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**BURST TEST QUALIFICATION ANALYSIS
OF DWPF CANISTER-PLUG WELD (U)**

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FEBRUARY 1995

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ASET D

APPLIED SCIENCE AND ENGINEERING TECHNOLOGY DEPARTMENT

DWPF Canister Plug
Stress Analysis
ASME Section IX
Resistance Weld
Shear Test
Retention - Permanent

**BURST TEST QUALIFICATION ANALYSIS
OF DWPF CANISTER-PLUG WELD (U)**

By

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SUMMARY

The DWPF canister closure system uses resistance welding for sealing the canister nozzle and plug to ensure leak tightness. The welding group at SRTC is using the burst test to qualify this seal weld in lieu of the shear test in ASME B&PV Code, Section IX, paragraph QW-196. The burst test is considered simpler and more appropriate than the shear test for this application. Although the geometry, loading and boundary conditions are quite different in the two tests, structural analyses show similarity in the failure mode of the shear test in paragraph QW-196 and the burst test on the DWPF canister nozzle.

Non-linear structural analyses are performed using finite element techniques to study the failure mode of the two tests. Actual test geometry and realistic stress strain data for the 304L stainless steel and the weld material are used in the analyses. The finite element models are loaded until failure strains are reached. The failure modes in both tests are shear at the failure points. Based on these observations, it is concluded that the use of a burst test in lieu of the shear test for qualifying the canister-plug weld is acceptable.

The burst test analysis for the canister-plug weld also yields the burst pressures which compare favorably with the actual pressures found during burst tests. Thus, the analysis also provides an estimate of the safety margins in the design of these vessels.

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1.0 INTRODUCTION

1.1 Background

The Defense Waste Processing Facility (DWPF) canisters will be filled with vitrified radioactive waste and sealed by welding a plug into the canister nozzle using a resistance upset welding process. These canisters are cylinders approximately 2 ft in diameter and 10 ft tall fabricated from 0.375" thick plate with forged bottom and nozzle joined to the cylinder by girth welds [1]. All components are fabricated from Type 304L stainless steel. The plug is 0.5" thick and 5" in diameter, slightly oversized for the nozzle opening (interference fit).

The canisters are fabricated in accordance with ASME B&PV Code, Section VIII, Division 1 [2]. ASME Code, Section IX [3] gives the requirements for qualifying the welding procedures, welding equipment, and the welding operators for resistance welding. Section IX requirements for resistance welding are geared towards thin plate and sheet metal components generally not used in the design of pressure vessels. Resistance weld qualification tests are called shear tests and are given in Section IX, paragraph QW-196 [3].

In the design of seal welds for the DWPF canisters, burst tests and metallography are used to qualify the seal welds to meet the Section IX requirements. In using the burst tests, rather than the shear tests as called out in Section IX, it is felt that burst tests are a better indicator of the structural integrity of the seal welds and are also easier to implement than the shear tests for this application.

Figure 1 shows a schematic of the DWPF canister, canister nozzle, and the plug arrangement.

1.2 Resistance Upset Welding

Resistance upset welding provides a reliable means of sealing the canisters and is well suited for use in the DWPF environment where its simplicity lends itself to remote, automatic operation. It is generally a solid-state process (no melting) using high temperature ($\approx 2100^{\circ}\text{F}$) and force to bond materials over a very short time period (< 2.5 seconds in this case). These short times at elevated temperature minimize the formation of heat affected zone in the material surrounding the weld. Bonding occurs due to the grain growth across the interface during high temperature deformation. Plastic deformation is, therefore, a key factor in the formation of these solid state welds. Solid-state welds thus formed are less susceptible to volumetric defects such as porosity from solubility changes during solidification of fusion welds.

1.3 Purpose

The analyses in this report will evaluate the seal weld in the canister-plug assembly and the spot weld in the shear test given in the ASME Code, Section IX [3]. This analysis will show that in spite of the differences in the two tests, the failure modes of the two tests are similar and the use of the burst test meets the intent of the Section IX requirements. The analyses will show the similarity of the state of strain at failure in both tests. However, no attempt is made to evaluate the weld soundness. The analyses will also be able to evaluate the safety margins in the canister seal weld, however, due to lack of stress strain data for the weld material, corroboration of test results is considered difficult.

2.0 MATERIALS OF CONSTRUCTION

2.1 Mechanical Properties for Base Metal

Mechanical properties of solution-annealed Type 304L stainless steel are obtained from Reference 4. True plastic strains are used for input into the finite element structural models. Appendix A gives the stress strain data used in the analyses in this report.

2.2 Mechanical Properties for Weld Metal

Mechanical properties of the weld metal (plastically deformed base metal at the interface) at the nozzle/plug interface change. Hardness values of the weld metal were determined from the actual samples of the canister weld. It is found that the hardness of the metal at the interface increases from the hardness of the annealed 304L hardness [5, 6]. An increase of 7% to 8% was found in Reference 5. No tensile tests were performed to obtain stress strain data for the weld metal. Since an increase in hardness corresponds with an increase in the tensile strength in general, annealed 304L stress values were increased by this amount (7 or 8%) to obtain the weld material properties. Appendix A gives the stress strain data used in the analyses for the weld metal.

3.0 LOADING CONDITIONS FOR THE ANALYSIS

3.1 Temperature and Pressure

The canister-plug weld is tested at room temperature (70°F). The shear test required in Section IX is also performed at room temperature. The canister-plug weld is pressure tested to failure. No pressure loading is present in the shear test.

3.2 Applied Loads

The canister-plug weld is subjected to pressure loading. No other loading such as thermal or seismic is considered important for the overall integrity of the canister weld. The shear test is subjected to tension loading only at the free ends of the plates.

4.0 STRUCTURAL ANALYSIS

4.1 Basic Assumptions

The following assumptions are made in the analyses.

1. The canister resistance weld is free of cracks or other fabrication flaws. *This is confirmed from the metallography of the welds [5] where no flaws are observed.*
2. The canister weld is vertical and lies at the interface of the nozzle and the plug. *The interface is distorted during the welding process. However, small geometrical deviations in the analysis model will not affect the results significantly in this study.*
3. For the shear test on the weld between two plates, reduction in thickness at the spot or projection weld is negligible.

4.2 Methodology

The finite element methodology was used to evaluate the response of the canister-plug weld and the shear test at failure loads. Since the strains are large at failure loads, nonlinear analyses are necessary to obtain the correct response. The general purpose structural analysis computer code ABAQUS [7] was used to perform the analyses.

4.3 Acceptance Criteria

1. For the canister weld, the burst pressure is equal to or greater than 2600 psig [8].
2. For the shear test, minimum weld strength is 5065 lbs as given in the ASME Code, Section IX, QW-462.10 [3] for material having ultimate strength between 90,000 and 149,000 psi.
3. Failure strain for the stainless steel base metal is estimated as 40% [4]. For the weld metal the failure strains are expected to be smaller due to increased hardness and presence of residual stresses.

