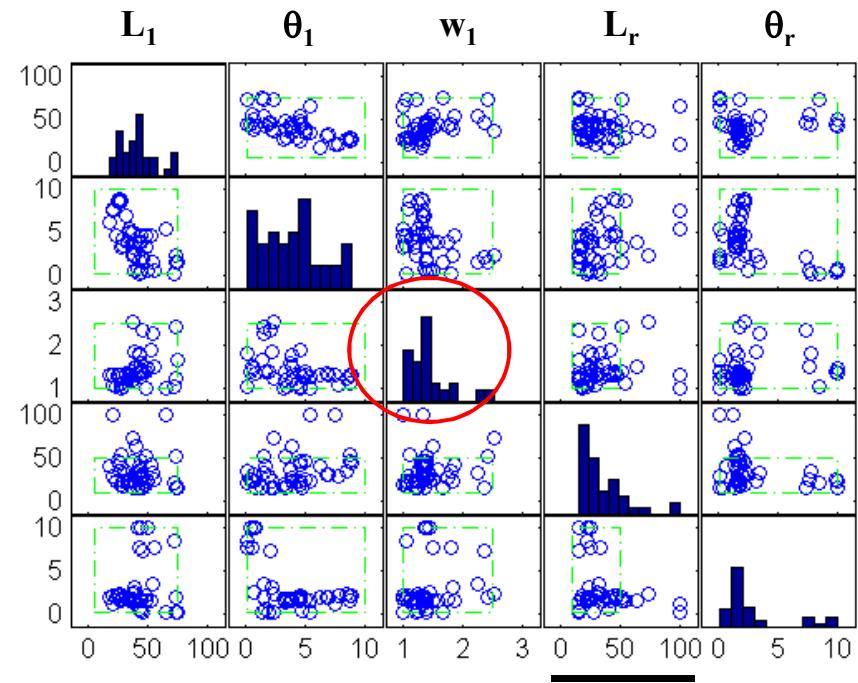
Optimized Designs

- Ran the first 50 points from the filtered set of 156 starting points
- 7 of these failed due to convergence issues
- 11 were very poor optimal designs (trapped in local minima)
- *No clear global optimum*

