

# Structural Dynamics Laboratory (ESP500 Structural Mechanics)

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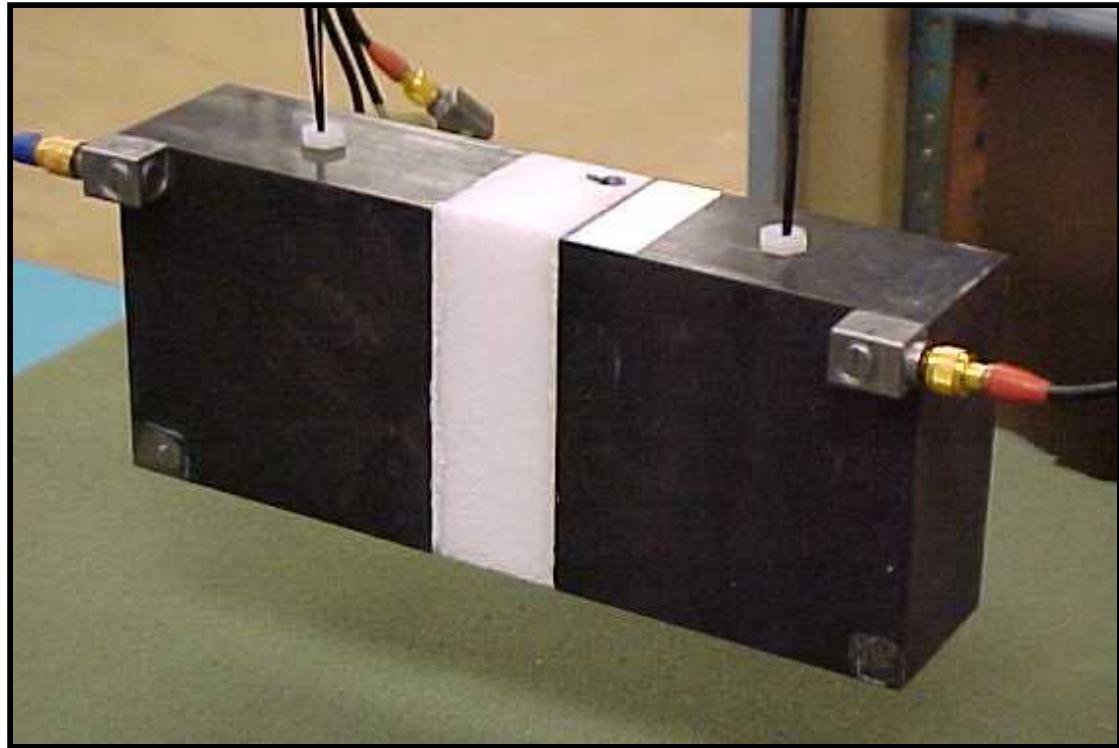
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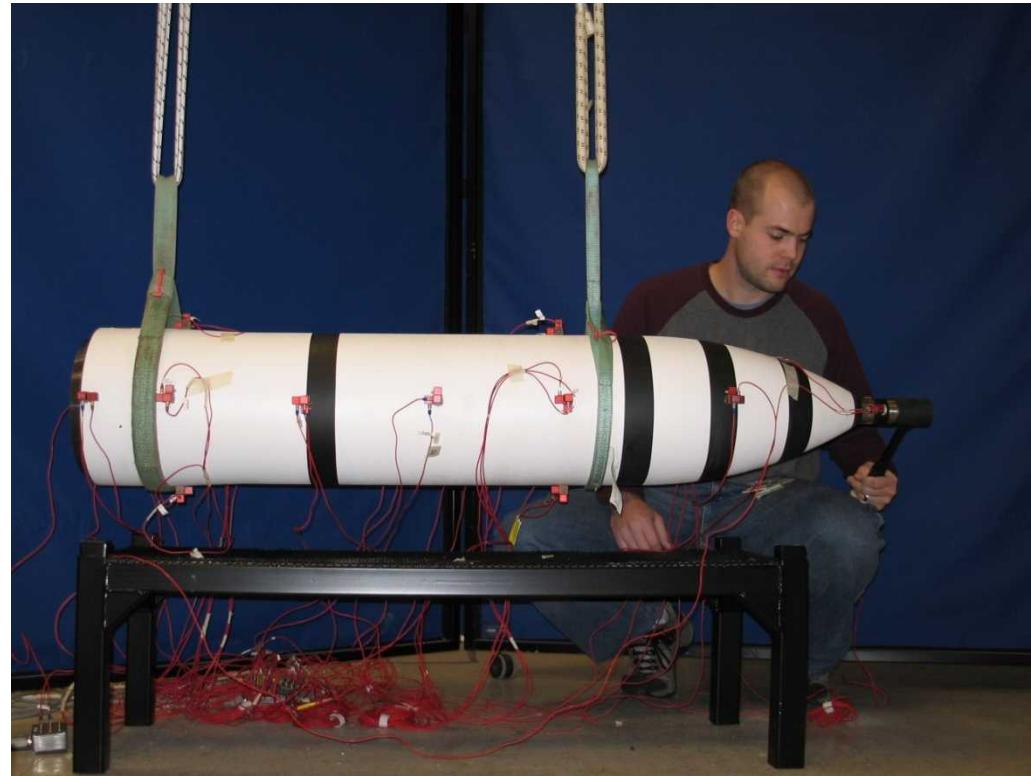
# Structural Dynamics Lab