5.0 FINITE ELEMENT MODEL

5.1 Finite Element (FE) Model for Canister-Plug Weld

The basic dimensions (Fig. 1) for generating the FE model were taken from drawings given in Reference 1. Since the canister-plug weld geometry is axisymmetric, axisymmetric solid elements CAX4 in the ABAQUS computer code were utilized. A total of 2370 elements were modeled to capture the detailed deformation patterns.

A review of the weld geometry shows that a crevice (see Figure 3 in Reference 5) is formed at the bottom of the plug during the welding operation. This crevice results in lack of bonding at the very bottom of the plug. This lack of bonding acts as sites for stress concentration. The finite element model is modified to reflect this lack of bonding. Figure 2 shows the finite element mesh of the model.

5.2 Finite Element (FE) Model for Shear Test

The basic dimensions (Fig. 1) for generating the FE model were taken from drawing details of the shear test in the ASME Code, Section IX, QW-462.9 [3]. Solid continuum elements C3D8R were used to capture the response for large deformation. Five elements were used through the thickness of each plate to get a fine mesh necessary for the large deformation problem. Figure 3 shows the finite element mesh of the model.

ASME Code, Section IX, QW-196 [3] requires that the spot weld size be at least $0.9\sqrt{t}$, where "t" is the thickness of the thinner plate. For a 1/8 inch thick plate, this is equal to a spot weld of 0.3182" diameter. A rectangular weld area of 0.3125" x 0.375" was modeled in the finite element model. A rectangular weld area in lieu of a circular area was modeled for simplicity. A total of 6000 solid 8 noded elements were used.

5.4 Boundary Conditions

5.4.1 Canister-Plug Weld Model

For the canister weld model, the lower end of the nozzle was restrained in the axial direction. This restraint was considered sufficient since it was far away from the weld area. Coding of exact boundary conditions to reflect connection of the nozzle to the canister body will have insignificant affect on the results in the weld area.

5.4.2 Shear Test Model

For the shear test model, one end of the lower plate (Plate 1 in Fig. 3) was fixed. The free end of the upper plate (Plate 2) was restrained in all directions except the longitudinal pull direction. This simulates the end being held in a vise and pulled in the longitudinal pull direction.

6.0 ANALYSIS RESULTS

Failure load is calculated as the load when the maximum equivalent plastic strain reaches failure strain for any element. Equivalent plastic strain is listed as the PEEQ value in the ABAQUS output results. The results are tabulated in Tables 1 and 2. Actual failure loads from burst tests for the canister weld and minimum failure loads from the ASME Code, Section IX, QW-462.10 [3] for the shear test are also listed.

Table 1 - Canister-Plug Weld Results

| Loading Condition | Element Number (†) | Pressure from Analysis (psig) | Failure Pressure from Burst Test (psig) | Maximum Plastic Strain, PEEQ [7] (%) |
|-------------------|--------------------|-------------------------------|---|--------------------------------------|
| Pressure | 230 | 9000 | ≈7000 | 24 |
| Pressure | 230 | 7000 | ≈7000 | 16 |

† See ABAQUS computer runs in Appendix B.

For the canister-plug weld it was found that the maximum plastic strain was only approximately 24% at 9000 psig internal pressure. The analysis was stopped at this stage. A review of the failed test samples shows that the failures occurred at the weld interface of the plug and the nozzle. Such failures could be attributed to increased hardness and residual stresses in the weld metal. A safety margin of 2.7 (≈ 7000/2600) over and above the required burst pressure of 2600 psig exists in the design. The test results appear to be consistent with the analysis in that mechanical tests performed at SRS on stainless steel weld metal coupons show failure strains to be in the range of 15 to 20% for the weld metal. Weld built-up (a stress riser) at the top of the plug-nozzle interface and shift in the weld interface were not modeled in the FE model for simplicity and could influence the analysis results.

Table 2 - Shear Test Spot Weld Results

| Loading Condition | Element Number (†) | Load from Analysis (lbs) | Minimum Failure Load from QW-462.10 (lbs) [3] | Maximum Plastic Strain, PEEQ [7] (%) |
|-------------------|--------------------|--------------------------|---|--------------------------------------|
| Tension | 4212 | 4713 | 5065 | 40 |

† See ABAQUS computer runs in Appendix B.

The load calculation in Table 2 is based on a **minimum** spot weld of 0.3182" diameter as required in the paragraph QW-462.10 of the ASME Code, Section IX [3]. No shear tests were performed. The difference in the failure load obtained in the analysis and the Section IX requirements can be explained from the lack of correct stress strain data for the plate material. However, the calculated load from the analysis gives good confidence in the finite element model to predict the failure mode for the shear test.

7.0 BURST TEST AND SHEAR TEST SIMILARITY

Due to the significant differences in the nature of the two tests, the similarity can be shown only qualitatively. Stress and strain magnitude and orientation are highly dependent on the geometry, type of loading, and the boundary conditions and as such a numerical comparison is difficult.

However, some strain comparisons can be made at the elements which experience large deformation to determine the failure mode, that is, shear, tension, or compression. Relative principal strains will be calculated at integration points of those elements which experience large deformations. It is expected that these relative strain values will exhibit some dominant failure mode.

Tables 3 and 4 give the principal strain values (Lep's) in the order of their magnitudes at all integration points for the elements (at the resistance weld interface) which undergo large deformation during the two tests. From the principal strains, relative principal strain values are calculated by dividing the principal strains by magnitude of total strain to assess the state of strain. The magnitude of total strain is calculated as the square root of the sum of the squares (vector sum) of Lep's.

$$\text{Magnitude of total strain} = \sqrt{\text{Lep1}^2 + \text{Lep2}^2 + \text{Lep3}^2}$$

Table 3 - State of Strain for Burst Test

| Minimum Principal Logarithmic Strain (Lep1) | | | | |
|---|-------------------|-------------------|-------------------|-------------------|
| Element Number | Integration Pt. 1 | Integration Pt. 2 | Integration Pt. 3 | Integration Pt. 4 |
| 110 | -0.017400 | -0.017400 | -0.017400 | -0.017400 |
| 170 | -0.083479 | -0.149700 | -0.135200 | -0.178800 |
| 230 | -0.194300 | -0.180500 | -0.131900 | -0.115300 |
| 290 | -0.100200 | -0.093427 | -0.083689 | -0.076060 |
| 350 | -0.065746 | -0.073466 | -0.056698 | -0.065750 |
| 410 | -0.054394 | -0.056830 | -0.053466 | -0.055958 |
| 470 | -0.041398 | -0.045943 | -0.046715 | -0.050855 |
| 530 | -0.045778 | -0.047600 | -0.049083 | -0.050793 |
| 590 | -0.063225 | -0.064260 | -0.073455 | -0.074369 |
| 650 | -0.096402 | -0.096967 | -0.105400 | -0.105900 |
| 2201 | -0.054180 | -0.046014 | -0.082483 | -0.077418 |
| 2241 | -0.157500 | -0.098970 | -0.156700 | -0.105100 |
| 2281 | -0.053553 | -0.059263 | -0.034724 | -0.043231 |
| 2321 | -0.018291 | -0.020902 | -0.018383 | -0.020996 |
| 2361 | -0.030386 | -0.031106 | -0.030146 | -0.030872 |
| 2401 | -0.024540 | -0.029561 | -0.030103 | -0.034428 |
| 2441 | -0.026610 | -0.027610 | -0.030506 | -0.031403 |
| 2481 | -0.038367 | -0.037794 | -0.046703 | -0.046252 |
| 2521 | -0.063511 | -0.063657 | -0.069488 | -0.069620 |
| 2561 | -0.092585 | -0.091575 | -0.104800 | -0.103900 |

