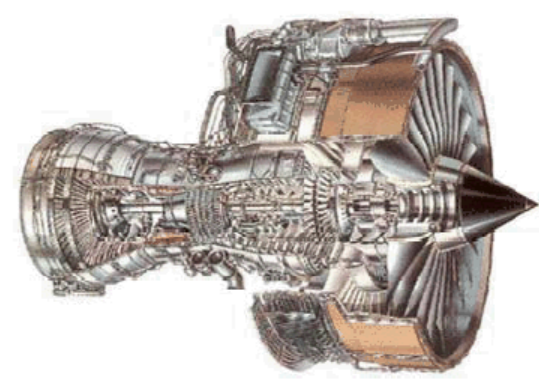
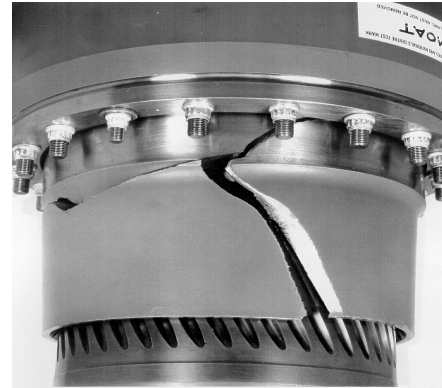
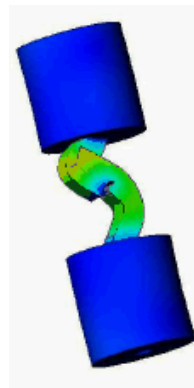
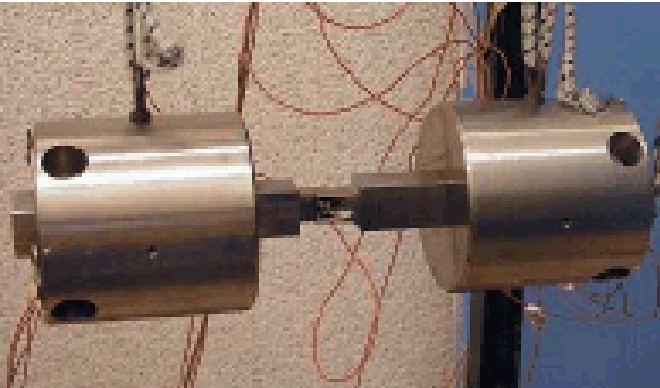


Exceptional service in the national interest



Reduced Order Modeling of Nonlinear Structures with Frictional Interfaces

M. R. Brake and D. J. Segalman, Sandia National Laboratories

P. Reuss and L. Gaul, the University of Stuttgart



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Overview

- How can we efficiently model systems with strong nonlinearities?
 - Frequency Based Substructuring
 - Discontinuous Basis Functions
- How can we compare two different nonlinear models?
 - Time histories
 - FRFs
 - Nonlinear normal modes

Frequency Based Substructuring

- Craig-Bampton reduction for the *linear* substructures

$$\begin{bmatrix} \mathbf{M}_{ff} & \mathbf{M}_{fi} \\ \mathbf{M}_{if} & \mathbf{M}_{ii} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{x}}_f \\ \ddot{\mathbf{x}}_i \end{bmatrix} + \begin{bmatrix} \mathbf{K}_{ff} & \mathbf{K}_{fi} \\ \mathbf{K}_{if} & \mathbf{K}_{ii} \end{bmatrix} \begin{bmatrix} \mathbf{x}_f \\ \mathbf{x}_i \end{bmatrix} = \begin{bmatrix} \mathbf{0} \\ \mathbf{f}_i \end{bmatrix}$$

- Frequency Based Substructuring based on a Harmonic Balance Method approach

$$\mathbf{M}\ddot{\mathbf{x}} + \mathbf{D}\dot{\mathbf{x}} + \mathbf{K}\mathbf{x} + \mathbf{f}_T(\dot{\mathbf{x}}, \mathbf{x}) = \mathbf{f}_{\text{exc}}$$

$$\mathbf{f}_T(\dot{\mathbf{x}}, \mathbf{x}) \approx \mathbf{D}_{\text{hbm}}\dot{\mathbf{x}} + \mathbf{K}_{\text{hbm}}\mathbf{x}$$