- ◆ Experimental Modal Analysis
- ◆ Frequency Response Functions
- ◆ Modal Test Setup
- ◆ Modal Examples
  - Beam Modal
  - Foam Modal
  - Health Monitoring
- ◆ Other Test Techniques



# Experimental Modal Analysis

- ◆ Characterize dynamic response of structures
- ◆ Uses input/output relationship (FRF)
- ◆ Useful for analytical model validation
- ◆ Extract material/structural parameters
- ◆ Non-destructive test
- ◆ **ASSUMES LINEARITY**



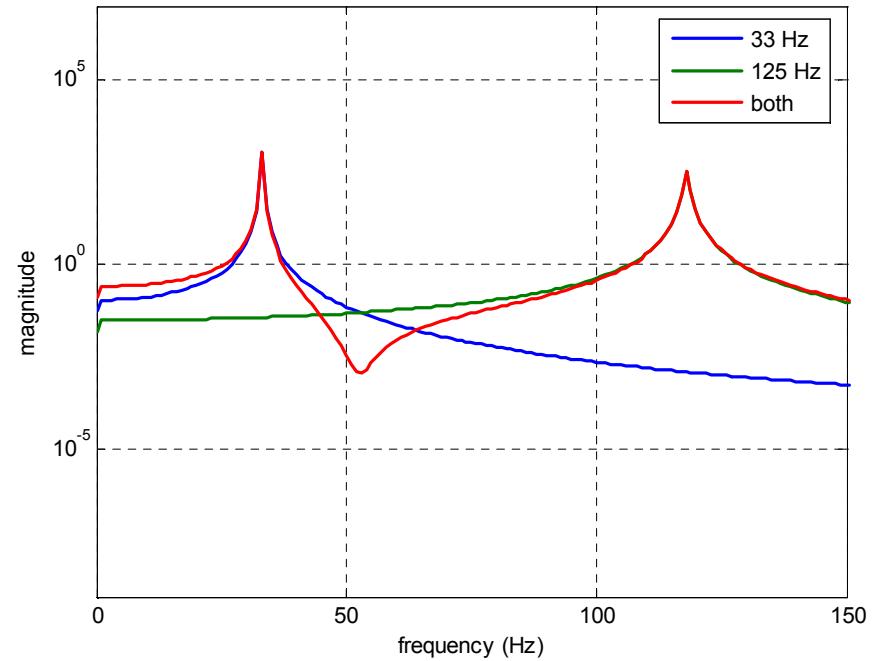
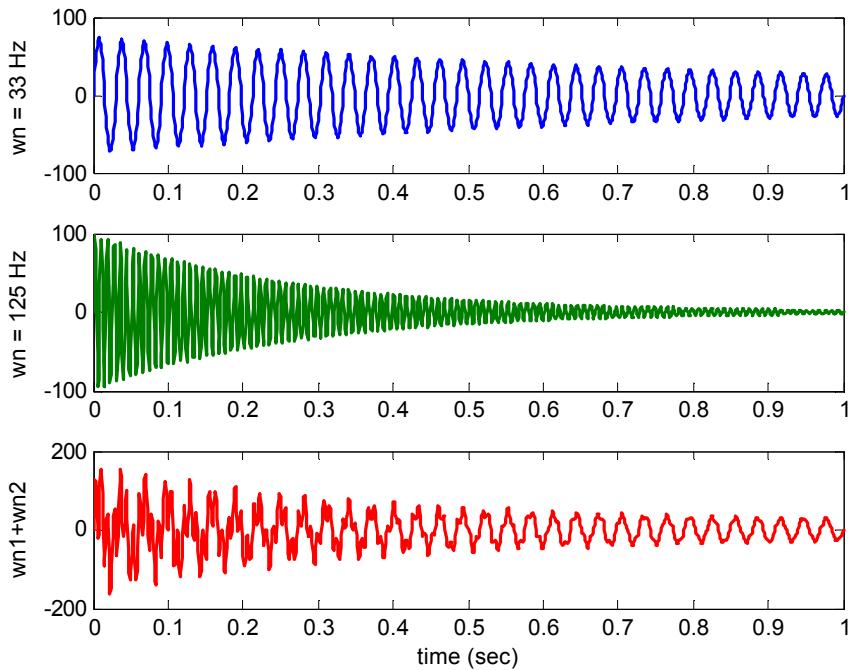
# Modal Analysis Parameters