Table 3 - State of Strain for Burst Test (cont'd)

| Intermediate Principal Logarithmic Strain (Lep2) | | | | |
|--|-------------------|-------------------|-------------------|-------------------|
| Element Number | Integration Pt. 1 | Integration Pt. 2 | Integration Pt. 3 | Integration Pt. 4 |
| 110 | 0.003976 | -0.001576 | -0.003454 | -0.006529 |
| 170 | -0.009834 | -0.009834 | -0.009834 | -0.009834 |
| 230 | -0.003830 | -0.003830 | -0.003830 | -0.003830 |
| 290 | -0.002123 | -0.002123 | -0.002123 | -0.002123 |
| 350 | -0.001349 | -0.001349 | -0.001349 | -0.001349 |
| 410 | -0.000506 | -0.000506 | -0.000506 | -0.000506 |
| 470 | 0.000763 | 0.000763 | 0.000763 | 0.000763 |
| 530 | 0.002796 | 0.002796 | 0.002796 | 0.002796 |
| 590 | 0.005421 | 0.005421 | 0.005421 | 0.005421 |
| 650 | 0.008425 | 0.008425 | 0.008425 | 0.008425 |
| 2201 | 0.000568 | 0.000568 | 0.000568 | 0.000568 |
| 2241 | -0.001266 | -0.001266 | -0.001266 | -0.001266 |
| 2281 | -0.002633 | -0.002633 | -0.002633 | -0.002633 |
| 2321 | -0.001894 | -0.001894 | -0.001894 | -0.001894 |
| 2361 | -0.001269 | -0.001269 | -0.001269 | -0.001269 |
| 2401 | -0.000772 | -0.000772 | -0.000772 | -0.000772 |
| 2441 | 0.000198 | 0.000198 | 0.000198 | 0.000198 |
| 2481 | 0.001834 | 0.001834 | 0.001834 | 0.001834 |
| 2521 | 0.004015 | 0.004015 | 0.004015 | 0.004015 |
| 2561 | 0.006426 | 0.006426 | 0.006426 | 0.006426 |

Table 3 - State of Strain for Burst Test (cont'd)

| Maximum Principal Logarithmic Strain (Lep3) | | | | |
|---|-------------------|-------------------|-------------------|-------------------|
| Element Number | Integration Pt. 1 | Integration Pt. 2 | Integration Pt. 3 | Integration Pt. 4 |
| 110 | 0.012420 | 0.017972 | 0.019850 | 0.022925 |
| 170 | 0.095502 | 0.161700 | 0.147200 | 0.190800 |
| 230 | 0.201600 | 0.187800 | 0.139300 | 0.122700 |
| 290 | 0.104600 | 0.097852 | 0.088113 | 0.080484 |
| 350 | 0.069177 | 0.076897 | 0.060129 | 0.069182 |
| 410 | 0.056315 | 0.058751 | 0.055386 | 0.057878 |
| 470 | 0.041827 | 0.046372 | 0.047144 | 0.051284 |
| 530 | 0.043175 | 0.044997 | 0.046479 | 0.048190 |
| 590 | 0.056767 | 0.057803 | 0.066998 | 0.067911 |
| 650 | 0.086228 | 0.086793 | 0.095230 | 0.095753 |
| 2201 | 0.054429 | 0.046263 | 0.082732 | 0.077667 |
| 2241 | 0.162200 | 0.103700 | 0.161400 | 0.109900 |
| 2281 | 0.060521 | 0.066231 | 0.041692 | 0.050199 |
| 2321 | 0.023027 | 0.025638 | 0.023119 | 0.025732 |
| 2361 | 0.033845 | 0.034564 | 0.033605 | 0.034331 |
| 2401 | 0.026994 | 0.032014 | 0.032556 | 0.036881 |
| 2441 | 0.027570 | 0.028570 | 0.031466 | 0.032363 |
| 2481 | 0.036309 | 0.035736 | 0.044645 | 0.044194 |
| 2521 | 0.058143 | 0.058289 | 0.064120 | 0.064252 |
| 2561 | 0.084419 | 0.083409 | 0.096591 | 0.095696 |

Table 3 - State of Strain for Burst Test (cont'd)

| Magnitude of total Strain | | | | |
|---------------------------|-------------------|-------------------|-------------------|-------------------|
| Element Number | Integration Pt. 1 | Integration Pt. 2 | Integration Pt. 3 | Integration Pt. 4 |
| 110 | 0.021745 | 0.025065 | 0.026622 | 0.029512 |
| 170 | 0.127225 | 0.220576 | 0.200109 | 0.261669 |
| 230 | 0.280017 | 0.260507 | 0.191877 | 0.168416 |
| 290 | 0.144864 | 0.135308 | 0.121541 | 0.110758 |
| 350 | 0.095445 | 0.106359 | 0.082656 | 0.095452 |
| 410 | 0.078297 | 0.081741 | 0.076984 | 0.080507 |
| 470 | 0.058855 | 0.065282 | 0.066373 | 0.072228 |
| 530 | 0.062988 | 0.065561 | 0.067655 | 0.070072 |
| 590 | 0.085143 | 0.086602 | 0.099568 | 0.100857 |
| 650 | 0.129613 | 0.130409 | 0.142299 | 0.143019 |
| 2201 | 0.076800 | 0.065252 | 0.116826 | 0.109663 |
| 2241 | 0.226090 | 0.143354 | 0.224959 | 0.152071 |
| 2281 | 0.080856 | 0.088913 | 0.054322 | 0.066301 |
| 2321 | 0.029468 | 0.033133 | 0.029597 | 0.033265 |
| 2361 | 0.045502 | 0.046517 | 0.045163 | 0.046188 |
| 2401 | 0.036489 | 0.043581 | 0.044347 | 0.050459 |
| 2441 | 0.038318 | 0.039732 | 0.043827 | 0.045095 |
| 2481 | 0.052856 | 0.052046 | 0.064635 | 0.063998 |
| 2521 | 0.086200 | 0.086406 | 0.094637 | 0.094823 |
| 2561 | 0.125459 | 0.124034 | 0.142668 | 0.141401 |
| | | | | |

