

Optimal Resistive Control Strategy for a Floating OWC

EWTEC 2013: Aalborg Denmark

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September 5th, 2013.

SNL Water Power Program

Marine Hydrokinetics, Offshore Wind, Conventional Hydropower

SF-Bay
Hydrodynamics

Unique Capabilities

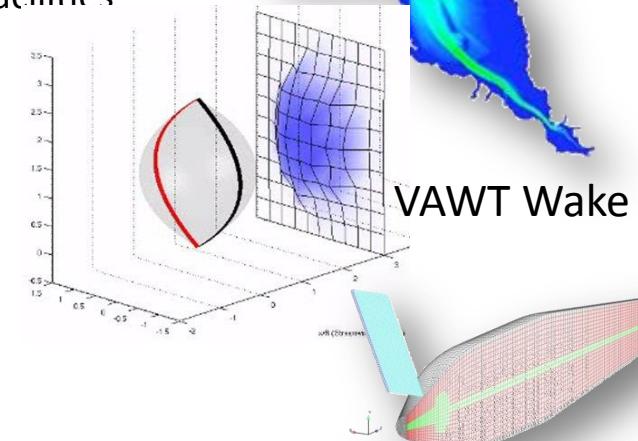
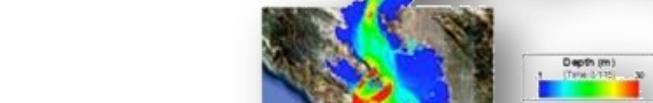
- MHK environmental circulation and performance code (SNL-EFDC)
- Composite structural materials and anti-biofouling coatings test facilities
- Sandia Lake Facility – TRL 6 appropriate for wave testing
- SEAWOLF lab/field oscillatory-flow sediment transport testing
- HydroSCOPE Seasonal Optimization Tool (CH)

Collaborative Projects

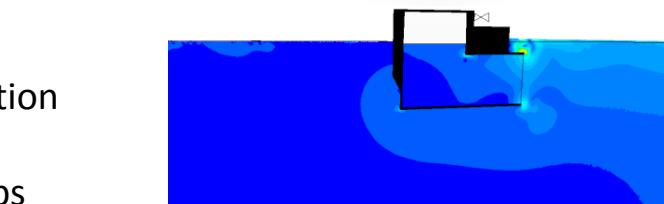
- Technical Industry Support
 - Ocean Renewable Power Company,
 - Ocean Power Technologies,
 - Snohomish PUD
- SNL-EFDC Technology Transfer to
 - Free Flow Power, NOAA, FERC, BOEM, Verdant, ORPC

Impact Examples

- Whale strike analysis (collaboration with PNNL) allowing demonstration project to proceed in Puget Sound
- Leading the techno-economic report to Congress detailing what steps need to be taken to ensure the growth of the WEC industry.
- Novel vertical axis wind turbine designs and structural health monitoring for offshore wind devices.



VAWT Wake



Whale Strike Analysis



Sandia Lake Facility



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Reference Model Project



Summary

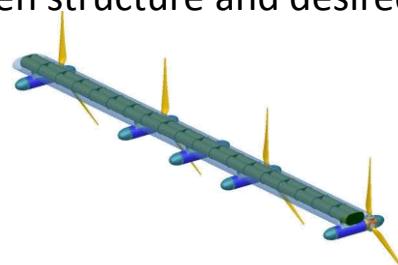
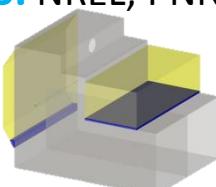
- Multi-Lab effort to obtain **baseline** performance and Cost Of Energy (**COE**) estimates for a variety of Marine Hydro-Kinetic (MHK) devices, sponsored by DOE.
- Method to achieve COE is to develop **public domain designs** incorporating the following:
 - Power performance models
 - Structural models
 - Anchor and mooring design
 - Economic Model
 - PTO Design
 - O&M / Installation
 - Permitting & Environment
- Designs are intended to be conservative, robust, and experimentally verified.



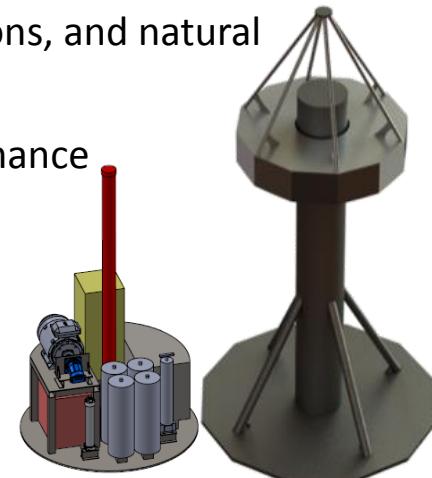
SNL Developed Models/Tools

- **Performance Models** — WEC: 3D model capable of handling 7DOF in Matlab; FEC: CACTUS
- **Survival Model** — utilize a Morison's Eq. approach to model extreme conditions
- **Structural Sizing Tool** — determination of weight, ballast, COG & COB locations, and natural frequencies
- **PTO Sizing Tool** — Turbine sizing tool for given structure and desired performance

Partners: NREL, PNNL, ARL/PSU, ORNL



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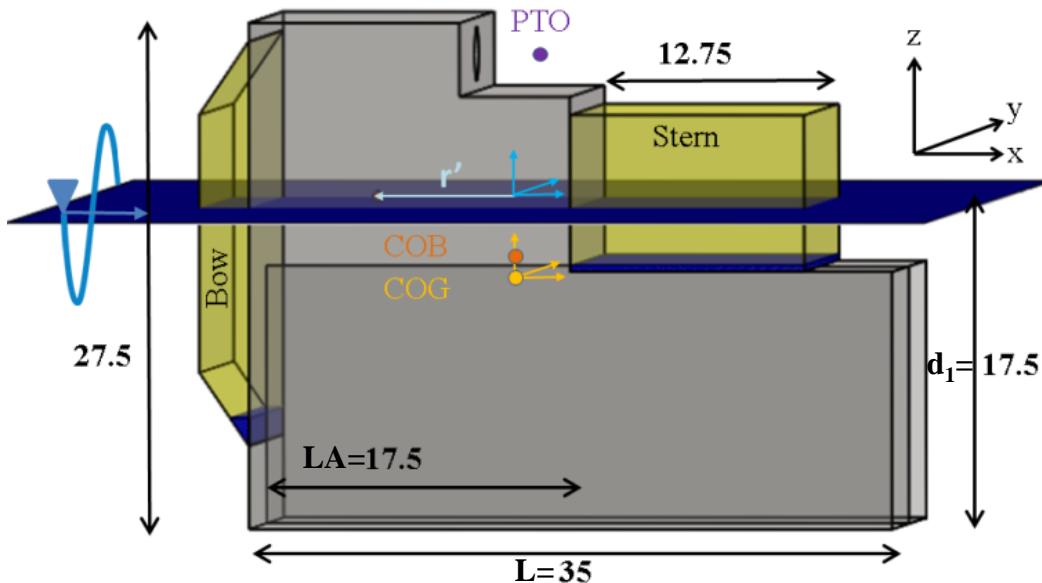


Backward Bent Duct Buoy

Profile

- Literature survey:

- $\lambda/L=4^1$ $L=35[m]$
 - 9[sec] $\lambda=140[m]$
- $L/LA=2.0^2$ $LA=17.5[m]$
- $d_1/LA=1.0^2$ $d_1=17.5[m]$
- $d_2/LA=0.2^2$ $d_2=3.5[m]$
- $L/B=1.3^{1,3}$ $B=27[m]$
 - Selected relationship based on most used in literature



Structural Design

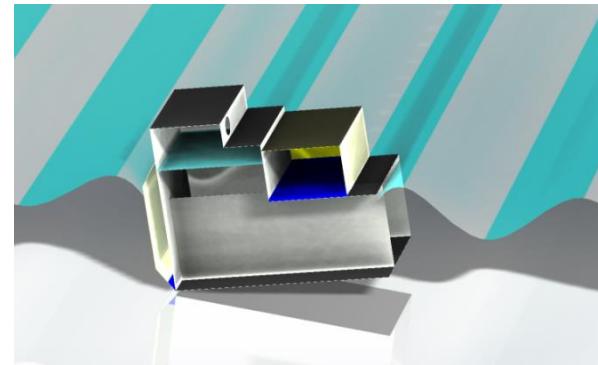
- Entire structure built to withstand hydrostatic pressure at 23.5 m
- Ballast chosen for stability
- Buoyancy chambers sized to support weight and obtain desired natural resonances

Displaced Mass [kg]	2,024,657		
Structural Mass [kg]	1,808,944		
Bow Ballast Mass [kg]	22,072		
Stern Ballast Mass [kg]	123,641		
Power Conversion Mass [kg]	70,000		
COG (x,y,z) [m]	0.00	0.00	-4.29
COB (x,y,z) [m]	0.00	0.00	-3.31
Free Surface Center (x,y,z) [m]	-5.12	0.00	0.00
Radius of Gyration at COG [m]	x	12.53	0.00
	y	0.00	14.33
	z	0.00	0.00
			14.54

Optimal Resistive Control of OWCs



Grounded



Floating

Power Absorption

- Wave activated water column

Power Conversion

- Pressure and volume in the air chamber
 - Wave activated motions linked through power conversion chain.

