

Fatigue Testing of 9 m Carbon Fiber Wind Turbine Research Blades

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Background, Purpose, and Overview

- **Background**

- SNL initiated a blade research program in 2002 to investigate the use of carbon in subscale 9 m blades
- 7 CX-100 and 7 TX-100 blades were manufactured
- Blades from each set have undergone modal and static tests

- **Purpose of Fatigue Tests**

- Verify that blades met their design criteria
- Investigate unique structural aspects of the blades
- Examine the use of advanced sensors

- **Overview**

- Carbon in blades
- 9 m Blade Designs
- Test Setup
- Test Results
- Conclusions

Carbon in Blades

- **Advantages:**

- High stiffness/
modulus
- Highly orthotropic
- Excellent fatigue
resistance
- Strong fibers

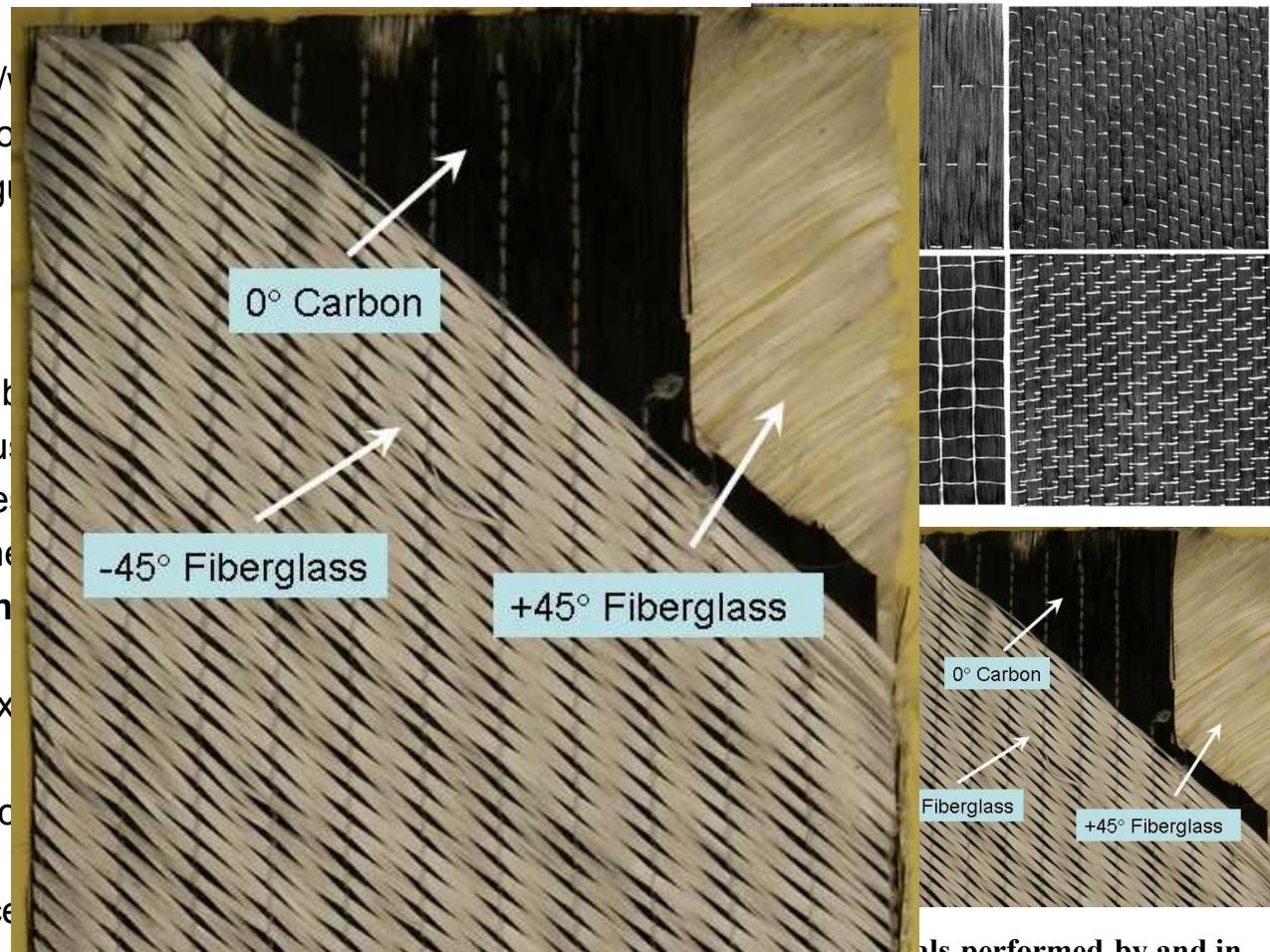
- **Disadvantages:**

- Higher cost
- Limited availability
- Difficult to infuse
- Poor properties
- Possible stiffness
anisotropy