- This yields a frequency domain equation that is solved linearly

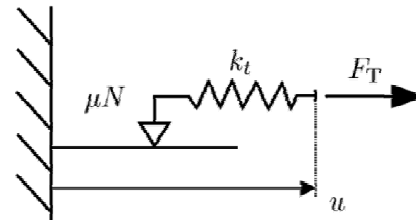
$$\left(\mathbf{L}_p^T (\mathbf{K} + \mathbf{B}_f^T \mathbf{K}_{\text{hbm}} \mathbf{B}_f) \mathbf{L}_p + i\omega \mathbf{L}_p^T (\mathbf{D} + \mathbf{B}_f^T \mathbf{D}_{\text{hbm}} \mathbf{B}_f) \mathbf{L}_p - \omega^2 \mathbf{L}_p^T \mathbf{M} \mathbf{L}_p \right) \hat{\mathbf{x}} = \mathbf{L}_p^T \hat{\mathbf{f}}$$

Interface Flexibility and Damping

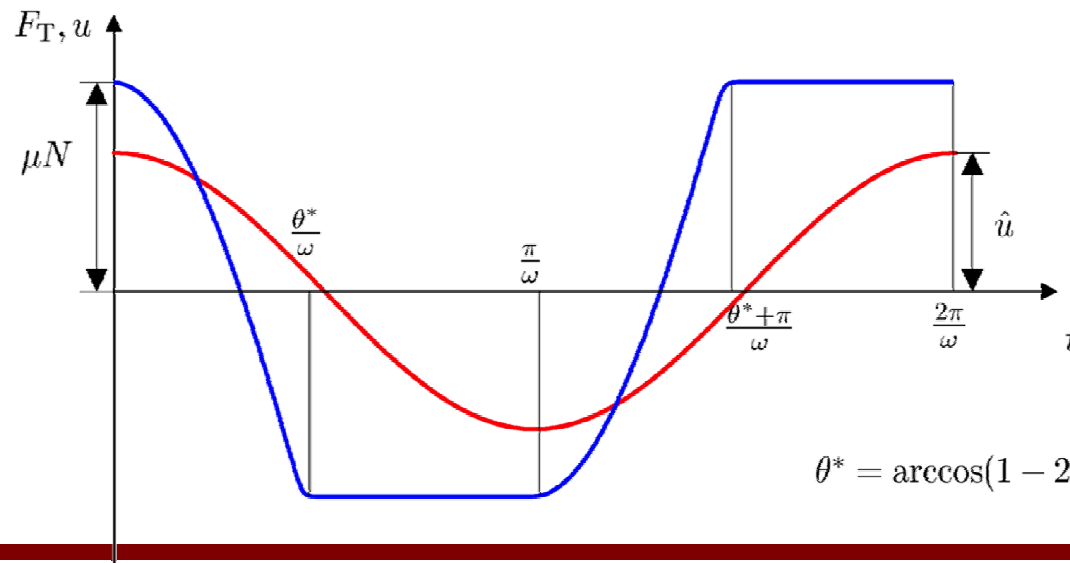
- Linearization of friction forces by Harmonic Balance Method

