

# **A Comparison of Mortar and Tied Contact Approaches for Acoustic and Structural Acoustic Meshes**

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# Motivation

- **Acoustic and structural meshes typically generated independently**
- **Acoustic and structural meshes almost always have different mesh density requirements**
- **Mesh tying methods have been researched extensively in solid mechanics – but not in acoustics or structural acoustics**
- **Fully coupled simulations are needed**
  - **Coupled modes, coupled frequency response**

**Mesh tying methods are needed for  
nonconforming wet interface**



# Acoustic-Structure Interaction

## Equations of motion of solid

$$\rho u_{tt} - \nabla \bullet \sigma = f(x, t) \quad \Omega_e x[0, T]$$

## Acoustic wave equation for fluid

$$\frac{1}{c^2} \phi_{tt} - \Delta \phi = 0 \quad \Omega_f x[0, T]$$

## Boundary conditions

$$\frac{\partial \phi}{\partial n} = \dot{u}_n \quad \text{Continuity of Displacement}$$

$$\sigma_n = -\hat{n} \bullet \dot{\psi} \quad \text{Continuity of Stress}$$

# Weak Formulation for Time Domain

Find  $(u, \phi) : [0, T] \rightarrow H_1(\Omega_f) \times (H_1(\Omega_e))^3$

$$\rho(u_{tt}, v)_{\Omega_e} - (\sigma, \nabla v)_{\Omega_e} + (\rho \dot{\phi}, v)_{\partial\Omega} = (f, v)_{\Omega_e} \quad \forall v \in (H^1(\Omega_f))^3$$

$$\frac{1}{c^2}(\phi_{tt}, \psi)_{\Omega_f} + (\nabla \phi, \nabla \psi)_{\Omega_f} + \frac{1}{c^2}(2\nabla \phi \bullet \nabla \dot{\phi}, \psi) + \frac{1}{c^2}\left(\frac{B}{c^2} \phi \dot{\phi}, \psi\right) +$$

$$b(\nabla \dot{\phi}, \nabla \psi) - (\dot{u}_n, \psi)_{\partial\Omega} = 0 \quad \forall \psi \in H^1(\Omega_f)$$

Discretized form:

$$\begin{bmatrix} M_s & 0 \\ 0 & -M_a \end{bmatrix} \begin{bmatrix} \ddot{u} \\ \ddot{\phi} \end{bmatrix} + \begin{bmatrix} C_s & L^T \\ L & -C_a \end{bmatrix} \begin{bmatrix} \dot{u} \\ \dot{\phi} \end{bmatrix} + \begin{bmatrix} K_s & 0 \\ 0 & -K_a \end{bmatrix} \begin{bmatrix} u \\ \phi \end{bmatrix} = \begin{bmatrix} f_s \\ f_a \end{bmatrix}$$



# Coupling Terms

- **The surface integral is the key to the coupling methods**

$$L_{ij} = \int_{\Gamma} N_{M_i} N_{S_j} d\Gamma$$

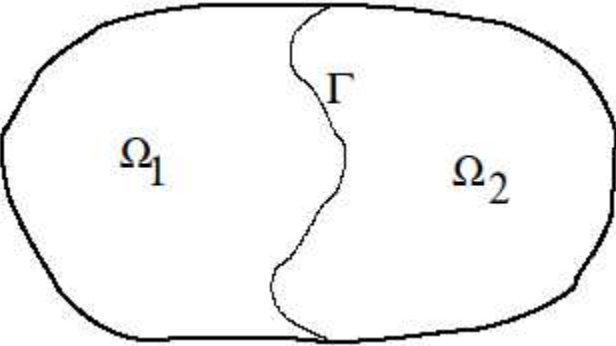
$N_{M_i}$  Surface shape function on master side

$N_{S_j}$  Surface shape function on slave side

- For conforming meshes, this is the classical structural-acoustic coupling matrix since  $N_M$  and  $N_S$  are the same functions and are on the same surface
- For mortar methods,  $N_M$  and  $N_S$  are different functions and are defined on different surfaces, but the integral is still the same
  - Since  $N_M$  and  $N_S$  are defined on different surfaces, a surface projection method is needed – we use approach of Laursen et al.

