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MAR 6 1958
Case No. 771.00
Ref. Sym. 1612 (486)
Project No. TM-548
File: TX-28, 3-2

SANDIA SYSTEMATIC DECLASSIFICATION REVIEW	
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5. Authority: <u>ADD</u>	6. Authority: <u>W.C. Layne</u>
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MR. W. J. DENISON - 1224

Attn: Mr. H. A. Mullin - 1224-2

Re: Static Test of TX-28/X1 Aft Case Parachute Lug Ring

COL No	
ACCOUNTABILITY CARD	<u>Bm</u>
FILE No	<u>TX-28</u> <u>3-2</u>

Summary of Results

One TX-28/X1 parachute lug ring and aft case, DS(1224)79328, was loaded to about 145,000 pounds with the simulated chute load pulling at 15 degrees with respect to the longitudinal axis of the case. No damage resulted to the unit from this load. The limit load at the time of this portion of the test was 107,500 pounds.

The second TX-28/X1 parachute lug ring and aft case, Dwg. No. DL-10021, was loaded to 110,000 pounds with the load pulling at 15 degrees with respect to the longitudinal axis of the case without failure. When the angle of the load was increased to 20 degrees, failure occurred at 106,000 pounds by buckling of the case which in turn was followed by rivet failure near the mounted end of the case. The limit load at the time this portion of the test was run was 80,000 pounds.

Object of Test

The test was performed to determine the structural adequacy of the TX-28/X1 parachute lug ring and aft case unit for a 15 degree pull (15 degree angle of attach) when loaded to 137.5 per cent of design limit load. Also, the unit was to be tested to failure by increasing the angle of pull by 5 degree increments beyond 15 degrees. The design limit load at the time of testing the first unit was 107,500 pounds. However, when the second unit was tested, the design limit load had decreased to 80,000 pounds due to reconsideration of aerodynamic forces.

Reason for Test

The test was performed as a result of the Work Order Authorization dated March 7, 1957 from Division 1224 to Division 1612.

Function of Object Tested

The aft case of the TX-28 contains the parachute and transmits to the weapon the loads induced by the parachute when it is deployed.

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PERSON CHANGING MARKING & DATE: <u>Amelda Selph 8/25/98</u>	RECORD ID: <u>98SN3792</u>
PERSON VERIFYING MARKING & DATE: <u>W.C. Layne 8/25/98</u>	DATED: <u>8/20/98</u>

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Project No. TM-548Summary of Past Tests

Numerous tests have been run on the TX-28 and its components. However, no previous static tests have been run in which the parachute loads were applied to the aft case. A previous static test was performed on a couple of parachute lugs, the results of which are given in the report, Static Test of TX-28 Parachute Lug, dated June 13, 1957, TM-577, Ref. Sym: 1612 (519), from Division 1612 to Division 1224. Following is the "Summary of Results" from that test:

"Two TX-28 lug and clevis assemblies were tested to destruction. In the first test the clevis failed in tension at the pin hole when subjected to a load of 137.5 per cent limit load (13,750 pounds). A heavier clevis was substituted for the second test and the lug failed in tension at the pin hole at a load of 159 per cent limit load (15,900 pounds)".

The lugs used in the current tests are very similar but not identical to the second clevis or lug tested in the above "Summary of Results".

Setup for Test

The setup for the test is shown in Figs. 1 and 2.

The drawing number for the first case was DS(1224)79328 and the drawing number for the second case was DL-10021.

Following is a description of the materials in the two cases: The first case had a 1/8-inch outer skin and 1/16-inch inner skin of 24S-T36 aluminum. The aluminum rings at both the mounted end and open end of the case was 14S-T6. The steel ring at the mounted end was 4130 steel (H.T. 125,000 - 145,000 psi). In the second case the inner and outer skins were 1/16-inch aluminum (61S-T6). The aluminum rings at each end of the case was also 61S-T6. The steel ring at the mounted end was 4130 steel (H.T. 125,000 - 145,000 psi).

The following equipment was used:

- 1 - Paytheon hydraulic cylinder, 400,000 pounds capacity
- 1 - Two-inch calibrated pull bar No. 7
- 8 - One-inch calibrated pull bars, Serial Nos. 2, 3, 4, 5, 6, 8, 9, and 10.
- 1 - Fairchild strain indicator, Model 101-C-2.
- 1 - Sadic automatic data processing system, Model 34-112 MI
- Baldwin Southwark testing machine, 5000-pounds capacity.

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The following instrumentation was used:

- 5 - Starrett dial indicators, graduated to .001 inch with a maximum range of 1.0 inch
- 28 - Type A-7 strain gages, six of which were used in two rosettes, Res. 120 ohms
- 10 - Type AX-7 strain gages, G.F. = $1.86 \pm 2\%$, Res. = 120 ohms
Stresscoat No. 1206, sensitivity at time of test was 800 to 850 microinches per inch.

Procedure

The first TX-28/X1 aft case, DS(1224)79328, was Stresscoated with Stresscoat No. 1206 and mounted in the static jig as shown in Figs. 1 and 2. The load was applied in a direction making an angle of 15 degrees with the longitudinal axis of the case. The load was applied in increments from 20,000 pounds to 125,000 pounds with each load being 1.2 times the preceding load. After each increment, the load was released to zero and the areas of Stresscoat cracks were marked. The eight shroud lines between the unit and the intermediate jig fixture were kept approximately equally loaded during this test and all subsequent tests of the unit by adjusting the tension in each shroud line as required before loading to the next increment.

After the Stresscoat test was completed, 25 strain gages were mounted, consisting of 19 uniaxial gages and 2 rosettes. The unit was again mounted in the static jig as shown in Fig. 1 and loaded in increments of 10,000 pounds to a load of about 145,000 pounds.

Strain readings were taken at each increment of load. At a load of 145,000 pounds, jig failure prohibited additional loading.

The second TX-28/X1 aft case, DL-10021, was Stresscoated with Stresscoat No. 1206 and mounted in the static jig as shown in Figs. 1 and 2. The load was applied in a direction making an angle of 15 degrees with the longitudinal axis of the case. The load was applied in increments from 20,000 pounds to 41,500 pounds with each load being 1.2 times the preceding load. After each increment, the load was released to zero and the areas of Stresscoat cracks were marked.

Ten biaxial strain gages and three uniaxial strain gages were mounted to this second case and the unit was again setup in the static jig as shown in Figs. 1 and 2. The unit was loaded in increments to a load of 110,000 pounds, pulling at an angle of 15 degrees from the longitudinal axis of the test unit. Strain readings were taken at the 23 strain gages and deflection readings were taken at the two dial indicators after each increment of load. In addition, as the test progressed, the load was dropped

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to 10 per cent of limit load after each increment of load and deflection readings were taken at this 10 per cent load. The tension on shroud lines was equalized each time when the load was released to 10 per cent of limit load.

A five-degree shim was inserted on the static jig mounting plate so that the outer end of the test unit was tilted downward five degrees from the preceding test. With the same setup for the hydraulic ram and pull rods as used previously, the direction of the load made an angle of 20 degrees with the longitudinal axis of the unit. The unit was then loaded in increments to about 106,000 pounds at which load failure prohibited additional loading. Strain readings at 23 strain gages and deflection readings at five dial indicators were taken at the increments of load. As in the previous test, the load was dropped to 10 per cent of limit load after each increment of load for equalizing the tension in the eight shroud lines. At this 10 per cent load, deflection readings were taken at the dial indicators for a portion of the test.

Results

The Stresscoat test on the first TX-28/X1 aft case, DS(1224)79328, produced the areas of Stresscoat cracks shown on Figs. 3 thru 6 when loaded to a maximum of 125,000 pounds. Table I shows the loads producing the Stresscoat cracks on the above figures and the sensitivity of the Stresscoat.

The first unit when loaded to about 145,000 pounds at a 15-degree pull failed the adapter plate between the test unit and the static jig. However, the TX-28/X1 case did not fail from this test. This load corresponds to 135 per cent of limit load based on the limit load of 107,500 pounds applicable at the time of this test. However, based on the limit load of 80,000 pounds used and the second case, the load of 145,000 pounds would be 182 per cent of the design limit load. The strain readings from the 25 strain gages and the computed stresses for the two rosettes are shown on Table II. The maximum computed stress was 3,200 psi (compression) at rosette (20, 21, 22) on 24S-T36 aluminum. The locations of the strain gages are shown on Figs. 7 thru 12.

The Stresscoat test on the second TX-28/X1 case (DL-10021) produced the areas of Stresscoat cracks shown on Figs. 13 thru 15 when loaded to a maximum of 41,500 pounds at an angle of 15 degrees with the longitudinal axis of case. Table III shows the loads producing the Stresscoat cracks on the preceding three figures.

The second unit when loaded to 110,000 pounds (137.5 per cent of a limit load of 80,000 pounds) with the load making an angle of 15 degrees with the longitudinal axis did not result in any damage to the unit. The

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strain readings and the computed principal stresses are shown on Table IV. The stresses at gages (3, 4) reached the yield stress of 61S-T6 aluminum (35,000 psi) at a load of 64,000 pounds (80 per cent limit load). The maximum stress on the steel ring was about 78,000 psi at gages (20, 21) at a load of 110,000 pounds. The stresses are computed on the assumption that the biaxial gages are oriented in the direction of the principal stresses. The location of the 23 strain gages are shown on Figs. 16 thru 19. The deflection readings at the two dial indicators are shown on Table V. Dial indicators 1 and 2 measure the vertical movement of the bottom of the case with respect to the floor at the mounted end and open end, respectively. The location of these two dial indicators is shown on a subsequent photograph.

When loaded with a 20-degree pull, the unit failed at a load of 106,000 pounds which is about 133 per cent of limit load based on a limit load of 80,000 pounds. The initial failure was the buckling of the skin of the case which was followed by the shearing of the rivets near the mounted end. The unit after failure is shown in Figs. 20 thru 23. Table VI shows the strain readings and the computed principal stresses. The computed stress on the aluminum ring (61S-T6) reached the yield point of 61S-T6 (35,000 psi) at about 50 per cent of limit load (gages 3, 4). The maximum computed stress in the steel ring (4130) was about 95,000 psi at 133 per cent of limit load (gages 20, 21). The strain gages are the same as used in the preceding test and are shown on Figs. 16 thru 19. The deflection readings for dial indicators 1 thru 5 is shown on Table V. The location of dial indicators 1 and 2 is the same for both the 15-degree and 20-degree pull.

After the test was completed, fourteen tensile specimens were cut from the unit at the locations shown in Figs. 16 and 17. These tensile specimens were pulled to failure in the Baldwin-Southwark testing machine with the results shown in Table VII.

The properties of the exterior and interior skins shown on Table VII indicate that this part of the structure is 61S-T6 which has an ultimate stress of 42,000 psi and a yield stress of 35,000 psi. The properties of the ring near the mounted end of the case shown on Table VII indicates that this part of the structure has a yield stress of about 43,000 psi and an ultimate stress of about 47,000 psi. This part is supposed to be 61S-T6. However, these values are somewhat above the minimum values of 61S-T6. The four tensile specimens from the ring at the open end of the case gave an average ultimate stress of 41,900 psi and an average yield stress of 39,100 psi. This corresponds to 61S-T6 aluminum except the yield stress is somewhat higher than the minimum yield stress for 61S-T6.

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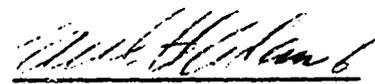
Project No. TM-548

Conclusions

It is concluded that the second unit had adequate strength for the 15-degree pull based on 137.5 per cent of a design limit load of 30,000 pounds. The first case has sufficient strength for at least 132 per cent of a limit load of 80,000 pounds when pulling at 15 degrees.


H. P. WHEELER - 1612-2

Approved by:


PAUL H. ADAMS - 1612

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W. A. Gardner, 1610

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C. L. Gomel, 5523

R. K. Smeltzer, Central Record File Section, 7221-3

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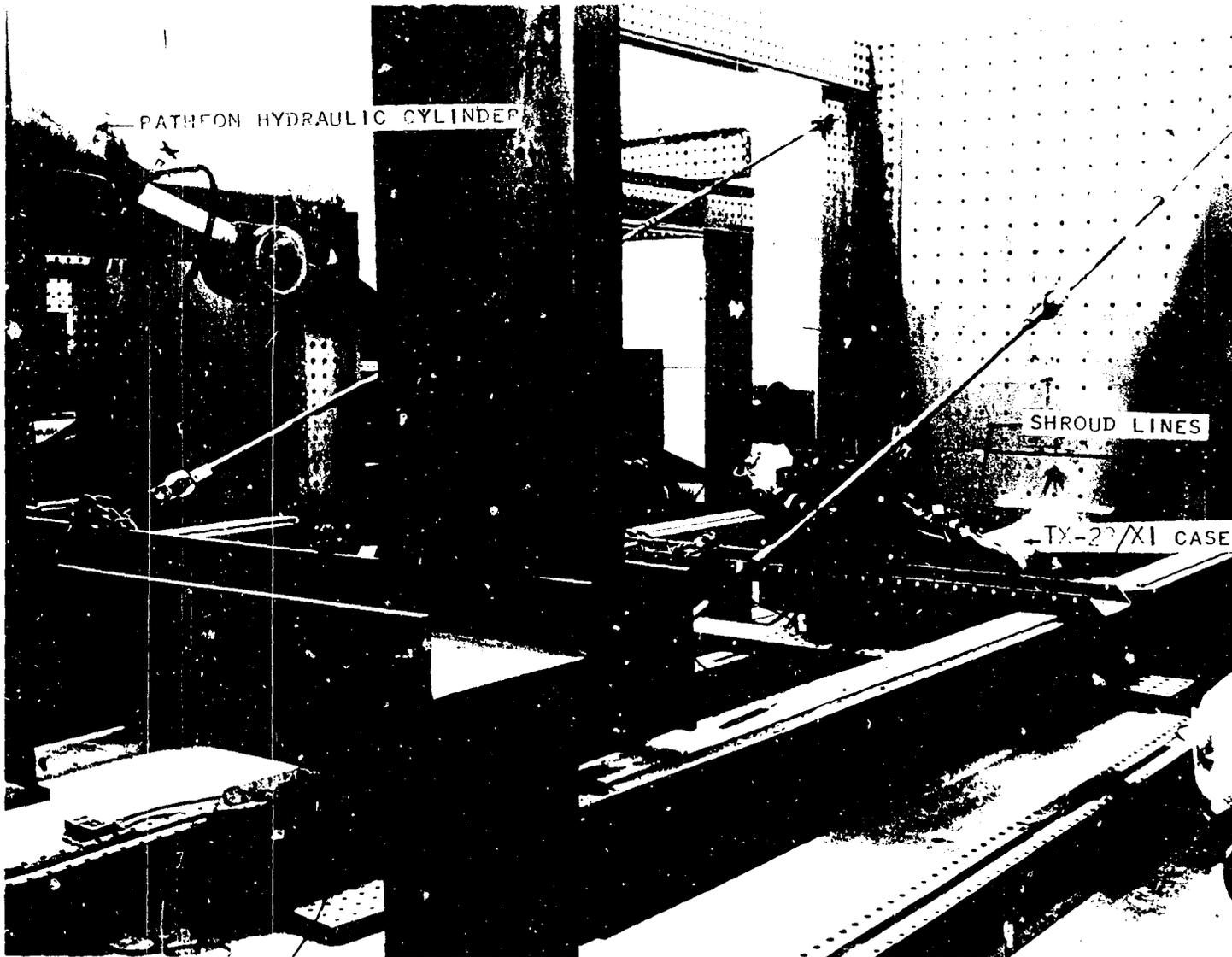


FIG. 1 -- TEST SETUP FOR STATIC TEST OF TX-28/XI AFT CASE PARACHUTE LUG RING.