- ◆ Natural Frequencies ( $\omega_n$ )
  - Frequencies at which structure resonates
- ◆ Damping ratio ( $\zeta_n$ )
  - Rate at which vibration at  $\omega_n$  decays to zero
- ◆ Mode Shapes ( $\Phi_n$ )
  - Shape structure forms at natural frequency



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# Welcome to the Frequency Domain! (Yes, I was scared too)

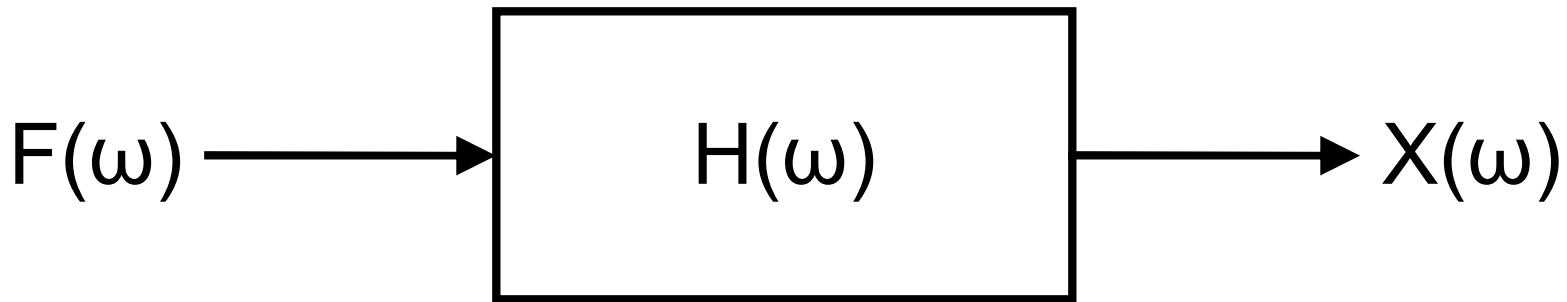


## ◆ Fast Fourier Transform

- Examine frequency content in time history signals
- Really fast!

# Frequency Response Function

## ◆ Linear System with Force Input and Displacement Response



$$H(\omega) = \frac{X(\omega)}{F(\omega)} = \frac{\phi\phi}{(\omega_n^2 - \omega^2) + i2\zeta_n\omega\omega_n}$$



# Frequency Response Functions

- ◆ Experimental modal testing typically measures accelerance due to easy-to-implement force transducers (most of the time) and the underlying physics of accelerometers.
- ◆ Can use any FRF for natural frequency and damping estimation, but mode shapes only scale properly for force input.