# Mesh Tying Methods for Acoustics

Weak formulations



The diagram shows a domain divided into two subdomains,  $\Omega_1$  and  $\Omega_2$ , by an interface  $\Gamma$ . The subdomains are labeled  $\Omega_1$  and  $\Omega_2$ , and the interface is labeled  $\Gamma$ .

$$\int_{\Omega_1} \left[ \frac{1}{c^2} \ddot{\psi} \phi + \nabla \psi \cdot \nabla \phi \right] d\Omega_1 = 0 \quad \int_{\Omega_2} \left[ \frac{1}{c^2} \ddot{\psi} \phi + \nabla \psi \cdot \nabla \phi \right] d\Omega_2 = 0$$

Constraint equations on interface

- Classical MPC equations
- Mortar method

$$\psi_S = \sum c_i \psi_M$$

$$\int_{\Gamma} (\psi_1 - \psi_2) \eta d\Gamma = 0$$



# Discretization of Boundary Constraint

Boundary Constraint Equation:

$$\int_{\Gamma} (\psi_1 - \psi_2) \eta d\Gamma = 0$$

Discretization:

$$\psi_M = \sum N_{M_i}(x) \psi_{M_i} \quad \psi_S = \sum N_{S_i}(x) \psi_{S_i}$$

$$\eta = N_{S_j}$$

$$\int_{\Gamma} (\psi_1 - \psi_2) \eta d\Gamma = \sum_i \int_{\Gamma} N_{M_i} N_{S_j} d\Gamma - \sum_i \int_{\Gamma} N_{S_i} N_{S_j} d\Gamma$$

Mortar method for acoustics involves same surface integrals as for conforming structural acoustics



# Mesh Tying Methods for Structural Acoustics

1. **Conforming finite element approach**
  - **Requires matching meshes**
2. **Classical multipoint constraint equations with ghost nodes**
3. **Mortar method with ghost nodes**
4. **Classical mortar method**
  - **Flemisch et al, 2006**

In all cases we need to evaluate integrals of the type:  $\int_{\Gamma} N_M N_S d\Gamma$

$N_M$  Surface shape function on master side

$N_S$  Surface shape function on slave side





# Mesh Tying Methods for Structural Acoustics

## Ghost nodes approach:

- Add “ghost” acoustic degrees of freedom to solid nodes on wet interface
- Use conforming coupling operators to couple solid nodes on wet interface to appended acoustic dof
- Couple acoustic dof on both sides of wet interface with mortar or standard MPC equations

For conforming meshes, this method reduces to a conforming structural acoustics

Same constraint equations for acoustic-acoustic coupling and structural-acoustic coupling

