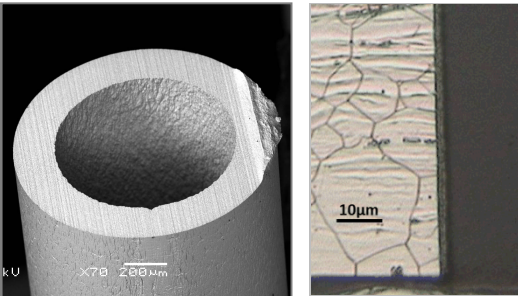


# Design & Direct Machining of Mesoscale Elastic Structures



ICOMM, March 11, 2012

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Sandia National Laboratories  
Materials Science & Engineering Center



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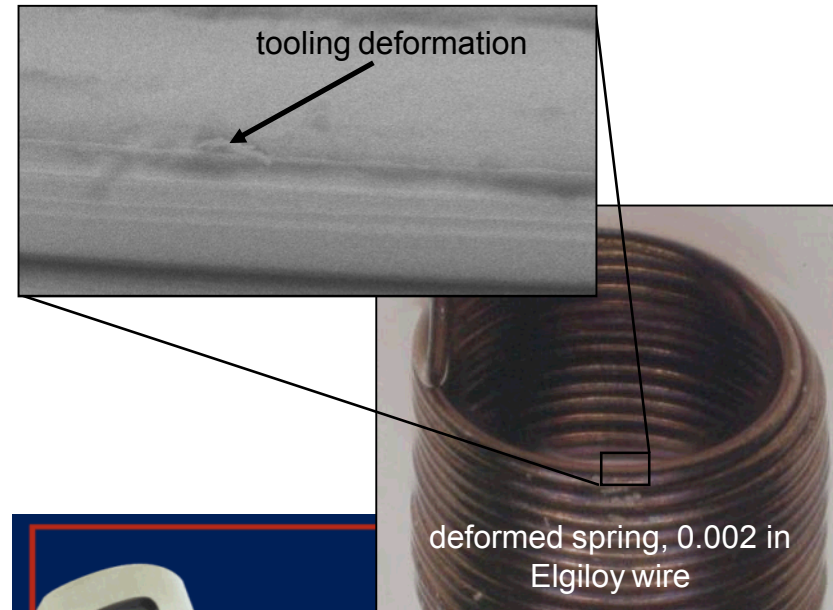
# Outline

- Motivation
  - Why directly machine mesoscale elastic structures?
- Design
  - What structure best meets performance requirements & leverages direct machining?
- Fabrication
  - How are mesoscale structures fabricated?
- Performance
  - How have prototypes performed?



# Motivation

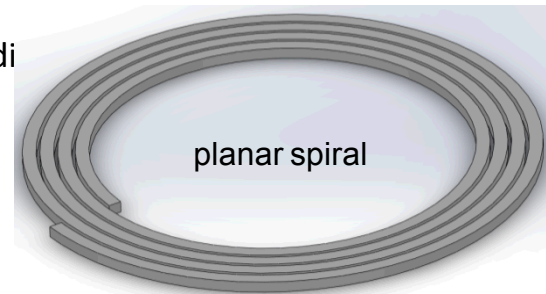
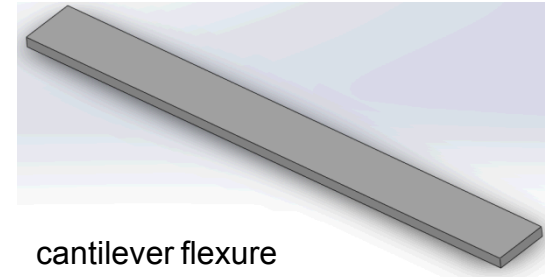
- Coil winding meso springs (wire dia.  $< 100 \mu\text{m}$ ) presents barriers to miniaturization
  - dimensional errors, tooling damage & material variability drive uncertainties
  - tolerance margins typically exceed 20%
  - design impacts counter miniaturization (increasing volume, force, power, stresses)
  - spring geometry limited by wire cross-section & material strengths
- Directly machined elastic structures present an alternative
  - reduce uncertainty margins
    - improve process accuracy
    - eliminate plasticity in processing
  - expand design space
    - unique design configurations
    - geometry driven by material form & fabrication
  - precedence
    - macro scale springs
    - medical stents



# Spring Structure Design

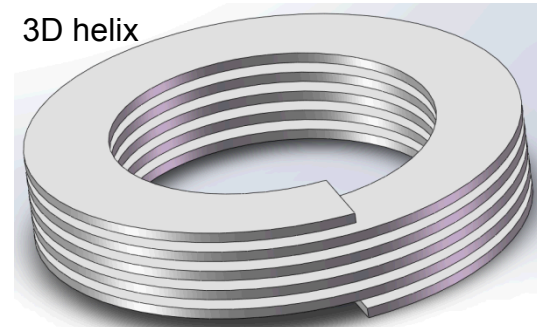
## Objectives

- explore design space & demonstrate mesoscale viability
- investigate optimization & trade-offs for performance & processing
- understand accuracy of macro scale models to meso springs
- equivalent spring requirements
  - single start, coil wound extension spring w/50  $\mu\text{m}$  wire diameter
  - specifications
    - stiffness from force (F1, F2) @ deflection (d1, d2)
    - working volume (min ID, max OD)



## Forms examined

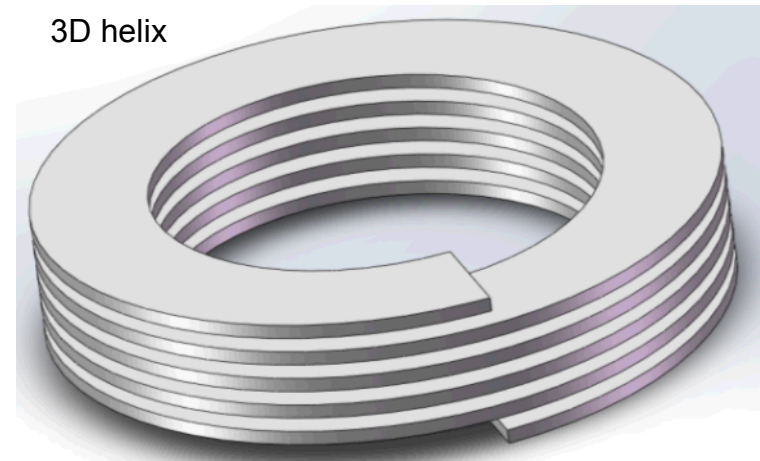
- normalized output for equivalent stiffness
  - cantilever beam flexure: unacceptable footprint
  - planar spiral spring: largest footprint, lowest natural frequency
  - 3D helical spring: best balance across requirements



form	element length	area	volume	$\omega_n$	deflected $\sigma$	ave.
cantilever	<u>1</u>	8.18	<u>1</u>	<u>1</u>	<u>1</u>	2.44
spiral	2.74	46.36	1.97	0.63	1.45	10.82
helix	3.73	<u>1</u>	1.56	0.84	1.42	<u>1.78</u>

