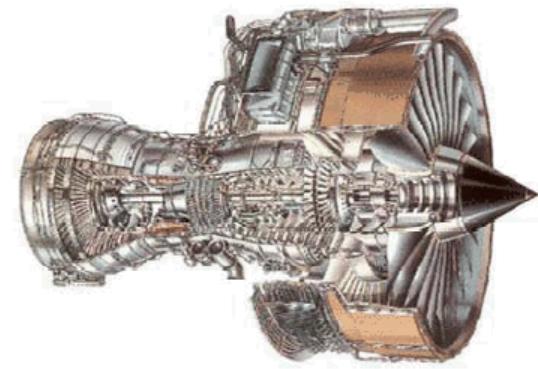
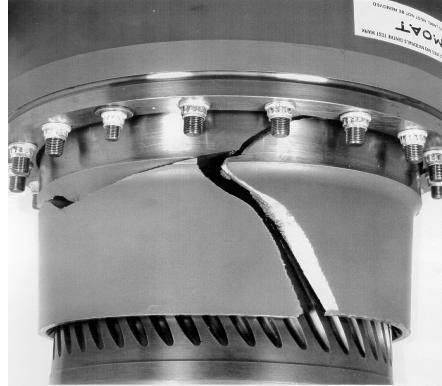
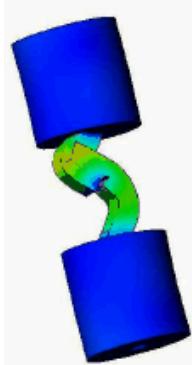
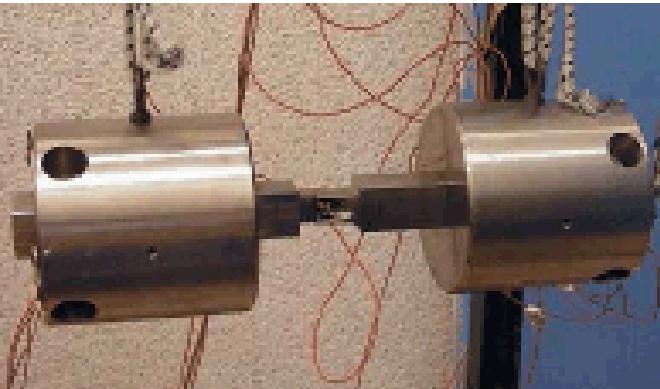


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# Reduced Order Modeling of Nonlinear Structures with Frictional Interfaces

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# Overview

- How can we efficiently model systems with strong nonlinearities?
  - Frequency Based Substructuring
  - Discontinuous Basis Functions
- How can we compare two different nonlinear models?
  - Time histories
  - FRFs
  - Nonlinear normal modes

# Frequency Based Substructuring

- Craig-Bampton reduction for the *linear* substructures

$$\begin{bmatrix} \mathbf{M}_{ff} & \mathbf{M}_{fi} \\ \mathbf{M}_{if} & \mathbf{M}_{ii} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{x}}_f \\ \ddot{\mathbf{x}}_i \end{bmatrix} + \begin{bmatrix} \mathbf{K}_{ff} & \mathbf{K}_{fi} \\ \mathbf{K}_{if} & \mathbf{K}_{ii} \end{bmatrix} \begin{bmatrix} \mathbf{x}_f \\ \mathbf{x}_i \end{bmatrix} = \begin{bmatrix} \mathbf{0} \\ \mathbf{f}_i \end{bmatrix}$$

- Frequency Based Substructuring based on a Harmonic Balance Method approach

$$\mathbf{M}\ddot{\mathbf{x}} + \mathbf{D}\dot{\mathbf{x}} + \mathbf{K}\mathbf{x} + \mathbf{f}_T(\dot{\mathbf{x}}, \mathbf{x}) = \mathbf{f}_{\text{exc}}$$

$$\mathbf{f}_T(\dot{\mathbf{x}}, \mathbf{x}) \approx \mathbf{D}_{\text{hbm}}\dot{\mathbf{x}} + \mathbf{K}_{\text{hbm}}\mathbf{x}$$

