

Nanostructured Materials for Solid Lubrication

Plenary Talk

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U.S. DEPARTMENT OF
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- Tony Ohlhausen (ToF-SIMS)
- John Jungk (FEM)
- Rand Garfield (Tribology Support)
- Michael Rye, Gary Bryant (FIB)
- Bonnie McKenzie (SEM)

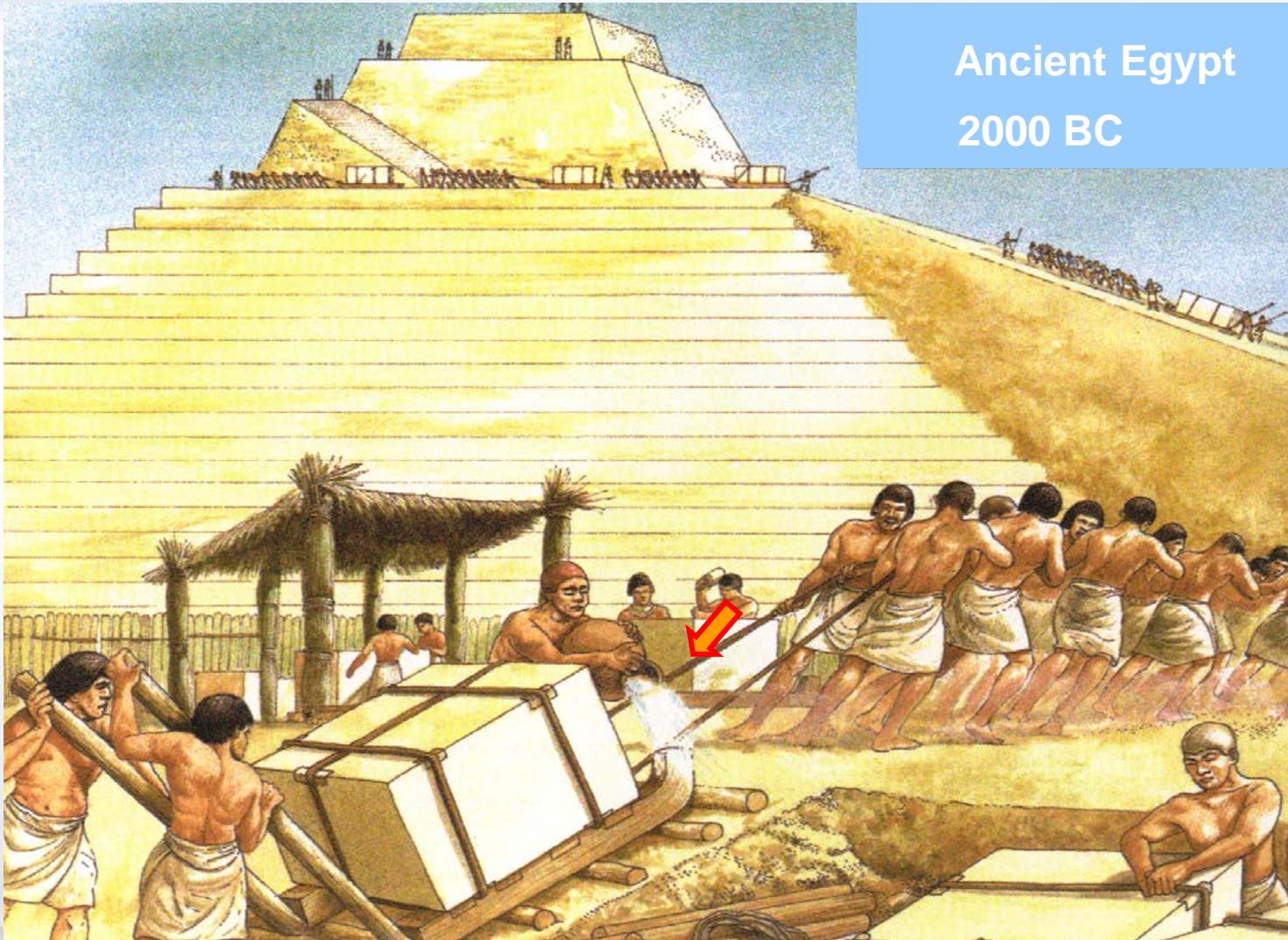
- Tom Scharf (U of North Texas)
- Jeff Zabinski (ARL)
- Jon-Erik Mogonye(UNT/Sandia)
- Greg Sawyer (U of Florida)
- Brandon Krick (Lehigh U)
- Rob Carpick (U Penn)
- Chandra Venkatraman (Entegris)
- Cindy Broadbeck (SulzerMetco)



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Lubrication in Pyramid Building

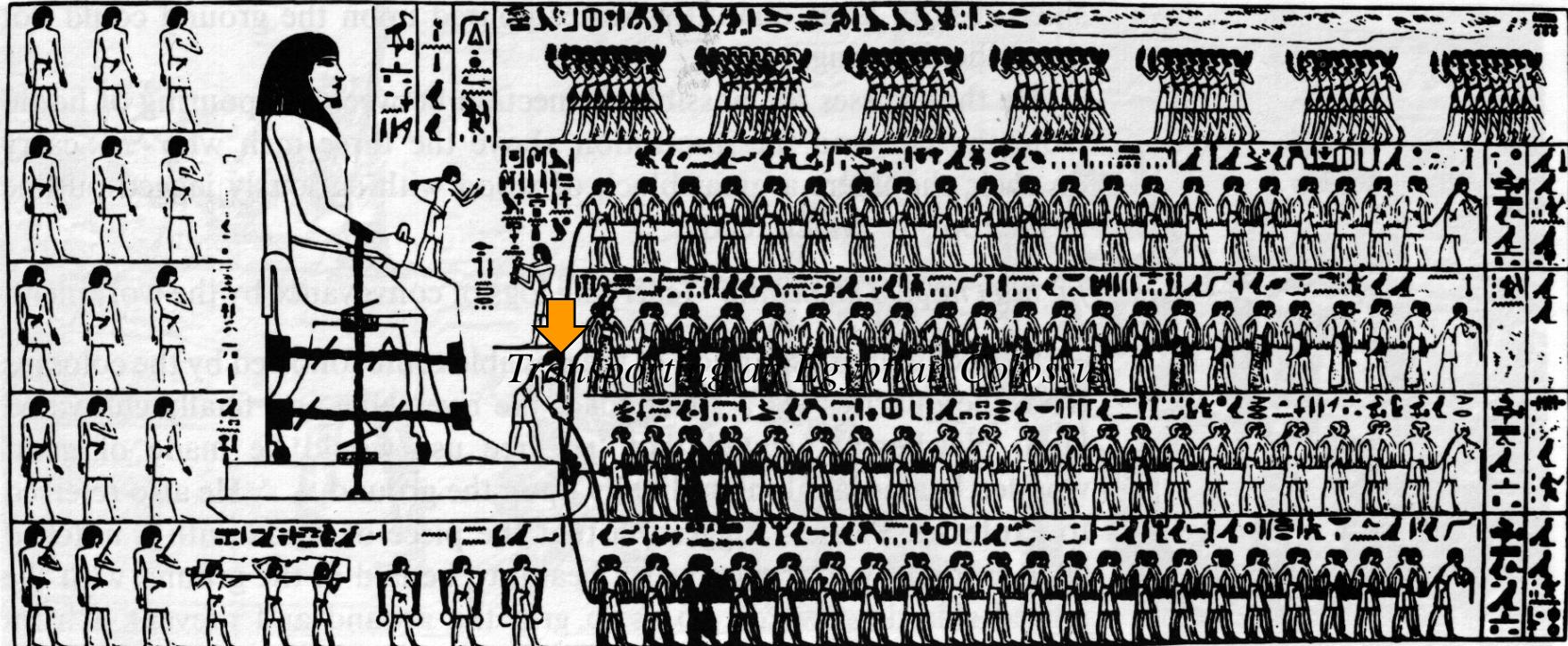


Ancient Egypt
2000 BC



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Transporting an Egyptian Colossus (Lubrication)

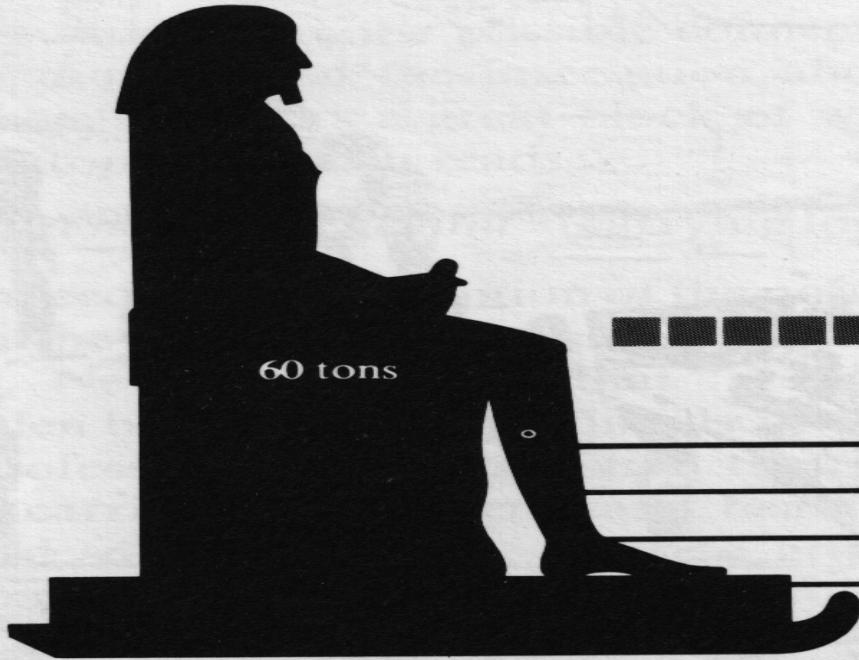


Painting from a Grotto at El-Bersheh

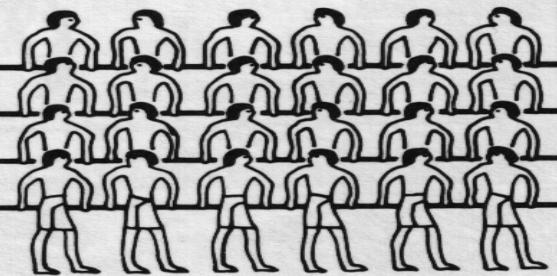
Source: Duncan Dowson, History of Tribology , Elsevier, 1979



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180 lbs



$$\mu = \frac{F}{W} = \frac{172 \times 180}{60 \times 2240} = 0.23$$

D. Dowson, "History of Tribology" Elsevier 1979

Bowden and Tabor, "Friction and Lubrication of Solids-Part I" Oxford 1950

- The coefficient of friction, μ , for hard wood sliding on wet/moist wood is 0.2
- The coefficient of friction of wood-on-wood in dry condition is 0.45-0.50

In ancient Egypt, about 100, 000 men were employed each year to transport massive objects.

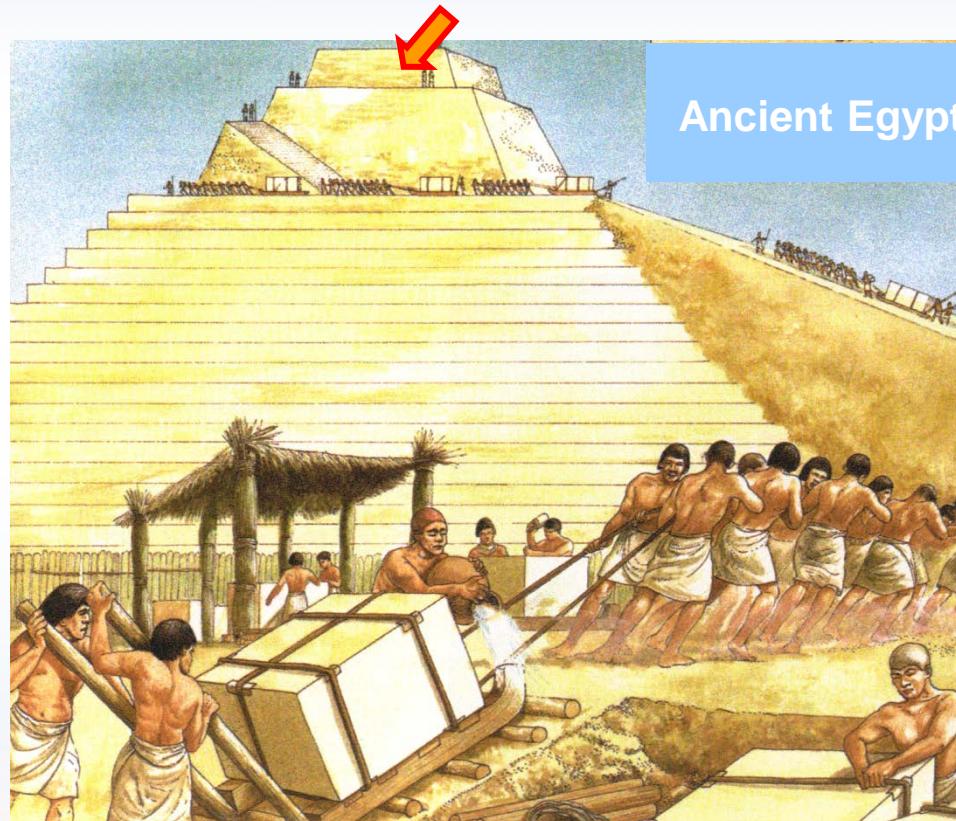
Without a lubricant, 200, 000 for a state in 2000 BC can be large expense even if the monarchs fed them only one meal and paid no wages!!!



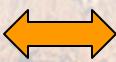
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Limited space as the apex is reached



Tribology

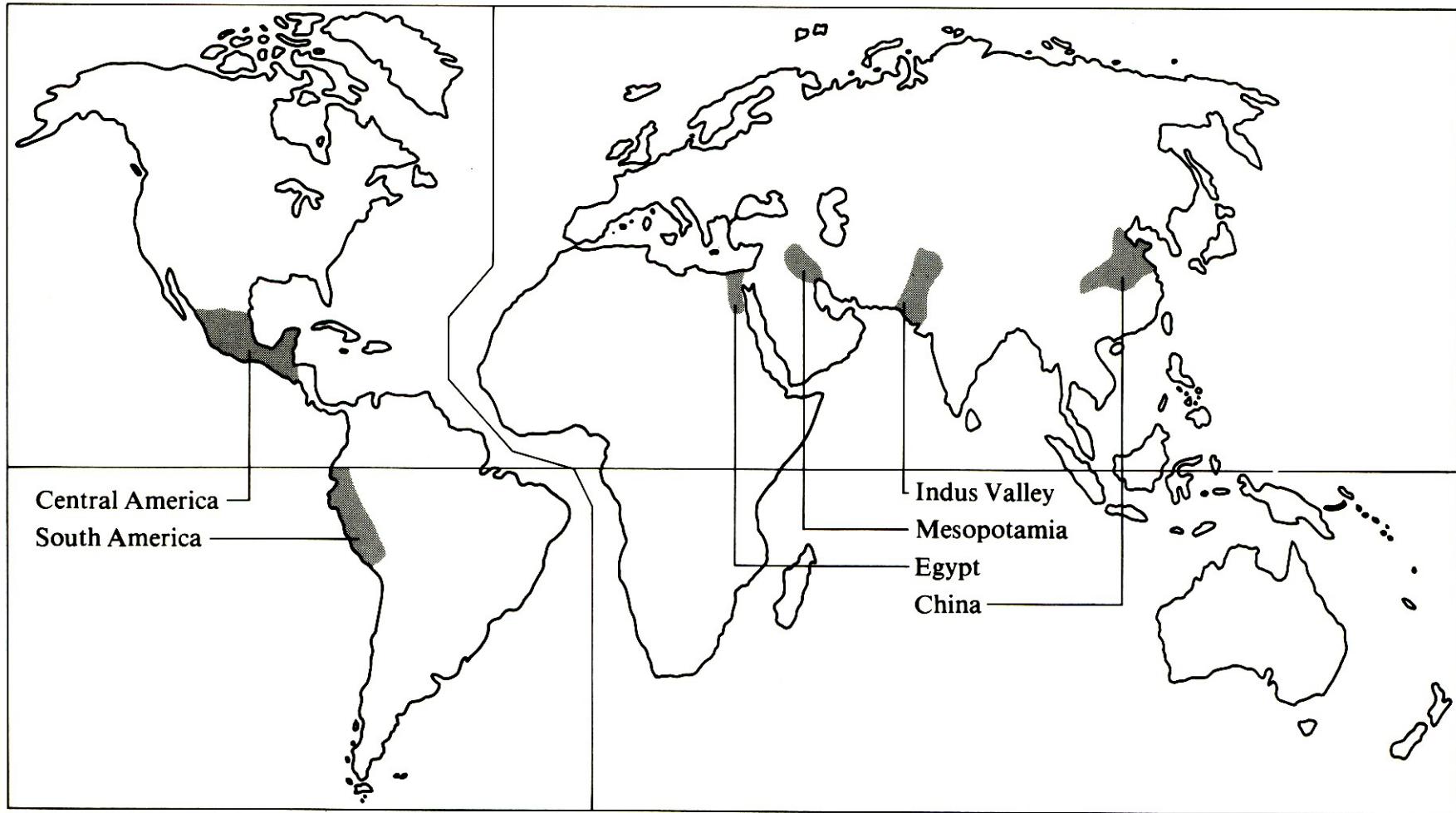
Cost and Energy Savings  Performance and Reliability



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Early Civilizations Across the World



Archeological Studies confirm tribological innovations during early civilizations

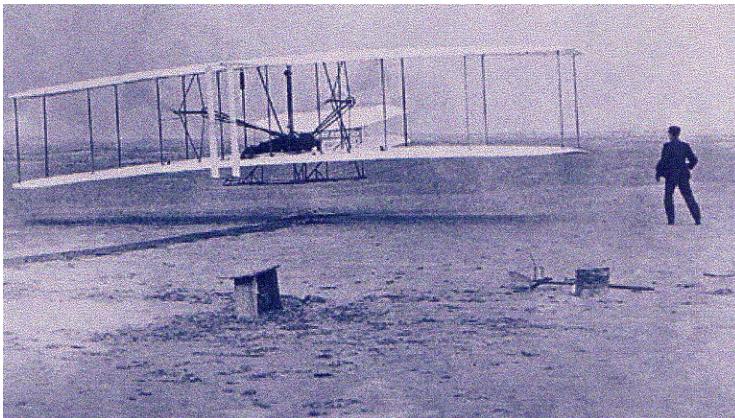


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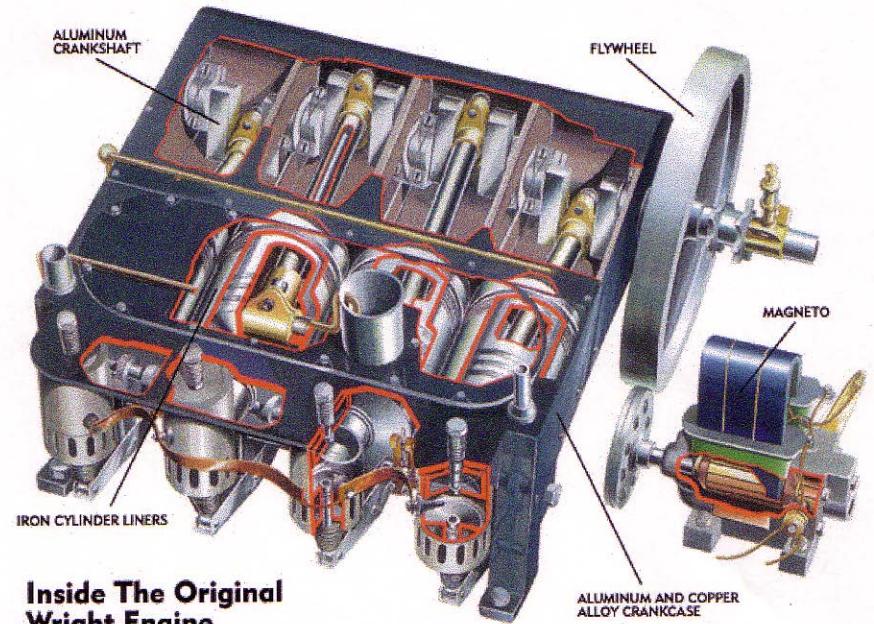
The Saga of Wright Brothers' Engine