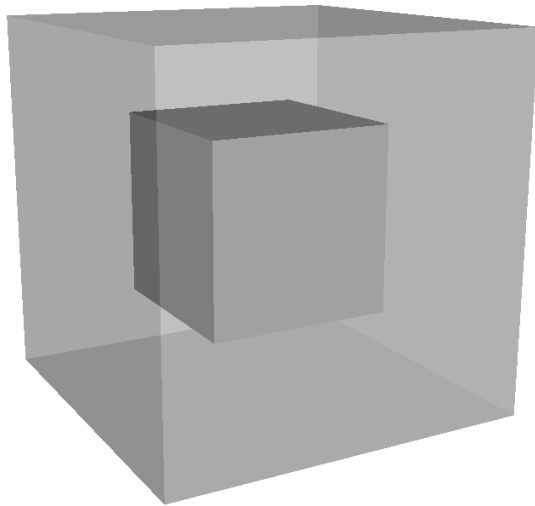


# **Mesh Tying for Structural Acoustics – Ghost Node Approach**

**(solid dof + ghost acoustic dof)**



# Cube-In-Cube Structural Acoustic Example

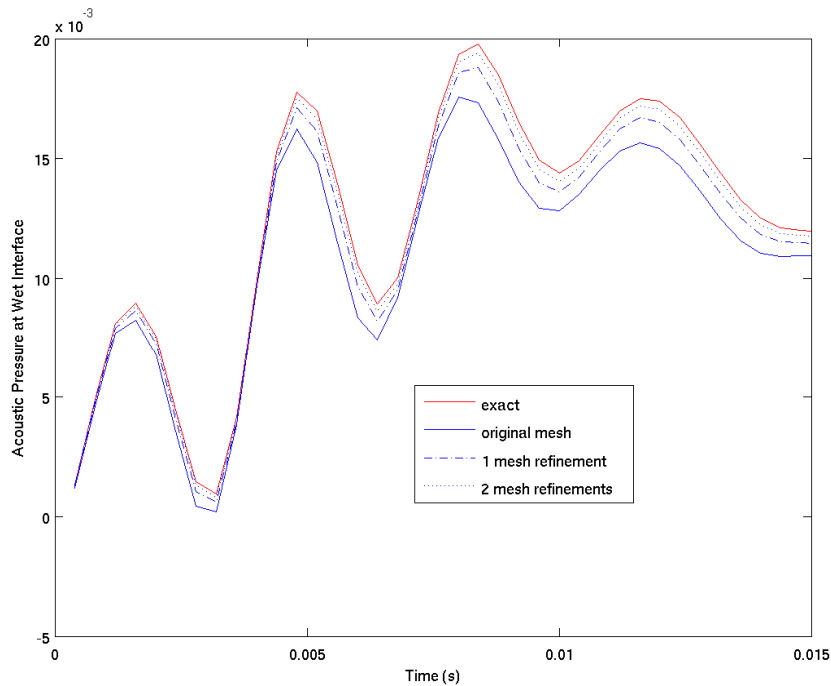


- Aluminum tank filled with water
- time-dependent pressure load (sinusoid) applied to end of tank
- far-end of tank fixed to rigid wall

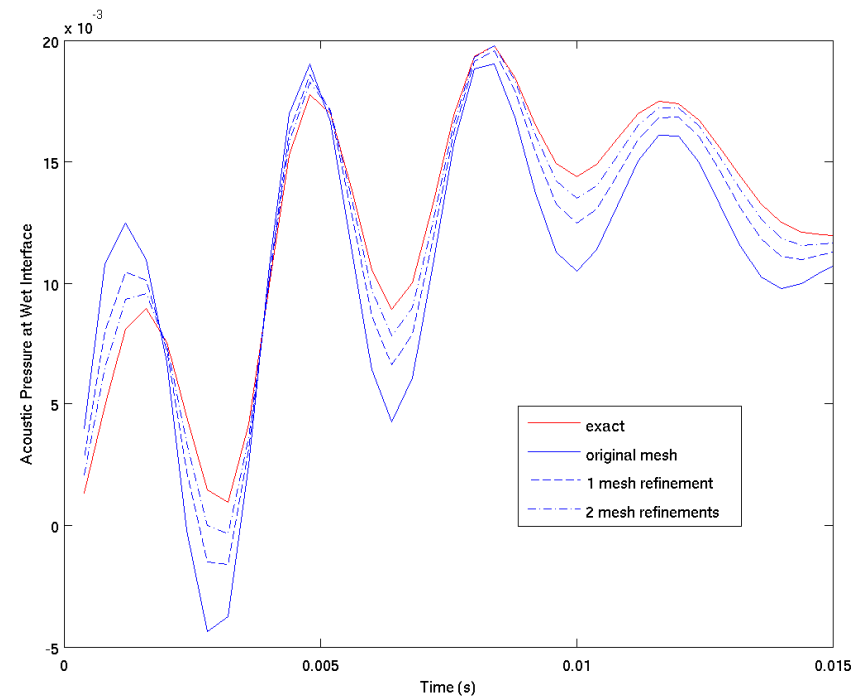
We compare the results using 3 methods:

- **Conforming meshes**
- **Nonconforming meshes with ghost nodes and classical MPCs**
- **Nonconforming meshes with ghost nodes and mortar constraints**