Coupled Power Absorption

- Wave activated water column
- Wave activated structure

Power Conversion

- Pressure and volume in the air chamber
 - **Coupled** wave activated motions linked through power conversion chain.



Modeling the Pressure Distribution of a Floating OWC

Approximation

- **Generalized Modes**

- Approximates full solution with user defined number of higher order modes

Explicit Solution

- **Free Surface Radiation Potential**

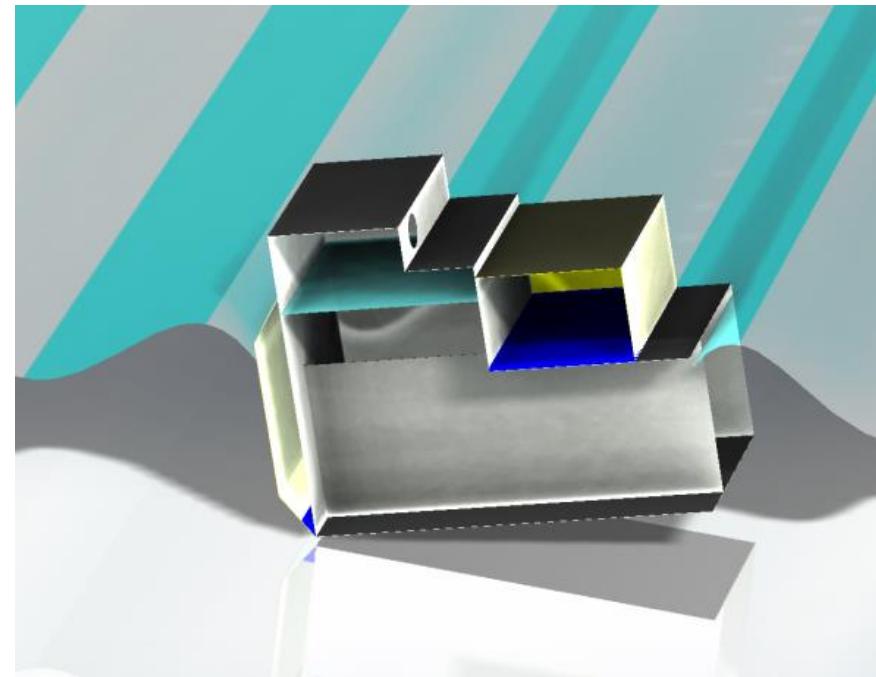
- Solves explicitly for the full velocity potential of i oscillating structures with k internal free surfaces.

$$\hat{\phi} = \hat{\phi}_o + \hat{\phi}_d + \sum_{ij} \varphi_{ij} \hat{u}_{ij} + \sum_k \varphi_k \hat{p}_k$$

Implicit Solution

- **Reciprocity Relations**

- Derive all of the free surface parameters from the oscillating structure potential
 - Requires an array of field points to define the internal free surface



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Modeling the Pressure Distribution of a Floating OWC

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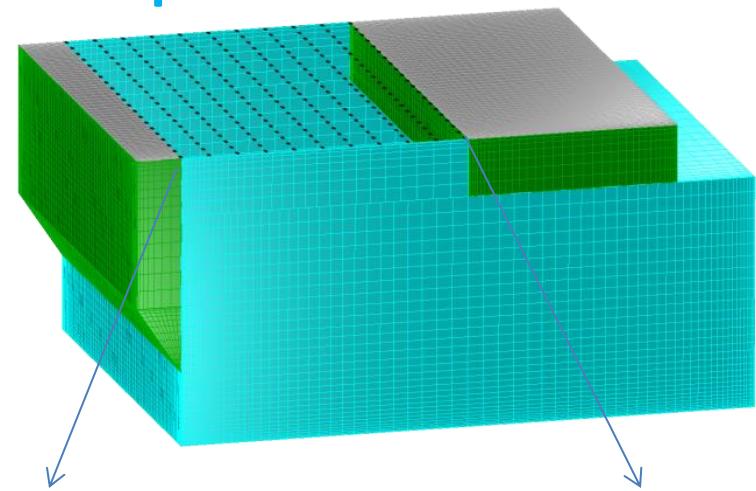
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Implicit Solution

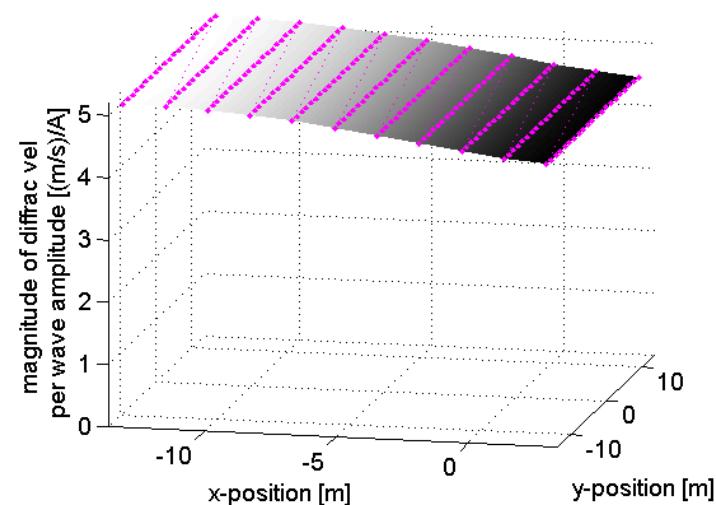
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Device Representation in WAMIT



OWC free surface diffraction velocity values for $\omega=0.46$.



Modeling the Pressure Distribution of a Floating OWC

Approximation

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Explicit Solution

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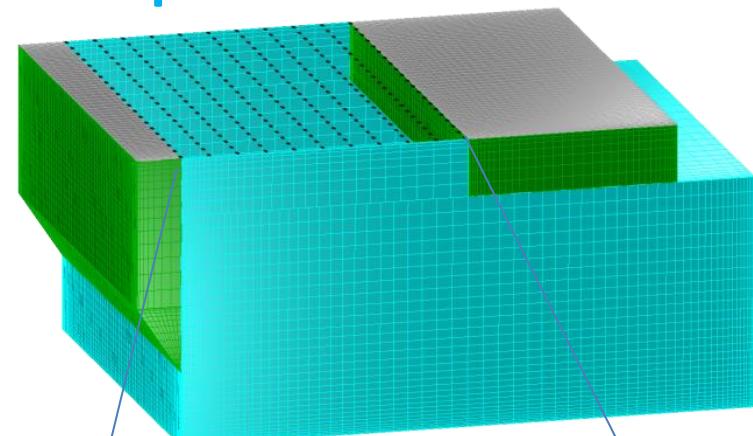
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Implicit Solution

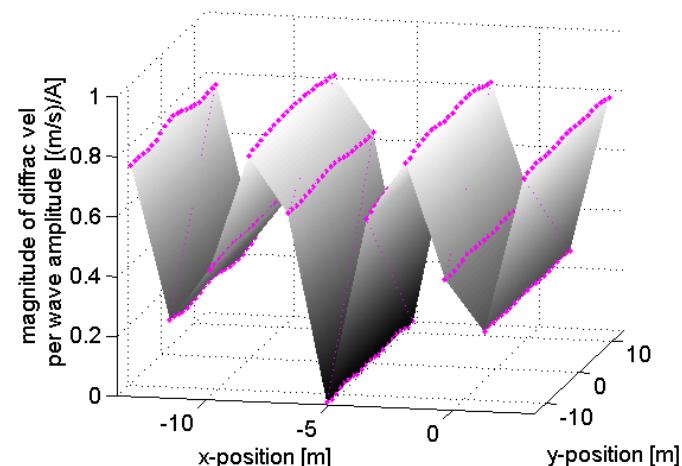
- **Reciprocity Relations**

- Derive all of the free surface parameters from the oscillating structure potential
 - Requires an array of field points to define the internal free surface

Device Representation in WAMIT



OWC free surface diffraction velocity values for $\omega=2.3$.