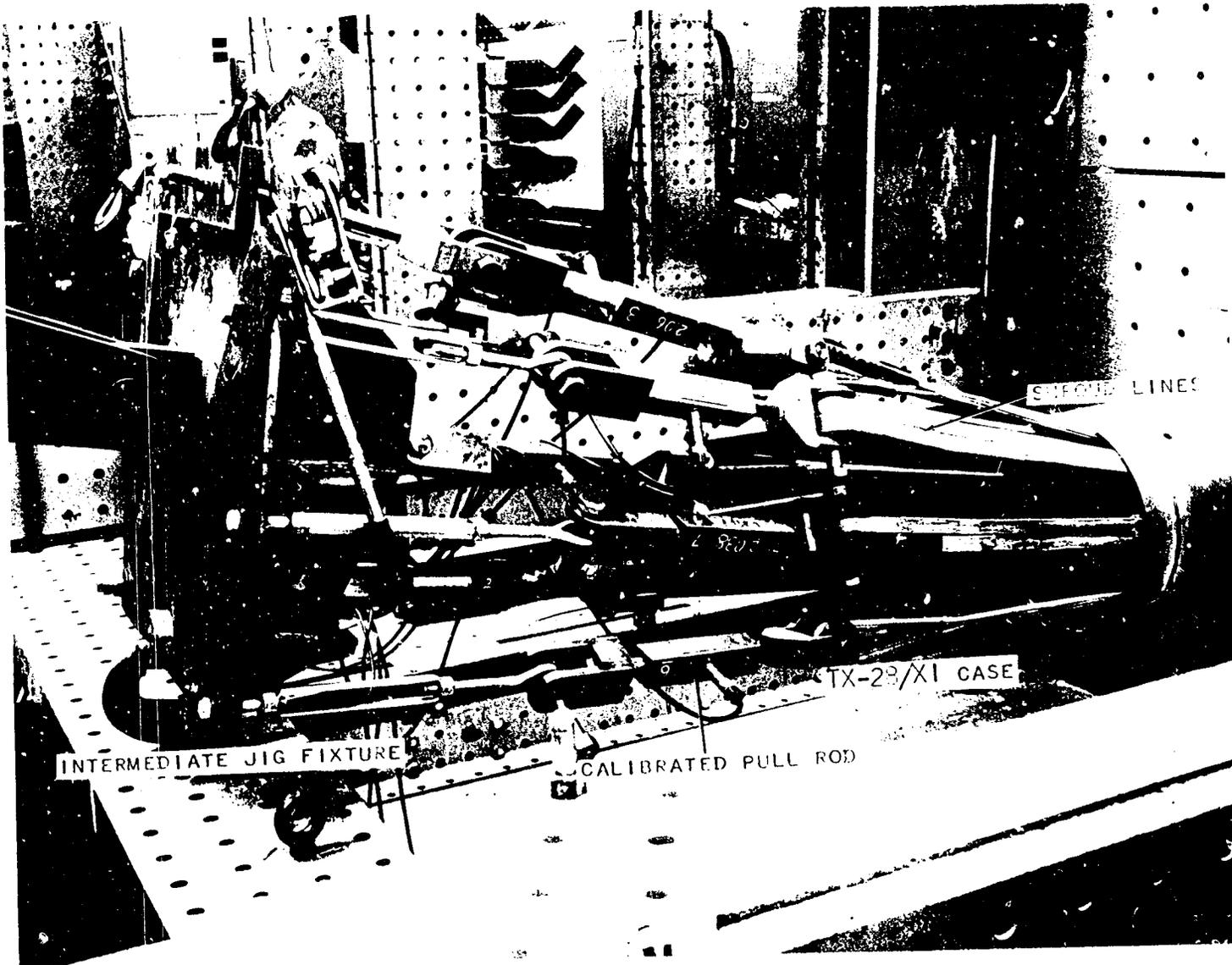
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FIG. 2 -- TEST SETUP FOR STATIC TEST OF TX-28/XI AFT CASE PARACHUTE LUG RING.

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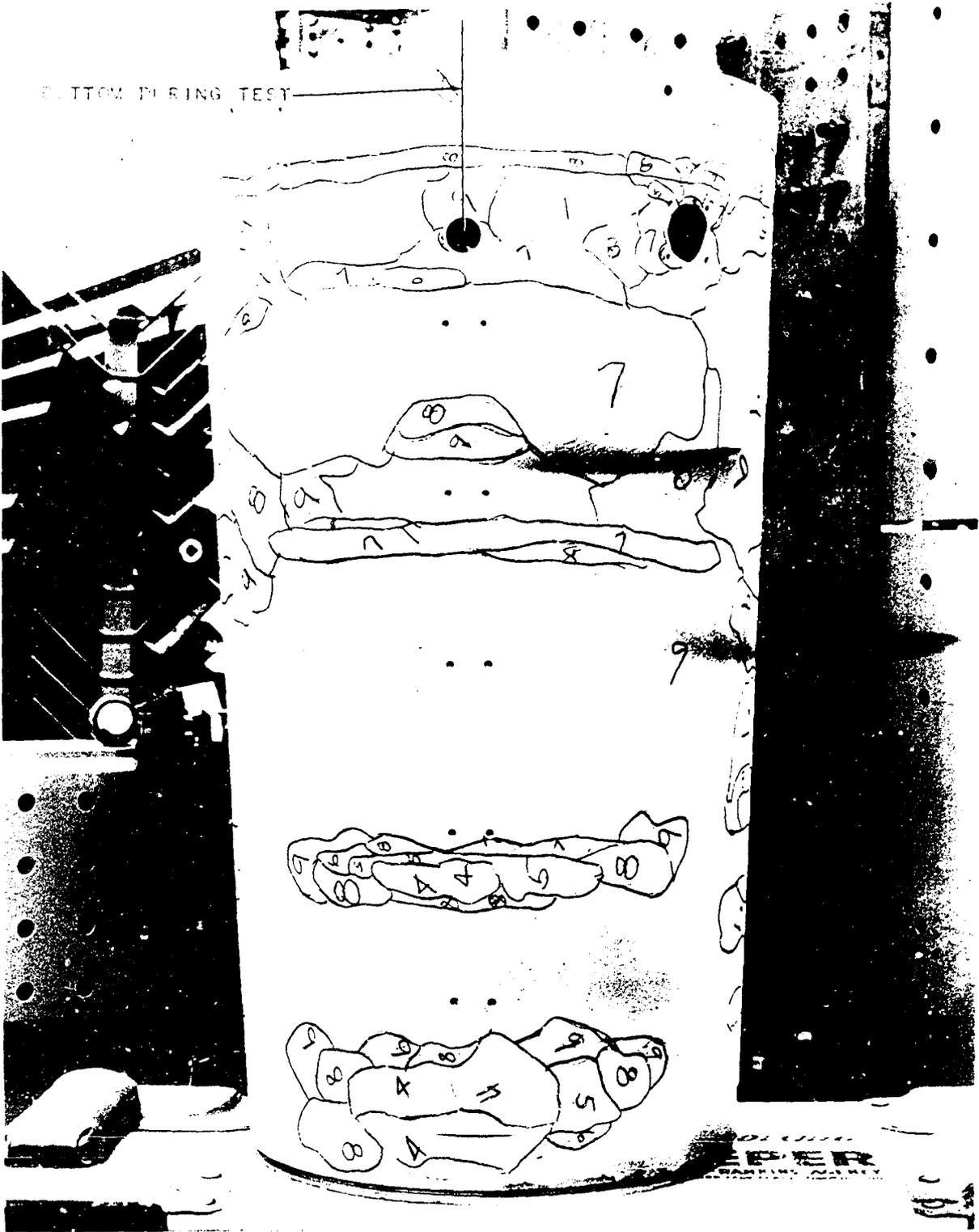


FIG. 3 -- AREAS OF STRESSCOAT CRACKS FOR 15° PULL (FIRST CASE) --
 STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

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REF. SYM: 1612 (486)
 PROJECT NO. TM-548

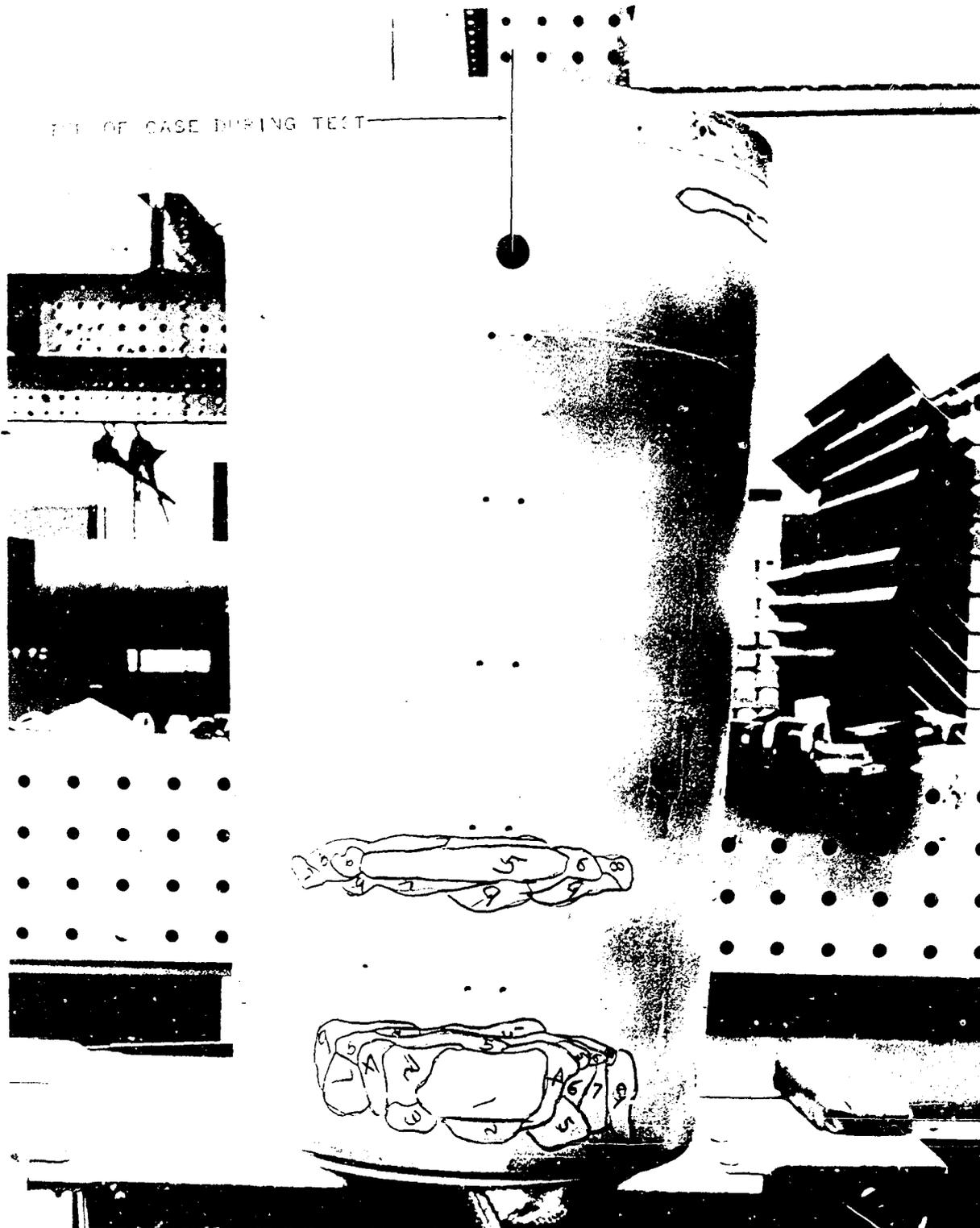


FIG. 4 -- AREAS OF STRESSCRACK CRACKS FOR 15° FULL (FIRST CASE) --
 STATIC TEST OF TX-2P/X1 AFT CASE PARACHUTE LUG RING.