- This yields a frequency domain equation that is solved linearly

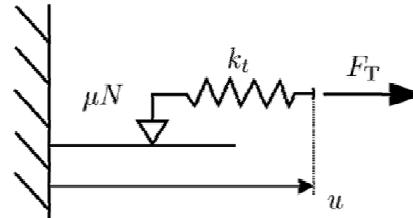
$$\left( \mathbf{L}_p^T (\mathbf{K} + \mathbf{B}_f^T \mathbf{K}_{\text{hbm}} \mathbf{B}_f) \mathbf{L}_p + i\omega \mathbf{L}_p^T (\mathbf{D} + \mathbf{B}_f^T \mathbf{D}_{\text{hbm}} \mathbf{B}_f) \mathbf{L}_p - \omega^2 \mathbf{L}_p^T \mathbf{M} \mathbf{L}_p \right) \hat{\mathbf{x}} = \mathbf{L}_p^T \hat{\mathbf{f}}$$

# Interface Flexibility and Damping

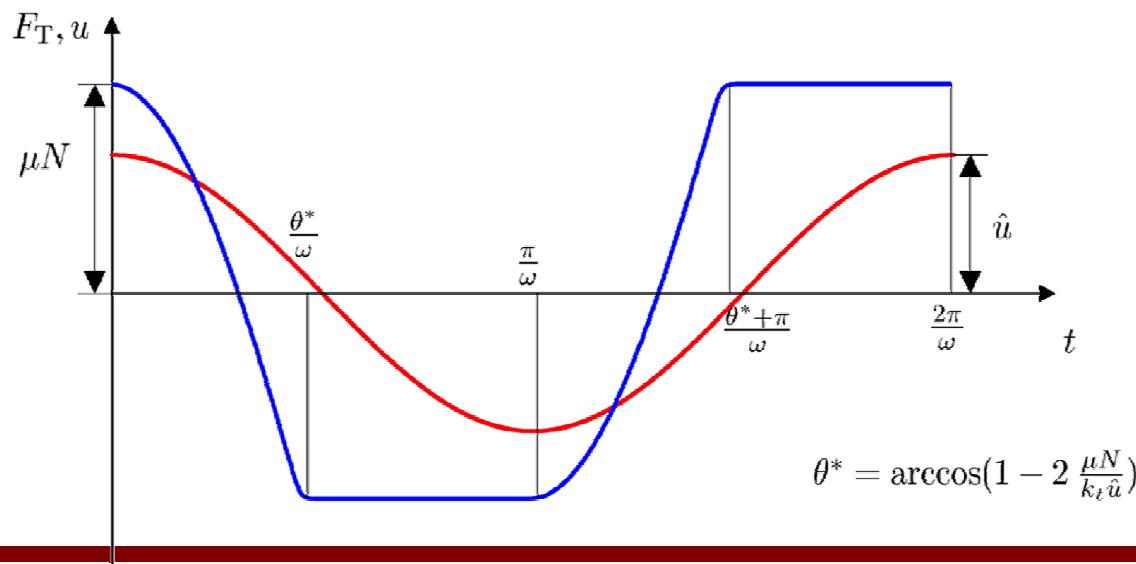
- Linearization of friction forces by Harmonic Balance Method

