

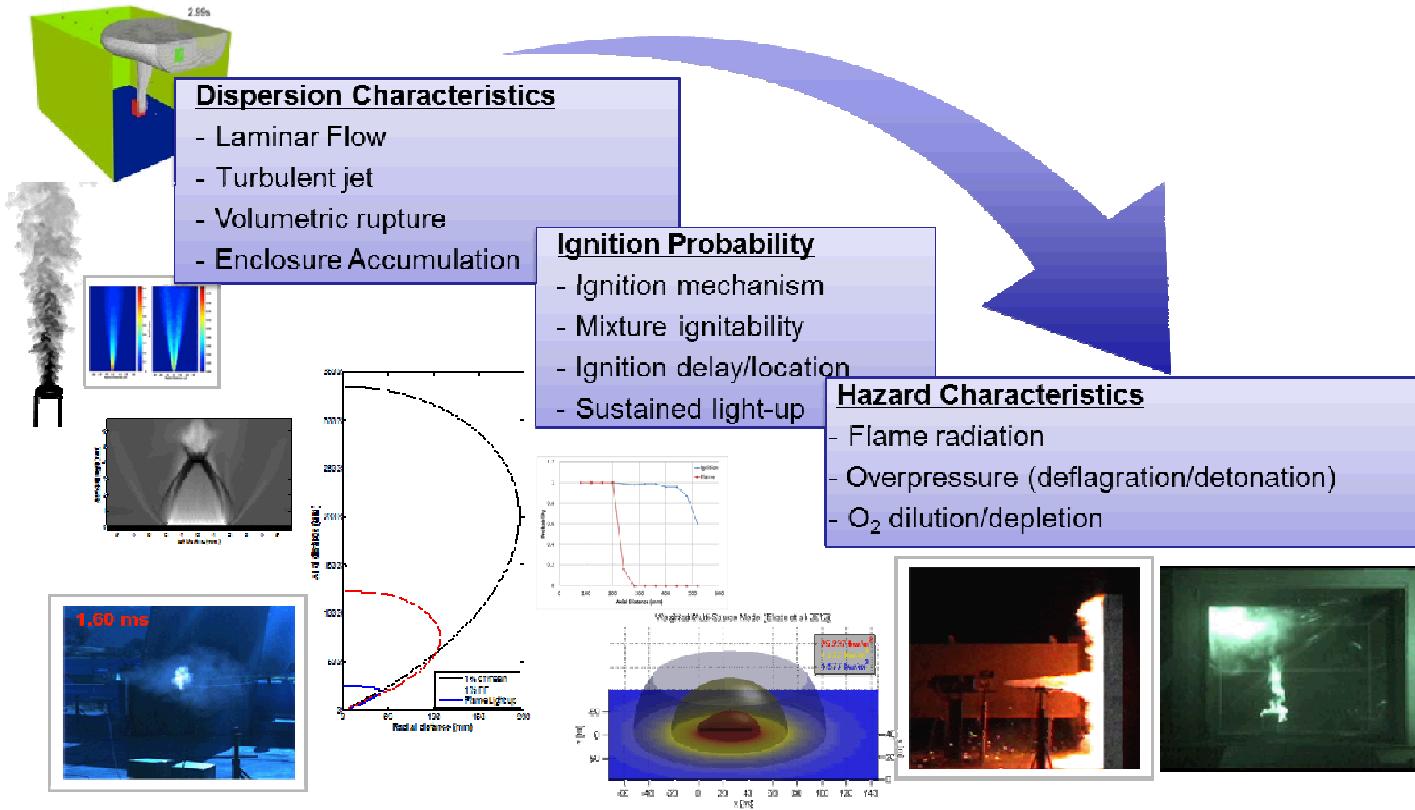
SNL Behavior Modeling — Research path & planned LH2 Experiments

Isaac Ekoto

Ethan Hecht

Chris San Marchi

Approach: Use high-fidelity data to develop & validate simplified models that augment quantitative risk modeling.



$$Risk \propto \sum_{i,j,k} P(\text{Release}_i) P(\text{Ignition}_j | \text{Release}_i) P(\text{Hazard}_k | \text{Ignition}_j \cap \text{Release}_i) P(\text{Harm} | \text{Hazard}_k)$$

Terms in red obtained from physical behavior models, while terms in black are based on scenario frequencies and probit harm models.

FY14 Accomplishment: Overpressure model development

$$\Delta p = p_0 \left\{ \left[\frac{V_T + V_{H2}}{V_T} \frac{V_T + V_{stoich}(\sigma - 1)}{V_T} \right]^\gamma - 1 \right\}$$

Bauwens & Dorofeev, ICHS, 2013.

p_0 : Ambient pressure
 V_T : Facility volume
 V_{H2} : Expanded volume of pure H₂
 V_{stoich} : Stoichiometric consumed H₂ volume
 σ : Stoichiometric H₂ expansion ratio
 γ : Air specific heat ratio (1.4)

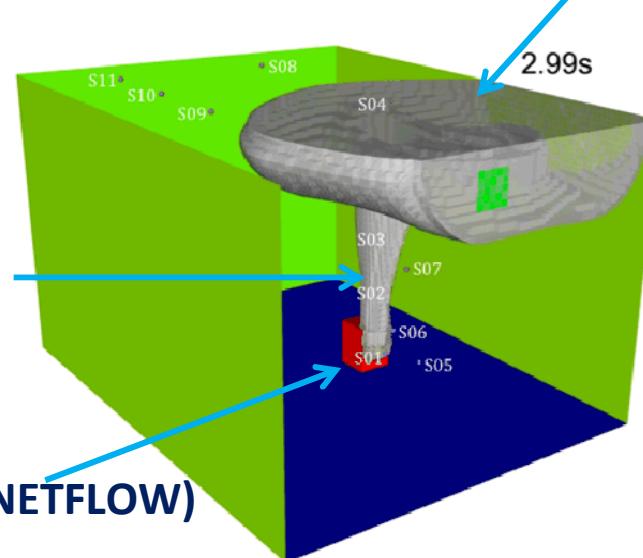
H₂ indoor refueling experiments & modeling

Ekoto et al. IJHE 2012

Houf et al. IJHE 2013

SNL H₂ Jet/Plume Model

Houf & Schefer, IJHE 2008

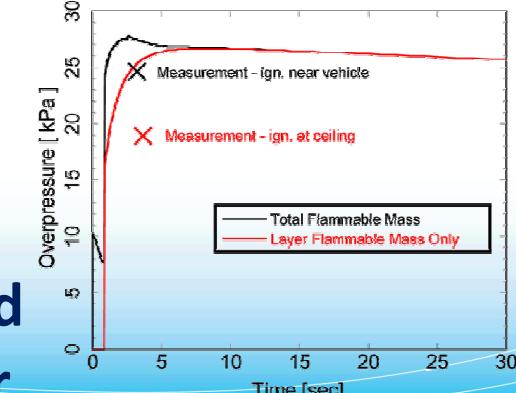
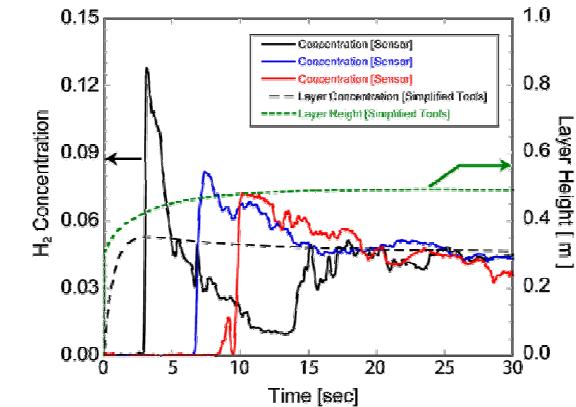


SNL Network Flow Model (NETFLOW)

Winters, SAND 2001-8422

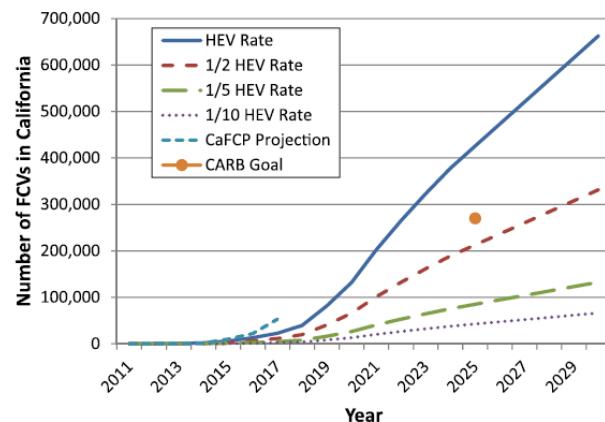
H₂ Layer Accumulation Model

Lowesmith et al. IJHE 2009



Exercise demonstrates how previous reduced-order model development work can be leveraged to quickly & accurately predict complex behavior.

Economic analyses for a proposed CA network of 68 H₂ fuel stations suggests LH₂ has long-term benefit over GH₂.