$$H(\omega) = \frac{X(\omega)}{F(\omega)}$$

receptance

$$H(\omega) = \frac{\dot{X}(\omega)}{F(\omega)}$$

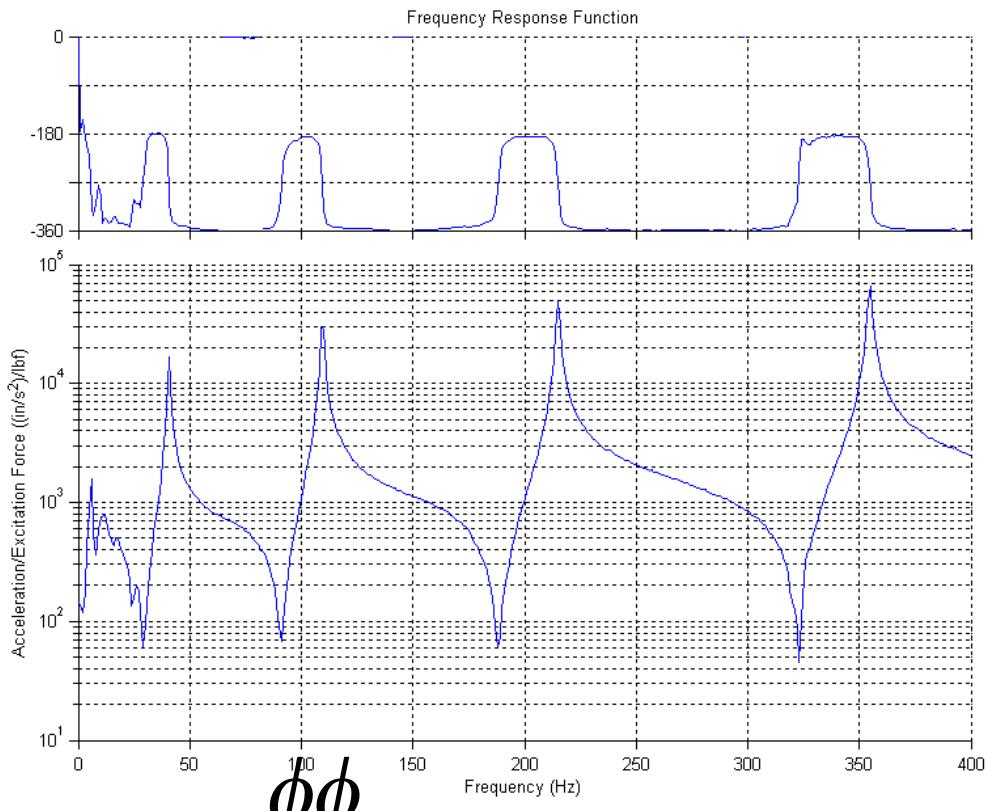
mobility

$$H(\omega) = \frac{\ddot{X}(\omega)}{F(\omega)}$$

accelerance

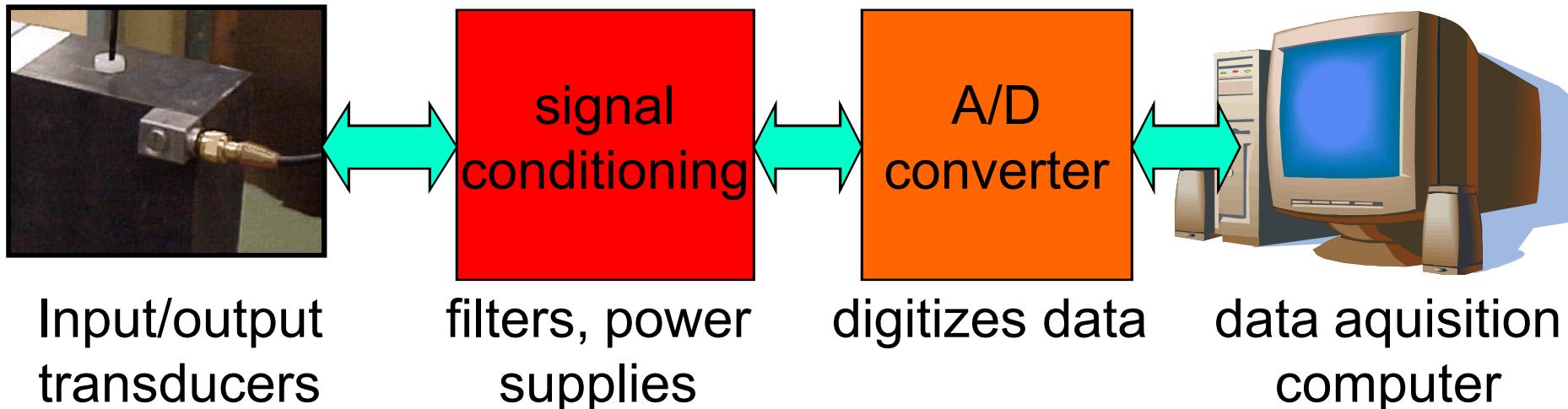
# Frequency Response Function

- ◆ FRFs are complex numbers, typically magnitude is plotted
- ◆ Frequencies at the peaks indicate natural frequencies
- ◆ Sharpness of peaks indicate damping
- ◆ Amplitudes at peaks used to calculate mode shapes