- **Potential solution:**

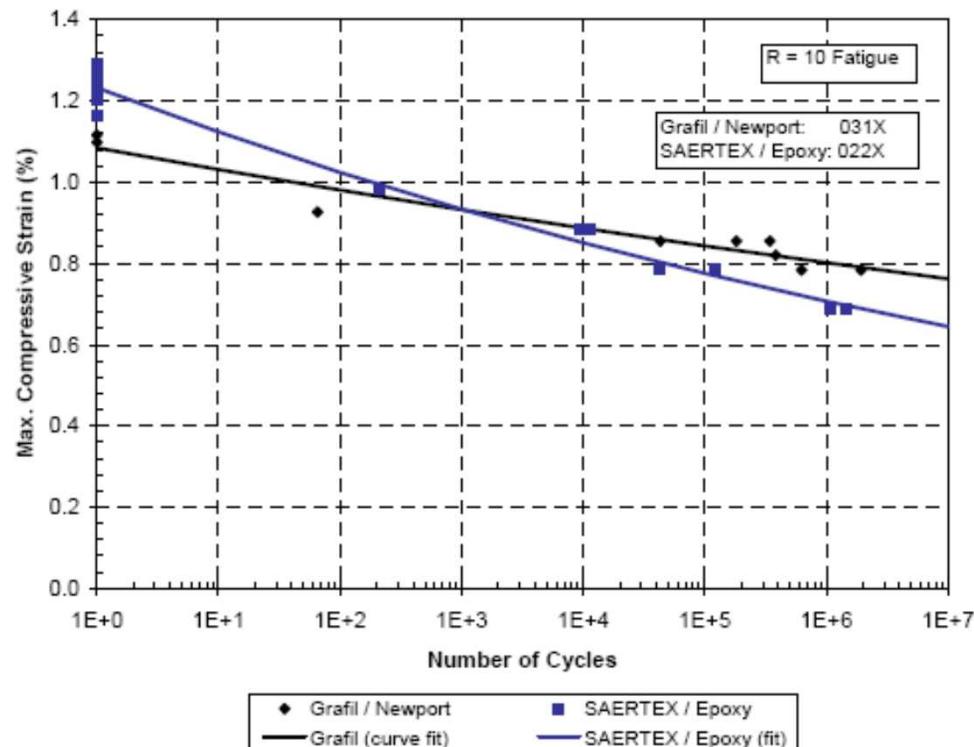
triax fabric

- Relatively inexpensive
- Infusible
- Dry fabric for cold
infusion techniques
- Maintains excellent
properties



Source: Sandia National Laboratories performed by and in
collaboration with GEC and MSU

SAERTEX Carbon Tri-ax Fatigue Performance

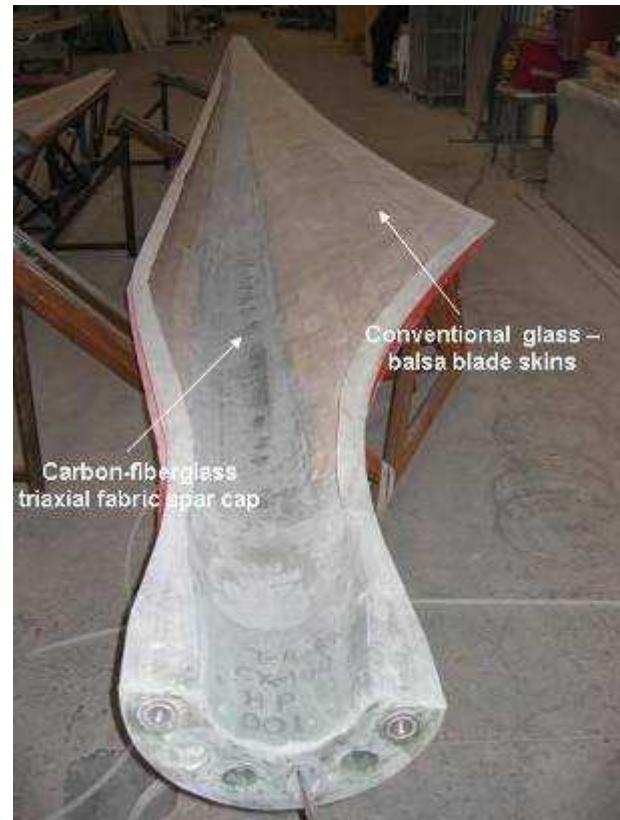


Source: Montana State University



CX-100

- CX-100 (Carbon Experimental 100 kW)
- Manufactured using existing 9 m molds
- Based on ERS-100 blade with non-scalloped root
- Glass-Epoxy blade with full length carbon spar cap