# Convergence Results for Cube-in-Cube

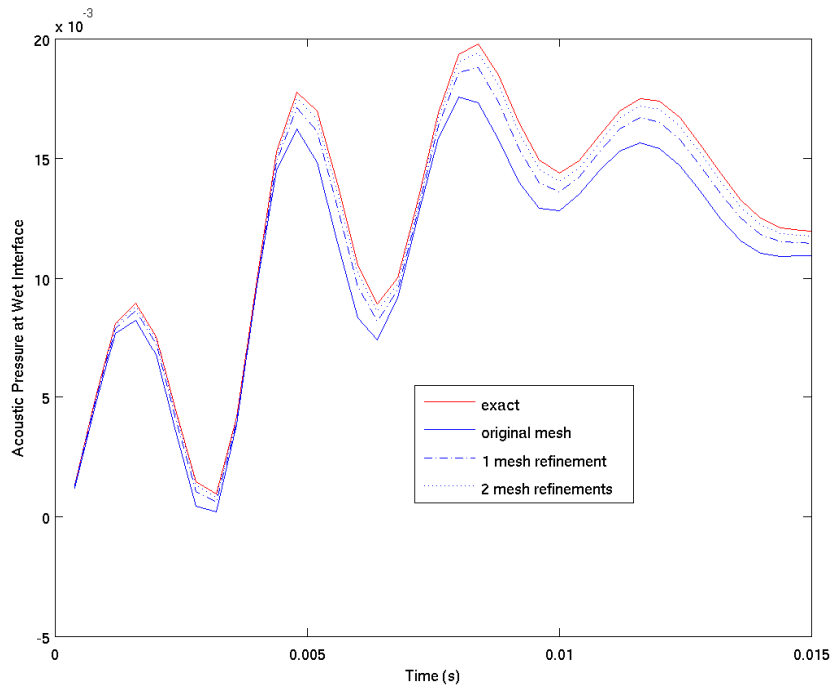


Conforming meshes

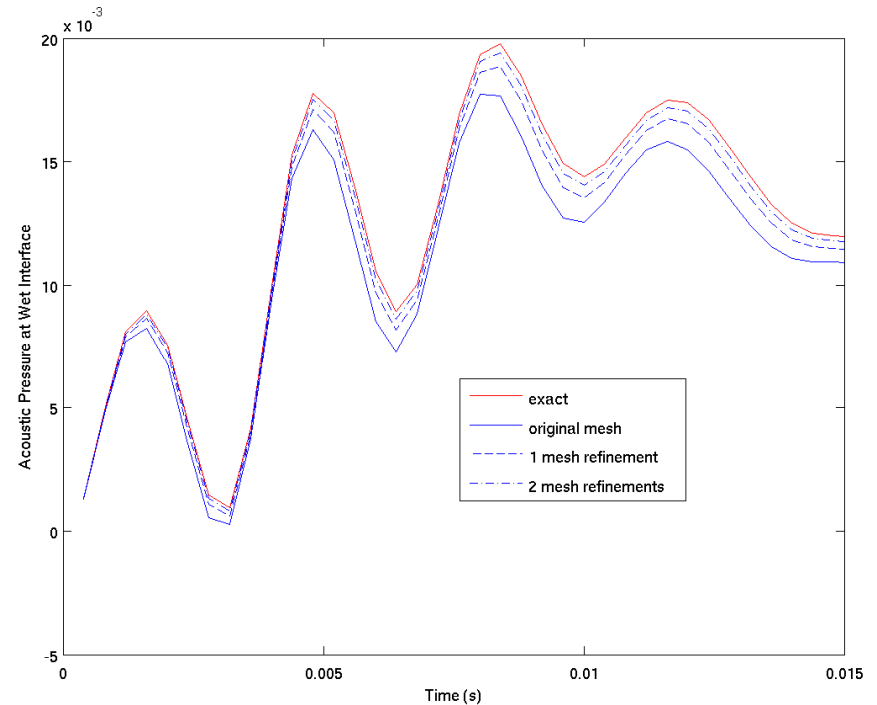


Ghost nodes approach  
with classical MPC's

# Convergence Results for Cube-in-Cube



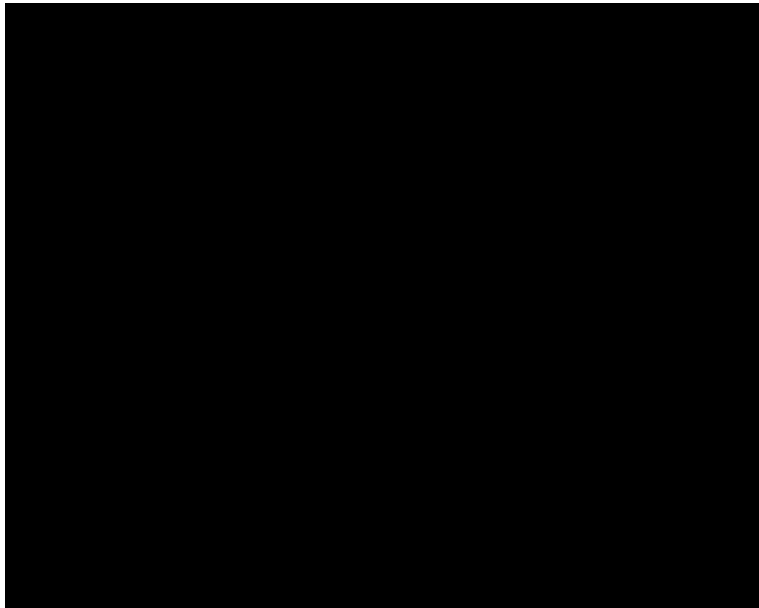
Conforming meshes



Ghost nodes approach  
with mortar method