1903



Mr. Charles Taylor (Mechanic)
Considered replacing CI with Al-Cu

- The Brothers needed an Engine with 8 HP weighing <180 lbs



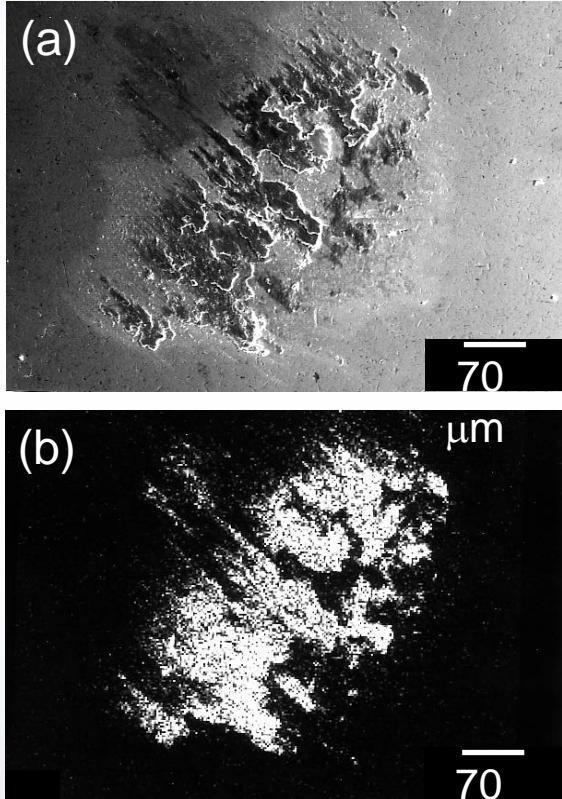
Inside The Original Wright Engine

- 16 HP-12HP 178 lbs
- The Brothers used the extra weight allowance to strengthen the wings and frame
- But Al has a tendency for seizure and galling in the absence of complete fluid film lubrication



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Mr. Taylor had the foresight to avoid Al in sliding contacts



Smearing of Al on a steel ball during a ball-on-disk wear test. SEM showing adhesive transfer of Al on steel ball (a) with corresponding X-ray map (b).

S. V. Prasad and K. R. Mecklenburg, *Lubrication Engineering*, 50 (1994) 511-518.

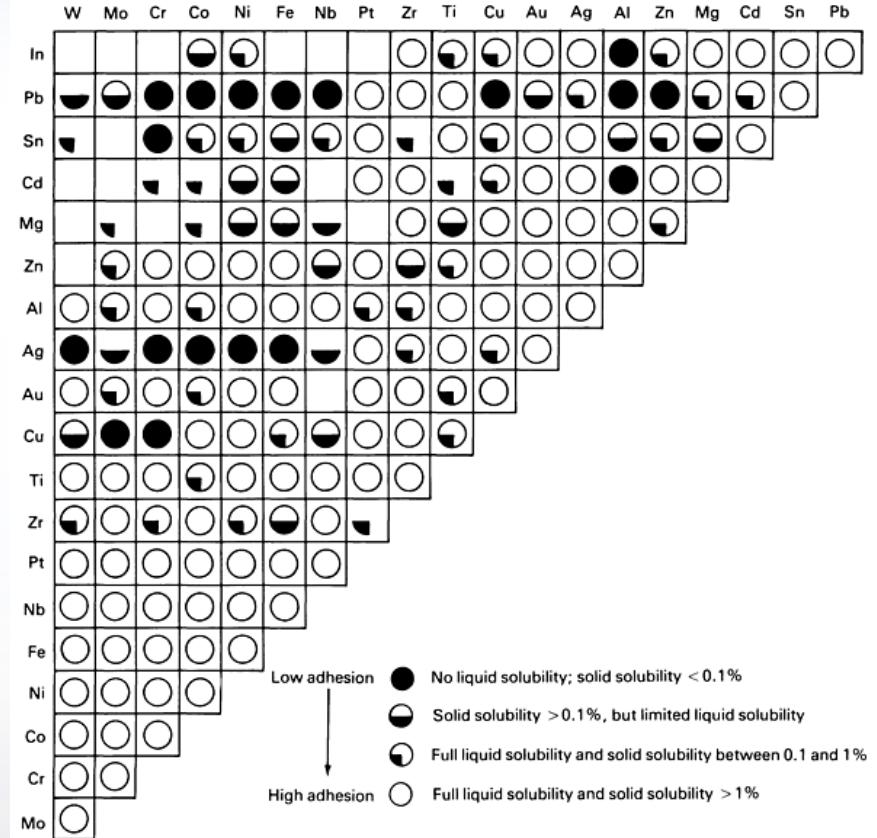


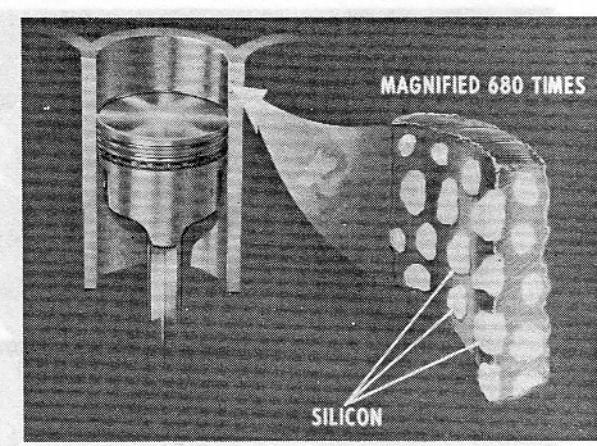
Chart indicates the degree of expected adhesion (and thus friction and wear) between the various metal combinations derived from binary equilibrium diagrams.

E. Rabinowicz (1971) ASLE Trans 14:198

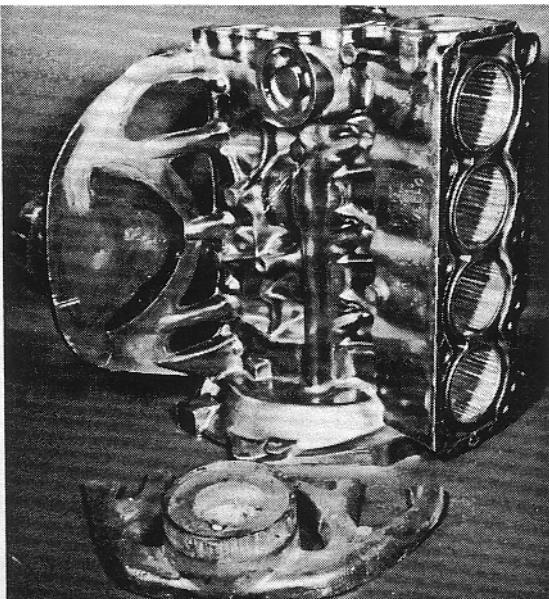


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The (Short) Legacy of the Vega Engine



Silicon surface cylinder bores



Vega engine block as removed from die

BASIC SPECIFICATIONS

VEGA 2300—140 cu in. Overhead Cam 4-cyl Engine

GENERAL

Type	In-Line OHC 4-cyl (L-4)
Gross horsepower	
Standard engine	90 at 4600-4800
Optional engine	110 at 4800
Gross torque	
Standard engine	136 at 2400
Optional engine	138 at 3200
Compression ratio	8.00:1
Bore and stroke	3.501 X 3.625
Firing order	1-3-4-2
Engine installation angle	3 deg 50 min
Fuel	Regular leaded and nonleaded 91 Octane

Carburetor

Standard engine	One-barrel, Monojet
Optional engine	Two-barrel, downdraft

CYLINDER BLOCK

Material	Die-cast high-silicon aluminum alloy
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Bore spacing (C/L to C/L)	4.00
Number of bulkheads	Five

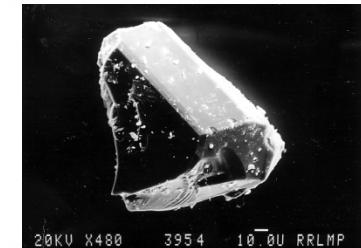
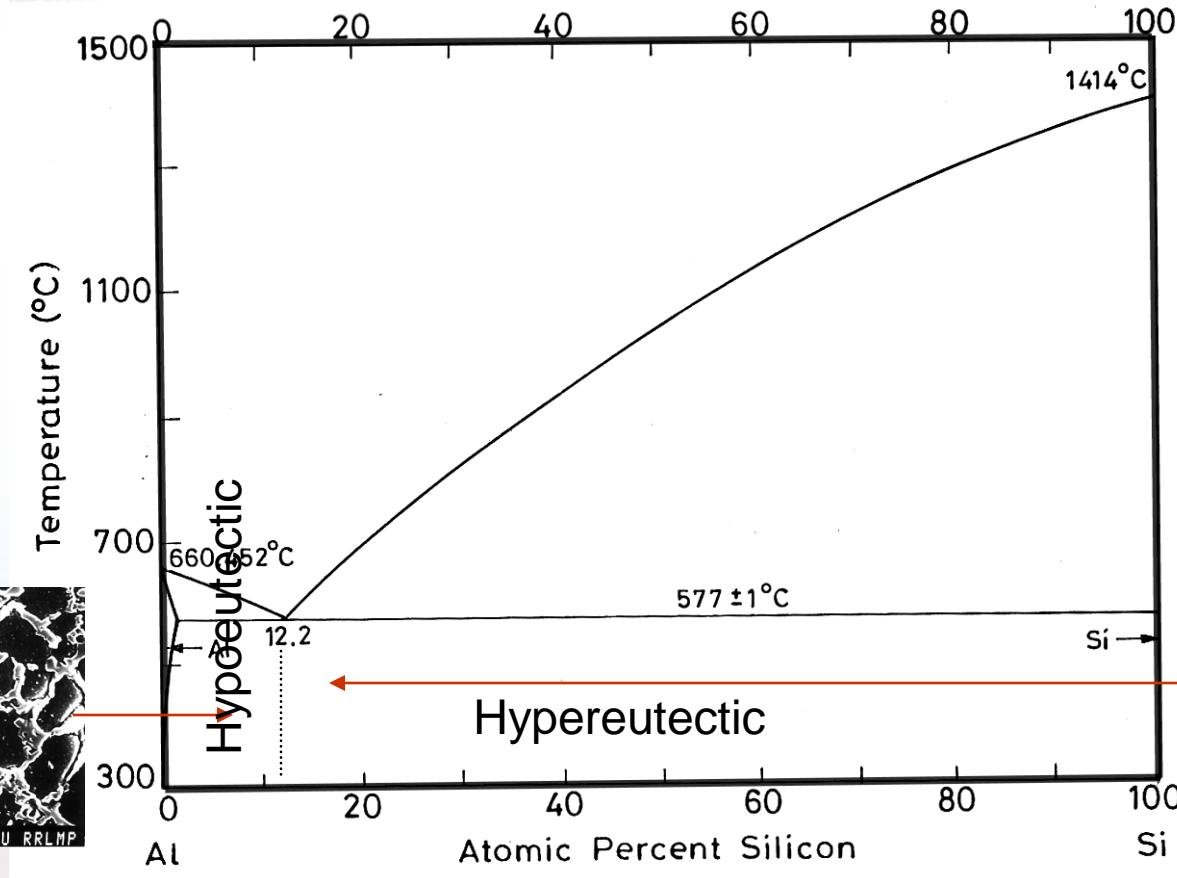
The Vega 2300 Engine, SAE 710147



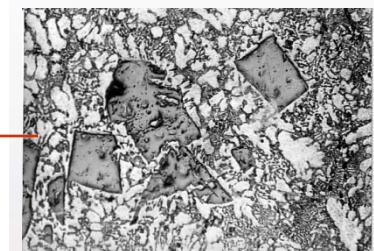
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Both the eutectic and the primary Si have undesirable morphology



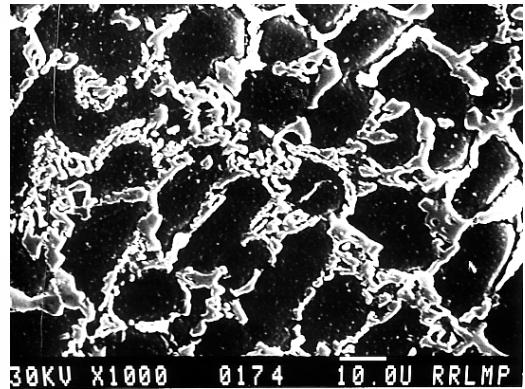
Silicon Debris
From Wear Test



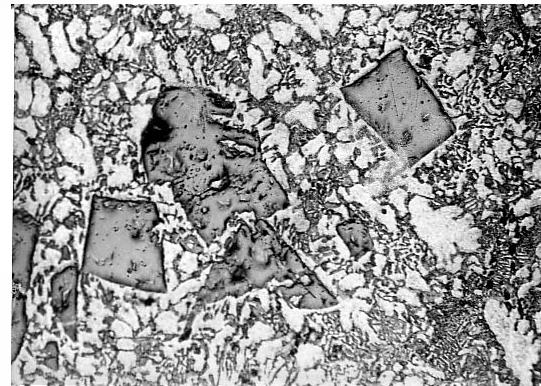
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Microstructural Modification of Al-Si Alloys for improved wear resistance

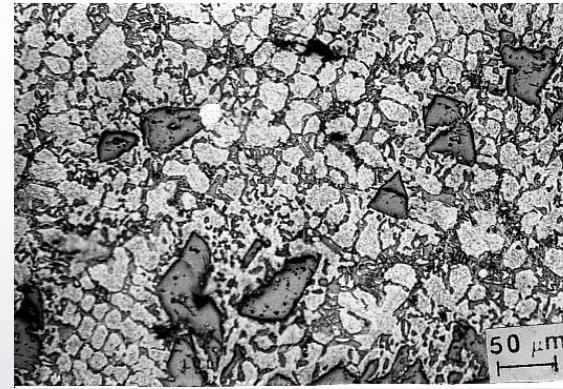
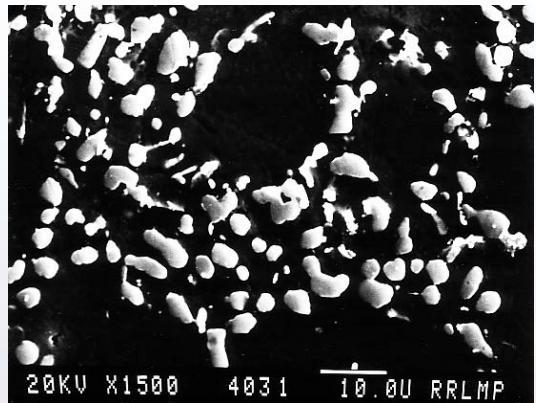
As-Cast



Hypereutectic



Modified
Refined

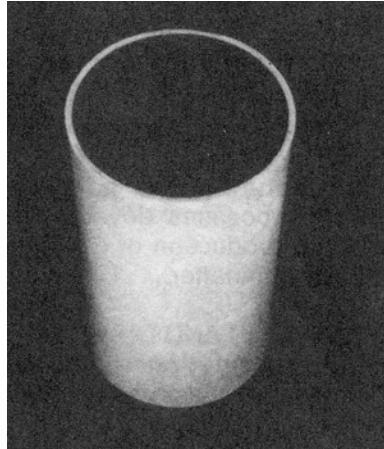


However, thermal mismanagement (arising from poor thermal conductivity of Si) was an issue that wasn't factored during the initial engine development, which essentially killed the engine.



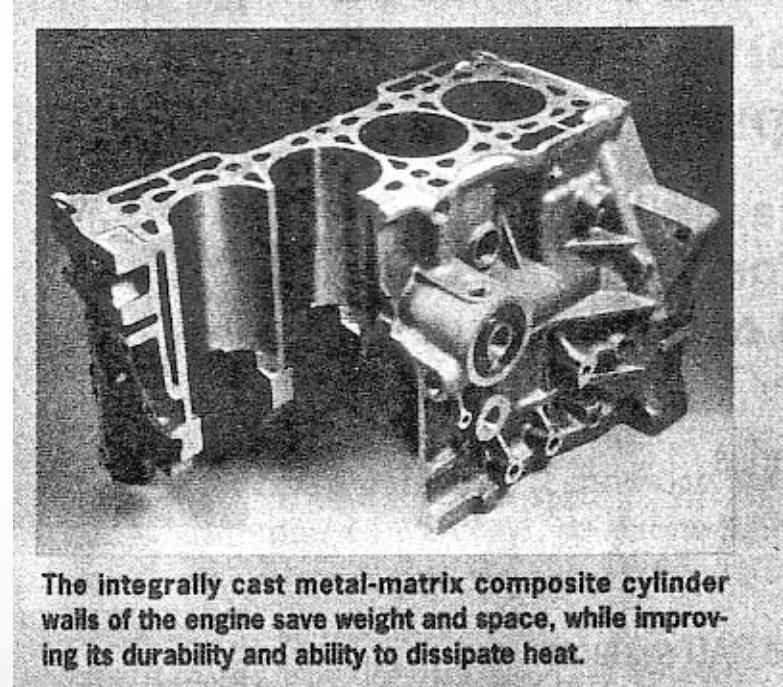


Integrally Cast MMC Cylinder: Honda Corporation (1980's)



Preform

A porous hybrid material made out of
Short alumina and Carbon fibers

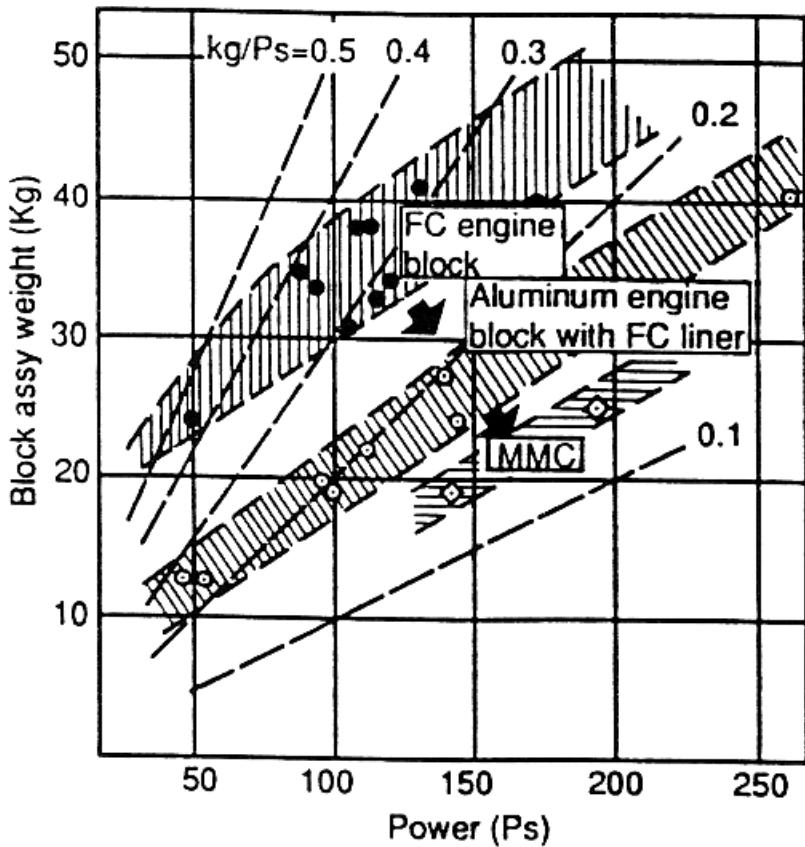


The integrally cast metal-matrix composite cylinder walls of the engine save weight and space, while improving its durability and ability to dissipate heat.