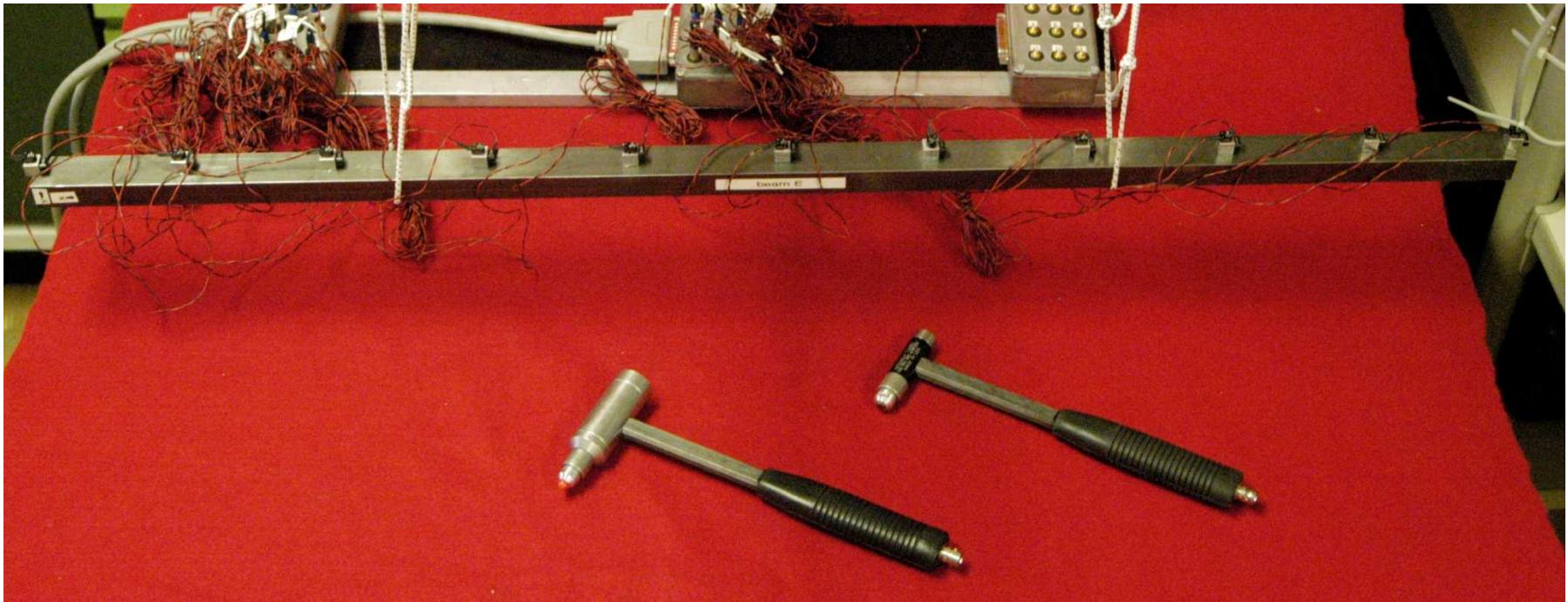
$$H(\omega) = \frac{\phi\phi}{(\omega_n^2 - \omega^2) + i2\zeta_n\omega\omega_n}$$

# Modal Test Setup



- ◆ Typically use force hammers and accelerometers.
- ◆ Data acquisition parameters must be selected: spectral lines, maximum frequency, frame size.
- ◆ Parameter selection determines: sampling frequency ( $2.56 \times \text{max freq.}$ ),  $\Delta t$  (1/samp.freq.),  $\Delta \text{freq}$  (1/frame size).