Brown et al., IJHE, 2013



	Total financial support required [\$MM]	Total potential profit [\$MM]	Weighted average cots of H ₂ [2011 \$/kg]	Onset of positive cash flow [year]	Total H ₂ dispensed [MMkg]	ROSP
LH ₂ Only						
100% of Historic HEV Sales	42.7	543.1	6.81	2017	155	11.72
50% of Historic HEV Sales	60.3	488.0	7.16	2018	141	7.09
20% of Historic HEV Sales	84.2	366.6	8.07	2019	112	3.35
10% of Historic HEV Sales	106.1	168.1	10.73	2021	67	0.58
GH ₂ Only						
100% of Historic HEV Sales	28.0	230.5	8.13	2016	87	7.23
50% of Historic HEV Sales	39.8	205.2	8.57	2018	79	4.16
20% of Historic HEV Sales	56.8	164.2	9.45	2019	66	1.89
10% of Historic HEV Sales	70.0	132.4	10.39	2020	57	0.89

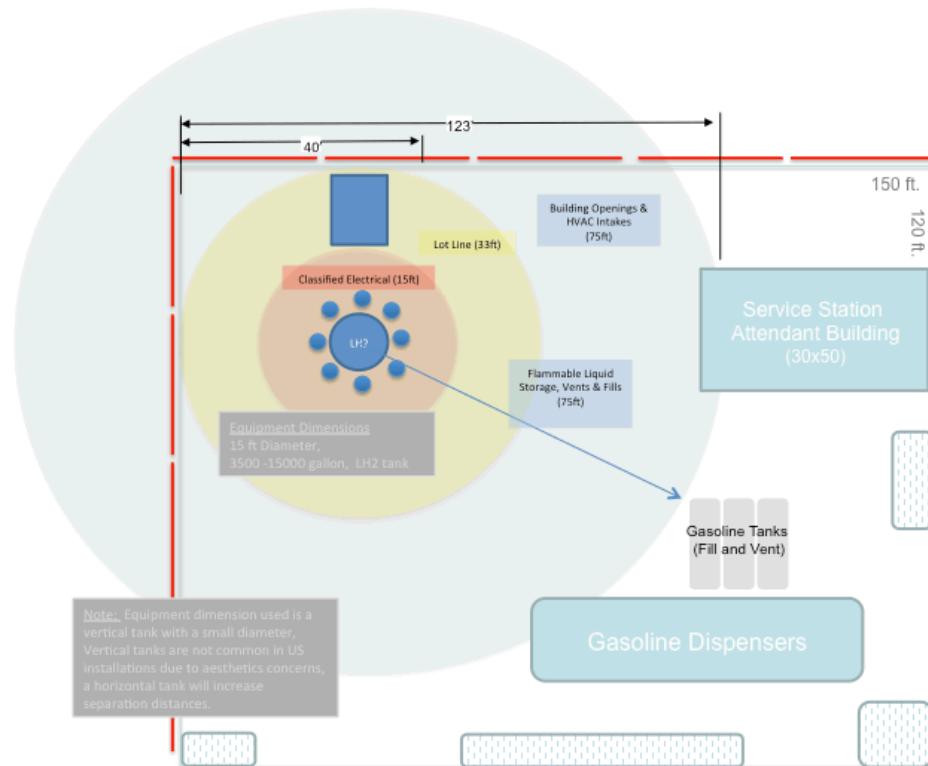
Analysis does not consider feasibility based on safety requirements.

Survey of California Energy Commission preferred stations suggest LH2 separation distances would be prohibitive.

Separation Distances (NFPA 2-2011) and Areas required for two station concepts (critical distances and areas emphasized)		
Fueling System Description	GH2	LH2
GH2: 12,500psi storage, 100kg, 0.4"ID tubing with a barrier wall LH2: 3500-15000 gallon (910-1300kg) with barrier wall and insulation		
Lot lines (ft)	24	33
Public Streets, Alleys (ft)	24	33
Parking (public assembly) (ft)	13	75
Buildings (sprinkled, fire rated) (ft)	10	5
Building Openings or air intakes (ft)	24	75
Flammable and Combustible liquid storage, vents or fill ports (ft)	10	50
Parking from fill connections on bulk storage (ft)	13	25
Class 1 Div. 2 area diameter (ft)	15	15
Max Bulk Storage Dimensions with Sep. Distances (ft)	78	123
Min Bulk Storage Dimension with Sep. Distances (ft)	68	123
Max Bulk Storage Equipment Dimension with lot lines (ft)	54	40
Min Bulk Storage Equipment Dimension with lot lines (ft)	49	40
Reference Bulk Storage Equipment Area with lot lines (sqft)	2646	1600
Reference Storage Area with Sep. Distances (sqft)	5304	15129
Note: Add 5 feet for vehicle protection on vehicle facing sides of equipment		

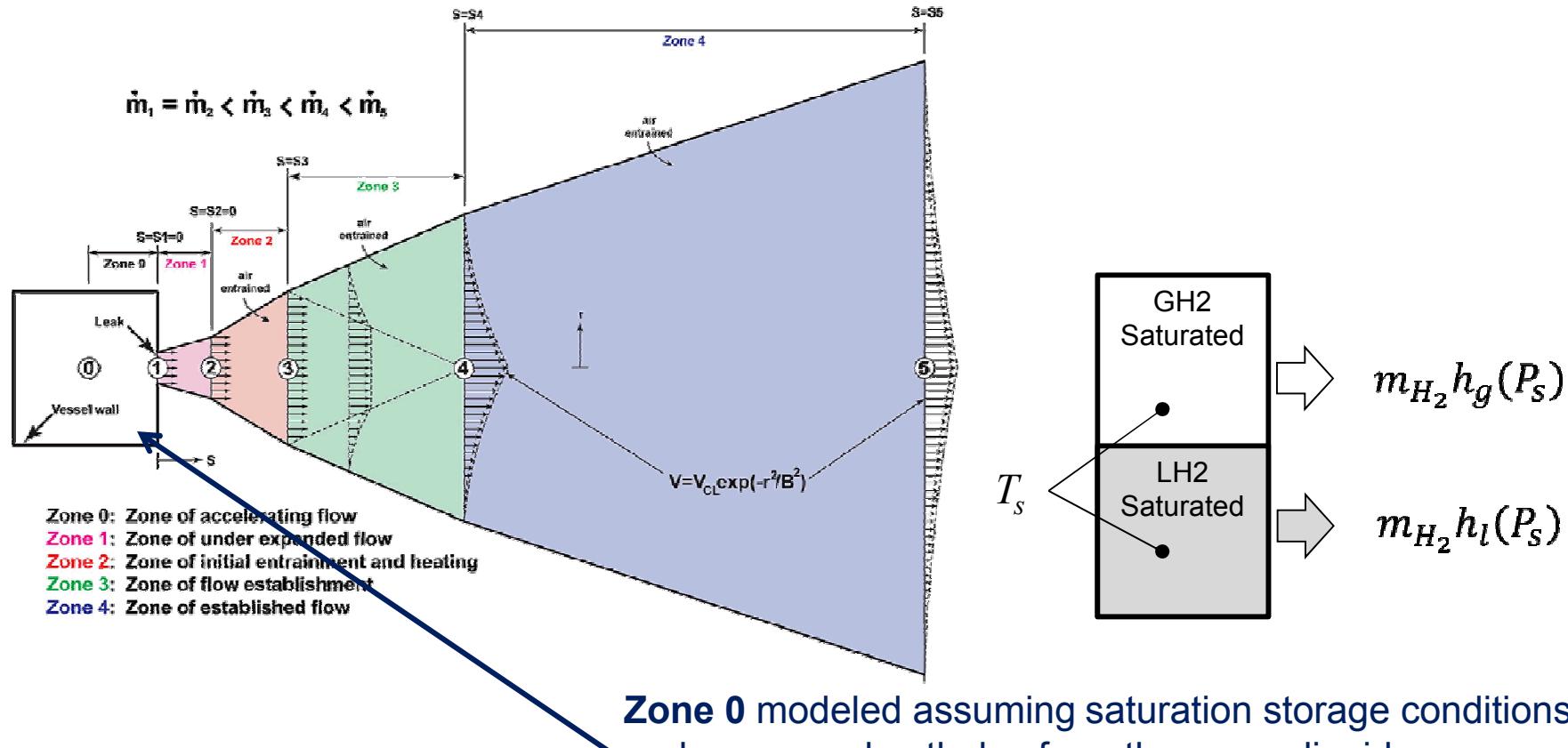
Harris, SAND-2014-XXXX

Of 70 stations surveyed (out of 343), none met the NFPA 2 Ch. 6 separation distance requirements.



Improved LH2 modeling needed to reduce separation distances and increase the viability of risk informed certification (e.g., NFPA 2 Ch. 5).

SNL conceptual model for LH₂ releases has already been developed – FY09



Winters, SAND Report 2009-0035

Winters & Houf, IJHE, 2011

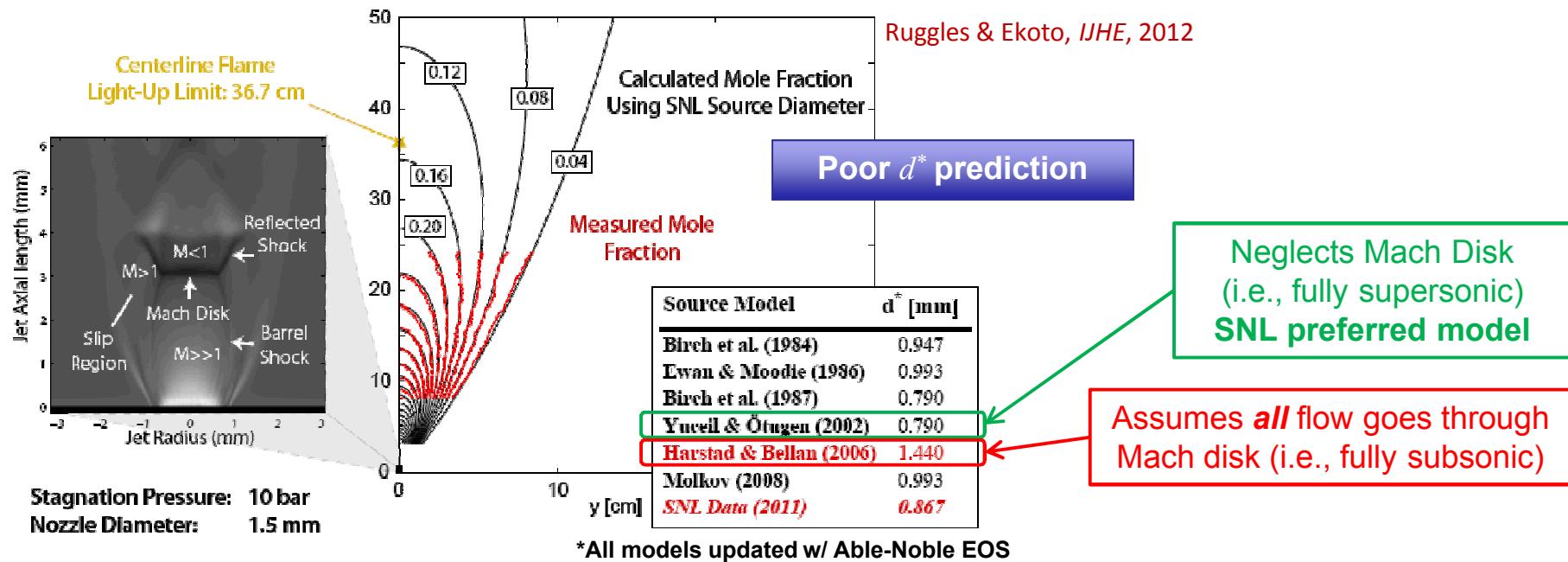
Houf & Winters, IJHE, 2013

Winters, SAND 2001-8422

SNL Network Flow Analysis Code (NETFLOW) used to model internal conditions in piping, valve, and tanks.