D# 102612

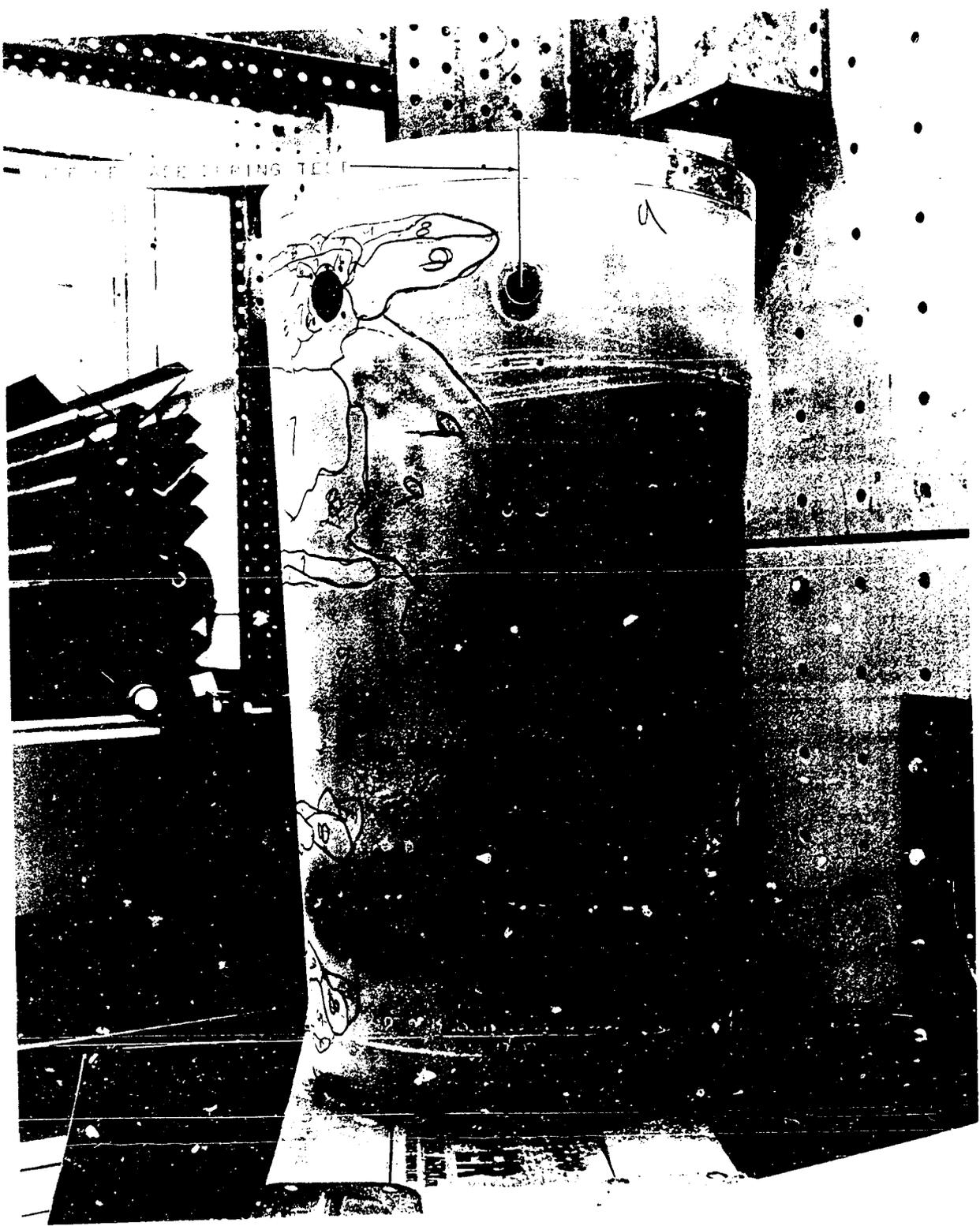
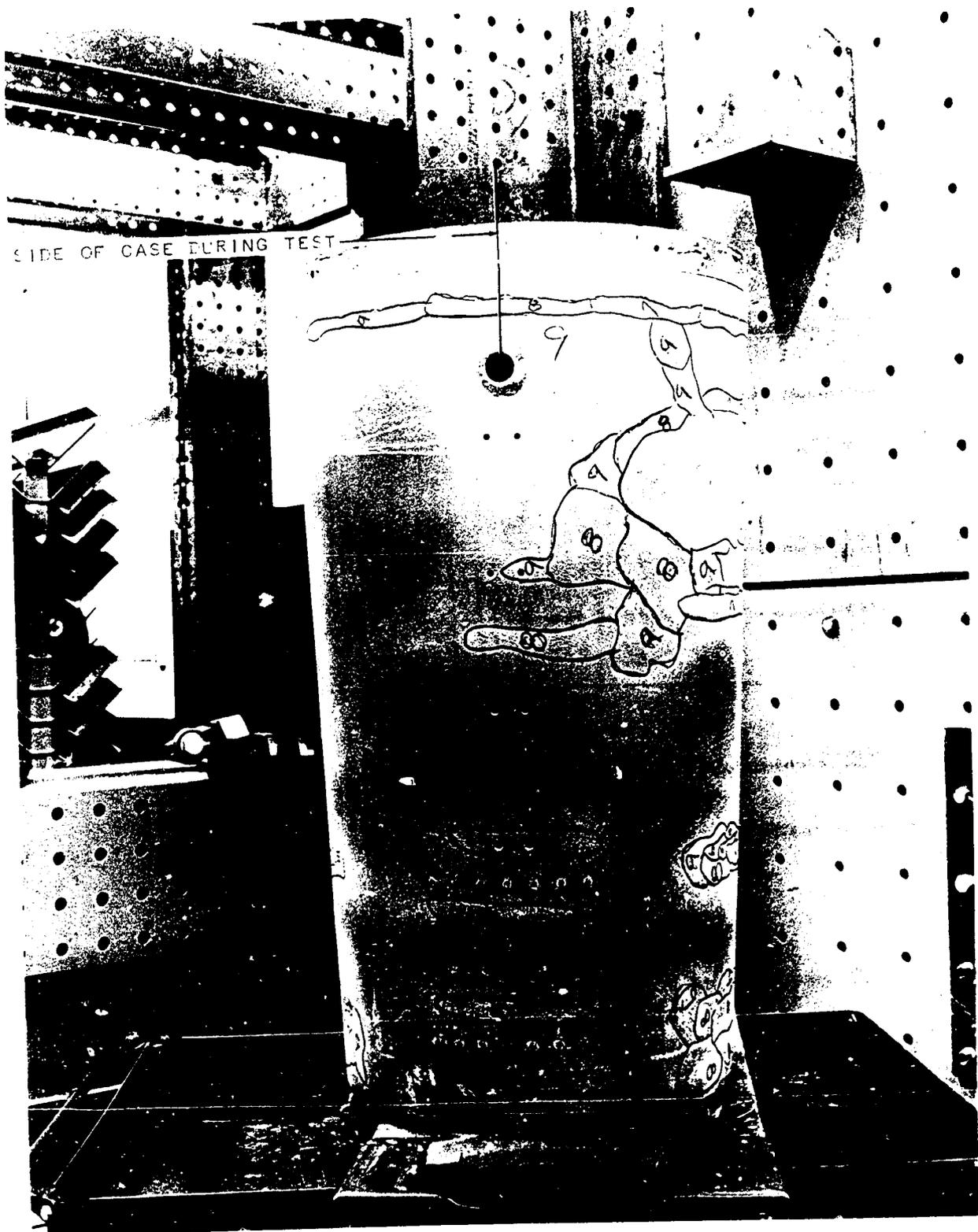


FIG. 5 -- LOCUS OF STRESSOCAT CRACKS FOR 15° HULL (FIRST CASE) --
 STATIC TEST OF TR-2 /X1 AFT CASE PARACHUTE LUG RING.

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REF. SYN: 1617 (4-6)
 PROJECT NO. TR-24



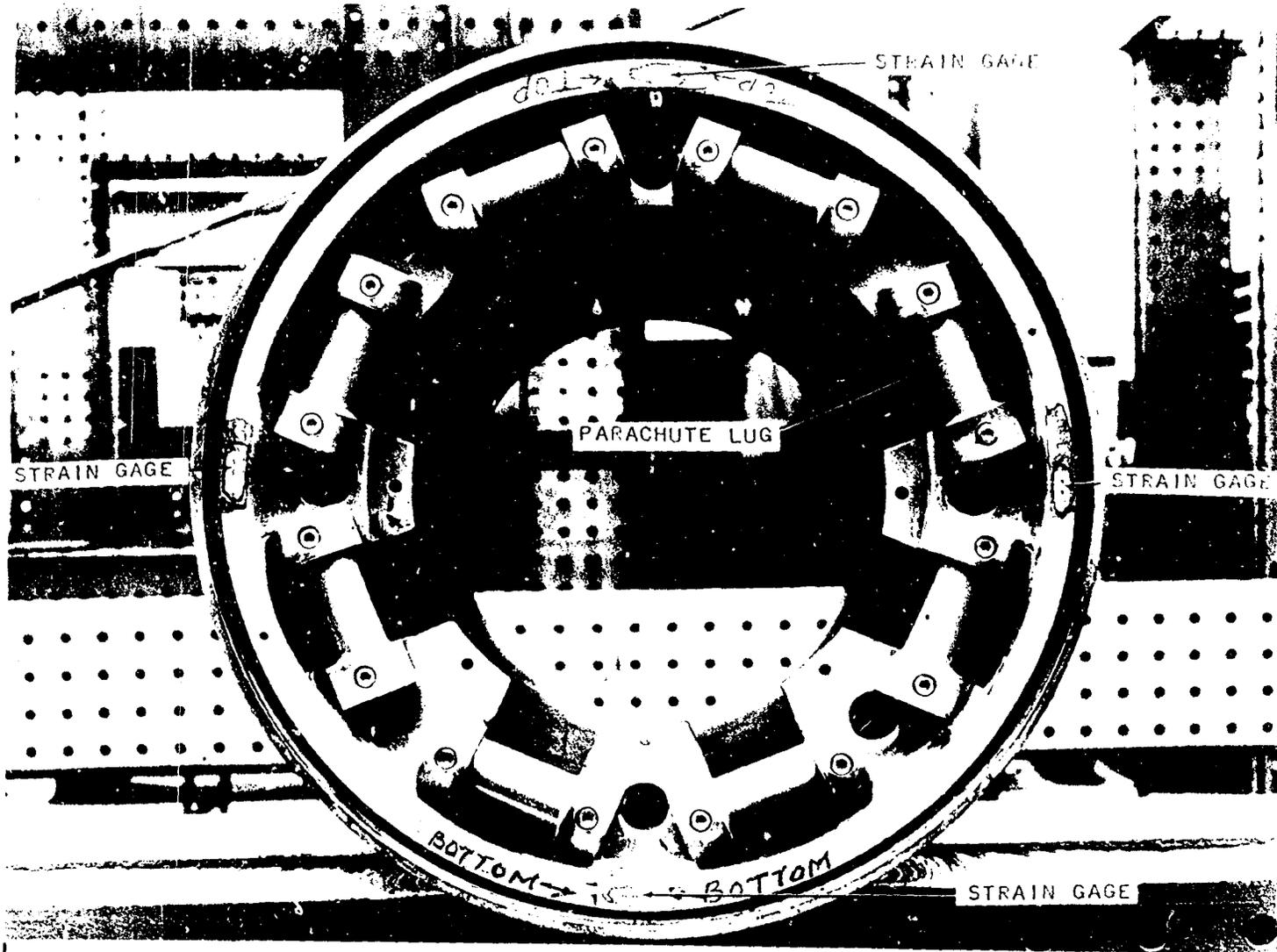
SIDE OF CASE DURING TEST

FIG. 6 -- AREAS OF STRESSCOAT CRACKS FOR 15° PULL (FIRST CASE) --
STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

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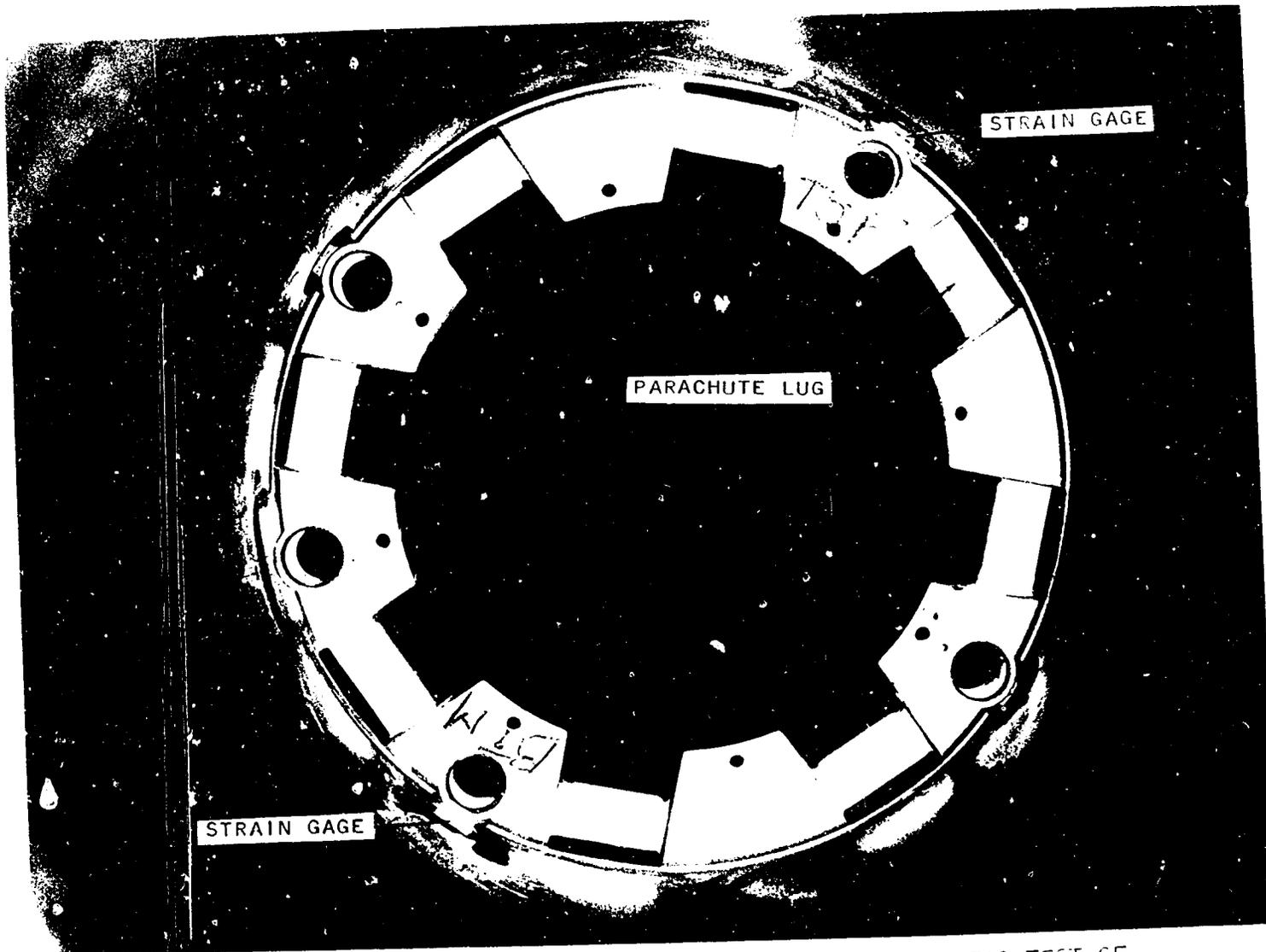
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FIG. 7 -- LOCATION OF STRAIN GAGES (FIRST CASE) -- STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

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FIG. 8 -- LOCATION OF STRAIN GAGES (FIRST CASE) -- STATIC TEST OF TX-2⁸/XI AFT CASE PARACHUTE LUG RING.