$$F_T(u) \approx k_{\text{hbm}}u + d_{\text{hbm}}\dot{u}$$

- Jenkins friction element



- Friction force for slip-stick



$$\theta^* = \arccos(1 - 2 \frac{\mu N}{k_t \hat{u}})$$

Harmonic Linearized Coefficients

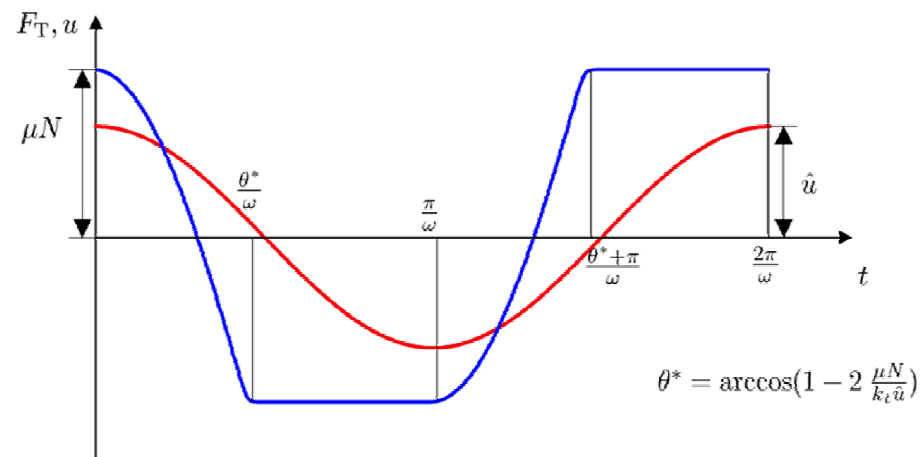
- Normalized critical amplitude

$$u^* = \frac{\mu N}{k_T \hat{u}}$$

- Sticking $u^* \geq 1$

$$k_{\text{hbm}} = k_T$$

$$d_{\text{hbm}} = 0$$



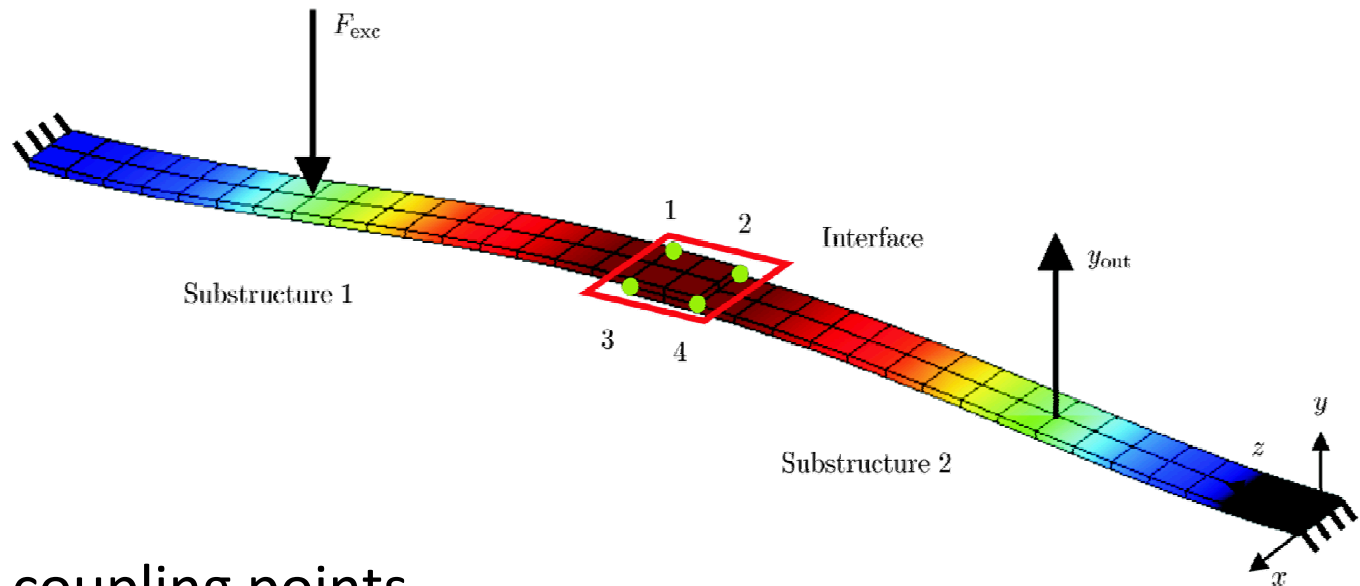
- Stick-slip $u^* < 1$

$$k_{\text{hbm}} = \frac{k_T}{\pi} \left(\arccos\left(1 - \frac{2}{u^*}\right) - \frac{2}{u^*} \left(1 - \frac{2}{u^*}\right) \sqrt{u^* - 1} \right)$$