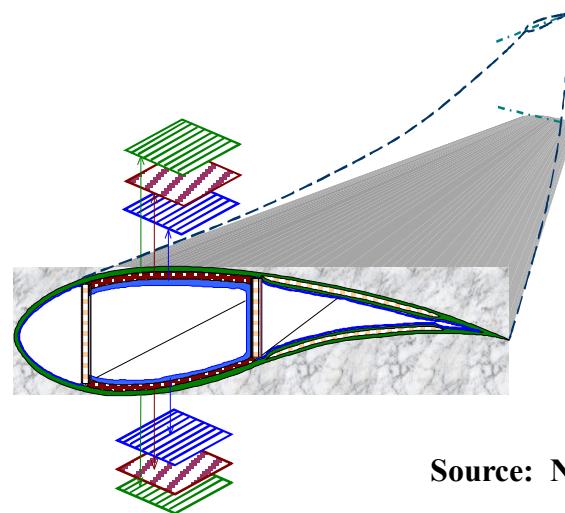


CX-100 Blade Skin



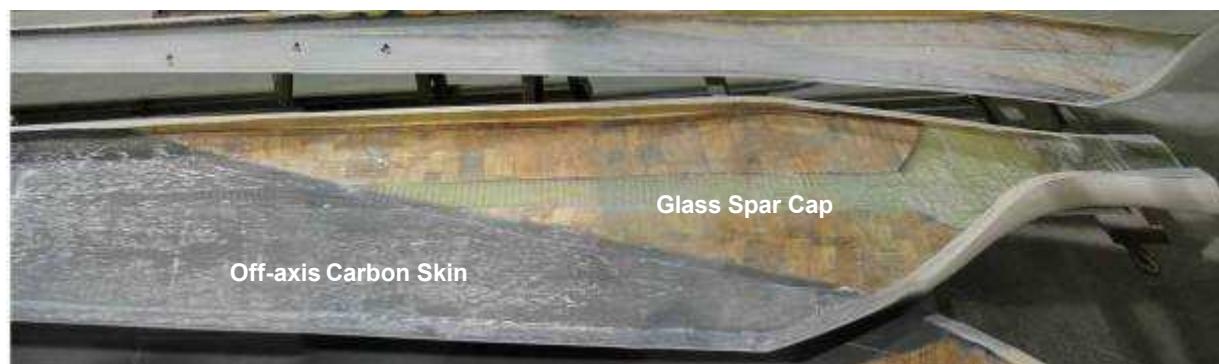
TX-100

- TX-100 (Twist-Bend Coupled
Experimental 100 kW)
- Identical geometry to CX-100
- Partial-length glass spar cap
- 20° off-axis carbon in outboard (~>3.5 m) skins to produce material-induced, passive aerodynamic load alleviation