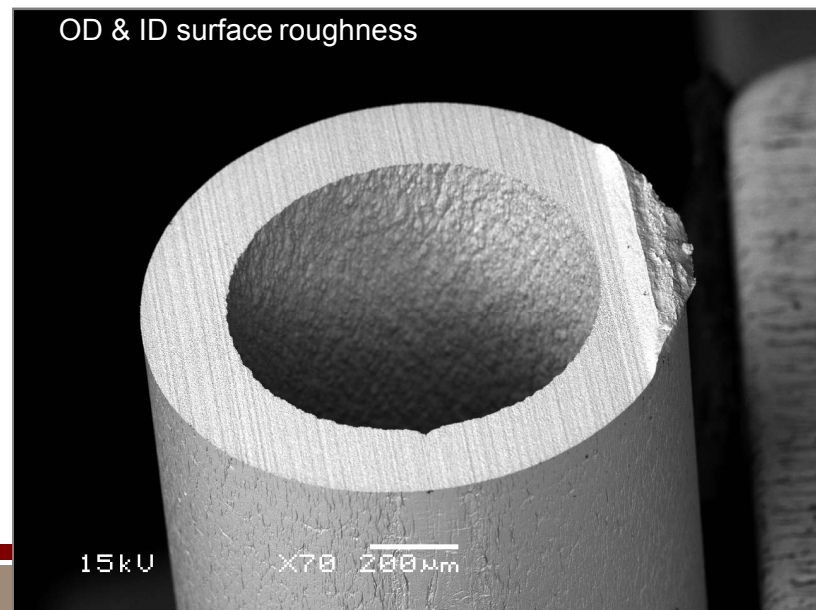
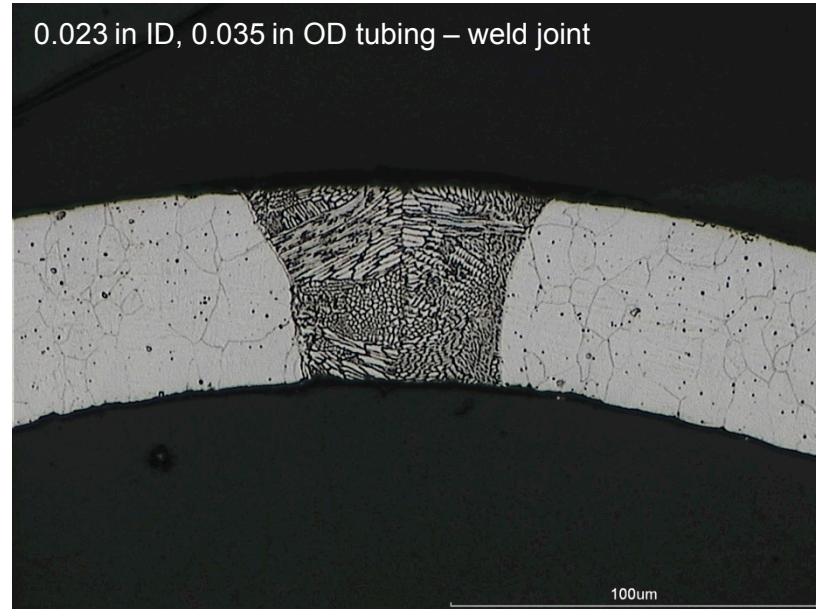
# Material Selection

- Mesoscale requirements
  - isotropic mechanical properties
  - large elastic modulus
  - high yield strength
  - small grains to insure multiple grains across spring coils
  - no microplasticity, ratcheting, or low-cycle fatigue issues at operating stresses
  - long-term environmental stability
    - high temperature
    - corrosion
  - form amenable to desired spring geometry
- Initial prototype material
  - 304L stainless steel hypodermic needle tubing



# 304L Stainless Hypodermic Tubing

- Readily available
- Well documented properties
  - isotropic
  - moderate strength (~75 ksi)
  - controllable microstructure
- Challenges
  - discrete tube sizes limit design space
  - fatigue vulnerabilities
    - rolled stock welded to form tube
    - ID surface roughness @ location of max stress



# 3D Helical Design

- Governing closed form equations
  - rectangular coil stiffness (Young)

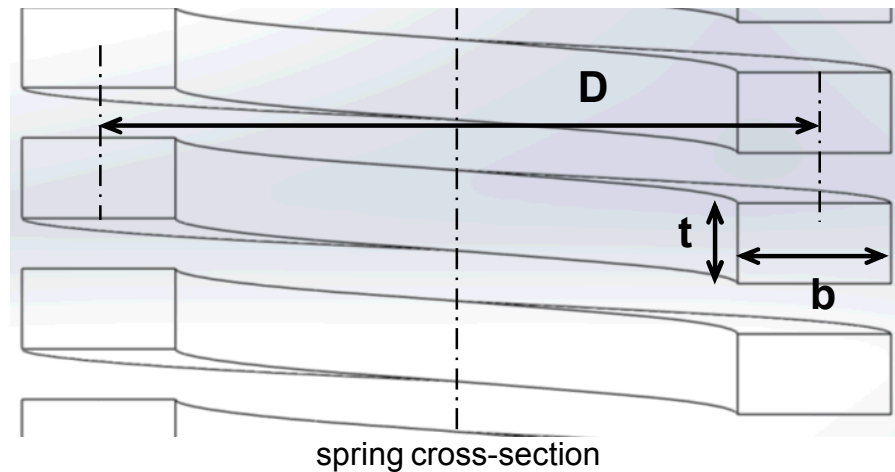
$$k = \frac{4G \cdot t^4}{3\pi \cdot D^3 \cdot N} \left( \frac{b}{t} - 0.627 \cdot \tanh \frac{\pi \cdot t}{2b} + 0.004 \right)$$

- stress under load (Wahl)

$$\tau = \frac{P \cdot D(3t + 1.8b)}{2t^2 \cdot b^2} \left( 1 + \frac{1.2t}{D} + \frac{0.56t^2}{D^2} + \frac{0.5t^3}{D^3} \right)$$