$$F_T(u) \approx k_{\text{hbm}}u + d_{\text{hbm}}\dot{u}$$

- Jenkins friction element



- Friction force for slip-stick



# Harmonic Linearized Coefficients

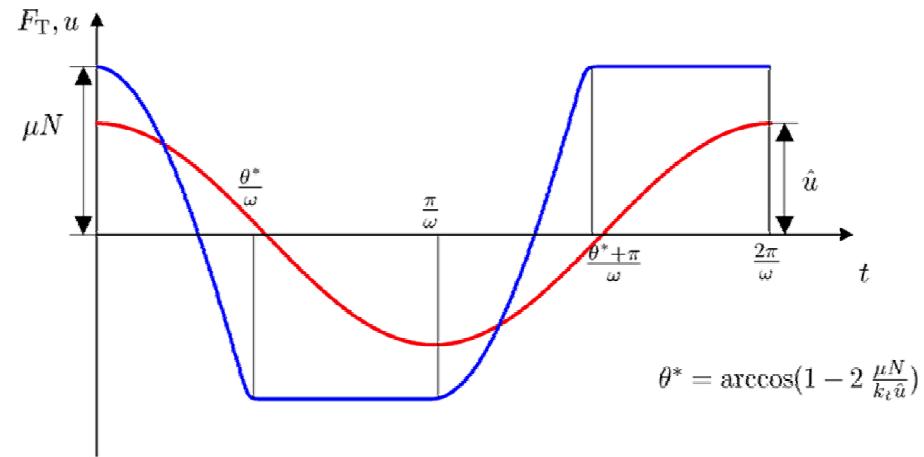
- Normalized critical amplitude

$$u^* = \frac{\mu N}{k_T \hat{u}}$$

- Sticking  $u^* \geq 1$

$$\begin{aligned}k_{\text{hbm}} &= k_T \\d_{\text{hbm}} &= 0\end{aligned}$$

- Stick-slip  $u^* < 1$

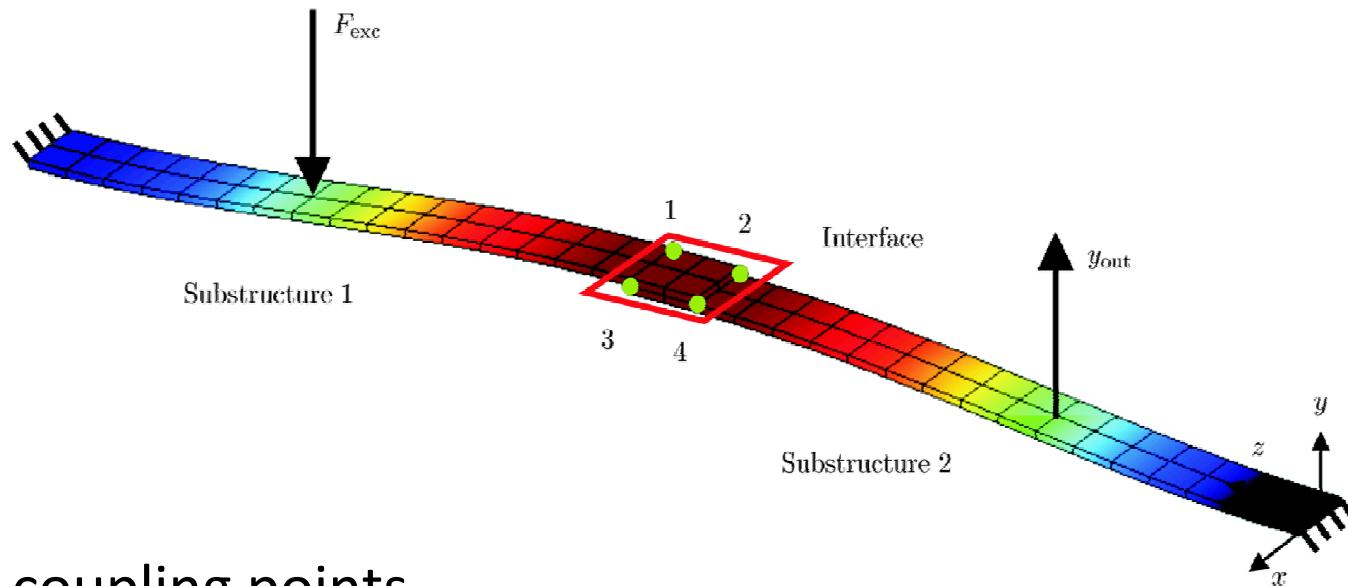


$$k_{\text{hbm}} = \frac{k_T}{\pi} \left( \arccos\left(1 - \frac{2}{u^*}\right) - \frac{2}{u^*} \left(1 - \frac{2}{u^*}\right) \sqrt{u^* - 1} \right)$$