Device Hydrodynamics: Wave Structure Interactions



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Free Surface Hydrodynamics

Total Hydrodynamic Volume Flow:

$$Q_{TH} = qA - Yp - \sum_j H_j^u u_j$$

Excitation Volume Flow

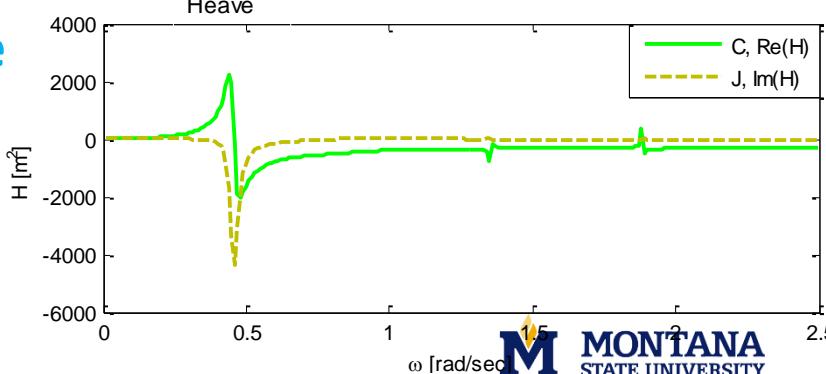
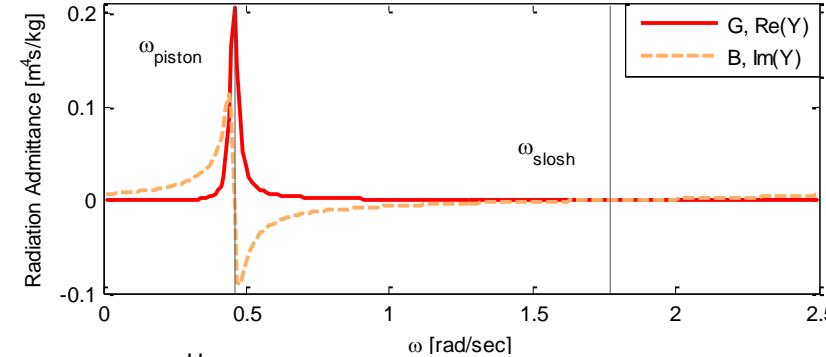
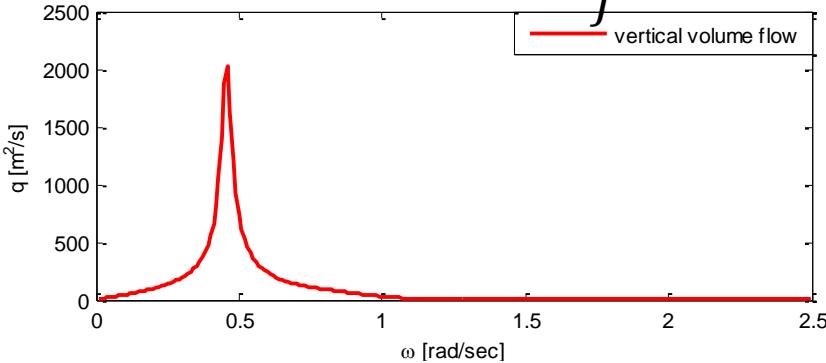
$$q = \frac{1}{A} \iint_S \frac{\partial(\phi_o + \phi_d)}{\partial z} dS$$

Radiation Admittance

$$Y = - \iint_S \frac{\partial \varphi}{\partial z} dS = G + iB$$

Coupling of the Oscillating Structure to the Oscillating Water Column

$$H_j^u = - \iint_S \frac{\partial \varphi_j}{\partial z} dS = C_j + iJ_j$$



Free Surface Hydrodynamics

Total Hydrodynamic Volume Flow:

$$Q_{TH} = qA - Yp - \sum_j H_j^u u_j$$

Excitation Volume Flow

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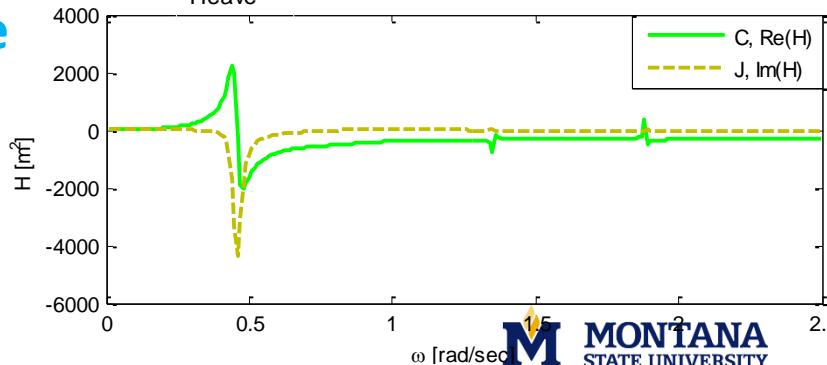
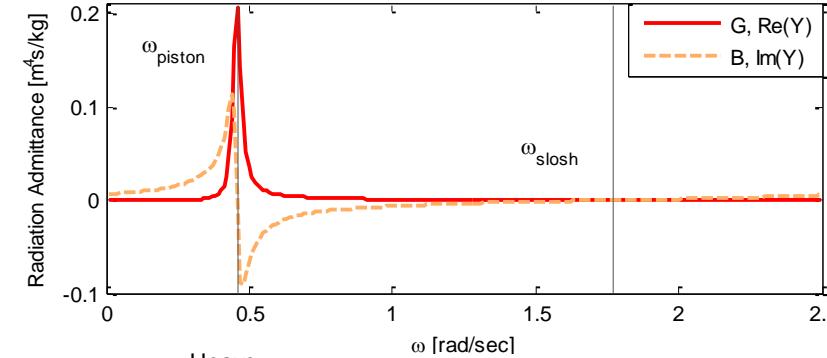
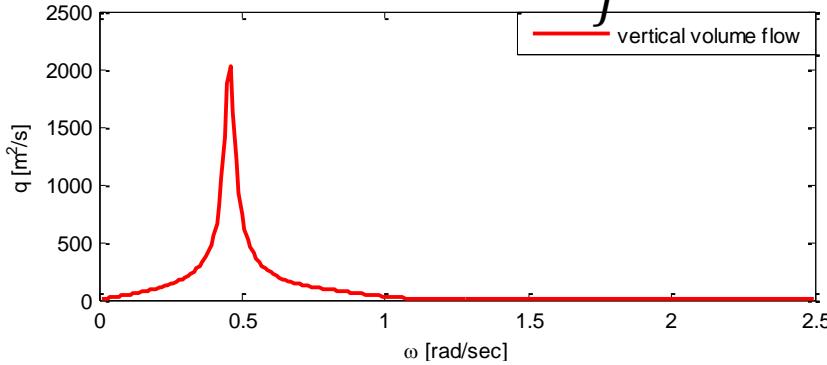
Radiation Admittance

- **Reciprocity Relation** $G = \frac{2k}{8\pi\rho g v_g} \int_0^\pi |q(\beta)|^2 d\beta$

- **Kramers-Kronig** $B(\omega) = -\frac{2\omega}{\pi} \int_0^\infty \frac{G(y)}{\omega^2 - y^2} dy$