$$d_{\text{hbm}} = \frac{4k_T}{\pi \omega u^*} \left(1 - \frac{1}{u^*}\right)$$

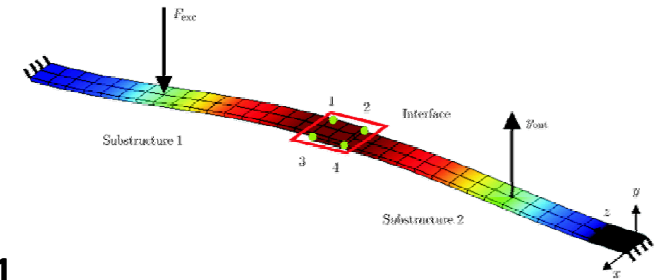
Numerical Test Cases

- Finite Element Model

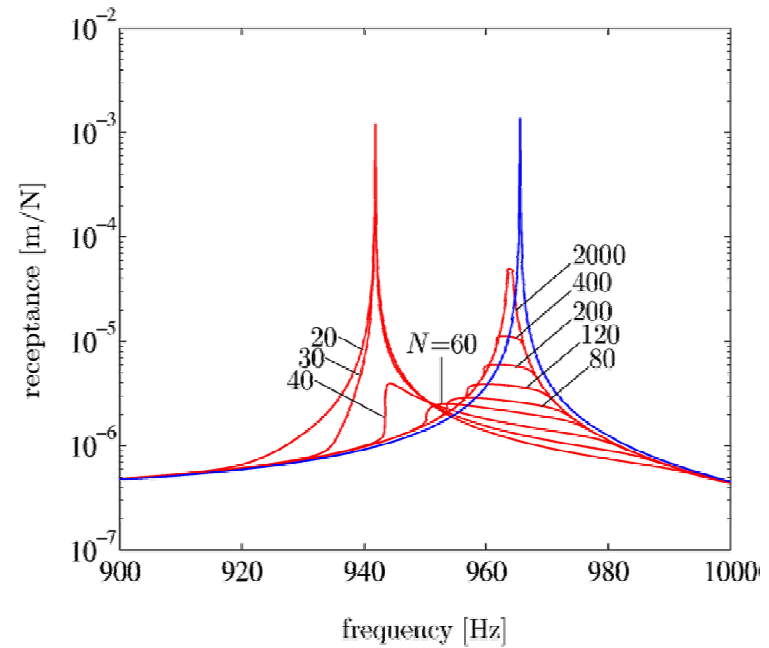
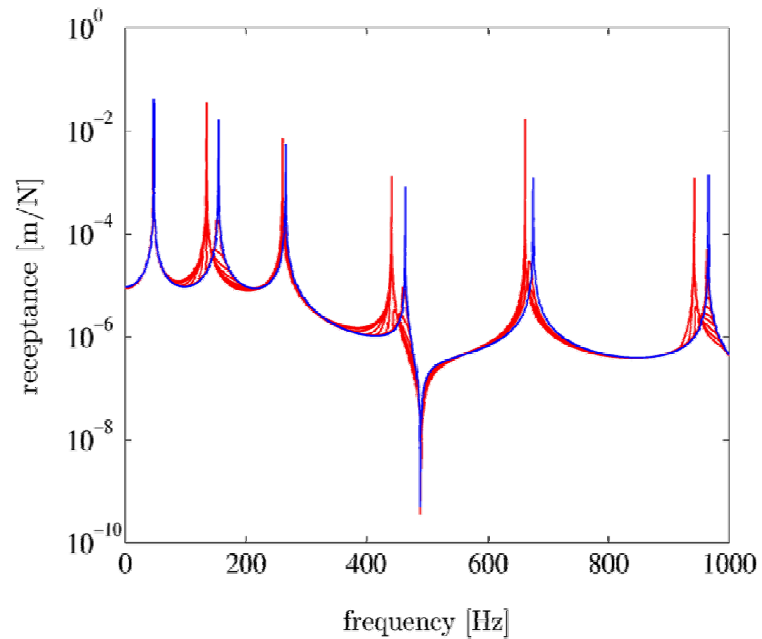