Table 3 - State of Strain for Burst Test (cont'd)

| State of Strain (%) | | | | | | |
|---------------------|-------------------|------------------|-------------|-------------------|------------------|-------------|
| Element Number | Integration Pt. 1 | | | Integration Pt. 2 | | |
| | Minimum (%) | Intermediate (%) | Maximum (%) | Minimum (%) | Intermediate (%) | Maximum (%) |
| 110 | -80.02 | 18.28 | 57.12 | -69.42 | -6.29 | 71.70 |
| 170 | -65.62 | -7.73 | 75.07 | -67.87 | -4.46 | 73.31 |
| 230 | -69.39 | -1.37 | 72.00 | -69.29 | -1.47 | 72.09 |
| 290 | -69.17 | -1.47 | 72.21 | -69.05 | -1.57 | 72.32 |
| 350 | -68.88 | -1.41 | 72.48 | -69.07 | -1.27 | 72.30 |
| 410 | -69.47 | -0.65 | 71.93 | -69.52 | -0.62 | 71.87 |
| 470 | -70.34 | 1.30 | 71.07 | -70.38 | 1.17 | 71.03 |
| 530 | -72.68 | 4.44 | 68.54 | -72.60 | 4.26 | 68.63 |
| 590 | -74.26 | 6.37 | 66.67 | -74.20 | 6.26 | 66.75 |
| 650 | -74.38 | 6.50 | 66.53 | -74.36 | 6.46 | 66.55 |
| 2201 | -70.55 | 0.74 | 70.87 | -70.52 | 0.87 | 70.90 |
| 2241 | -69.66 | -0.56 | 71.74 | -69.04 | -0.88 | 72.34 |
| 2281 | -66.23 | -3.26 | 74.85 | -66.65 | -2.96 | 74.49 |
| 2321 | -62.07 | -6.43 | 78.14 | -63.09 | -5.72 | 77.38 |
| 2361 | -66.78 | -2.79 | 74.38 | -66.87 | -2.73 | 74.30 |
| 2401 | -67.25 | -2.11 | 73.98 | -67.83 | -1.77 | 73.46 |
| 2441 | -69.45 | 0.52 | 71.95 | -69.49 | 0.50 | 71.91 |
| 2481 | -72.59 | 3.47 | 68.69 | -72.62 | 3.52 | 68.66 |
| 2521 | -73.68 | 4.66 | 67.45 | -73.67 | 4.65 | 67.46 |
| 2561 | -73.80 | 5.12 | 67.29 | -73.83 | 5.18 | 67.25 |

Table 3 - State of Strain for Burst Test (cont'd)

| State of Strain (%) | | | | | | |
|---------------------|-------------------|------------------|-------------|-------------------|------------------|-------------|
| Element Number | Integration Pt. 3 | | | Integration Pt. 4 | | |
| | Minimum (%) | Intermediate (%) | Maximum (%) | Minimum (%) | Intermediate (%) | Maximum (%) |
| 110 | -65.36 | -12.97 | 74.56 | -58.96 | -22.12 | 77.68 |
| 170 | -67.56 | -4.91 | 73.56 | -68.33 | -3.76 | 72.92 |
| 230 | -68.74 | -2.00 | 72.60 | -68.46 | -2.27 | 72.86 |
| 290 | -68.86 | -1.75 | 72.50 | -68.67 | -1.92 | 72.67 |
| 350 | -68.60 | -1.63 | 72.75 | -68.88 | -1.41 | 72.48 |
| 410 | -69.45 | -0.66 | 71.95 | -69.51 | -0.63 | 71.89 |
| 470 | -70.38 | 1.15 | 71.03 | -70.41 | 1.06 | 71.00 |
| 530 | -72.55 | 4.13 | 68.70 | -72.49 | 3.99 | 68.77 |
| 590 | -73.77 | 5.44 | 67.29 | -73.74 | 5.37 | 67.33 |
| 650 | -74.07 | 5.92 | 66.92 | -74.05 | 5.89 | 66.95 |
| 2201 | -70.60 | 0.49 | 70.82 | -70.60 | 0.52 | 70.82 |
| 2241 | -69.66 | -0.56 | 71.75 | -69.11 | -0.83 | 72.27 |
| 2281 | -63.92 | -4.85 | 76.75 | -65.20 | -3.97 | 75.71 |
| 2321 | -62.11 | -6.40 | 78.11 | -63.12 | -5.69 | 77.35 |
| 2361 | -66.75 | -2.81 | 74.41 | -66.84 | -2.75 | 74.33 |
| 2401 | -67.88 | -1.74 | 73.41 | -68.23 | -1.53 | 73.09 |
| 2441 | -69.61 | 0.45 | 71.80 | -69.64 | 0.44 | 71.77 |
| 2481 | -72.26 | 2.84 | 69.07 | -72.27 | 2.87 | 69.06 |
| 2521 | -73.43 | 4.24 | 67.75 | -73.42 | 4.23 | 67.76 |
| 2561 | -73.46 | 4.50 | 67.70 | -73.48 | 4.54 | 67.68 |
| | | | | | | |

Table 4 - State of Strain for Shear Test

| Element Number (†) | Minimum Principal Log Strain Lep1 | Intermediate Principal Log Strain Lep2 | Maximum Principal Log Strain Lep3 | Magnitude of Total Strain | State of Strain | | |
|-----------------------|--------------------------------------|---|--------------------------------------|---------------------------|-----------------|------------------|-------------|
| | | | | | Minimum (%) | Intermediate (%) | Maximum (%) |
| 1488 | -0.0945 | -0.0109 | 0.1070 | 0.1432 | -66.02 | -7.61 | 74.72 |
| 1489 | -0.0938 | -0.0031 | 0.0981 | 0.1358 | -69.10 | -2.25 | 72.25 |
| 1490 | -0.0800 | -0.0004 | 0.0805 | 0.1135 | -70.50 | -0.34 | 70.92 |
| 1491 | -0.1016 | -0.0003 | 0.1029 | 0.1446 | -70.26 | -0.24 | 71.16 |
| 1492 | -0.1072 | -0.0015 | 0.1080 | 0.1522 | -70.44 | -1.00 | 70.97 |
| 1788 | -0.1013 | 0.0089 | 0.0944 | 0.1387 | -73.01 | 6.41 | 68.03 |
| 1789 | -0.0877 | 0.0049 | 0.0819 | 0.1201 | -73.02 | 4.09 | 68.20 |
| 1790 | -0.0634 | 0.0031 | 0.0598 | 0.0872 | -72.71 | 3.53 | 68.56 |
| 1791 | -0.0947 | 0.0058 | 0.0882 | 0.1295 | -73.07 | 4.51 | 68.12 |
| 1792 | -0.1094 | 0.0082 | 0.1010 | 0.1491 | -73.36 | 5.49 | 67.73 |
| 2088 | -0.0653 | -0.0081 | 0.0749 | 0.0997 | -65.48 | -8.17 | 75.14 |
| 2089 | -0.0576 | -0.0026 | 0.0614 | 0.0843 | -68.36 | -3.08 | 72.92 |
| 2090 | -0.0479 | -0.0021 | 0.0499 | 0.0692 | -69.21 | -3.01 | 72.12 |
| 2091 | -0.0689 | -0.0036 | 0.0735 | 0.1008 | -68.34 | -3.58 | 72.91 |
| 2092 | -0.0744 | -0.0050 | 0.0785 | 0.1083 | -68.72 | -4.63 | 72.50 |
| 4208 | -0.2989 | -0.0154 | 0.3171 | 0.4360 | -68.55 | -3.54 | 72.72 |
| 4209 | -0.1945 | -0.0048 | 0.1993 | 0.2785 | -69.83 | -1.72 | 71.56 |
| 4210 | -0.1680 | -0.0004 | 0.1692 | 0.2384 | -70.46 | -0.16 | 70.96 |
| 4211 | -0.1859 | -0.0071 | 0.1933 | 0.2683 | -69.29 | -2.66 | 72.05 |
| 4212 | -0.3020 | -0.0168 | 0.3224 | 0.4421 | -68.31 | -3.80 | 72.93 |
| 4508 | -0.1784 | 0.0148 | 0.1637 | 0.2426 | -73.54 | 6.11 | 67.48 |
| 4509 | -0.0577 | 0.0065 | 0.0515 | 0.0776 | -74.33 | 8.32 | 66.37 |
| 4510 | -0.0310 | 0.0014 | 0.0289 | 0.0424 | -73.08 | 3.22 | 68.18 |
| 4511 | -0.0597 | 0.0094 | 0.0507 | 0.0789 | -75.68 | 11.87 | 64.27 |
| 4512 | -0.1924 | 0.0195 | 0.1736 | 0.2599 | -74.04 | 7.50 | 66.80 |
| 4808 | -0.2064 | -0.0128 | 0.2217 | 0.3032 | -68.08 | -4.24 | 73.13 |
| 4809 | -0.1300 | -0.0041 | 0.1338 | 0.1866 | -69.67 | -2.22 | 71.70 |
| 4810 | -0.1075 | -0.0009 | 0.1091 | 0.1532 | -70.19 | -0.56 | 71.23 |