Starting Values

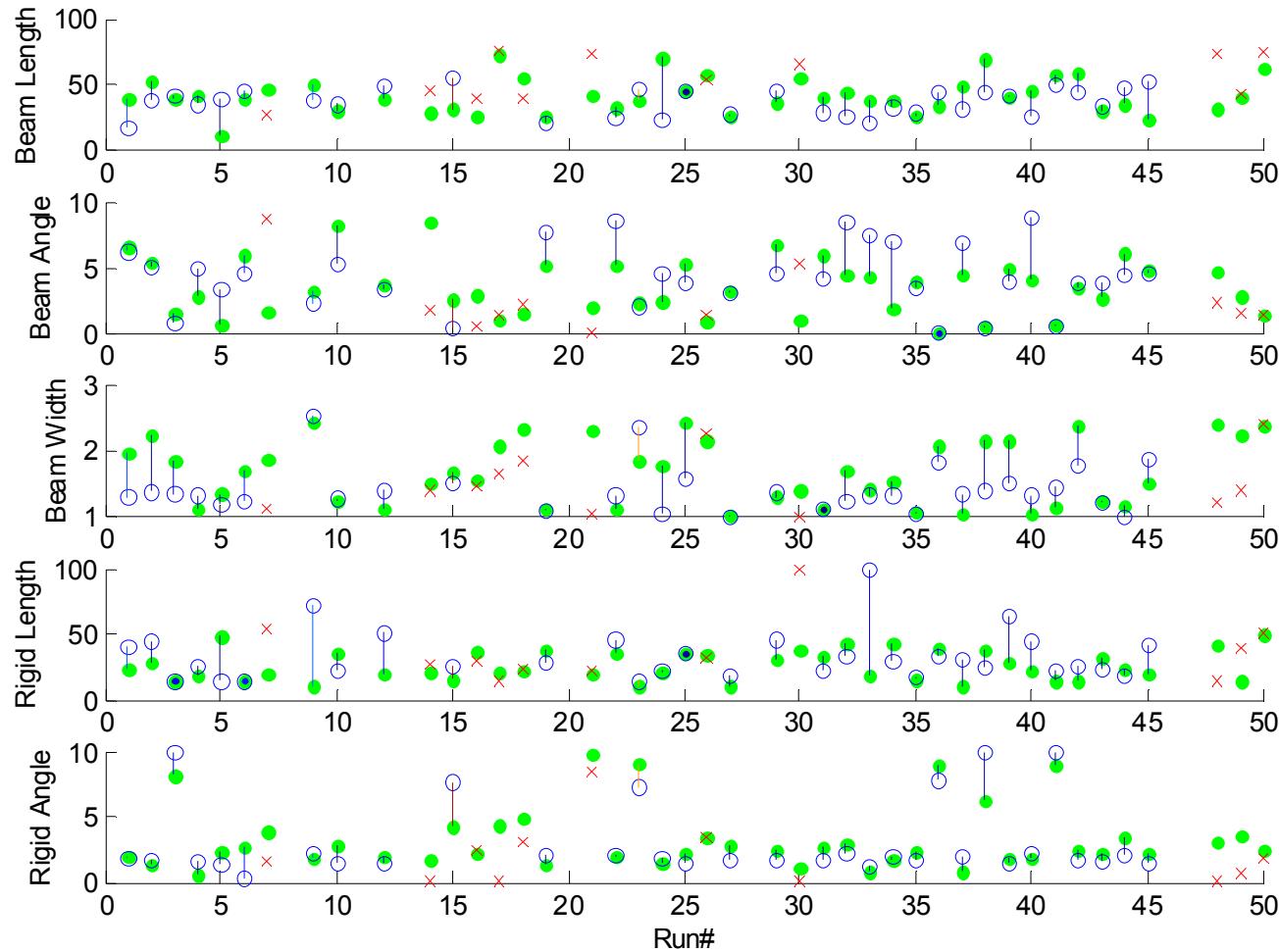


After Optimization

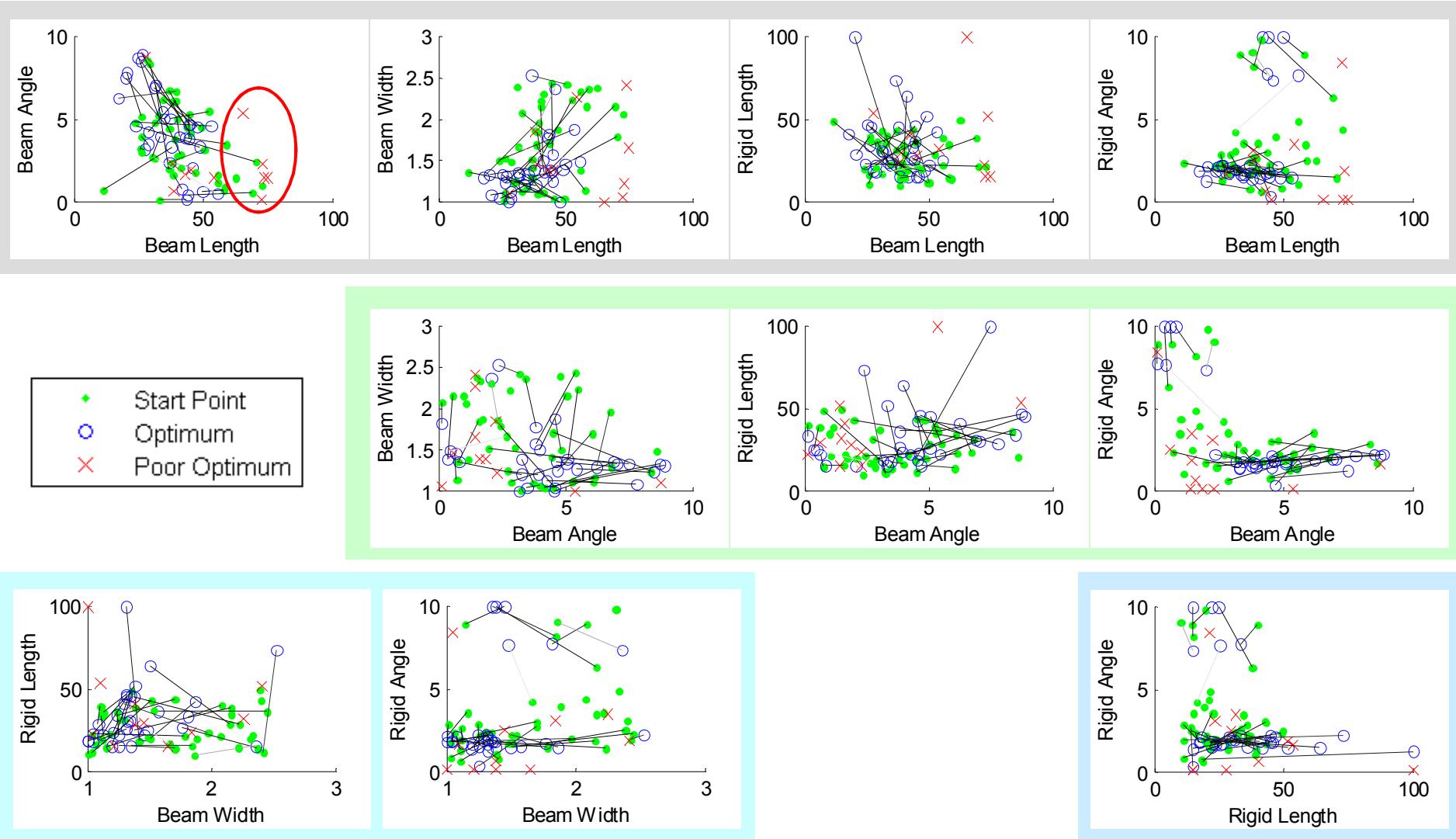


1D Projections (5 Variables = 5 graphs)