Coupling of the Oscillating Structure to the Oscillating Water Column

$$H_j^u = - \iint_S \frac{\partial \phi_j}{\partial z} dS = C_j + iJ_j$$

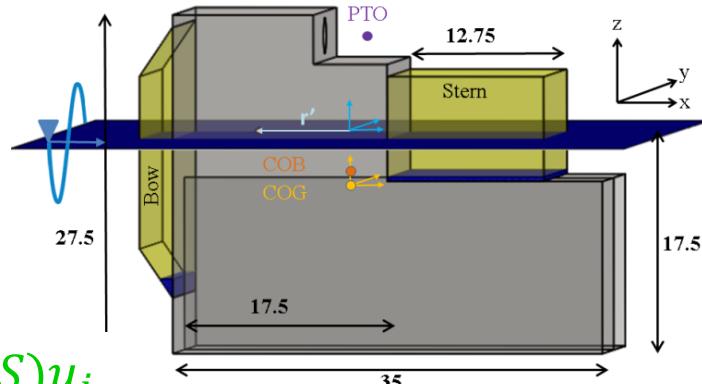


Hydrodynamically Coupled Relative Pressure

Relativizing the pressure

- Apply transformation vector to account for body oscillations

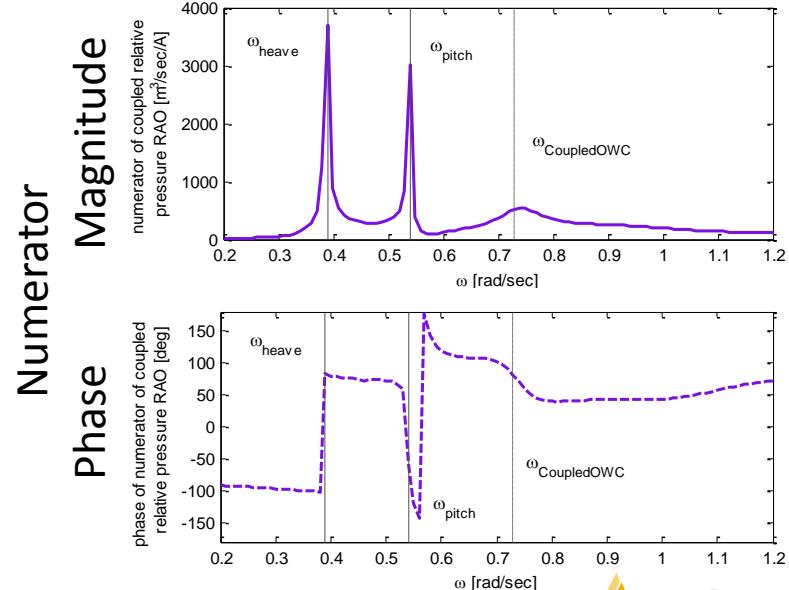
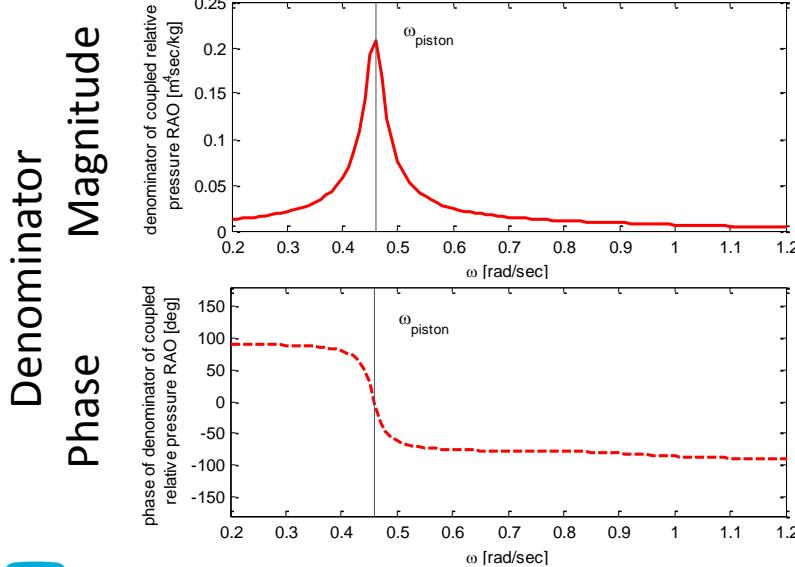
$$T = [0 \ 0 \ 1 \ 0 \ -r' \ 0]^T$$



Coupled relative pressure

$$\frac{p}{A} = \frac{q - \sum_j (H_j^u + T_j S) u_j}{Y}$$

Determining Resonances: Couple Relative Pressure



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Device Performance: Linear Wells Turbine Linking the Structures Oscillations to the OWC

Linked Governing Equations

Linked Matrix Representation

- Structure velocity
- Relative Pressure in the Air Column

$$\begin{pmatrix} f \\ q \end{pmatrix} A = \begin{pmatrix} \mathbf{Z}_i & -\mathbf{H}_i \\ \mathbf{H}_i^T & Y_i + \frac{1}{R_{load}} \end{pmatrix} \begin{pmatrix} u \\ p \end{pmatrix}$$

- Radiation Impedance of the Structure

$$\mathbf{Z}_i = \mathbf{b} + \mathbf{b}_{vis} + i\omega \left(\mathbf{m} + \mathbf{a} - \frac{(\mathbf{C} + \mathbf{K})}{\omega^2} \right)$$

- Relative Coupling Term

$$\mathbf{H}_i = \mathbf{H} + \mathbf{T} \mathbf{S}$$

- Radiation Admittance of the Compressible Air Column

$$Y_i = \left(G + \frac{1}{R_{vis}} \right) + i \left(B + \frac{\omega \mathbb{A}_o}{\gamma p_{atm}} \right)$$

Linked Total Volume Flow

$$Q_T = qA - Y_i p - \mathbf{H}_i^T \mathbf{u} = \frac{p}{R_{load}}$$



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Externally Applied Forces: Viscous Damping and Mooring

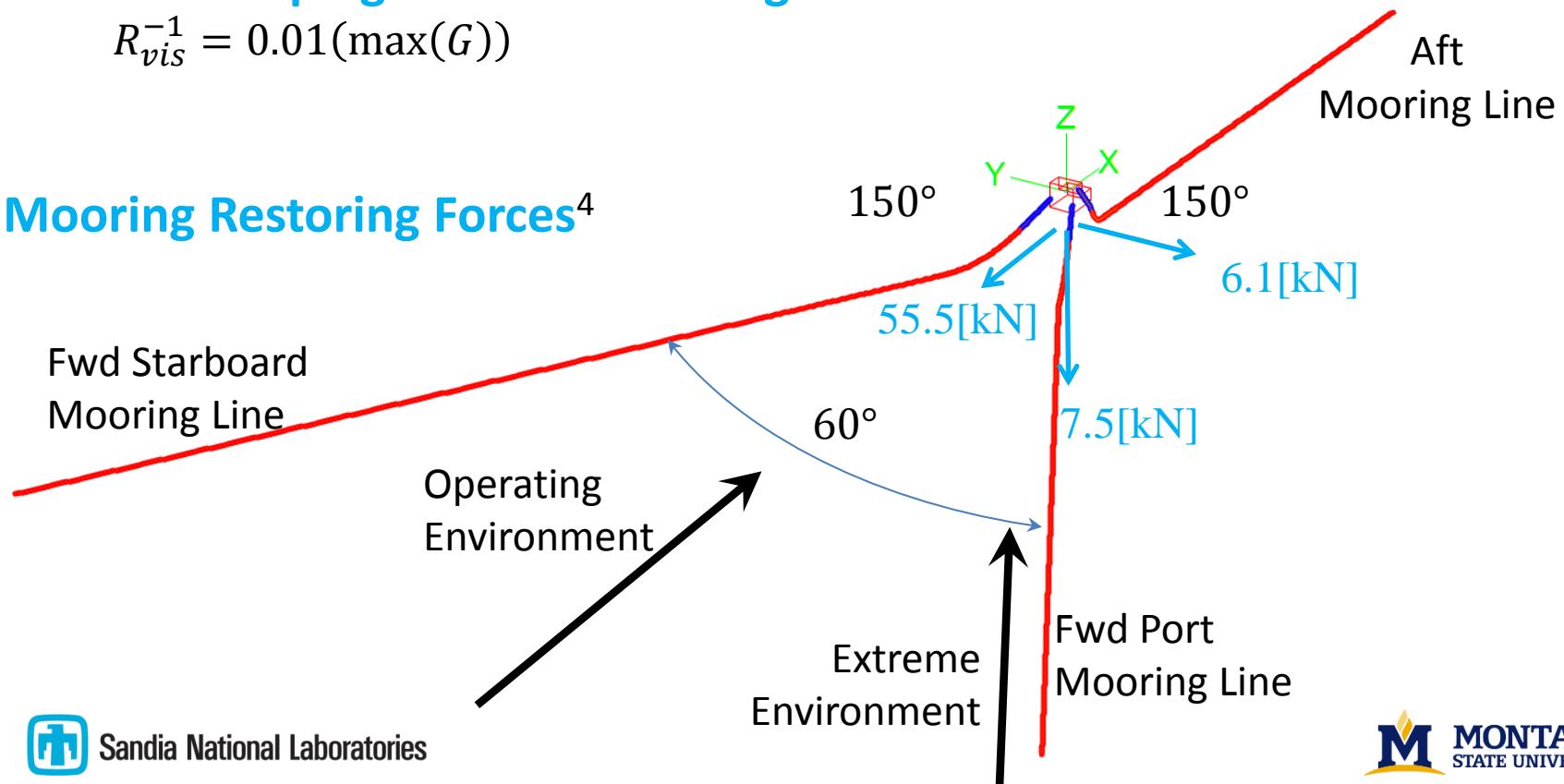
Viscous Damping on the Floating Structure