| | | | | | | | |
|------|---------|---------|--------|--------|--------|-------|-------|
| 4811 | -0.1194 | -0.0075 | 0.1269 | 0.1744 | -68.46 | -4.32 | 72.76 |
| 4812 | -0.2030 | -0.0176 | 0.2242 | 0.3030 | -67.01 | -5.82 | 74.00 |

† These are the elements which form the weld spot. Element 4212 experiences the maximum total strain.

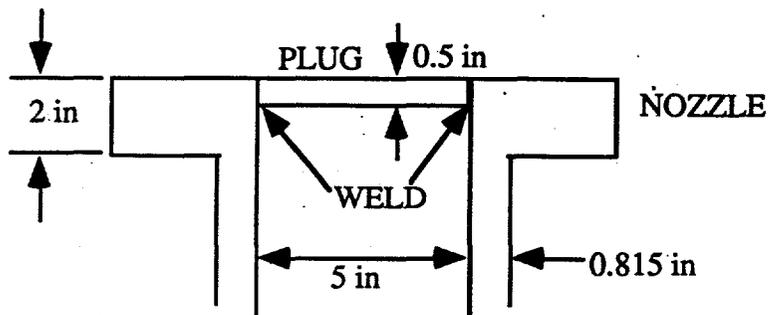
A review of the strain values in the state of strain columns shows that minimum and maximum strains are approximately equal and opposite in magnitude. The third strain is negligible in magnitude as compared to the other two strains. A review of the Mohr circle shows that the shear strain is maximum when the principal strains are approximately equal and opposite in magnitude. This state of strain clearly indicates that a shear dominant failure mode exists. This trend exists in both the tests and, therefore, it can be concluded that the failure mode in both the tests is shear. On this basis, the two tests are similar in nature.

8.0 CONCLUSIONS

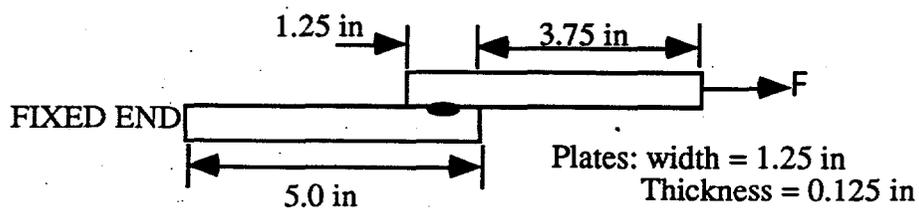
- 1) Based on the detailed structural analyses of the burst test for the canister-plug weld and the shear test as required in the ASME Code, Section IX, QW-196 [3], it is found that the dominant failure mode for the two tests is shear. On this basis, the use of burst test for the canister weld in lieu of shear test is acceptable.
- 2) Based on the realistic material properties of 304L stainless steel and the weld metal, it is found that the canister-plug weld is structurally sound and can safely withstand 2600 psig of internal pressure.

9.0 REFERENCES

- [1] Design Drawings:
DWPF - Canister Assembly - W832094 Rev. B1
Weld Plug Details - D199208 Rev. B1
- [2] ASME Boiler and Pressure Vessel Code, Sections VIII, Division 1, Edition 1986.
- [3] ASME Boiler and Pressure Vessel Code, Sections IX, Edition 1992.
- [4] SRT-MTS-93-3113, Tensile Properties of Type 304/304L Stainless Steel for Impact Deformation Analysis of Nuclear Materials Containers (U), R. L. Sindelar, Westinghouse Savannah River Co., Aiken, SC, November 1993.
- [5] WSRC-TR-94-002, Characterization of DWPF Canister Closure Weld Microstructure (U), S. L. West and W. R. Kanne, Jr., Westinghouse Savannah River Co., Aiken, SC, January 1994.
- [6] Kanne Jr., W. R., Solid State Resistance Welding of Cylinders and Spheres, *Welding Journal*, May 1986 pp 33-38.
- [7] ABAQUS, Version 5-3, Hibbitt, Karlsson, & Sorensen, Inc., Pawtucket, Rhode Island, 1994.
- [8] SRT-SPS-940066, Qualification of Resistance Welding Processes in the Defense Waste Processing Facility (U), J. A. Morin, Westinghouse Savannah River Co., Aiken, SC, May 1994.