- **Ceramic “preform” production**
- **Pressure casting process**
- **Honing**
- Hybrid preforms: Carbon for thermal conduction, Aluminosilicate for strength
- Honing to minimize direct contact between Al and the piston ring

First introduced in Honda Prelude

Relationship between Power and Engine Block Weight



The new engine block features higher performance, further compactness and weight reduction compared to cast-iron engine blocks and those made out of Al alloy with cast-iron liners

M. Ebisawa et. al, "The Production Process for MMC Engine Block", SAE 910835



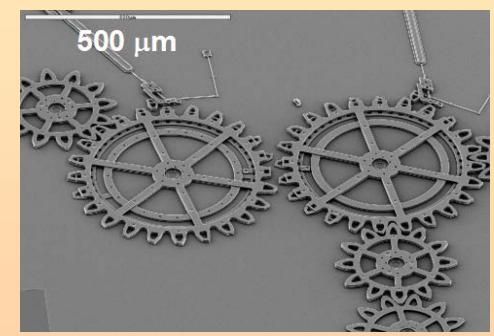
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Outline

- Examples from Ancient and Modern History
- Solid Lubricants
 - Environmental Effects
 - Synthesis of Nanocomposites (MoS_2 -Au, DLC- SiO_2)
 - Subsurface Deformation (FIB-EBSD-TEM)
 - Load Bearing and Barrier Layers

Harsh Environments



Energy and Communications



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Solid Lubricants

- *The major of lubrication needs still met by fluids and greases*
- *When the operating conditions are beyond the liquid realm (e.g., high temperature or vacuum), or situations where liquids cannot be introduced, attention turns to solids*

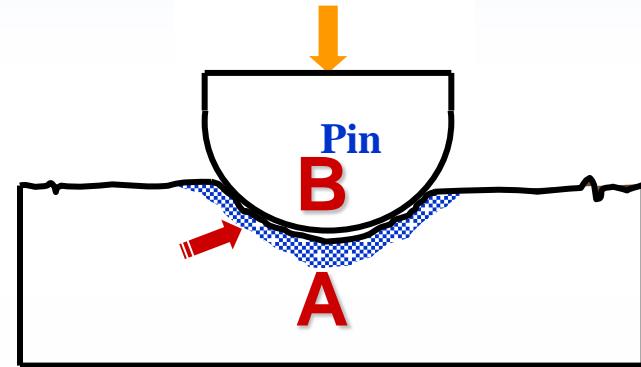
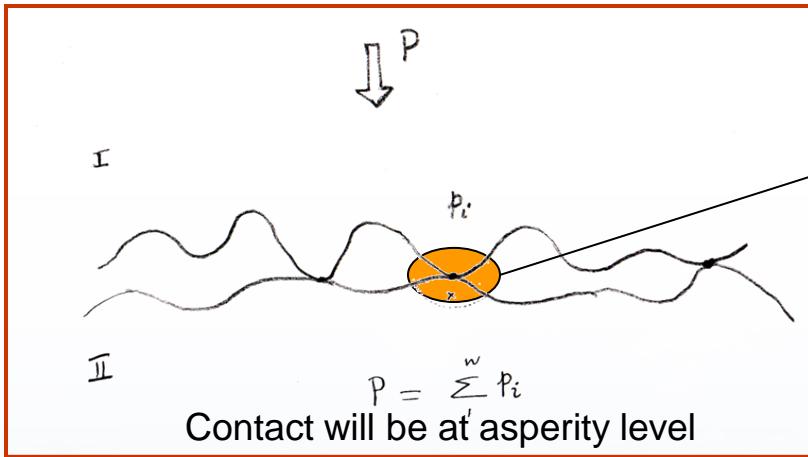
1. Carbon-based materials (graphite, DLCs, nanocrystalline diamond)
2. Transition metal dichalcogenides (MoS_2 , WS_2)
3. Polymers (e.g., PTFE)
4. Soft Metals (Ag, Sn, Pb, In , Au)



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Frictional Contact Typically Results in Plastic Deformation Below Wear Surfaces

Engineering surfaces are not flat



- Sliding contact results:
 - Plastic deformation
 - Diffusion

Barrier layers may be necessary to prevent the plastic deformation and diffusion (substrate species into coating)



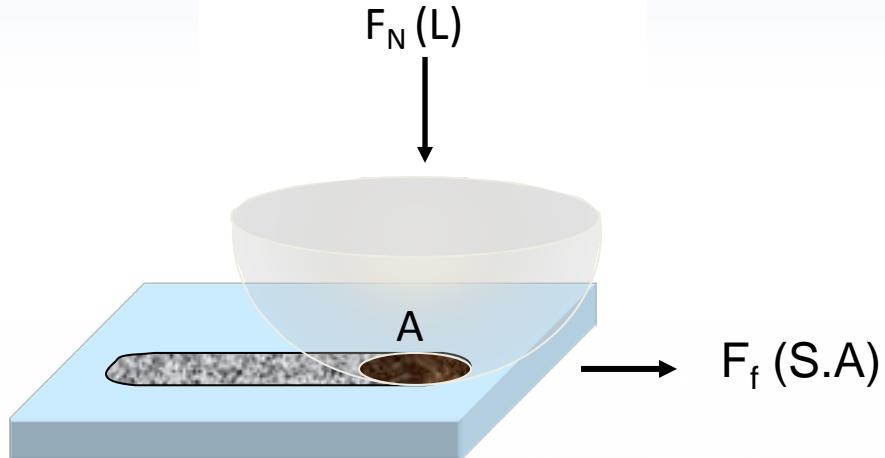
J. F. Archard, *J. Appl. Phys.* **24** (1953) 981
R. Holm, *Electrical Contacts* (Springer, 1946)

Rigney, D.A., Hirth, J.P. *Wear* **53** (1979) 345



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Many Solid Lubricant Coatings Exhibit Load Dependence on Friction



$$P = F_N / A$$

$$F_f = S A$$

$$S = S_o + \alpha P$$

Elastic Contact (Sphere-on-Flat)

$$\mu = S_o \pi (3R/4E)^{2/3} L^{-1/3} + \alpha$$

$$\mu = F_f/F_N = S.A/P.A$$

- S_o is the interfacial shear strength (a 'velocity accommodation parameter'), a property of the interface.
- α is a fit constant (the pressure-dependence of 'S')

Friction is NOT independent of Load

F. P. Bowden and D. Tabor, "The Friction and Lubrication of Solids", Oxford Science Publications, 1986

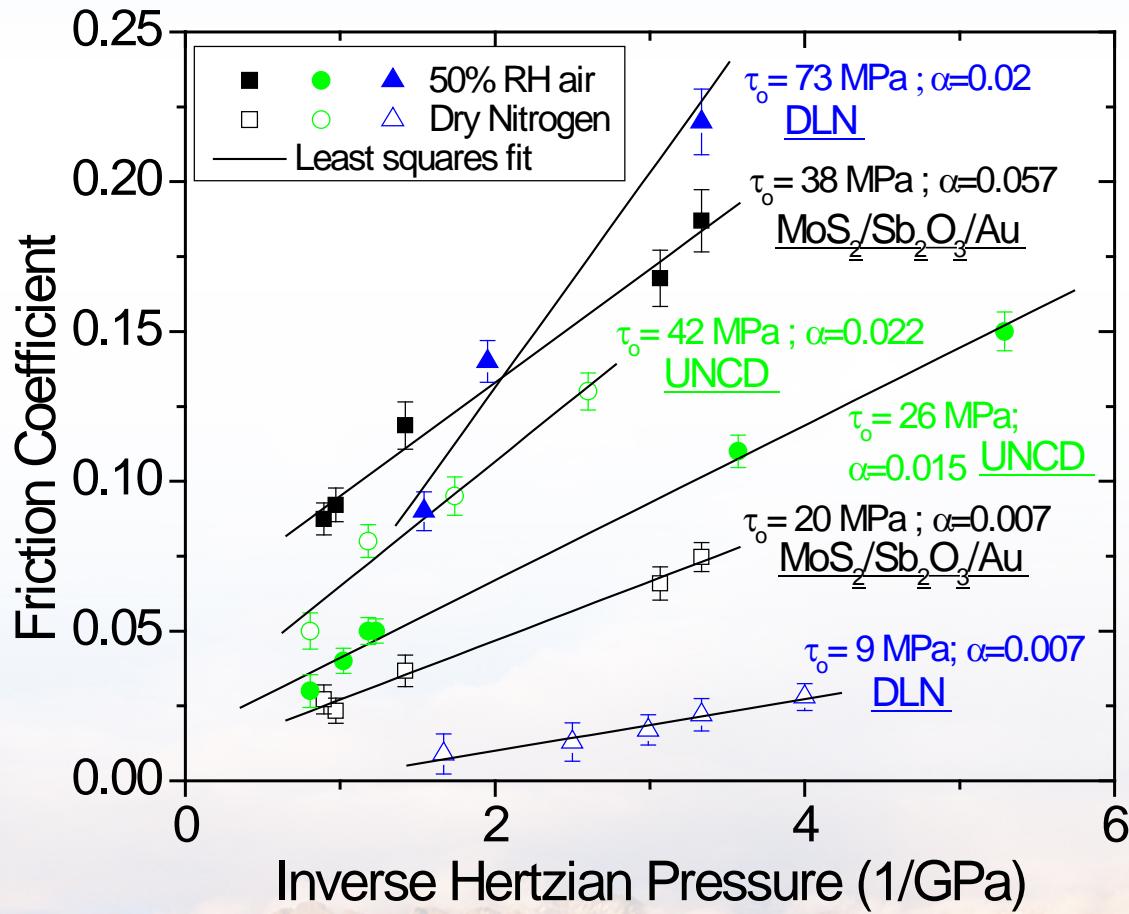
I.L. Singer, et al. *Applied Physics Letters* 57, 995 (1990).

B.J. Briscoe and D.C.B. Evans, *Proc. R. Soc. Lond. A* 380, 398 (1982).



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Inverse Hertzian Behavior— Non-Amontonian



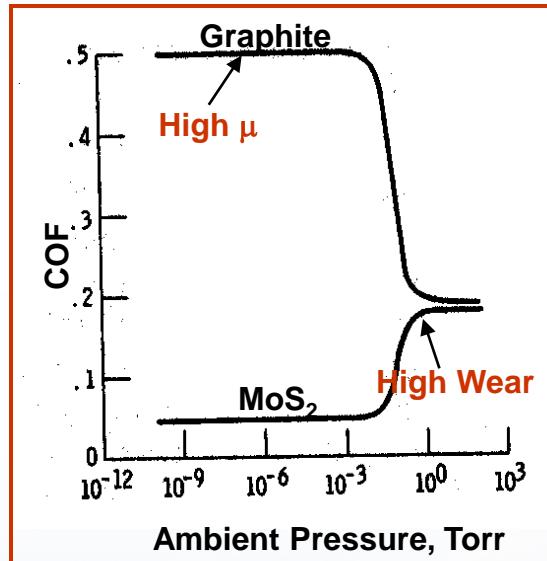
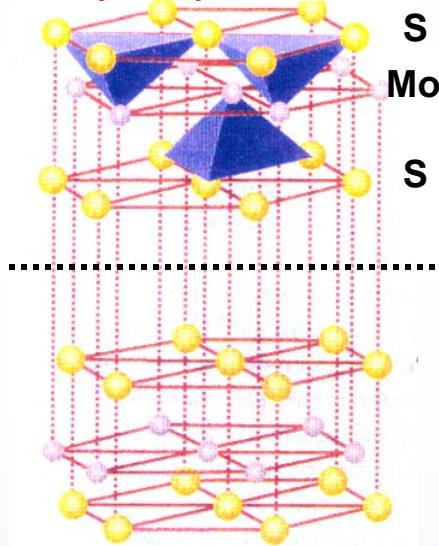
T. W. Scharf and S. V. Prasad, Solid Lubricants: A Review, *J. Maters. Science* (2013) 48:511-531



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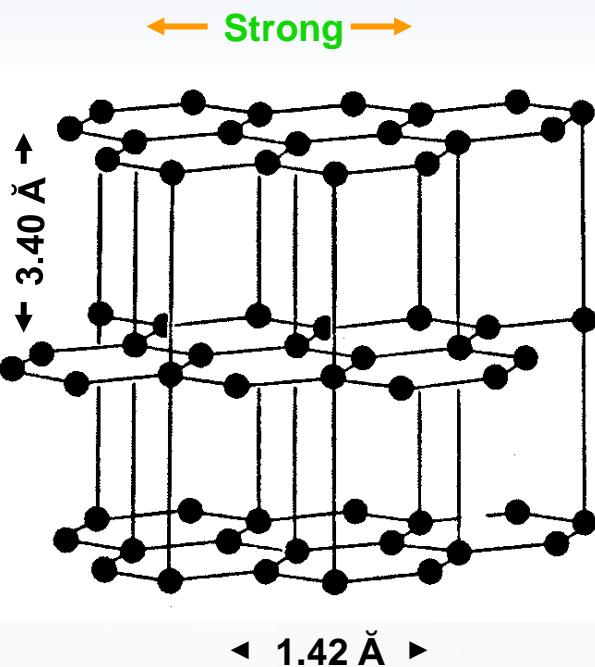
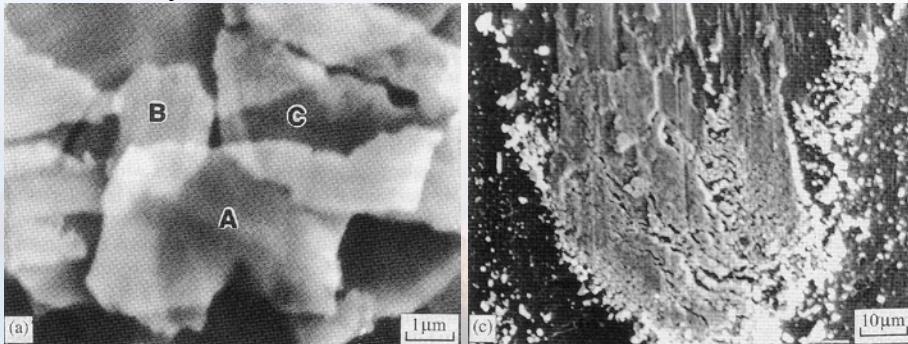
Classic Examples of Environmental Effects on Transfer Films & Friction

MoS₂: Extremely low COF (0.01-0.05) and long wear life, **but only in dry environments**.



Mo/W Disulfide

They form thin transfer films on the counterpart

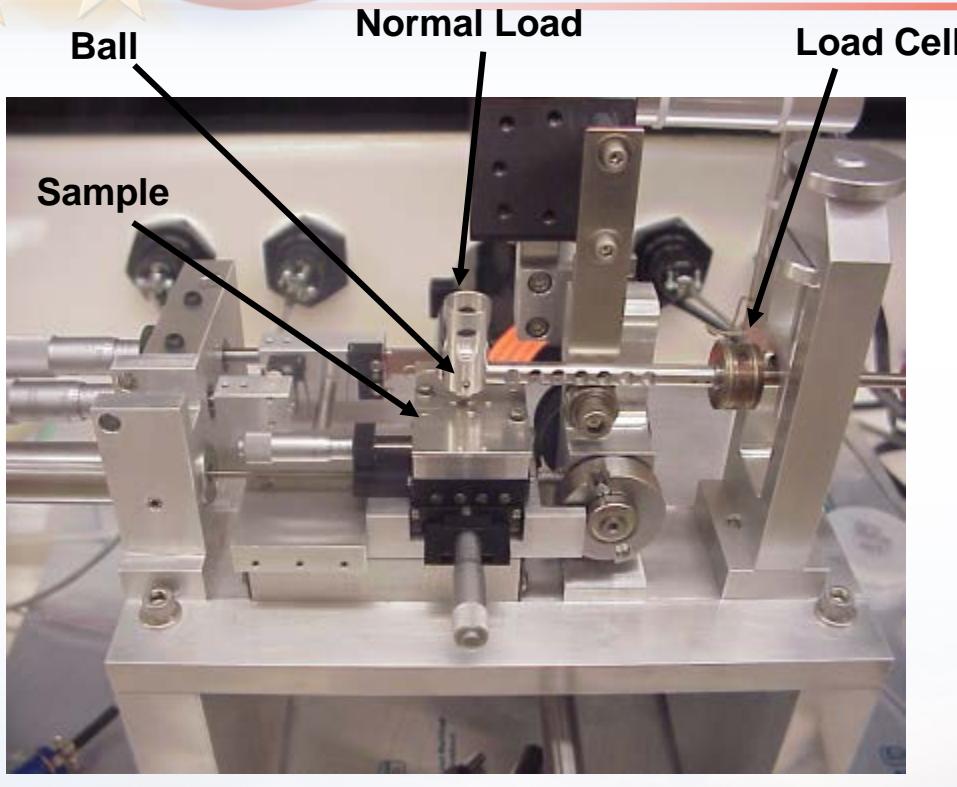


Graphite

- Graphite needs moisture or adsorbed gases in the environment (>100 ppm) (they either act as intercalants, or passivate the dangling covalent bonds) to lubricate.
- In vacuum, graphite exhibits high friction and wear—a phenomenon known as “dusting”, first observed in the late 1930’s on graphite brushes in aircrafts that exhibited accelerated wear at high altitudes.