# Convergence Results for Water-Castor Oil System



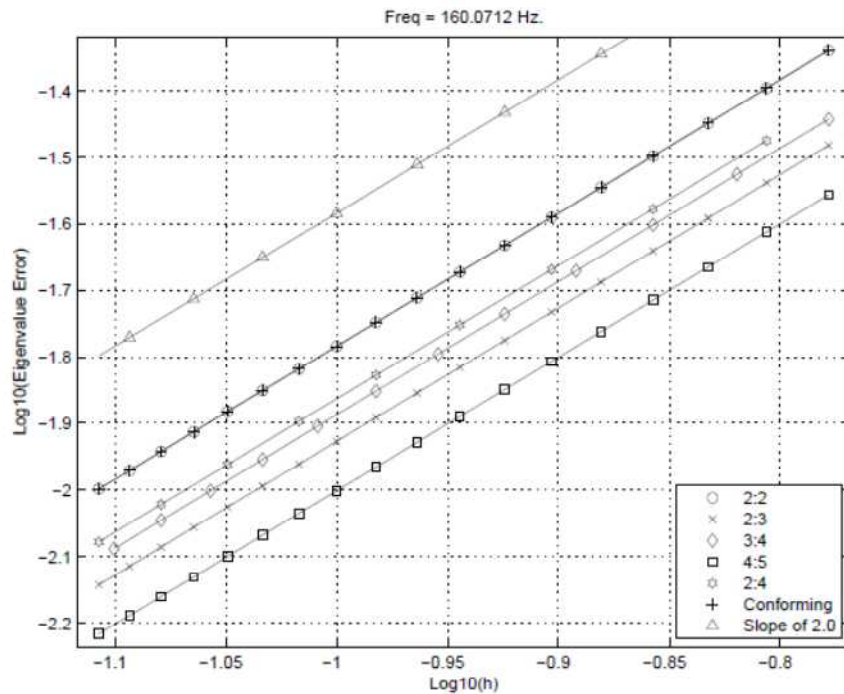
- **Two-fluid tank filled with water and castor oil**
- **Assumed that no mixing occurs**

We compare the results using 3 methods:

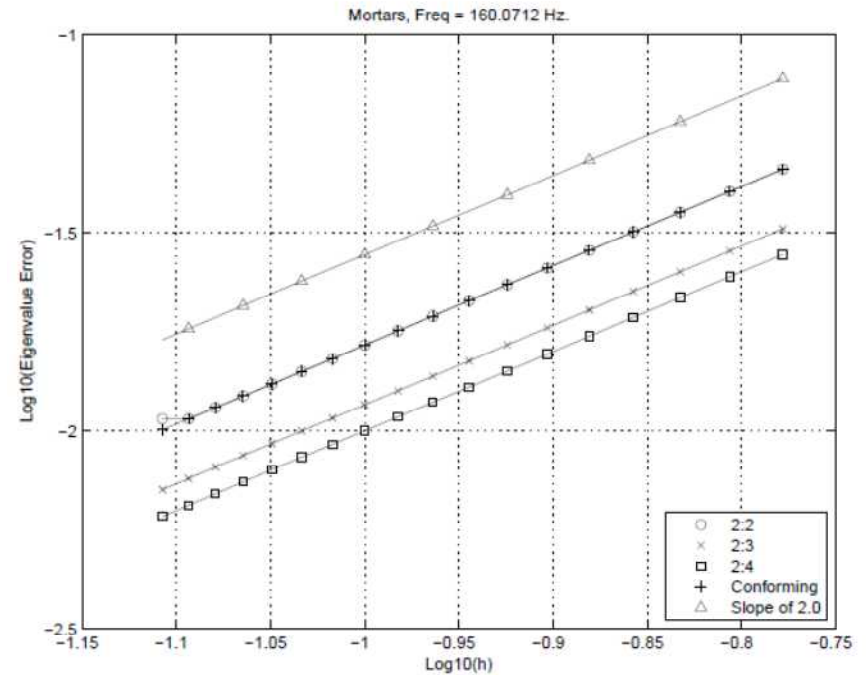
- **Conforming meshes**
- **Nonconforming meshes with ghost nodes and classical MPCs**
- **Nonconforming meshes with ghost nodes and mortar constraints**