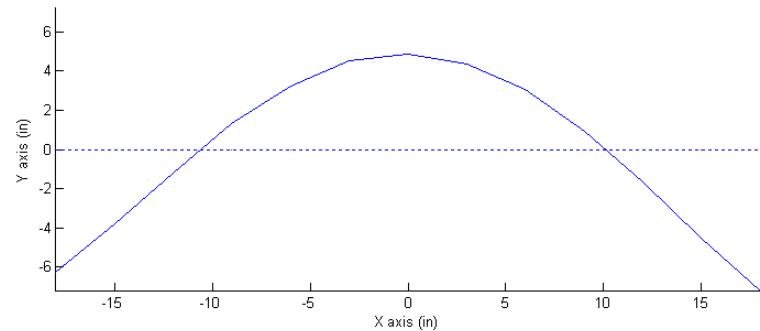
# Modal Analysis of Beam



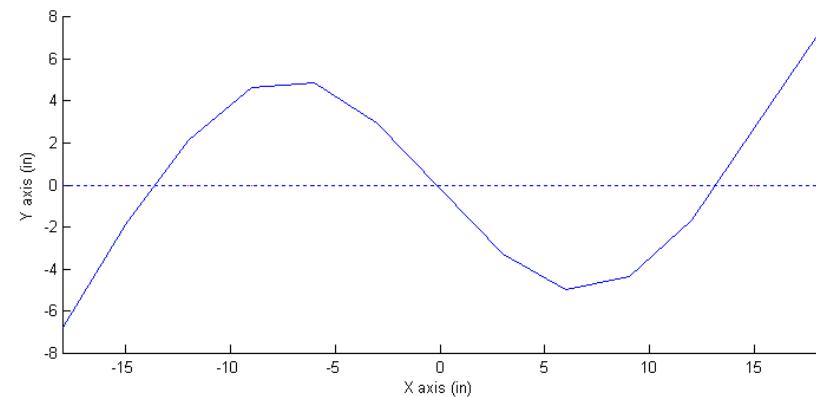
- ◆ Find natural frequencies, mode shape of beam
- ◆ Look at time histories, FRF, coherence
- ◆ Use FRF real/imag plot and smac (MATLAB program) to find results

# Modal Analysis of Beam

Mode 1  
Frequency: 40.713 Hz  
Damping: 0.870 %Cr  
IDLLine 1: Generated from reference 1Y-



Mode 2  
Frequency: 109.545 Hz  
Damping: 0.432 %Cr  
IDLLine 1: Generated from reference 1Y-



« « ? >>

Animate Stop

Close

« « ? >>

Animate Stop

Close

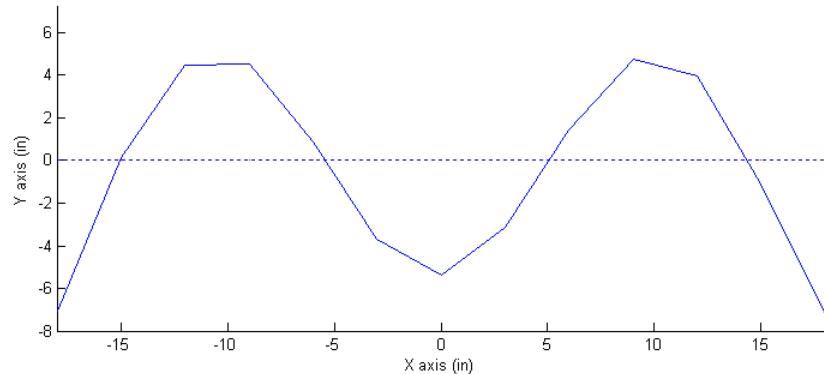
$$\omega_n = 40 \text{ Hz}, \zeta = 0.87\%$$

$$\omega_n = 109 \text{ Hz}, \zeta = 0.43\%$$

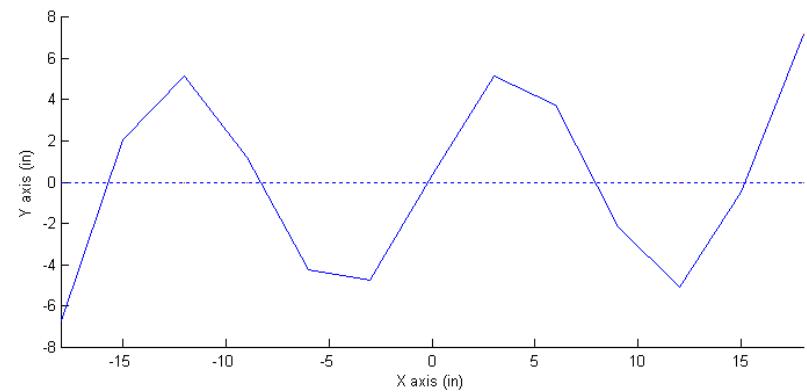


# Modal Analysis of Beam

Mode 3  
Frequency: 215.107 Hz  
Damping: 0.367 %Cr  
IDLine 1: Generated from reference 1Y-



Mode 4  
Frequency: 354.822 Hz  
Damping: 0.231 %Cr  
IDLine 1: Generated from reference 1Y-



« « ? >> » »