Pseudo source models are used to account for choked flow behavior (Zone 1).

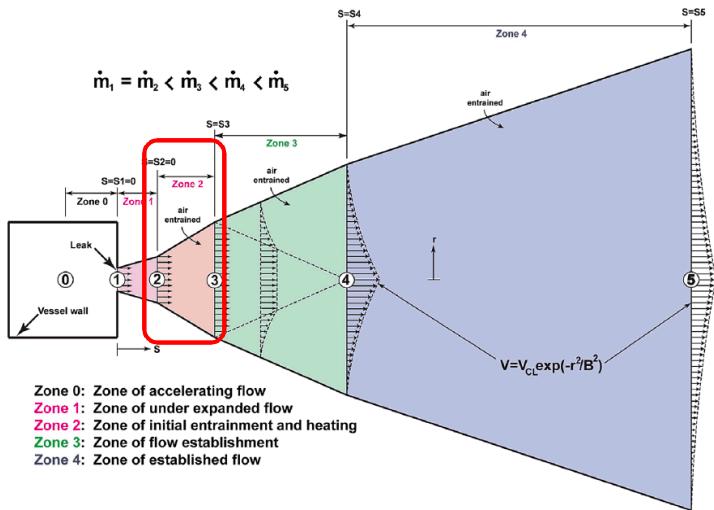
Several source models have been developed to predict the mass weighted effective diameter, (i.e., the critical scaling parameter): $d^* \equiv d_{eff} \sqrt{\rho_{eff}/\rho_{amb}}$



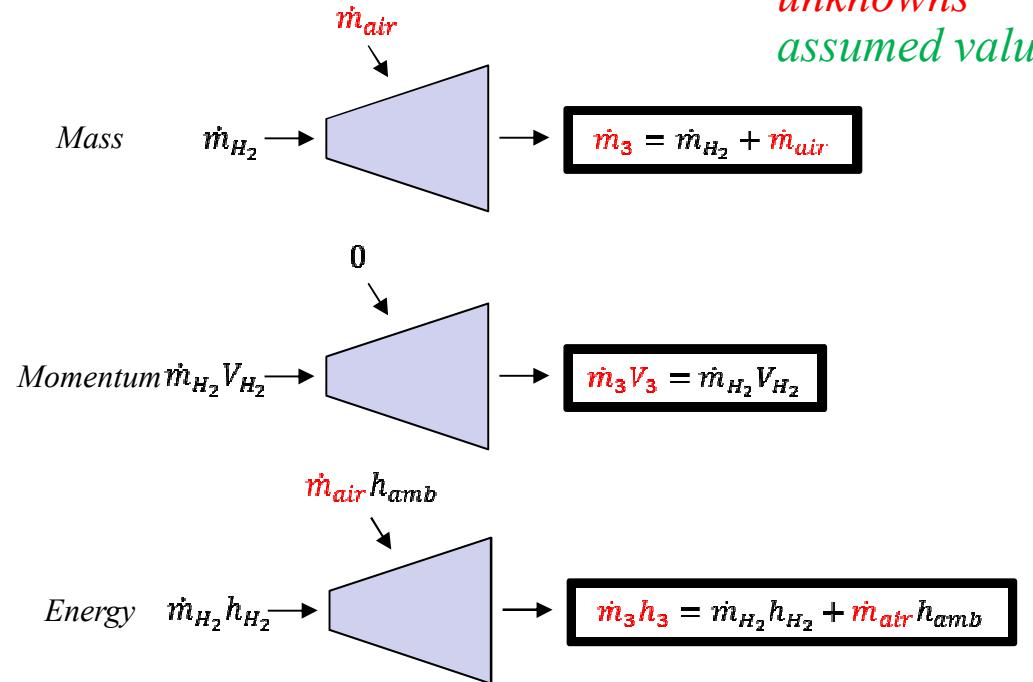
Reality is that fluid is split between the slip and Mach disk regions

Two-zone source model that accounts for the fluid split ratio between the slip region & Mach disk region is needed.

Plug flow assumption invoked for Zone 2.



Winters, SAND Report 2009-0035



State modeling by NIST H₂ EOS:

$$h_3 = f(Y_{H_2,3}, p_{amb}, T_3)$$

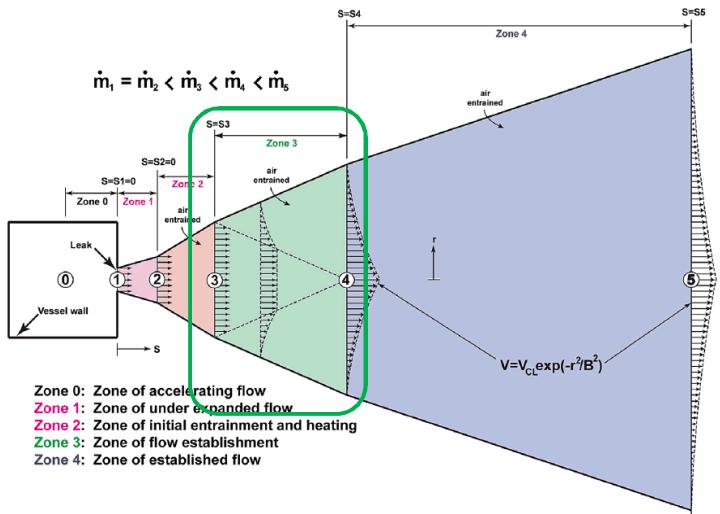
Species conservation used to close system of equations:

$$\dot{m}_{air} = \dot{m}_{H_2} \frac{1 - Y_{H_2,3}}{Y_{H_2,3}}$$

Turbulent jet entrainment rate used to estimate zone length:

$$E_{mom} \equiv \frac{1}{\rho_{amb}} \frac{d\dot{m}}{dS} \approx \frac{1}{\rho_{amb}} \frac{\dot{m}_{air}}{S_3} \Rightarrow S_3 = \frac{\dot{m}_{air}}{E_{mom} \rho_{amb}}, \text{ where } E_{mom} = \alpha_m \left(\frac{\pi D_{H_2}^2 \rho_{H_2} V_{H_2}^2}{4 \rho_{amb}} \right)^{1/2}$$

Zone 3 treated as discrete region w/ boundary conditions specified from self-similar profiles at Zone 4.



Winters, SAND Report 2009-0035

unknowns
assumed value

$$V_{CL,4} = V_3$$

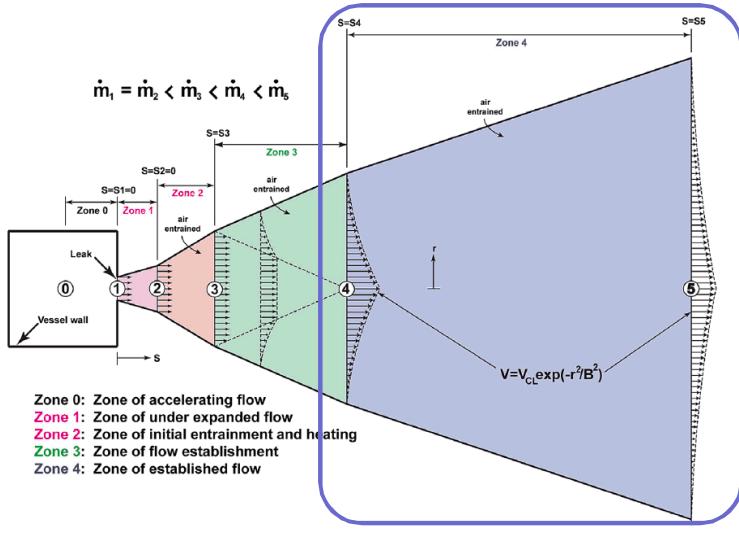
Mass
Momentum

$$\rho_3 \frac{D_3^2}{4} = B_4^2 \left[\rho_{amb} - \frac{\lambda^2}{\lambda^2 + 1} (\rho_{amb} - \rho_{CL,4}) \right]$$

$$(\rho_{amb} - \rho_3) \frac{D_3^2}{4} = B_4^2 \left[\frac{\rho_{amb}}{2} - \frac{\lambda^2}{2\lambda^2 + 1} (\rho_{amb} - \rho_{CL,4}) \right]$$

$\underbrace{\phantom{\rho_3 \frac{D_3^2}{4}}}_{S_3}$ $\underbrace{\phantom{(\rho_{amb} - \rho_3) \frac{D_3^2}{4}}}_{S_4}$

Zone 4 modeled with previous SNL 1D integral jet/plume models that invoke self-similarity – FY08.