Source: NREL

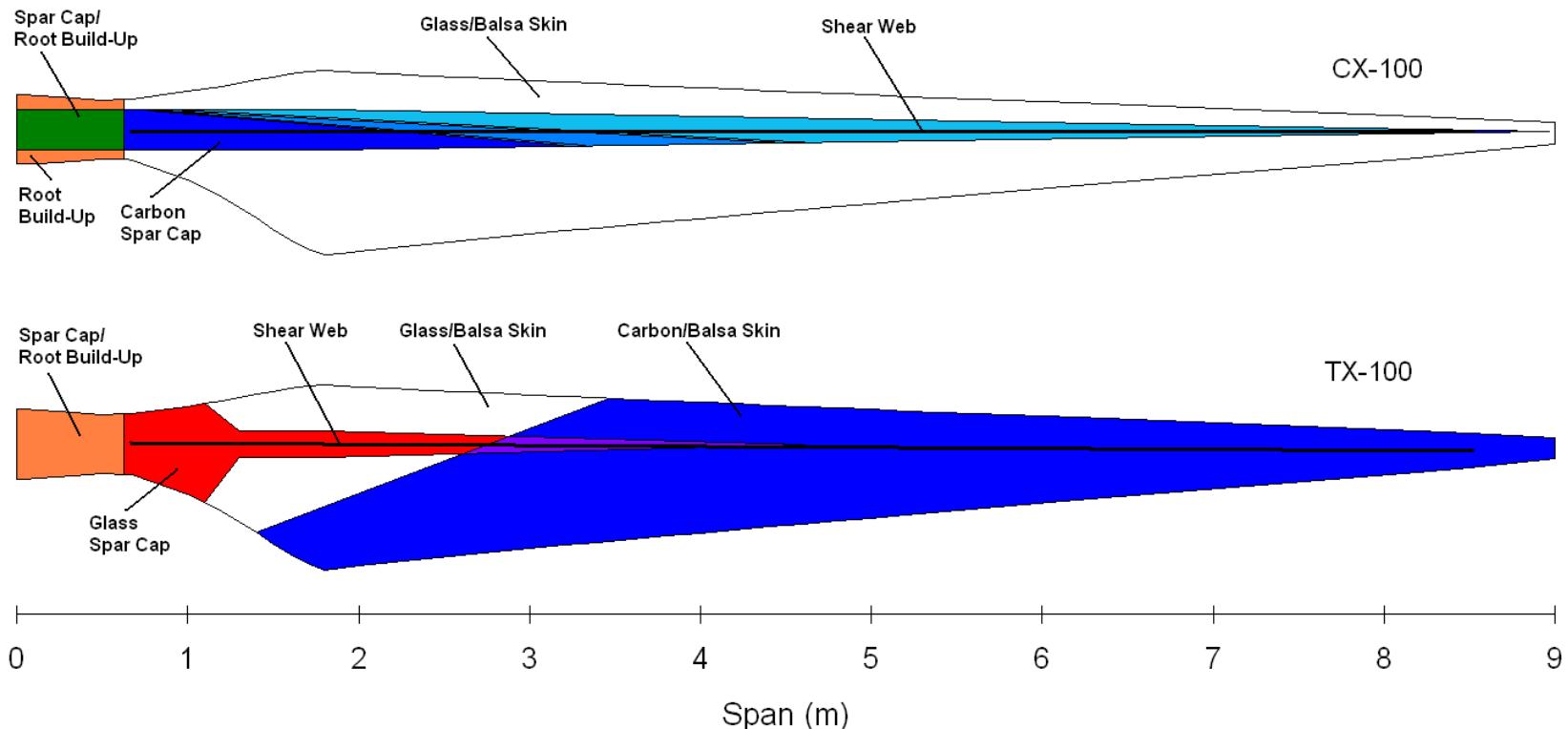
Material Induced Twist-Bend Coupling



TX-100 Blade Skin



9 m Blade designs



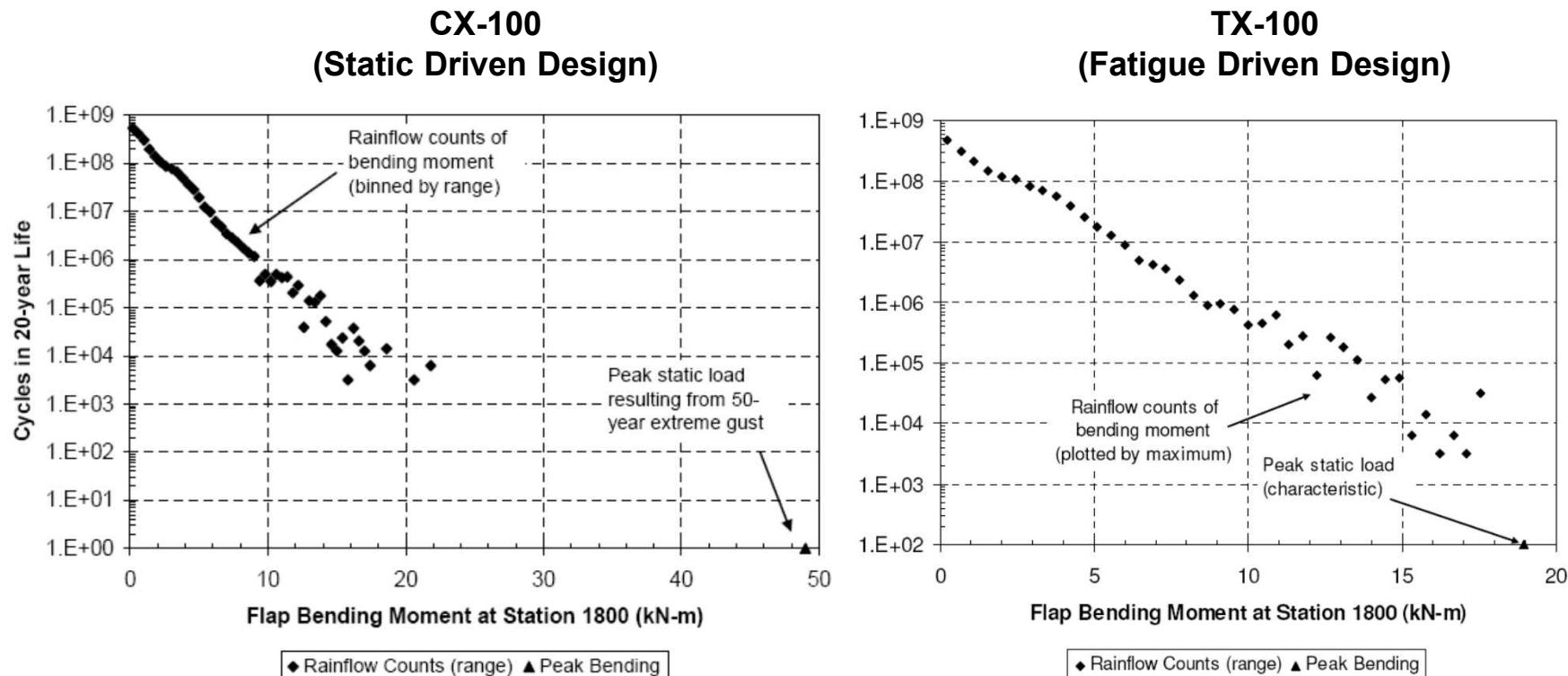
CX-100 (top) and TX-100 (bottom) Geometry and Major Laminate Regions



Fatigue Test Methodology

- **Test objective**
 - Demonstrate 20-year fatigue equivalent life
 - Complete test in 1-4 million cycles
- **Fatigue Equivalent Life Calculation Procedure**
 1. Perform system dynamics simulations
 2. Count fatigue cycles/second
 3. Extrapolate to 20-years
 4. Compute damage fraction using damage model along with material data, appropriate safety factors, and damage accumulation counting method

CX-100 and TX-100 Simulations





Test Setup: CX-100

- **Fatigue analysis focused on carbon spar cap**
- **Slope parameter of 12 used (GL standards: 10 for glass, 14 for carbon)**
- **Single-axis flapwise point loading**
 - Hydraulic cylinder used to apply oscillating load at single point
 - Robust, simple setup
 - Only allows for target load matching in limited area
- **1.25-12.5 kN applied at saddle for 1M cycles, then increased by 10% every 500k cycles**
- **20-year fatigue equivalent life demonstrated in 6k cycles**