- 1st mode resonance

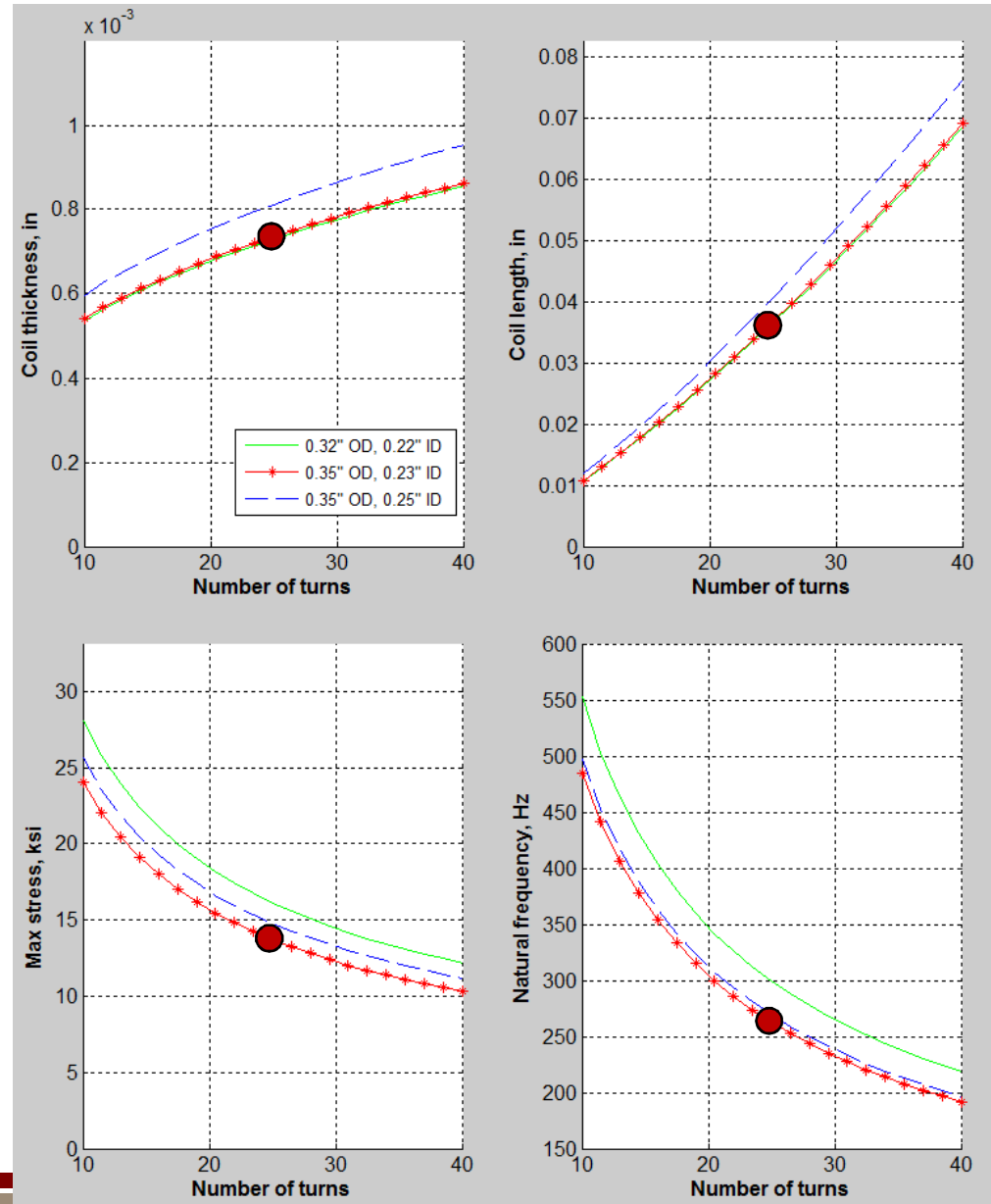
$$\omega_n = \frac{1}{2\pi} \sqrt{\frac{k}{\rho \cdot t \cdot b \cdot \pi D \cdot N}}$$



- Used for “optimization” design curves
  - coil thickness, spring length, working stress & natural frequency solved numerically across range of tubing diameters & number of coil turns
  - expect to capture trends for meso structures

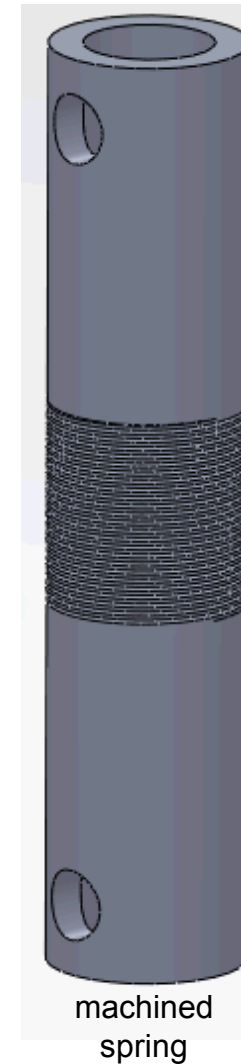
# Design Optimization

- Design balanced
  - working stress
  - machinable coil thickness
  - minimum active coil length
  - maximum 1st mode resonance
- Trends
  - increasing tubing OD & thickness
    - reduces stress
    - reduces natural frequency
    - increases fabrication difficulty
  - discretization of tube sizes limits design space



# Prototype Design

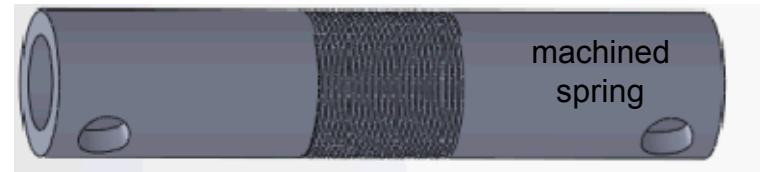
- Description
  - 304L hypodermic tubing
  - 25 coils
  - 0.023 in ID, 0.035 in OD, 0.00074 in coil thickness
  - 13.7 ksi working stress
  - 263 Hz natural frequency
  - closed form agreement to FEA model within 10%
- Wound spring comparison
  - natural frequency decreased 25%
  - end geometry
    - stronger tangs
    - enables reduced length (i.e. volume)
    - enables improved and/or integrated assembly



# Fabrication

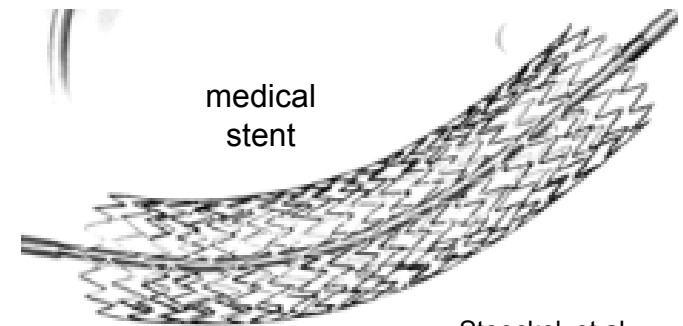
- Meso scale spring requirements

- “traditional” machining inadequate for structural elements  $< 50 \mu\text{m}$
- process accuracy =  $1\text{-}5 \mu\text{m}$
- minimum feature “resolution” =  $10\text{-}20 \mu\text{m}$
- minimal change in material microstructure or introduction of surface effects
- candidate processes
  - pulse laser machining
  - $\mu$ -wire EDM