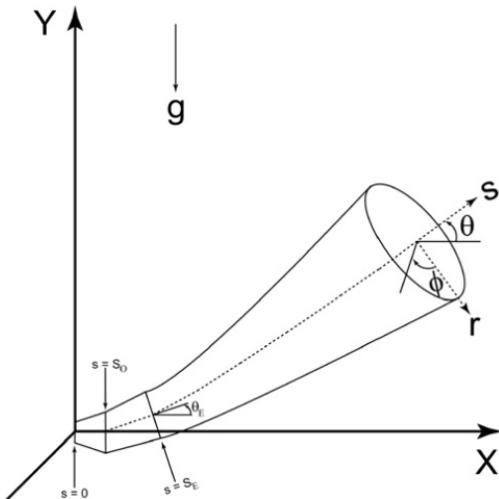
Houf & Schefer, IJHE, 2008

$$\text{Mass: } \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V r dr d\phi = \rho_{amb} E$$

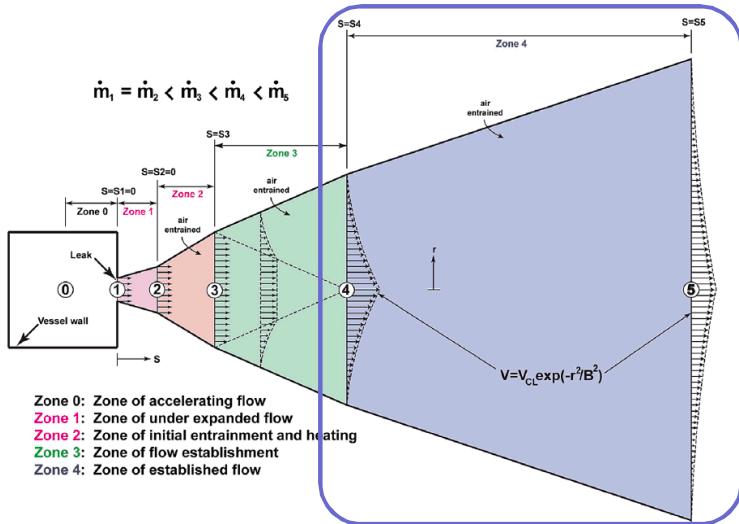
$$\text{x-Mom: } \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V^2 \cos \theta r dr d\phi = 0$$

$$\text{y-Mom: } \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V^2 \sin \theta r dr d\phi = \int_0^{2\pi} \int_0^{\infty} (\rho_{amb} - \rho) g r dr d\phi$$

$$\text{Species: } \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V Y r dr d\phi = 0$$

$$\text{Energy: } \frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V (h - h_{amb}) r dr d\phi = 0$$


Zone 4 modeled using previously developed SNL integral jet/plume models – FY08



Zone 0: Zone of accelerating flow
Zone 1: Zone of under expanded flow
Zone 2: Zone of initial entrainment and heating
Zone 3: Zone of flow establishment
Zone 4: Zone of established flow

Entrainment modeled to include buoyancy & momentum effects

F_{rL} : Jet Froude length

α_b : Buoyancy entrainment coefficient

α_m : Momentum entrainment coefficient

g : Gravity constant

Houf & Schefer, IJHE, 2008

$$E_{mom} = \alpha_m \left(\frac{\pi D^2}{4} \frac{\rho V^2}{\rho_{amb}} \right)^{\frac{1}{2}}$$

$$E_{buoy} = \frac{\alpha_b}{F_{rL}} (2\pi V_{CL} B) \sin \theta$$

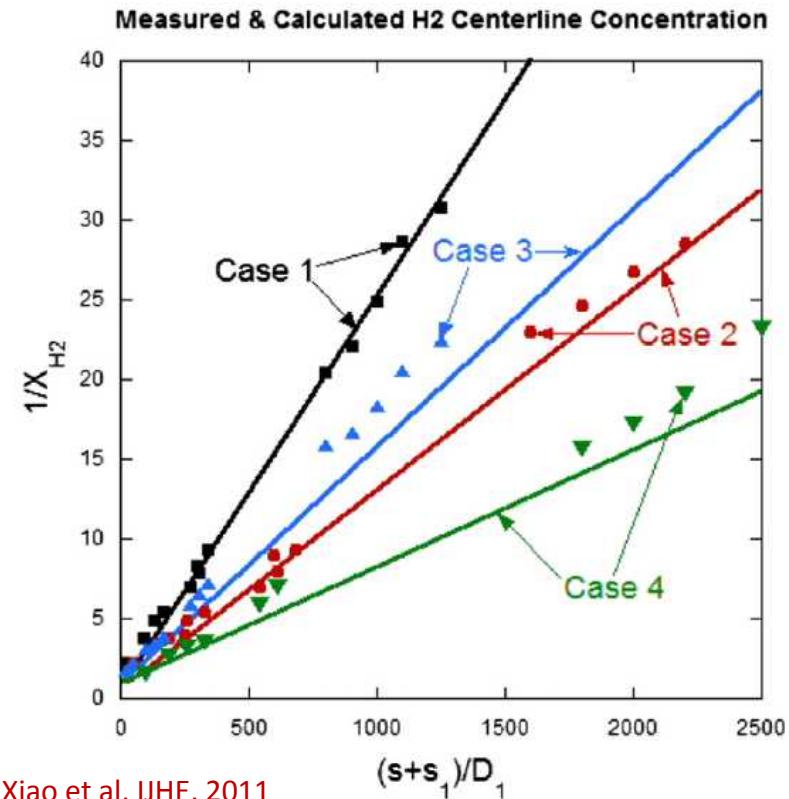
$$F_{rL} = \frac{V_{CL}^2 \rho_{exit}}{g B (\rho_{amb} - \rho_{CL})}$$

$$\begin{aligned} \text{Mass} \quad & \frac{d}{dS} \left\{ V_{CL} B^2 \left[\rho_{amb} - \frac{\lambda^2}{\lambda^2 + 1} (\rho_{amb} - \rho_{CL}) \right] \right\} = \frac{\rho_{amb} (E_{mom} + E_{buoy})}{\pi} \\ x\text{-Mom} \quad & \frac{d}{dS} \left\{ V_{CL}^2 B^2 \cos \theta \left[\rho_{amb} - \frac{2\lambda^2}{2\lambda^2 + 1} (\rho_{amb} - \rho_{CL}) \right] \right\} = 0 \\ y\text{-Mom} \quad & \frac{d}{dS} \left\{ V_{CL}^2 B^2 \sin \theta \left[\frac{\rho_{amb}}{2} - \frac{\lambda^2}{2\lambda^2 + 1} (\rho_{amb} - \rho_{CL}) \right] \right\} = (\rho_{amb} - \rho_{CL}) \lambda^2 B^2 \\ \text{Species} \quad & \frac{d}{dS} \{ V_{CL} B^2 (\rho_{amb} Y_{amb} - \rho_{CL} Y_{CL}) \} = 0 \\ \text{Energy} \quad & \frac{\partial}{\partial S} \left\{ \int_0^{\infty} \rho V [h(P_{amb}, \rho) - h_{amb}(P_{amb}, \rho_{amb})] r dr \right\} = 0 \end{aligned}$$

Solution possible via commonly available 1D ODE solvers.

Model results compare favorably to experiments from Karlsruhe Institute of Technology.

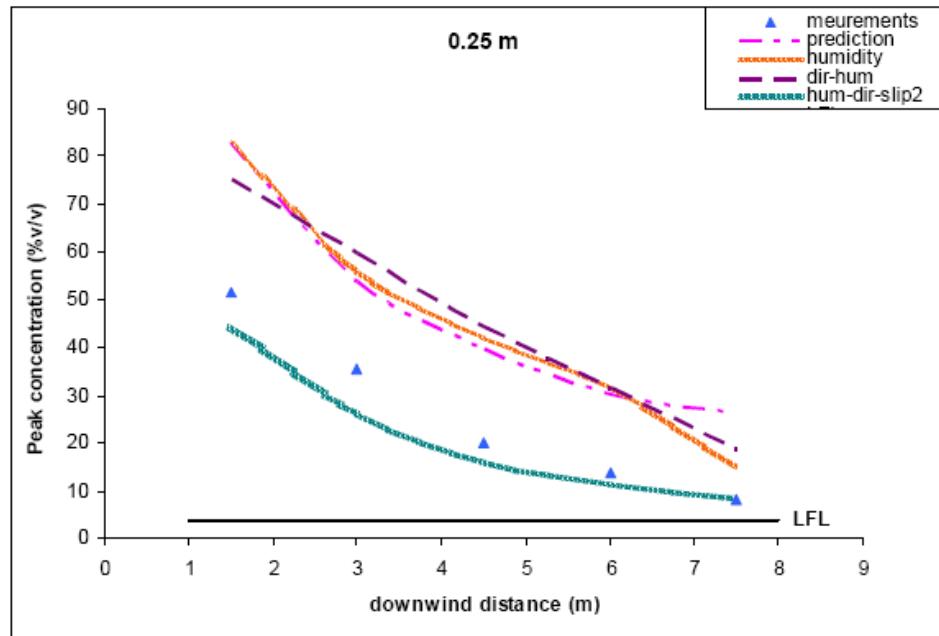
Case	Reservoir pressure [MPa]	Reservoir temperature [K]	Leak diameter [mm]
1	1.7	298	2
2	6.85	298	1
3	0.825	80	2
4	3.2	80	1



However, no well-controlled validation data is available at lower temperatures where multi-phase flows are expected (i.e., T < 77 K).

Multi-phase behavior—particularly for high-humidity conditions—is important.

Liquid and vapor phases have different velocities due to density differences — slip models have captured these effects in CFD simulations.