$$b_{vis} = 0.02\sqrt{M_{tot}c_{tot}}$$

Viscous Damping on the Oscillating Water Column

$$R_{vis}^{-1} = 0.01(\max(G))$$

Mooring Restoring Forces⁴



Optimal Resistive Damping: Derivation

Pneumatic Power

- $\langle P \rangle = \overline{p(t)Q_T(t)} = \frac{1}{2} \operatorname{Re}\{pQ_T^*\}$

Pneumatic Power In Monochromatic Waves

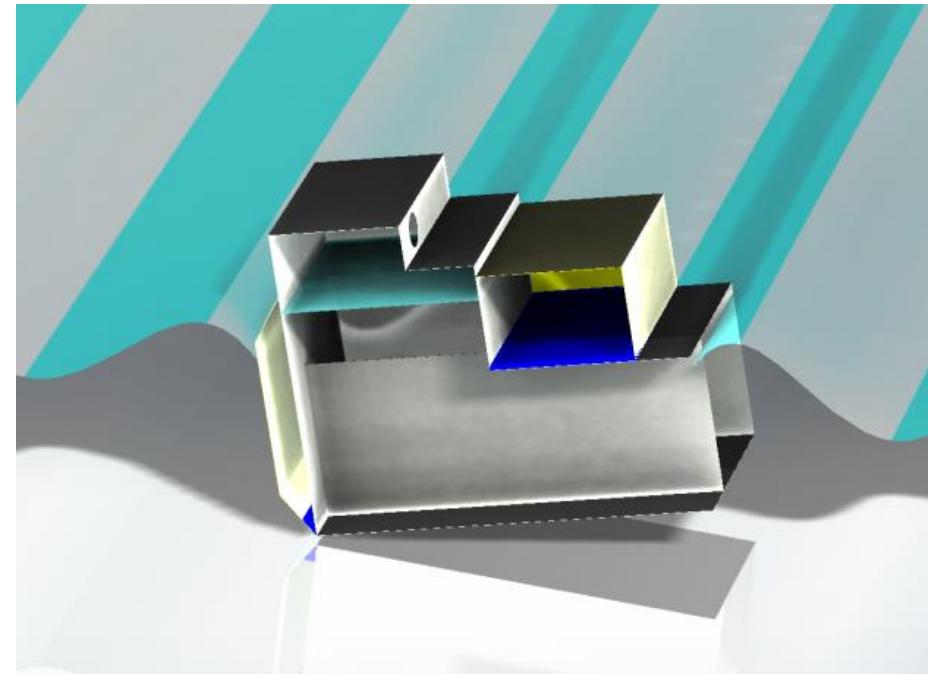
- $\langle P \rangle = \frac{1}{2} \frac{1}{R_{load}} |p|^2$

Optimization Condition

- $\frac{\partial \langle P \rangle}{\partial R_{load}} = 0$

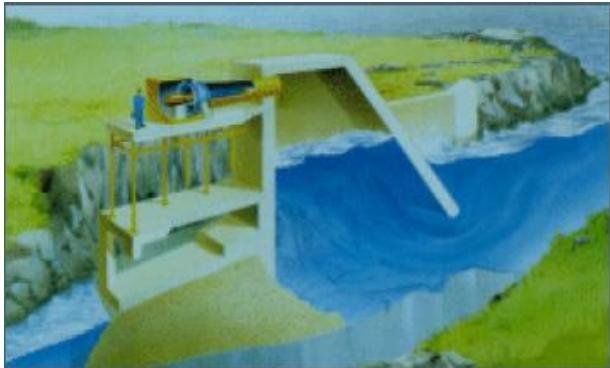
Optimal Resistive Damping for Hydrodynamically Coupled OWC

- $R_{l_{opt}} = \left(|Y_i + \mathbf{H}_i^T \mathbf{Z}_i^{-1} \mathbf{H}_i|^2 \right)^{-\frac{1}{2}}$

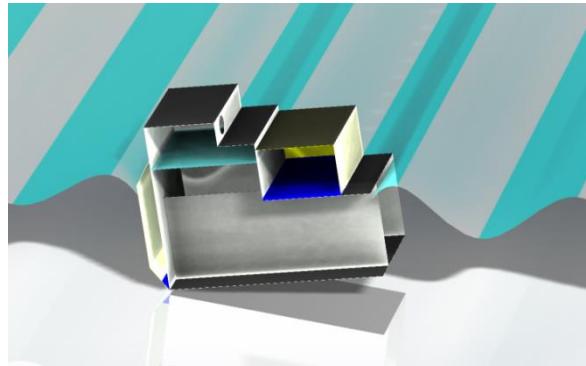


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Optimal Resistive Damping: with Wells Turbine