Animate Stop

Close « ? >>

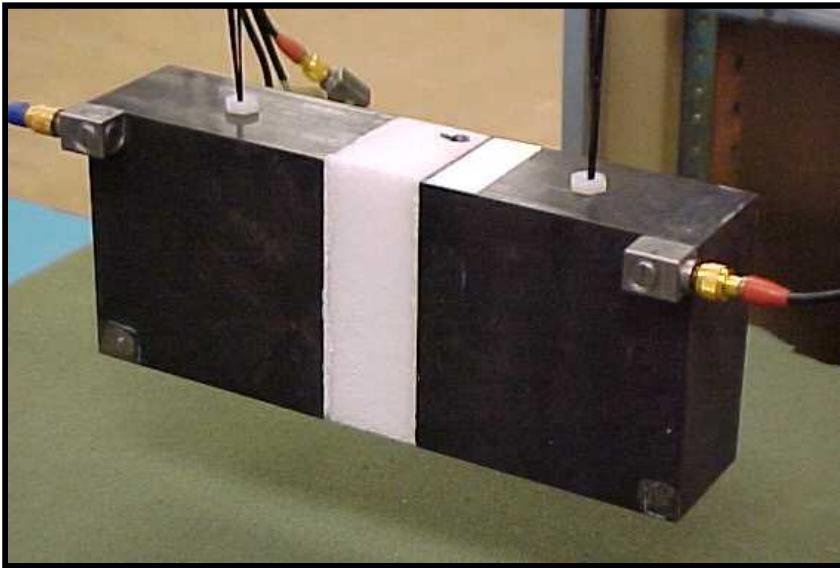
Animate Stop Close

$\omega_n = 215 \text{ Hz}$ ,  $\zeta = 0.36\%$

$\omega_n = 354 \text{ Hz}$ ,  $\zeta = 0.231\%$   
(requires more measurement points)

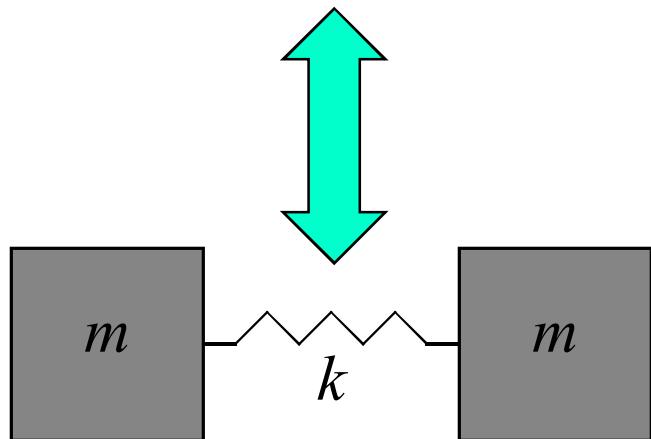
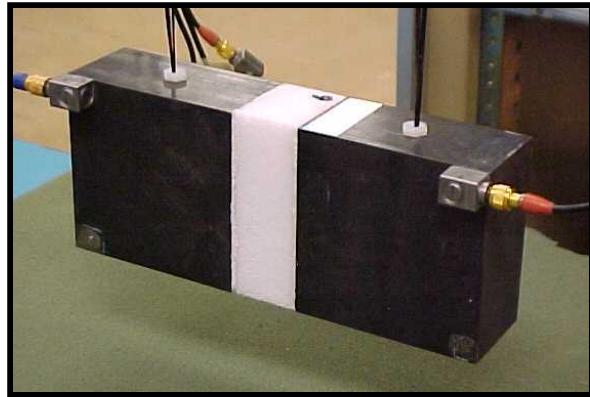


# Foam Example



- ◆ Two large masses connected with foam
- ◆ Model validation experiment for the foam
- ◆ Calibrate modulus used in material model

# Simple Calibration method

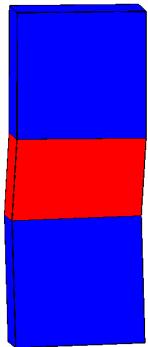


- ◆ View foam experiment as simple model
- ◆ Derive natural frequencies from simple model
- ◆ Find axial, bending, torsion frequencies experimentally (from mode shapes)
- ◆ Solve for stiffness,  $k$

$$\omega_{axial} = \sqrt{\frac{2k}{m}}$$

↔ unknown

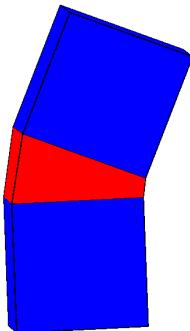
# Simple Calibration method



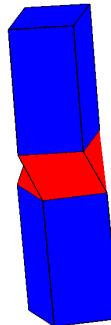
$$k_{axial} = \frac{EI}{l}$$

unknown

- ◆ Use various stiffness models to calculate modulus
- ◆ Show modal movies
- ◆ Congrats, you are a Calibration Expert!!!!