HSL Measurements: Sample probes
Hooker et al, ICHS, 2011

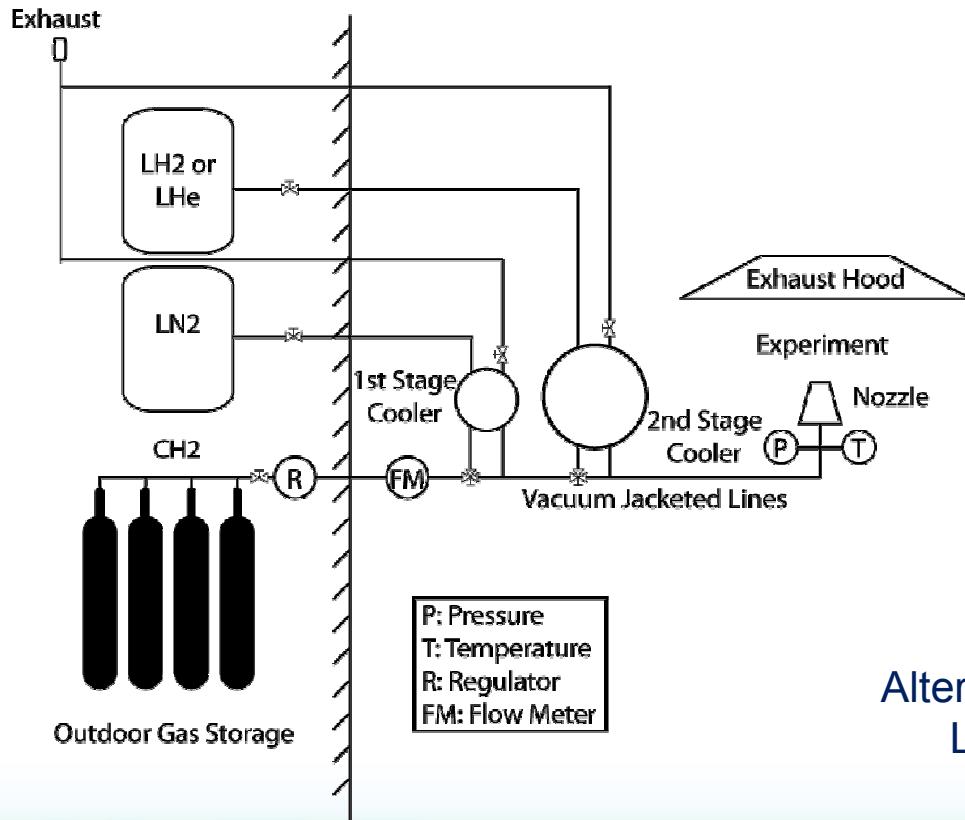
ADREA-HF CFD Simulations
Giannissi et al, ICHS, 2013

Substantial differences in model results suggest 2-phase effects cannot be neglected for LH₂ releases

Experiments had poor control of release and environmental boundary conditions, which are needed for suitable benchmark data.

Proposal is to build an LH₂ delivery system that can be integrated w/ existing SNL laboratory infrastructure.

Delivery system would generate ultra-cold vapor and liquid H₂ jets

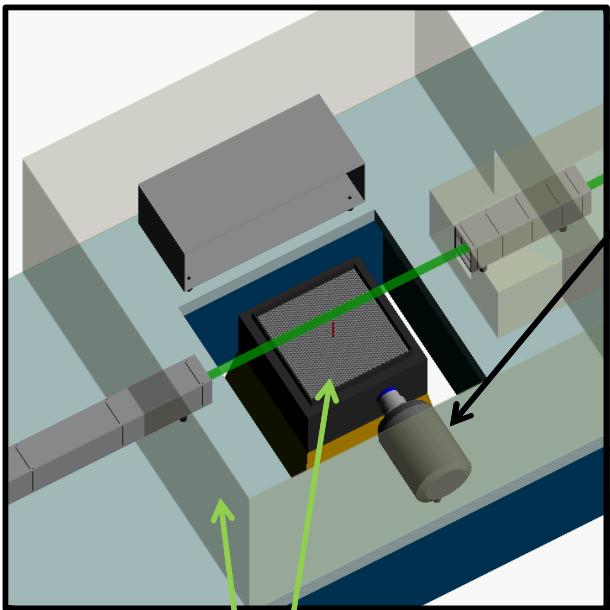


LH₂ or LHe generation via a commercial cryo-refrigerator

Alternatively, LH₂ could be transported from LLNL to SNL from an available LH₂ tank

System design, construction, & verification ~12 months, w/ another 6-12 months for measurements – system cost ~\$150K plus labor.

Scalar field to be measured via Rayleigh scatter imaging in established flow zone to validate LH2 release model.

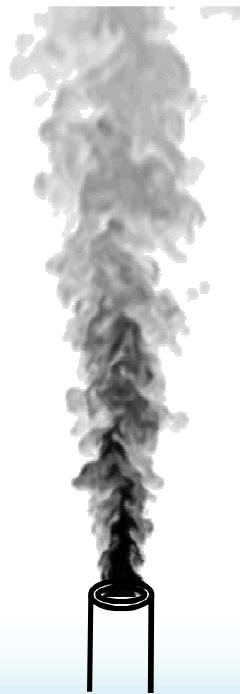


Air co-flow & barriers to minimize impact of room currents

PIXIS 400B low noise CCD Camera

- 2 x 2 binning for high signal-to-noise (~400:1)
- Multiple interrogation regions to image full jet
- Multiple images for converged statistics

Nd:YAG injection seeded laser
(1 J/pulse @ 532 nm)



Opportunity for additional upstream measurements using complementary Raman diagnostics in an adjacent lab.

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Backup Slides

Jet Flame Radiation Modeling

Three jet flame radiative heat flux model categories:

Single Point Source (SPS) models

Flame shape:

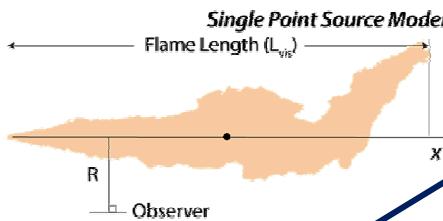
- Non-dimensional radiant power to estimate radiant load distribution

Radiant fraction models:

- Empirical function: temperature, composition, release rate, soot, residence time, heat release

Sivathanu & Gore, Combust Flame, 1993

Molina et al., Proc Combust Inst, 2007



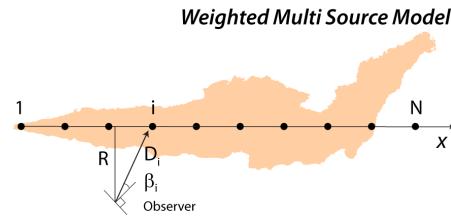
Flame shape:

- Weighted source emitters along flame centerline

De-Faveri et al., Hydrocarbon Processing, 1985

Hankinson & Lowesmith, Combust Flame, 2012

Multi Source Models (MSM)



Radiant fraction models:

- Same as SPS models

Flame shape:

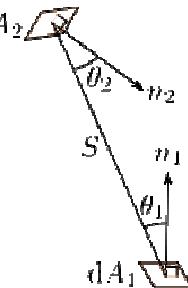
- Assumed flame shape (e.g., cone) w/ empirically tuned radiating surface
- Geometric View Factors to calculate radiation transfer
- Empirical wind/buoyancy corrections

Chamberlain, Chem Eng Res Des, 1987

Johnson et al., Process Safety Environ Prot, 1994

$$VF_{1 \rightarrow 2} = \frac{\cos \theta_1 \cos \theta_2}{\pi S^2}$$

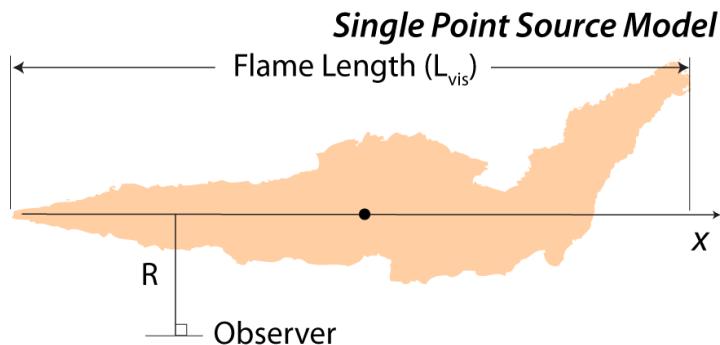
Single Surface models



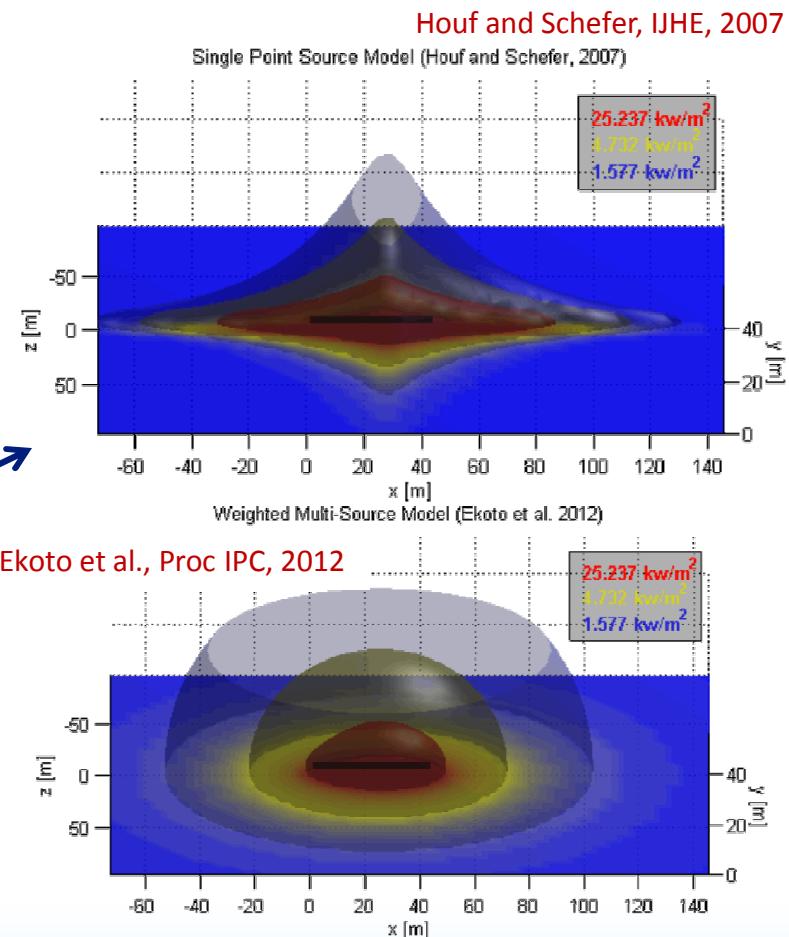
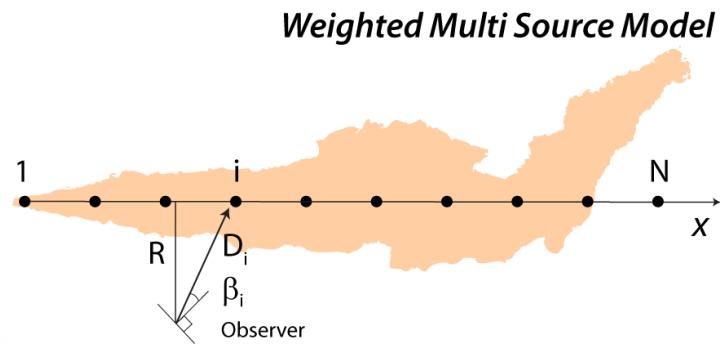
Previous Accomplishments