CANISTER-PLUG WELD



ASME SECTION IX SHEAR TEST

FIGURE 1 - Canister-Plug Weld and Shear Test Geometry

ABAQUS

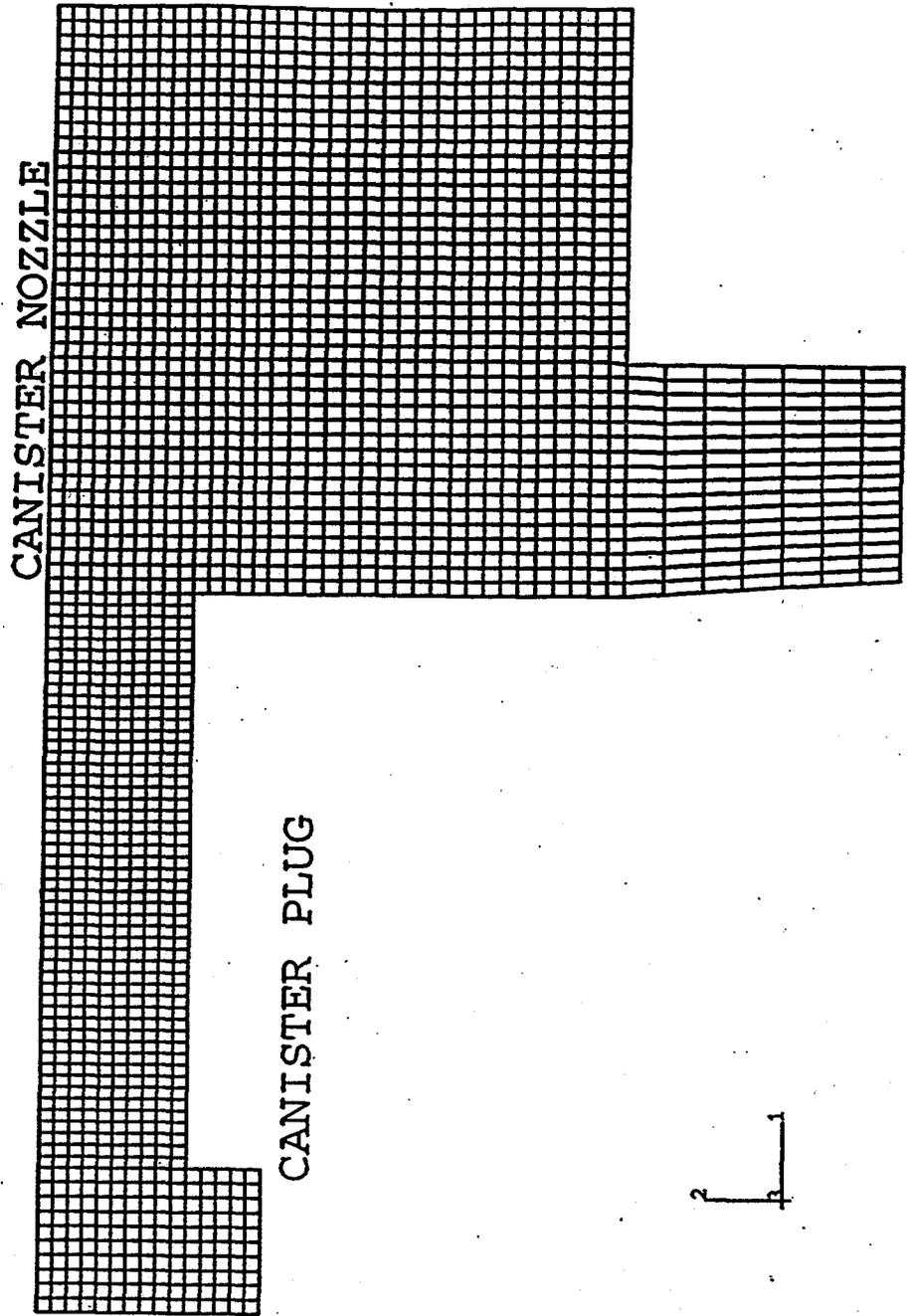


FIGURE 2 - Finite Element Mesh of the Canister-Plug Weld

ABAQUS

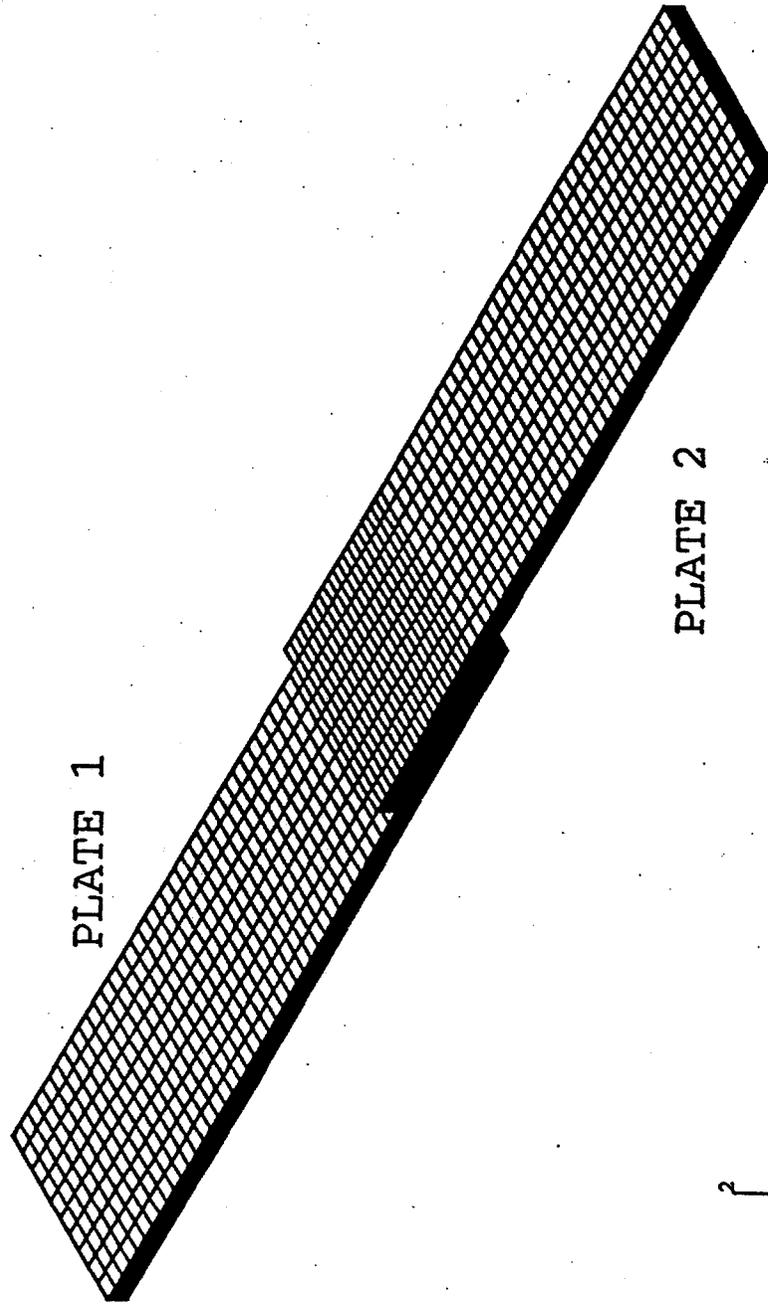