- Stent differences

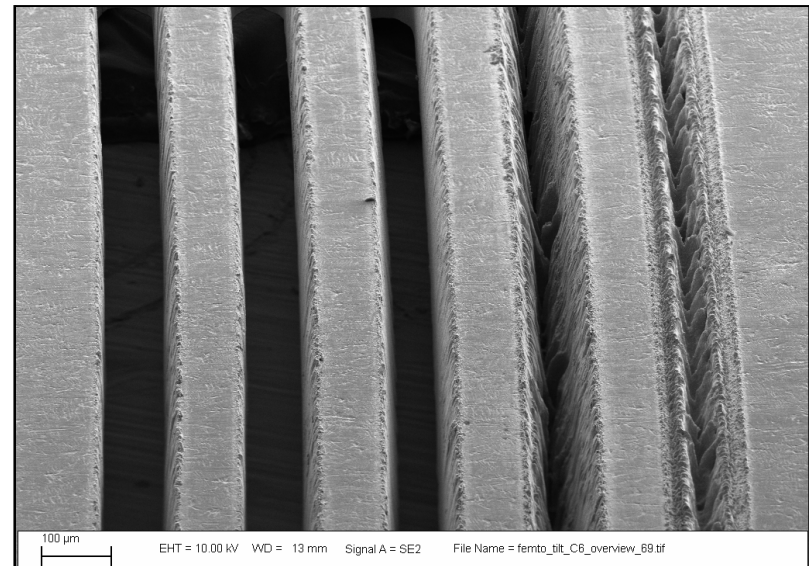
- provide primarily radial motion
- require smaller range of motion
- smaller aspect ratio cuts & features



Stoeckel, et al

# Material Impacts

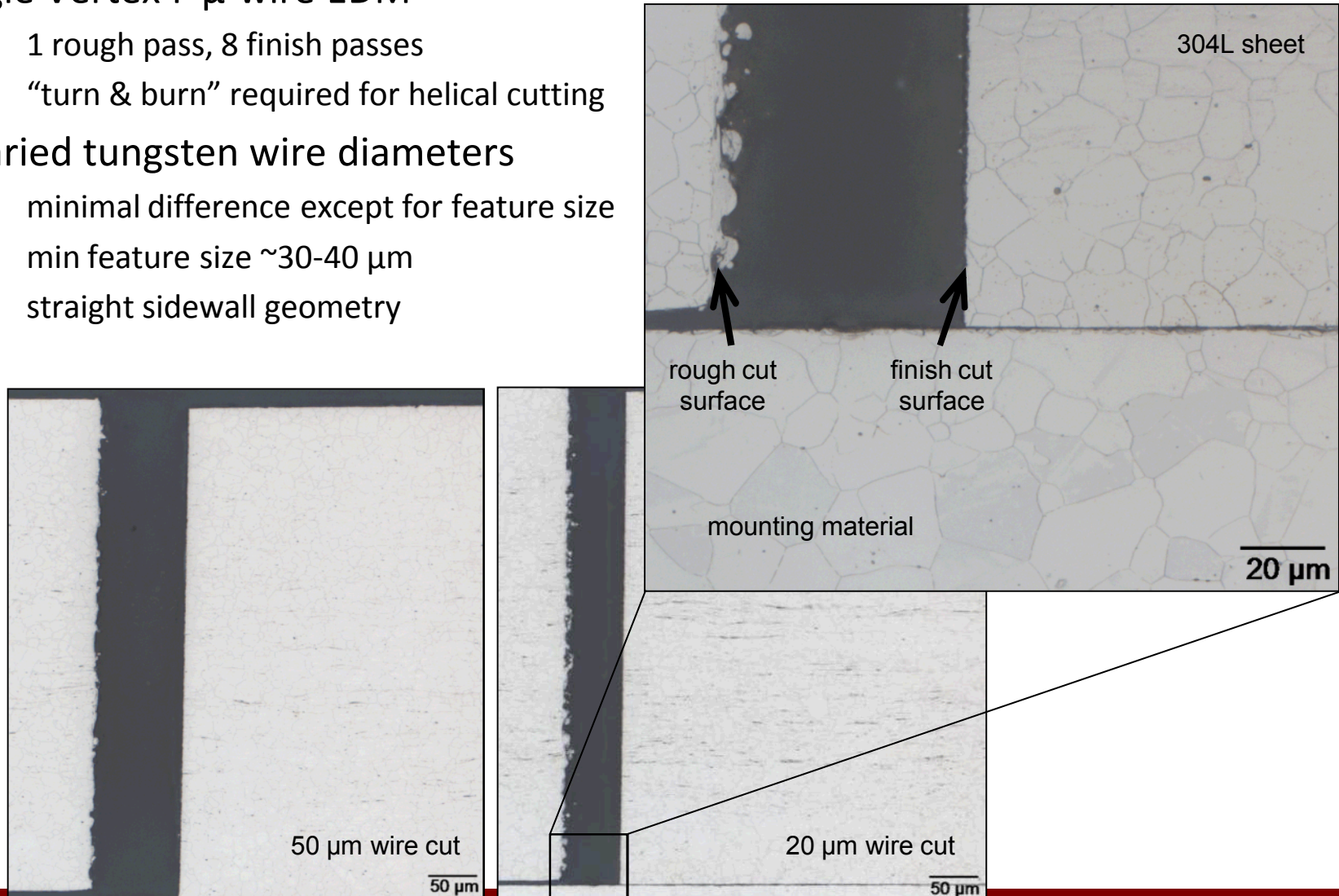
- Process damage is critical for mesoscale structures
  - surface layers & microstructure can approach feature sizes
    - subtractive processes generate heat affected zones, recast, internal stresses, etc.
  - design models assume bulk properties & continuum material behavior
- Process experiments
  - cut 250  $\mu\text{m}$  thick 304L sheet
    - fs & ns pulsed laser machining
    - $\mu$ -EDM
    - variable process parameters
  - investigating
    - heat affected zone
    - cut geometry
    - surface quality



100 fs, 1 kHz, 0.49 mJ, 20 passes, 0.1-5 mm/sec

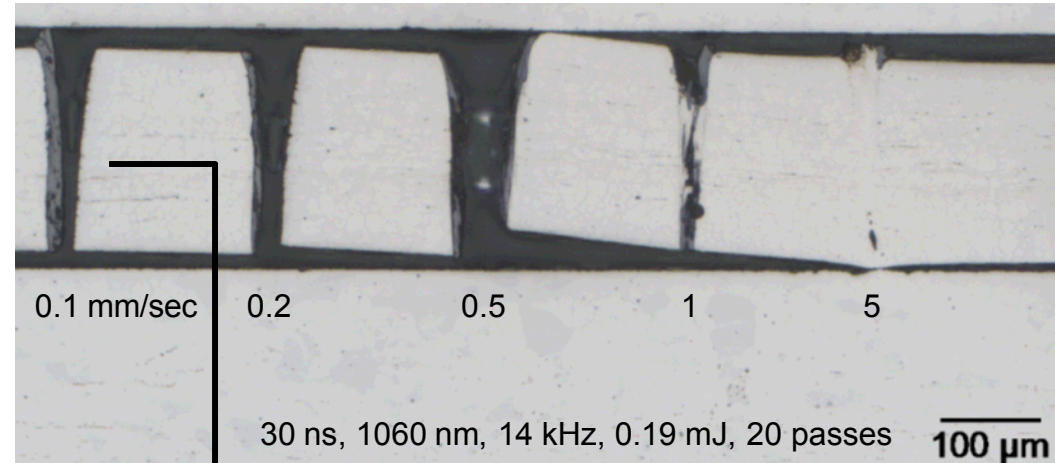
# $\mu$ -Wire EDM

- Agie Vertex F  $\mu$ -wire EDM
  - 1 rough pass, 8 finish passes
  - “turn & burn” required for helical cutting
- Varied tungsten wire diameters
  - minimal difference except for feature size
  - min feature size  $\sim 30\text{-}40\ \mu\text{m}$
  - straight sidewall geometry