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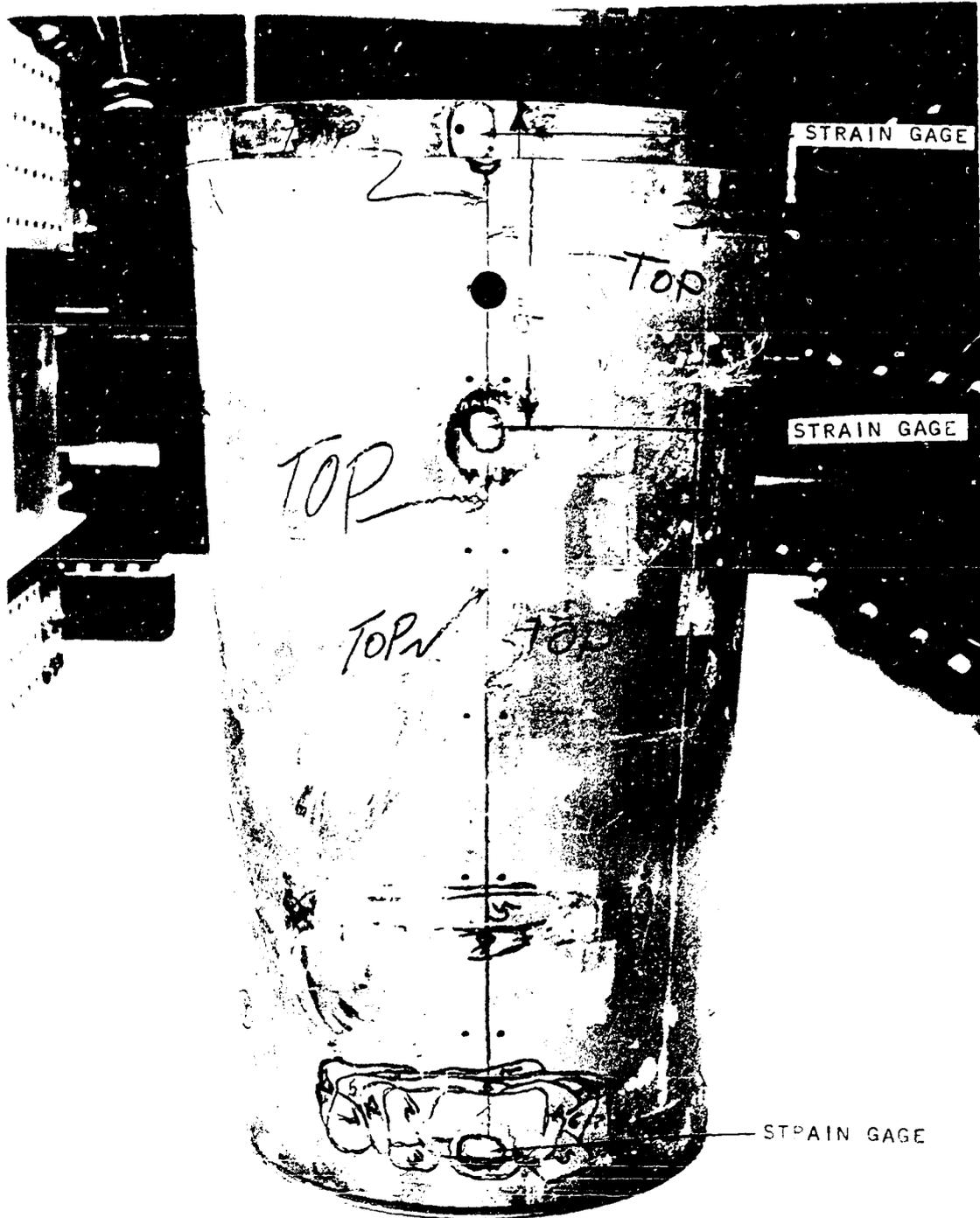
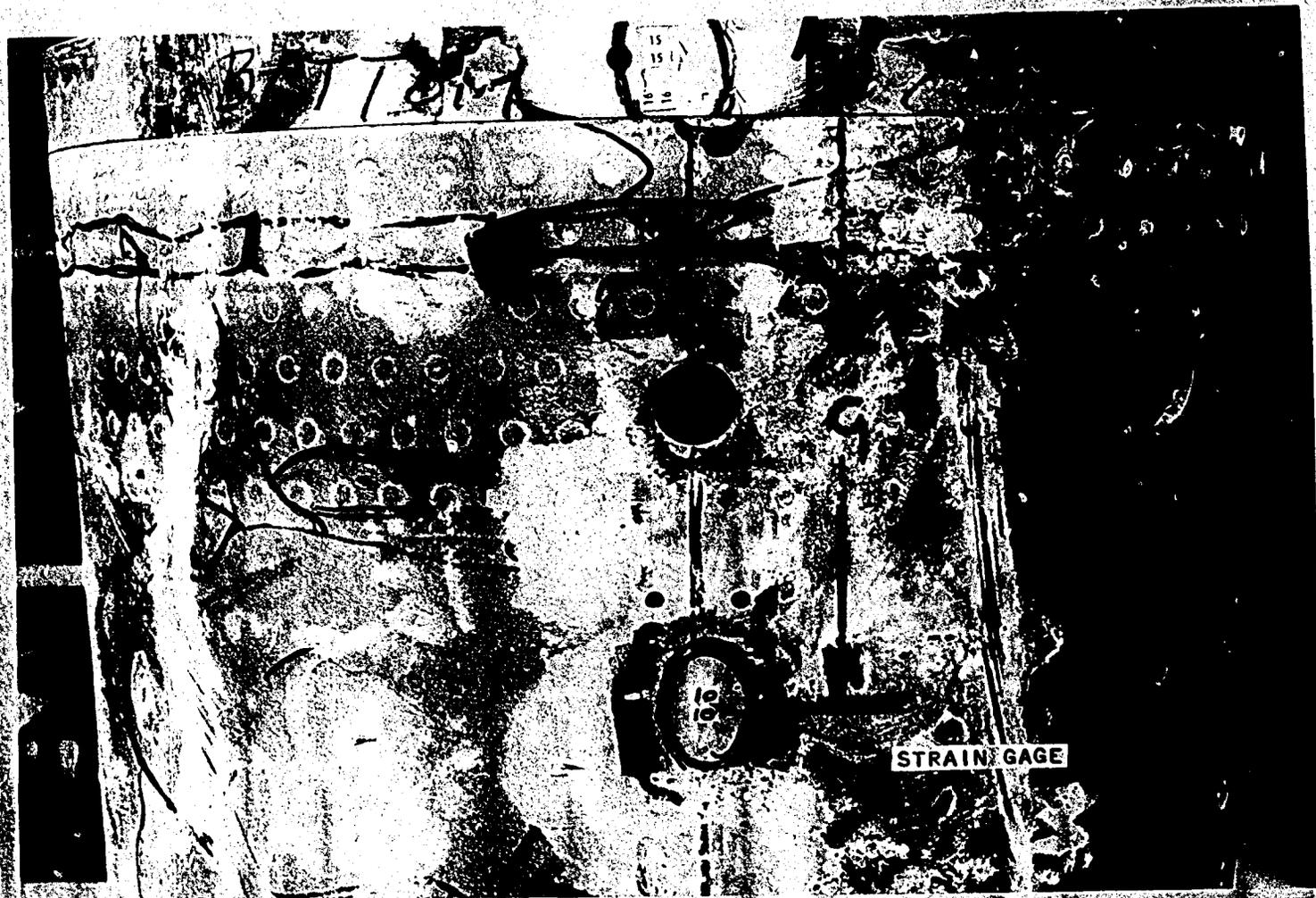


FIG. 9 -- LOCATION OF STRAIN GAGES (FIRST CASE) -- STATIC TEST OF
 TX-2P/XI AFT CASE FATIGUE TEST RING.

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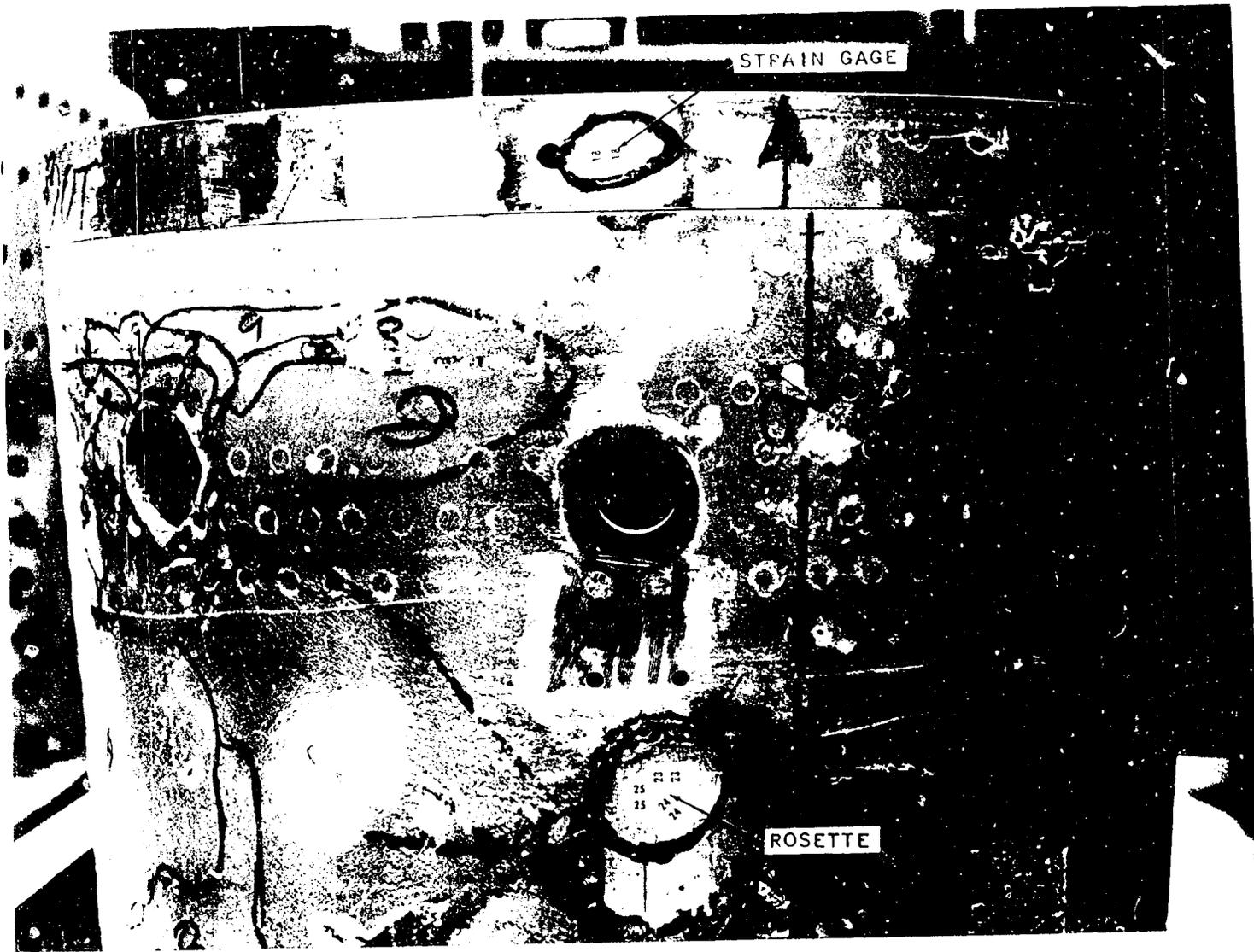
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FIG. 10 -- LOCATION OF STRAIN GAGES (FIRST CASE) -- STATIC TEST OF TX-28/XI AFT CASE PARACHUTE LUG RING.

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FIG. 11 -- LOCATION OF STRAIN GAGES (FIRST CASE) -- STATIC TEST OF TX-20/X1 AFT CASE FAPACHITE LUG RING.

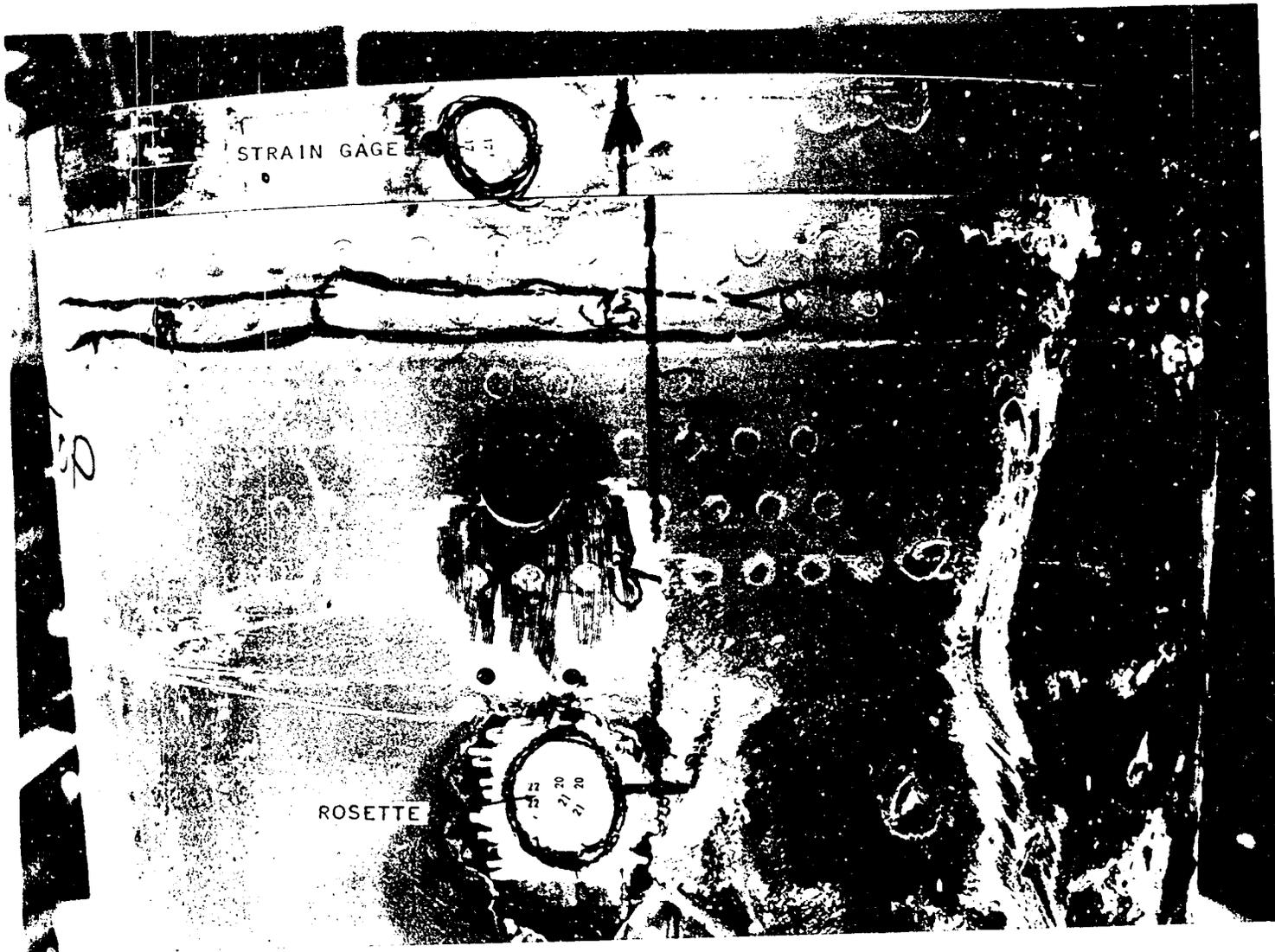
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FIG. 12 -- LOCATION OF STRAIN GAGES (FIRST CASE) -- STATIC TEST OF TX-28/X1 AFT CASE FAPACHITE LUG RING.