Friction and Wear Measurements



Linear Wear Tester
(Ball-on-Flat configuration)



- Contact Pressures: 0.44 GPa to 1.7 GPa
- Environment: Dry Nitrogen; Air (50% RH)



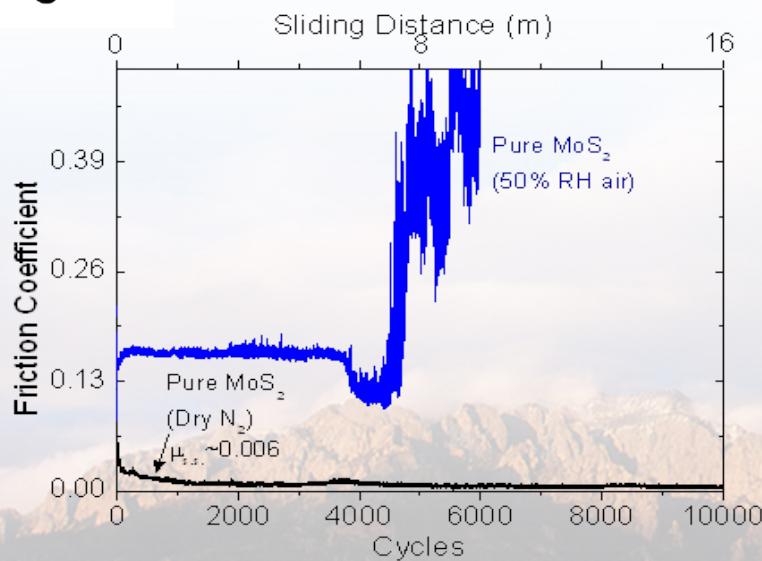
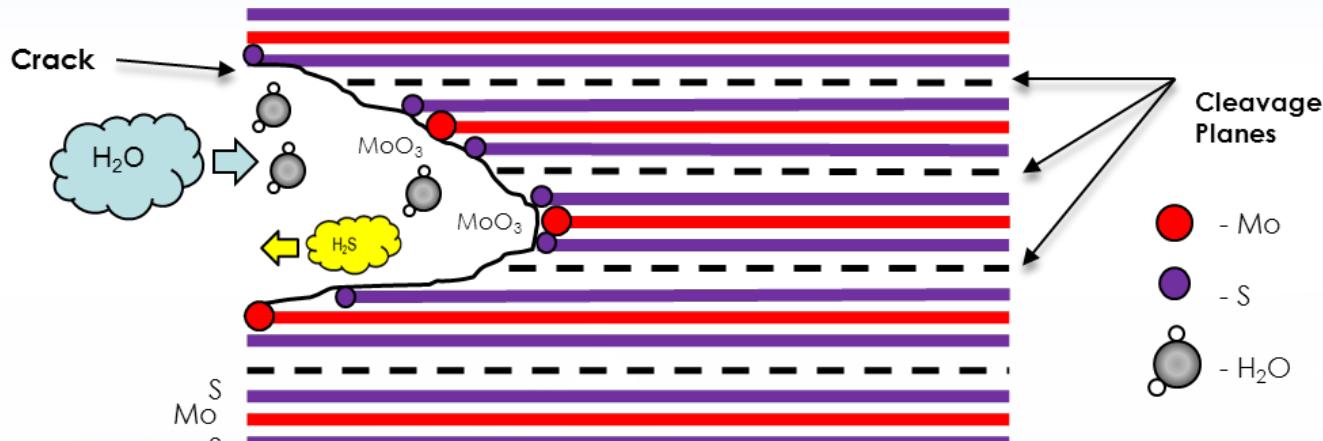
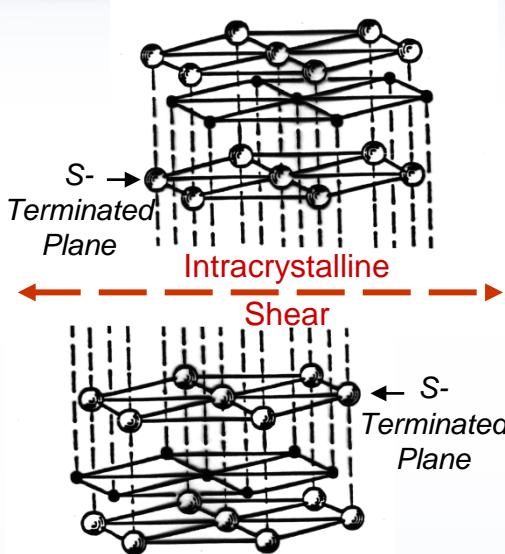
Oxygen Analyzer
Dew Point Measurement

Environmental Control



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Effect of humidity and water vapor on the friction behavior of MoS_2

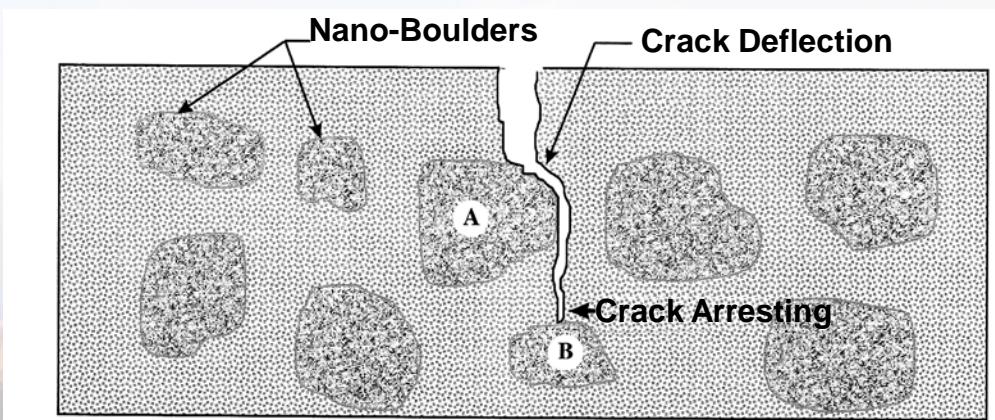
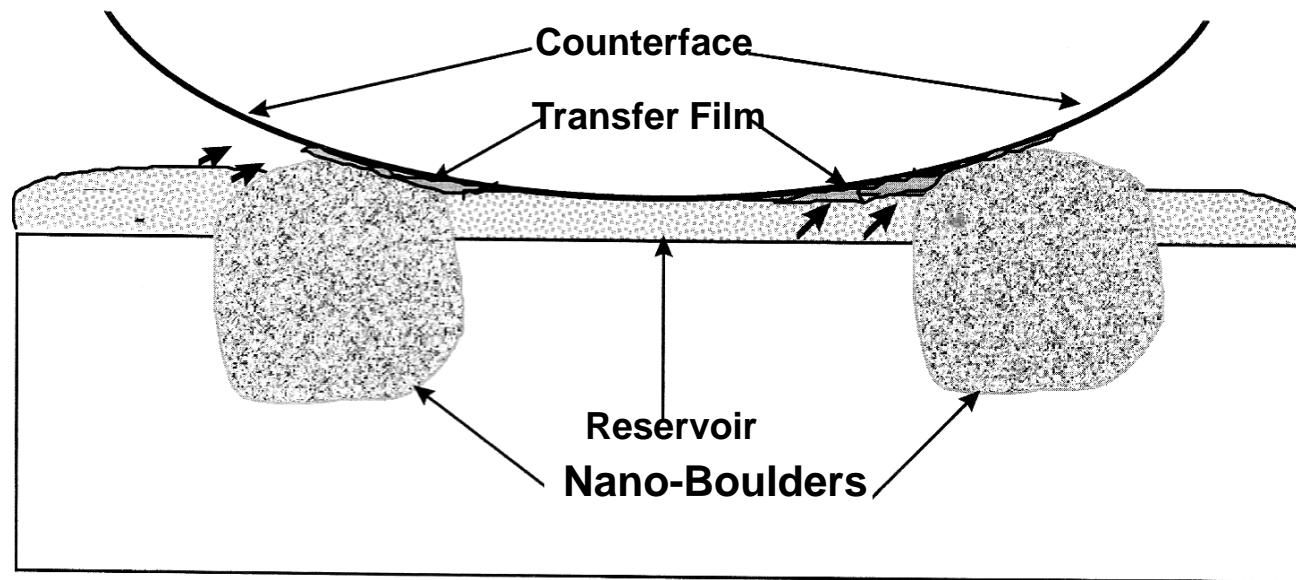


Reactions with water
Tribo-oxidation (MoO_3 formation)
Increase in the COF
Reduced Wear Life



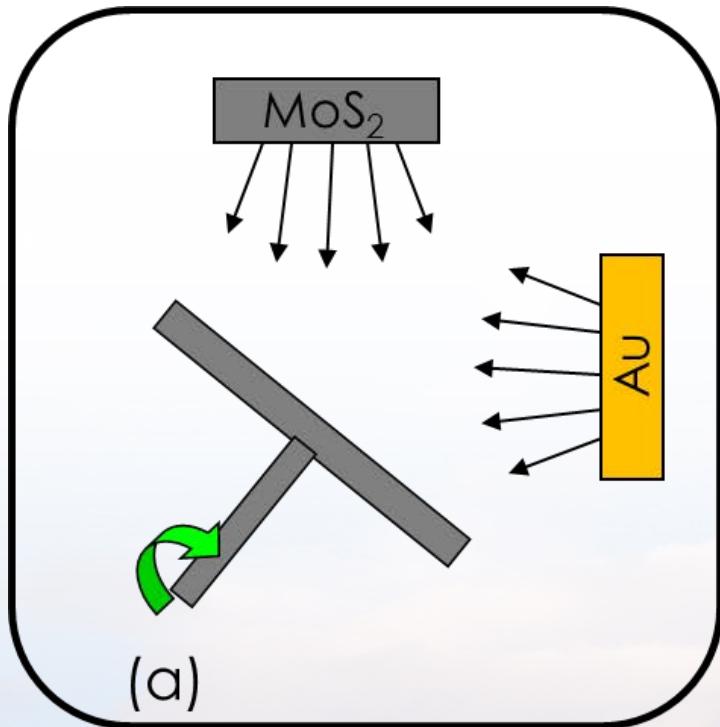
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Novel Concepts for Materials Synthesis



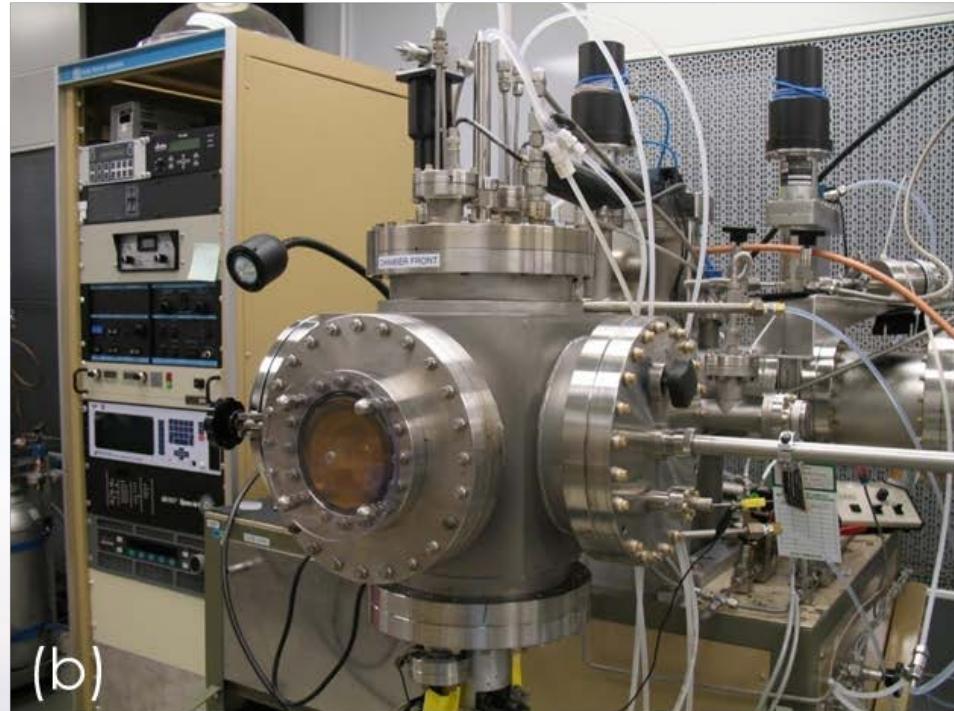
PVD-Co-Deposition for the Synthesis of MoS₂-Au Nanocomposites

Sputter targets located at 90 ° to each other and ~45° to the substrate stage



Independent rate control with stage rotation for uniformity

Experimental sputter co-deposition system used in this study



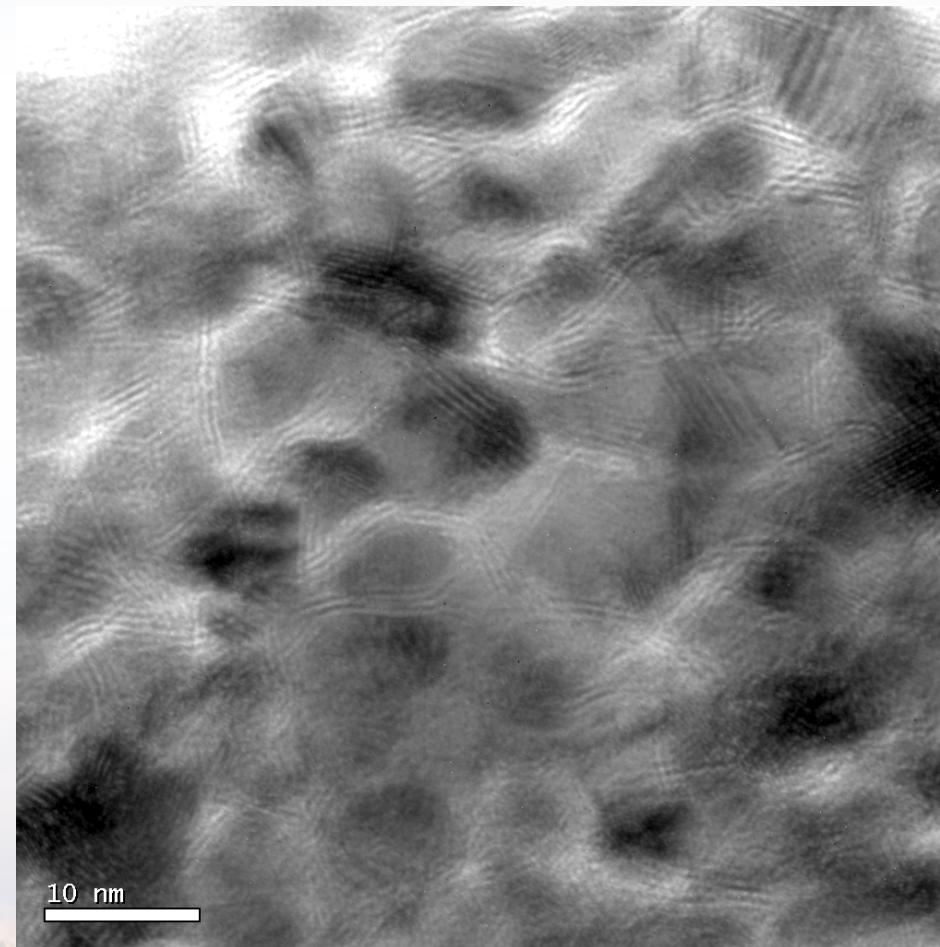
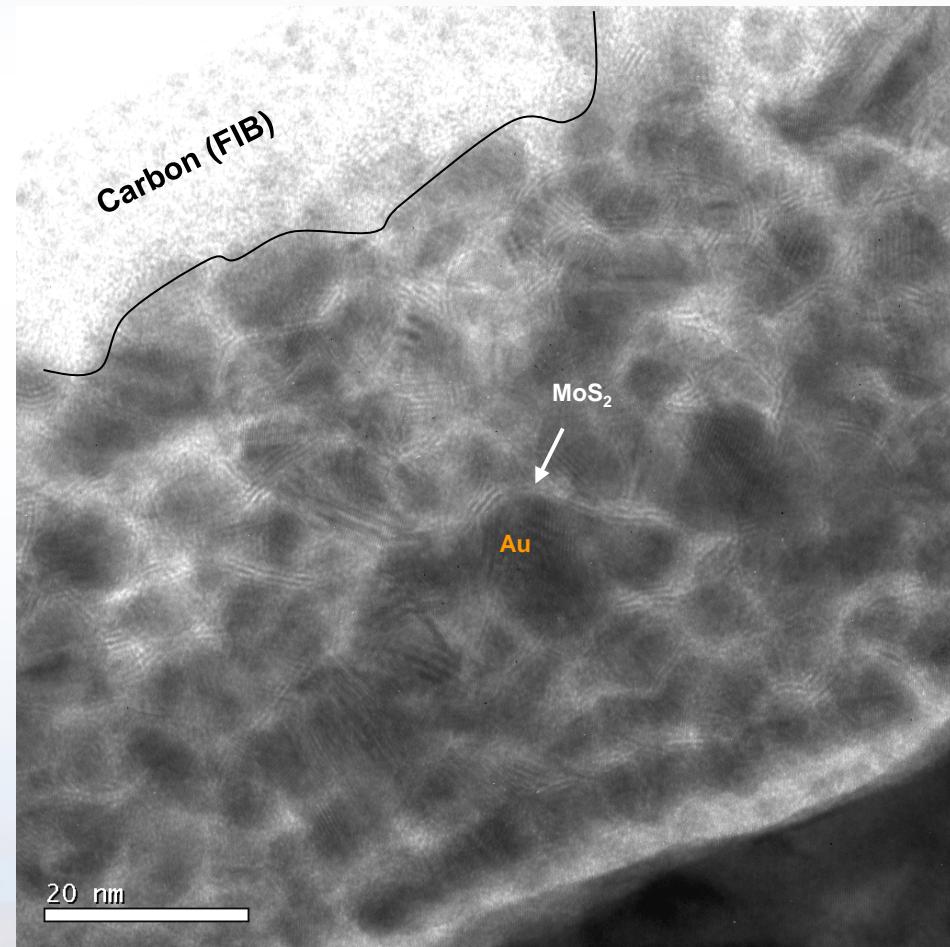
Independent QCM for rate monitor and control
Hot stage
DC Bias



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XTEM of MoS₂-Au nanocomposite grown at 200°C