Large-scale flame data supplied by Air Products and Chemicals Inc.

d_j [mm]	[kg/s]	L_{vis} [m]	p_0 [barg]	T_0 [K]	T_{amb} [K]	p_{amb} [bar]
50.8	7.4	48.5	62.1	288	280	1.01



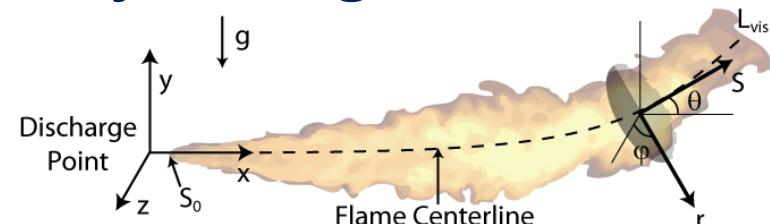
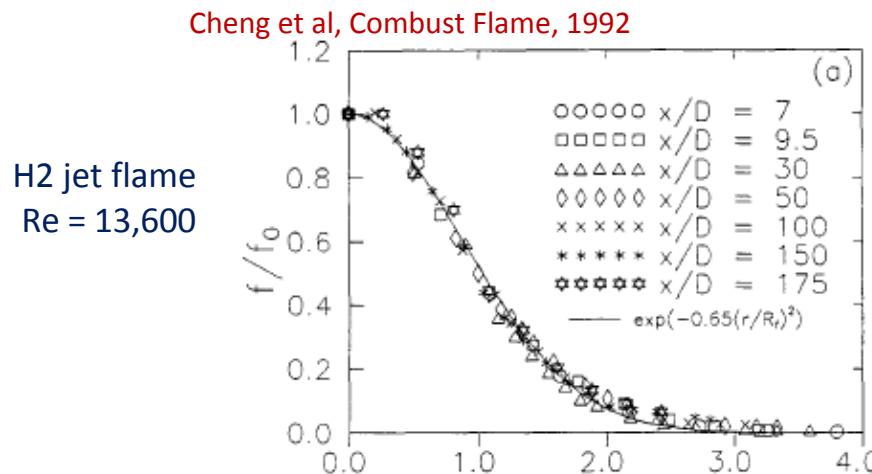
Old model used to inform NFPA 2/55



Improved radiative heat flux boundaries for more accurate harm & improved recommendations for reduced separation distances.

- Model can be improved with a better prediction of flame trajectory to better

Flame integral model is similar to 1D jet integral models



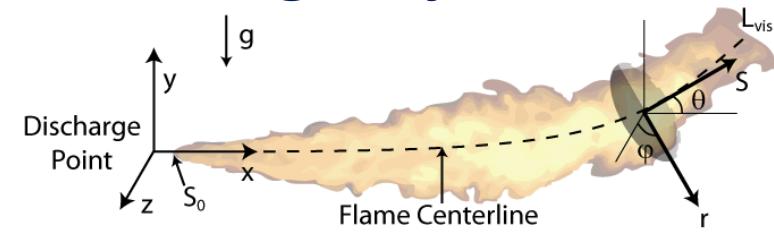
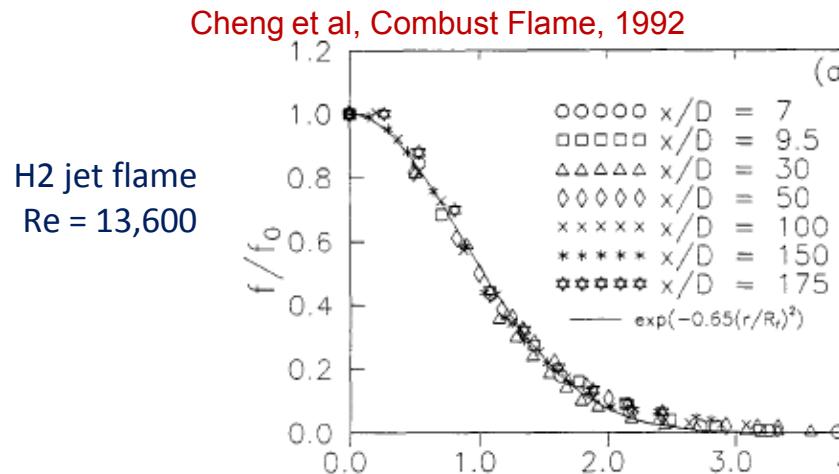
Species are no longer conserved & excess state variable profiles depend on the degree of chemical reaction

Self-similar, Gaussian mixture fraction profiles observed at all radials throughout the flame

$$\underbrace{\frac{\partial}{\partial S} \int_0^{2\pi} \int_0^{\infty} \rho V Y r dr d\phi}_{H_2 \text{ Species Conservation}} \text{ replaced by, } \underbrace{\frac{d}{dS} \int_0^{2\pi} \int_0^{\infty} \rho V f r dr d\phi = 0}_{\text{Mixture Fraction Conservation}} \text{ where, } f = Y_{H_2} + Y_{H_2O} \frac{MW_{H_2}}{MW_{H_2O}}$$

Mixture fraction is a conserved scalar that can replace H₂ mass fraction in the conservation equations

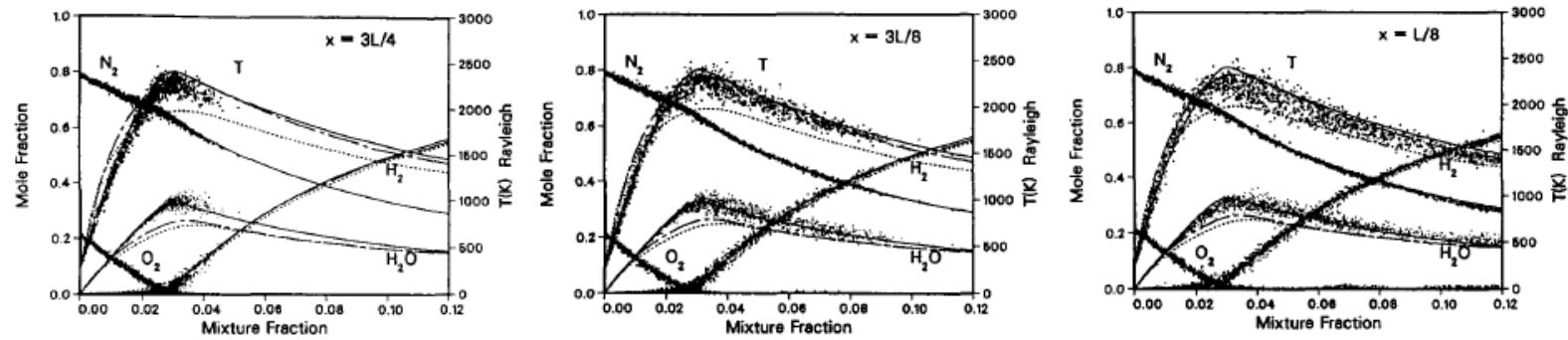
Flame integral model is similar to existing 1D jet models



Species are no longer conserved & **excess state variable profiles depend on the degree of chemical reaction**

Self-similar, Gaussian mixture fraction profiles observed at all radials throughout the flame

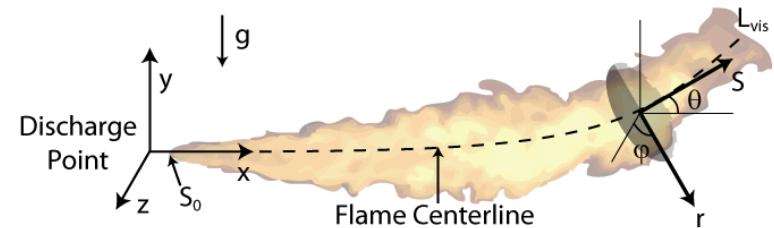
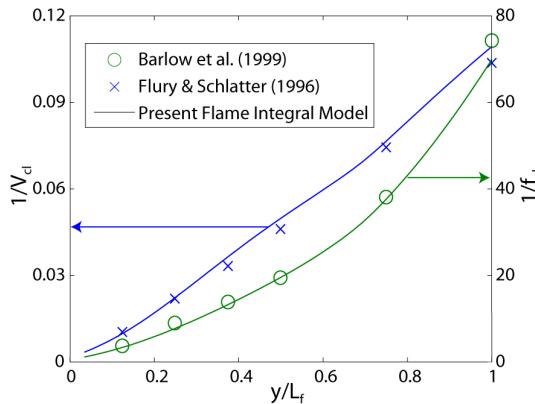
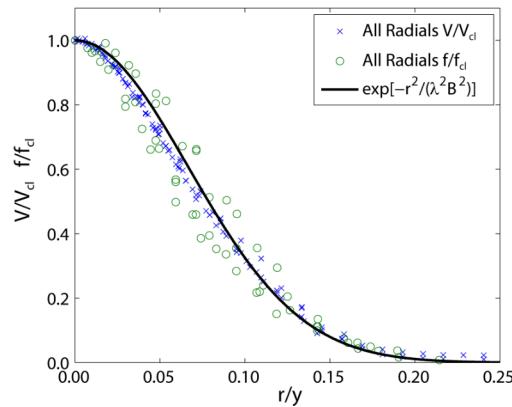
Barlow & Carter, Combust Flame, 1994