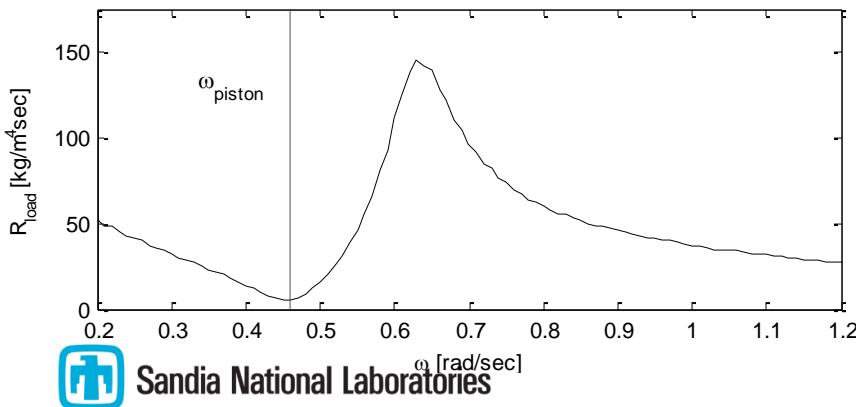
Grounded



Floating

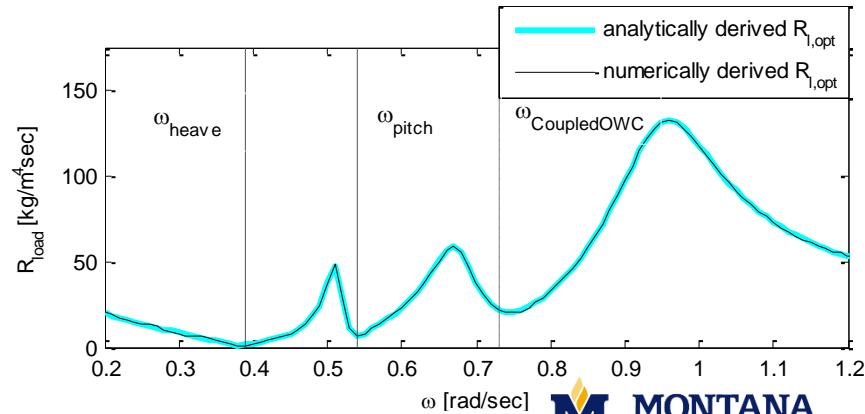
Optimal Resistive Damping

$$\bullet \quad R_{l,op} = (|Y_i|^2)^{-\frac{1}{2}}$$



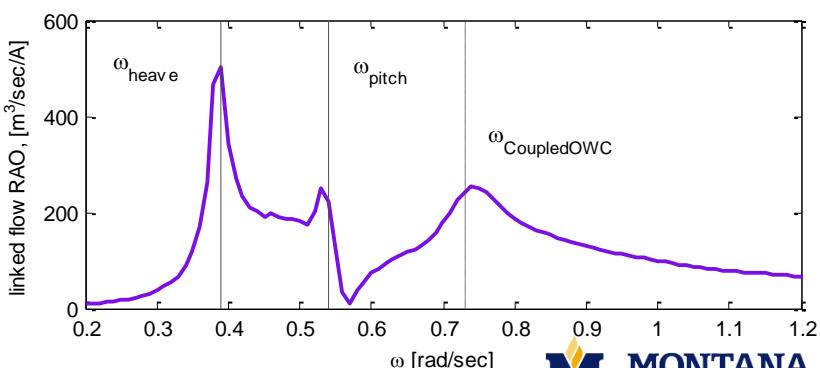
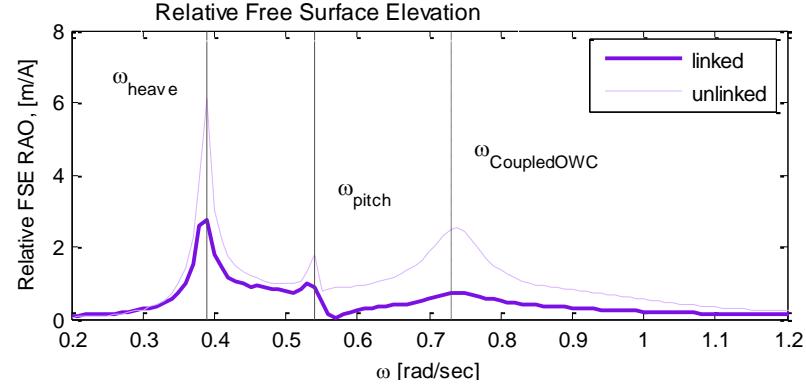
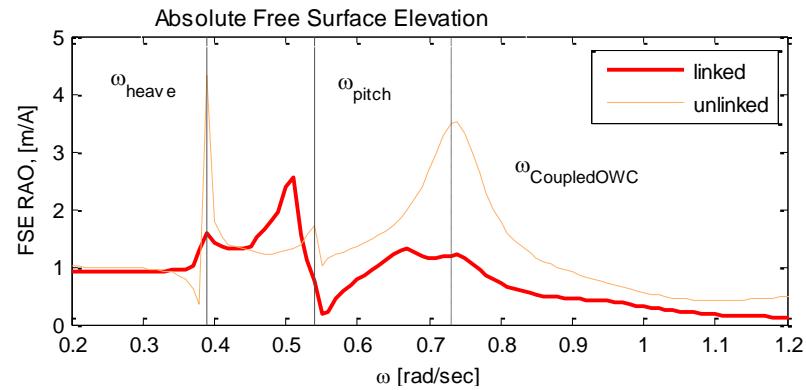
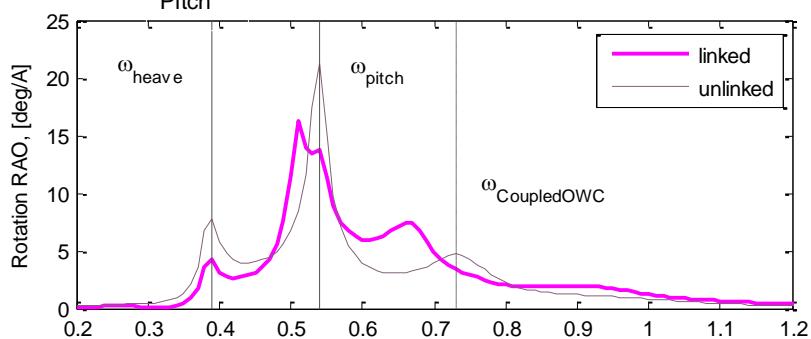
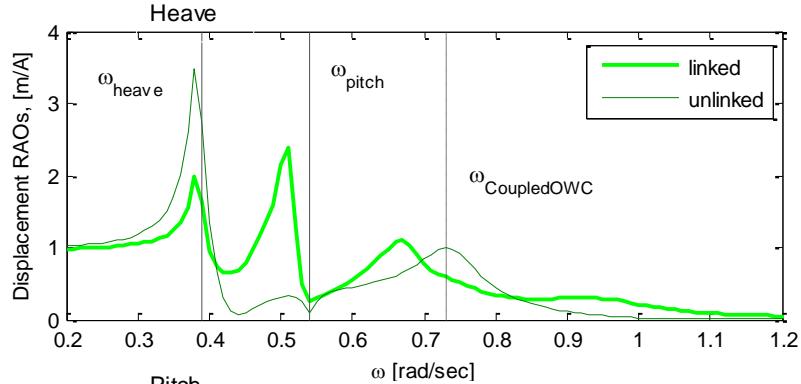
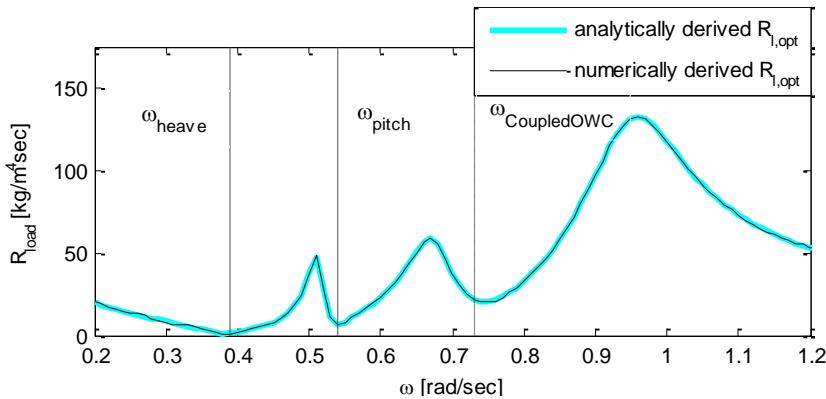
Optimal Coupled Resistive Damping

$$\bullet \quad R_{l,op} = \left(|Y_i + H_i^T Z_i^{-1} H_i|^2 \right)^{-\frac{1}{2}}$$



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Influence of Wells Turbine on Device Motion: Troughs

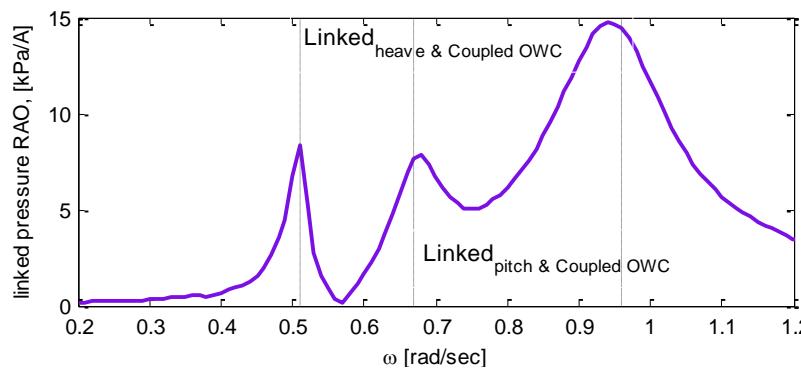
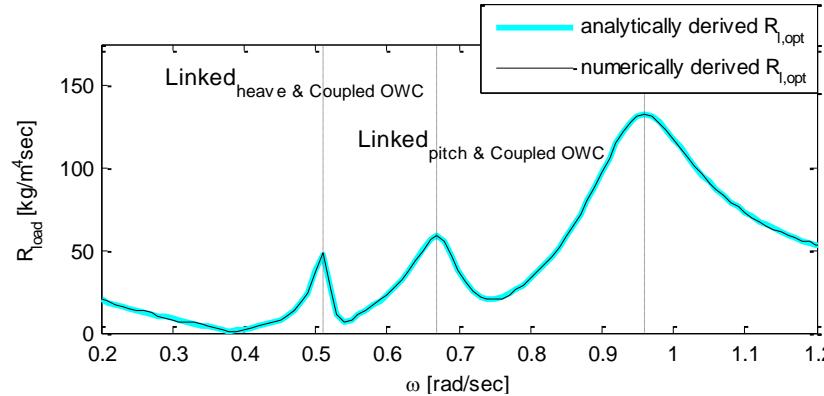