$$d_{\text{hbm}} = \frac{4k_T}{\pi \omega u^*} \left(1 - \frac{1}{u^*}\right)$$

# Numerical Test Cases

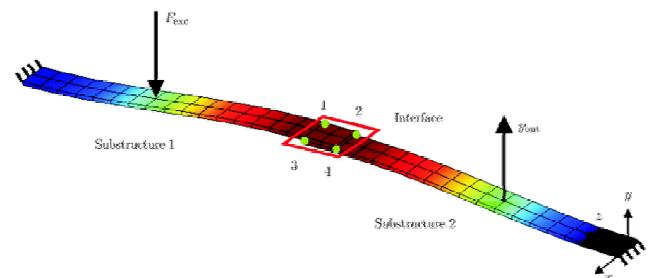
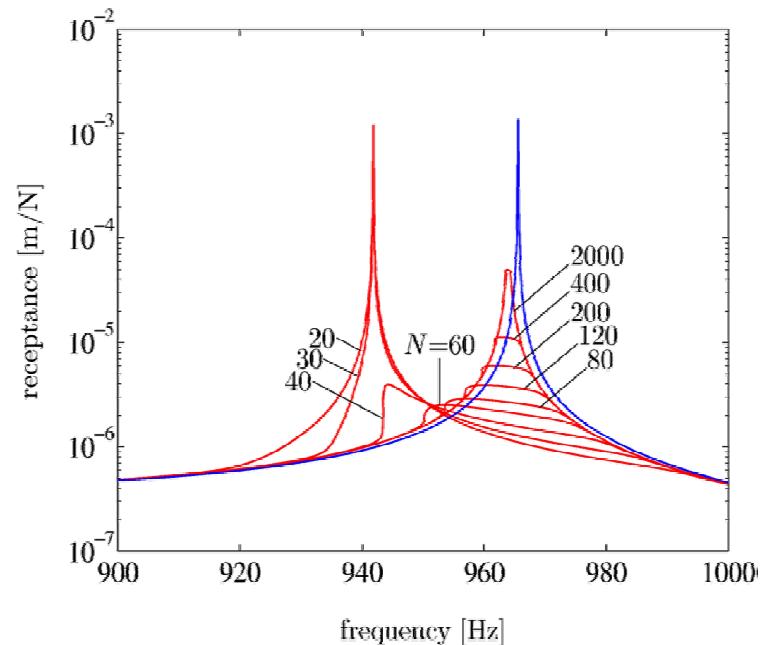
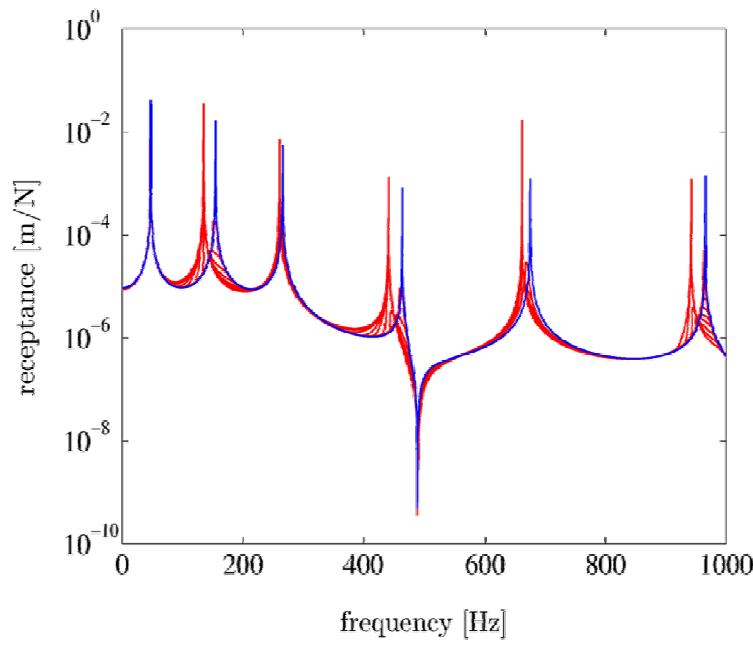
- Finite Element Model



- 4 coupling points
- Excitation of bending modes

# Numerical Test Cases

- FRF with excitation at substructure 1



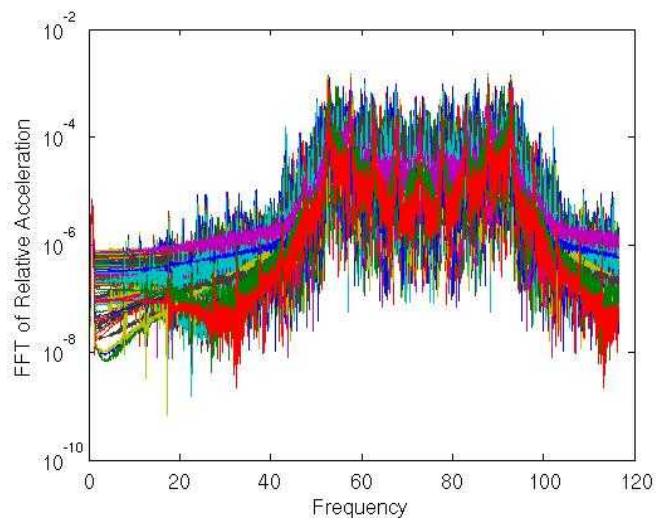
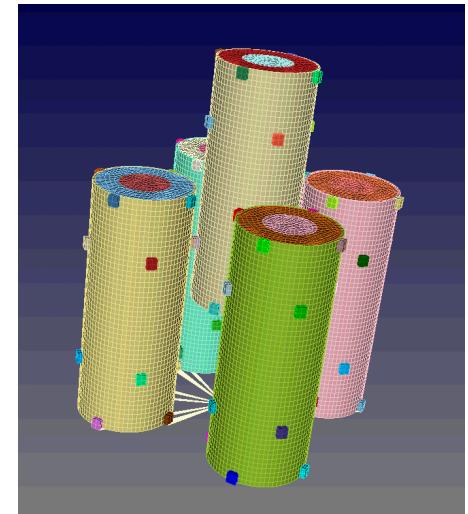
# Discontinuous Basis Functions