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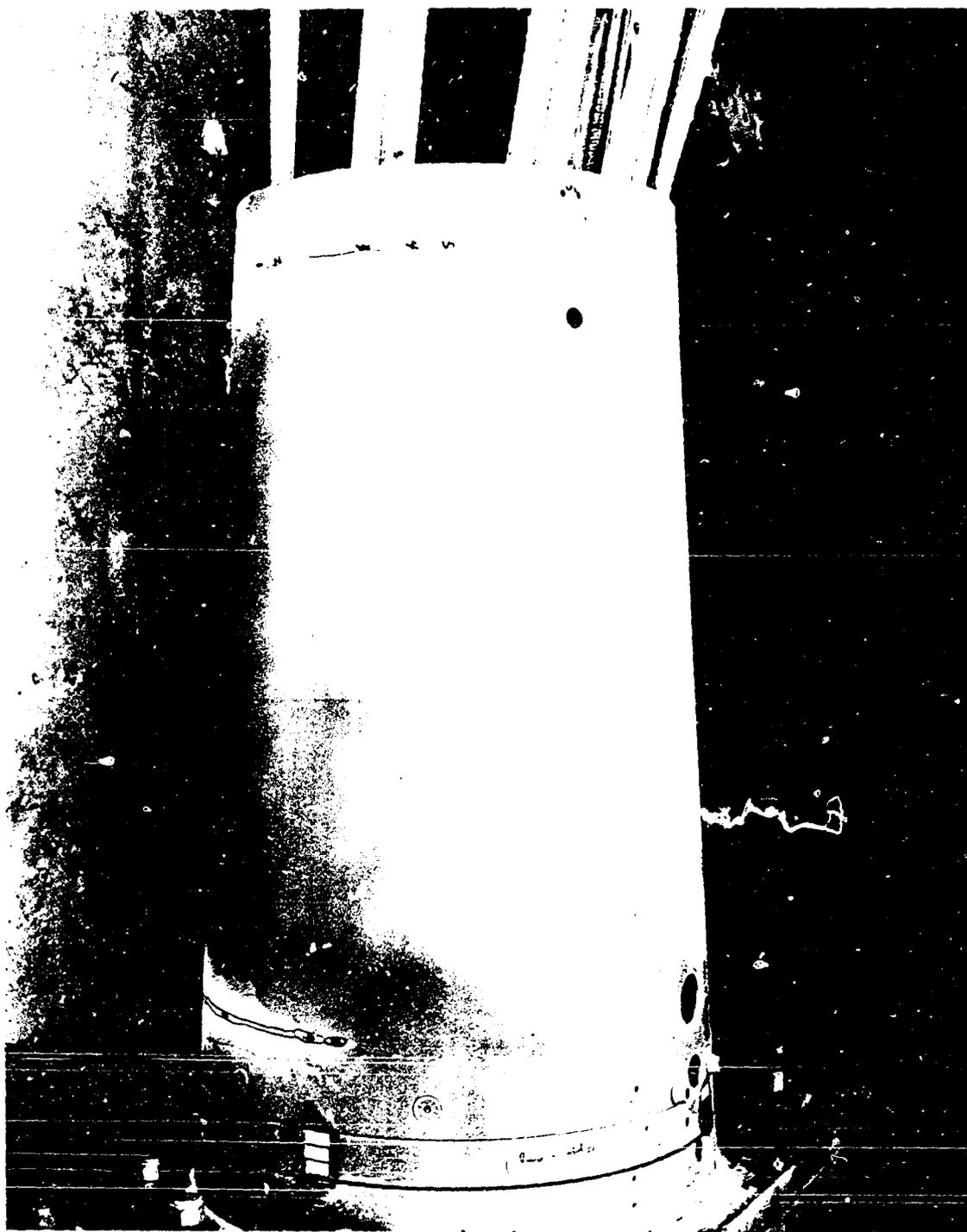


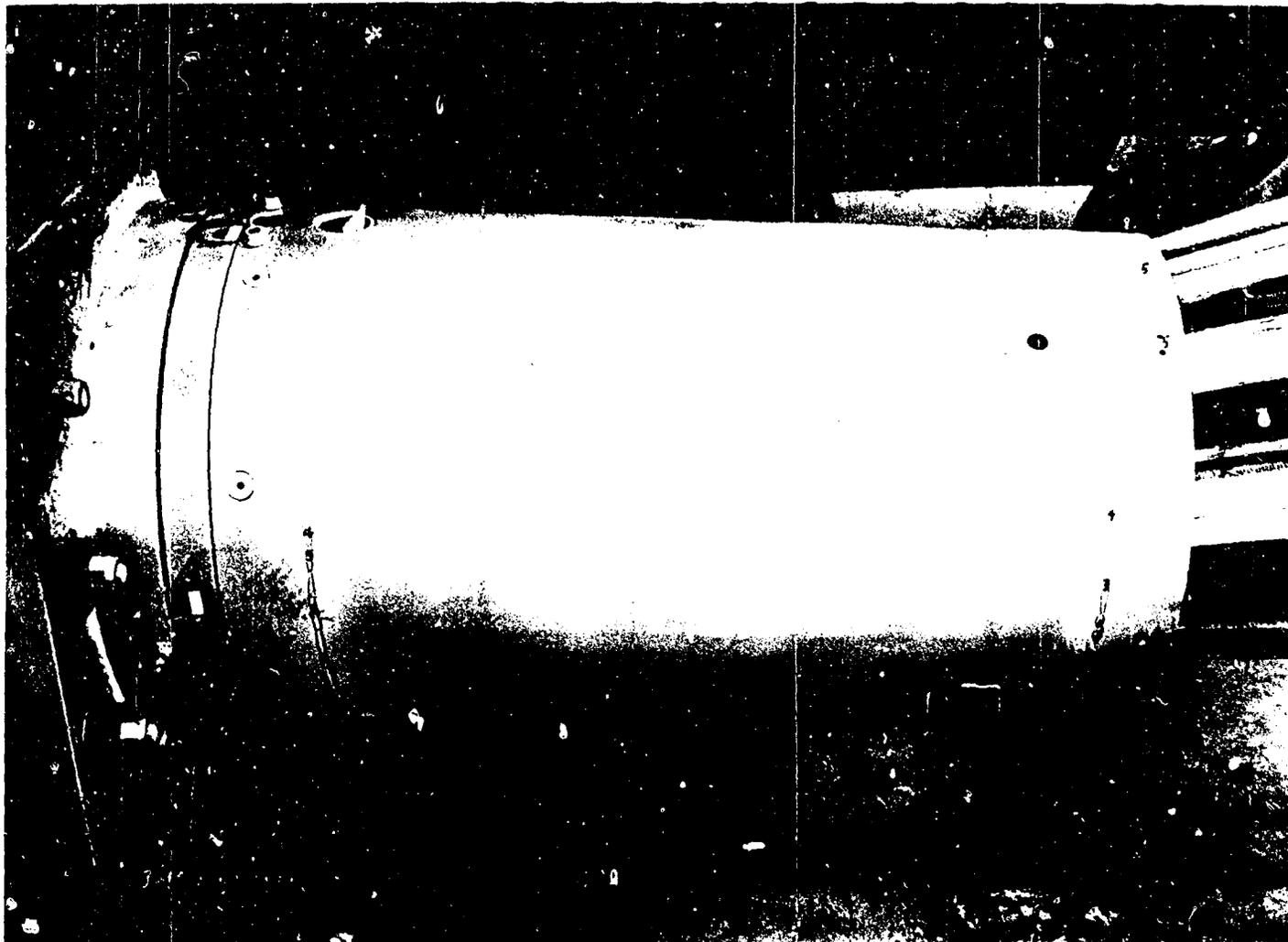
FIG. 13 -- AREAS OF STRESSCOAT CRACKS FOR 15° FULL (SECOND CASE) --
STATIC TEST OF TX-2E/XI AFT CASE PARACHUTE LUG RING.

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FIG. 14 -- AREAS OF STRESSCOAT CRACKS FOR 15° PULL (SECOND CASE) --
STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

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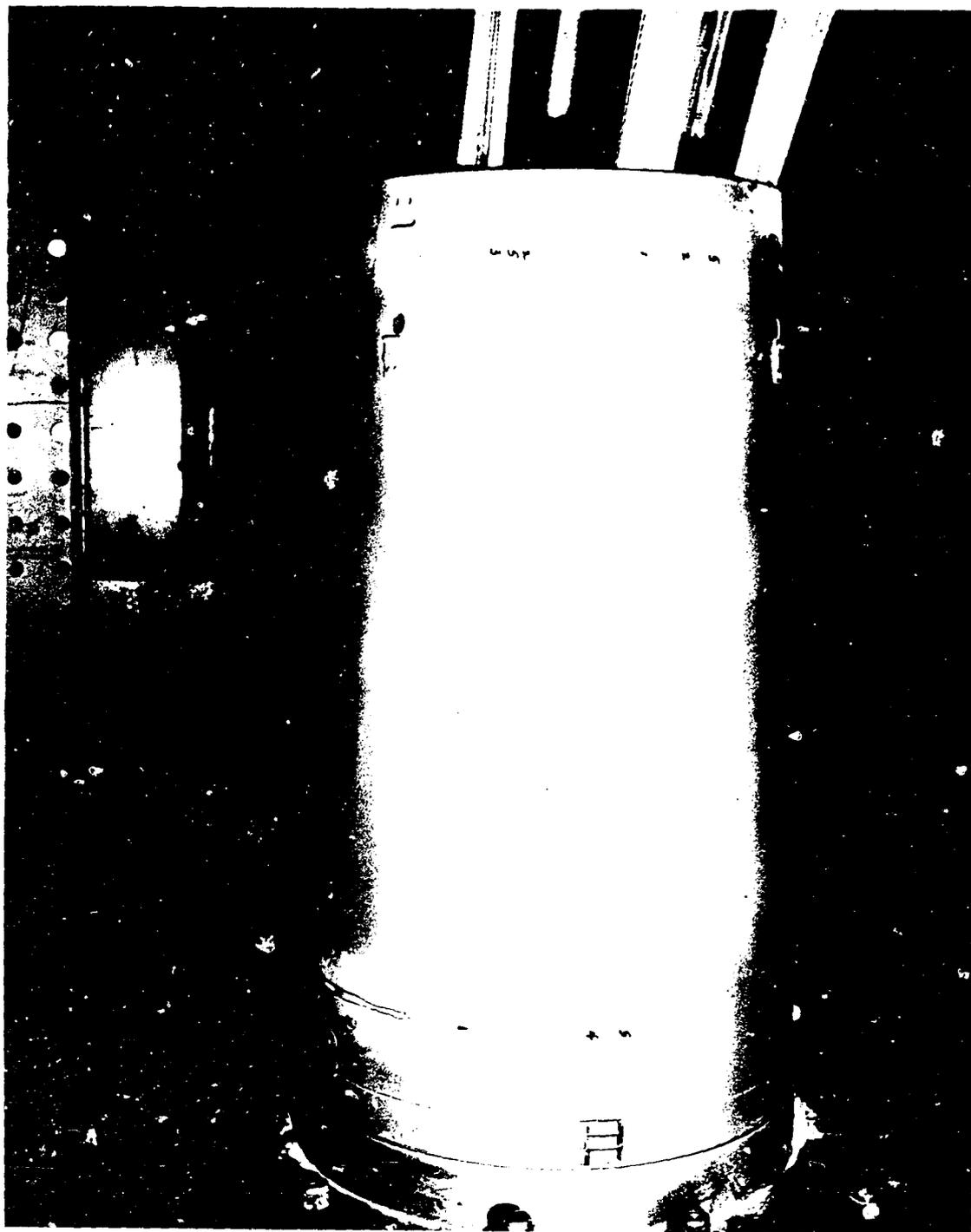


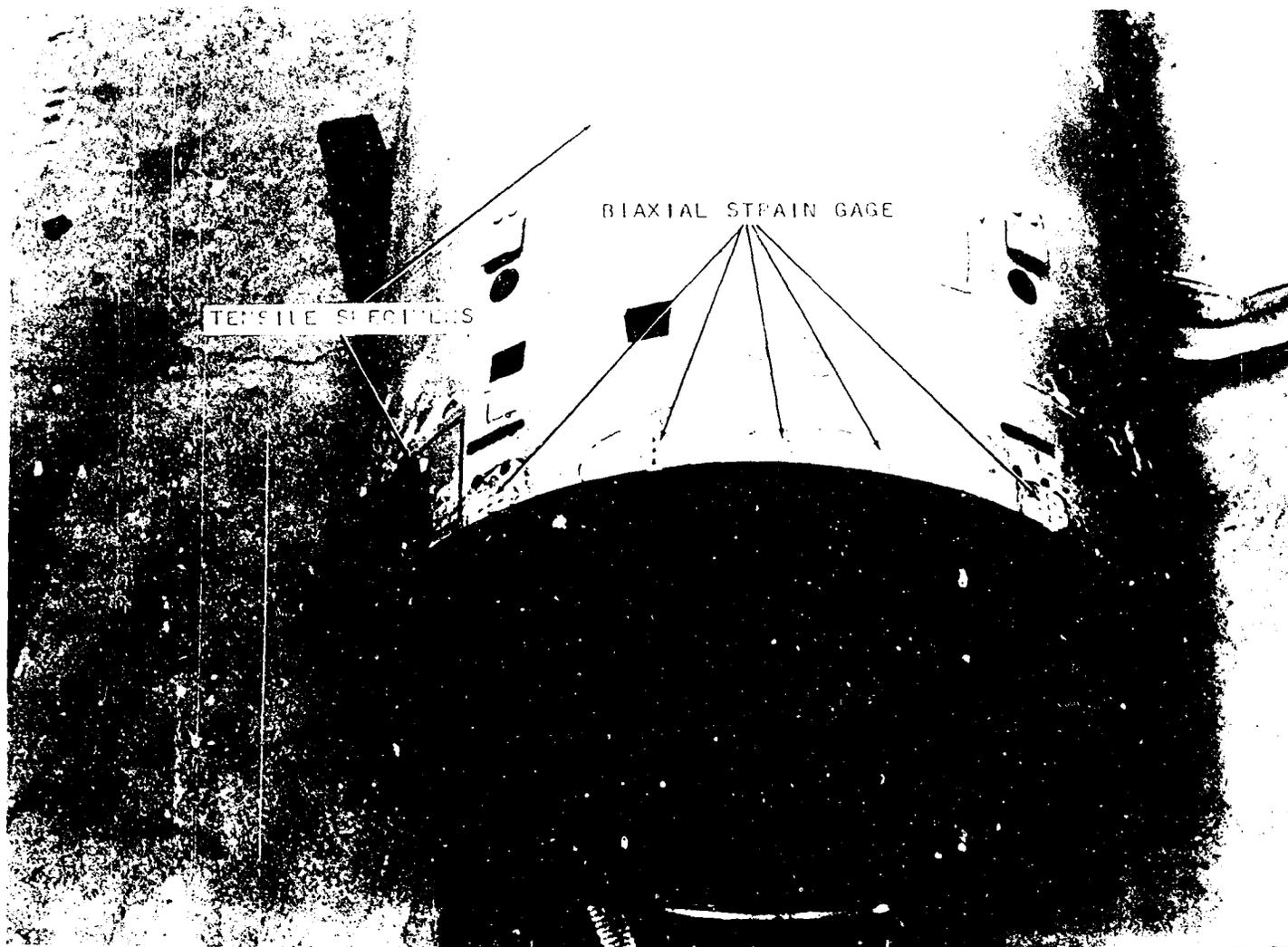
FIG. 15 -- AREAS OF STRESSCOAT CRACKS FOR 15° PULL (SECOND CASE) -->
STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