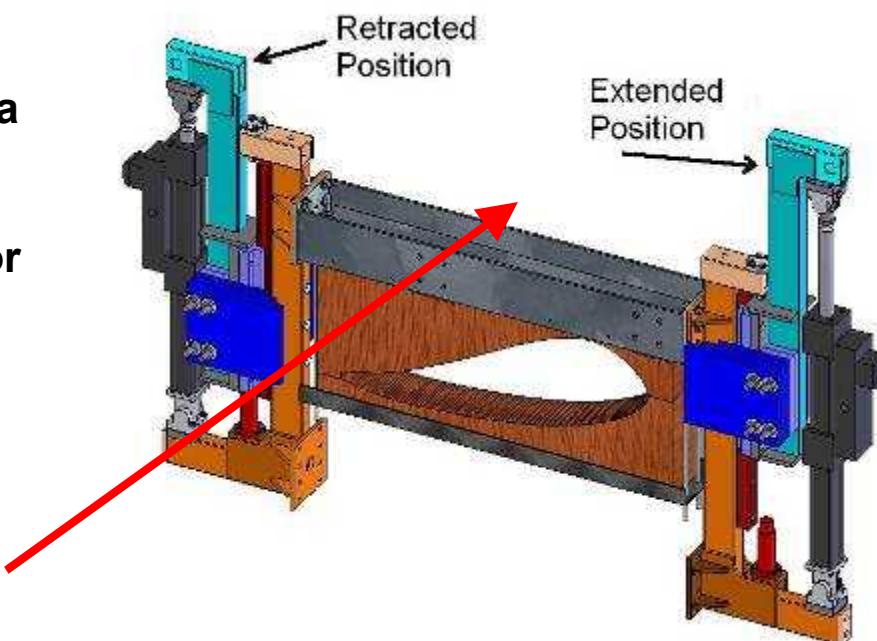
Test Setup: TX-100

- **Fatigue analysis focused on both glass and carbon areas**
- **Slope parameter of 10 used (for off-axis loading)**
- **Single-axis flapwise resonant loading**
 - Uses oscillating mass to excite natural frequencies of blade-mass system
 - Mean load adjusted by exciter and ballast masses
 - Amplitude adjusted by exciter displacement
 - Complicated setup required to produce correct shape and amplitude
 - Potentially allows for load matching for large portion of blade span
- **1M, 2M, and 4M cycle test loads calculated**
- **Test began with 4M load and then increased 10% beginning at 1M cycle count and repeating every 500k cycles**
- **Unable to increase at 2.5M cycles, load was held constant thereafter**
- **20-year fatigue equivalent life demonstrated at 2M cycles**



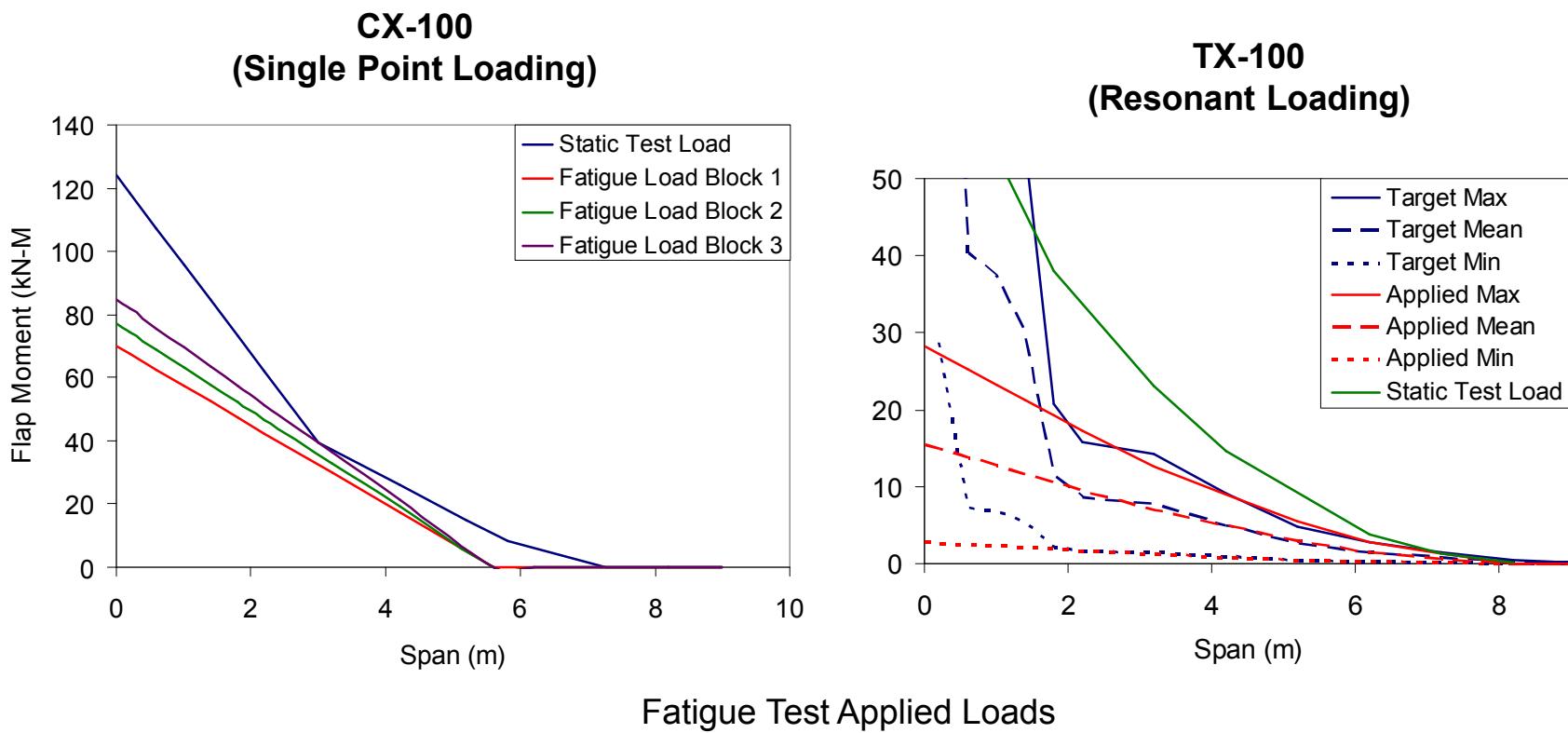
UREX

- Developed specifically for the unique aspects of testing bend-twist coupled blades
- Pair of hydraulic actuators mounted to the blade through a ballast saddle
- Rotational inertia minimized compared to mounting actuator and resonant mass above the blade
- Possible to apply torsional loading by adjusting actuator phases
- Horizontally mounted cylinder can be used to excite edge movement