FIGURE 3 - Finite Element Mesh of the Shear Test Weld

ABAQUS

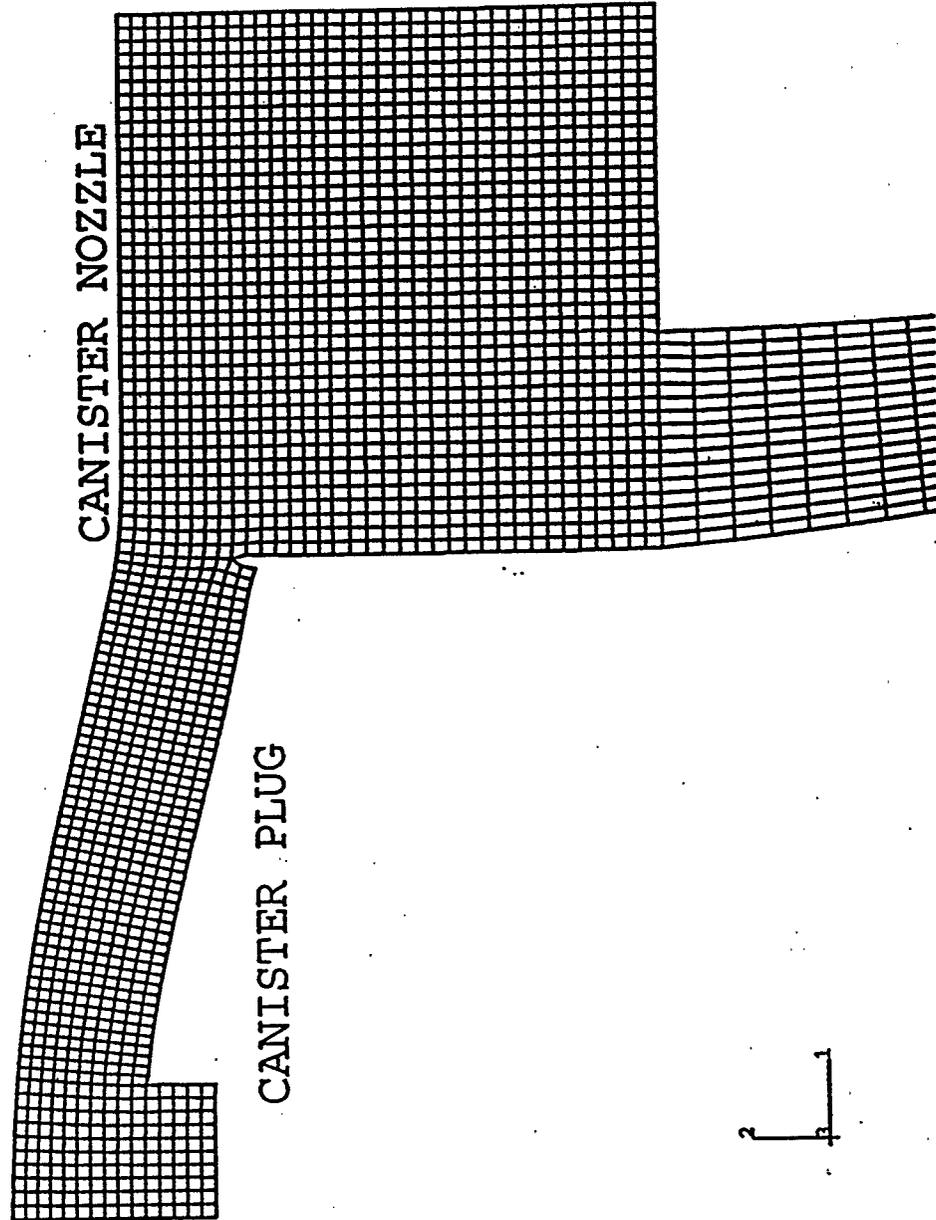


FIGURE 4 - Deformed Shape of Canister-Plug Weld
(Magnification Factor = 1)

ABAQUS

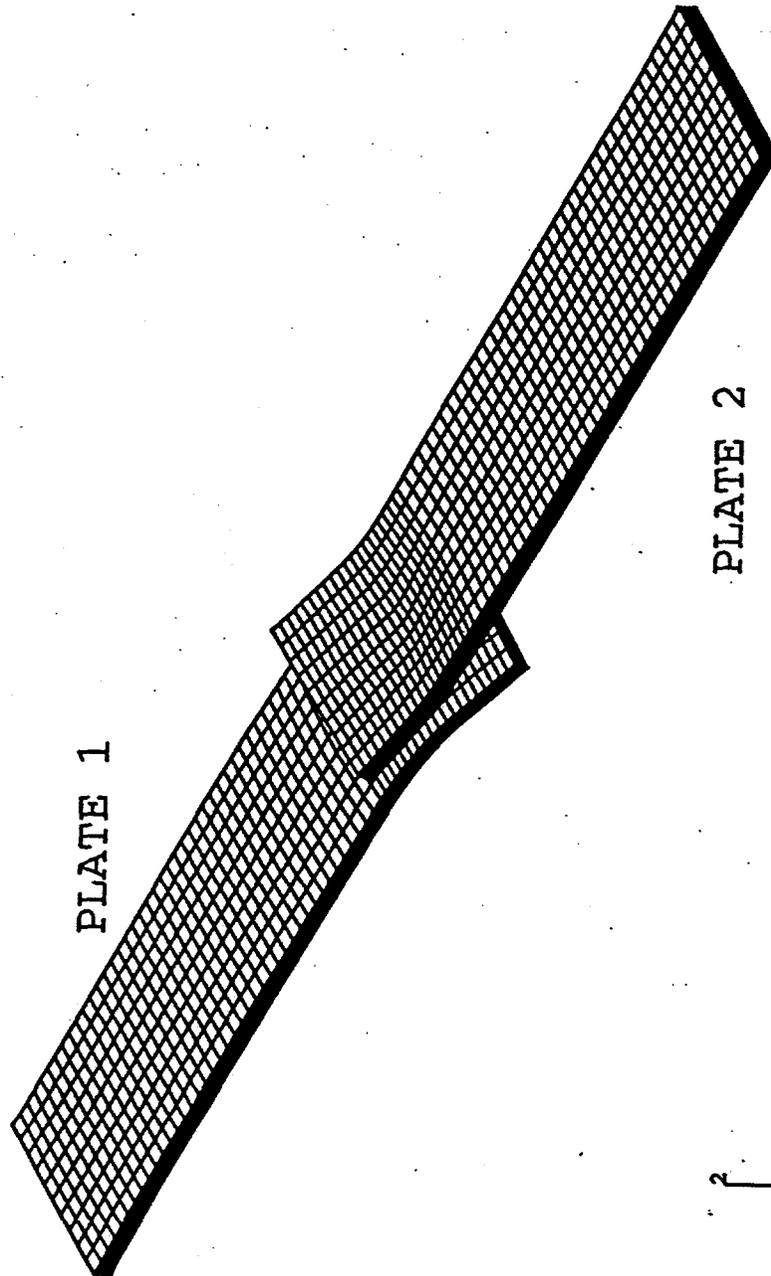


FIGURE 5 - Deformed Shape of the Shear Test Weld
(Magnification Factor = 1)