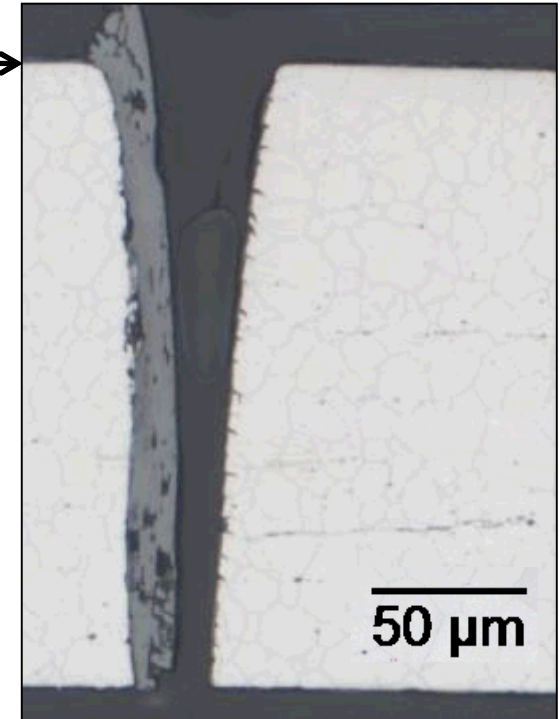
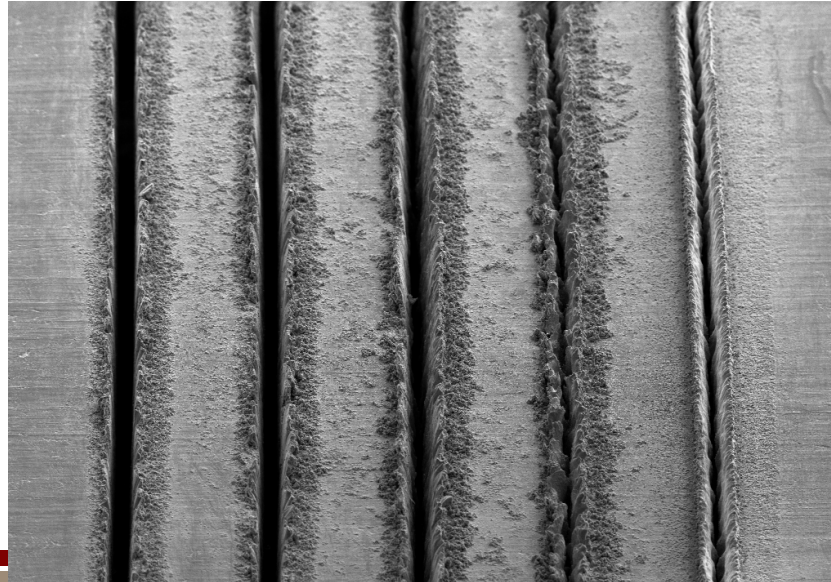


# Ns Pulsed Laser

- Parameters examined
  - power, pulse rate
  - feed rate, number of scans
  - wavelength
- Damage
  - significant HAZ & recast
  - material distortion due to stresses
  - unacceptable for mesoscale structures



30 ns, 800 nm, 1 kHz, 0.466 mJ, 20 passes, 0.1-5 mm/sec



# Fs Pulsed Laser

- Fs pulses provide better cutting
  - less thermal damage & recast
  - straight sidewalls
  - high power & slow velocity improve cut depth & quality
  - extra cuts cleanup surface
- Selected prototype process