- 4 coupling points
- Excitation of bending modes

Numerical Test Cases



- FRF with excitation at substructure 1



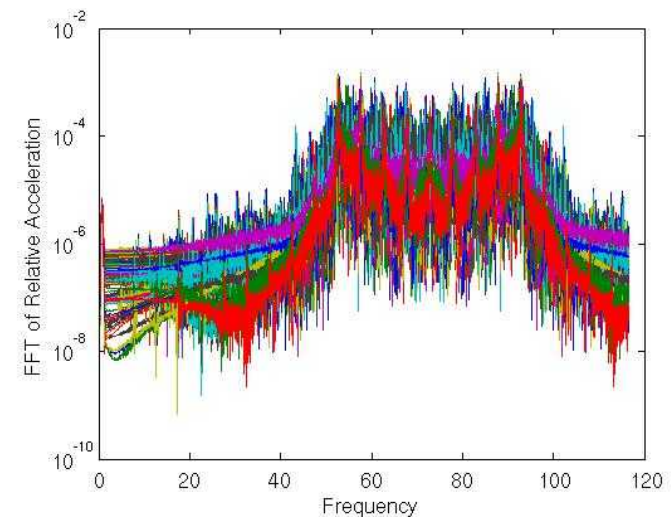
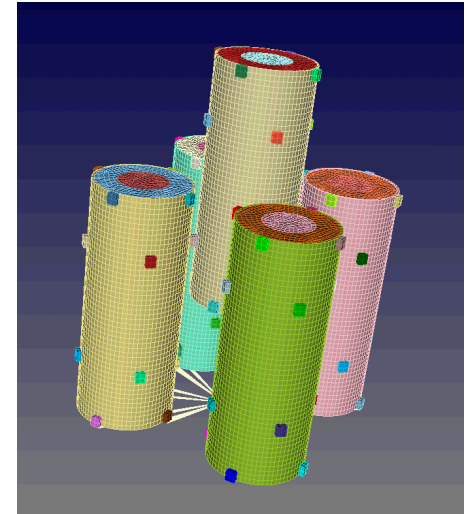
Discontinuous Basis Functions

- Based on component mode synthesis
- CMS model augmented by a set of Milman Chu modes
- Only a few discontinuous basis functions needed for convergence,
- Can easily handle arbitrary nonlinearities

Assessment of Efficiency

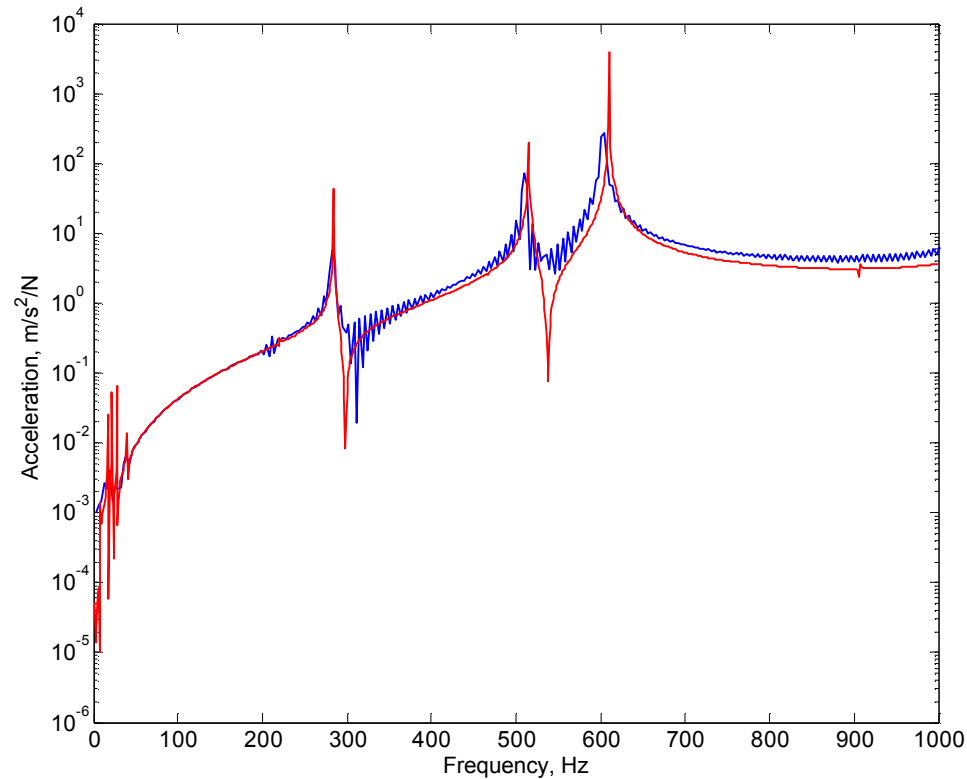
- 5 Body Example: 5 linear substructures connected via Iwan elements
- Effective computational savings of 24,000x.