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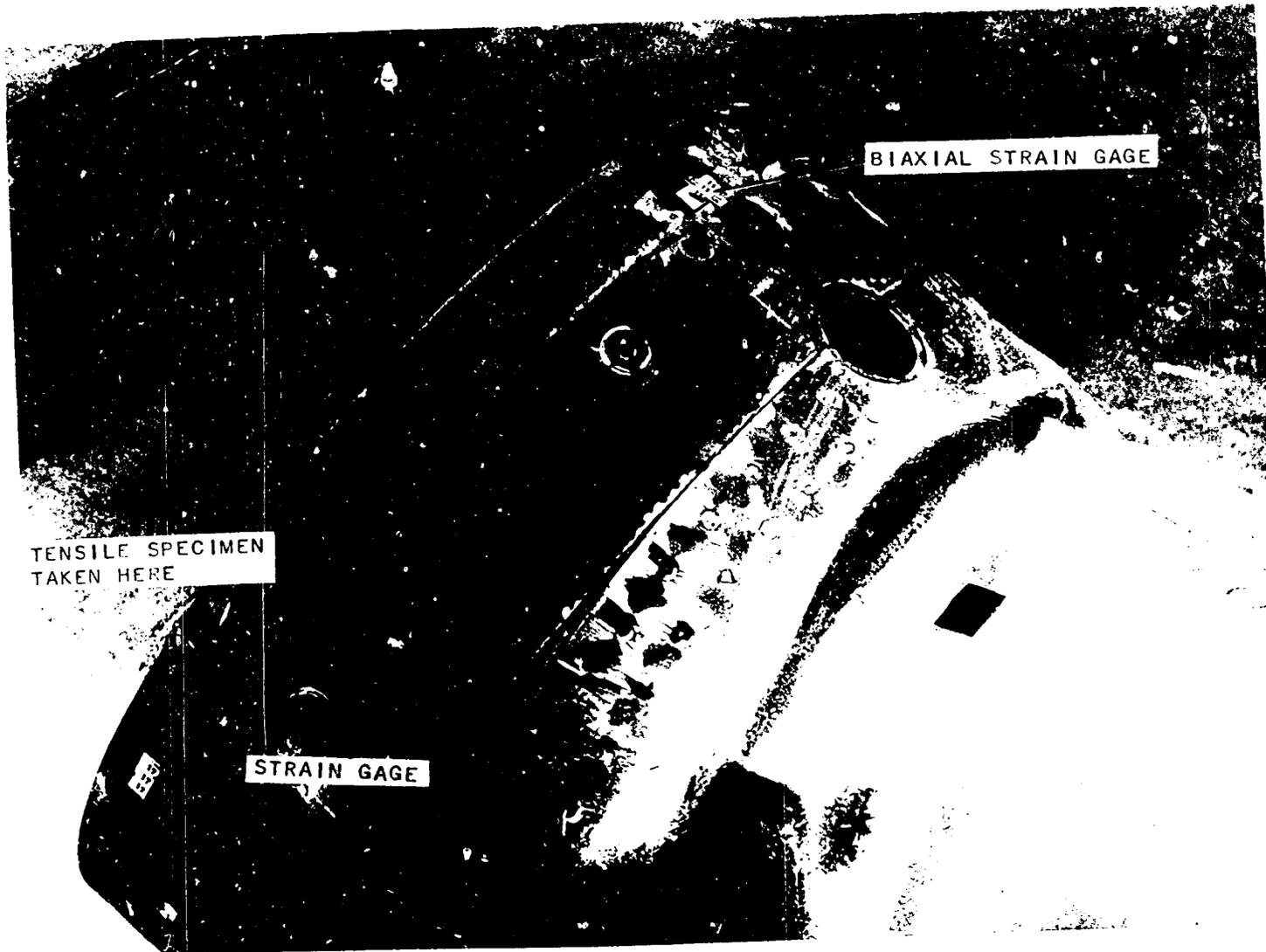
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FIG. 16 -- LOCATION OF STRAIN GAGES (SECOND CASE) -- STATIC TEST OF TX-2⁰/XI AFT CASE PARACHUTE LIG RING.

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FIG. 17 -- LOCATION OF STRAIN GAGES (SECOND CASE) -- STATIC TEST OF TX-28/X1 AFT CASE PAPACHITE LUG RING.

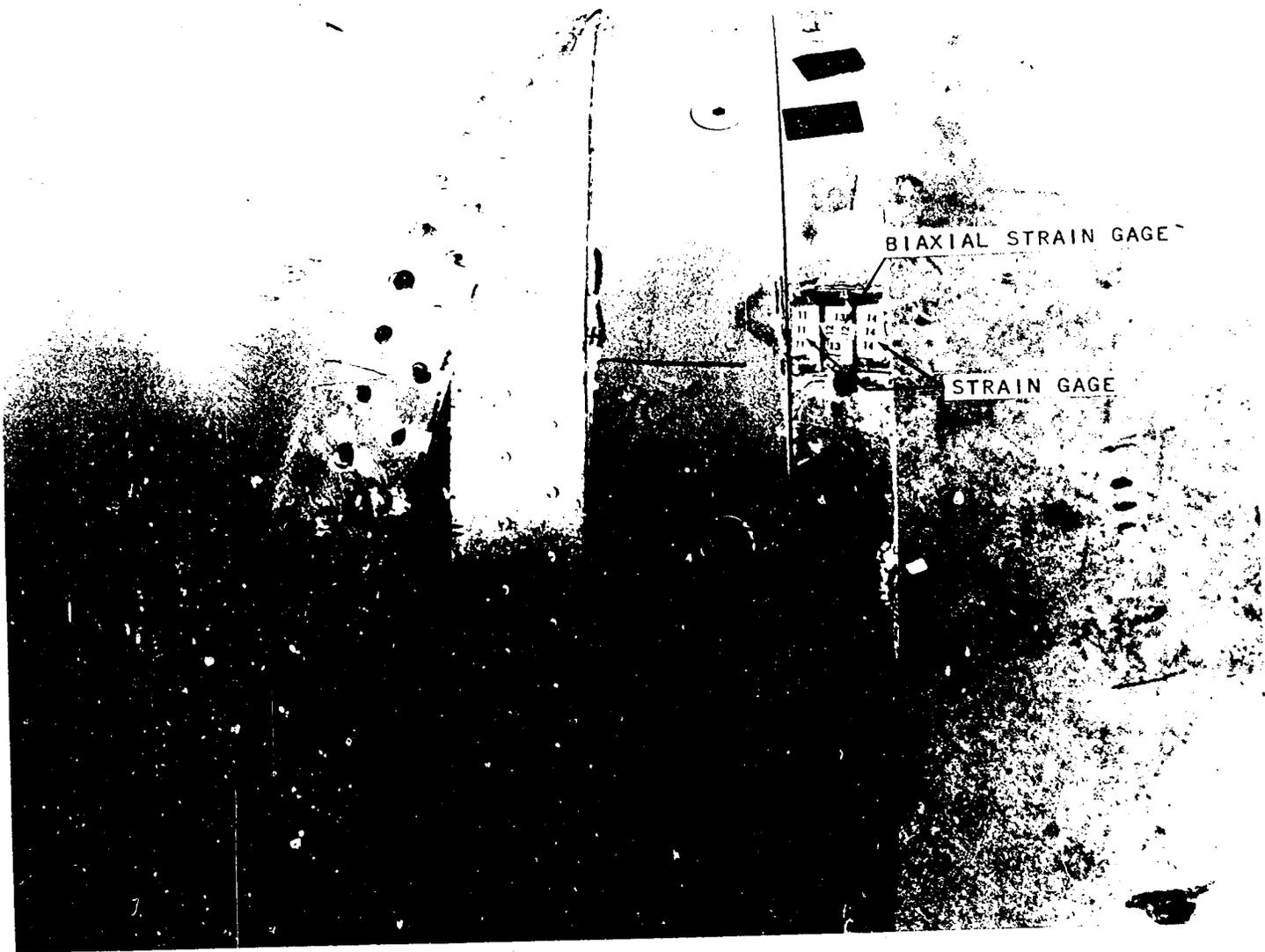
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PROJECT NO. TM-548

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FIG. 18 -- LOCATION OF STRAIN GAGES (SECOND CASE) -- STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

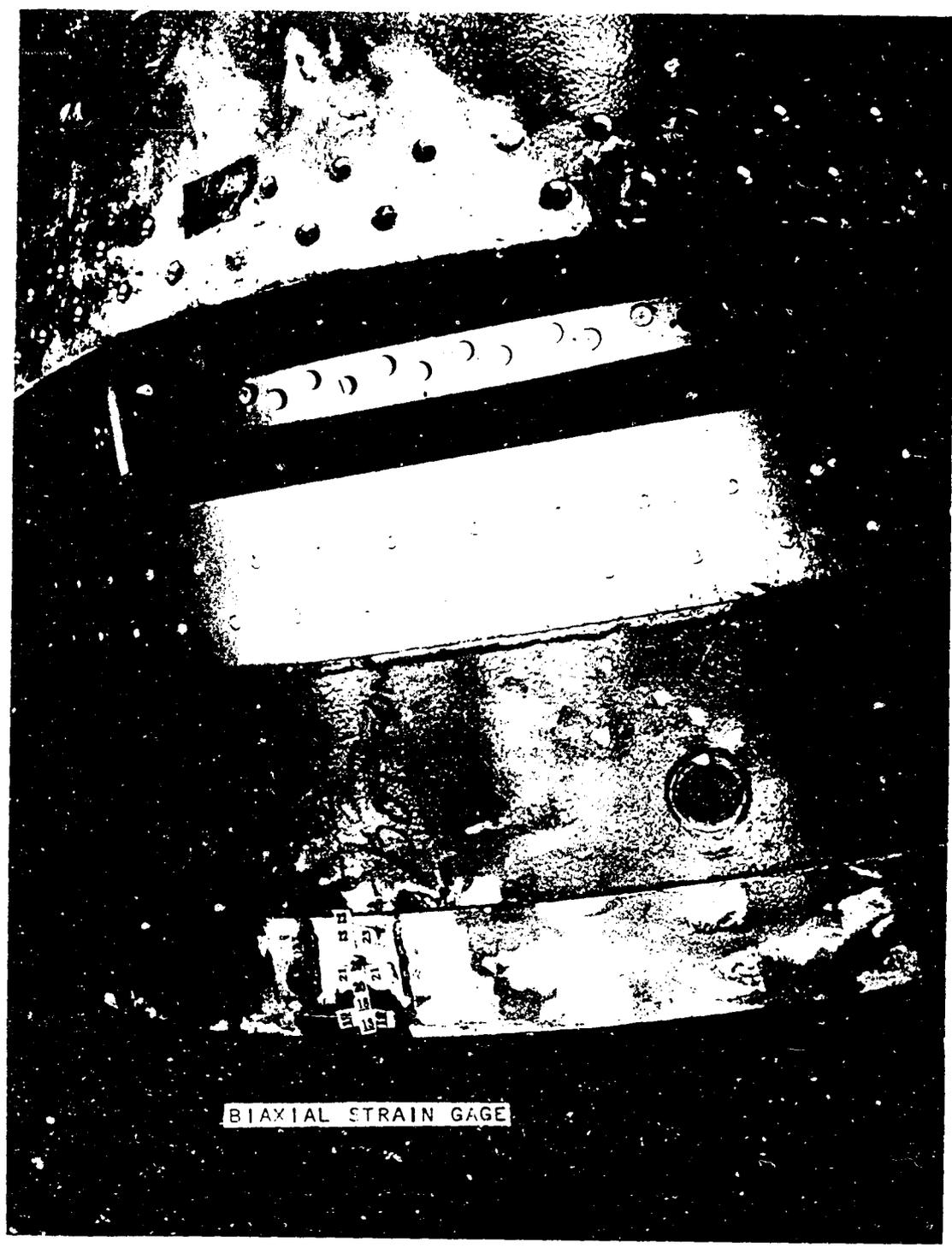
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PROJECT NO. TM-548

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BIAXIAL STRAIN GAGE

FIG. 19 -- LOCATION OF STRAIN GAGES (SECOND CASE) -- STATIC TEST OF TX-28/XI AFT CASE FAPACHITE LUG PING.

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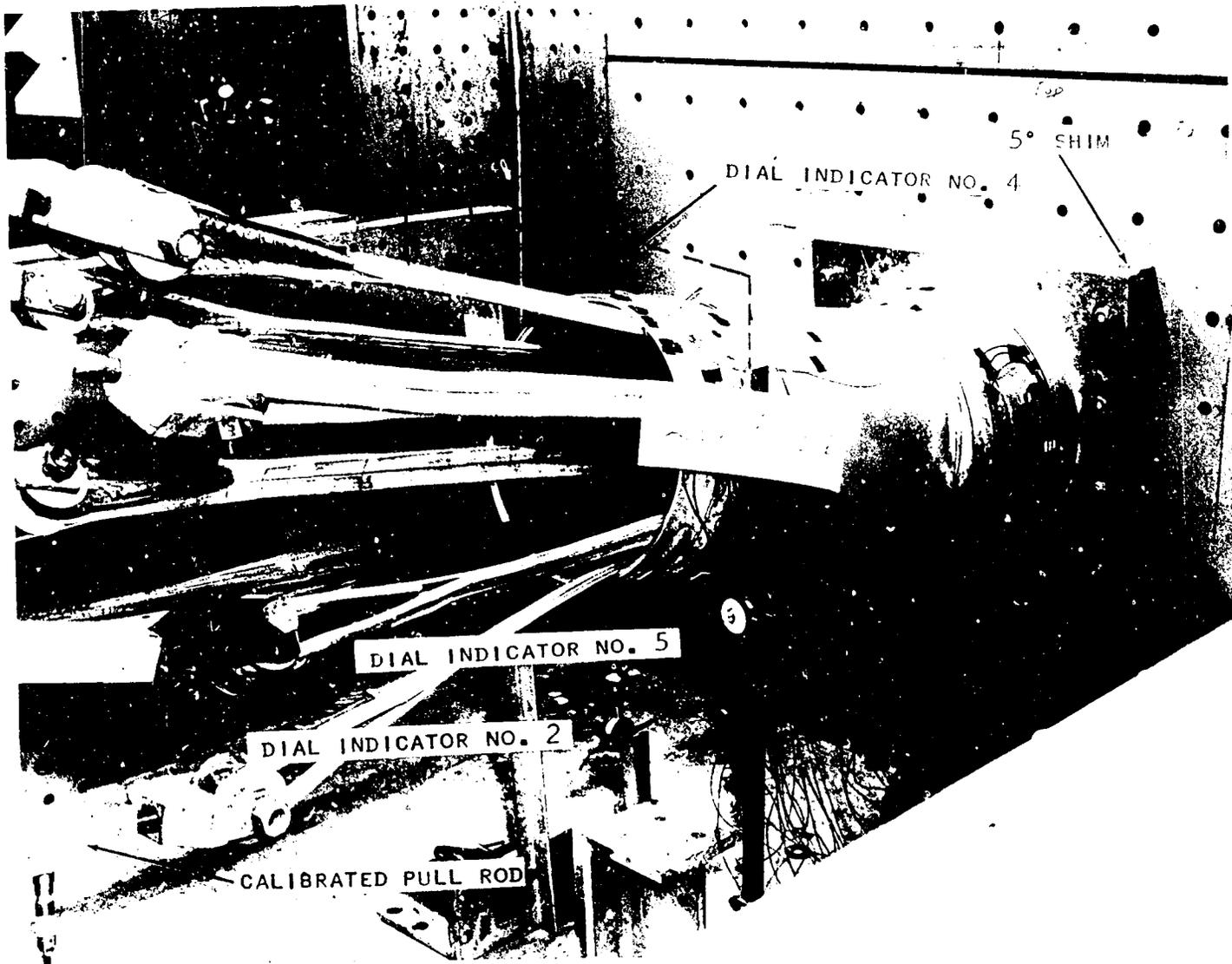
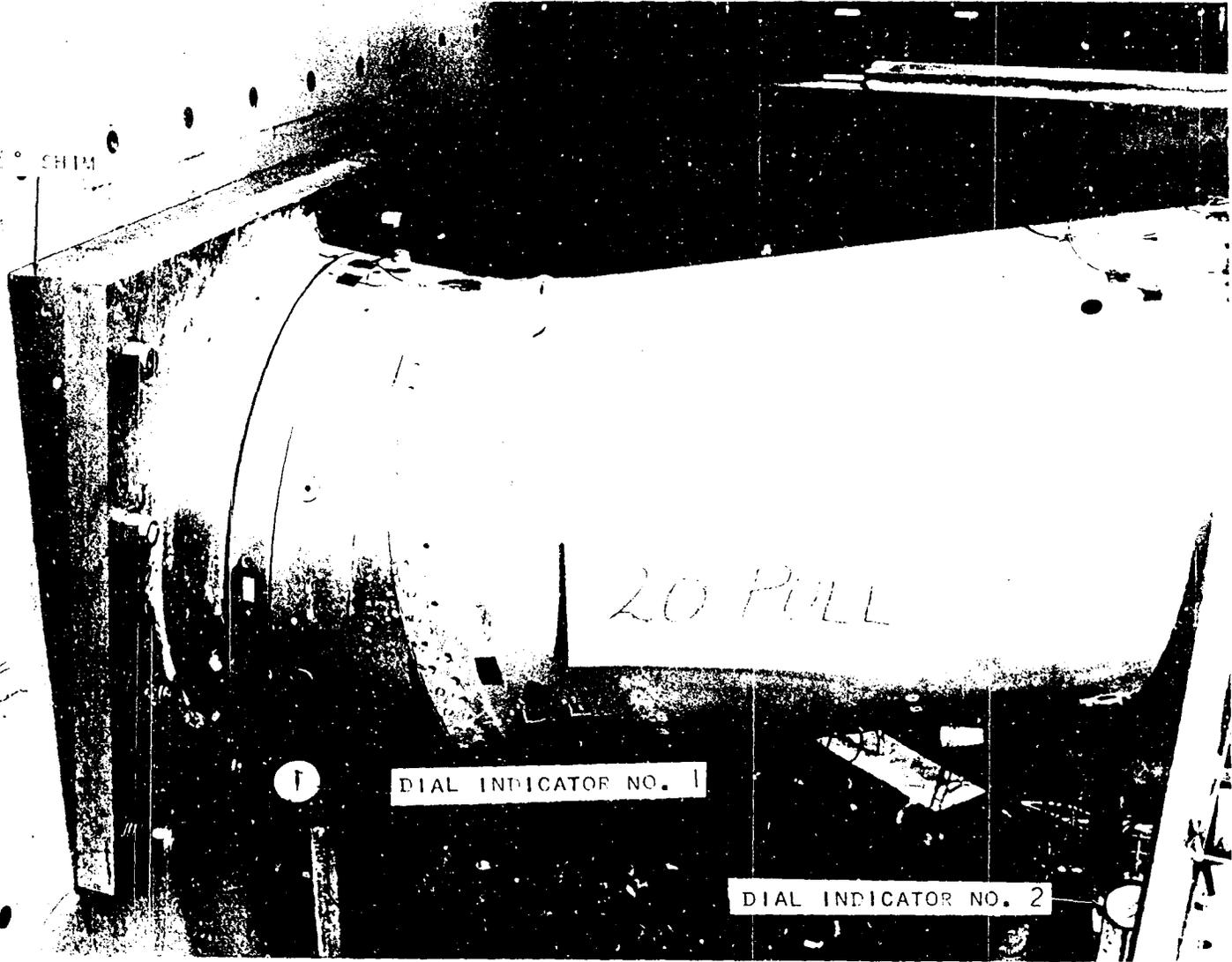