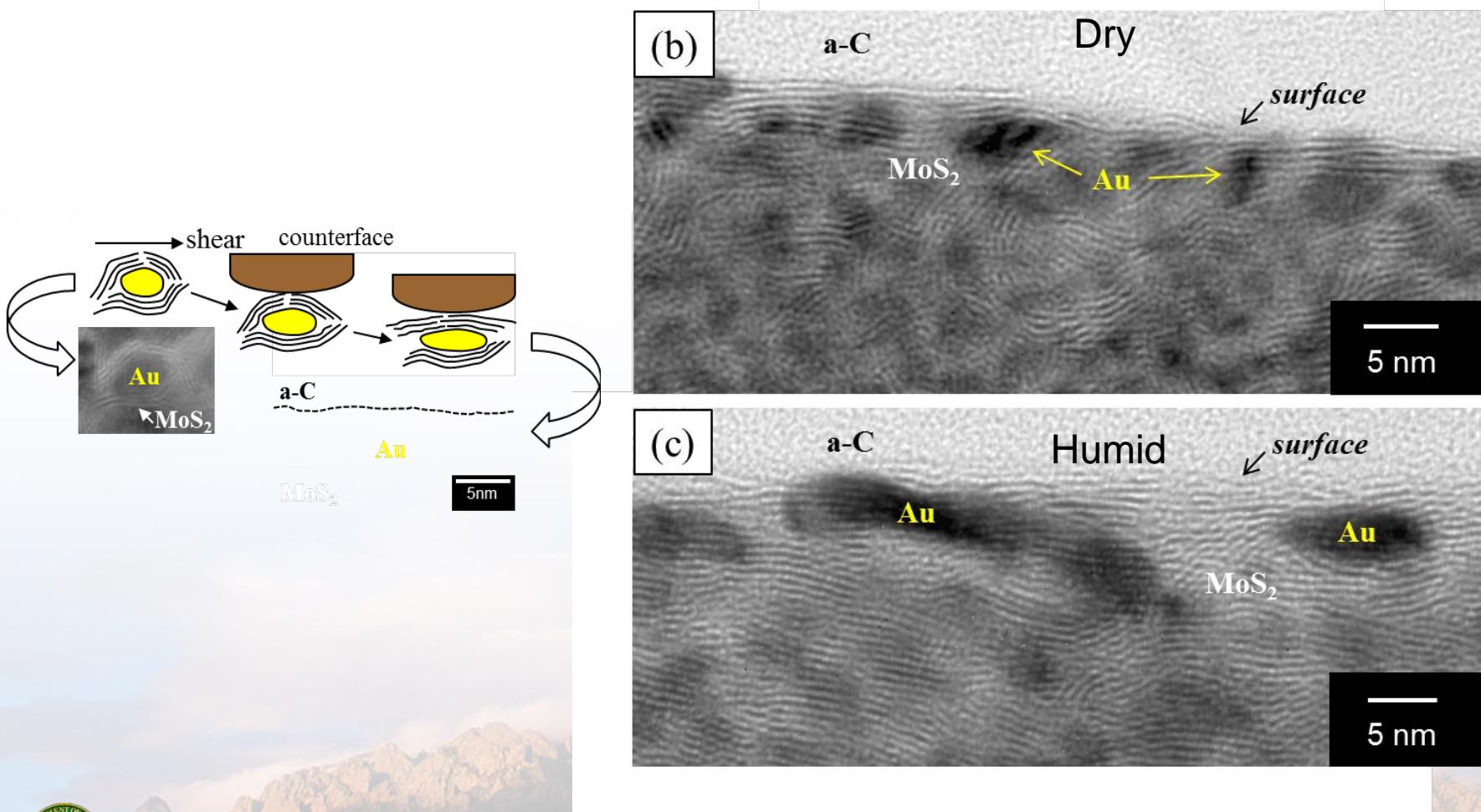


Nanocrystalline Au (~10 nm) core with MoS₂ basal planes encircling



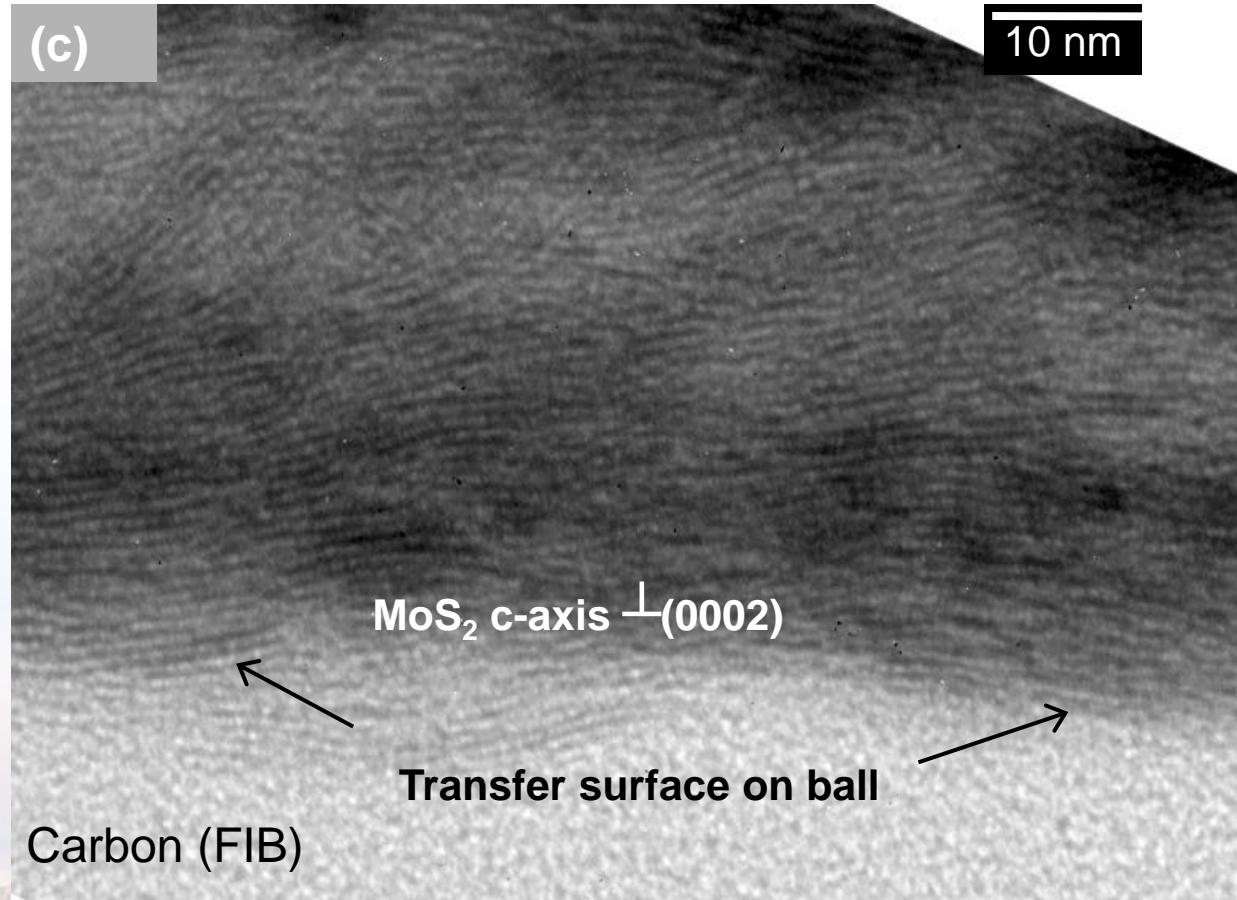
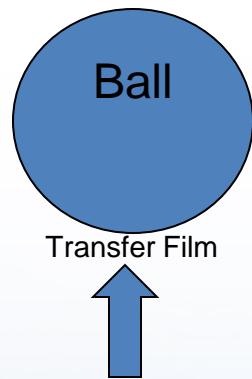
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Shear-induced exfoliation of MoS₂ (0002) with basal planes parallel to the sliding direction



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Cross-section of a Transfer Film Air (50% RH)

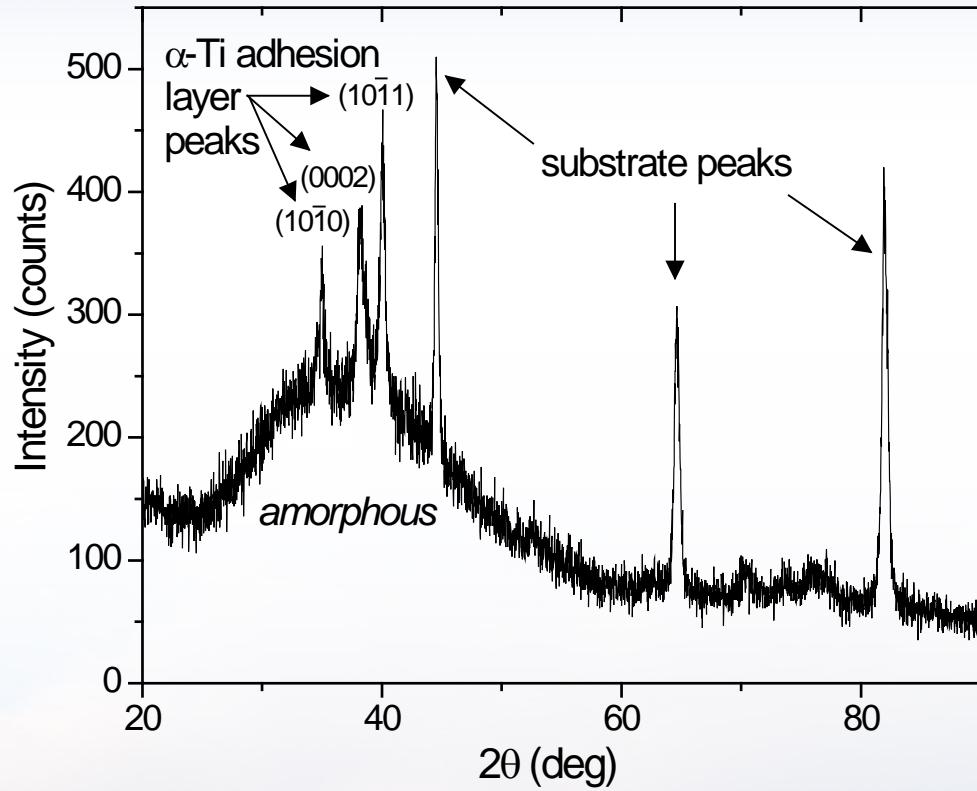
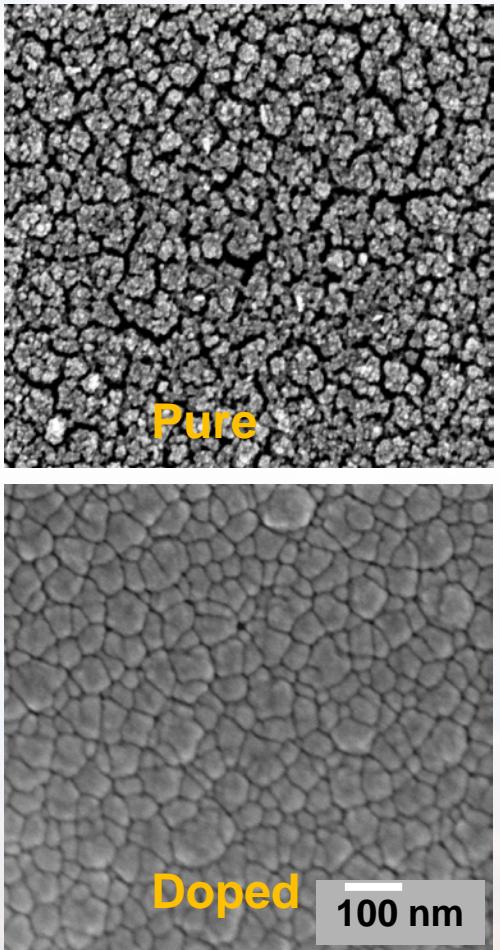


HRTEM



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Doped MoS₂ (Sb₂O₃ and Au) from Tribologix



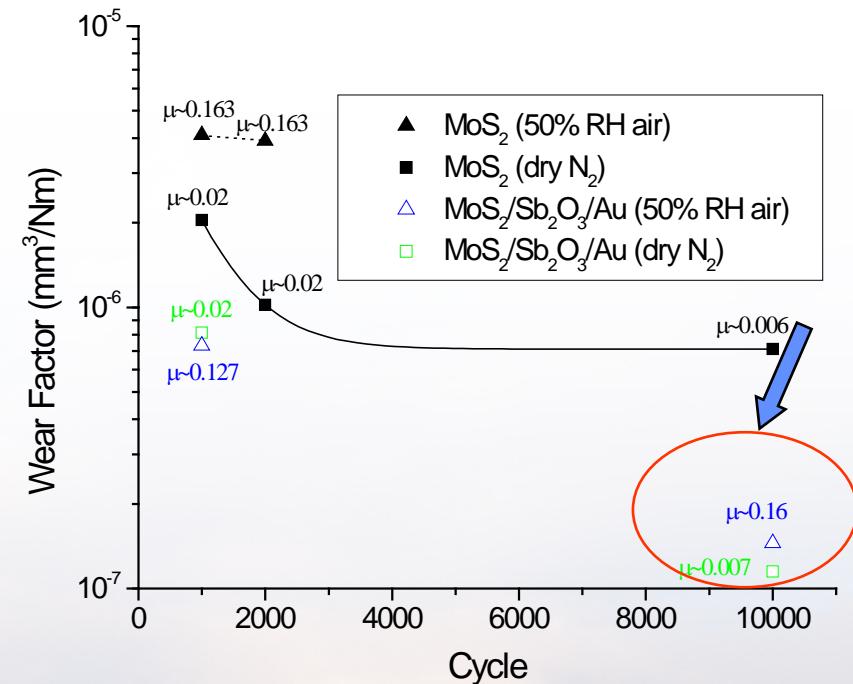
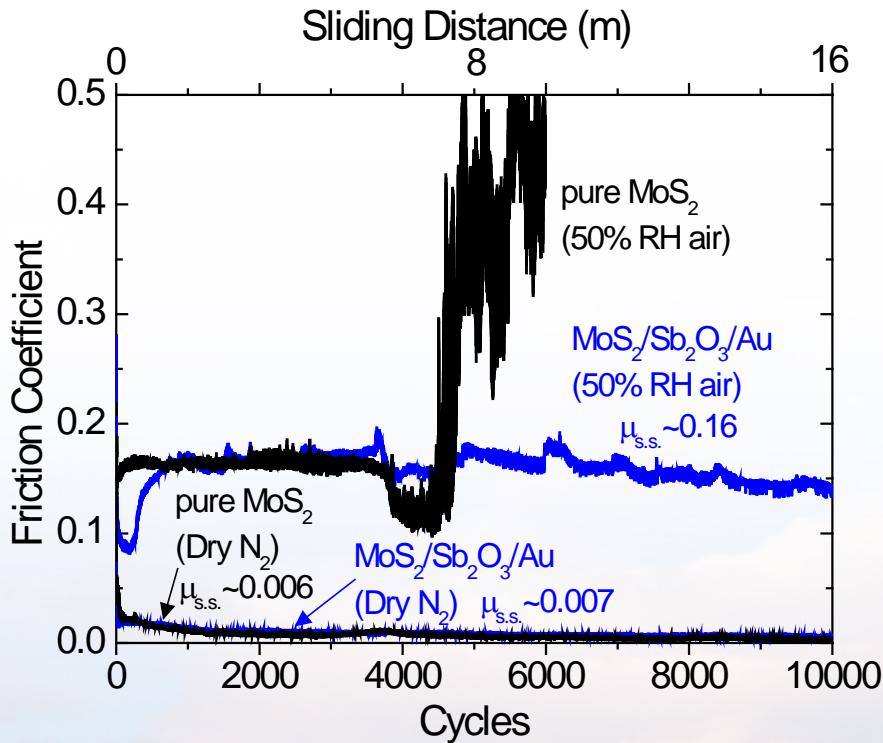
- Note the absence of peaks characteristic of any of the three constituent phases
- Instead there is a broad indicating the presence of an amorphous phase
- Doped films are much denser



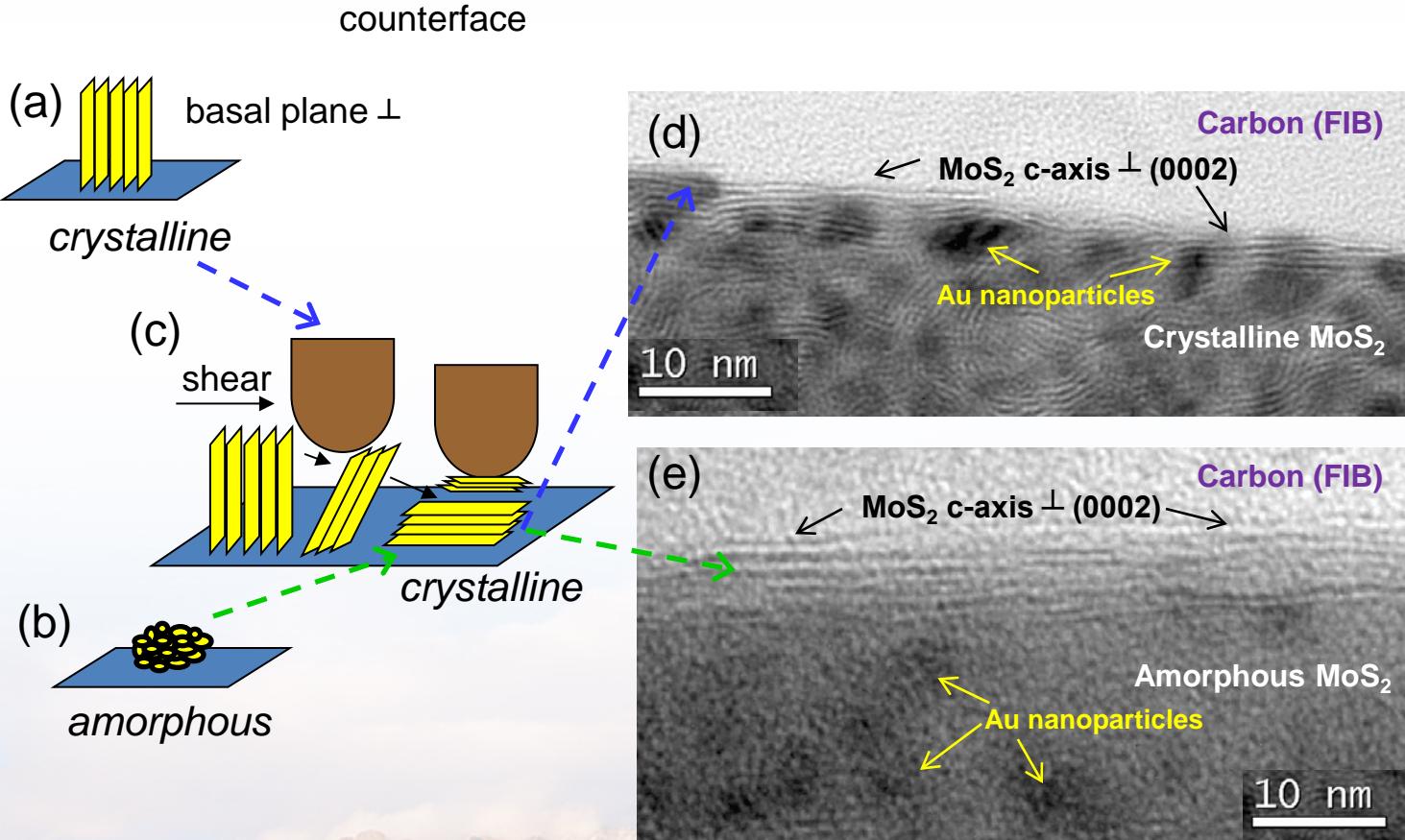
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Friction and Wear of Doped and Undoped MoS_2

Whether the MoS_2 is in nanocrystalline form or amorphous, friction induces crystallization with basal planes aligning parallel to the surface

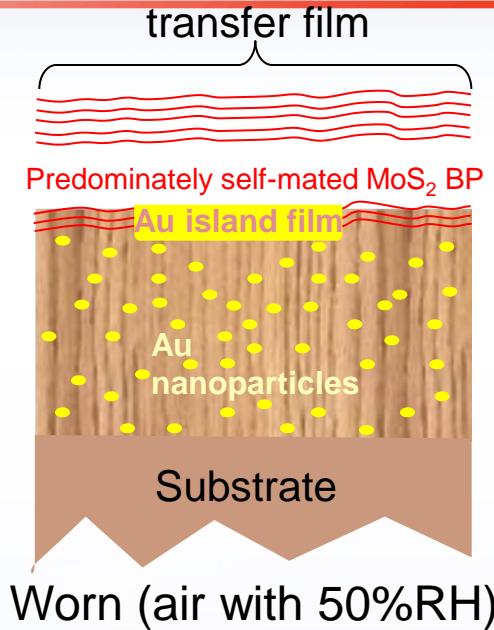
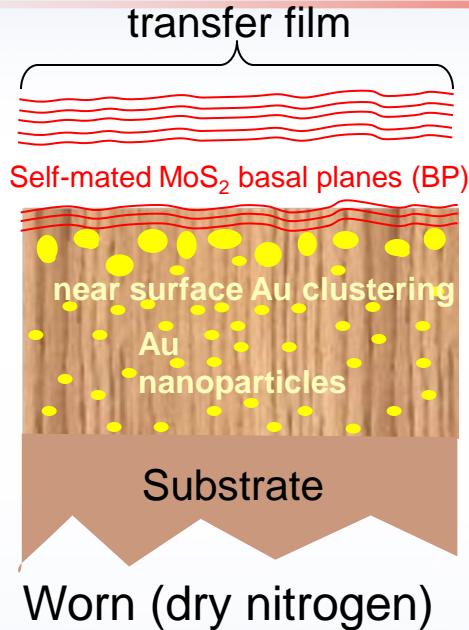