Time Step	Salinas	Reduced Order Model
1e-4	Still running	312 minutes
2e-4	Still running	156 minutes
4e-4	7347 minutes	79 minutes
8e-4	Unstable	39 minutes
2e-3	Unstable	19.5 minutes
4e-3	Unstable	Unstable



Comparison of Models

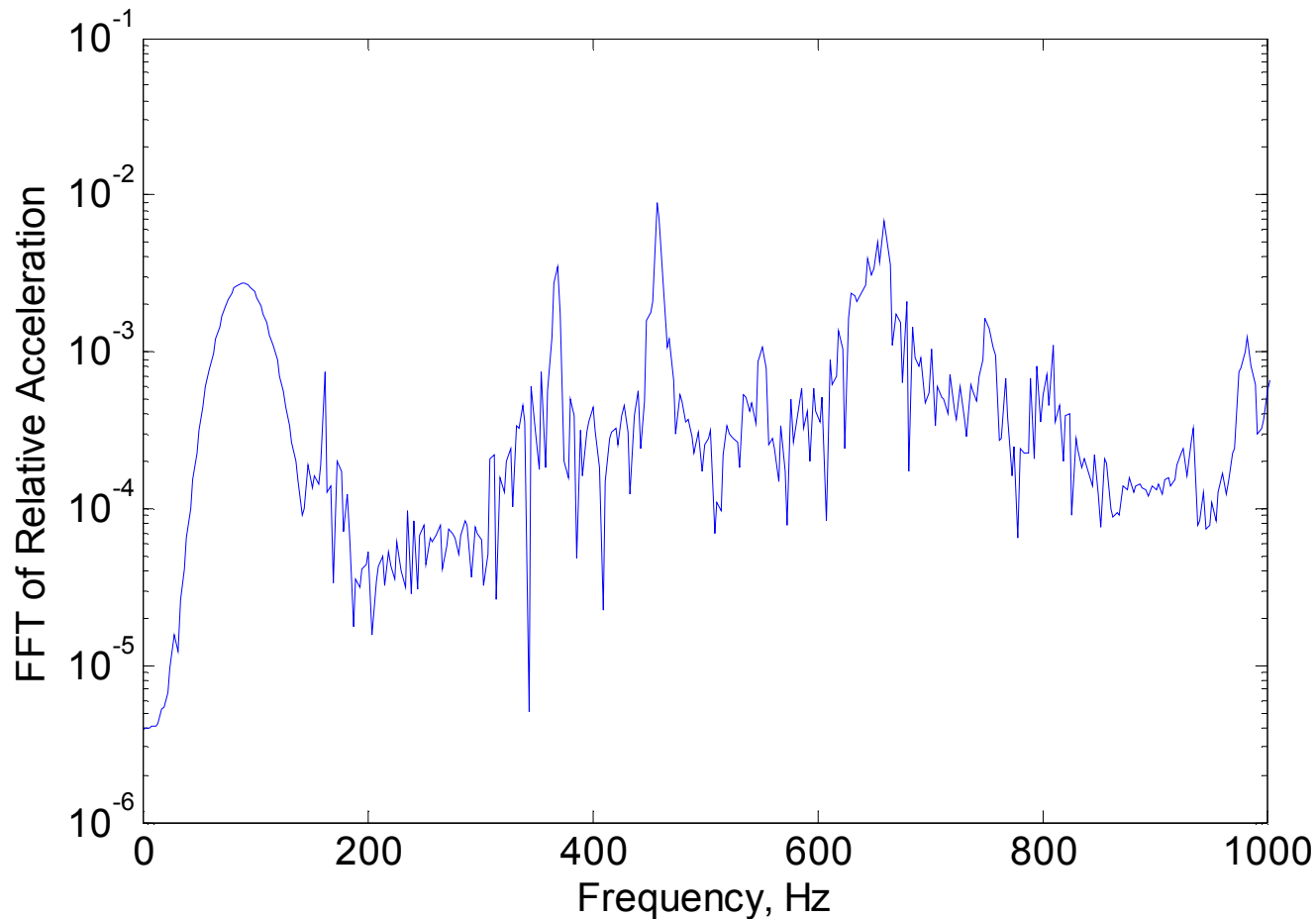
- Good agreement seen for linear springs connecting the interfaces



- Red: HBM, Blue: DBF.