0.1 mm/sec

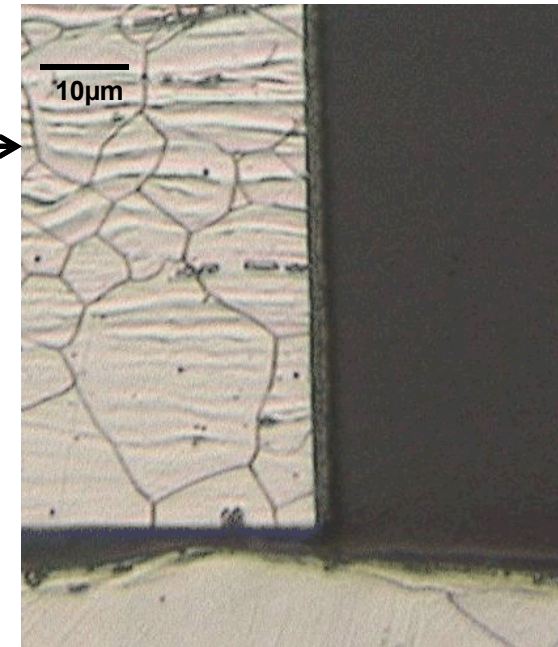
0.2

0.5

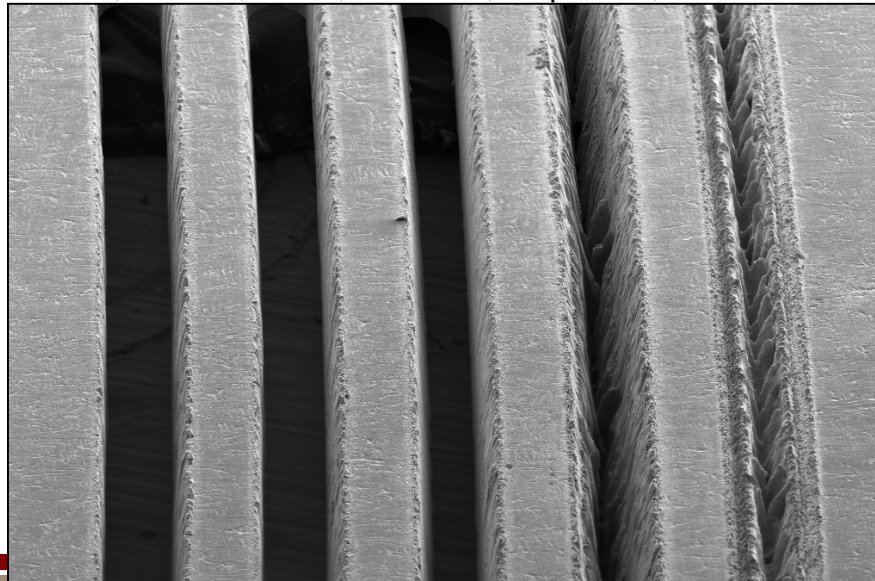
1

5

100 fs, 800nm, 1 kHz, 0.493 mJ, 20 passes



100 fs, 800 nm, 1 kHz, 0.493 mJ, 20 passes, 0.1-5 mm/sec

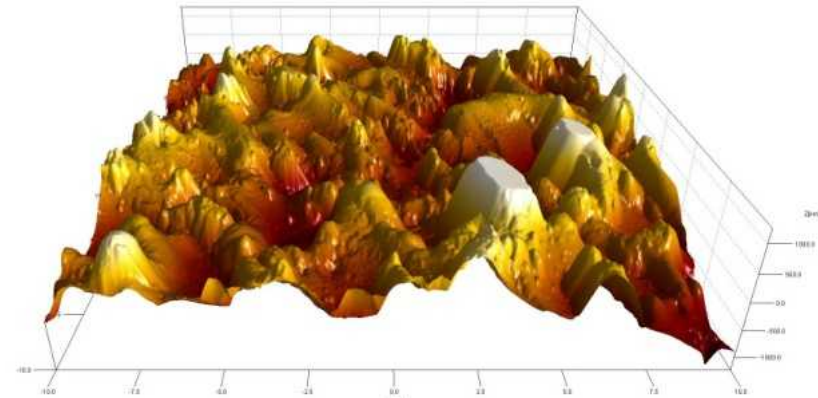


100 µm

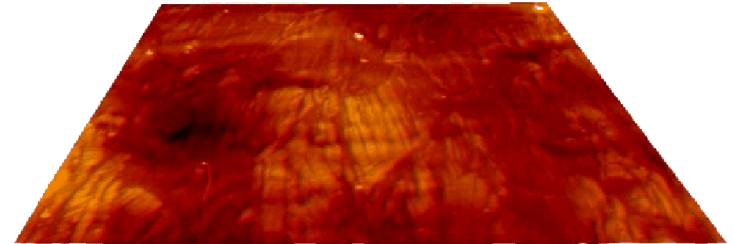
EHT = 10.00 kV WD = 13 mm Signal A = SE2 File Name = femto\_tilt\_C6\_overview\_69.tif

# Electropolishing

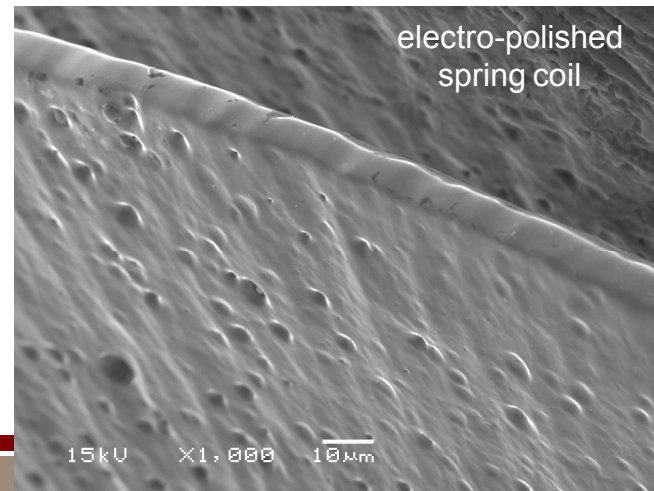
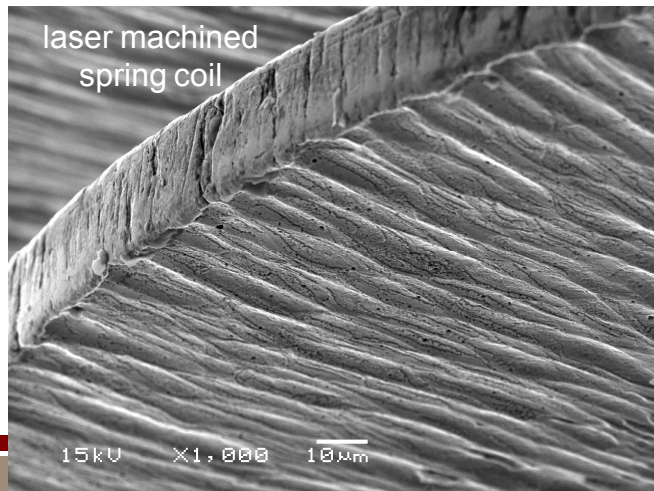
- Prompted by concern that process damage & tube roughness would reduce fatigue strength
- Optimization experiments
  - examined EP of high aspect ratio features
    - 20-30 $\mu\text{m}$  gaps w/aspect ratios  $\sim 5$
    - thru cuts on 0.008in thick, 0.050in OD tube
    - measured throwing power, surface finish & material removal
  - presentation by colleague L. Serna
    - “Effect of Electropolishing on the Surface Topography of Micro-Wire Electrodischarge Machined Simulated Coil Gaps”, ICOMM 2012



$\mu$ -EDM surface,  $R_a = 254\text{nm}$

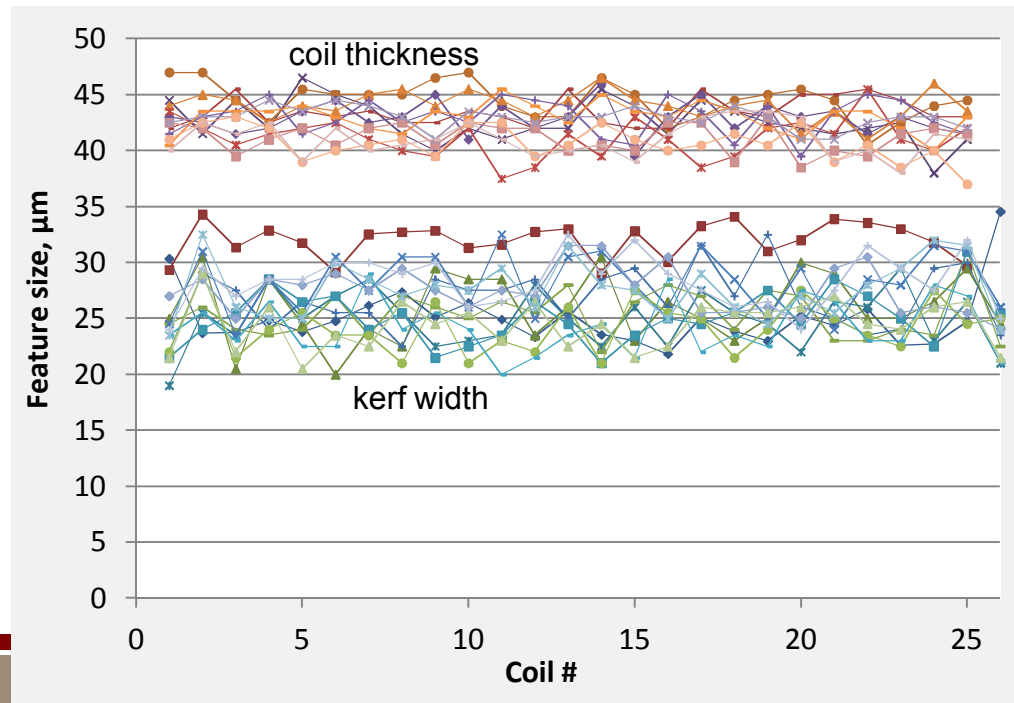
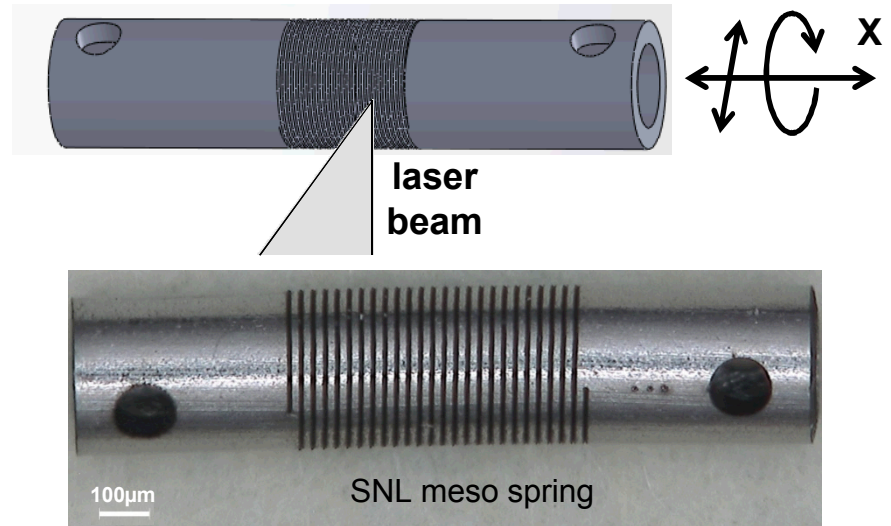