UREX Schematic

Applied Loads





CX-100 Test Results



CX-100 Early in Fatigue Test



CX-100 Test Results



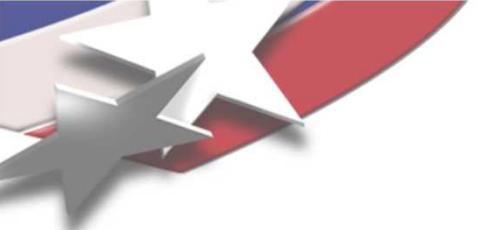
CX-100 Dimple (left) and Tip Movement (right) just before Failure



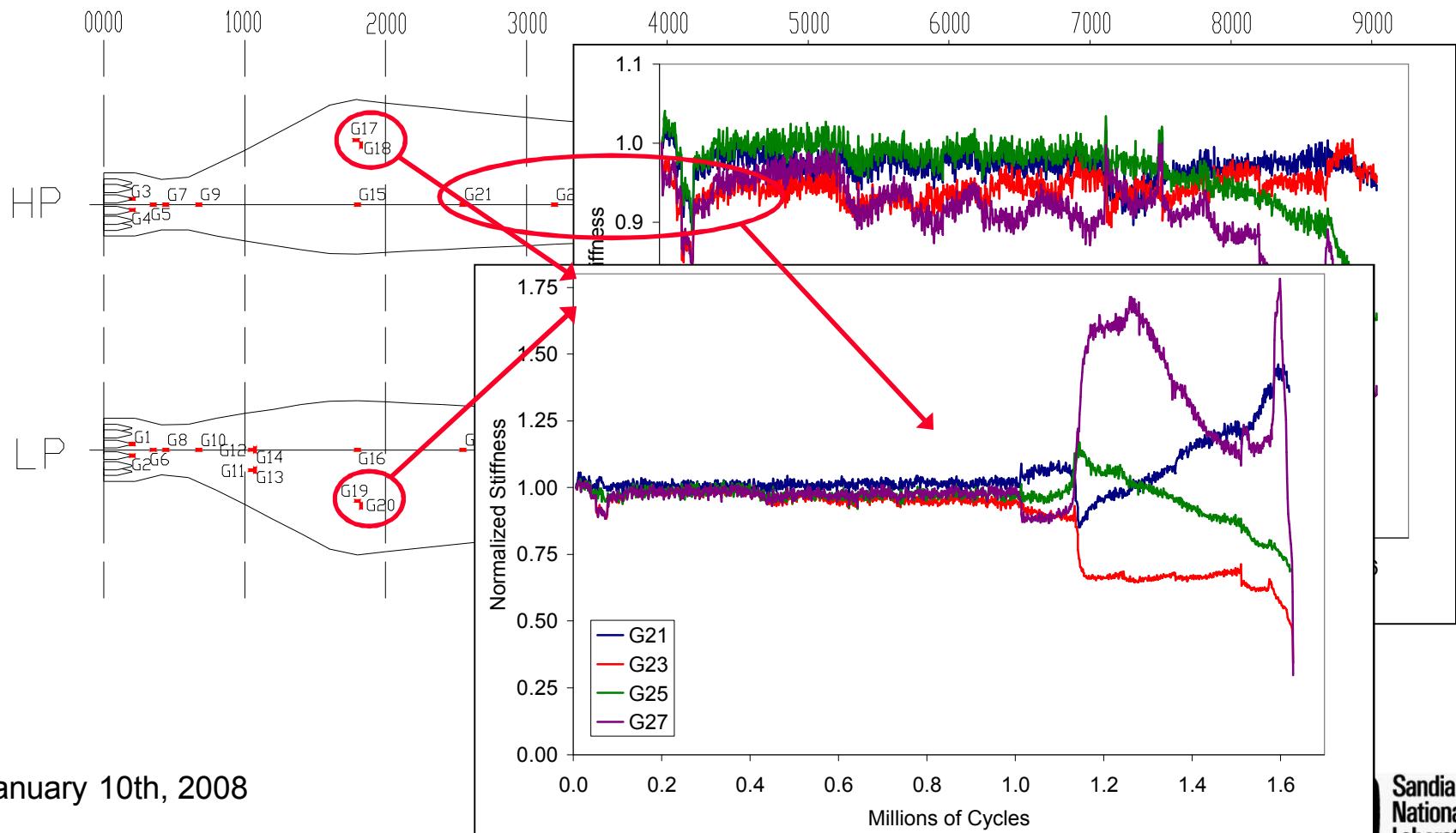
CX-100 Test Results



CX-100 Dimple (left) and Tip Movement (right) at Failure



CX-100 Test Results

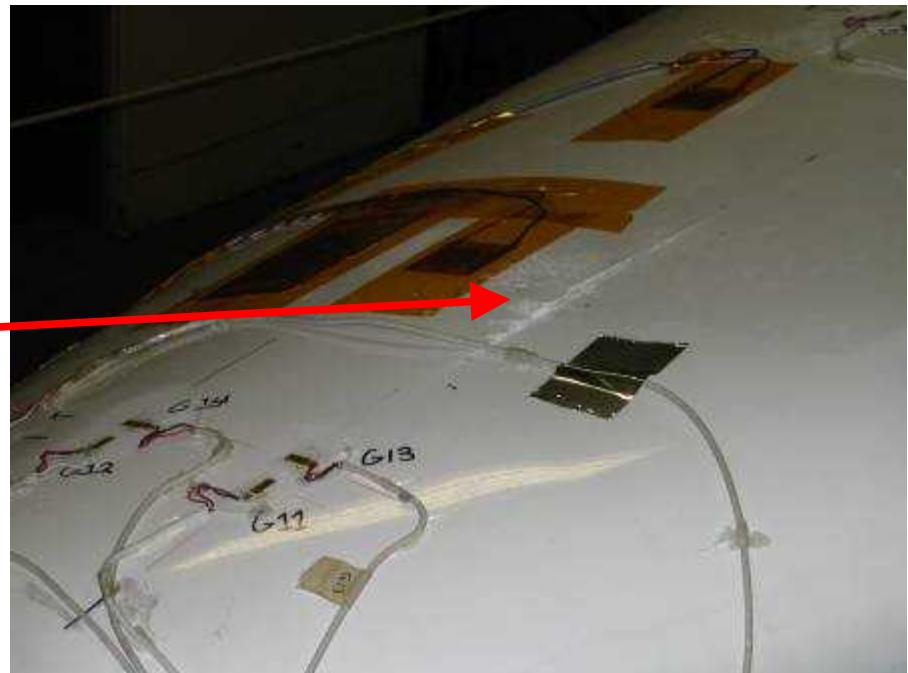


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CX-100 Failure Mechanism

- Dimple formed early during test around max chord
- Low pressure skin pushed outward aft of sparcap and inward forward of sparcap
- At 1.5M cycles, crack began to grow along sparcap/aft-panel intersection
- Crack resulted in greatly decreased stiffness in the area and cause severe edgewise movement