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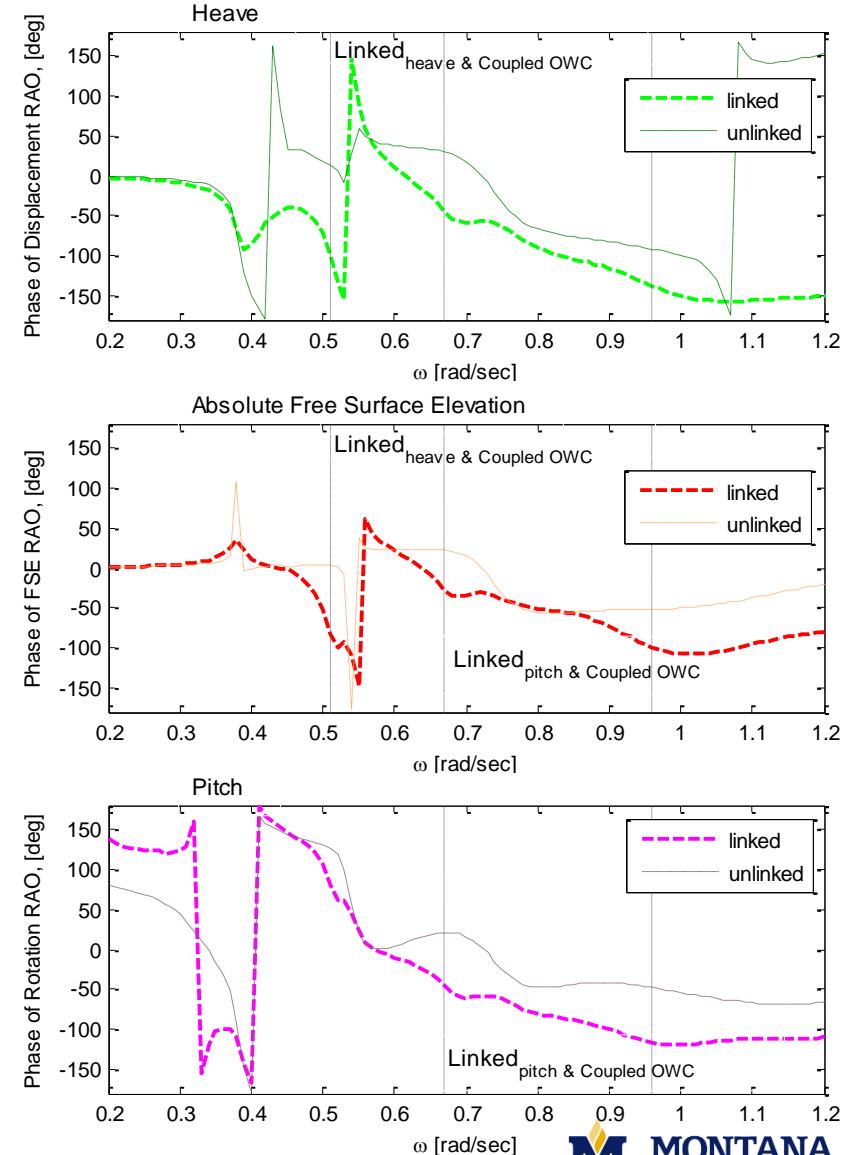
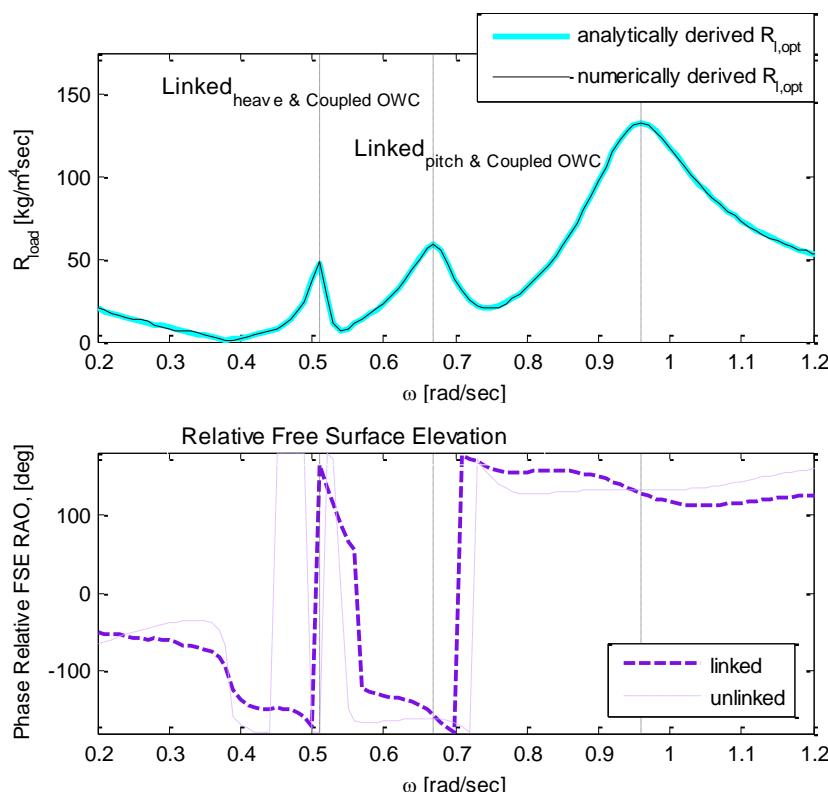


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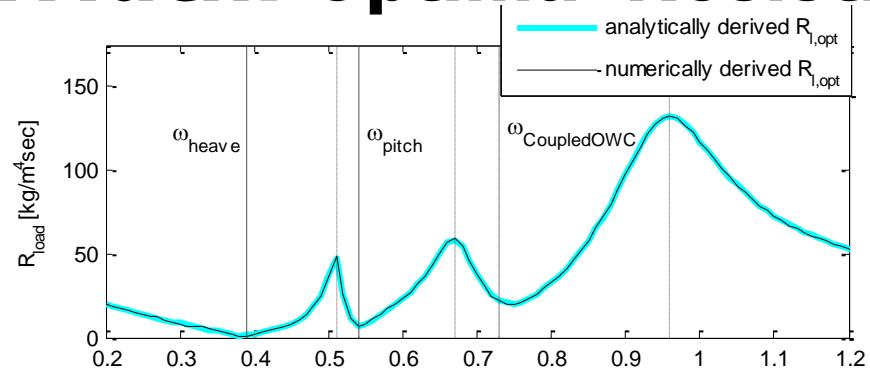
Influence of Wells Turbine on Device Motion: Peaks



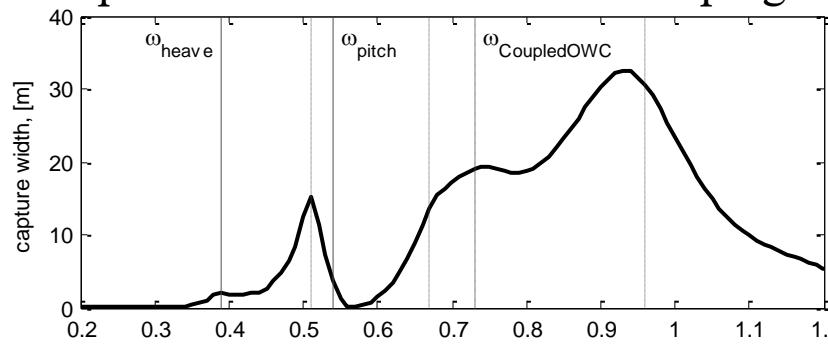
Influence of Wells Turbine on Device Motion: Understanding Peaks