- **Green = Start Points, Blue = Optimum, Red = Poor Optimum**

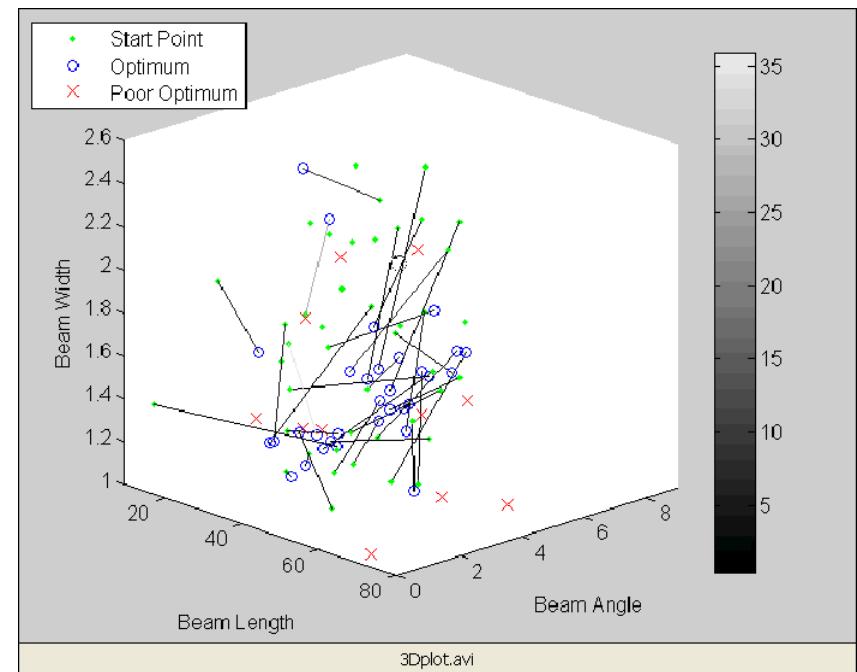
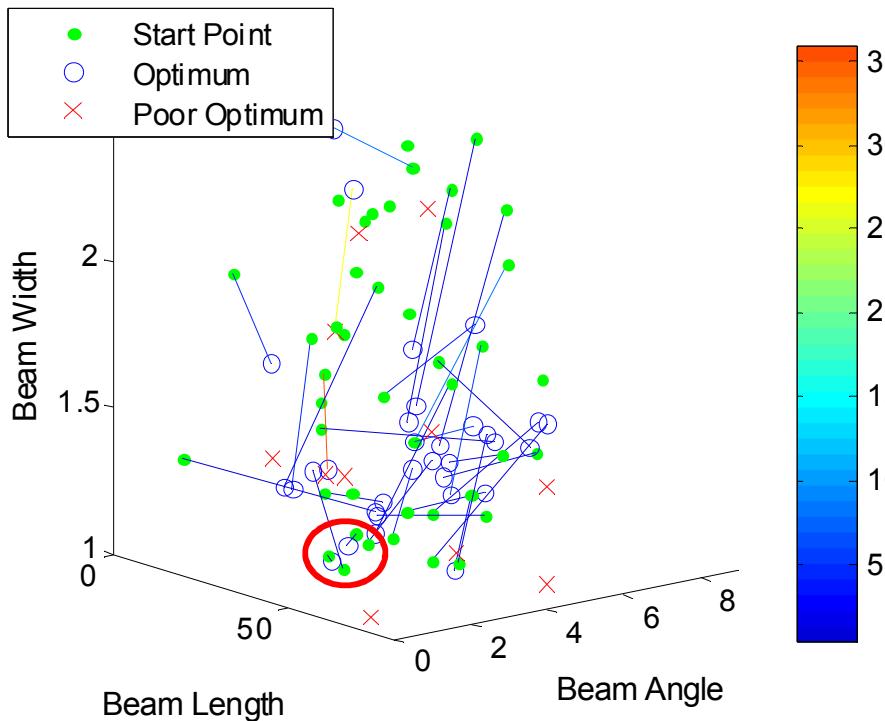