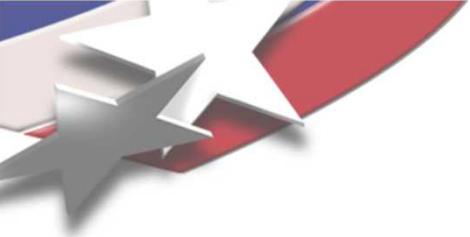
CX-100 Crack Growth



TX-100 Test Results



TX-100 Early in Fatigue Test



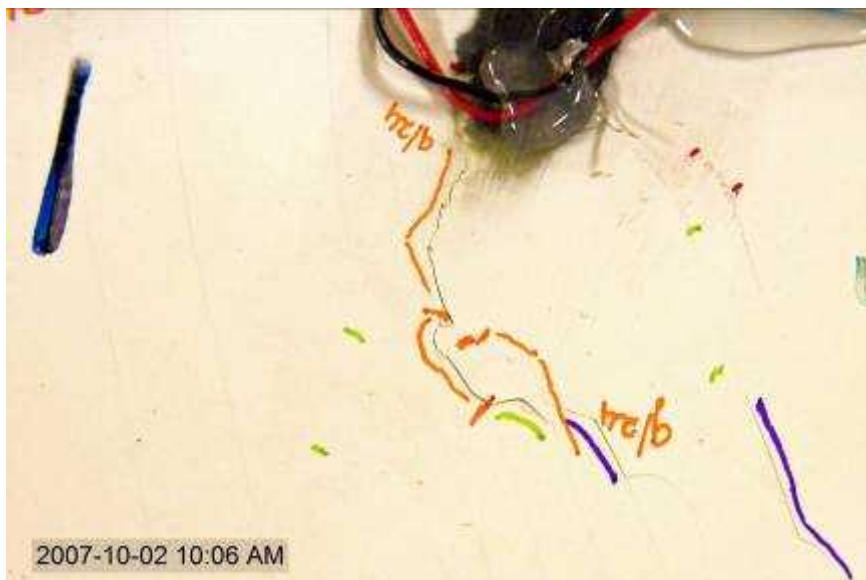
TX-100 Test Results



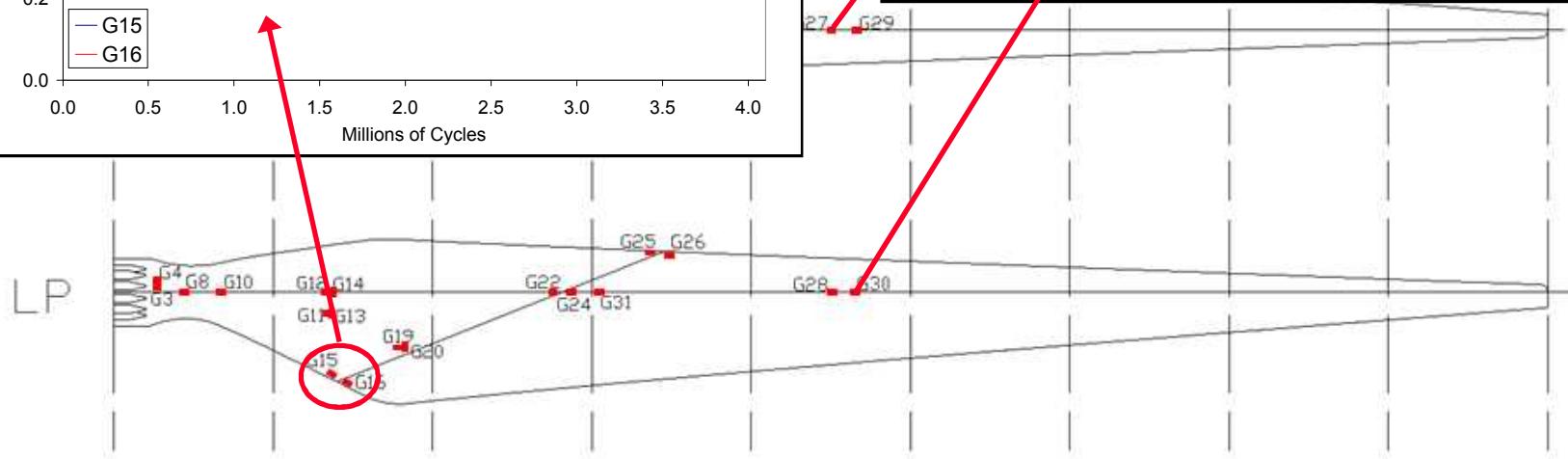
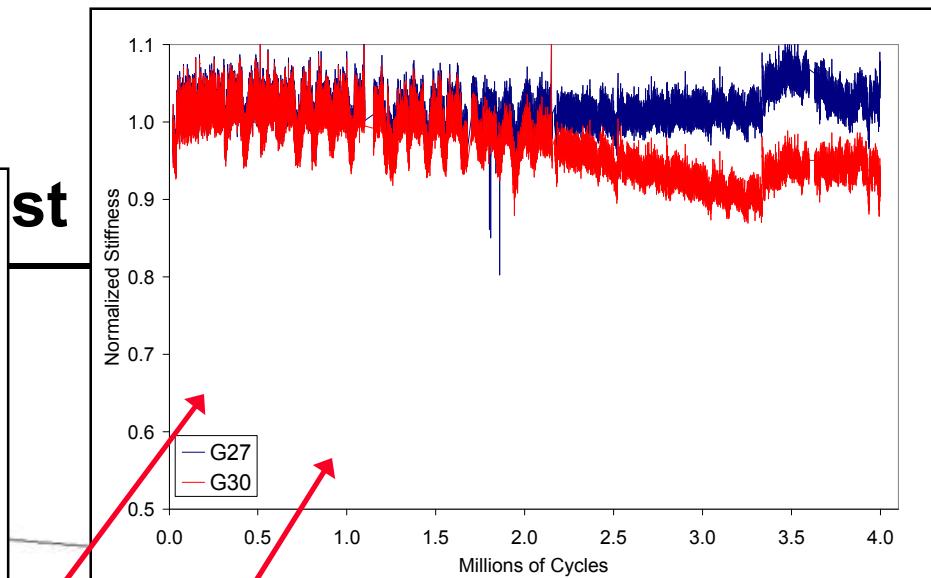
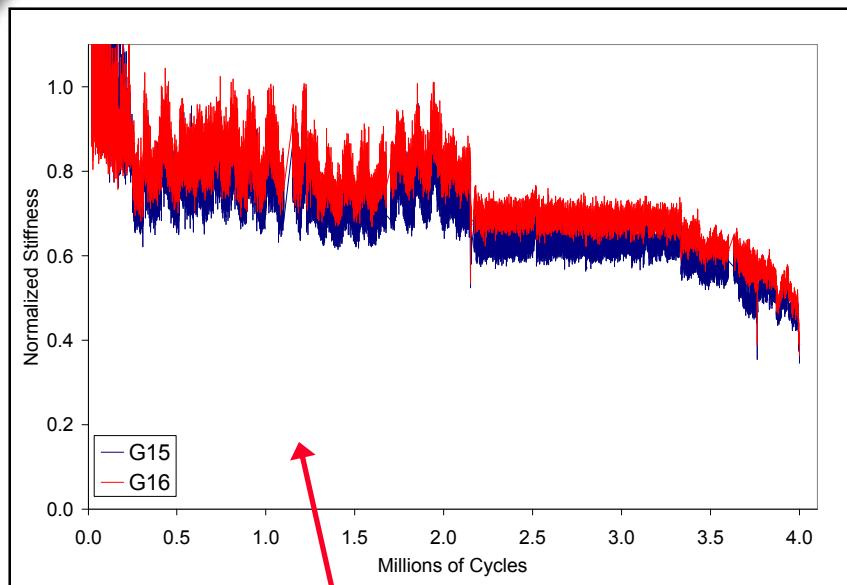
TX-100 Sparcap Tip Stress Contours



TX-100 Test Results



TX-100 Crack Growth Beginning (left) and Progression (right)



TX-100 Strain Gage Layout



TX-100 Failure Mechanism

- At 723k cycle count, crack began to grow just outboard of HP sparcap termination
- Cracks grew at 65° angle from blade axis until 2.4M cycles
- Crack then changed direction and grew along 20° direction corresponding to carbon fiber direction
- Growth of crack continued until 4M cycles when excessive torsional movement of the blade tip occurred



TX-100 HP Crack Growth



Conclusions

- CX-100 failed due to buckle formation near max-chord which caused a fracture between the sparcap and aft balsa panel leading to excessive edge movement
- TX-100 failed due to crack which grew from sparcap termination on HP surface along carbon fiber direction causing excessive tip rotation
- Infused carbon was effectively implemented in a CX-100 and TX-100 blade designs
- Both blades failed in carbon areas
- Blades failed due to damage in off-axis directions, showing the difficulty in using fiber-direction fatigue calculations



Acknowledgements

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