electropolished,  $R_a = 20\text{nm}$



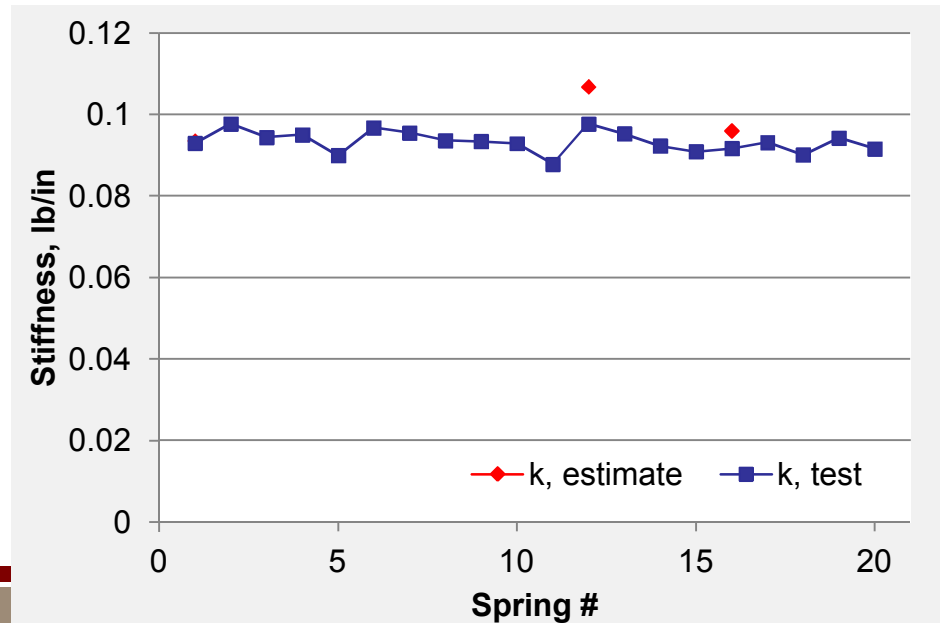
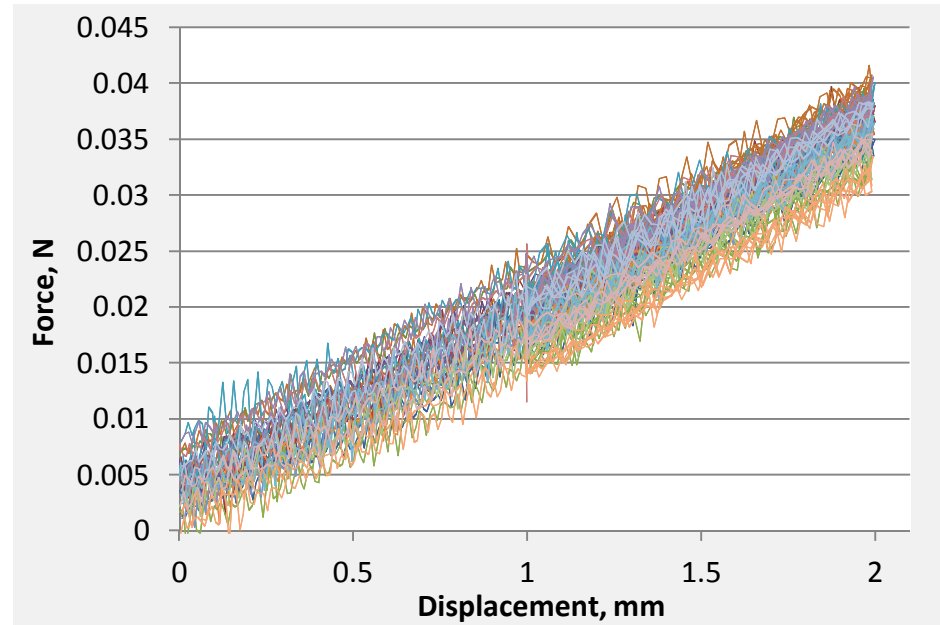
# Prototype Fabrication

- Prototype fabrication
  - Spectra Physics Hurricane fs laser
  - process description
    - stationary beam
    - tubing translates in XY, rotates about  $\theta$
- Process repeatability
  - 19 prototype parts
  - coil thickness
    - spec =  $40\mu\text{m}$
    - average =  $42.5\mu\text{m}$
    - range =  $10.0\mu\text{m}$
    - std dev =  $1.9\mu\text{m}$
  - kerf thickness
    - spec =  $25\mu\text{m}$
    - average =  $26.0\mu\text{m}$
    - range =  $15.6\mu\text{m}$
    - std dev =  $2.8\mu\text{m}$