Friction-induced crystallization and reorientation of basal planes

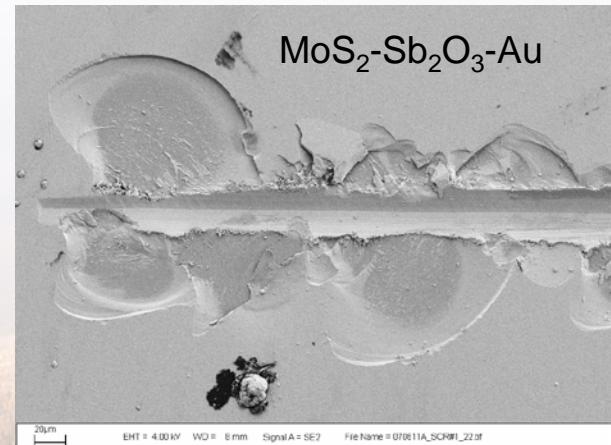


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Mechanisms of Friction



- Whether the MoS₂ is in nanocrystalline form or amorphous, friction induces crystallization with basal planes aligning parallel to the surface
- COF in air is higher but the wear rate is almost identical, indicating the environmental robustness of wear for the nanocomposite (MoS₂/Sb₂O₃/Au) coating
- Sb₂O₃ makes the coating brittle



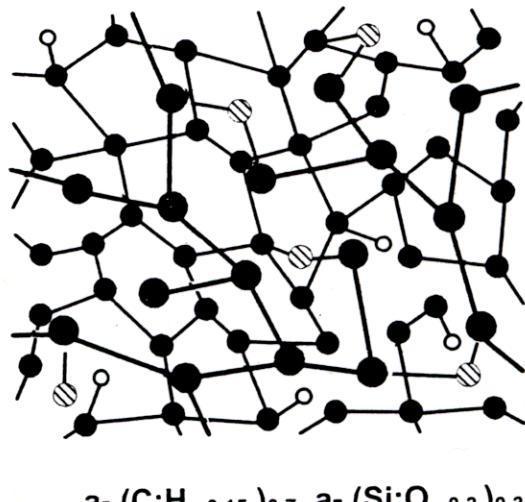
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- Diamond-Like Carbon
- Subsurface Deformation
- Load Bearing and Barrier Layers

Diamond-Like Carbon Nanocomposites

Plasma Enhanced CVD

Ployphenylmethylsiloxane precursor



Schematic of DLN atomic structure.

Interpenetrating random networks
DLC (a-C:H) and glass like a-Si:O

- Conformal coatings could provide coverage of sidewalls
- Substrate temperatures do not typically exceed 150 to 200 °C

Hardness: 9-17 GPa
Modulus: 90-140 GPa

V. F. Dorfman, *Thin Solid Films*, 212 (1992) 267-273

D. J. Kester, C. L. Brodbeck, I. L. Singer and A. Kyriakopoulos, *Surface and Coatings Tech.* 113 (1999) 268-273.

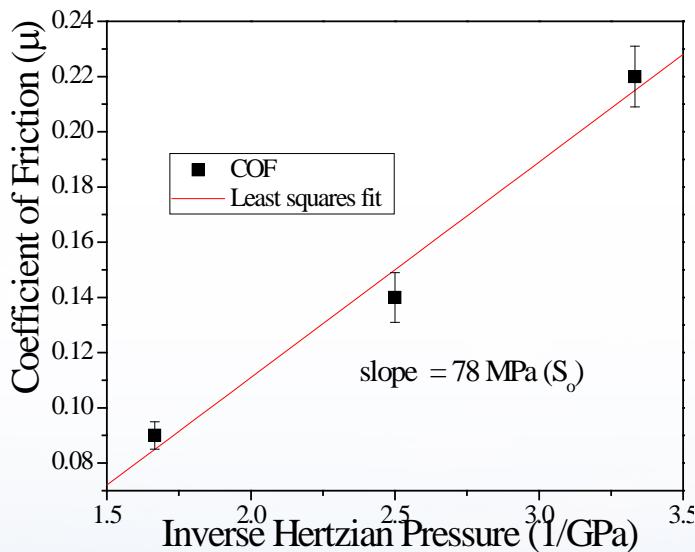
C. Venkatraman, C. Brodbeck and R. Lei, *Surface and Coatings Tech.* 115 (1999) 215-221.



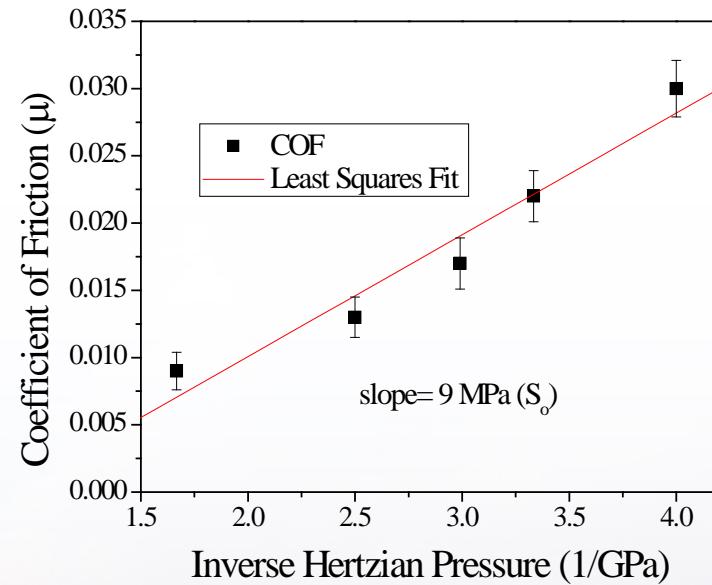
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The Friction Behavior is non-Amontonian

Air (50% RH)



Dry Nitrogen



- Load Dependence
- Environmental Effects
- (Note the difference in the slopes)

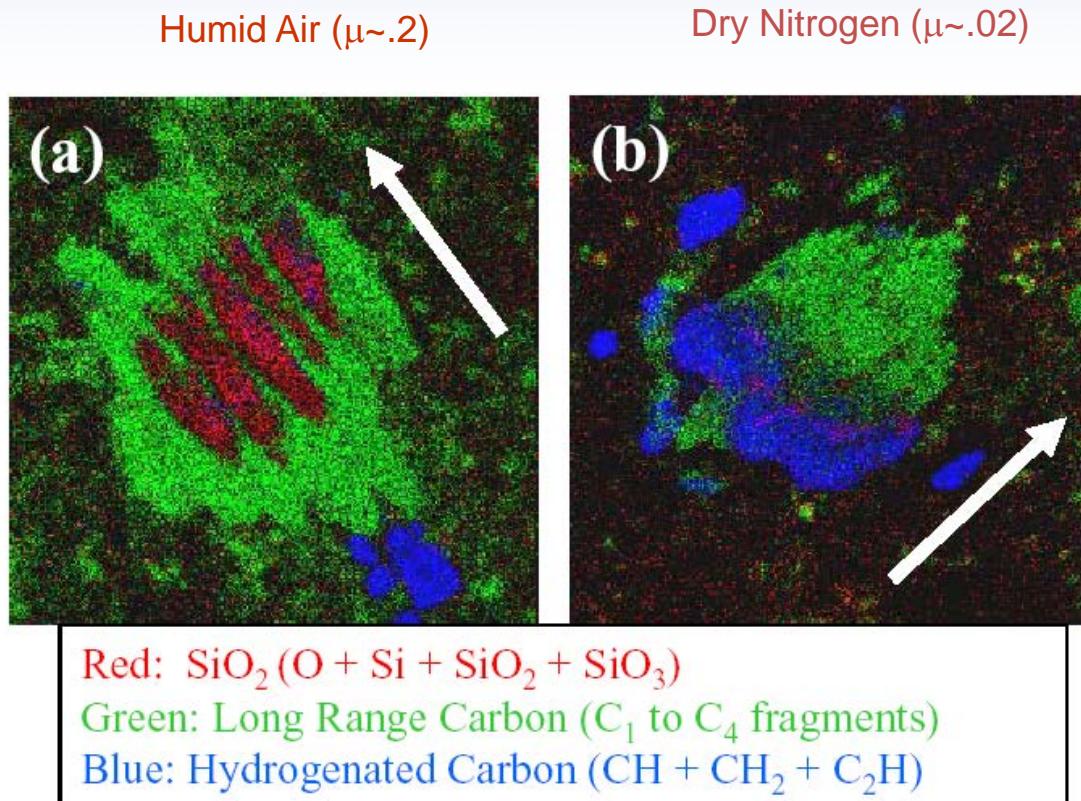
T. W. Scharf, J. A. Ohlhausen, D. R. Tallant, S. V. Prasad, J. Appl. Phys. 101 (2007)



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ToF-SIMS of Transfer Films illustrating the Chameleon Nature

ToF-SIMS



78 MPa

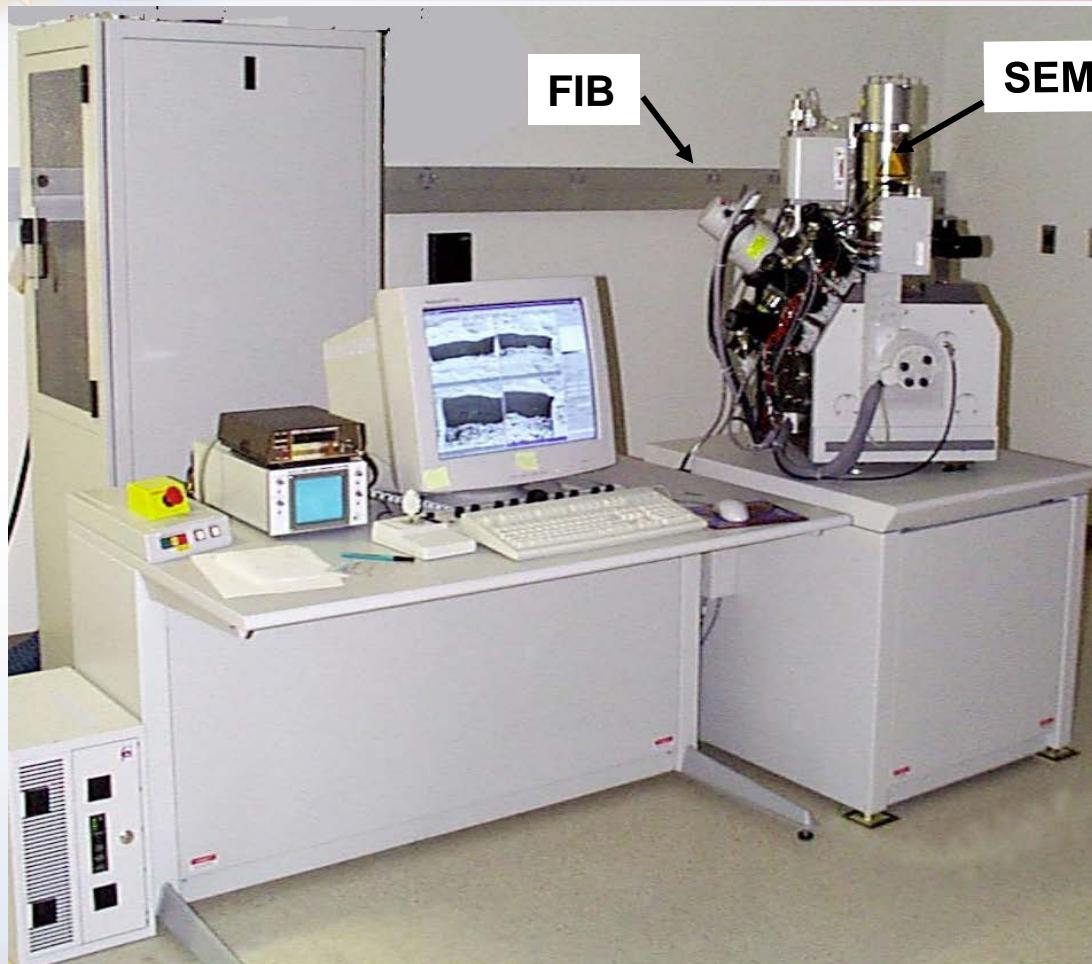
9 MPa



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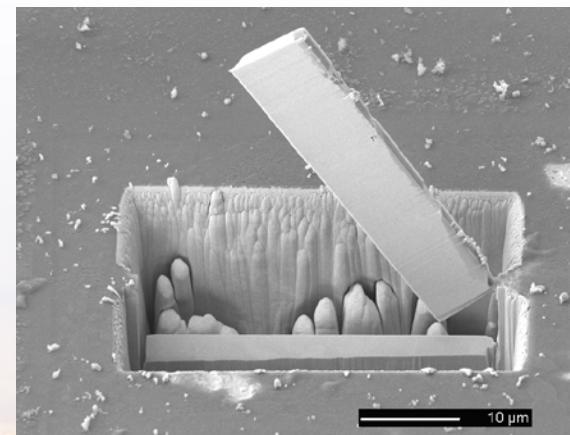
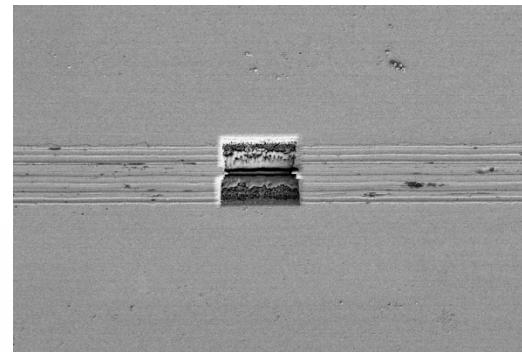


FIB-EBSD-TEM were used for analyzing the friction-induced damage below the wear surfaces



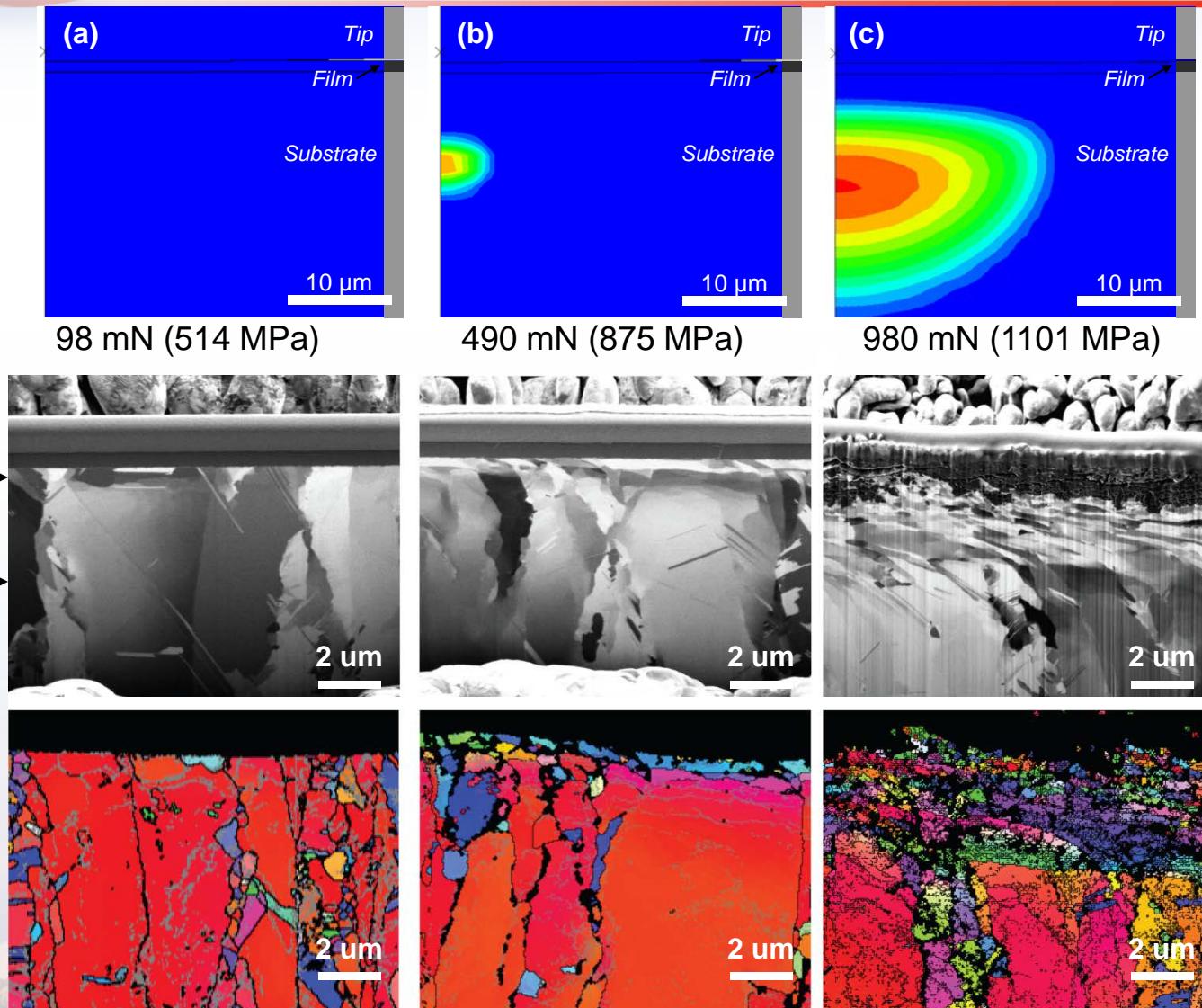
Dual-beam system: Both a FIB column and a SEM column are present on one sample chamber.