2D Projections (5 Variables = 10 graphs)



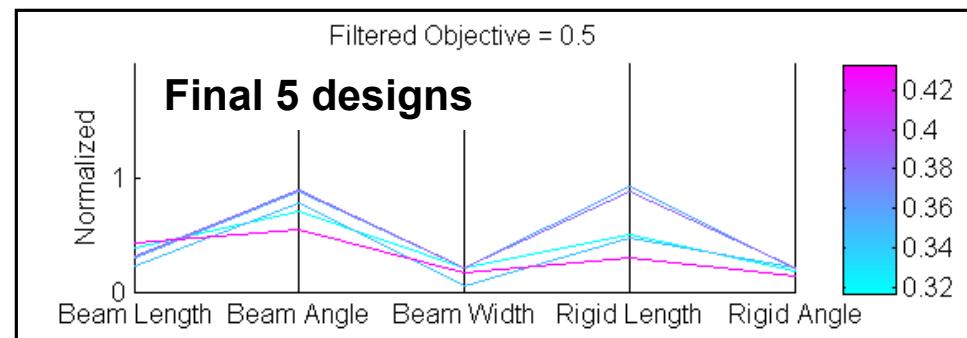
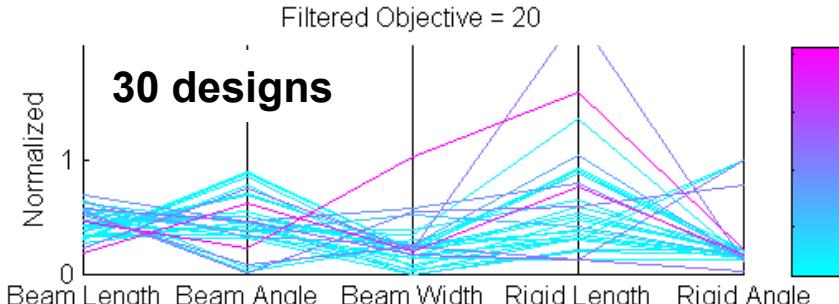
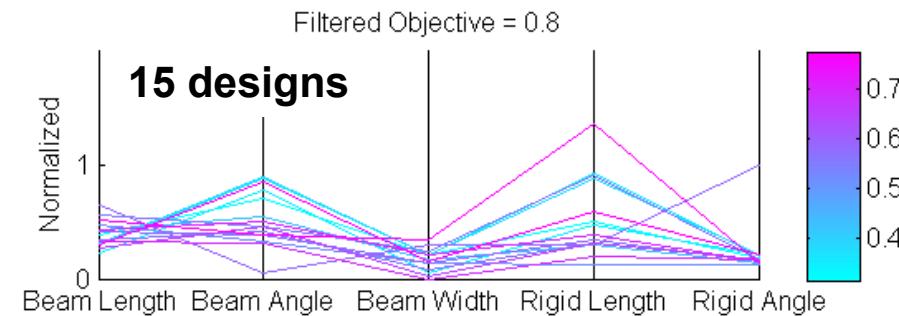
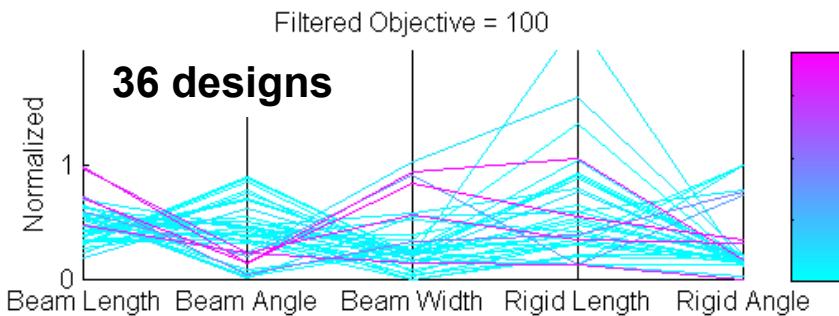
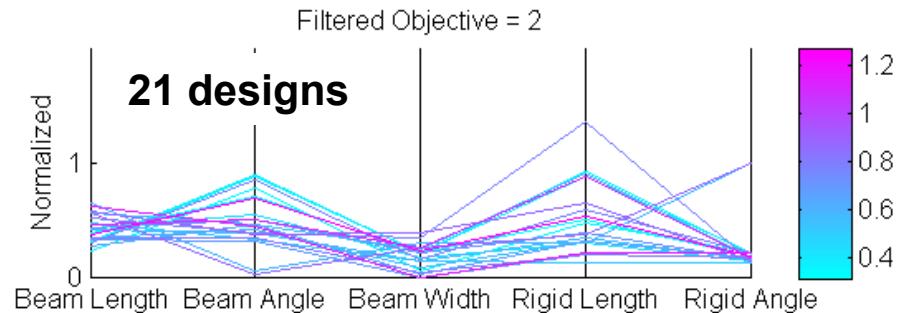
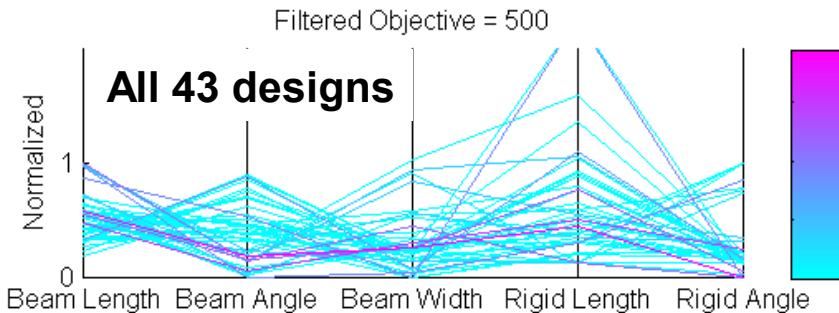
3D Projections (5 Variables = 9 graphs)