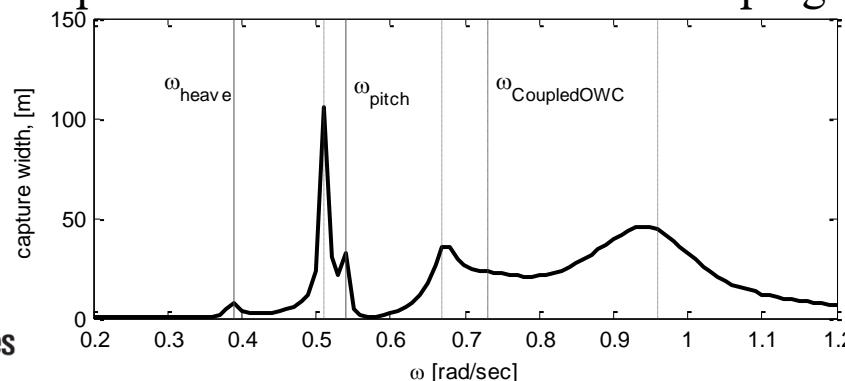
Influence of Wells Turbine on Device Capture Width: Optimal Resistive Damping



Capture Width With Viscous Damping

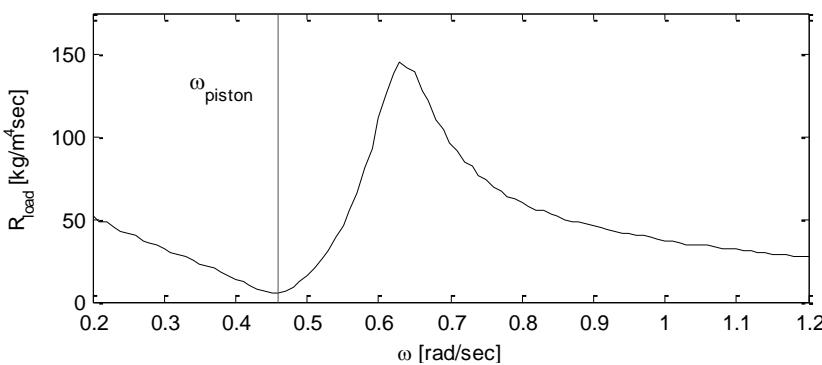


Capture Width *Without* Viscous Damping

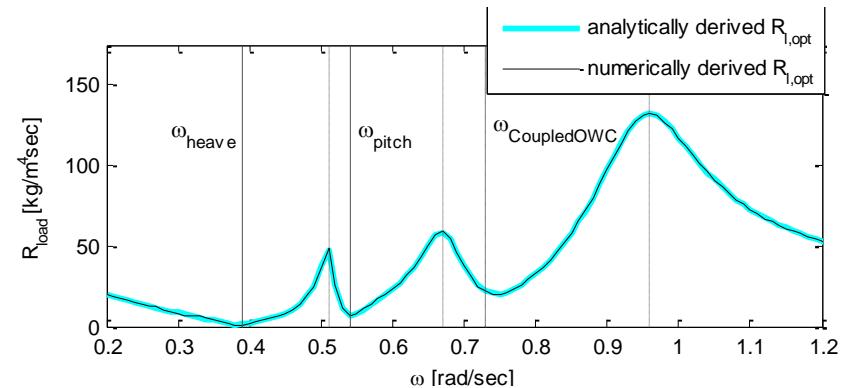


Comparison of Damping

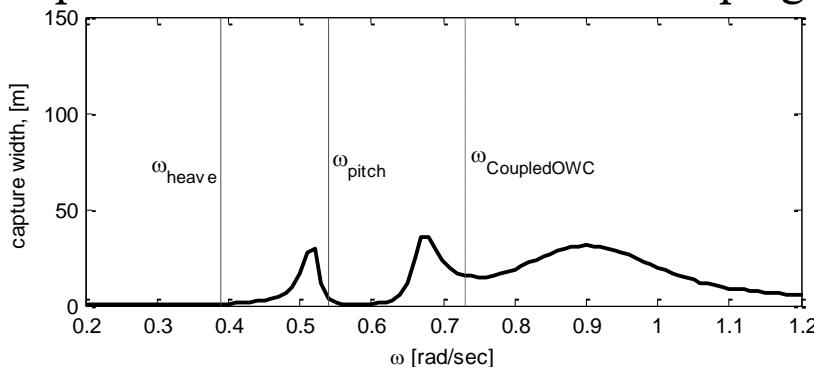
Applying Grounded $R_{l, \text{opt}}$ to Floating Device



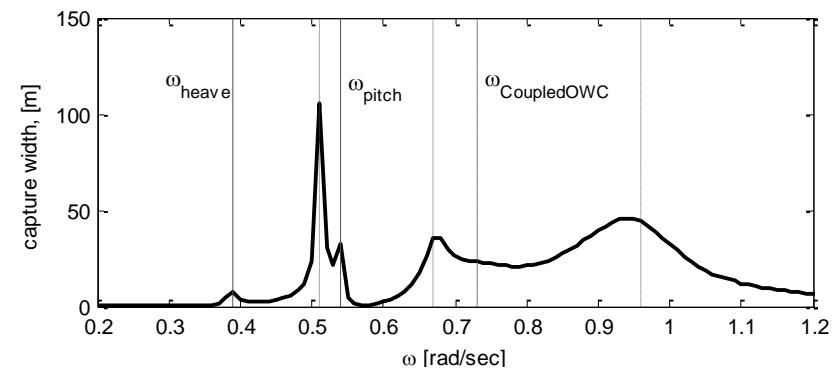
Applying Coupled $R_{l, \text{opt}}$ to Floating Device



Capture Width *Without* Viscous Damping



Capture Width *Without* Viscous Damping



Conclusions

Grounded vs. Floating OWCs

- Floating OWCs must be modeled such that BOTH the wave activated water column and the wave activated structure are included

Coupled Water Column Resonance for Floating OWCs

- Floating OWCs are coupled between the wave activated column and the wave activated structure.
- This coupling results in a new resonance location for the coupled relative free surface in the device.

Optimal Resistive Control for Floating OWCs

$$\bullet \quad R_{l_{opt}} = \left(\left| Y_i + \mathbf{H}_i^T \mathbf{Z}_i^{-1} \mathbf{H}_i \right|^2 \right)^{-\frac{1}{2}}$$

Controls' Influence on Device Motion and Power

- Applying $R_{l_{opt}}$ to the floating OWC will preserve the natural resonances of the coupled system.
- This optimization clearly takes advantage of the coupling between the OWC and the structure.

Next Steps

Experimental Verification at HMRC

- Determine realistic viscous damping values
- Verify the shape of the optimal R_{load} curve in monochromatic waves
 - A linear scaled PTO has been designed
- Verify the predicted power absorption of the device in Northern California



Incorporation of Wells Turbine

- ARL at PennState is working with SNL to size the Wells Turbine and power electronics that should be associated with this design
 - Incorporation will include sea state – by – sea state derived efficiencies

Economic Model

- NREL will be working to determine the LCOE of this device
 - The cost model will incorporate sensitivity analysis on some of the estimates.



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References

- [1] Y. Imai, K. Toyota, S. Nagata, T. Setoguchi, and M. Takao, "An Experimental Study on Generating Efficiency of a Wave Energy Converter" Backward Bent Duct Buoy," presented at the EWTEC.
- [2] M. Suzuki, T. Kuboki, S. Nagata, and T. Setoguchi, "Numerical Investigation of 2D Optimal Profile of Backward-Bent Duct Type Wave Energy Converter," *J. Offshore Mech. Arct. Eng.*, vol. 133, no. 4, pp. 041602–8, Nov. 2011.
- [3] D. Hong, S. Hong, and S. Hong, "Numerical study on the reverse drift force of floating BBDB wave energy absorbers," *Ocean engineering*, vol. 31, no. 10, pp. 1257–1294, 2004.
- [4] D. Bull and P. Jacob, "Methodology for creating nonaxisymmetric WECs to screen mooring designs using a Morison Equation approach," in *OCEANS '12. "Harnessing the Power of the Ocean". Proceedings*, Hampton Roads, VA, 2012, pp. 1–9



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Thank you.

Sandia National Laboratories is a multiprogram laboratory managed and operated by Sandia Corporation, a wholly owned subsidiary of Lockheed Martin Corporation, for the U.S. Department of Energy's National Nuclear Security Administration under contract DE-AC04-94AL85000.

This research was made possible by support from the Department of Energy's Energy Efficiency and Renewable Energy Office's Wind and Water Power Program. The research was in support of the Reference Model Project.

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