Sliding Direction

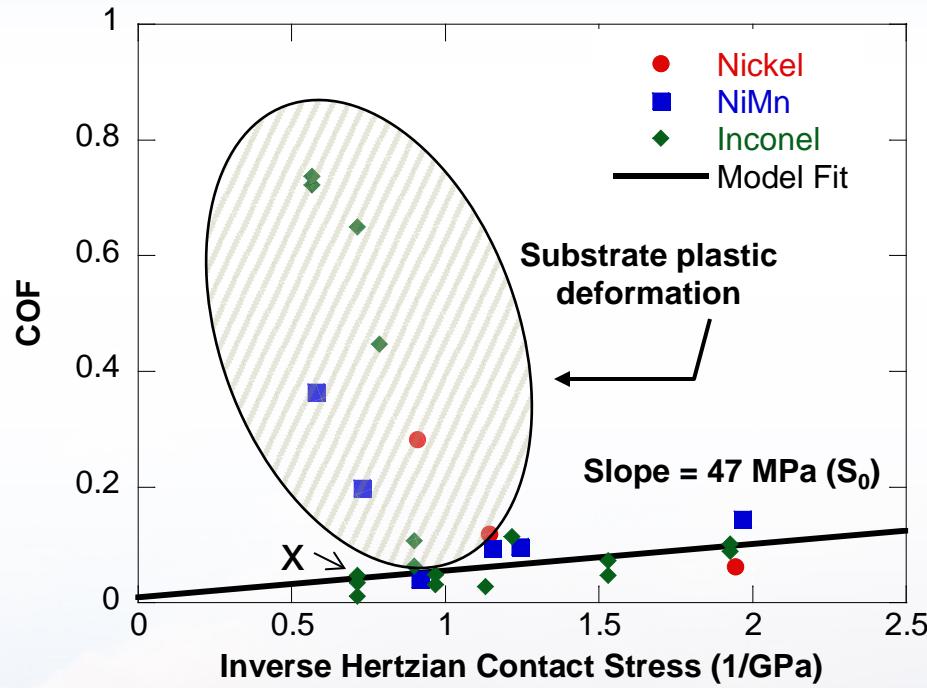
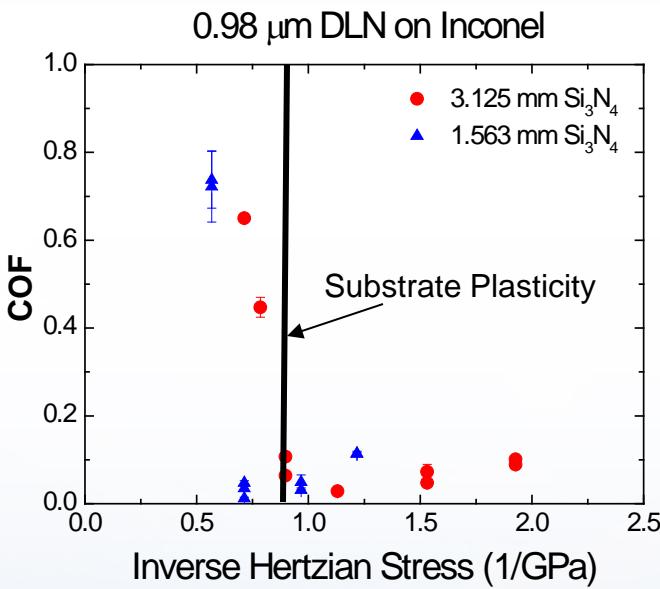


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Friction Induced Plastic Deformation in the Substrate



Deviations from Predicted Behavior



FEM predictions of substrate plastic deformation are in general agreement with observed deviations in Hertzian contact model

Multi-Layer Architectures may be necessary to mitigate substrate plastic deformation





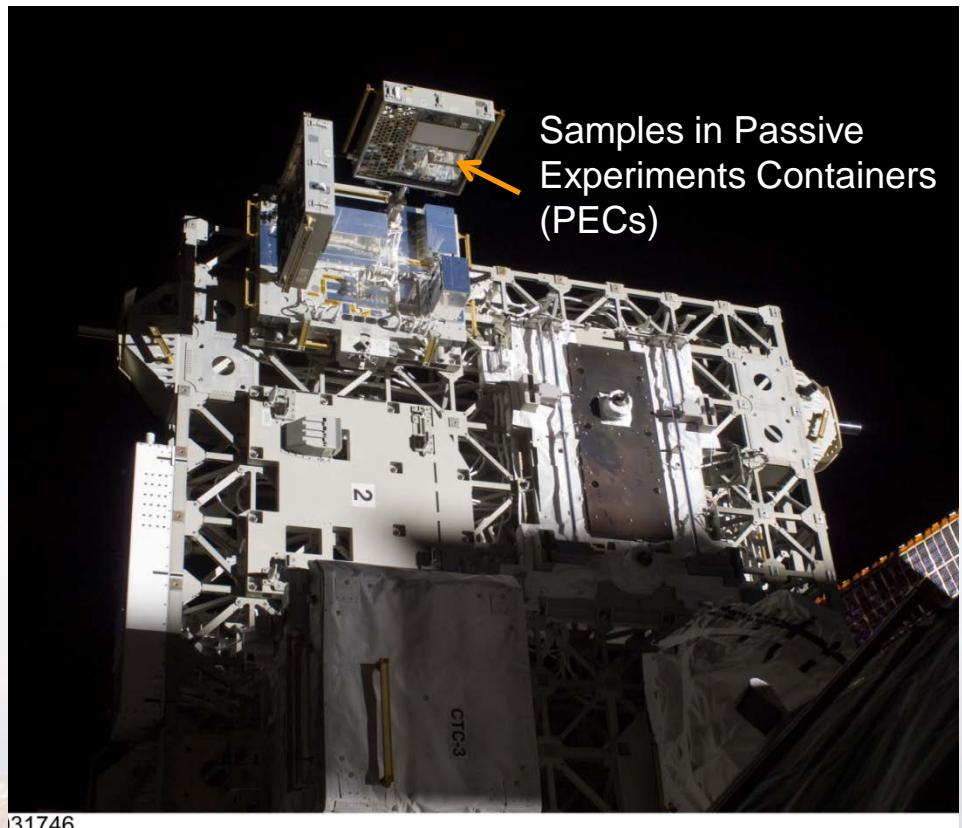
Space



Materials on the International Space Station Experiment (MISSE-7)

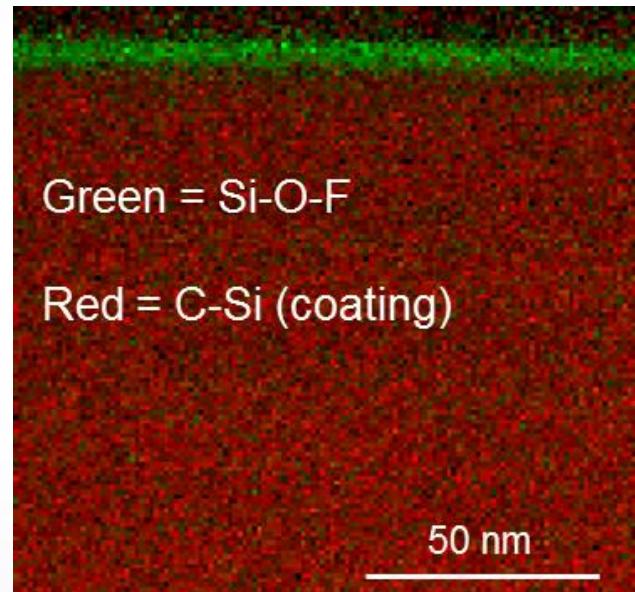
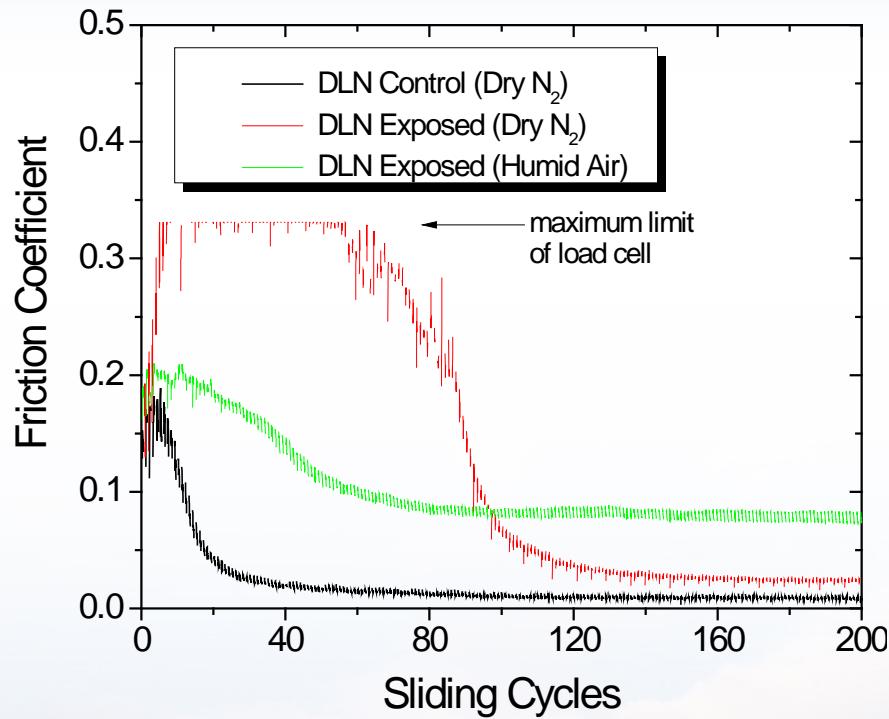
STS-129 (November 16, 2009): To ISS
STS-133 (March 1, 2011): Recovery

1. DLN
2. $\text{MoS}_2\text{-Sb}_2\text{O}_3\text{-Au}$



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The Effects of LEO environments: DLN

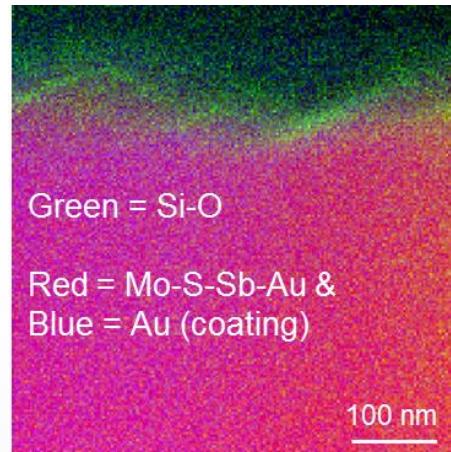
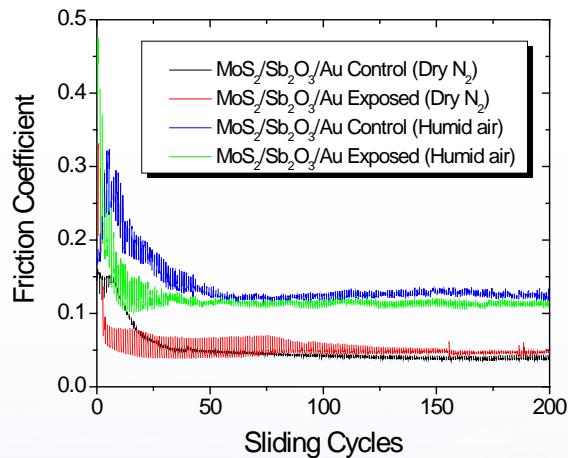


- Surface Contamination
- Increase in the Run-in Period



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The Effects of LEO environments: $\text{MoS}_2\text{-Sb}_2\text{O}_3\text{-Au}$



- Surface Contamination
- But the increase in the Run-in Period is significantly less

M.T. Dugger, T.W. Scharf and S.V. Prasad. "The Effects of Low Earth Orbit on Thin Film Solid Lubricants" Advanced Materials and Processes, May 2014



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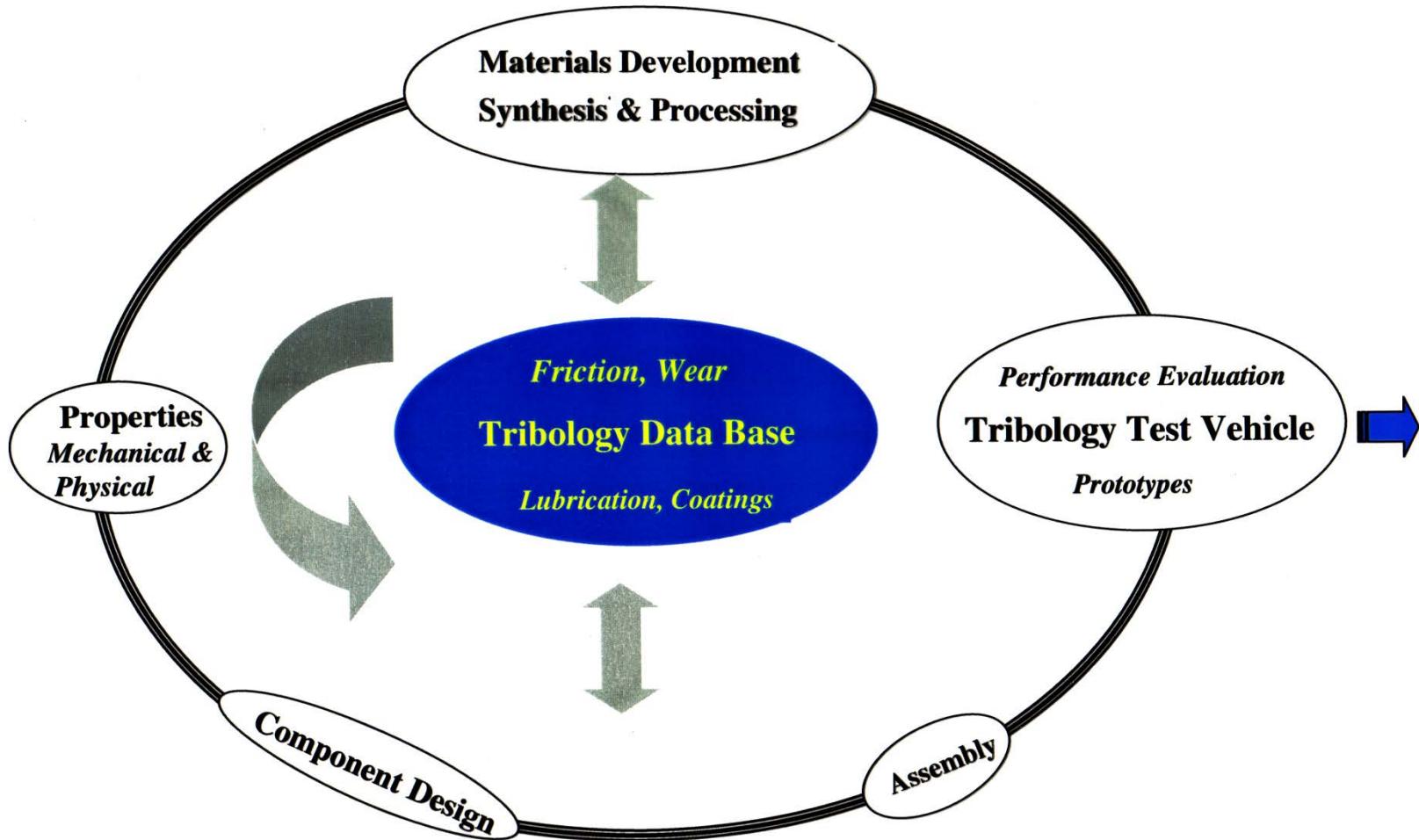
Concluding Remarks

- Solid lubricant coatings are needed, specifically when the operating conditions are beyond the liquid realm or in situations where liquids cannot be introduced.
- No single phase material can give low friction and wear in all environments and under all operating conditions
 - Many Solid Lubricant Coatings Exhibit Load Dependence on Friction
 - Environment plays a very significant role in determining the tribological behavior
 - Introduce multilayer coatings if subsurface plastic deformation cannot be avoided
- Materials designed for friction and wear mitigation, commonly referred to as tribological materials, must also meet mechanical and physical property requirements. From a practical design perspective, it becomes almost impossible to balance these diverse requirements and mitigate friction from bulk materials that are used to build tribological components.
- Avoid using metallurgically compatible metals/alloys in sliding contacts
- Use systems approach for the design of tribological contacts



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Systems approach is an integral part of Tribology