FIG. 20 -- FAILURE OF CASE RESULTING FROM 20° FULL (SECOND CASE) --
STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG RING.

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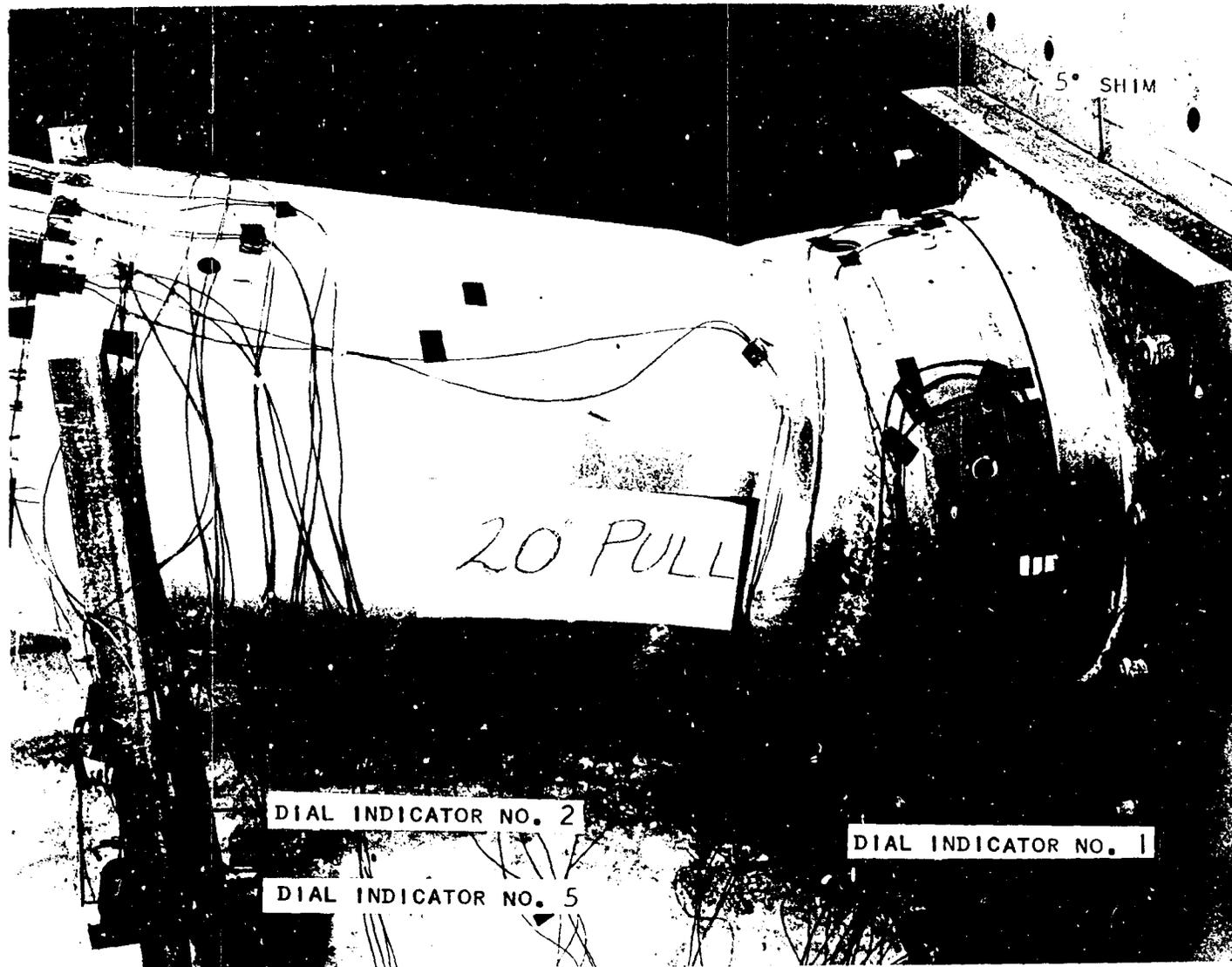
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FIG. 21 -- FAILURE OF CASE RESULTING FROM 20° FULL (SECOND CASE) --
 STATIC TEST OF TX-22/X1 AFT CASE PARACHUTE LUG RING.

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FIG. 22 -- FAILURE OF CASE RESULTING FROM 20° FULL (SECOND CASE) --
STATIC TEST OF TX-28/X1 AFT CASE PARACHUTE LUG PING.

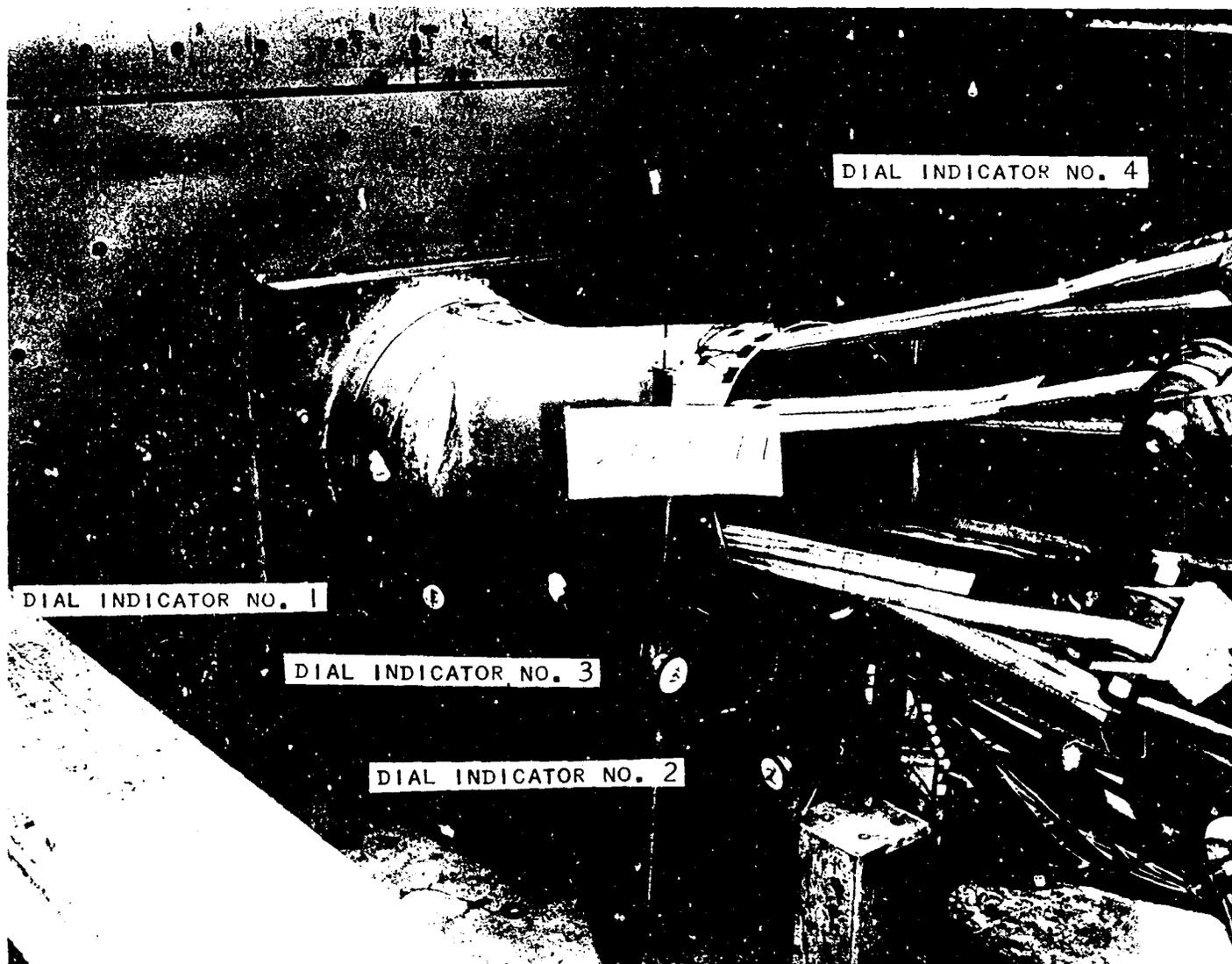
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FIG. 23 -- FAILURE OF CASE RESULTING FROM 20° FULL (SECOND CASE) --
STATIC TEST OF TX-2^o/X1 AFT CASE PARACHUTE LUG RING.

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Project No. TM-542

TABLE I

STRESSCOAT DATA (PILOT CASE) -- STATIC TEST OF TX-2B/X1 AFT CASE
PARACHUTE LUG RING

Load in Pounds	Stresscoat Sensitivity in <u>Microinches per Inch</u>	Stresscoat Designations <u>(Figs. 2 through 5)</u>
20,000	850	--
24,000	850	--
29,000	850	1
35,000	850	2
42,000	850	3
50,000	850	4
60,000	850	5
72,000	850	6
87,000	850	7
104,000	850	8
125,000	850	9

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TABLE II

STRAIN READINGS AND COMPUTED PRINCIPAL STRESSES (10° FULL) (FIRST CASE) -- STATIC TEST OF TX-28(X) AFT CASE PARACHUTE LUG RING

Load in Pounds	Strain Readings at the Following Gages - Microinches per Inch																								
	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20-21-22	23-24-25				
10,000	-177	-131	42	281	-279	-216	48	101	-143	41	7	-32	35	14	1	-72	34	-9	227	-26	54	-22	-32	57	53
20,000	-321	-313	72	362	-473	-255	228	134	-259	69	33	-99	183	156	27	-220E	80	6	468	-51	102	-44	-57	109	104
30,000	-532	-553	125	423	-715	-452	369	224	-361	133	45	-128	251	253	126	-309	112	30	755	-65	144	-64	-76	151	143
40,000	-669	-675	159	483	-913	-637	446	285	-459	210	44	-147	295	310	116	-647	122	47	1110	-73	175	-38	-93	186	171
50,000	-750	-710	190	490	-1060	-1010	547	323	-540	230	40	-157	330	379	140	-860	130	59	1440	-80	200	-120	-120	230	210
60,000	-830	-820	220	503	-1230	-1170	620	320	-620	230	40	-157	360	470	220	-1080	130	69	1680	-90	230	-130	-130	260	220
70,000	-1030	-1030	250	550	-1420	-1310	750	1170	-770	430	30	-167	380	560	320	-1360	160	100	1920	-100	230	-120	-120	280	240
80,000	-1130	-1100	290	600	-1620	-1470	870	1340	-740	530	30	-180	440	660	420	-1550	190	130	2280	-120	230	-130	-130	320	250
90,000	-1230	-1240	330	650	-1810	-1560	980	1470	-820	590	40	-170	470	730	460	-1750	220	140	2520	-130	240	-240	-160	320	250
100,000	-1430	-1400	370	710	-1920	-2350	1190	1650	-850	680	40	-220	560	730	550	-1760	260	170	2730	-130	240	-260	-160	370	260
110,000	-1580	-1570	390	730	-2150	-2920	1240	1870	-900	780	40	-240	630	840	600	-2150	280	230	2960	-140	250	-270	-160	370	260
120,000	-1700	-1710	440	840	-2340	-2170	1370	1950	-950	790	30	-240	660	870	730	-2330	290	240	3140	-140	250	-290	-130	380	260
130,000	-1850	-1860	470	920	-2520	-2320	1540	2150	-1020	860	30	-240	730	390	780	-2450	330	250	3330	-150	270	-330	-130	390	300
140,000	-2070	-2080	530	760	-2620	-2240	1740	2400	-1070	830	-10	-260	760	360	840	-2420	330	270	3420	-170	340	-300	-130	440	270

Load in Pounds	Maximum and Minimum Principal Stresses - psi			
	20-21-22		23-24-25	
	Su	Sv	Su	Sv
10,000	+200	-1000	+300	-600
20,000	+500	-1900	+550	-1300
30,000	+600	-2600	+700	-1700
40,000	+700	-3350	+950	-2200
50,000	+800	-3900	+1250	-2650
60,000	+900	-4510	+1500	-2900
70,000	+900	-5700	+1500	-3100
80,000	+600	-5500	+1650	-3200
90,000	+450	-6200	+2000	-3400
100,000	+125	-6500	+2150	-3500
110,000	+400	-6800	+2250	-3300
120,000	+400	-7100	+2600	-3350
130,000	+300	-7200	+2450	-4150
140,000	+800	-8200	+1600	-5500

Gages 1 thru 8 and No. 19 are mounted on 14S-T6 aluminum.
Gages 9, 10, and 20 thru 25 are mounted on 24S-T36
Gages 11 thru 18 are mounted on 4130 steel (H.T. 125,000 - 145,000 psi).