H₂ jet flame; Re = 10,000

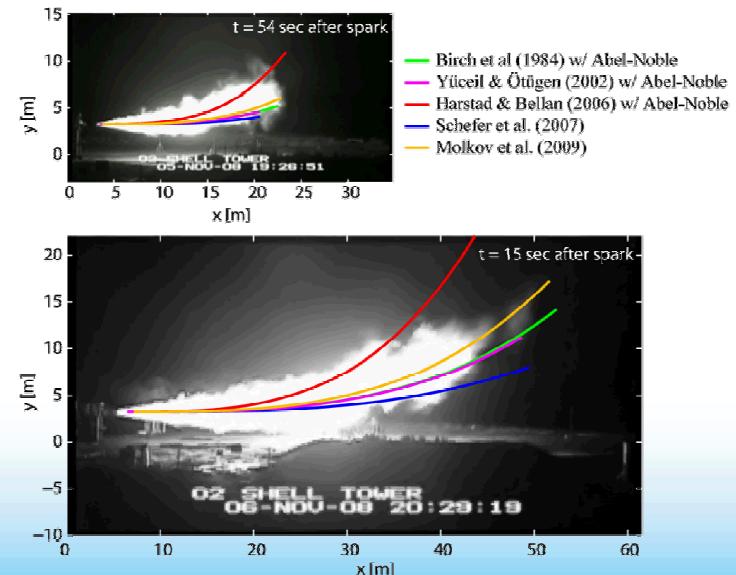
Composition/Temperature at most points was close to equilibrium solution — energy conservation can be neglected if equilibrium kinetics are assumed

Model performs well when applied to large-scale flames



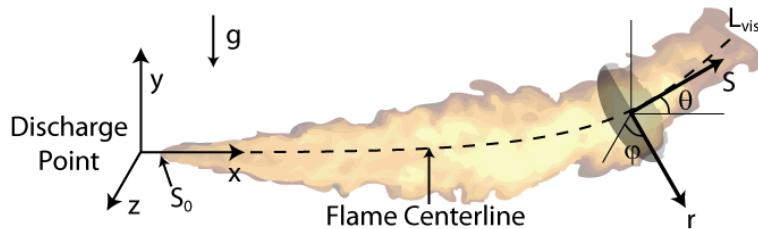
Entrainment rate coefficients & jet spreading ratios adjusted to match experimental data

Centerline trajectory results highly dependent on choked source model



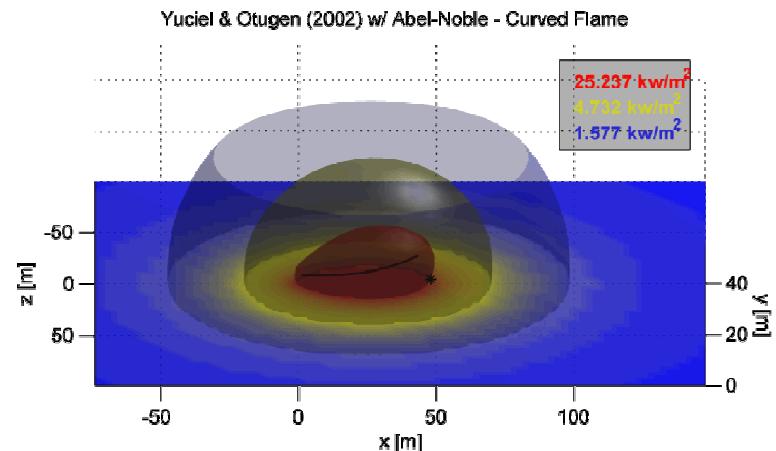
Integral flame model developed to improve downstream H₂ jet flame heat flux prediction.

Ekoto et al. ICHS 2013



Adjusts placement of radiative emitters for new multi-source model – FY12 accomplishment

d_j [mm]	\dot{m} [kg/s]	p_0 [barg]	T_0 [K]	RH	T_{amb} [K]	p_{amb} [mbar]	U_{wind} [m/s]	Wind dir [°]
52.5	7.4	62.1	287.8	94.5	280	1011	0.83	34.0



SNL preferred source model

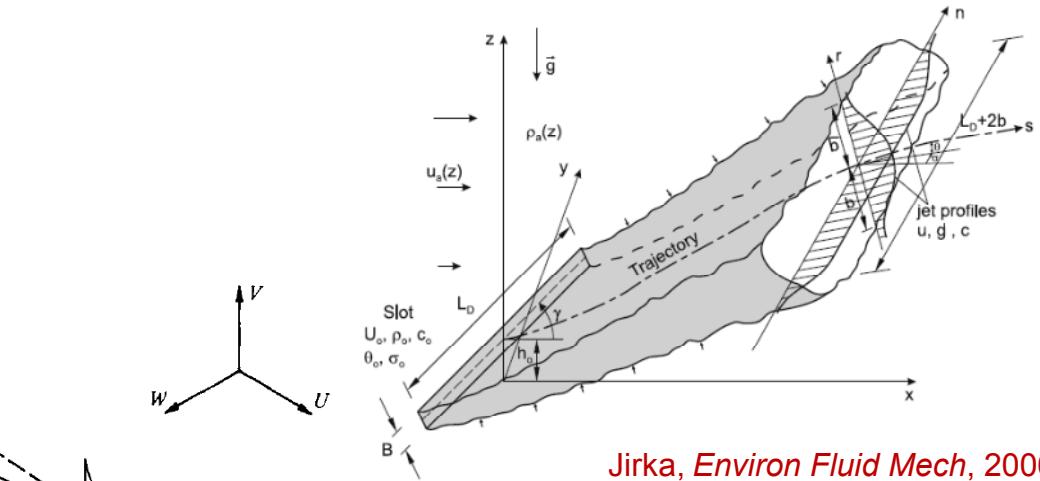
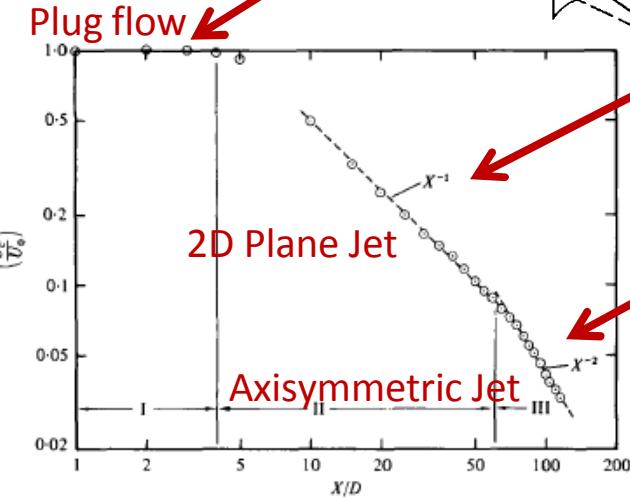
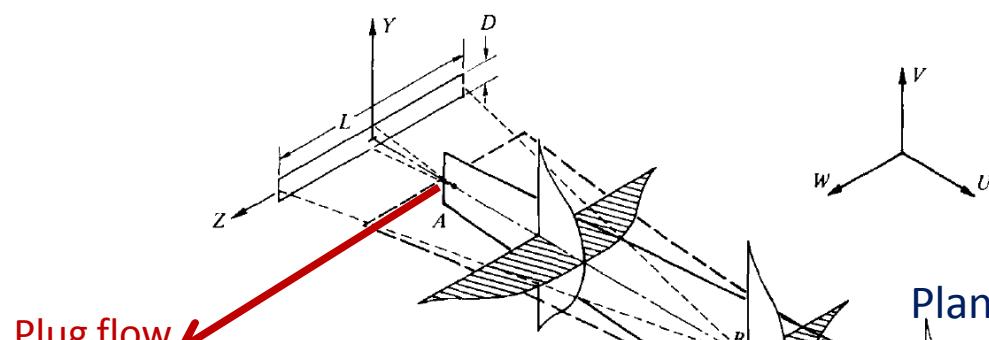


Notional Nozzle Model	L_f [m]	q_{rad} (Straight) [kW/m ²]	q_{rad} (Curved) [kW/m ²]
Measurement	45.9	–	23.9
Birch et al. (1984) w/ Abel-Noble	49.3	97.3	29.9
Yuceil & Otuken (2002) w/ Abel-Noble	44.6	34.8	23.8
Schefer et al. (2007)	44.6	34.8	28.1
Harstad & Bellan (2006) w/ Abel-Noble	52.7	189.8	13.2
Molkov et al. (2009)	49.9	113.2	25.6

Similar wind corrections in progress

Slot Nozzle Modeling

Unchoked slot jets have been thoroughly investigated



Jirka, *Environ Fluid Mech*, 2006

Planar jet integral model has been developed

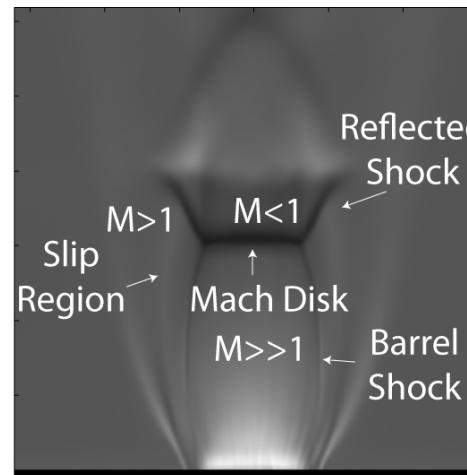
Krothapalli et al., *J Fluid Mech*, 1981

Distinct 2D region with inverse $\frac{1}{2}$ power centerline decay rate exists

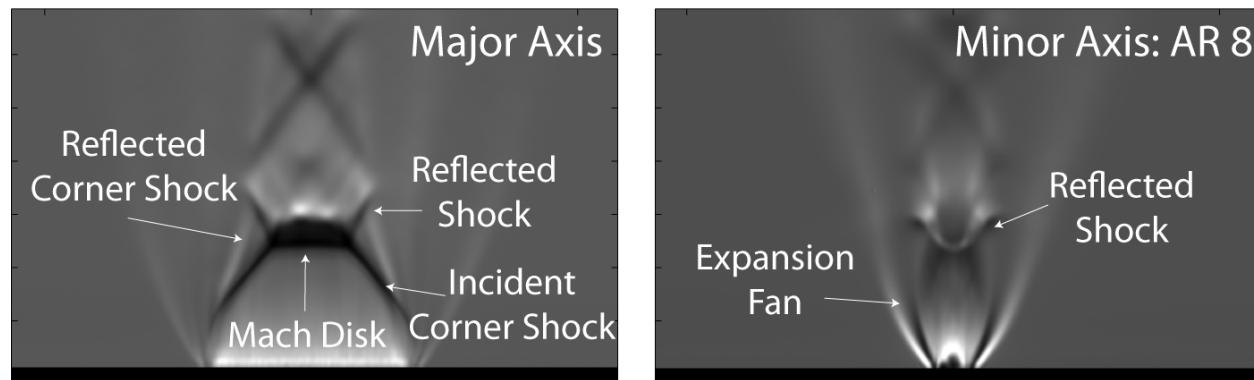
Unclear if models are applicable to choked slot jets