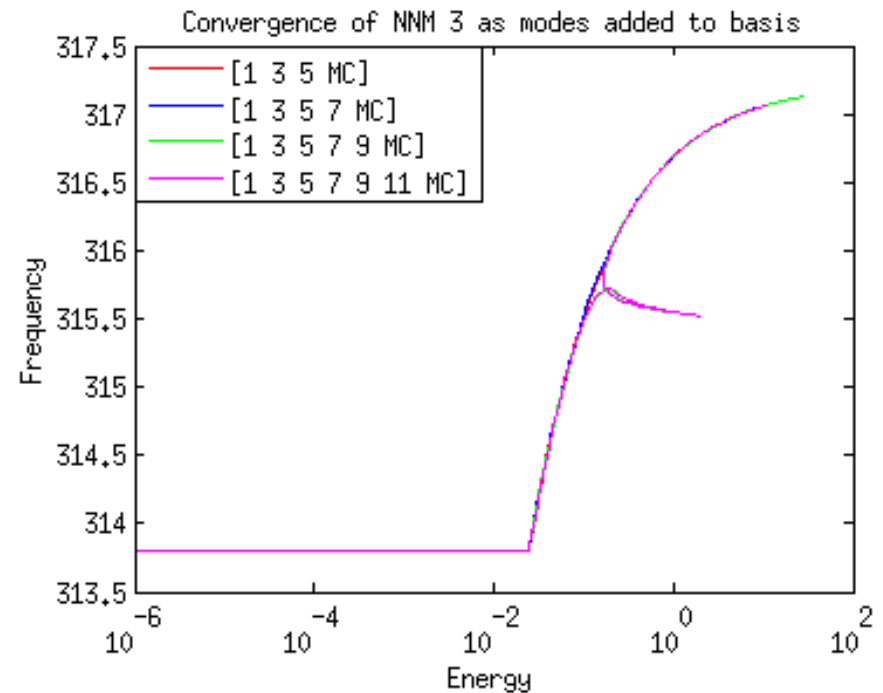
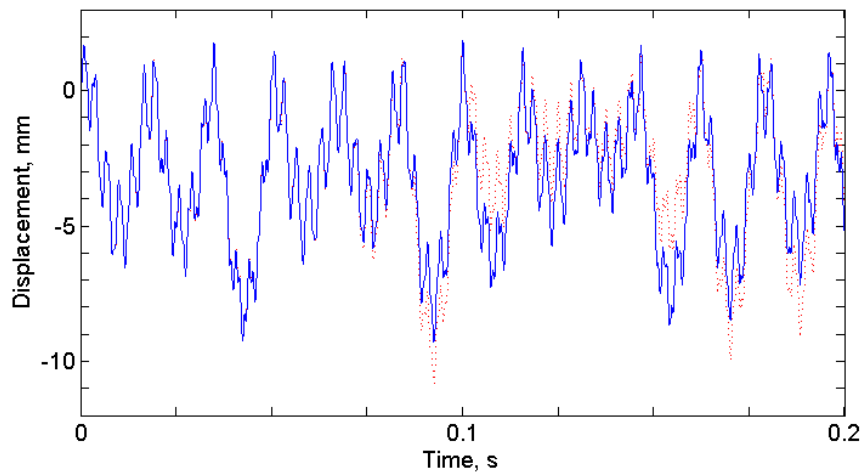
Comparison of Nonlinear Models in Progress

- Sample result for a shock response with Iwan elements using DBF:



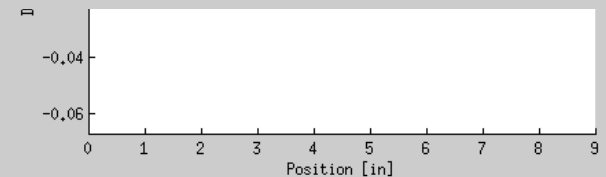
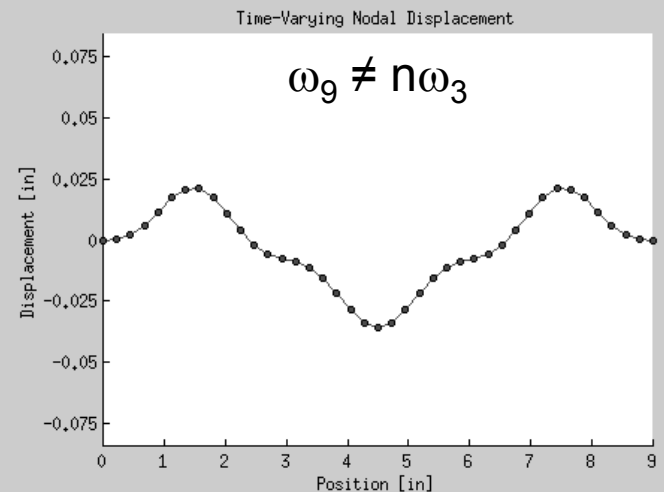
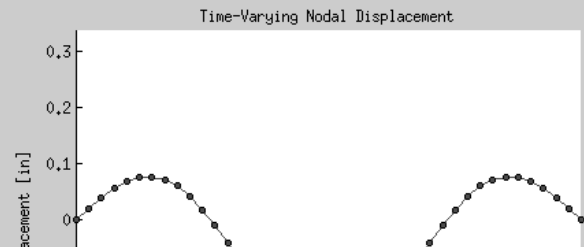
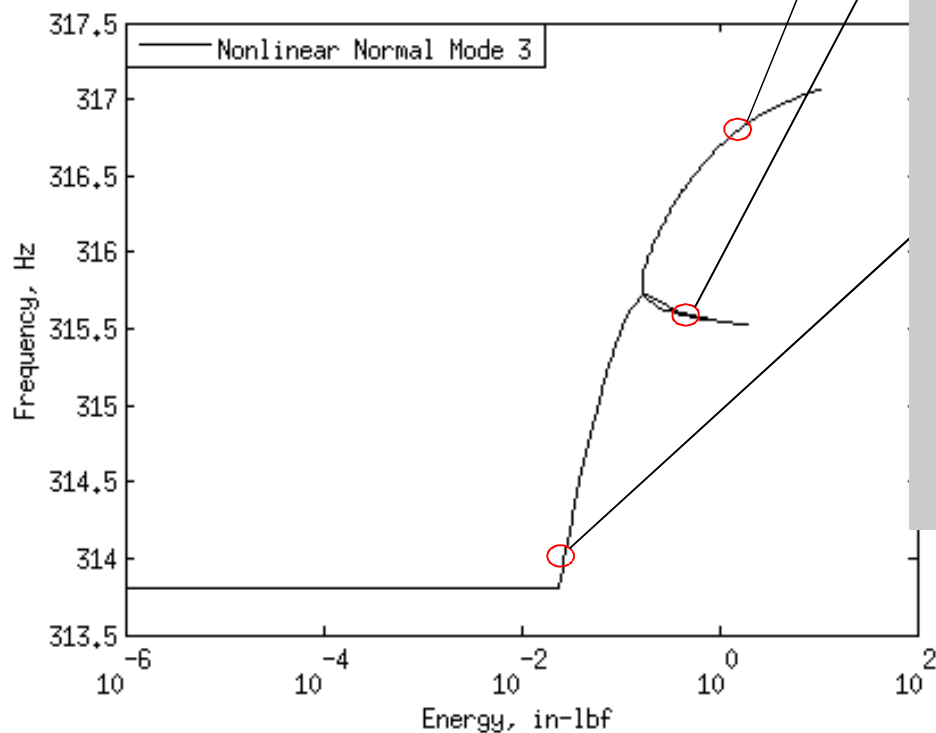
Comparison of Nonlinear Systems

- How do you compare two different models of the same nonlinear system?
 - Time histories, dissipation, strain energy, L_2 norm, etc.
 - Use of nonlinear normal modes to measure convergence



Nonlinear Normal Modes Applied to a Beam Example

- A nonlinear normal mode is defined as a nontrivial periodic response to the conservative equations of motion
- For a nonlinear conservative system with N degrees of freedom, there are N NNM branches that initiate from each linear normal mode



Summary

- Two methods compared for modeling nonlinear systems
- Frequency based substructuring very efficient when excitation and nonlinearities are able to be expressed as harmonic terms
- Method of discontinuous basis functions better suited for transient simulations and arbitrary nonlinearities
- Both methods approximately 24,000x faster than full FEA model