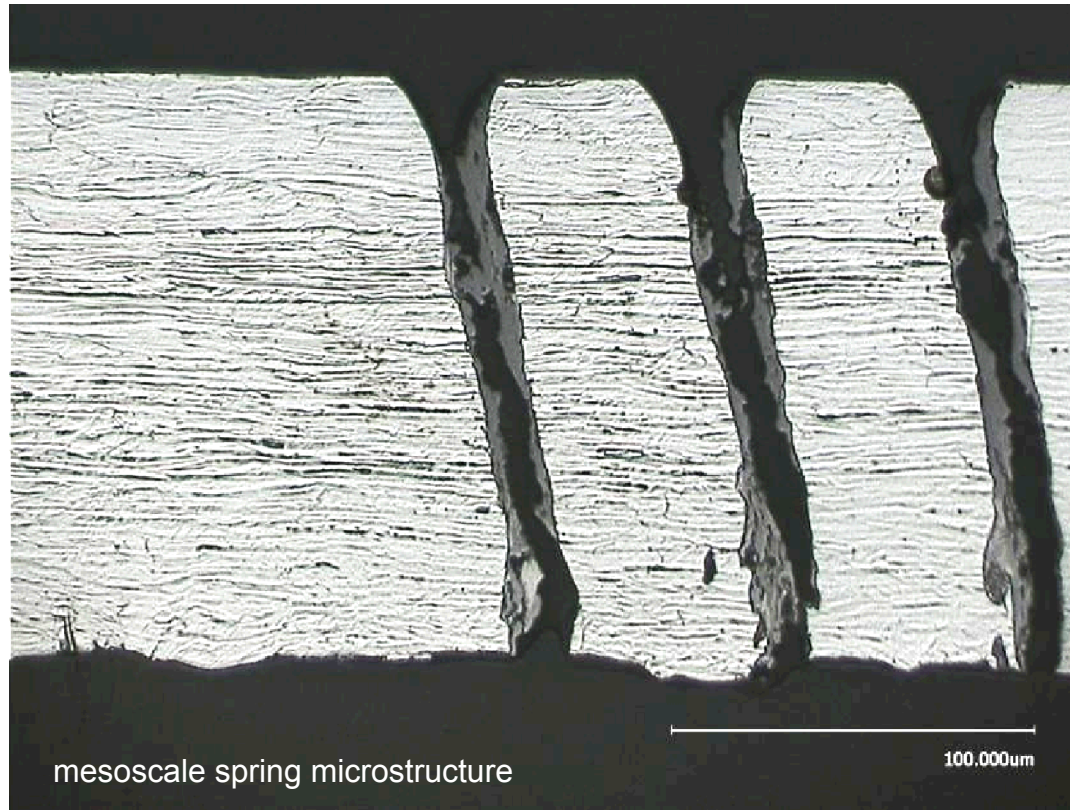


# Spring Performance

- Design models
  - proven for “macro” springs
  - assume bulk properties & continuum material behavior
  - what is accuracy @ mesoscale?
- Prototype meso springs
  - set of 20 springs
  - ~10% variability in stiffness
    - correlates directly to similar variation in coil thickness
    - normal distribution
  - closed form prediction accurate to ~5%
  - suggests minimal impact to material modulus due to processing & feature size



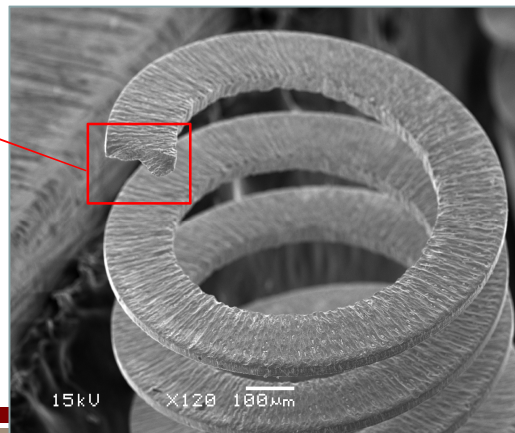
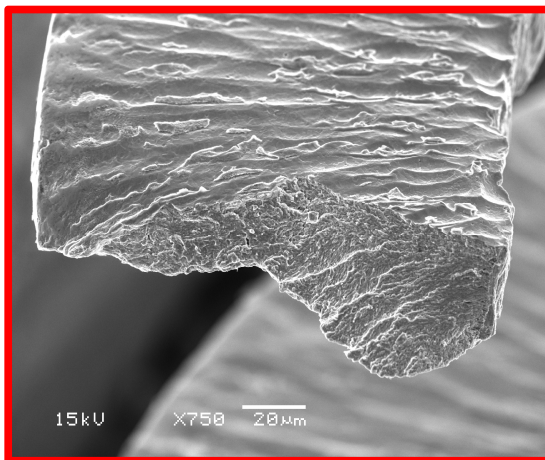
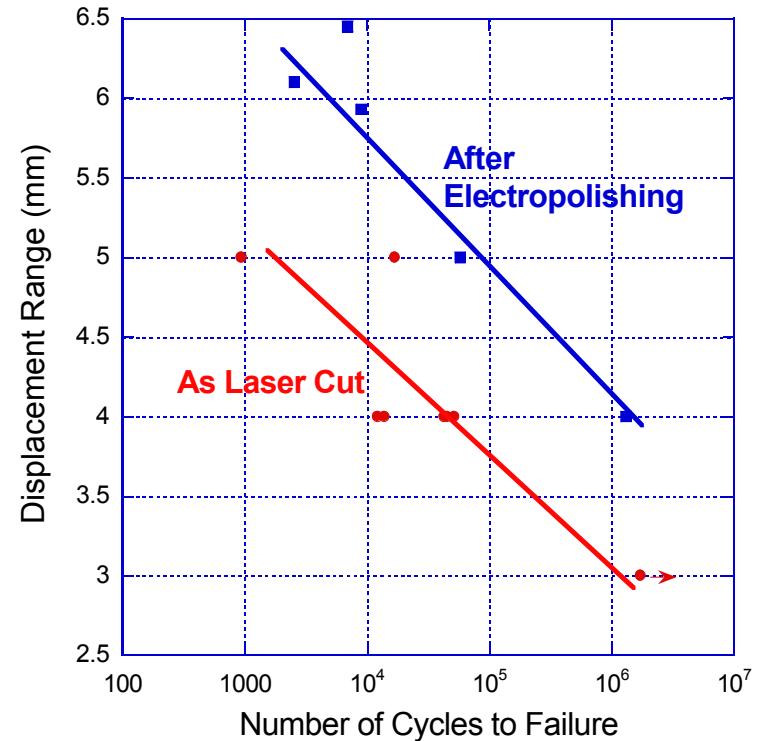
# Continuum Behavior



- Stiffness consistent with continuum models
  - coil thicknesses  $\sim 20\text{-}40\mu\text{m}$
  - work hardened grain orientation is advantageous
    - multiple grains observed across coil width
  - torsion loading averages stresses across grains

# Fatigue Resistance

- Pulsed laser surfaces are rough & are sources for crack initiation
- Electropolishing provides order-of-magnitude improved fatigue life



# Conclusions

- Directly machined mesoscale springs have been investigated
- Design trades have been studied based on closed form solutions
- Feasibility & impact of fabrication processes have been examined
- Prototype springs have been fabricated & shown to provide stiffness variations of only 10% with model agreement within 5%



# Acknowledgements

- Process operation
  - Carter Hodges, Michael Saavedra
- Design
  - Ernie Wilson, Ron Wild
  
- This document has been reviewed and approved for unclassified, unlimited release under SAND2012-**xxx**C.