- Rigid Length & Angle (would need 9 to see all combinations)



Design Selection

- How are the best-of-the-best optima clustered?



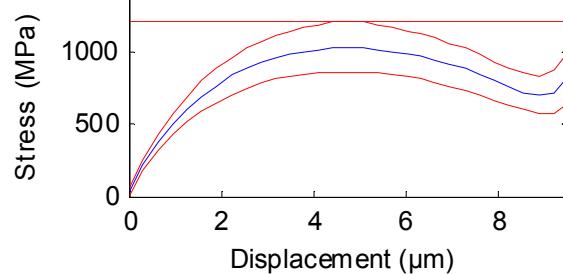
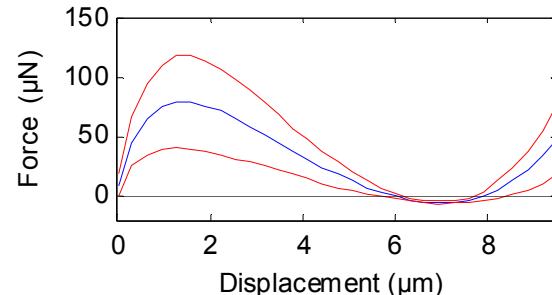
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Design Selection & Verification

- Top 5 optimum designs
- Compare performance metrics

| Run# | Obj | Size (μm) | Z Stiff | F_{\min} (μN) | F_{\max}/F_{\min} | SSP (μm) | σ/F_{\min} | $S_{\max}+2\sigma$ (MPa) |
|------|-------|---------------------------|---------|---------------------------------|---------------------|--------------------------|-------------------|-----------------------------|
| 34 | 0.317 | 93.09 | 0.90 | -4.98 | 12.90 | 10.5 | 10.2% | 947.3 |
| 19 | 0.345 | 69.26 | 0.96 | -4.97 | 11.00 | 7.8 | 11.3% | 1197.0 |
| 22 | 0.357 | 96.50 | 0.91 | -4.97 | 12.22 | 10.8 | 11.5% | 1193.2 |
| 40 | 0.384 | 96.95 | 0.90 | -5.09 | 12.60 | 11.4 | 10.8% | 1197.0 |
| 10 | 0.433 | 91.11 | 0.88 | -4.87 | 10.03 | 8.9 | 10.5% | 718.1 |

- Verify analysis after rounding the design variables
- Verify results with a higher fidelity model





Summary

- **Confirmed that the design space is highly nonlinear, with many local minima**
- **Used filtered set of starting points to come up with 7 different designs**
- **Recommendations on visualization methods for these types of multi-dimensional analyses would be appreciated**
- **Future work might include performing other types of global robust optimization algorithms**