- Based on component mode synthesis
- CMS model augmented by a set of Milman Chu modes
- Only a few discontinuous basis functions needed for convergence,
- Can easily handle arbitrary nonlinearities

# Assessment of Efficiency

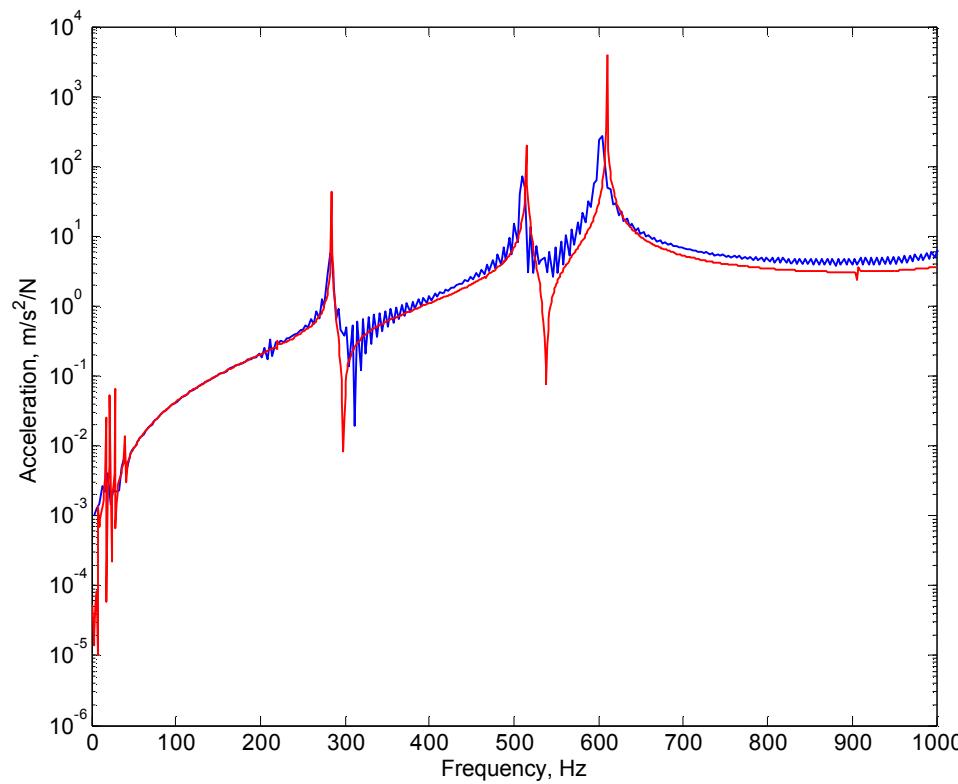
- 5 Body Example: 5 linear substructures connected via Iwan elements
- Effective computational savings of 24,000x.

Time Step	Salinas	Reduced Order Model
1e-4	Still running	312 minutes
2e-4	Still running	156 minutes
4e-4	7347 minutes	79 minutes
8e-4	Unstable	39 minutes
2e-3	Unstable	19.5 minutes
4e-3	Unstable	Unstable