# Convergence Results Two-Fluid System

160 Hz



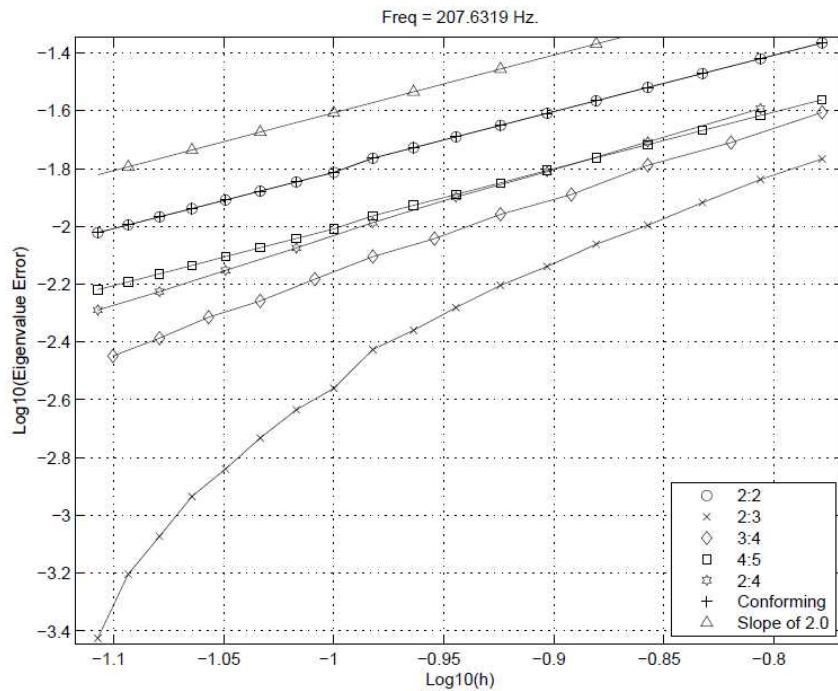
Classical MPCs



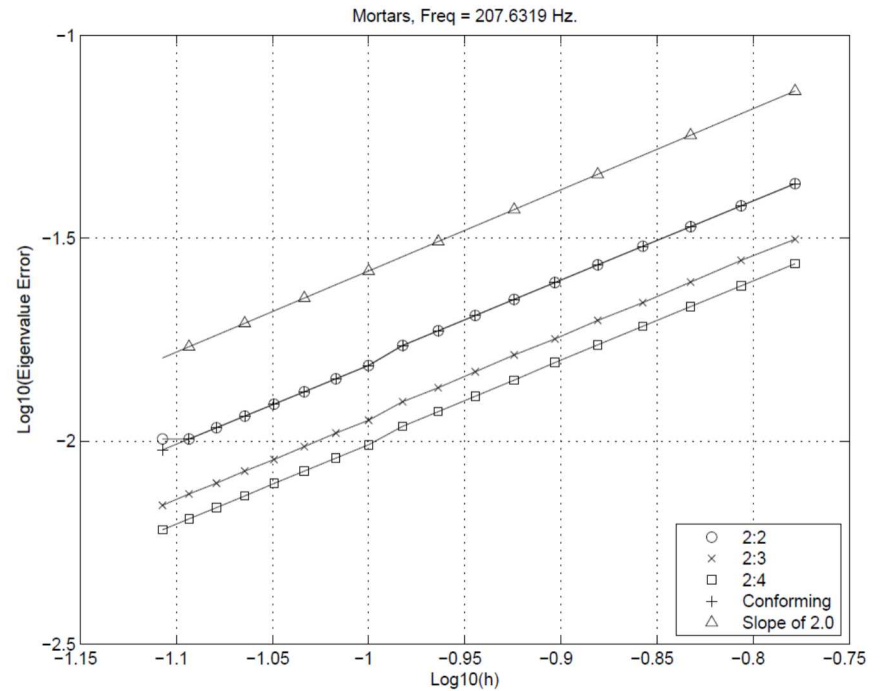
Mortar method

# Convergence Results Two-Fluid System

207 Hz



Classical MPCs

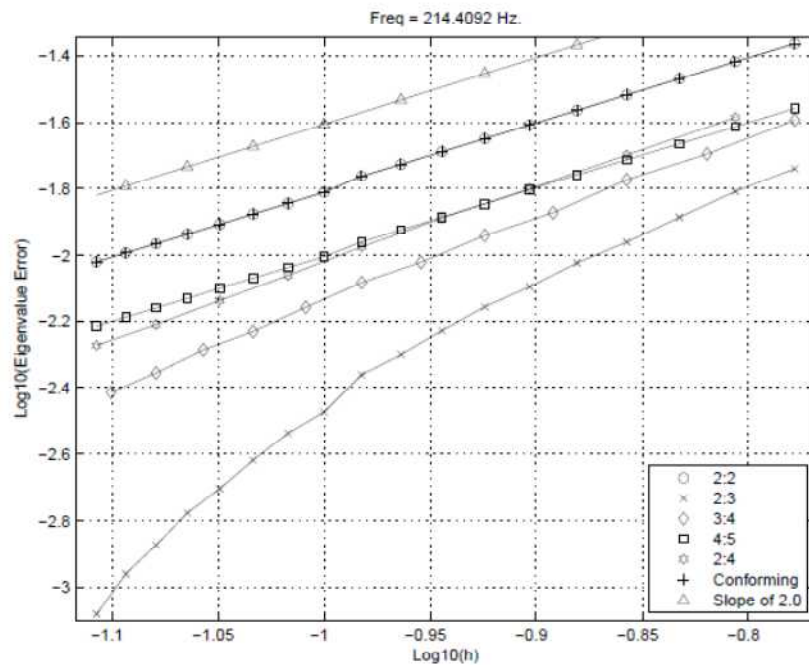


Mortar method

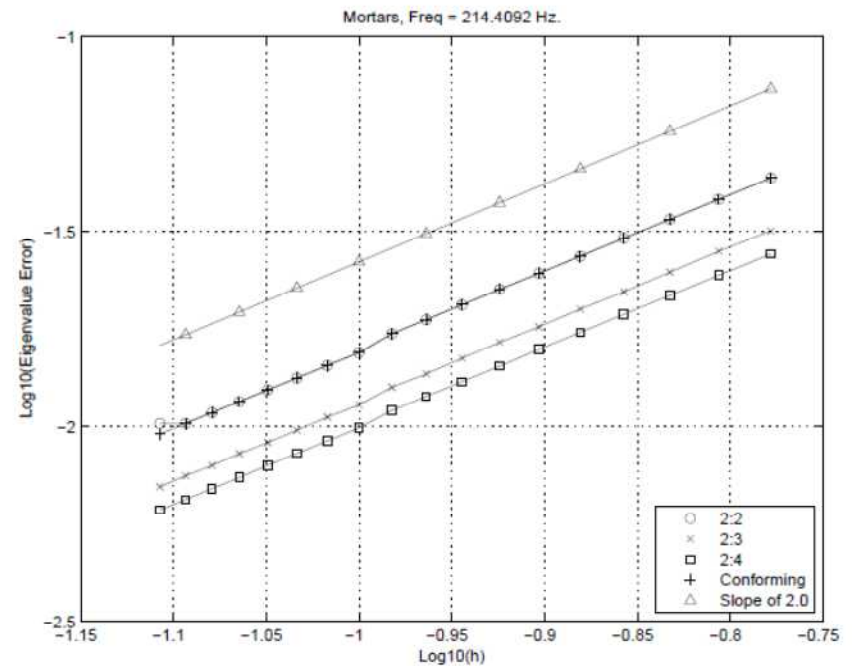


# Convergence Results for Two-Fluid System

## 214 Hz



Classical MPCs



Mortar method



# Observations and Conclusions

- **Both Classical MPCs and mortar method demonstrate convergence for both acoustic and structural acoustic mesh tying**
- **Mortar method appears to converge faster on cube-in-cube transient structural acoustic problem**
- **For two-fluid system, mortar method appears to give convergence rates that are consistent with theory**
- **For two-fluid system, classical MPCs show consistent convergent rates for some modes, and inconsistent rates for other modes**