Su = Maximum principal stress (largest algebraic values).
Sv = Minimum principal stress (smallest algebraic values).

Ref. Syn: 1612 (486)
Project No. TM-548

TABLE III

STRESSCOAT DATA (SECOND CASE, DWG. NO: DL-10021) -- STATIC TEST OF
TX-28/X1 AFT CASE PARACHUTE LUG RING

Load in Pounds	Stresscoat Sensitivity in Microinches per Inch	Stresscoat Designations (Figs. 13 through 15)
20,000	800	1
24,000	800	2
28,800	800	3
34,600	800	4
41,500	800	5

TABLE IV

STRAIN READINGS AND COMPUTED PRINCIPAL STRESSES (15° FULL) (SECOND CASE) -- STATIC TEST OF TX-24/X1 AFT CASE PARACHUTE LUG RING

Load in Pounds	Strain Readings at the Following Gages - Microinches per Inch															
	1-2	3-4	5-6	7-8	9-10	11	12-13	14	15-16	17	18-19	20-21	22-23	24	25	
3,000	156	-38	329	-41	106	-50	373	-118	313	-65	-11	31	-39	-52	16	-2
16,000	470	-207	1120	-171	424	-114	1270	-411	1070	-158	9	-50	-111	56	-2	37
24,000	720	-377	1670	-250	714	-173	1840	-571	1540	-224	-136	114	-12	-121	125	-20
32,000	1000	-550	2270	-350	1000	-240	2470	-770	2070	-300	-210	250	50	-141	180	-30
40,000	1400	-820	2970	-490	1300	-330	3170	-1000	2770	-390	-280	370	100	-190	250	-40
48,000	1800	-1100	3770	-650	1600	-430	3970	-1300	3570	-490	-370	470	150	-240	320	-50
56,000	2200	-1400	4570	-830	1900	-540	4770	-1600	4370	-590	-460	570	200	-290	400	-60
64,000	2600	-1700	5370	-1030	2200	-660	5570	-1900	5170	-690	-550	670	250	-340	490	-70
72,000	3000	-2000	6170	-1250	2500	-790	6370	-2200	5970	-800	-630	770	300	-390	540	-80
80,000	3400	-2300	6970	-1480	2800	-930	7170	-2500	6770	-920	-710	870	350	-440	600	-90
100% L.L.	4200	-2800	8170	-1800	3400	-1100	8370	-3000	8170	-1100	-800	1000	400	-500	700	-100
88,000	3800	-2500	7370	-1580	3000	-930	7570	-2700	7370	-930	-700	900	450	-450	750	-110
96,000	4200	-2800	8170	-1800	3400	-1100	8370	-3000	8170	-1100	-800	1000	400	-500	700	-100
104,000	4600	-3100	8970	-2050	3800	-1280	9170	-3300	8970	-1280	-900	1100	450	-550	750	-110
112,000	5000	-3400	9770	-2300	4200	-1480	9970	-3600	9770	-1480	-1000	1200	500	-600	800	-120

Load in Pounds	Maximum and Minimum Principal Stresses - psi																			
	1-2		3-4		5-6		7-8		9-10		12-13		15-16		18-19		20-21		22-23	
	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv
3,000	1500	130	360	360	2100	160	3930	100	3430	470	1600	-940	430	-130	-170	-150	-1450	-3560	-520	-2320
16,000	5120	620	7350	710	4550	360	8150	270	7410	970	-210	-1510	1600	-50	-1220	-1240	-3220	-10600	50	-6070
24,000	8700	1120	12070	1100	7660	590	12430	470	10250	1150	5020	750	3380	210	-1630	-2300	-5270	-17820	10	-9920
32,000	11780	1460	18060	1460	11580	910	16760	770	13250	1330	8470	1070	5450	320	-1070	-2630	-7420	-23930	-520	-12350
40,000	15240	1840	22420	2410	15570	1330	20670	1000	14060	1330	12390	1170	7580	1200	-2170	-3270	-10640	-29230	-1070	-15820
48,000	19380	2130	27200	2910	19510	1520	24130	1300	18180	1700	17630	1370	9710	1570	-2640	-4000	-11640	-34750	-1450	-18610
56,000	23970	2430	31930	3450	22350	1820	27730	1700	20970	2120	24780	15760	13660	2220	-3300	-5000	-15540	-45470	-770	-24520
64,000	28100	2670	36600	3700	25380	2030	30360	1800	23170	2450	26700	17310	15390	2570	-3570	-5700	-17120	-51220	-1620	-28470
72,000	31770	3170	40200	3900	27900	2260	32570	1910	25310	2790	28560	14660	17610	2870	-3820	-5800	-18730	-56860	-390	-31710
80,000	35300	3560	43600	4100	30570	2490	34770	2070	27370	3410	30240	2080	19530	3460	-3320	-5800	-20240	-62500	-800	-35140
100% L.L.	42000	40200	50200	47000	37270	3410	42770	3470	34700	4130	34490	21580	21560	3510	-4230	-6200	-24320	-58240	250	-38270
88,000	36460	4540	44200	41200	33200	3740	37700	3740	34320	4320	34320	23230	23770	4270	-4550	-6300	-23260	-71110	560	-41930
96,000	39700	4920	47600	44200	34900	4070	39800	4070	35850	4150	34850	24070	25210	4360	-5170	-6500	-24230	-73160	1160	-44300

Stresses below this line are above Y.P. of material.

Gages 1 thru 10 are mounted on 61S-T6 aluminum.

Gages 11 thru 23 are mounted on 4130 steel (H.T. 125,000 - 125,000 psi).

Stresses at biaxial gages are computed on assumption that gages are oriented in direction of principal stresses.

Su = maximum principal stress (largest algebraic value).
Sv = minimum principal stress (smallest algebraic value).

TABLE V

DEFLECTION READINGS (SECOND CASE, DL-10021) -- STATIC TEST OF TX-28/X1 AFT

Per Cent of Limit Load	Deflections in Inches						
	15° Pull		20° Pull				
	1	2	1	2	3	4	5
10	0	.013	-.001	.051	--	--	--
20	-.003	.057	-.003	.103	.040	.117	.036
10	--	--	-.002	.075	.012	.034	.007
30	-.002	.091	.004	.165	.077	.245	.077
10	--	--	.006	.095	.014	.056	.012
40	.002	.131	.033	.240	--	.375	.112
10	.003	.038	.032	.136	.019	.097	.016
50	.006	.175	.062	.314	--	.509	.144
10	.006	.053	.058	.165	.027	.125	.020
60	.013	.225	.078	.363	--	.599	.166
10	.011	.071	.071	.160	.027	.147	.020
70	.021	.274	.092	.433	--	.740	.199
10	.017	.090	.085	.208	.039	.197	.030
80	.030	.322	.106	.486	--	--	.229
10	.024	.110	.093	.216	--	--	.030
90	.041	.383	.117	.537	.688	--	--
10	.032	.132	.098	.221	.049	--	--
100	.046	.415	.130	.588	.675	--	--
10	.037	.148	--	--	--	--	--
110	.054	.463	.138	.703	--	--	--
10	.042	.163	--	--	--	--	--
120	.061	.509	--	--	--	--	--
10	.048	.181	--	--	--	--	--
130	.071	.562	--	--	--	--	--
10	.053	.214	--	--	--	--	--
137.5	.075	.598	--	--	--	--	--
10	.061	.260	--	--	--	--	--

All dial indicators are supported on the floor.

A plus deflection for dial indicators 1, 2, and 4 indicates a vertical movement upward.

A plus deflection for dial indicators 3 and 5 indicates a movement toward the longitudinal axis of the case.

Readings discontinued on 20° pull at about 100 per cent design limit load.

TABLE VI

STRAIN READINGS AND COMPUTED PRINCIPAL STRESSES (20° FULL) (SECOND CASE) — STATIC TEST OF TX-23/41 AFT CASE FA. ACUTE LUG FING

Load in Pounds	Strain Readings at the Following Gages - Microinches per Inch																						
	1-2		3-4		5-6		7-8		9-10		11	12-13		14	15-16		17	18-19		20-21		22-23	
3,000	356	-37	575	-141	379	-111	166	-142	204	-50	334	-36	-32	-29	-24	-1	-42	21	-105	-52	-56	-40	-52
16,000	-12	-100	1210	-370	930	-280	1700	-200	620	-150	270	-50	-30	-141	60	-20	-50	0	-70	-320	-50	-210	20
24,000	120	-310	2320	-430	1530	-440	1610	-50	990	-230	230	130	10	-120	150	-20	-10	-20	-70	-500	-30	-350	50
32,000	1450	-370	2670	-430	2110	-610	260	-70	1370	-320	-240	370	130	-20	240	-50	20	-40	-70	-210	-10	-490	110
40,000	2060	-430	3200	-70	2630	-730	2510	-730	1710	-210	-240	540	240	-50	330	-60	70	-50	-30	-1020	-20	-620	170
48,000	2330	-540	3640	-360	2000	-360	2830	-320	1920	-440	-330	640	300	1	410	-70	110	-70	-30	-1470	-20	-390	220
56,000	2790	-510	4250	-990	3760	-1020	3410	-1020	2270	-120	-490	740	330	100	540	-30	150	-70	-30	-1630	-10	-1030	280
64,000	3100	-710	4720	-1120	4260	-1130	3750	-1150	2430	-570	-300	820	430	160	600	-110	190	-30	-30	-1830	-10	-1160	330
72,000	3400	-710	5210	-1220	4730	-1240	4340	-1230	2590	-580	-430E	910	430	200	740	-120	240	-30	-30	-1910	0	-1300	390
80,000	3770	-840	5970	-1360	5340	-1290	4230	-1320	2840	-840	-570	970	540	250	840	-140	280	-110	-70	-2110	0	-1300	390
(100% L.L.)																							
88,000	4010	-320	6670	-1420	5240	-1340	4770	-1460	3080	-630	-1050	1080	590	200	930	-150	300	-120	-900	-2320	20	-1450	460
96,000	4250	-420	7350	-1510	5500	-1430	5330	-1690	3290	-730	-1160	1170	640	310	1040	-170	380	-140	-30	-2560	30	-1610	540
104,000	4570	-740	8520	-1610	6270	-1510	6380	-1360	3590	-770	-1300	1330	790	370	1130	-180	500	-150	-110	-2770	20	-1270	660
106,000	Unit failed at this load. No readings taken after failure.																						

Load in Pounds	Maximum and Minimum Principal Stresses - psi																					
	1-2		3-4		5-6		7-8		9-10		12-13		15-16		18-19		20-21		22-23			
	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv	Su	Sv		
3,000	3850	400	6220	640	4260	290	4730	190	2210	230	-3470	-3550	-170	-460	-370	-3140	-2310	-2620	-1770	-2050		
16,000	1800	1310	13900	1600	10280	620	11230	790	6710	720	-2430	-3370	1720	-50	-830	-2870	-4760	-13700	-1440	-6500		
24,000	12620	1400	21010	2440	16290	950	17100	1150	10760	1260	4250	1610	4390	210	-1430	-2750	-6790	-19120	-1370	-10670		
32,000	17910	2020	28780	3280	22450	1270	23860	1070	15120	1800	13090	7810	7160	780	-1170	-2630	-3130	-25940	-1340	-14540		
40,000	22390	2600	34760	3790	28110	1340	27520	1740	18530	2020	19600	13000	9030	1350	-2390	-3050	-10920	-33370	-1980	-13470		
48,000	25330	2980			32010	2250	30690	1280	20390	2510	23380	15900	12330	1820	-3020	-3240	-12600	-36800	-1700	-21230		
56,000	30310	3240					36150	1650	24700	2970	27360	13440	16340	2470	-3120	-3560	-15170	-47300	-1730	-26120		
64,000	32730	3460							26700	3220	30410	21330	17010	2720	-3440	-3660	-16330	-53600	-1250	-30090		
72,000	37210	4200							28240	3550	32780	24310	22420	3490	-3760	-4360	-19210	-61030	-750	-33740		
80,000									30940	3940	36920	27020	25410	3840	-4060	-4400	-23870	-67310	-420	-37610		
(100% L.L.)																						
88,000									34620	4330	40270	29500	27120	4410	-4360	-4720	-23530	-74210	320	-4170		
96,000									38870	4720	43650	31990	31500	4260	-4710	-5260	-26270	-81060	1310	-16720		
104,000									40640	37050			34270	5420	-4790	-5370	-30770	-9240	2560	-53130		

Stresses below this line are above Y.P. of material

Gages 1 thru 10 are mounted on 618-T6 aluminum.

Gages 11 thru 23 are mounted on A177 steel (H.T. 125,000 - 1145,000 psi).

Stresses at biaxial gages are computed on assumption that gages are oriented in direction of principal stresses.

Su = maximum principal stress (largest algebraic value).

Sv = minimum principal stress (smallest algebraic value).

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Ref. Sym: 1612 (486)
Project No. TM-548

TABLE VII

RESULTS FROM PULLING TENSILE SPECIMENS TO FAILURE -- STATIC TEST OF
TX-28/X1 AFT CASE PARACHUTE LUG RING

Specimen No.	Size of Cross Section	Area Sq. In.	Ultimate		Yield	
			Load (lbs.)	Stress (psi)	Load (lbs.)	Stress (psi)
<u>Interior Skin on Case</u>						
1	.064 x .50	.0320	1360	42,500	1130	36,700
2	.064 x .50	.0320	1395	43,600	1210	37,800
3	.064 x .497	.0313	1390	43,600	1215	38,200
Average				43,200		37,600
<u>Exterior Skin on Case</u>						
1	.069 x .498	.0344	1500	43,600	1190	36,600
2	.071 x .498	.0354	1490	42,100	1200	34,900
3	.071 x .497	.0353	1505	42,600	1205	34,200
Average				42,800		34,300
<u>Ring that Attaches to Case Skin at Mounted End</u>						
1	.127 x .499	.0634	3035	47,900	2750	43,400
2	.127 x .502	.0636	3010	47,300	2750	43,200
3	.129 x .501	.0646	3085	47,800	2760	42,700
4	.127 x .501	.0637	2945	46,300	2700	42,400
Average				47,300		42,900
<u>Ring that Attaches to Case Skin at Open End</u>						
1	.126 x .502	.0633	2665	42,100	2500	39,500
2	.129 x .503	.0649	2760	42,500	2480	38,200
3	.130 x .502	.0653	2680	41,300	2550	39,100
4	.128 x .500	.0640	2655	41,500	2540	39,700
Average				41,900		39,100

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