# Comparison of Models

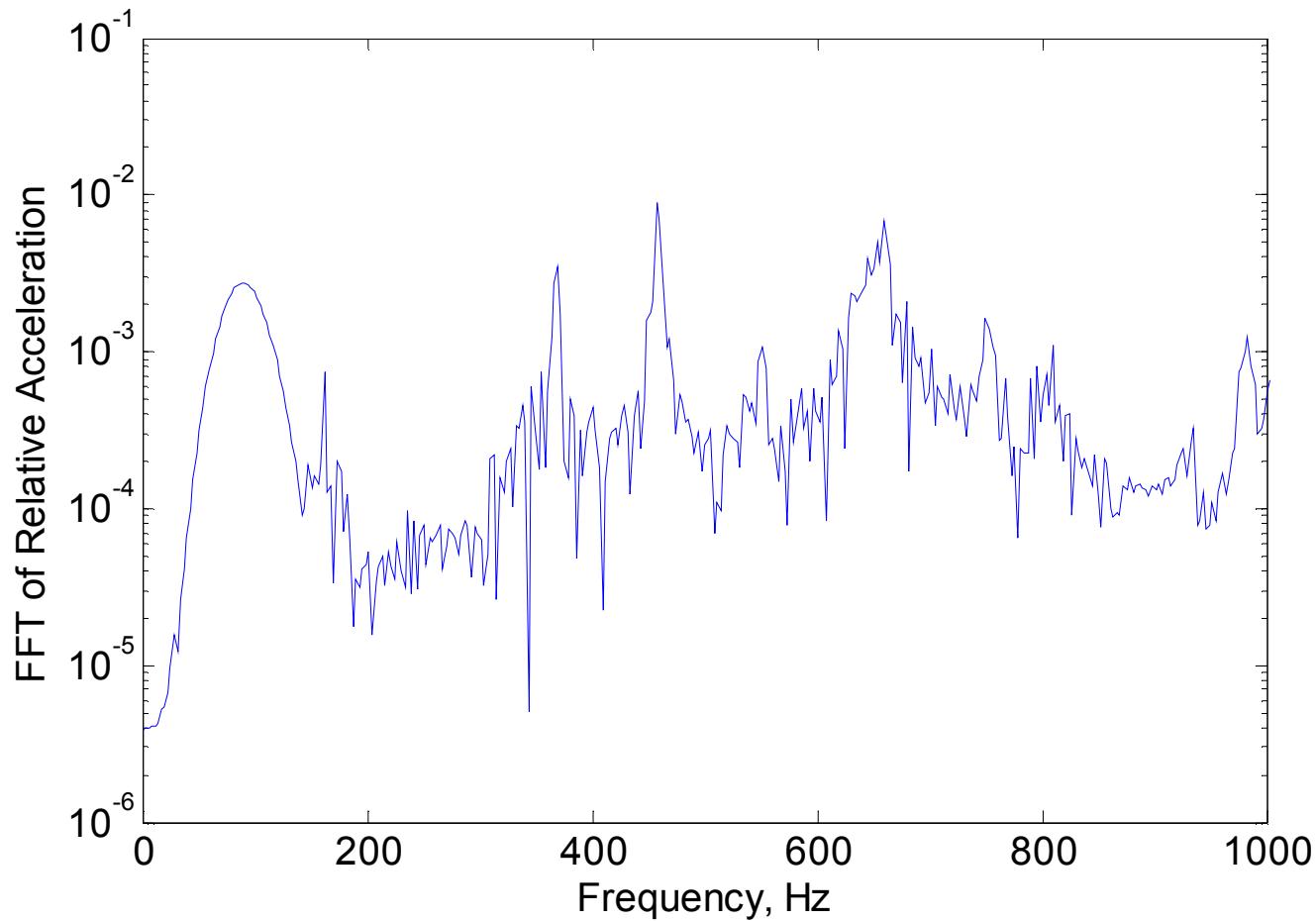
- Good agreement seen for linear springs connecting the interfaces



- Red: HBM, Blue: DBF.