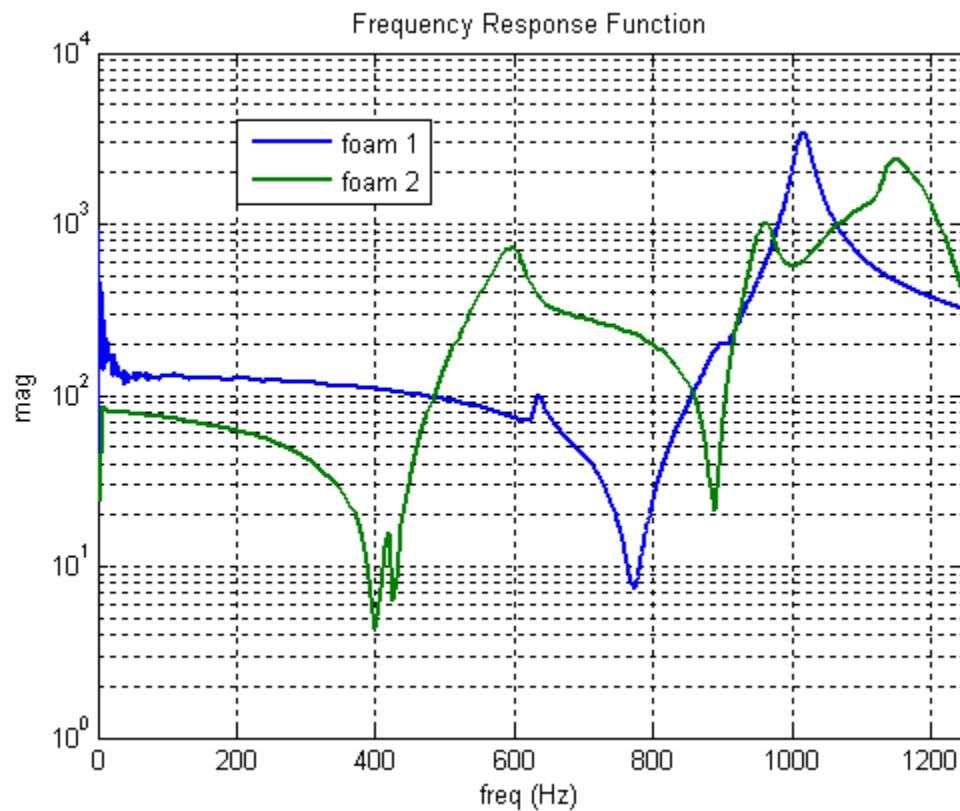


$$k_{bend} = \frac{3EI}{l^3}$$

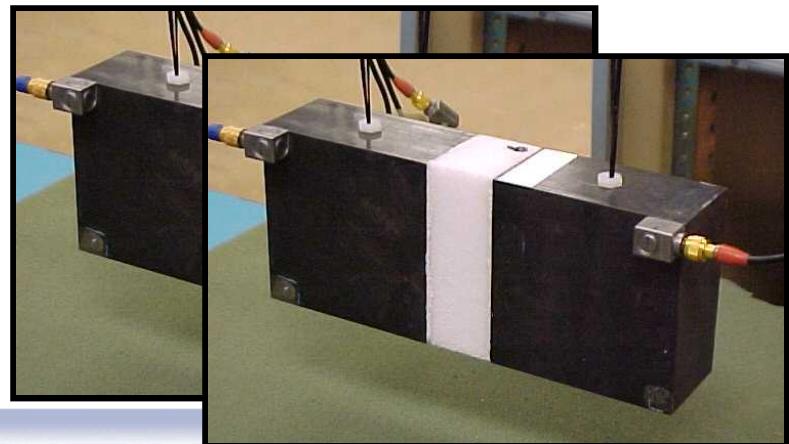


$$k_{torsion} = \frac{GJ_P}{l}$$

# Health Monitoring Example



- ◆ Similar structures, same foam
- ◆ Which one has been tested in thermal cycles?
- ◆ Look for non-symmetric peaks (non-linearity), increased damp.
- ◆ Answer: *foam 2*



# Other: Vibration Shaker Testing



- ◆ Electro-dynamic shakers used with closed-loop control system and control/limit accels to provide vibration input environment.
- ◆ PSDs used for control, but acceleration admittance ( $\ddot{X}/\ddot{X}$ ) can be

# Other: Acoustic Testing

- ◆ Loud-speakers used to provide acoustic/vibration environment
- ◆ Closed-loop control system used with control and limit mics for input.
- ◆ Several variations of setup exists: DFAT, reverberant /anechoic chamber.
- ◆ Not typically used for modal analysis, mainly for model validation.



MSI loudspeakers in Building 6560 reverberant chamber



# The End

## Any Questions?

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