Appendix A

Stress Strain (True) Data for SS304L and Weld Metal

| Stress-Strain Data for SS304L | | Stress-Strain Data for 304L Weld Metal | |
|-------------------------------|----------------|--|----------------|
| Stress (psi) | Plastic Strain | Stress (psi) | Plastic Strain |
| 37840 | 0.0000 | 40754 | 0.0000 |
| 39710 | 0.0014 | 42768 | 0.0014 |
| 41301 | 0.0029 | 44481 | 0.0029 |
| 42332 | 0.0046 | 45592 | 0.0046 |
| 43157 | 0.0062 | 46480 | 0.0062 |
| 43848 | 0.0079 | 47224 | 0.0079 |
| 44614 | 0.0095 | 48049 | 0.0095 |
| 45176 | 0.0111 | 48655 | 0.0111 |
| 45586 | 0.0127 | 49096 | 0.0127 |
| 46291 | 0.0143 | 49855 | 0.0143 |
| 46720 | 0.0162 | 50317 | 0.0162 |
| 47650 | 0.0192 | 51319 | 0.0192 |
| 48644 | 0.0222 | 52390 | 0.0222 |
| 49451 | 0.0253 | 53259 | 0.0253 |
| 50533 | 0.0284 | 54424 | 0.0284 |
| 51316 | 0.0313 | 55267 | 0.0313 |
| 51814 | 0.0343 | 55804 | 0.0343 |
| 52882 | 0.0373 | 56954 | 0.0373 |
| 53592 | 0.0404 | 57719 | 0.0404 |
| 55850 | 0.0490 | 60150 | 0.0490 |
| 58275 | 0.0573 | 62762 | 0.0573 |
| 60387 | 0.0654 | 65037 | 0.0654 |
| 62375 | 0.0735 | 67178 | 0.0735 |
| 64570 | 0.0816 | 69542 | 0.0816 |
| 66366 | 0.0895 | 71476 | 0.0895 |
| 68384 | 0.0974 | 73650 | 0.0974 |
| 70116 | 0.1053 | 75515 | 0.1053 |
| 71568 | 0.1131 | 77079 | 0.1131 |
| 73404 | 0.1208 | 79056 | 0.1208 |
| 75421 | 0.1285 | 81228 | 0.1285 |
| 77335 | 0.1362 | 83290 | 0.1362 |
| 78646 | 0.1438 | 84702 | 0.1438 |
| 80354 | 0.1513 | 86541 | 0.1513 |
| 81762 | 0.1587 | 88058 | 0.1587 |
| 83521 | 0.1662 | 89952 | 0.1662 |
| 85116 | 0.1736 | 91670 | 0.1736 |
| 86583 | 0.1810 | 93250 | 0.1810 |
| 88233 | 0.1882 | 95027 | 0.1882 |
| 89525 | 0.1955 | 96418 | 0.1955 |

| | | | |
|--------|--------|--------|--------|
| 91036 | 0.2027 | 98046 | 0.2027 |
| 92629 | 0.2098 | 99761 | 0.2098 |
| 94249 | 0.2168 | 101506 | 0.2168 |
| 95515 | 0.2238 | 102870 | 0.2238 |
| 96607 | 0.2308 | 104046 | 0.2308 |
| 98231 | 0.2377 | 105795 | 0.2377 |
| 99625 | 0.2446 | 107296 | 0.2446 |
| 101206 | 0.2514 | 108999 | 0.2514 |
| 102325 | 0.2582 | 110204 | 0.2582 |
| 103756 | 0.2650 | 111745 | 0.2650 |
| 105025 | 0.2717 | 113112 | 0.2717 |
| 106120 | 0.2784 | 114291 | 0.2784 |
| 107976 | 0.2852 | 116290 | 0.2852 |
| 108557 | 0.2918 | 116916 | 0.2918 |
| 109886 | 0.2984 | 118347 | 0.2984 |
| 111634 | 0.3051 | 120230 | 0.3051 |
| 111005 | 0.3117 | 119552 | 0.3117 |
| 114262 | 0.3183 | 123060 | 0.3183 |
| 115154 | 0.3248 | 124021 | 0.3248 |
| 116689 | 0.3314 | 125674 | 0.3314 |
| 117520 | 0.3379 | 126569 | 0.3379 |
| 118480 | 0.3445 | 127603 | 0.3445 |
| 120036 | 0.3510 | 129279 | 0.3510 |
| 121236 | 0.3575 | 130571 | 0.3575 |
| 122230 | 0.3640 | 131642 | 0.3640 |
| 123151 | 0.3697 | 132634 | 0.3697 |
| 123702 | 0.3719 | 133227 | 0.3719 |
| 124088 | 0.3741 | 133643 | 0.3741 |
| 124428 | 0.3763 | 134009 | 0.3763 |
| 124797 | 0.3783 | 134406 | 0.3783 |
| 125510 | 0.3806 | 135174 | 0.3806 |
| 125973 | 0.3828 | 135673 | 0.3828 |
| 126296 | 0.3849 | 136021 | 0.3849 |
| 126573 | 0.3871 | 136319 | 0.3871 |
| 126868 | 0.3893 | 136637 | 0.3893 |
| 127427 | 0.3914 | 137239 | 0.3914 |
| 127988 | 0.3936 | 137843 | 0.3936 |
| 128124 | 0.3957 | 137990 | 0.3957 |
| 128410 | 0.3979 | 138298 | 0.3979 |
| 128496 | 0.4001 | 138390 | 0.4001 |
| 129202 | 0.4023 | 139151 | 0.4023 |
| 129594 | 0.4045 | 139573 | 0.4045 |
| 129804 | 0.4067 | 139799 | 0.4067 |
| 130445 | 0.4089 | 140489 | 0.4089 |
| 131070 | 0.4111 | 141162 | 0.4111 |
| 131165 | 0.4133 | 141265 | 0.4133 |

| | | | |
|--------|--------|--------|--------|
| 131298 | 0.4155 | 141408 | 0.4155 |
| 131691 | 0.4177 | 141831 | 0.4177 |
| 132647 | 0.4200 | 142861 | 0.4200 |
| 132702 | 0.4221 | 142920 | 0.4221 |
| 132737 | 0.4243 | 142958 | 0.4243 |
| 133034 | 0.4265 | 143278 | 0.4265 |
| 134036 | 0.4288 | 144357 | 0.4288 |
| 133980 | 0.4310 | 144296 | 0.4310 |
| 134365 | 0.4333 | 144711 | 0.4333 |
| 134721 | 0.4355 | 145095 | 0.4355 |
| 135494 | 0.4377 | 145927 | 0.4377 |
| 135781 | 0.4399 | 146236 | 0.4399 |
| 136029 | 0.4422 | 146503 | 0.4422 |
| 136385 | 0.4445 | 146887 | 0.4445 |
| 136723 | 0.4467 | 147251 | 0.4467 |
| 137371 | 0.4490 | 147949 | 0.4490 |
| 137757 | 0.4513 | 148364 | 0.4513 |
| 138114 | 0.4536 | 148749 | 0.4536 |
| 138865 | 0.4559 | 149558 | 0.4559 |
| 138869 | 0.4582 | 149562 | 0.4582 |
| 139248 | 0.4605 | 149970 | 0.4605 |
| 139615 | 0.4628 | 150365 | 0.4628 |
| 140083 | 0.4652 | 150869 | 0.4652 |
| 140227 | 0.4675 | 151024 | 0.4675 |
| 140635 | 0.4698 | 151464 | 0.4698 |
| 141105 | 0.4721 | 151970 | 0.4721 |
| 141494 | 0.4745 | 152389 | 0.4745 |
| 141768 | 0.4768 | 152684 | 0.4768 |
| 142216 | 0.4793 | 153167 | 0.4793 |

Appendix B
ABAQUS Files

| Item No. | Input/Output Files | Restart (.res) File | File Description |
|----------|--------------------|---------------------|--|
| 1 | weld | w3 | Shear Test Analysis |
| 2 | "O" | "O" | Canister-Plug Weld Burst Test Analysis |

- Notes: 1. Shear test analysis files are stored in the SRS computer files storage system CFS in directory /y6203/weld.
2. Canister-Plug burst test analysis files are stored in the SRS computer files storage system CFS in directory /y6749/dwpcf-plug.