Close-up schlieren imaging reveals unique slot nozzle behavior

– FY13

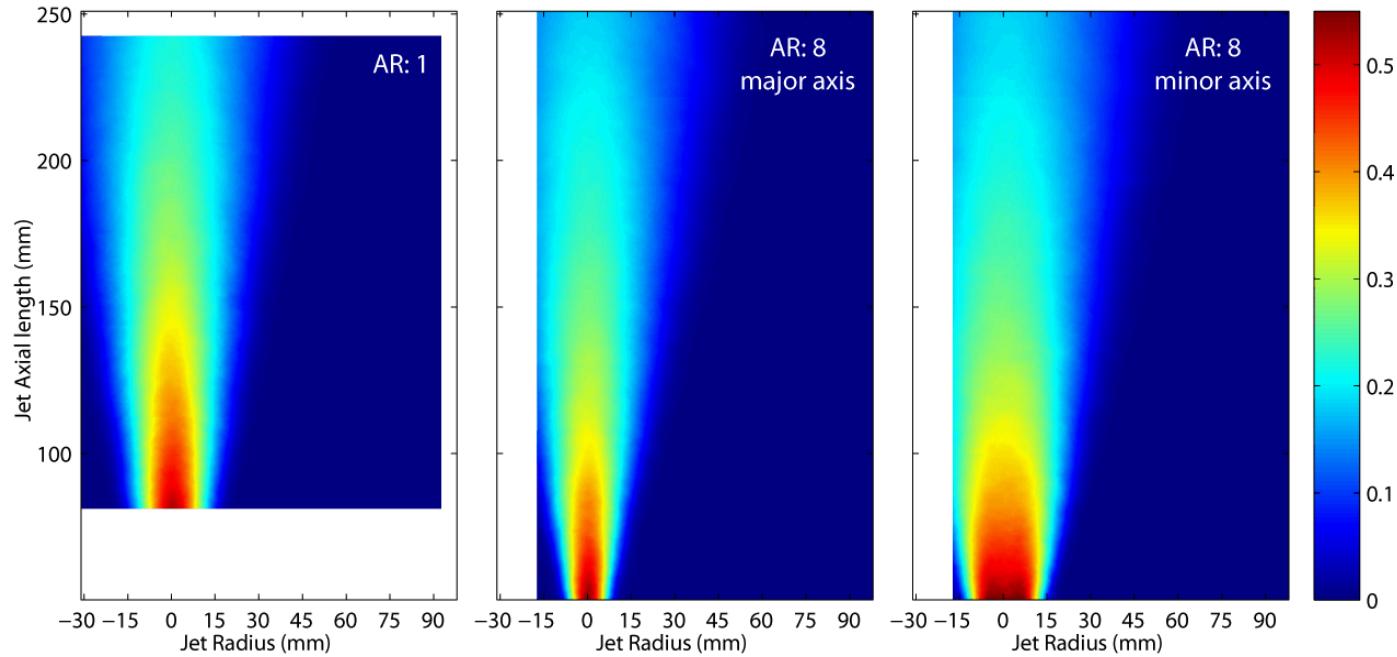


Strong, sharply converging incident corner shock is missing from the minor axis plane



Unclear if existing choked flows notional nozzle models predict correct effective diameters and densities

Mean mass fraction slot jet contours confirm axis switching in the scalar field



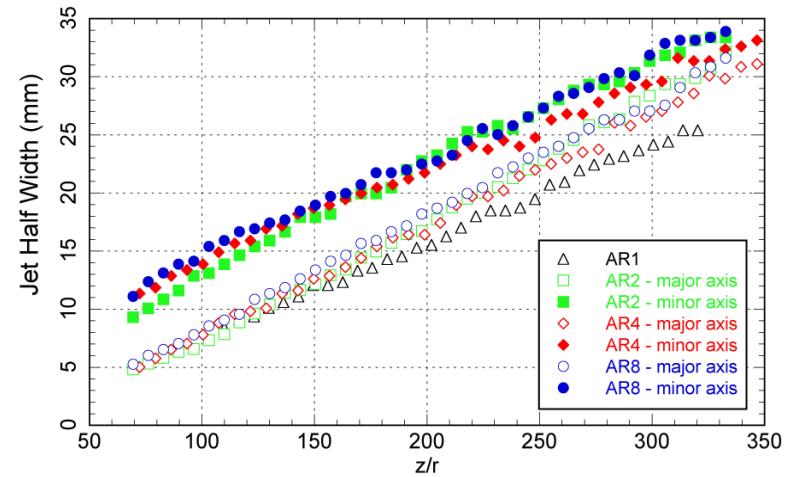
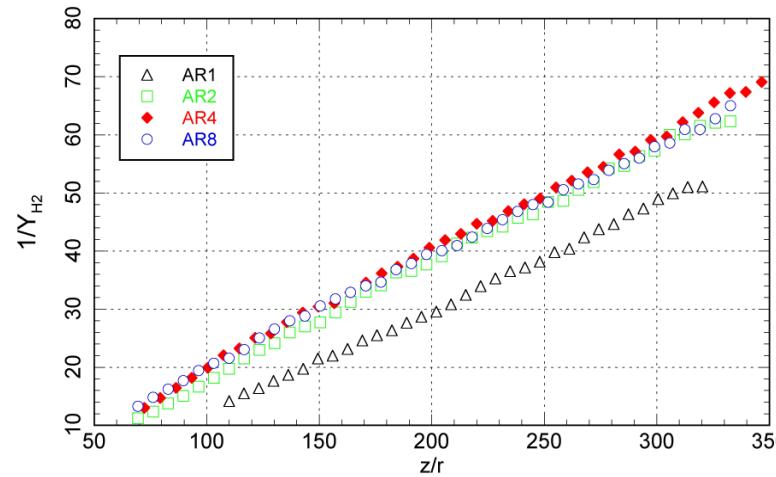
Elevated downstream mass fraction contours for the axisymmetric jet

$$\begin{aligned} D_{eq} &= 1.5 \text{ mm} \\ p_0 &= 10 \text{ bar} \end{aligned}$$

Concentration decay rates remained relatively linear throughout the measurement region

Planar decay region (half-power) not observed

- Upstream of interrogation region?



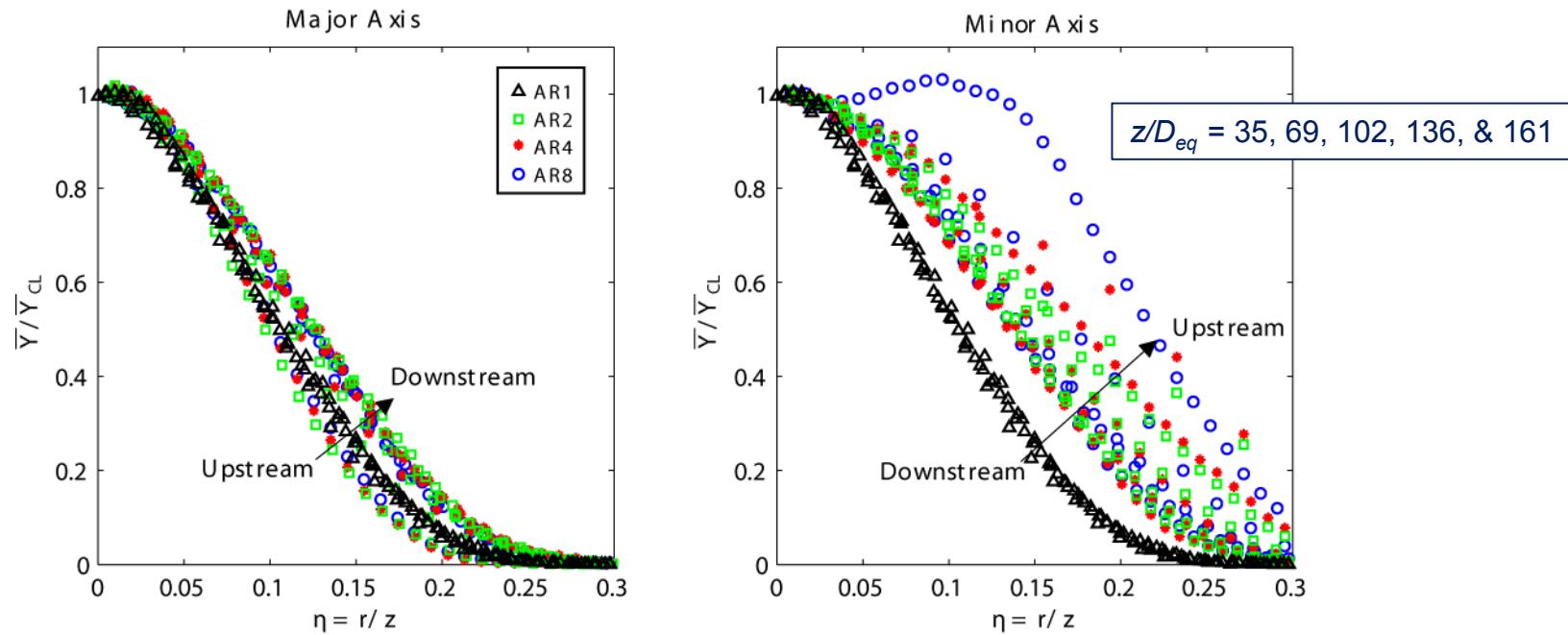
Major & minor axis jet half widths appear to converge

- Half widths larger than for corresponding axisymmetric jet
- Slightly non-linear growth rates – unclear when convergence occurs

Normalized concentration radial profiles along the major/minor axes do not collapse

Axisymmetric profiles collapsed to uniform curves as expected

Normalized profiles grew **wider along the major axis** and **narrower along the minor axis**



Minor axis peak H₂ **near-field** concentrations observed away from the centerline – not predicted by planar integral models

These data will be used to refine jet release characteristics for different release morphologies