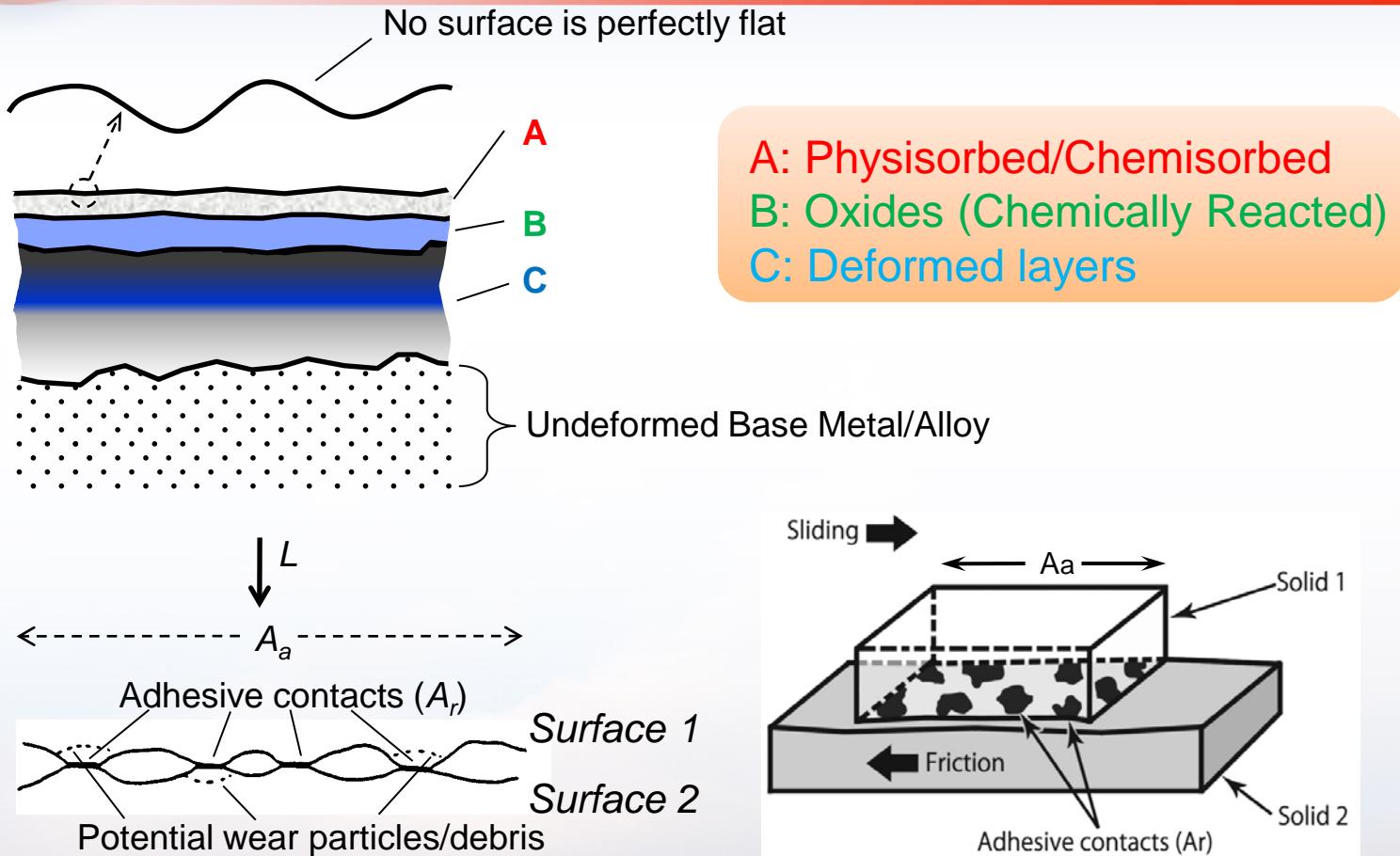


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Backup Slides

Nature of Metallic Surfaces

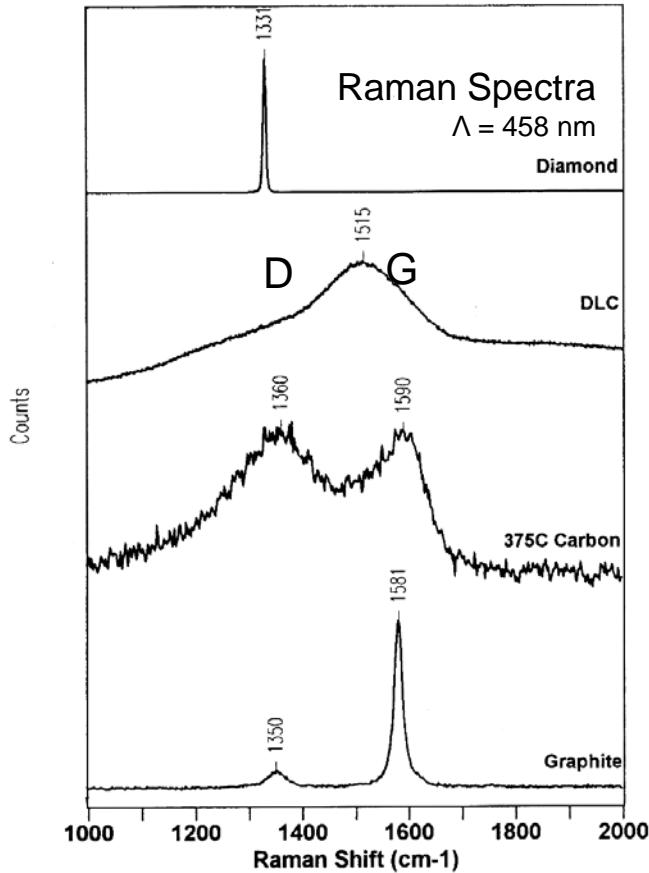
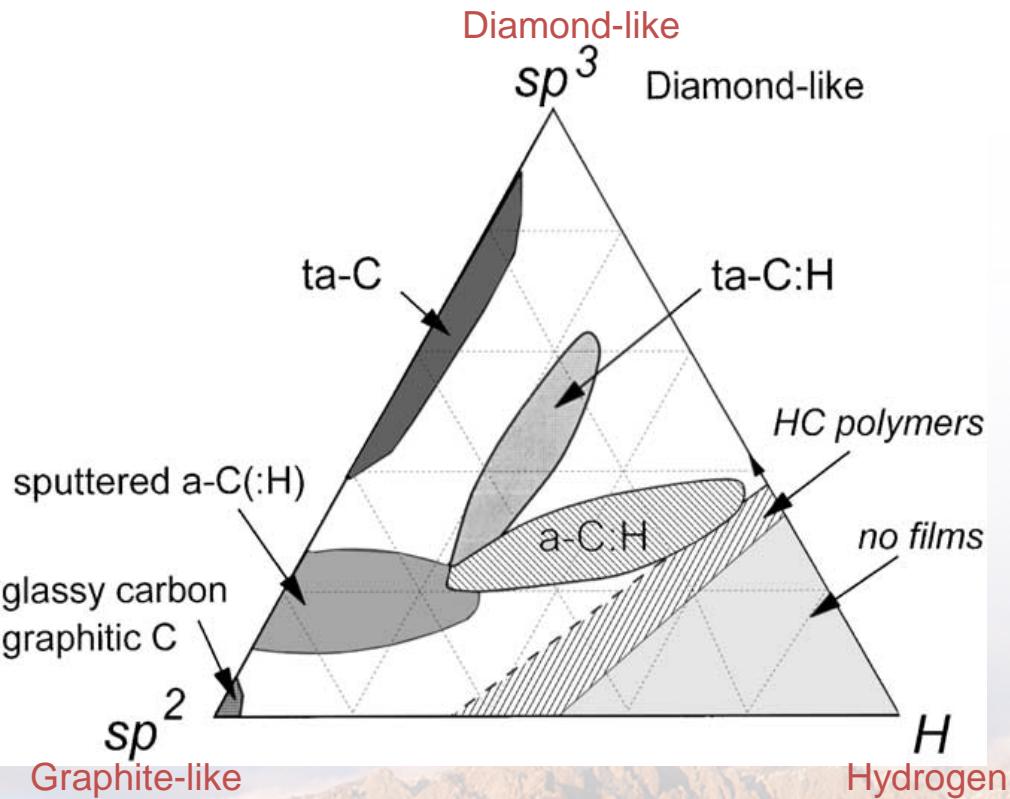


The real area of contact is a small fraction of the apparent area of contact.

- Surface topography and cleanliness are critical to performance and reliability of the coated component

Ternary phase diagram of bonding in amorphous carbon-hydrogen materials

- Diamond-like Carbon has mixed sp^2/sp^3 bonding with majority being metastable sp^3 , unless it is stabilized with C-H bonds.
- DLC can be amorphous (a-C) or hydrogenated amorphous carbon (a-C:H) (typically 10-50 atomic % H).



From J. Robertson (2002) Mater Sci Eng R 37: 129.

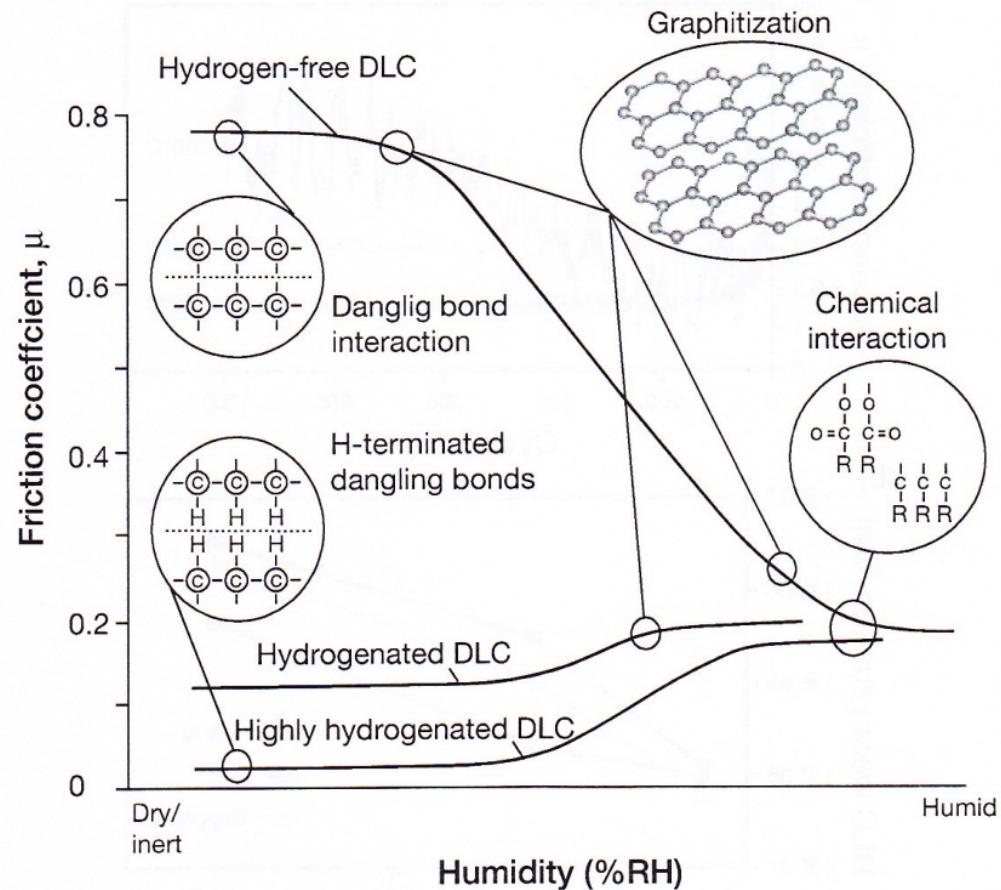
D. R. Tallant et. al, Diamond and Related Materials 4 (1995) 191-199



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Synthesizing an Environmentally Robust DLC is Still a Challenge!

- Hydrogenated DLCs give low friction in dry environments.
- Hydrogen free DLCs require moisture to terminate dangling bonds.

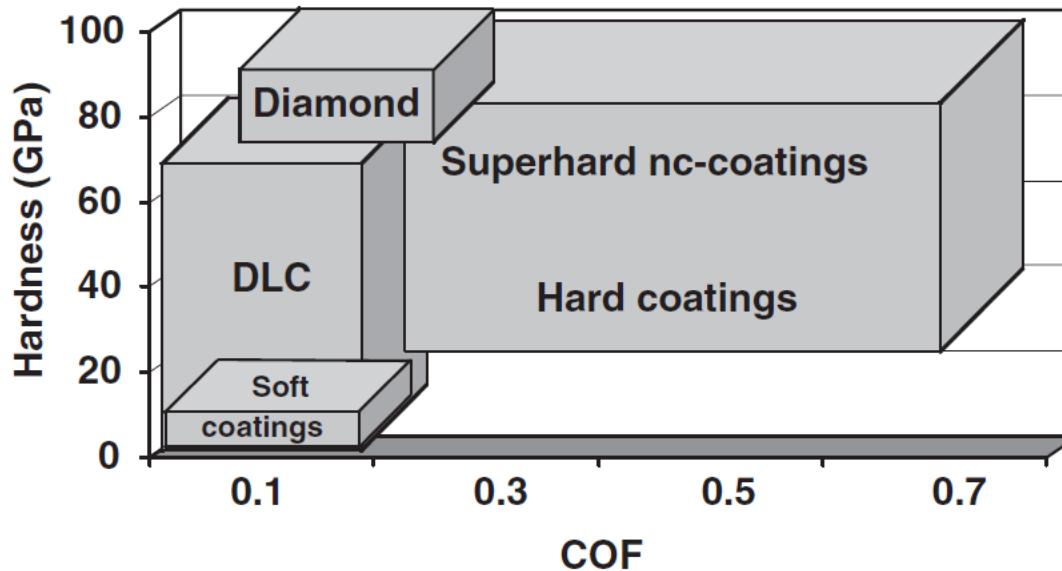


H. Ronkainen and K. Holmberg, "Environmental and Thermal Effects on the Tribological performance of Coatings", In: C. Donnet and A. Erdemir (eds.), *Tribology of Diamond-Like Carbon Films: Fundamentals and Applications*. Springer 2008



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DLC can provide both high hardness and low friction



- DLCs exhibit the unusual combination of high hardness and elastic modulus in conjunction with low friction.
- This appears like an exception to the Bowden and Tabor Model which requires a low hardness for shear accommodation to achieve low friction.

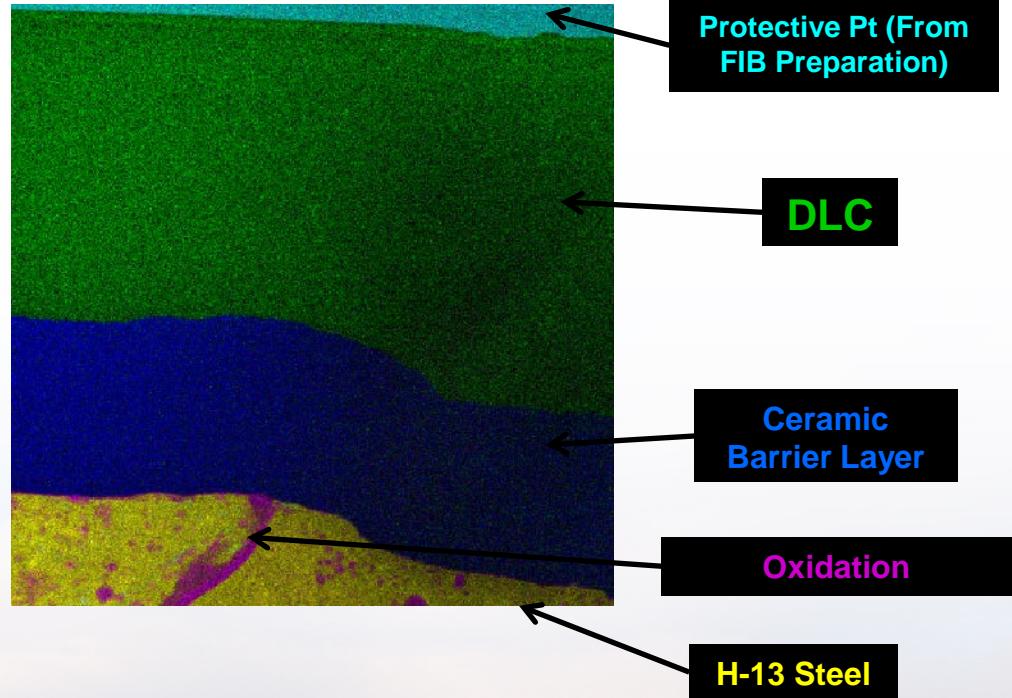
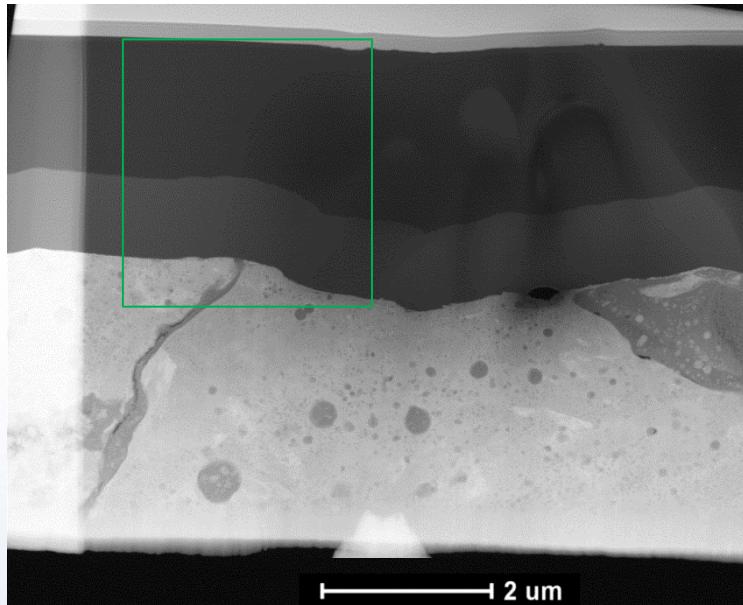


C. Donnet and A. Erdemir “Diamond-like Carbon Films: A Historical Overview,” from C. Donnet and A. Erdemir (eds.), *Tribology of Diamond-Like Carbon Films: Fundamentals and Applications*. Springer 2008



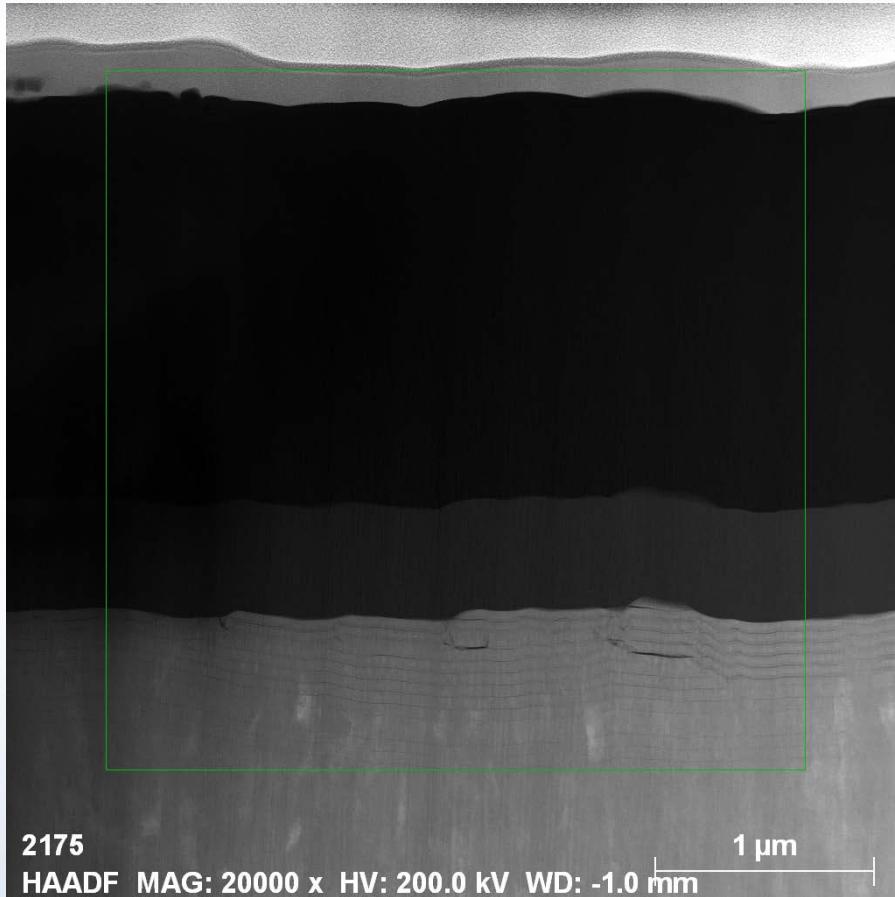
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Ceramic Barrier Layers may be needed for high temperature applications

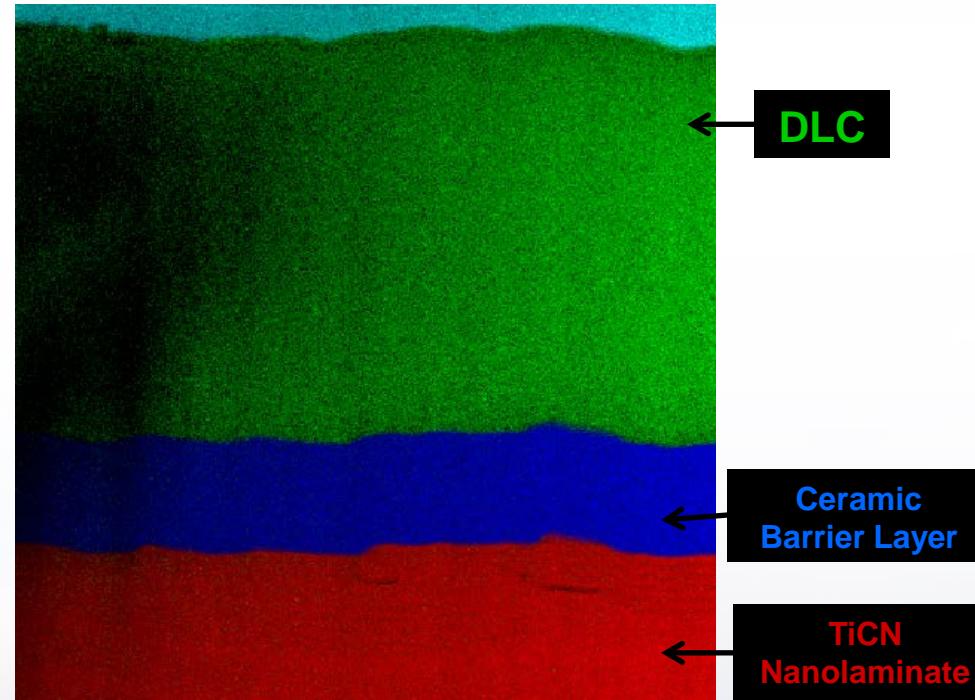


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Multilayer Coatings: Adhesion, Load Bearing, Diffusion Barrier



Micro-cracks between nanolaminate layers



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Tribo-Oxidation seen from the TEM of H-13 tested at 300C