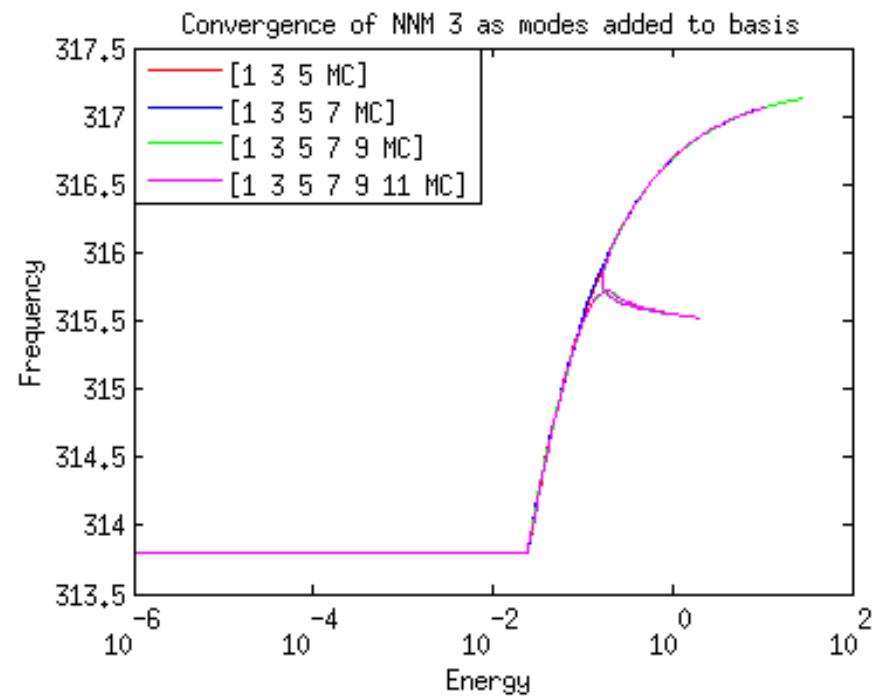
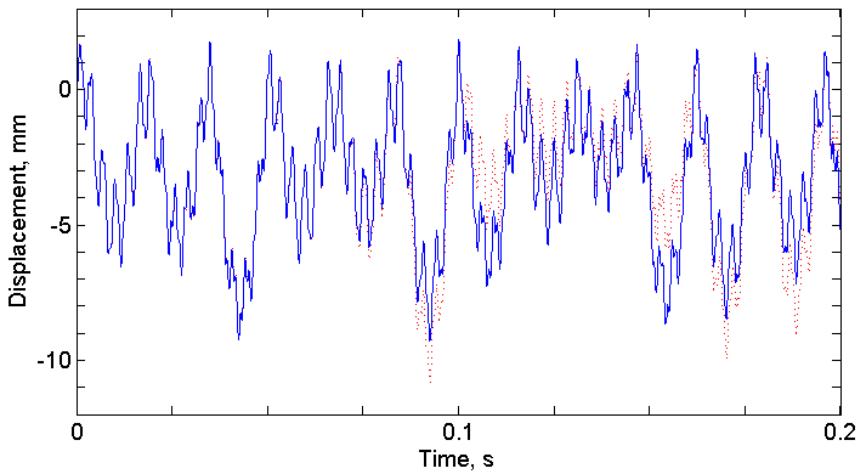
# Comparison of Nonlinear Models in Progress

- Sample result for a shock response with Iwan elements using DBF:



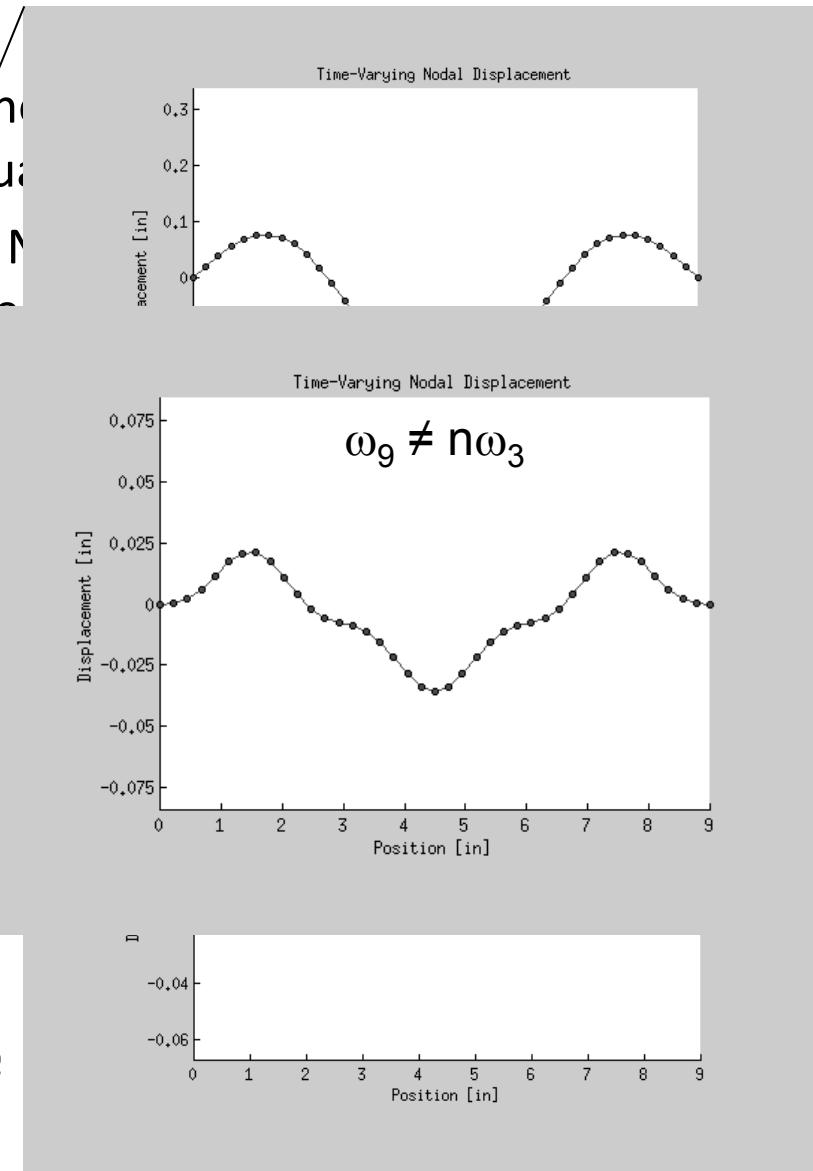
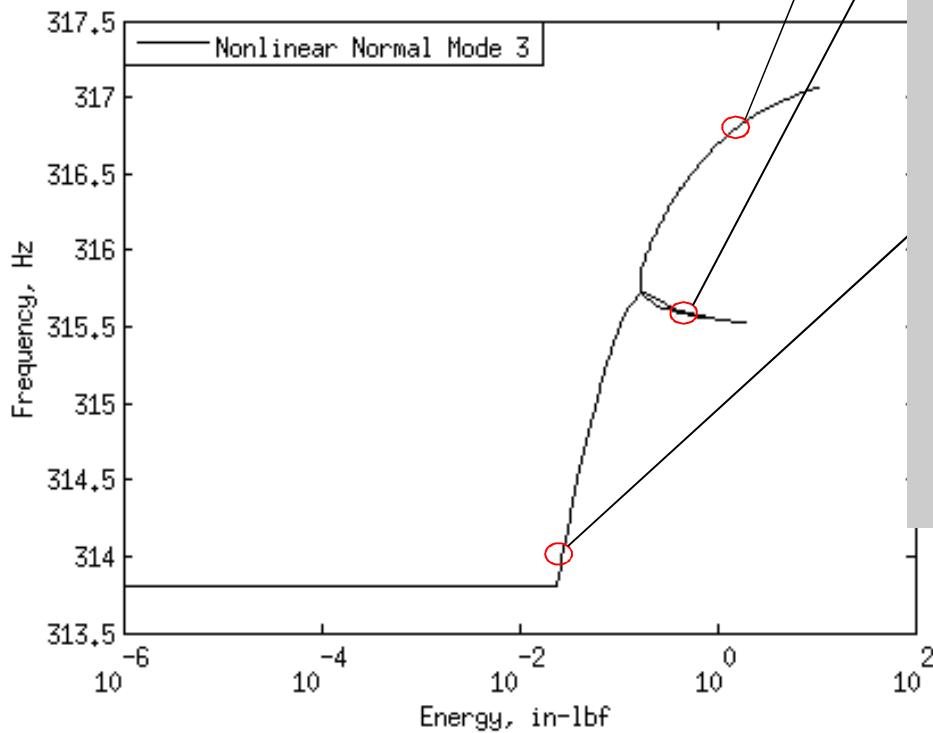
# Comparison of Nonlinear Systems

- How do you compare two different models of the same nonlinear system?
  - Time histories, dissipation, strain energy,  $L_2$  norm, etc.
  - Use of nonlinear normal modes to measure convergence



# Nonlinear Normal Modes Applied to a Beam Example

- A nonlinear normal mode is defined as a nonlinear periodic response to the conservative equations of motion
- For a nonlinear conservative system with  $N$  degrees of freedom, there are  $N$  NNM branches that initiate from each linear mode



# Summary

- Two methods compared for modeling nonlinear systems
- Frequency based substructuring very efficient when excitation and nonlinearities are able to be expressed as harmonic terms
- Method of discontinuous basis functions better suited for transient simulations and arbitrary nonlinearities
- Both methods approximately 24,000x faster than full FEA model